# Illinois Statewide

# Technical Reference Manual

# for Energy Efficiency

# Version 4.0

## Final

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## Acknowledgements

This document was created through collaboration amongst the members of the Illinois Energy Efficiency Stakeholder Advisory Group (SAG). The SAG is an open forum where interested parties may participate in the evolution of Illinois' energy efficiency programs. Parties wishing to participate in the SAG process may do so by visiting http://www.ilsag.info/questions.html and contacting the Independent Facilitator at Annette.Beitel@FutEE.biz. Parties wishing to participate in the Technical Advisory Committee (TAC), a subcommittee of the SAG, may do so by contacting the TRM Administrator at iltrmadministrator@veic.org.

| SAG Stakeholders[1] |
|---|
| Ameren Illinois Company (Ameren) |
| Citizen's Utility Board (CUB) |
| City of Chicago |
| Commonwealth Edison Company (ComEd) |
| Elevate Energy |
| Energy Resources Center at the University of Illinois, Chicago (ERC) |
| Environment IL |
| Environmental Law and Policy Center (ELPC) |
| Future Energy Enterprises LLC |
| Illinois Attorney General's Office (AG) |
| Illinois Commerce Commission Staff (ICC Staff) |
| Illinois Department of Commerce and Economic Opportunity (DCEO) |
| Independent Evaluators (ADM, Cadmus, Itron, Navigant) |
| Integrys (Peoples Gas and North Shore Gas) |
| Metropolitan Mayor's Caucus (MMC) |
| Midwest Energy Efficiency Association (MEEA) |
| Natural Resources Defense Council (NRDC) |
| Nicor Gas |

[1] Being an open forum, this list of SAG stakeholders and participants may change at any time.

**Table 1.1: Document Revision History**

| Document Title | Applicable to PY Beginning |
|---|---|
| Illinois_Statewide_TRM_Effective_060112_Version_1.0_091412_Clean.doc | 6/1/12 |
| Illinois_Statewide_TRM_Effective_060113_Version_2.0_060713_Clean.docx | 6/1/13 |
| Illinois_Statewide_TRM_Effective_060114_Version_3.0_022414_Clean.docx | 6/1/14 |
| Illinois_Statewide_TRM_Effective_060115_Version_4.0_021315_Clean.docx | 6/1/15 |

## Summary of Measure Revisions

The following tables summarize the evolution of measures that are new, revised or errata. This version of the TRM contains 82 measure-level changes as described in the following table.

**Table 1.2: Summary of Measure Level Changes**

| Change Type | # Changes |
|---|---|
| Errata | 16 |
| Revision | 43 |
| New Measure | 23 |
| Total Changes | 82 |

The 'Change Type' column indicates what kind of change each measure has gone through. Specifically, when a measure error was identified and the TAC process resulted in a consensus, the measure is identified here as an 'Errata'. In these instances the measure code indicates that a new version of the measure has been published, and that the effective date of the measure dates back to June 1st, 2014. Measures that are identified as 'Revised' were included in the third edition of the TRM, and have been updated for this edition of the TRM. Both 'Revised' and 'New Measure(s)' have an effective date of June 1st, 2015.

The following table provides an overview of the 82 measure-level changes that are included in this version of the TRM.

**Table 1.3: Summary of Measure Revisions**

| Mkt | End Use | Measure Name | Measure Code | Change Type | Explanation | Impact on Savings |
|---|---|---|---|---|---|---|
| C&I | 4.2 Food Service Equipment | 4.2.19 ENERGY STAR Electric Convection Oven | CI-FSE-ECON-V01-150601 | New | n/a | n/a |
| C&I | 4.3 Hot Water | 4.3.2 Low Flow Faucet Aerators | CI-HWE-LFFA-V05-140601 | Errata | Fixing Reference | None |
| | | 4.3.7 Multifamily Central Domestic Hot Water Plants | CI-HW_-MDHW-V01-150601 | New | n/a | n/a |
| | | 4.3.8 Controls for Central Domestic Hot Water | CI-HW_-CDHW-V01-150601 | New | n/a | n/a |
| C&I | 4.4 HVAC | 4.4 HVAC End Use | | Errata | Added EFLH for Heat Pump Systems, PTAC/PTHP and Unitary AC | Unknown |
| | | 4.4 HVAC End Use | | Revision | Updated EFLH | Dependent on building type |
| | | 4.4.1 Air Conditioner Tune-Up | CI-HVC-ACTU-V02-150601 | Revision | Removed some tune-up requirements Removing deemed savings assumption - | Unknown |

| Mkt | End Use | Measure Name | Measure Code | Change Type | Explanation | Impact on Savings |
|---|---|---|---|---|---|---|
| | | | | | replaced with algorithm | |
| | | 4.4.2 Space Heating Boiler Tune-Up | CI-HVC-BLRT-V05-150601 | Revision | Removing deemed savings factor | Unknown |
| | | 4.4.3 Process Boiler Tune-Up | CI-HVC-PBTU-V04-150601 | Revision | Removing deemed savings factor | Unknown |
| | | 4.4.4 Boiler Lockout/Reset Controls | CI-HVC-BLRC-V03-150601 | Revision | Updated example with new EFLH | Dependent on building type |
| | | 4.4.6 Electric Chiller | CI-HVC-CHIL-V03-150601 | Revision | Updated example with new EFLH | Dependent on building type |
| | | 4.4.8 Guest Room Energy Management | CI-HVC-GREM-V05-150601 | Revision | Updated Hotel savings values based on modeling | Dependent on inputs |
| | | 4.4.9 Heat Pump Systems | CI-HVC-HPSY-V03-150601 | Revision | Updated code reference<br>Updated example with new EFLH | Dependent on building type |
| | | 4.4.10 High Efficiency Boiler | CI-HVC-BOIL-V05-150601 | Revision | Updated example with new EFLH | Dependent on building type |
| | | 4.4.11 High Efficiency Furnace | CI-HVC-FRNC-V04-150601 | Revision | Updated example with new EFLH | Dependent on building type |
| | | 4.4.13 Package Terminal Air Conditioner and Package Terminal Heat Pump | CI-HVC-PTAC-V05-140601 | Errata | Retrofit cost information updated | None |
| | | 4.4.13 Package Terminal Air Conditioner and Package Terminal Heat Pump | CI-HVC-PTAC-V05-150601 | Revision | Updated example with new EFLH<br>Added default early replacement costs | Dependent on building type |
| | | 4.4.14 Pipe Insulation | CI-HVC-PINS-V03-150601 | Revision | Clarifications of algorithm<br>Model results updated with new EFLH | Dependent on building type |
| | | 4.4.15 Single-Package and Split System Unitary Air Conditioning | CI-HVC-SPUA-V03-150601 | Revision | Updated code reference<br>Updated example with new EFLH | Dependent on building type |
| | | 4.4.17 Variable Speed Drives for HVAC Pumps and Cooling Tower Fans | CI-HVC-VSDHP-V02-150601 | Revision | Seperated Supply / Return fans in to separate measure.<br>Updated Hours table | Dependent on building type |
| | | 4.4.18 Small Commercial Programmable Thermostats | CI-HVC-PROG-V02-150601 | Revision | Costs clarified.<br>Complete re-work of savings estimate | Unknown |

| Mkt | End Use | Measure Name | Measure Code | Change Type | Explanation | Impact on Savings |
|---|---|---|---|---|---|---|
| | | 4.4.19 Demand Control Ventilation | CI-HVC-DCV-V02-140601 | Errata | Therm_Saving_Factor and Elec_Saving_Factor updated with final values | Unknown |
| | | 4.4.24 Small Pipe Insulation | CI-HVC-SPIN-V01-150601 | New | n/a | n/a |
| | | 4.4.25 Small Programmable Thermostat Adjustments | CI-HVC- PRGA -V01-150601 | New | n/a | n/a |
| | | 4.4.26 Variable Speed Drives for HVAC Supply and Return Fans | CI-HVC-VSDF-V01-150601 | New | n/a | n/a |
| | | 4.4.27 Energy Recovery Ventilator | CI-HVC-ERVE-V01-150601 | New | n/a | n/a |
| | | 4.4.28 Stack Economizer for Boilers Serving HVAC Loads | CI-HVC-BECO-V01-150601 | New | n/a | n/a |
| | | 4.4.29 Stack Economizer for Boilers Serving Process Loads | CI-HVC-PECO-V01-150601 | New | n/a | n/a |
| | | 4.4.30 Notched V Belts for HVAC Systems | CI-HVC-NVBE-V01-150601 | New | n/a | n/a |
| | | 4.4.31 Small Business Furnace Tune Up | CI-HVC-FTUN-V01-150601 | New | n/a | n/a |
| | | 4.4.32 Combined Heat and Power | CI-HVC-CHAP-V01-150601 | New | n/a | n/a |
| C&I | 4.5 Lighting | 4.5 Lighting | | Revision | Update of WHF to match new models | Dependent on building type |
| | | 4.5.1 Commercial ENERGY STAR Compact Fluorescent | CI-LTG-CCFL-V04-140601 | Errata | Update to RES v C&I Split | Increase in kWh savings (more C&I) |
| | | 4.5.1 Commercial ENERGY STAR Compact Fluorescent | CI-LTG-CCFL-V05-150601 | Revision | Update to RES v C&I Split, ISR | Increase in kWh savings (more C&I) |
| | | 4.5.3 High Performance and Reduced Wattage T8 Fixtures | CI-LTG-T8FX-V03-140601 | Errata | Update to RES v C&I Split | Increase in kWh savings (more C&I) |
| | | 4.5.3 High Performance and Reduced Wattage T8 Fixtures | CI-LTG-T8FX-V04-150601 | Revision | Clarification of Direct Install assumptions. Measure life clarification Update to ISR | Increase in kWh savings (higher ISR) |
| | | 4.5.4 LED Bulbs and Fixtures | CI-LTG-LEDB-V03-140601 | Errata | Update to RES v C&I Split | Increase in kWh savings |

| Mkt | End Use | Measure Name | Measure Code | Change Type | Explanation | Impact on Savings |
|---|---|---|---|---|---|---|
| | | | | | | (more C&I) |
| | | 4.5.4 LED Bulbs and Fixtures | CI-LTG-LEDB-V04-150601 | Revision | Update to RES v C&I Split | Increase in kWh savings (more C&I) |
| | | 4.5.14 Commercial ENERGY STAR Specialty Compact Fluorescent Lamp | CI-LTG-SCFL-V01-150601 | New | n/a | n/a |
| C&I | 4.6 Refrigeration | 4.6.6 Evaporator Fan Control | CI-RFG-EVPF-V02-140601 | Errata | Replacing CA estimates with Illinois | Approximately the same |
| | | 4.6.9 Night Covers for Open Refrigerated Display Cases | CI-RFG-NCOV-V01-150601 | New | n/a | n/a |
| C&I | 4.7 Miscellaneous | 4.7.4 Pump Optimization | CI-MSC-PMPO-V01-150601 | New | n/a | n/a |
| | | 4.7.5 Efficient Compressed Air Nozzles | CI-MSC-CNOZ-V01-150601 | New | n/a | n/a |
| | | 4.7.6 Roof Insulation for C&I Facilities | CI-MSC-RINS-V01-150601 | New | n/a | n/a |
| | | 4.7.7 Computer Power Management Software | CI-MSC-CPMS-V01-150601 | New | n/a | n/a |
| Res | 5.1 Appliances | 5.1.2 ENERGY STAR and ENERGY STAR Most Efficient Clothes Washer | RS-APL-ESCL-V03-150601 | Revision | Updatated methodology based on new Federal and ESTAR specifications | Reduction in savings |
| | | 5.1.8 Refrigerator and Freezer Recycling | RS-APL-RFRC-V04-140601 | Errata | Fixing typo in Freezer coefficient table. | Unknown |
| | | 5.1.8 Refrigerator and Freezer Recycling | RS-APL-RFRC-V05-150601 | Revision | Updated coefficient values | Dependent on inputs |
| | | 5.1.10 Residential ENERGY STAR Clothes Dryer | RS-APL-ESDR-V01-150601 | New | n/a | n/a |
| Res | 5.3 HVAC | 5.3.1 Air Source Heat Pump | RS-HVC-ASHP-V04-150601 | Revision | Added deemed early replacement rate. Updated Federal Standard and costs | Reduction in savings |
| | | 5.3.3 Central Air Conditioning | RS-HVC-CAC1-V04-150601 | Revision | Added deemed early replacement rate. | None |
| | | 5.3.4 Duct Insulation and Sealing | RS-HVC-DINS-V03-140601 | Errata | Input capacity update Addition of furnace fan savings | Addition of kWh savings |
| | | 5.3.4 Duct Insulation and Sealing | RS-HVC-DINS-V05-150601 | Revision | Updated HP Federal Standard | Reduction in savings |

| Mkt | End Use | Measure Name | Measure Code | Change Type | Explanation | Impact on Savings |
|---|---|---|---|---|---|---|
| | | 5.3.6 Gas High Efficiency Boiler | RS-HVC-GHEB-V03-150601 | Revision | Added deemed early replacement rate. | None |
| | | 5.3.7 Gas High Efficiency Furnace | RS-HVC-GHEF-V04-150601 | Revision | Added deemed early replacement rate. | None |
| | | 5.3.8 Ground Source Heat Pump | RS-HVC-GSHP-V04-150601 | Revision | Complete rework of methodology | Increase in savings |
| | | 5.3.12 Ductless Heat Pumps | RS-HVC-DHP-V02-150601 | Revision | Updated HP Federal Standard | Reduction in savings |
| | | 5.3.13 Residential Furnace Tune up | RS-HVC-FTUN-V01-150601 | New | n/a | n/a |
| | | 5.3.14 Boiler Reset Controls | RS-HVC-BREC-V01-150601 | New | n/a | n/a |
| | | 5.3.15 ENERGY STAR Ceiling Fan | RS-HVC-CFAN-V01-150601 | New | n/a | n/a |
| Res | 5.4 Hot Water | 5.4.2 Gas Water Heater | RS-HWE-GWHT-V03-140601 | Errata | Removing link and reference to DeOreo study | None |
| | | 5.4.2 Gas Water Heater | RS-HWE-GWHT-V04-150601 | Revision | Added Early Replacement Updated Federal Baseline | Reduction in savings |
| | | 5.4.3 Heat Pump Water Heaters | RS-HWE-HPWH-V03-140601 | Errata | Coincidence factor reference update Removing link and reference to DeOreo study | None |
| | | 5.4.3 Heat Pump Water Heaters | RS-HWE-HPWH-V04-150601 | Revision | Updated Federal Baseline | Reduction in savings |
| | | 5.4.4 Low Flow Faucet Aerators | RS-HWE-LFFA-V04-140601 | Errata | Removing link and reference to DeOreo study | None |
| | | 5.4.6 Water Heater Temperature Setback | RS-HWE-TMPS-V04-150601 | Revision | Complete rework of methodology | Unknown |
| | | 5.4.7 Water Heater Wrap | RS-HWE-WRAP-V02-150601 | revision | Fixed algorithm typo | None |
| | | 5.4.8 Thermostatic Restrictor Shower Valve | RS-HWE-TRVA-V01-150601 | New | n/a | n/a |
| Res | 5.5 Lighting | 5.5.1 ENERGY STAR Compact Fluorescent Lamp | RS-LTG-ESCF-V04-150601 | Revision | Update to RES v C&I Split, ISR, Hours, Coincidence Factors. Added leakage variable | Unknown |

| Mkt | End Use | Measure Name | Measure Code | Change Type | Explanation | Impact on Savings |
|---|---|---|---|---|---|---|
| | | | | | Added Efficiency Kit ISRs | |
| | | 5.5.2 ENERGY STAR Specialty Compact Fluorescent Lamp | RS-LTG-ESCC-V03-150601 | Revision | Update to RES v C&I Split, ISR, Hours, Coincidence Factors. Added leakage variable Added Efficiency Kit ISRs | Unknown |
| | | 5.5.3 ENERGY STAR Torchiere | RS-LTG-ESTO-V02-150601 | Revision | Update to Coincidence Factors. Added leakage variable | Unknown |
| | | 5.5.4 Exterior Hardwired Compact Fluorescent Lamp | RS-LTG-EFIX-V04-150601 | Revision | Update to Hours, Coincidence Factors. Added leakage variable | Increase in savings |
| | | 5.5.5 Interior Hardwired Compact Fluorescent Lamp | RS-LTG-IFIX-V04-150601 | Revision | Update to Hours, Coincidence Factors. Added leakage variable | Reduction in savings |
| | | 5.5.6 LED Downlights | RS-LTG-LEDD-V03-140601 | Errata | Fixing typo in delta watts table | None |
| | | 5.5.6 LED Downlights | RS-LTG-LEDD-V04-150601 | Revision | Update to Hours, Coincidence Factors. Added leakage variable | Dependent on inputs |
| | | 5.5.8 LED Screw Based Omnidirectional Bulbs | RS-LTG-LEDA-V03-150601 | Revision | Update to ISR, Hours, Coincidence Factors. Added leakage variable Added Efficiency Kit ISRs | Unknown |
| Res | 5.6 Shell | 5.6.1 Air Sealing | RS-SHL-AIRS-V03-150601 | Revision | Updated HP Federal Standard | Reduction in savings |
| | | 5.6.2 Basement Sidewall Insulation | RS-SHL-BINS-V05-140601 | Errata | Removing Low Income distinction due to very small sample size. | Increase in LI savings |
| | | 5.6.2 Basement Sidewall Insulation | RS-SHL-BINS-V06-150601 | Revision | Updated HP Federal Standard | Reduction in savings |
| | | 5.6.3 Floor Insulation Above Crawlspace | RS-SHL-FINS-V05-140601 | Errata | Removing Low Income distinction due to very small sample size. | Increase in LI savings |
| | | 5.6.3 Floor Insulation Above Crawlspace | RS-SHL-FINS-V06-150601 | Revision | Updated HP Federal Standard | Reduction in savings |
| | | 5.6.4 Wall and Ceiling/Attic Insulation | RS-SHL-AINS-V05-150201 | Revision | Updated HP Federal Standard | Reduction in savings |

# 1 Purpose of the TRM

The purpose of the Illinois Statewide Technical Reference Manual (TRM) is to provide a transparent and consistent basis for calculating energy (electric kilowatt-hours (kWh) and natural gas therms) and capacity (electric kilowatts (kW)) savings generated by the State of Illinois' energy efficiency programs[2] which are administered by the Department of Commerce and Economic Opportunity (DCEO) and the state's largest electric and gas Utilities[3] (collectively, Program Administrators).

The TRM is a technical document that is filed with the Illinois Commerce Commission (Commission or ICC) and is intended to fulfill a series of objectives, including:

- "Serve as a common reference document for all… stakeholders, [Program Administrators], and the Commission, so as to provide transparency to all parties regarding savings assumptions and calculations and the underlying sources of those assumptions and calculations.
- Support the calculation of the Illinois Total Resource Cost test[4] ("TRC"), as well as other cost-benefit tests in support of program design, evaluation and regulatory compliance. Actual cost-benefit calculations and the calculation of avoided costs will not be part of this TRM.
- Identify gaps in robust, primary data for Illinois, that can be addressed via evaluation efforts and/or other targeted end-use studies.
- [Provide] a process for periodically updating and maintaining records, and preserve a clear record of what deemed parameters are/were in effect at what times to facilitate evaluation and data accuracy reviews.
- …[S]upport coincident peak capacity (for electric) savings estimates and calculations for electric utilities in a manner consistent with the methodologies employed by the utility's Regional Transmission Organization ("RTO"), as well as those necessary for statewide Illinois tracking of coincident peak capacity impacts."[5]

[2] 220 ILCS 5/8-103, 220 ILCS 5/16-111.5B and 220 ILCS 5/8-104.
[3] In addition to DCEO, the Program Administrators include: Ameren Illinois, ComEd, Peoples Gas, North Shore Gas, and Nicor Gas (collectively, the Utilities).
[4] The Illinois TRC test is defined in 220 ILCS 5/8-104(b) and 20 ILCS 3855/1-10.
[5] Illinois Statewide Technical Reference Manual Request for Proposals, August 22, 2011, pages 3-4, http://ilsag.org/yahoo_site_admin/assets/docs/TRM_RFP_Final_part_1.230214520.pdf

## 1.1 Enabling ICC Policy

This Illinois Statewide Technical Reference Manual (TRM) was developed to comply with the Illinois Commerce Commission (ICC or Commission) Final Orders from the electric and gas Utilities'[6] Energy Efficiency Plan dockets. In the Final Orders, the ICC required the utilities to work with DCEO and the Illinois Energy Efficiency Stakeholder Advisory Group (SAG) to develop a statewide TRM. *See, e.g.,* ComEd's Final Order *(Docket No. 10-0570, Final Order[7] at 59-60, December 21, 2010);* Ameren's Final Order *(Docket No. 10-0568, Order on Rehearing[8] at 19, May 24, 2011);* Peoples Gas/North Shore Gas' Final Order *(Docket No. 10-0564, Final Order[9] at 76, May 24, 2011),* and Nicor's Final Order *(Docket No. 10-0562, Final Order[10] at 30, May 24, 2011).*

As directed in the Utilities' Efficiency Plan Orders, the SAG had the opportunity to, and also participated in, every aspect of the development of the TRM. Interested members of the SAG participated in weekly teleconferences to review, comment, and participate in the development of the TRM. The active participants in the TRM were designated as the "Technical Advisory Committee" (TAC). The TAC participants include representatives from the following organizations:

- the Utilities (ComEd, Ameren IL, Nicor Gas, Peoples Gas/North Shore Gas),
- DCEO, Implementation contractors (Applied Proactive Technologies (APT), CLEAResult, Conservation Services Group, Elevate Energy, Franklin Energy, GDS Associates, PECI, 360 Energy Group),
- Illinois Department of Commerce and Economic Opportunity (DCEO),
- the independent evaluators (ADM Associates, The Cadmus Group, Itron, Navigant Consulting, Michael's Engineering, Opinion Dynamics Corporation),
- ICC Staff,
- the Illinois Attorney General's Office (AG),
- Natural Resources Defense Council (NRDC),
- the Environmental Law and Policy Center (ELPC),
- the Citizen's Utility Board (CUB),
- The University of Illinois at Chicago,
- Future Energy Enterprises,
- Geothermal Alliance of Illinois,
- the Geothermal Exchange Organization.

[6] The Illinois Utilities subject to this TRM include: Ameren Illinois Company d/b/a Ameren Illinois (Ameren), Commonwealth Edison Company (ComEd), The Peoples Gas Light and Coke Company and North Shore Gas Company (Integrys), and Northern Illinois Gas Company d/b/a Nicor Gas.
[7] http://www.icc.illinois.gov/docket/files.aspx?no=10-0570&docId=159809
[8] http://www.icc.illinois.gov/docket/files.aspx?no=10-0568&docId=167031
[9] http://www.icc.illinois.gov/docket/files.aspx?no=10-0564&docId=167023
[10] http://www.icc.illinois.gov/docket/files.aspx?no=10-0562&docId=167027

## 1.2 Development Process

The first edition of the IL-TRM was approved by the Commission in ICC Docket No. 12-0528[11]. The second edition of the IL-TRM was approved by the Commission in ICC Docket No. 13-0437[12]. The policies surrounding the applicability and use of the IL-TRM in planning, implementation, and evaluation were established by the Commission in ICC Docket No. 13-0077[13]. The Commission extended these policies, including the applicability of the IL-TRM, to the Section 16-111.5B energy efficiency programs in ICC Docket No. 14-0588[14] in order to increase certainty for all parties. The third edition of the IL-TRM was approved by the Commission in ICC Docket No. 14-0189[15]. This document represents the fourth edition of the IL-TRM. It contains a series of new measures, as well as a series of errata items[16] and updates to existing measures that were already present in the first, second and third editions. Like the previous editions, it is a result of an ongoing review process involving the Illinois Commerce Commission (ICC) Staff (Staff or ICC Staff), the Utilities, DCEO, the Evaluators, the SAG TAC, and the SAG. VEIC meets with the SAG and/or the TRM TAC at least once each month to create a high level of transparency and vetting in the development of this TRM.

Measure requests that are submitted by interested parties are ranked based on the following criteria to determine the approximate priority level for order of inclusion in the TRM:

1. High Priority
   - For those existing measures that make up a significant portion of a utilities' portfolio and/or where the impact of the requested change is high
   - For new measures where plans are in place to implement in the next program year
2. Medium Priority
   - For existing measures that are a less significant percent of a utilities' portfolio and value change will not have a significant impact
   - For new measures where a savings value is estimated but implementation plans not yet developed
3. Low Priority
   - For existing measures that represent a very small percent of a utilities' portfolio

[11] http://www.icc.illinois.gov/docket/files.aspx?no=12-0528&docId=187554
[12] http://www.icc.illinois.gov/docket/files.aspx?no=13-0437&docId=200492
[13] http://www.icc.illinois.gov/docket/files.aspx?no=13-0077&docId=203903; http://www.icc.illinois.gov/docket/files.aspx?no=13-0077&docId=195913; http://www.icc.illinois.gov/downloads/public/edocket/339744.pdf
[14] ICC Docket No. 14-0588, Final Order at 227, December 17, 2014.
The adopted consensus language concerning the IL-TRM and its applicability to future Section 16-111.5B energy efficiency programs can be accessed from the following link:
http://www.icc.illinois.gov/downloads/public/June%2018%202014%20Consensus%20Language%20for%20Section%2016-111.5B%20Oversight%20and%20Evaluation%20Responsibility%20Energy%20Efficiency%20Issues.pdf
[15] http://www.icc.illinois.gov/docket/files.aspx?no=14-0189&docId=210478
http://www.icc.illinois.gov/downloads/public/Illinois_Statewide_TRM_Effective_060114_Version_3.0_022414_Clean.pdf
[16] Errata as well as links to the official IL-TRM documents, dockets, and policy documents are available on the following ICC webpage: http://www.icc.illinois.gov/electricity/TRM.aspx

- For new measures that are just beginning to be explored and will not be implemented in the next program year

These rankings are used to align budget and schedule constraints with desired updates from the TRM.

As measure requests are finalized leading up to the next update of the TRM, weekly TAC meetings are often scheduled to maximize the level of collaboration and visibility into the measure characterization process. Where consensus does not emerge on specific measures or issues, those items are identified in a memo, and are not included in the TRM. As a result, this TRM represents a broad consensus amongst the SAG and TAC participants. In keeping with the goal of transparency, all of the comments and their status to-date are available through the TAC SharePoint web site, https://portal.veic.org.

For each measure characterization, this TRM includes engineering algorithm(s) and a value(s) for each parameter in the equation(s). These parameters have values that fall into one of three categories: a single deemed value, a lookup table of deemed values or an actual value such as the capacity of the equipment. The TRM makes extensive use of lookup tables because they allow for an appropriate level of measure streamlining and customization within the context of an otherwise prescriptive measure.

Accuracy is the overarching principle that governs what value to use for each parameter. When it is explicitly allowed within the text of the measure characterization, the preferred value is the actual or on-site value for the individual measure being implemented. The *deemed values*[17] in the lookup tables are the next most accurate choice, and in the absence of either an actual value or an appropriate value in a lookup table, the single, *deemed value* should be used. As a result, this single, *deemed value* can be thought of as a default value for that particular input to the algorithm.

A single *deemed savings estimate* is produced by any given combination of an algorithm and the allowable input values for each of its parameters. In cases where lookup tables are provided, there is a range of deemed savings estimates that are possible, depending on site-specific factors such as equipment capacity, location and building type.

Algorithms and their parameter values are included for calculating estimated:

- Gross annual electric energy savings (kWh)
- Gross annual natural gas energy savings (therms)
- Gross electric summer coincident peak demand savings (kW)

To support cost-effectiveness calculations, parameter values are also included for:

- Incremental costs ($)
- Measure life (years)
- Operation and maintenance costs ($)
- Water (gal) and other resource savings where appropriate.

[17] Emphasis has been added to denote the difference between a "deemed value" and a "deemed savings estimate". A deemed value refers to a single input value to an algorithm, while a deemed savings estimate is the result of calculating the end result of all of the values in the savings algorithm.

To facilitate the use of the TRM as measures are revised, updated, and removed, a unique code is provided for each measure that identifies the measure and the applicable installed program year.

# 2 Organizational Structure

The organization of this document follows a three-level format, each of which is a major heading in the Table of Contents. These levels are designed to define and clarify what the measure is and where it is applied.

1. **Market Sectors**[18]

   - This level of organization specifies the type of customer the measure applies to, either Commercial and Industrial or Residential.

   - Answers the question, "What category best describes the customer?"

2. **End-use Category**

   - This level of organization represents most of the major end-use categories for which an efficient alternative exists. The following table lists all of the end-use categories in this version of the TRM.

   - Answers the question, "To what end-use category does the measure apply?"

**Table 2.1: End-Use Categories in the TRM**[19]

| Residential Market Sector | Commercial and Industrial Market Sector |
|---|---|
| Appliances | Agricultural Equipment |
| Consumer Electronics | Food Service Equipment |
| Hot Water | Hot Water |
| HVAC | HVAC |
| Lighting | Lighting |
| Shell | Miscellaneous |
| | Refrigeration |

3. **Measure & Technology**

   - This level of organization represents individual efficient measures such as CFL lighting and LED lighting, both of which are individual technologies within the Lighting end-use category.

   - Answers the question, "What technology defines the measure?"

This organizational structure is silent on which fuel the measure is designed to save; electricity or natural gas. By organizing the TRM this way, measures that save on both fuels do not need to be repeated. As a result, the TRM will be easier to use and to maintain.

[18] Note that the Public sector buildings and low income measures that DCEO administers are not listed as a separate Market Sector. The Public building type is one of a series of building types that are included in the appropriate measures in the Commercial and Industrial Sector.

[19] Please note that this is not an exhaustive list of end-uses and that others may be included in future versions of the TRM.

## 2.1 Measure Code Specification

In order to uniquely identify each measure in the TRM, abbreviations for the major organizational elements of the TRM have been established. When these abbreviations are combined and delimited by a dash ('-') a unique, 18-character alphanumeric code is formed that can be used for tracking the measures and their associated savings estimates. Measure codes appear at the end of each measure and are structured using five parts.

**Code Structure = Market + End-use Category + Measure + Version # + Effective Date**

For example, the commercial boiler measure is coded: "CI-HVC-BLR_-V01-120601"

**Table 2.2: Measure Code Specification Key**

| Market (@@) | End-use (@@@) | Measure (@@@@) | Version (V##) | Effective Date |
|---|---|---|---|---|
| CI (C&I) | AGE (Agricultural Equipment) | BLR_ | V01 | YYMMDD |
| RS (Residential) | APL (Appliances) | T5F_ | V02 | YYMMDD |
| | CEL (Consumer Electronics) | T8F_ | V03 | YYMMDD |
| | FSE (Food Service Equipment) | … | … | … |
| | HVC (HVAC) | | | |
| | HW_ (Hot Water) | | | |
| | LTG (Lighting) | | | |
| | MSC (Miscellaneous) | | | |
| | RFG (Refrigeration) | | | |
| | SHL (Shell) | | | |

## 2.2 Components of TRM Measure Characterizations

Each measure characterization uses a standardized format that includes at least the following components. Measures that have a higher level of complexity may have additional components, but also follow the same format, flow and function.

### DESCRIPTION

Brief description of measure stating how it saves energy, the markets it serves and any limitations to its applicability.

### DEFINITION OF EFFICIENT EQUIPMENT

Clear definition of the criteria for the efficient equipment used to determine delta savings. Including any standards or ratings if appropriate.

### DEFINITION OF BASELINE EQUIPMENT

Clear definition of the efficiency level of the baseline equipment used to determine delta savings including any standards or ratings if appropriate. If a Time of Sale measure the baseline will be new base level equipment (to replace existing equipment at the end of its useful life or for a new building). For Early Replacement or Early Retirement measures the baseline is the existing working piece of equipment that is being removed.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected duration in years (or hours) of the savings. If an early replacement measure, the assumed life of the existing unit is also provided.

### DEEMED MEASURE COST

For time of sale measures, incremental cost from baseline to efficient is provided. Installation costs should only be included if there is a difference between each efficiency level. For Early Replacement the full equipment and install cost of the efficient installation is provided in addition to the full deferred hypothetical baseline replacement cost.

### LOADSHAPE

The appropriate loadshape to apply to electric savings is provided.

### COINCIDENCE FACTOR

The summer coincidence factor is provided to estimate the impact of the measure on the utility's system peak – defined as 1PM to hour ending 5PM on non-holiday weekdays, June through August.

**Algorithm**

### CALCULATION OF ENERGY SAVINGS

Algorithms are provided followed by list of assumptions with their definition.

If there are no Input Variables, there will be a finite number of Output values. These will be identified and listed in a table. Where there are custom inputs, an example calculation is often provided to illustrate the algorithm and provide context.

### Electric Energy Savings

### Summer Coincident Peak Demand Savings

### Natural Gas Savings

### Water Impact Descriptions and Calculation

### Deemed O&M Cost Adjustment Calculation

Only required if the operation and maintenance cost for the efficient case is different to the baseline.

### Measure Code

## 2.3 Variable Input Tables

Many of the measures in this TRM require the user to select the appropriate input value from a list of inputs for a given parameter in the savings algorithm. Where the TRM asks the user to select the input, look-up tables of allowable values are provided. For example, a set of input parameters may depend on building type; while a range of values may be given for each parameter, only one value is appropriate for any specific building type. If no table of alternative inputs is provided for a particular parameter, then the single deemed value will be used, unless the measure has a custom allowable input.

### 2.3.1 C&I Custom Value Use in Measure Implementation

This section defines the requirements for capturing Custom variables that can be used in place of defaults for select assumptions within the prescriptive measures defined in this statewide TRM. This approach is to be used when a variable in a measure formula can be replaced by a verifiable and documented value that is not presented in the TRM. This approach assumes that the algorithms presented in the measure are used as stated and only allows changes to certain variable values and is not a replacement algorithm for the measure. A custom variable is when customer input is provided to define the number or the value is measured at the site. Custom values can also be supplied from product data of the measure installed. In certain cases the custom data can be provided from a documented study or report that is applicable to the measure. Custom variables and potential sources are clearly defined in the specific measures where “Actual” or “Custom” is noted.

In exceptional cases where the participant, program administrator, and independent evaluator all agree that the TRM algorithm for a particular energy efficiency measure does not accurately characterize the energy efficiency measure within a project due to the complexity in the design and configuration of the particular energy efficiency project, a more comprehensive custom engineering and financial analysis may be used that more accurately incorporates the attributes of the measure in the complex energy efficiency project. In such cases and consistent with Commission policy adopted in ICC Docket No. 13-0077, Program Administrators are subject to retrospective evaluation risk (retroactive adjustments to savings based on ex post evaluation findings) for such projects utilizing customized savings calculations.

## 2.4 Program Delivery & Baseline Definitions

The measure characterizations in this TRM are not grouped by program delivery type. As a result, the measure characterizations provided include information and assumptions to support savings calculations for the range of program delivery options commonly used for the measure. The organizational significance of this approach is that multiple baselines, incremental costs, O&M costs, measure lives and in-service rates are included in the measure characterization(s) that are delivered under two or more different program designs. Values appropriate for each given program delivery type are clearly specified in the algorithms or in look-up tables within the characterization.

Care has been taken to clearly define in the measure's description the types of program delivery that the measure characterization is designed to support. However, there are no universally accepted definitions for a particular program type, and the description of the program type(s) may differ by measure. Nevertheless, program delivery types can be generally defined according to the following table. These are the definitions used in the measure descriptions, and, when necessary, individual measure descriptions may further refine and clarify these definitions of program delivery type.

**Table 2.3: Program Delivery Types**

| Program | Attributes |
|---|---|
| Time of Sale (TOS) | Definition: A program in which the customer is incented to purchase or install higher efficiency equipment than if the program had not existed. This may include retail rebate (coupon) programs, upstream buydown programs, online store programs or contractor based programs as examples.<br>Baseline = New equipment.<br>Efficient Case = New, premium efficiency equipment above federal and state codes and standard industry practice.<br>Example: CFL rebate |
| New Construction (NC) | Definition: A program that intervenes during building design to support the use of more-efficient equipment and construction practices.<br>Baseline = Building code or federal standards.<br>Efficient Case = The program's level of building specification<br>Example: Building shell and mechanical measures |
| Retrofit (RF) | Definition: A program that upgrades existing equipment before the end of its useful life.<br>Baseline = Existing equipment or the existing condition of the building or equipment. A single baseline applies over the measure's life.<br>Efficient Case = New, premium efficiency equipment above federal and state codes and standard industry practice.<br>Example: Air sealing and insulation |
| Early Replacement (EREP) | Definition: A program that replaces existing equipment before the end of its expected life.<br>Baseline = Dual; it begins as the existing equipment and shifts to new baseline equipment after the expected life of the existing equipment is over.<br>Efficient Case = New, premium efficiency equipment above federal and state codes and standard industry practice.<br>Example: Refrigerators, freezers |
| Early Retirement (ERET) | Definition: A program that retires duplicative equipment before its expected life is over.<br>Baseline = The existing equipment, which is retired and not replaced.<br>Efficient Case = Zero because the unit is retired.<br>Example: Appliance recycling |
| Direct Install (DI) | Definition: A program where measures are installed during a site visit.<br>Baseline = Existing equipment.<br>Efficient Case = New, premium efficiency equipment above federal and state codes and standard industry practice. |

| Program | Attributes |
|---|---|
| | Example: Lighting and low-flow hot water measures |
| Efficiency Kits (KITS) | Definition: A program where measures are provided free of charge to a customer in an Efficiency Kit.<br>Baseline = Existing equipment.<br>Efficient Case = New, premium efficiency equipment above federal and state codes and standard industry practice.<br>Example: Lighting and low-flow hot water measures |

The concept and definition of the baseline is a key element of every measure characterization and is directly related to the program delivery type. Without a clear definition of the baseline, the savings algorithms cannot be adequately specified and subsequent evaluation efforts would be hampered. As a result, each measure has a detailed description (and in many cases, specification) of the specific baseline that should be used to calculate savings. Baselines in this TRM fall into one of the following four categories, and are organized within each measure characterization by the program delivery type to which it applies.

1. **Building Code:** As defined by the minimum specifications required under state energy code or applicable federal standards.

2. **Existing Equipment**: As determined by the most representative (or average) example of equipment that is in the existing stock. Existing equipment baselines apply over the equipment's remaining useful life.

3. **New Equipment:** As determined by the equipment that represents standard practice in the current market environment. New equipment baselines apply over the effective useful life of the measure.

4. **Dual Baseline:** A baseline that begins as the existing equipment and shifts to new equipment after the expected life of the existing equipment is over.

# 3 Assumptions

The information contained in this TRM contains VEIC's recommendations for the content of the Illinois TRM. Sources that are cited within the TRM have been chosen based on two priorities, geography and age. Whenever possible and appropriate, VEIC has incorporated Illinois-specific information into each measure characterization. The Business TRM documents from Ameren and ComEd were reviewed, as well as program and measure specific data from evaluations, efficiency plans, and working documents.

The assumptions for these characterizations rest on our understanding of the information available. In each case, the available Illinois and Midwest-specific information was reviewed, including evaluations and support material provided by the Illinois Utilities.

When Illinois or region-specific evaluations or data were not available, best practice research and data from other jurisdictions was used, often from west and east-coast states that have allocated large amounts of funding to evaluation work and to refining their measure characterization parameters. As a result, much of the most-defensible information originates from these regions. In every case, VEIC used the most recent, well-designed, and best-supported studies and only if it was appropriate to generalize their conclusions to the Illinois programs.

## 3.1 Footnotes & Documentation of Sources

Each new and updated measure characterization is supported by a work paper, which is posted to the SharePoint web site (https://portal.veic.org).[20] Both the work paper and the measure characterizations themselves use footnotes to document the references that have been used to characterize the technology. The reference documents are too numerous to include in an Appendix and have instead been posted to the TRM's Sharepoint website. These files can be found in the 'Sources and Reference Documents' folder in the main directory, and are also be posted to the SAG's public web site (http://www.ilsag.info/technical-reference-manual.html).

## 3.2 General Savings Assumptions

The TRM savings estimates are expected to serve as average, representative values, or ways to calculate savings based on program-specific information. All information is presented on a per-measure basis. In using the measure-specific information in the TRM, it is helpful to keep the following notes in mind.

- All estimates of energy (kWh or therms) and peak (kW) savings are for first-year savings, not lifetime savings.
- Unless otherwise noted, measure life is defined to be the life of an energy consuming measure, including its equipment life and measure persistence.
- Where deemed values for savings are provided, they represent the average energy (kWh or therms) or peak (kW) savings that could be expected from the average of all measures that might be installed in Illinois in the program year.
- In general, the baselines included in the TRM are intended to represent average conditions in Illinois. Some are based on data from the state, such as household consumption characteristics provided by the Energy Information Administration. Some are extrapolated from other areas, when Illinois data are not available.

[20] To gain access to the SharePoint web site, please contact the TRM Administrator, Nikki Clace at iltrmadministrator@veic.org.

## 3.3 Shifting Baseline Assumptions

The TRM anticipates the effects of changes in efficiency codes and standards on affected measures. When these changes take effect, a shift in the baseline is usually required. This complicates the measure savings estimation somewhat, and will be handled in future versions of the TRM by describing the choice of and reasoning behind a shifting baseline assumption. In this version of the TRM, this applies to CFLs and T5/T8 Linear Fluorescents, Furnaces and Early Replacement Measures.

### 3.3.1 CFL and T5/T8 Linear Fluorescents Baseline Assumptions

Specific reductions in savings have been incorporated for CFL measures that relate to the shift in appropriate baseline due to changes in Federal Standards for lighting products. Federal legislation (stemming from the Energy Independence and Security Act of 2007) mandates a phase-in process beginning in 2012 for all general-purpose light bulbs between 40W and 100W to be approximately 30% more energy efficient than current incandescent bulbs, in essence beginning the phase-out of the current style, or "standard", incandescent bulbs. In 2012, standard 100W incandescent bulbs will no longer be manufactured, followed by restrictions on standard 75W bulbs in 2013 and 60W and 40W bulbs in 2014. The baseline for the CFL measure in the corresponding program years starting June 1 each year will therefore become bulbs (improved or "efficient" incandescent, or halogen) that meet the new standard and have the same lumen equivalency. Those products can take several different forms we can envision now and perhaps others we do not yet know about. Halogens are one of those possibilities and have been chosen to represent a baseline at that time. To account for this shifting baseline, annual savings are reduced within the lifetime of the measure. Other lighting measures will also have baseline shifts (for example screw based LED and CFL fixtures) that will result in significant impacts to annual estimated savings in later years.

In July 14, 2012, Federal Standards were enacted that were expected to eliminate T-12s as an option for linear fluorescent fixtures. Through v3.0 of the TRM, it was assumed that the T-12 would no longer be baseline for retrofits from 1/1/2016. However, due to significant loopholes in the legislation, T-12 compliant product is still freely available and in Illinois T-12s continue to hold a significant share of the existing and replacement lamp market. Therefore the timing of the sunsetting of T-12s as a viable baseline has been pushed back in v4.0 until 6/1/2016 and will be revisited in future update sessions.

### 3.3.2 Early Replacement Baseline Assumptions

A series of measures have an option to choose an Early Replacement Baseline. For these measures, the baseline assumption of the existing unit efficiency is based upon the average efficiency of units that were classified in Ameren's PY3-PY4 as functioning and that meet efficiency and cost of replacement criteria in the following table.

**Table 3.1: Early Replacement Baseline Criteria[21]**

| Measure | Section | Criteria |
|---|---|---|
| Air Source Heap Pump | 5.3.1 | SEER <=10 and cost of any repairs <$249 per ton |
| Central Air Conditioner | 5.3.3 | SEER <=10 and cost of any repairs <$190 per ton |
| Boiler | 5.3.6 | AFUE <= 75% and cost of any repairs <$709 |
| Furnace | 4.4.11, 5.3.7 | AFUE <= 75% and cost of any repairs <$528 |
| Ground Source Heat Pump | 5.3.8 | SEER <=10 and cost of any repairs <$249 per ton |

It is only appropriate to use these Early Replacement assumptions where these conditions are met. The TAC

[21] These criteria were documented in a memo entitled, "Early Replacement Measure Issue Summary_0409.docx."

defined "functioning" as the unit is fully operational – providing sufficient space conditioning (i.e. heat exchanger, compressors, pumps work effectively) and/or the cost of repair is under 20% of the new baseline replacement cost. Therefore in order to apply early replacement assumptions the programs should apply the following eligibility criteria for the existing heating or cooling system in the home:

### 3.3.3 Furnace Baseline

"The prior national standard for residential oil and gas furnaces was 78% AFUE. DOE raised the standard in 2007 to 80% AFUE, effective 2015. However, virtually all furnaces on the market have an AFUE of 80% or better, which prompted states and environmental and consumer groups to sue DOE over its 2007 decision. In April 2009, DOE accepted a "voluntary remand" in that litigation. In October 2009, manufacturers and efficiency advocates negotiated an agreement that, for the first time, included different standard levels in three climate regions: the North, South, and Southwest. DOE issued a direct final rule (DFR) in June 2011 reflecting the standard levels in the consensus agreement. The DFR became effective on October 25, 2011 establishing new standards: In the North, most furnaces will be required to have an AFUE of 90%.The 80% AFUE standard for the South and Southwest will remain unchanged at 80%. Oil furnaces will be required to have an AFUE of 83% in all three regions. The amended standards will become effective in May 2013 for non-weatherized furnaces and in January 2015 for weatherized furnaces. DOE estimates that the standards will save about 3.3 quads (quadrillion Btu) of energy over 30 years and yield a net present value of about $14 billion at a 3 percent discount rate.

Update*:* On January 14th 2013, the U.S. Department of Energy (DOE) proposed to settle a lawsuit brought by the American Public Gas Association (APGA) that seeks to roll back gas furnace efficiency standards. As a result, the new standards, completed in 2011 and slated to take effect in May 2013, would be eliminated in favor of yet another round of DOE hearings and studies. Even if DOE completes a new rulemaking in two years, it's unlikely to take effect before 2020."[22]

As a result, each of the furnace measures contains the following language describing the baseline assumption:

"Although the current Federal Standard for gas furnaces is an AFUE rating of 78%, based upon review of available product in the AHRI database, the baseline efficiency for this characterization is assumed to be 80%. The baseline will be adjusted when the Federal Standard is updated."

[22] Appliance Standards Awareness Project, http://www.appliance-standards.org/product/furnaces

## 3.4 Glossary

**Baseline Efficiency:** The assumed standard efficiency of equipment, absent an efficiency program.

**Building Types[23]:**

| Building Type | Definition |
|---|---|
| Assisted Living MultiFamily | Applies to residential buildings of three of more units with staff to assist the occupants. Gross Floor Area should include all fully-enclosed space within the exterior walls of the building(s) including individual rooms or units, wellness centers, exam rooms, community rooms, small shops or service areas for residents and visitors (e.g. hair salons, convenience stores), staff offices, lobbies, atriums, cafeterias, kitchens, storage areas, hallways, basements, stairways, corridors between buildings, and elevator shafts. |
| Auditorium/Assembly | Applies to any performance space such as a theater, arena, or hall. Gross Floor Area should include all space within the building(s), including seating, stage and backstage areas, food service areas, retail areas, rehearsal studios, administrative/office space, mechanical rooms, storage areas, elevator shafts, and stairwells. |
| College/University | Applies to facility space used for higher education. Relevant buildings include administrative headquarters, residence halls, athletic and recreation facilities, laboratories, etc. The total gross floor area should include all supporting functions such as kitchens used by staff, lobbies, atria, conference rooms and auditoria, fitness areas for staff, storage areas, stairways, elevator shafts, etc. |
| Convenience Store | Applies to facility space used for the retail sale of a limited selection of food and beverage products. The total gross floor area should include all supporting functions such as kitchens and break rooms used by staff, storage areas (refrigerated and non-refrigerated), and administrative areas. |
| Elementary School | Applies to a school serving children In any grades from Kindergarten through sixth grade. The total gross floor area should include all supporting functions such as administrative space, conference rooms, kitchens used by staff, lobbies, cafeterias, gymnasiums, auditoria, laboratory classrooms, portable classrooms, greenhouses, stairways, atria, elevator shafts, small landscaping sheds, storage areas, etc. |
| Exterior | Applies to unconditioned spaces that are outside of the building envelope. |
| Garage | Applies to unconditioned spaces either attached or detached from the primary building envelope that are not used for living space. |
| Grocery | Applies to facility space used for the retail sale of food and beverage products. It should not be used by restaurants. The total gross floor area should include all supporting functions such as kitchens and break rooms used by staff, storage areas (refrigerated and non-refrigerated), administrative areas, stairwells, atria, lobbies, etc. |
| Healthcare Clinic | Applies to a facility space used to provide diagnosis and treatment for medical, dental, or psychiatric outpatient care. Gross Floor Area should include all space within the building(s) including offices, exam rooms, laboratories, lobbies, atriums, conference rooms and auditoriums, employee break rooms and kitchens, rest rooms, elevator shafts, stairways, mechanical rooms, and storage areas. |
| Heavy Industry | Heavy industry buildings are typically characterized by a plant that includes a main production area that has high-ceilings and contains heavy equipment used for assembly |

[23] Source: US EPA, www.energystar.gov, Space Type Definitions

| Building Type | Definition |
|---|---|
| | line production. These building types may be distinguished by categorizing NIACS (SIC) codes according to the needs of the Program Administrator. |
| High School/Middle School | Applies to facility space used as a school building for 7th through 12th grade students. This does not include college or university classroom facilities and laboratories, vocational, technical, or trade schools. The total gross floor area should include all supporting functions such as administrative space, conference rooms, kitchens used by staff, lobbies, cafeterias, gymnasiums, auditoria, laboratory classrooms, portable classrooms, greenhouses, stairways, atria, elevator shafts, small landscaping sheds, storage areas, etc. |
| Hospital | Applies to a general medical and surgical hospital (including critical access hospitals and children's hospitals) that is either a stand-alone building or a campus of buildings. Spaces more accurately characterized as a Healthcare Clinic should use that definition. The definition of Hospital accounts for all space types that are located within the Hospital building/campus, such as medical offices, administrative offices, and skilled nursing. The total floor area should include the aggregate floor area of all buildings on the campus as well as all supporting functions such as: stairways, connecting corridors between buildings, medical offices, exam rooms, laboratories, lobbies, atria, cafeterias, storage areas, elevator shafts, and any space affiliated with emergency medical care, or diagnostic care. |
| Hotel/Motel Combined (All Spaces) | Applies to buildings that rent overnight accommodations on a room/suite basis, typically including a bath/shower and other facilities in guest rooms. The total gross floor area should include all interior space, including guestrooms, halls, lobbies, atria, food preparation and restaurant space, conference and banquet space, health clubs/spas, indoor pool areas, and laundry facilities, as well as all space used for supporting functions such as elevator shafts, stairways, mechanical rooms, storage areas, employee break rooms, back-of-house offices, etc. Hotel does not apply to fractional ownership properties such as condominiums or vacation timeshares. Hotel properties should be owned by a single entity and have rooms available on a nightly basis. |
| Hotel/Motel Common Areas | All the common areas open to guests of the hotel such as the lobby, corridors and stairways, and other spaces that may have continuous or large lighting and HVAC hours. |
| Hotel/Motel Guest Room | Applies to the guest rooms of the hotel or motel. These spaces are occupied intermittantly. |
| Light Industry | Applies to buildings that are dedicated to manufacturing activities. Light industry buildings are characterized by consumer product and component manufacturing .These building types may be distinguished by categorizing NIACS (SIC) codes according to the needs of the Program Administrator. |
| Miscellaneous | Applies to spaces that do not fit clearly within any available categories should be designated as "miscellaneous". |
| Multifamily-Mid Rise | Applies to residential buildings with up to four floors, including all public and multiuse spaces within the building envelope. Small Multifamily buildings best described as a house should use the residential measure characterizations. |
| Multifamily-High Rise Combined (All Spaces) | Applies to residential buildings with five or more floors, including all public and multiuse spaces within the building envelope. Gross Floor Area should include all fully-enclosed space within the exterior walls of the building(s) including living space in each unit (including occupied and unoccupied units), interior common areas (e.g. lobbies, offices, community rooms, common kitchens, fitness rooms, indoor pools), hallways, stairwells, elevator shafts, connecting corridors between buildings, storage areas, and mechanical space such as a boiler room. Open air stairwells, breezeways, and other similar areas that are not fully-enclosed should not be included in the Gross Floor Area. |

| Building Type | Definition |
|---|---|
| Multifamily-High Rise Common Areas | All the common areas open to occupants of the building such as the lobby, corridors and stairways, and other spaces that may have continuous or high lighting and HVAC hours. |
| Multifamily-High Rise Residential Units | Applies to the residential units in the building only. |
| Movie Theater | Applies to buildings used for public or private film screenings. Gross Floor Area should include all space within the building(s), including seating areas, lobbies, concession stands, bathrooms, administrative/office space, mechanical rooms, storage areas, elevator shafts, and stairwells. |
| Office-Low Rise | Applies to facility spaces in buildings with four floors or fewer used for general office, professional, and administrative purposes. The total gross floor area should include all supporting functions such as kitchens used by staff, lobbies, atria, conference rooms and auditoria, fitness areas for staff, storage areas, stairways, elevator shafts, etc. |
| Office-Mid Rise | Applies to facility spaces in buildings with ffive to nine floors used for general office, professional, and administrative purposes. The total gross floor area should include all supporting functions such as kitchens used by staff, lobbies, atria, conference rooms and auditoria, fitness areas for staff, storage areas, stairways, elevator shafts, etc. |
| Office-High Rise | Applies to facility spaces in buildings with ten floors or more used for general office, professional, and administrative purposes. The total gross floor area should include all supporting functions such as kitchens used by staff, lobbies, atria, conference rooms and auditoria, fitness areas for staff, storage areas, stairways, elevator shafts, etc. |
| Religious Worship/Church | Applies to buildings that are used as places of worship. This includes churches, temples, mosques, synagogues, meetinghouses, or any other buildings that primarily function as a place of religious worship. Gross Floor Area should include all areas inside the building that includes the primary worship area, including food preparation, community rooms, classrooms, and supporting areas such as restrooms, storage areas, hallways, and elevator shafts. |
| Restaurant | Applies to a subcategory of Retail/Service space that is used to provide commercial food services to individual customers, and includes kitchen, dining, and common areas. |
| Retail/Service-Department store | Applies to facility space used to conduct the retail sale of consumer product goods. Stores must be at least 30,000 square feet and have an exterior entrance to the public. The total gross floor area should include all supporting functions such as kitchens and break rooms used by staff, storage areas, administrative areas, elevators, stairwells, etc. Retail segments typically included under this definition are: Department Stores, Discount Stores, Supercenters, Warehouse Clubs, Drug Stores, Dollar Stores, Home Center/Hardware Stores, and Apparel/Hard Line Specialty Stores (e.g., books, clothing, office products, toys, home goods, electronics). Retail segments excluded under this definition are: Grocery, Convenience Stores, Automobile Dealerships, and Restaurants. |
| Retail/Service- Strip Mall | Applies to facility space used to conduct the retail sale of consumer product goods. Stores must less than 30,000 square feet and have an exterior entrance to the public. The total gross floor area should include all supporting functions such as kitchens and break rooms used by staff, storage areas, administrative areas, elevators, stairwells, etc. Retail segments excluded under this definition are: Grocery, Convenience Stores, Automobile Dealerships, and Restaurants. |
| Warehouse | Applies to unrefrigerated or refrigerated buildings that are used to store goods, manufactured products, merchandise or raw materials. The total gross floor area of Refrigerated Warehouses should include all temperature controlled area designed to store perishable goods or merchandise under refrigeration at temperatures below 50 degrees Fahrenheit. The total gross floor area of Unrefrigerated Warehouses should |

| Building Type | Definition |
|---|---|
| | include space designed to store non-perishable goods and merchandise. Unrefrigerated warehouses also include distribution centers. The total gross floor area of refrigerated and unrefrigerated warehouses should include all supporting functions such as offices, lobbies, stairways, rest rooms, equipment storage areas, elevator shafts, etc. Existing atriums or areas with high ceilings should only include the base floor area that they occupy. The total gross floor area of refrigerated or unrefrigerated warehouse should not include outside loading bays or docks. Self-storage facilities, or facilities that rent individual storage units, are not eligible for a rating using the warehouse model. |

**Coincidence Factor** (CF): Coincidence factors represent the fraction of connected load expected to be coincident with a particular system peak period, on a diversified basis. Coincidence factors are provided for summer peak periods.

**Commercial & Industrial:** The market sector that includes measures that apply to any of the building types defined in this TRM, which includes multifamily common areas and public housing[24].

**Connected Load**: The maximum wattage of the equipment, under normal operating conditions.

**Deemed Value:** A value that has been assumed to be representative of the average condition of an input parameter.

**Default Value**: When a measure indicates that an input to a prescriptive saving algorithm may take on a range of values, an average value is also provided in many cases. This value is considered the default input to the algorithm, and should be used when the other alternatives listed in the measure are not applicable.

**End-use Category:** A general term used to describe the categories of equipment that provide a service to an individual or building. See Table 2.1.1 for a list of the end-use categories that are incorporated in this TRM.

**Energy Efficiency:** "Energy efficiency" means measures that reduce the amount of electricity or natural gas required to achieve a given end use. "Energy efficiency" also includes measures that reduce the total Btus of electricity and natural gas needed to meet the end use or uses (20 ILCS 3855/1-10). For purposes of this Section, "energy efficiency" means measures that reduce the amount of energy required to achieve a given end use. "Energy efficiency" also includes measures that reduce the total Btus of electricity and natural gas needed to meet the end use or uses (220 ILCS 5/8-104(b)).

**Equivalent Full Load Hours** (EFLH): The equivalent hours that equipment would need to operate at its peak capacity in order to consume its estimated annual kWh consumption (annual kWh/connected kW) or therms.

**High Efficiency**: General term for technologies and processes that require less energy, water, or other inputs to operate.

**Lifetime**: The number of years (or hours) that the new high efficiency equipment is expected to function. These are generally based on engineering lives, but sometimes adjusted based on expectations about frequency of removal, remodeling or demolition. Two important distinctions fall under this definition; Effective Useful Life (EUL) and Remaining Useful Life (RUL).

[24] Measures that apply to the multifamily and public housing building types describe how to handle tenant versus master metered buildings.

**EUL** – EUL is based on the manufacturers rating of the effective useful life; how long the equipment will last. For example, a CFL that operates x hours per year will typically have an EUL of y. A house boiler may have a lifetime of 20 years but the EUL is only 15 years since after that time it may be operating at a non-efficient point. An estimate of the median number of years that the measures installed under a program are still in place and operable.

**RUL** – Applies to retrofit or replacement measures. For example, if an existing working refrigerator is replaced with a high efficiency unit, the RUL is an assumption of how many more years the existing unit would have lasted. As a general rule the RUL is usually assumed to be 1/3 of the EUL.

**Load Factor** (LF): The fraction of full load (wattage) for which the equipment is typically run.

**Measure Cost**: The incremental (for time of sale measures) or full cost (both capital and labor for retrofit measures) of implementing the High Efficiency equipment.

**Measure Description**: A detailed description of the technology and the criteria it must meet to be eligible as an energy efficient measure.

**Measure:** An efficient technology or procedure that results in energy savings as compared to the baseline efficiency.

**Residential:** The market sector that includes measures that apply only to detached, residential buildings or duplexes.

**Operation and Maintenance (O&M) Cost Adjustments:** The dollar impact resulting from differences between baseline and efficient case Operation and Maintenance costs.

**Operating Hours** (HOURS): The annual hours that equipment is expected to operate.

**Program:** The mode of delivering a particular measure or set of measures to customers. See Table 2.4 for a list of program descriptions that are presently operating in Illinois.

**Rating Period Factor** (RPF): Percentages for defined times of the year that describe when energy savings will be realized for a specific measure.

**Stakeholder Advisory Group (SAG):** The Illinois Energy Efficiency Stakeholder Advisory Group (SAG) was first defined in the electric utilities' first energy efficiency Plan Orders to include "… the Utility, DCEO, Staff, the Attorney General, BOMA and CUB and representation from a variety of interests, including residential consumers, business consumers, environmental and energy advocacy organizations, trades and local government... [and] a representative from the ARES (alternative retail electric supplier) community should be included."[25] A group of stakeholders who have an interest in Illinois' energy efficiency programs and who meet regularly to share information and work toward consensus on various energy efficiency issues. The Utilities in Illinois have been directed by the ICC to work with the SAG on the development of a statewide TRM.

[25] ICC Docket No. 07-0540, Final Order at 32-33, February 6, 2008.
http://www.icc.illinois.gov/downloads/public/edocket/215193.pdf

**Table 3.2: Degree-Day Zones and Values by Market Sector**

| | Residential | | C&I | | |
|---|---|---|---|---|---|
| Zone | HDD | CDD | HDD | CDD | Weather Station / City |
| 1 | 5,352 | 820 | 4,272 | 2,173 | Rockford AP / Rockford |
| 2 | 5,113 | 842 | 4,029 | 2,181 | Chicago O'Hare AP / Chicago |
| 3 | 4,379 | 1,108 | 3,406 | 2,666 | Springfield #2 / Springfield |
| 4 | 3,378 | 1,570 | 2,515 | 3,358 | Belleville SIU RSCH / Belleville |
| 5 | 3,438 | 1,370 | 2,546 | 3,090 | Carbondale Southern IL AP / Marion |
| Average | 4,860 | 947 | 3,812 | 2,362 | Weighted by occupied housing units |
| Base Temp | 60F | 65F | 55F | 55F | Year climate normals, 1981-2010 |

## 3.5 Electrical Loadshapes (kWh)

Loadshapes are an integral part of the measure characterization and are used to divide energy savings into appropriate periods using Rating Period Factors (RPFs) such that each have variable avoided cost values allocated to them for the purpose of estimating cost effectiveness.

For the purposes of assigning energy savings (kWh) periods, the TRM TAC has agreed to use the industry standards for wholesale power market transactions as shown in the following table.

**Table 3.3: On and Off Peak Energy Definitions**

| Period Category | Period Definition (Central Prevailing Time) |
|---|---|
| Winter On-Peak Energy | 8AM - 11PM, weekdays, Oct – Apr, No NERC holidays |
| Winter Off-Peak Energy | All other hours |
| Summer On-Peak Energy | 8AM - 11PM, weekdays, May – Sept, No NERC holidays |
| Summer Off-Peak Energy | All other hours |

Loadshapes have been developed for each end-use by assigning Rating Period Factor percentages to each of the four periods above. Two methodologies were used:

1. Itron eShapes data for Missouri, reconciled to Illinois loads and provided by Ameren, were used to calculate the percentage of load in to the four categories above.

2. Where the Itron eShapes data did not provide a particular end-use or specific measure load profile, loadshapes that have been developed over many years by Efficiency Vermont and that have been reviewed by the Vermont Department of Public Service, were adjusted to match Illinois period definitions. Note – no weather sensitive loadshapes were based on this method. Any of these load profiles that relate to High Impact Measures should be an area of future evaluation.

The following pages provide the loadshape values for most measures provided in the TRM[26]. To distinguish the source of the loadshape, they are color coded. Rows that are shaded in green are Efficiency Vermont loadshapes adjusted for Illinois periods. Rows that are unshaded and are left in white are Itron eShapes data provided by Ameren.

ComEd uses the DSMore™ (Integral Analytics DSMore™ Demand Side Management Option/Risk Evaluator) software to screen the efficiency measures for cost effectiveness. Since this tool requires a loadshape value for weekdays and weekends in each month (i.e., 24 inputs), the percentages for the four period categories above were calculated by weighting the proportion of weekdays/weekends in each month to the total within each period. The results of these calculations are also provided below.

[26] All loadshape information has been posted to the VEIC Sharepoint site, and is publically accessible through the Stakeholder Advisory Group's web site. http://www.ilsag.info/technical-reference-manual.html
http://ilsagfiles.org/SAG_files/Technical_Reference_Manual/Residential_Loadshapes_References.zip
http://ilsagfiles.org/SAG_files/Technical_Reference_Manual/Commercial_Loadshapes_References.zip
http://ilsagfiles.org/SAG_files/Technical_Reference_Manual/Version_3/Final_Draft/Sources%20and%20References%20-%20Loadshapes/TRM_Version_3_Loadshapes_2.24.zip

**Table 3.4: Loadshapes by Season**

| | | Winter Peak | Winter Off-peak | Summer Peak | Summer Off-peak |
|---|---|---|---|---|---|
| | Loadshape Reference Number | Oct-Apr, M-F, non-holiday, 8AM - 11PM | Oct-Apr, All other time | May-Sept, M-F, non-holiday, 8AM - 11PM | May- Sept, All other time |
| Residential Clothes Washer | R01 | 47.0% | 11.1% | 34.0% | 8.0% |
| Residential Dish Washer | R02 | 49.3% | 8.7% | 35.7% | 6.3% |
| Residential Electric DHW | R03 | 43.2% | 20.6% | 24.5% | 11.7% |
| Residential Freezer | R04 | 38.9% | 16.4% | 31.5% | 13.2% |
| Residential Refrigerator | R05 | 37.0% | 18.1% | 30.1% | 14.7% |
| Residential Indoor Lighting | R06 | 48.1% | 15.5% | 26.0% | 10.5% |
| Residential Outdoor Lighting | R07 | 18.0% | 44.1% | 9.4% | 28.4% |
| Residential Cooling | R08 | 4.1% | 0.7% | 71.3% | 23.9% |
| Residential Electric Space Heat | R09 | 57.8% | 38.8% | 1.7% | 1.7% |
| Residential Electric Heating and Cooling | R10 | 35.2% | 22.8% | 31.0% | 11.0% |
| Residential Ventilation | R11 | 25.8% | 32.3% | 18.9% | 23.0% |
| Residential - Dehumidifier | R12 | 12.9% | 16.2% | 31.7% | 39.2% |
| Residential Standby Losses - Entertainment Center | R13 | 26.0% | 32.5% | 18.9% | 22.6% |
| Residential Standby Losses - Home Office | R14 | 23.9% | 34.6% | 17.0% | 24.5% |
| | | | | | |
| Commercial Electric Cooking | C01 | 40.6% | 18.2% | 28.7% | 12.6% |
| Commercial Electric DHW | C02 | 40.5% | 18.2% | 28.5% | 12.8% |
| Commercial Cooling | C03 | 4.9% | 0.8% | 66.4% | 27.9% |
| Commercial Electric Heating | C04 | 53.5% | 43.2% | 1.9% | 1.4% |
| Commercial Electric Heating and Cooling | C05 | 19.4% | 13.5% | 47.1% | 19.9% |

| | | Winter Peak | Winter Off-peak | Summer Peak | Summer Off-peak |
|---|---|---|---|---|---|
| | Loadshape Reference Number | Oct-Apr, M-F, non-holiday, 8AM - 11PM | Oct-Apr, All other time | May-Sept, M-F, non-holiday, 8AM - 11PM | May- Sept, All other time |
| Commercial Indoor Lighting | C06 | 40.1% | 18.6% | 28.4% | 12.9% |
| Grocery/Conv. Store Indoor Lighting | C07 | 31.4% | 26.4% | 22.8% | 19.3% |
| Hospital Indoor Lighting | C08 | 29.1% | 29.0% | 21.0% | 20.9% |
| Office Indoor Lighting | C09 | 42.1% | 16.0% | 30.4% | 11.5% |
| Restaurant Indoor Lighting | C10 | 32.1% | 25.7% | 23.4% | 18.8% |
| Retail Indoor Lighting | C11 | 35.5% | 22.3% | 25.8% | 16.3% |
| Warehouse Indoor Lighting | C12 | 39.4% | 18.5% | 28.6% | 13.5% |
| K-12 School Indoor Lighting | C13 | 45.8% | 22.6% | 20.2% | 11.4% |
| Indust. 1-shift (8/5) (e.g., comp. air, lights) | C14 | 50.5% | 7.2% | 37.0% | 5.3% |
| Indust. 2-shift (16/5) (e.g., comp. air, lights) | C15 | 47.5% | 10.2% | 34.8% | 7.4% |
| Indust. 3-shift (24/5) (e.g., comp. air, lights) | C16 | 34.8% | 23.2% | 25.5% | 16.6% |
| Indust. 4-shift (24/7) (e.g., comp. air, lights) | C17 | 25.8% | 32.3% | 18.9% | 23.0% |
| Industrial Indoor Lighting | C18 | 44.3% | 13.6% | 32.4% | 9.8% |
| Industrial Outdoor Lighting | C19 | 18.0% | 44.1% | 9.4% | 28.4% |
| Commercial Outdoor Lighting | C20 | 23.4% | 35.3% | 13.0% | 28.3% |
| Commercial Office Equipment | C21 | 37.7% | 20.9% | 26.7% | 14.7% |
| Commercial Refrigeration | C22 | 38.5% | 20.6% | 26.7% | 14.2% |
| Commercial Ventilation | C23 | 38.1% | 20.6% | 29.7% | 11.6% |
| Traffic Signal - Red Balls, always changing or flashing | C24 | 25.8% | 32.3% | 18.9% | 23.0% |
| Traffic Signal - Red Balls, changing day, off night | C25 | 37.0% | 20.9% | 27.1% | 14.9% |
| Traffic Signal - Green Balls, always changing | C26 | 25.8% | 32.3% | 18.9% | 23.0% |

| | | Winter Peak | Winter Off-peak | Summer Peak | Summer Off-peak |
|---|---|---|---|---|---|
| | Loadshape Reference Number | Oct-Apr, M-F, non-holiday, 8AM - 11PM | Oct-Apr, All other time | May-Sept, M-F, non-holiday, 8AM - 11PM | May- Sept, All other time |
| Traffic Signal - Green Balls, changing day, off night | C27 | 37.0% | 20.9% | 27.1% | 14.9% |
| Traffic Signal - Red Arrows | C28 | 25.8% | 32.3% | 18.9% | 23.0% |
| Traffic Signal - Green Arrows | C29 | 25.8% | 32.3% | 18.9% | 23.0% |
| Traffic Signal - Flashing Yellows | C30 | 25.8% | 32.3% | 18.9% | 23.0% |
| Traffic Signal - “Hand” Don’t Walk Signal | C31 | 25.8% | 32.3% | 18.9% | 23.0% |
| Traffic Signal - “Man” Walk Signal | C32 | 25.8% | 32.3% | 18.9% | 23.0% |
| Traffic Signal - Bi-Modal Walk/Don’t Walk | C33 | 25.8% | 32.3% | 18.9% | 23.0% |
| Industrial Motor | C34 | 47.5% | 10.2% | 34.8% | 7.4% |
| Industrial Process | C35 | 47.5% | 10.2% | 34.8% | 7.4% |
| HVAC Pump Motor (heating) | C36 | 38.7% | 48.6% | 5.9% | 6.8% |
| HVAC Pump Motor (cooling) | C37 | 7.8% | 9.8% | 36.8% | 45.6% |
| HVAC Pump Motor (unknown use) | C38 | 23.2% | 29.2% | 21.4% | 26.2% |
| VFD - Supply fans <10 HP | C39 | 38.8% | 16.1% | 28.4% | 16.7% |
| VFD - Return fans <10 HP | C40 | 38.8% | 16.1% | 28.4% | 16.7% |
| VFD - Exhaust fans <10 HP | C41 | 34.8% | 23.2% | 20.3% | 21.7% |
| VFD - Boiler feedwater pumps <10 HP | C42 | 42.9% | 44.2% | 6.6% | 6.3% |
| VFD - Chilled water pumps <10 HP | C43 | 11.2% | 5.5% | 40.7% | 42.6% |
| VFD Boiler circulation pumps <10 HP | C44 | 42.9% | 44.2% | 6.6% | 6.3% |
| Refrigeration Economizer | C45 | 36.3% | 50.8% | 5.6% | 7.3% |
| Evaporator Fan Control | C46 | 24.0% | 35.9% | 16.7% | 23.4% |
| Standby Losses - Commercial Office | C47 | 8.2% | 50.5% | 5.6% | 35.7% |

| | | Winter Peak | Winter Off-peak | Summer Peak | Summer Off-peak |
|---|---|---|---|---|---|
| | Loadshape Reference Number | Oct-Apr, M-F, non-holiday, 8AM - 11PM | Oct-Apr, All other time | May-Sept, M-F, non-holiday, 8AM - 11PM | May- Sept, All other time |
| VFD Boiler draft fans <10 HP | C48 | 37.3% | 48.9% | 6.4% | 7.3% |
| VFD Cooling Tower Fans <10 HP | C49 | 7.9% | 5.2% | 54.0% | 32.9% |
| Engine Block Heater Timer | C50 | 26.5% | 61.0% | 4.1% | 8.5% |
| Door Heater Control | C51 | 30.4% | 69.6% | 0.0% | 0.0% |
| Beverage and Snack Machine Controls | C52 | 10.0% | 48.3% | 7.4% | 34.3% |
| Flat | C53 | 36.3% | 21.8% | 26.2% | 15.7% |
| Religious Indoor Lighting | C54 | 26.8% | 31.4% | 18.9% | 22.8% |

**Table 3.5: Loadshapes by Month and Day of Week**

| | | Jan | | Feb | | Mar | | Apr | | May | | Jun | | Jul | | Aug | | Sep | | Oct | | Nov | | Dec | |
|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|
| | | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S |
| Residential Clothes Washer | R01 | 7.0% | 1.6% | 6.3% | 1.5% | 6.6% | 1.7% | 6.7% | 1.5% | 6.9% | 1.6% | 6.5% | 1.6% | 7.1% | 1.5% | 6.8% | 1.7% | 6.6% | 1.6% | 7.0% | 1.5% | 6.5% | 1.7% | 6.9% | 1.6% |
| Residential Dish Washer | R02 | 7.3% | 1.2% | 6.6% | 1.2% | 7.0% | 1.4% | 7.1% | 1.2% | 7.3% | 1.2% | 6.9% | 1.3% | 7.4% | 1.2% | 7.1% | 1.3% | 7.0% | 1.2% | 7.4% | 1.2% | 6.8% | 1.3% | 7.2% | 1.3% |
| Residential Electric DHW | R03 | 6.4% | 2.9% | 5.8% | 2.7% | 6.1% | 3.3% | 6.2% | 2.8% | 5.0% | 2.3% | 4.7% | 2.4% | 5.1% | 2.2% | 4.9% | 2.5% | 4.8% | 2.3% | 6.5% | 2.8% | 6.0% | 3.1% | 6.3% | 3.0% |
| Residential Freezer | R04 | 5.8% | 2.3% | 5.2% | 2.2% | 5.5% | 2.6% | 5.6% | 2.2% | 6.4% | 2.6% | 6.1% | 2.7% | 6.6% | 2.5% | 6.3% | 2.8% | 6.1% | 2.6% | 5.8% | 2.2% | 5.4% | 2.4% | 5.7% | 2.4% |
| Residential Refrigerator | R05 | 5.5% | 2.6% | 4.9% | 2.4% | 5.2% | 2.9% | 5.3% | 2.5% | 6.2% | 2.9% | 5.8% | 3.0% | 6.3% | 2.8% | 6.0% | 3.1% | 5.9% | 2.9% | 5.5% | 2.5% | 5.1% | 2.7% | 5.4% | 2.6% |
| Residential Indoor Lighting | R06 | 7.1% | 2.2% | 6.4% | 2.1% | 6.8% | 2.4% | 6.9% | 2.1% | 5.3% | 2.1% | 5.0% | 2.2% | 5.4% | 2.0% | 5.2% | 2.2% | 5.1% | 2.1% | 7.2% | 2.1% | 6.6% | 2.3% | 7.0% | 2.2% |
| Residential Outdoor Lighting | R07 | 2.7% | 6.2% | 2.4% | 5.9% | 2.6% | 7.0% | 2.6% | 6.0% | 1.9% | 5.7% | 1.8% | 5.8% | 2.0% | 5.3% | 1.9% | 6.0% | 1.8% | 5.7% | 2.7% | 6.0% | 2.5% | 6.6% | 2.6% | 6.4% |
| Residential Cooling | R08 | 0.6% | 0.1% | 0.5% | 0.1% | 0.6% | 0.1% | 0.6% | 0.1% | 14.6% | 4.8% | 13.7% | 4.9% | 14.9% | 4.5% | 14.2% | 5.0% | 13.9% | 4.8% | 0.6% | 0.1% | 0.6% | 0.1% | 0.6% | 0.1% |
| Residential Electric Space Heat | R09 | 8.6% | 5.5% | 7.7% | 5.1% | 8.2% | 6.1% | 8.3% | 5.3% | 0.3% | 0.3% | 0.3% | 0.3% | 0.4% | 0.3% | 0.3% | 0.4% | 0.3% | 0.3% | 8.7% | 5.3% | 8.0% | 5.8% | 8.5% | 5.6% |
| Residential Electric Heating and Cooling | R10 | 5.2% | 3.2% | 4.7% | 3.0% | 5.0% | 3.6% | 5.0% | 3.1% | 6.3% | 2.2% | 6.0% | 2.3% | 6.5% | 2.1% | 6.2% | 2.3% | 6.0% | 2.2% | 5.3% | 3.1% | 4.9% | 3.4% | 5.2% | 3.3% |
| Residential Ventilation | R11 | 3.8% | 4.6% | 3.4% | 4.3% | 3.6% | 5.1% | 3.7% | 4.4% | 3.8% | 4.6% | 3.6% | 4.7% | 3.9% | 4.3% | 3.8% | 4.8% | 3.7% | 4.6% | 3.9% | 4.4% | 3.6% | 4.8% | 3.8% | 4.7% |
| Residential - Dehumidifier | R12 | 1.9% | 2.3% | 1.7% | 2.2% | 1.8% | 2.6% | 1.8% | 2.2% | 6.5% | 7.8% | 6.1% | 8.0% | 6.6% | 7.3% | 6.3% | 8.2% | 6.2% | 7.8% | 1.9% | 2.2% | 1.8% | 2.4% | 1.9% | 2.4% |
| Residential Standby Losses - Entertainment Center | R13 | 3.8% | 4.6% | 3.5% | 4.3% | 3.7% | 5.1% | 3.7% | 4.4% | 3.9% | 4.5% | 3.7% | 4.6% | 4.0% | 4.2% | 3.8% | 4.8% | 3.7% | 4.5% | 3.9% | 4.4% | 3.6% | 4.8% | 3.8% | 4.7% |
| Residential Standby | R14 | 3.5% | 4.9% | 3.2% | 4.6% | 3.4% | 5.5% | 3.4% | 4.7% | 3.5% | 4.9% | 3.3% | 5.0% | 3.5% | 4.6% | 3.4% | 5.2% | 3.3% | 4.9% | 3.6% | 4.7% | 3.3% | 5.2% | 3.5% | 5.0% |

| | | Jan | | Feb | | Mar | | Apr | | May | | Jun | | Jul | | Aug | | Sep | | Oct | | Nov | | Dec | |
|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|
| | | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S |
| Losses - Home Office | | | | | | | | | | | | | | | | | | | | | | | | | |
| | | | | | | | | | | | | | | | | | | | | | | | | | |
| Commercial Electric Cooking | C01 | 6.0% | 2.6% | 5.4% | 2.4% | 5.7% | 2.9% | 5.8% | 2.5% | 5.9% | 2.5% | 5.5% | 2.6% | 6.0% | 2.4% | 5.7% | 2.6% | 5.6% | 2.5% | 6.1% | 2.5% | 5.6% | 2.7% | 5.9% | 2.6% |
| Commercial Electric DHW | C02 | 6.0% | 2.6% | 5.4% | 2.4% | 5.7% | 2.9% | 5.8% | 2.5% | 5.8% | 2.5% | 5.5% | 2.6% | 6.0% | 2.4% | 5.7% | 2.7% | 5.6% | 2.5% | 6.1% | 2.5% | 5.6% | 2.7% | 5.9% | 2.6% |
| Commercial Cooling | C03 | 0.7% | 0.1% | 0.6% | 0.1% | 0.7% | 0.1% | 0.7% | 0.1% | 13.6% | 5.5% | 12.8% | 5.7% | 13.9% | 5.2% | 13.3% | 5.9% | 13.0% | 5.5% | 0.7% | 0.1% | 0.7% | 0.1% | 0.7% | 0.1% |
| Commercial Electric Heating | C04 | 7.9% | 6.1% | 7.1% | 5.7% | 7.6% | 6.8% | 7.7% | 5.9% | 0.4% | 0.3% | 0.4% | 0.3% | 0.4% | 0.3% | 0.4% | 0.3% | 0.4% | 0.3% | 8.0% | 5.9% | 7.4% | 6.5% | 7.8% | 6.3% |
| Commercial Electric Heating and Cooling | C05 | 2.9% | 1.9% | 2.6% | 1.8% | 2.8% | 2.1% | 2.8% | 1.9% | 9.6% | 4.0% | 9.1% | 4.1% | 9.8% | 3.7% | 9.4% | 4.2% | 9.2% | 4.0% | 2.9% | 1.9% | 2.7% | 2.0% | 2.8% | 2.0% |
| Commercial Indoor Lighting | C06 | 5.9% | 2.6% | 5.3% | 2.5% | 5.7% | 2.9% | 5.7% | 2.6% | 5.8% | 2.6% | 5.5% | 2.6% | 5.9% | 2.4% | 5.7% | 2.7% | 5.5% | 2.6% | 6.0% | 2.6% | 5.5% | 2.8% | 5.9% | 2.7% |
| Grocery/Conv. Store Indoor Lighting | C07 | 4.7% | 3.7% | 4.2% | 3.5% | 4.4% | 4.2% | 4.5% | 3.6% | 4.7% | 3.8% | 4.4% | 3.9% | 4.8% | 3.6% | 4.6% | 4.1% | 4.5% | 3.8% | 4.7% | 3.6% | 4.3% | 3.9% | 4.6% | 3.8% |
| Hospital Indoor Lighting | C08 | 4.3% | 4.1% | 3.9% | 3.8% | 4.1% | 4.6% | 4.2% | 4.0% | 4.3% | 4.2% | 4.0% | 4.3% | 4.4% | 3.9% | 4.2% | 4.4% | 4.1% | 4.2% | 4.4% | 4.0% | 4.0% | 4.3% | 4.3% | 4.2% |
| Office Indoor Lighting | C09 | 6.2% | 2.3% | 5.6% | 2.1% | 6.0% | 2.5% | 6.0% | 2.2% | 6.2% | 2.3% | 5.9% | 2.4% | 6.4% | 2.2% | 6.1% | 2.4% | 5.9% | 2.3% | 6.3% | 2.2% | 5.8% | 2.4% | 6.2% | 2.3% |
| Restaurant Indoor Lighting | C10 | 4.8% | 3.6% | 4.3% | 3.4% | 4.5% | 4.1% | 4.6% | 3.5% | 4.8% | 3.7% | 4.5% | 3.8% | 4.9% | 3.5% | 4.7% | 4.0% | 4.6% | 3.7% | 4.8% | 3.5% | 4.4% | 3.8% | 4.7% | 3.7% |
| Retail Indoor Lighting | C11 | 5.3% | 3.1% | 4.7% | 3.0% | 5.0% | 3.5% | 5.1% | 3.1% | 5.3% | 3.2% | 5.0% | 3.3% | 5.4% | 3.1% | 5.2% | 3.4% | 5.0% | 3.2% | 5.3% | 3.1% | 4.9% | 3.3% | 5.2% | 3.2% |
| Warehouse Indoor Lighting | C12 | 5.8% | 2.6% | 5.2% | 2.5% | 5.6% | 2.9% | 5.6% | 2.5% | 5.8% | 2.7% | 5.5% | 2.8% | 6.0% | 2.5% | 5.7% | 2.8% | 5.6% | 2.7% | 5.9% | 2.5% | 5.4% | 2.8% | 5.8% | 2.7% |
| K-12 School Indoor Lighting | C13 | 6.8% | 3.2% | 6.1% | 3.0% | 6.5% | 3.6% | 6.6% | 3.1% | 4.1% | 2.3% | 3.9% | 2.3% | 4.2% | 2.1% | 4.0% | 2.4% | 3.9% | 2.3% | 6.9% | 3.1% | 6.3% | 3.4% | 6.7% | 3.3% |

| | | Jan | | Feb | | Mar | | Apr | | May | | Jun | | Jul | | Aug | | Sep | | Oct | | Nov | | Dec | |
|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|
| | | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S |
| Indust. 1-shift (8/5) (e.g., comp. air, lights) | C14 | 7.5% | 1.0% | 6.7% | 1.0% | 7.1% | 1.1% | 7.2% | 1.0% | 7.5% | 1.1% | 7.1% | 1.1% | 7.7% | 1.0% | 7.4% | 1.1% | 7.2% | 1.1% | 7.6% | 1.0% | 7.0% | 1.1% | 7.4% | 1.0% |
| Indust. 2-shift (16/5) (e.g., comp. air, lights) | C15 | 7.0% | 1.4% | 6.3% | 1.4% | 6.7% | 1.6% | 6.8% | 1.4% | 7.1% | 1.5% | 6.7% | 1.5% | 7.3% | 1.4% | 6.9% | 1.6% | 6.8% | 1.5% | 7.1% | 1.4% | 6.6% | 1.5% | 7.0% | 1.5% |
| Indust. 3-shift (24/5) (e.g., comp. air, lights) | C16 | 5.1% | 3.3% | 4.6% | 3.1% | 4.9% | 3.7% | 5.0% | 3.2% | 5.2% | 3.3% | 4.9% | 3.4% | 5.3% | 3.1% | 5.1% | 3.5% | 5.0% | 3.3% | 5.2% | 3.2% | 4.8% | 3.5% | 5.1% | 3.4% |
| Indust. 4-shift (24/7) (e.g., comp. air, lights) | C17 | 3.8% | 4.6% | 3.4% | 4.3% | 3.6% | 5.1% | 3.7% | 4.4% | 3.8% | 4.6% | 3.6% | 4.7% | 3.9% | 4.3% | 3.8% | 4.8% | 3.7% | 4.6% | 3.9% | 4.4% | 3.6% | 4.8% | 3.8% | 4.7% |
| Industrial Indoor Lighting | C18 | 6.6% | 1.9% | 5.9% | 1.8% | 6.3% | 2.1% | 6.3% | 1.9% | 6.6% | 1.9% | 6.2% | 2.0% | 6.8% | 1.8% | 6.5% | 2.0% | 6.3% | 1.9% | 6.6% | 1.9% | 6.1% | 2.0% | 6.5% | 2.0% |
| Industrial Outdoor Lighting | C19 | 2.7% | 6.2% | 2.4% | 5.9% | 2.6% | 7.0% | 2.6% | 6.0% | 1.9% | 5.7% | 1.8% | 5.8% | 2.0% | 5.3% | 1.9% | 6.0% | 1.8% | 5.7% | 2.7% | 6.0% | 2.5% | 6.6% | 2.6% | 6.4% |
| Commercial Outdoor Lighting | C20 | 3.5% | 5.0% | 3.1% | 4.7% | 3.3% | 5.6% | 3.3% | 4.8% | 2.7% | 5.6% | 2.5% | 5.8% | 2.7% | 5.3% | 2.6% | 5.9% | 2.5% | 5.6% | 3.5% | 4.8% | 3.2% | 5.3% | 3.4% | 5.1% |
| Commercial Office Equipment | C21 | 5.6% | 3.0% | 5.0% | 2.8% | 5.3% | 3.3% | 5.4% | 2.9% | 5.4% | 2.9% | 5.1% | 3.0% | 5.6% | 2.7% | 5.3% | 3.1% | 5.2% | 2.9% | 5.6% | 2.9% | 5.2% | 3.1% | 5.5% | 3.0% |
| Commercial Refrigeration | C22 | 5.7% | 2.9% | 5.1% | 2.7% | 5.4% | 3.2% | 5.5% | 2.8% | 5.5% | 2.8% | 5.1% | 2.9% | 5.6% | 2.7% | 5.3% | 3.0% | 5.2% | 2.8% | 5.8% | 2.8% | 5.3% | 3.1% | 5.6% | 3.0% |
| Commercial Ventilation | C23 | 5.6% | 2.9% | 5.1% | 2.7% | 5.4% | 3.3% | 5.4% | 2.8% | 6.1% | 2.3% | 5.7% | 2.4% | 6.2% | 2.2% | 5.9% | 2.4% | 5.8% | 2.3% | 5.7% | 2.8% | 5.3% | 3.1% | 5.6% | 3.0% |
| Traffic Signal - Red Balls, always changing or flashing | C24 | 3.8% | 4.6% | 3.4% | 4.3% | 3.6% | 5.1% | 3.7% | 4.4% | 3.8% | 4.6% | 3.6% | 4.7% | 3.9% | 4.3% | 3.8% | 4.8% | 3.7% | 4.6% | 3.9% | 4.4% | 3.6% | 4.8% | 3.8% | 4.7% |
| Traffic Signal - Red Balls, | C25 | 5.5% | 2.9% | 4.9% | 2.8% | 5.2% | 3.3% | 5.3% | 2.9% | 5.5% | 3.0% | 5.2% | 3.1% | 5.7% | 2.8% | 5.4% | 3.1% | 5.3% | 3.0% | 5.5% | 2.9% | 5.1% | 3.1% | 5.4% | 3.0% |

| | | Jan | | Feb | | Mar | | Apr | | May | | Jun | | Jul | | Aug | | Sep | | Oct | | Nov | | Dec | |
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| | | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S |
| changing day, off night | | | | | | | | | | | | | | | | | | | | | | | | | |
| Traffic Signal - Green Balls, always changing | C26 | 3.8% | 4.6% | 3.4% | 4.3% | 3.6% | 5.1% | 3.7% | 4.4% | 3.8% | 4.6% | 3.6% | 4.7% | 3.9% | 4.3% | 3.8% | 4.8% | 3.7% | 4.6% | 3.9% | 4.4% | 3.6% | 4.8% | 3.8% | 4.7% |
| Traffic Signal - Green Balls, changing day, off night | C27 | 5.5% | 2.9% | 4.9% | 2.8% | 5.2% | 3.3% | 5.3% | 2.9% | 5.5% | 3.0% | 5.2% | 3.1% | 5.7% | 2.8% | 5.4% | 3.1% | 5.3% | 3.0% | 5.5% | 2.9% | 5.1% | 3.1% | 5.4% | 3.0% |
| Traffic Signal - Red Arrows | C28 | 3.8% | 4.6% | 3.4% | 4.3% | 3.6% | 5.1% | 3.7% | 4.4% | 3.8% | 4.6% | 3.6% | 4.7% | 3.9% | 4.3% | 3.8% | 4.8% | 3.7% | 4.6% | 3.9% | 4.4% | 3.6% | 4.8% | 3.8% | 4.7% |
| Traffic Signal - Green Arrows | C29 | 3.8% | 4.6% | 3.4% | 4.3% | 3.6% | 5.1% | 3.7% | 4.4% | 3.8% | 4.6% | 3.6% | 4.7% | 3.9% | 4.3% | 3.8% | 4.8% | 3.7% | 4.6% | 3.9% | 4.4% | 3.6% | 4.8% | 3.8% | 4.7% |
| Traffic Signal - Flashing Yellows | C30 | 3.8% | 4.6% | 3.4% | 4.3% | 3.6% | 5.1% | 3.7% | 4.4% | 3.8% | 4.6% | 3.6% | 4.7% | 3.9% | 4.3% | 3.8% | 4.8% | 3.7% | 4.6% | 3.9% | 4.4% | 3.6% | 4.8% | 3.8% | 4.7% |
| Traffic Signal - “Hand” Don’t Walk Signal | C31 | 3.8% | 4.6% | 3.4% | 4.3% | 3.6% | 5.1% | 3.7% | 4.4% | 3.8% | 4.6% | 3.6% | 4.7% | 3.9% | 4.3% | 3.8% | 4.8% | 3.7% | 4.6% | 3.9% | 4.4% | 3.6% | 4.8% | 3.8% | 4.7% |
| Traffic Signal - “Man” Walk Signal | C32 | 3.8% | 4.6% | 3.4% | 4.3% | 3.6% | 5.1% | 3.7% | 4.4% | 3.8% | 4.6% | 3.6% | 4.7% | 3.9% | 4.3% | 3.8% | 4.8% | 3.7% | 4.6% | 3.9% | 4.4% | 3.6% | 4.8% | 3.8% | 4.7% |
| Traffic Signal - Bi-Modal Walk/Don’t Walk | C33 | 3.8% | 4.6% | 3.4% | 4.3% | 3.6% | 5.1% | 3.7% | 4.4% | 3.8% | 4.6% | 3.6% | 4.7% | 3.9% | 4.3% | 3.8% | 4.8% | 3.7% | 4.6% | 3.9% | 4.4% | 3.6% | 4.8% | 3.8% | 4.7% |
| Industrial Motor | C34 | 7.0% | 1.4% | 6.3% | 1.4% | 6.7% | 1.6% | 6.8% | 1.4% | 7.1% | 1.5% | 6.7% | 1.5% | 7.3% | 1.4% | 6.9% | 1.6% | 6.8% | 1.5% | 7.1% | 1.4% | 6.6% | 1.5% | 7.0% | 1.5% |
| Industrial Process | C35 | 7.0% | 1.4% | 6.3% | 1.4% | 6.7% | 1.6% | 6.8% | 1.4% | 7.1% | 1.5% | 6.7% | 1.5% | 7.3% | 1.4% | 6.9% | 1.6% | 6.8% | 1.5% | 7.1% | 1.4% | 6.6% | 1.5% | 7.0% | 1.5% |
| HVAC Pump Motor (heating) | C36 | 5.7% | 6.9% | 5.2% | 6.4% | 5.5% | 7.7% | 5.5% | 6.6% | 1.2% | 1.4% | 1.1% | 1.4% | 1.2% | 1.3% | 1.2% | 1.4% | 1.2% | 1.4% | 5.8% | 6.6% | 5.3% | 7.3% | 5.7% | 7.1% |
| HVAC Pump Motor (cooling) | C37 | 1.2% | 1.4% | 1.0% | 1.3% | 1.1% | 1.5% | 1.1% | 1.3% | 7.5% | 9.1% | 7.1% | 9.3% | 7.7% | 8.5% | 7.3% | 9.6% | 7.2% | 9.1% | 1.2% | 1.3% | 1.1% | 1.5% | 1.1% | 1.4% |
| HVAC Pump Motor | C38 | 3.4% | 4.1% | 3.1% | 3.9% | 3.3% | 4.6% | 3.3% | 4.0% | 4.4% | 5.2% | 4.1% | 5.4% | 4.5% | 4.9% | 4.3% | 5.5% | 4.2% | 5.2% | 3.5% | 4.0% | 3.2% | 4.4% | 3.4% | 4.2% |

| | | Jan | | Feb | | Mar | | Apr | | May | | Jun | | Jul | | Aug | | Sep | | Oct | | Nov | | Dec | |
|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|
| | | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S |
| (unknown use) | | | | | | | | | | | | | | | | | | | | | | | | | |
| VFD - Supply fans <10 HP | C39 | 5.7% | 2.3% | 5.2% | 2.1% | 5.5% | 2.5% | 5.6% | 2.2% | 5.8% | 3.3% | 5.5% | 3.4% | 5.9% | 3.1% | 5.7% | 3.5% | 5.5% | 3.3% | 5.8% | 2.2% | 5.4% | 2.4% | 5.7% | 2.3% |
| VFD - Return fans <10 HP | C40 | 5.7% | 2.3% | 5.2% | 2.1% | 5.5% | 2.5% | 5.6% | 2.2% | 5.8% | 3.3% | 5.5% | 3.4% | 5.9% | 3.1% | 5.7% | 3.5% | 5.5% | 3.3% | 5.8% | 2.2% | 5.4% | 2.4% | 5.7% | 2.3% |
| VFD - Exhaust fans <10 HP | C41 | 5.1% | 3.3% | 4.6% | 3.1% | 4.9% | 3.7% | 5.0% | 3.2% | 4.1% | 4.3% | 3.9% | 4.4% | 4.2% | 4.1% | 4.1% | 4.6% | 4.0% | 4.3% | 5.2% | 3.2% | 4.8% | 3.5% | 5.1% | 3.4% |
| VFD - Boiler feedwater pumps <10 HP | C42 | 6.4% | 6.2% | 5.7% | 5.9% | 6.1% | 7.0% | 6.1% | 6.0% | 1.3% | 1.3% | 1.3% | 1.3% | 1.4% | 1.2% | 1.3% | 1.3% | 1.3% | 1.3% | 6.4% | 6.0% | 5.9% | 6.6% | 6.3% | 6.4% |
| VFD - Chilled water pumps <10 HP | C43 | 1.7% | 0.8% | 1.5% | 0.7% | 1.6% | 0.9% | 1.6% | 0.8% | 8.3% | 8.5% | 7.8% | 8.7% | 8.5% | 8.0% | 8.1% | 8.9% | 7.9% | 8.5% | 1.7% | 0.8% | 1.6% | 0.8% | 1.6% | 0.8% |
| VFD Boiler circulation pumps <10 HP | C44 | 6.4% | 6.2% | 5.7% | 5.9% | 6.1% | 7.0% | 6.1% | 6.0% | 1.3% | 1.3% | 1.3% | 1.3% | 1.4% | 1.2% | 1.3% | 1.3% | 1.3% | 1.3% | 6.4% | 6.0% | 5.9% | 6.6% | 6.3% | 6.4% |
| Refrigeration Economizer | C45 | 5.4% | 7.2% | 4.8% | 6.7% | 5.1% | 8.0% | 5.2% | 7.0% | 1.1% | 1.5% | 1.1% | 1.5% | 1.2% | 1.4% | 1.1% | 1.5% | 1.1% | 1.5% | 5.4% | 7.0% | 5.0% | 7.6% | 5.3% | 7.4% |
| Evaporator Fan Control | C46 | 3.6% | 5.1% | 3.2% | 4.8% | 3.4% | 5.7% | 3.4% | 4.9% | 3.4% | 4.7% | 3.2% | 4.8% | 3.5% | 4.4% | 3.3% | 4.9% | 3.3% | 4.7% | 3.6% | 4.9% | 3.3% | 5.4% | 3.5% | 5.2% |
| Standby Losses - Commercial Office | C47 | 1.2% | 7.1% | 1.1% | 6.7% | 1.2% | 8.0% | 1.2% | 6.9% | 1.1% | 7.1% | 1.1% | 7.3% | 1.2% | 6.7% | 1.1% | 7.5% | 1.1% | 7.1% | 1.2% | 6.9% | 1.1% | 7.5% | 1.2% | 7.3% |
| VFD Boiler draft fans <10 HP | C48 | 5.5% | 6.9% | 5.0% | 6.5% | 5.3% | 7.7% | 5.3% | 6.7% | 1.3% | 1.5% | 1.2% | 1.5% | 1.3% | 1.4% | 1.3% | 1.5% | 1.2% | 1.5% | 5.6% | 6.7% | 5.2% | 7.3% | 5.5% | 7.1% |
| VFD Cooling Tower Fans <10 HP | C49 | 1.2% | 0.7% | 1.1% | 0.7% | 1.1% | 0.8% | 1.1% | 0.7% | 11.0% | 6.5% | 10.4% | 6.7% | 11.3% | 6.2% | 10.8% | 6.9% | 10.5% | 6.5% | 1.2% | 0.7% | 1.1% | 0.8% | 1.2% | 0.8% |
| Engine Block Heater Timer | C50 | 3.9% | 8.6% | 3.5% | 8.1% | 3.7% | 9.6% | 3.8% | 8.3% | 0.8% | 1.7% | 0.8% | 1.7% | 0.8% | 1.6% | 0.8% | 1.8% | 0.8% | 1.7% | 4.0% | 8.3% | 3.7% | 9.1% | 3.9% | 8.9% |
| Door Heater Control | C51 | 4.5% | 9.8% | 4.0% | 9.2% | 4.3% | 11.0% | 4.3% | 9.5% | 0.0% | 0.0% | 0.0% | 0.0% | 0.0% | 0.0% | 0.0% | 0.0% | 0.0% | 0.0% | 4.5% | 9.5% | 4.2% | 10.4% | 4.4% | 10.1 % |
| Beverage and Snack | C52 | 1.5% | 6.8% | 1.3% | 6.4% | 1.4% | 7.6% | 1.4% | 6.6% | 1.5% | 6.8% | 1.4% | 7.0% | 1.5% | 6.4% | 1.5% | 7.2% | 1.4% | 6.8% | 1.5% | 6.6% | 1.4% | 7.2% | 1.5% | 7.0% |

| | | Jan | | Feb | | Mar | | Apr | | May | | Jun | | Jul | | Aug | | Sep | | Oct | | Nov | | Dec | |
|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|
| | | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S | M-F | S-S |
| Machine Controls | | | | | | | | | | | | | | | | | | | | | | | | | |
| Flat | C53 | 5.4% | 3.1% | 4.8% | 2.9% | 5.1% | 3.4% | 5.2% | 3.0% | 5.3% | 3.1% | 5.0% | 3.2% | 5.5% | 2.9% | 5.2% | 3.3% | 5.1% | 3.1% | 5.4% | 3.0% | 5.0% | 3.3% | 5.3% | 3.2% |
| Religious Indoor Lighting | C54 | 4.0% | 4.4% | 3.6% | 4.2% | 3.8% | 5.0% | 3.8% | 4.3% | 3.9% | 4.5% | 3.6% | 4.7% | 3.9% | 4.3% | 3.8% | 4.8% | 3.7% | 4.5% | 4.0% | 4.3% | 3.7% | 4.7% | 3.9% | 4.6% |

## 3.6 Summer Peak Period Definition (kW)

To estimate the impact that an efficiency measure has on a utility's system peak, the peak itself needs to be defined. Illinois spans two different electrical control areas, the Pennsylvania – Jersey – Maryland (PJM) and the Midwest Independent System Operators (MISO). As a result, there is some disparity in the peak definition across the state. However, only PJM has a forward capacity market where an efficiency program can potentially participate. Because ComEd is part of the PJM control area, their definition of summer peak is being applied statewide in this TRM.

Because Illinois is a summer peaking state, only the summer peak period is defined for the purpose of this TRM. The coincident summer peak period is defined as 1:00-5:00 PM Central Prevailing Time on non-holiday weekdays, June through August.

Summer peak coincidence factors can be found within each measure characterization. The source is provided and is based upon evaluation results, analysis of load shape data (e.g., the Itron eShapes data provided by Ameren), or through a calculation using stated assumptions.

For measures that are not weather-sensitive, the summer peak coincidence factor is estimated whenever possible as the average of savings within the peak period defined above. For weather sensitive measures such as cooling, the summer peak coincidence factor is provided in two different ways. The first method is to estimate demand savings during the utility's peak hour (as provided by Ameren). This is likely to be the most indicative of actual peak benefits. The second way represents the average savings over the summer peak period, consistent with the non-weather sensitive end uses, and is presented so that savings can be bid into PJM's Forward Capacity Market.

## 3.7 Heating and Cooling Degree-Day Data

Many measures are weather sensitive. Because there is a range of climactic conditions across the state, VEIC engaged the Utilities to provide their preferences for what airports and cities are the best proxies for the weather in their service territories. The result of this engagement is in the table below. All of the data represents 30-year normals[27] from the National Climactic Data Center (NCDC). Note that the base temperature for the calculation of heating degree-days in this document does not follow the historical 65F degree base temperature convention. Instead VEIC used several different temperatures in this TRM to more accurately reflect the outdoor temperature when a heating or cooling system turns on.

Residential heating is based on 60F, in accordance with regression analysis of heating fuel use and weather by state by the Pacific Northwest National Laboratory[28]. Residential cooling is based on 65F in agreement with a field study in Wisconsin[29]. These are lower than typical thermostat set points because internal gains such as appliances, lighting, and people provide some heating. In C&I settings, internal gains are often much higher; the base temperatures for both heating and cooling is 55F[30]. Custom degree-days with building specific base temperatures are recommended for large C&I projects.

[27] 30-year normals have been used instead of Typical Meteorological Year (TMY) data due to the fact that few of the measures in the TRM are significantly affected by solar insolation, which is one of the primary benefits of using the TMY approach.
[28] Belzer and Cort, Pacific Northwest National Laboratory in "Statistical Analysis of Historical State-Level Residential Energy Consumption Trends," 2004.
[29] Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research", p. 32 (amended in 2010).
[30] This value is based upon experience, and it is preferable to use building-specific base temperatures when available.

**Table 3.6: Degree-Day Zones and Values by Market Sector**

| | Residential | | C&I | | |
|---|---|---|---|---|---|
| Zone | HDD | CDD | HDD | CDD | Weather Station / City |
| 1 | 5,352 | 820 | 4,272 | 2,173 | Rockford AP / Rockford |
| 2 | 5,113 | 842 | 4,029 | 3,357 | Chicago O'Hare AP / Chicago |
| 3 | 4,379 | 1,108 | 3,406 | 2,666 | Springfield #2 / Springfield |
| 4 | 3,378 | 1,570 | 2,515 | 3,090 | Belleville SIU RSCH / Belleville |
| 5 | 3,438 | 1,370 | 2,546 | 2,182 | Carbondale Southern IL AP / Marion |
| Average | 4,860 | 947 | 3,812 | 3,051 | Weighted by occupied housing units |
| Base Temp | 60F | 65F | 55F | 55F | 30 year climate normals, 1981-2010 |

This table assigns each of the proxy cities to one of five climate zones. The following graphics from the Illinois State Water Survey show isobars (lines of equal degree-days) and we have color-coded the counties in each of these graphics using those isobars as a dividing line. Using this approach, the state divides into five cooling degree-day zones and five heating degree-day zones. Note that although the heating and cooling degree-day maps are similar, they are not the same, and the result is that there are a total of 10 climate zones in the state. The counties are listed in the tables following the figures for ease of reference.

**Figure 3.1: Cooling Degree-Day Zones by County**

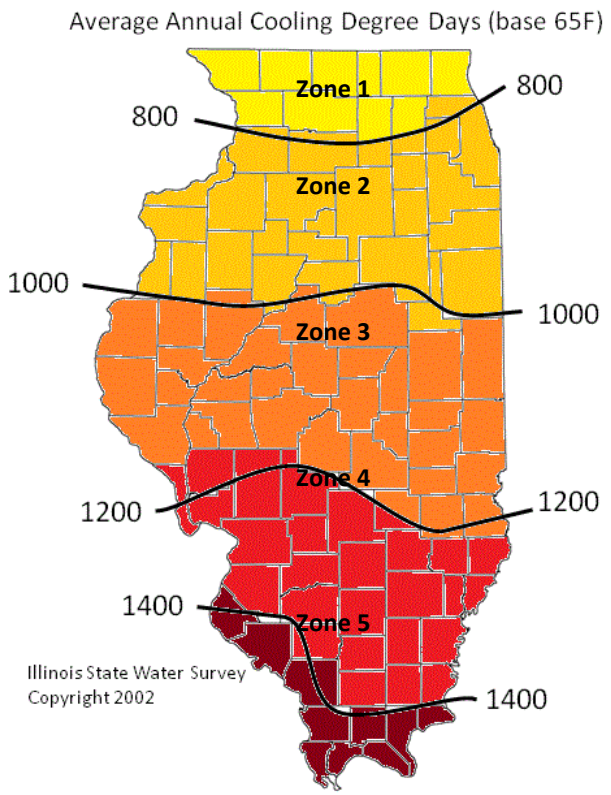

**Figure 3.2: Heating Degree-Day Zones by County**

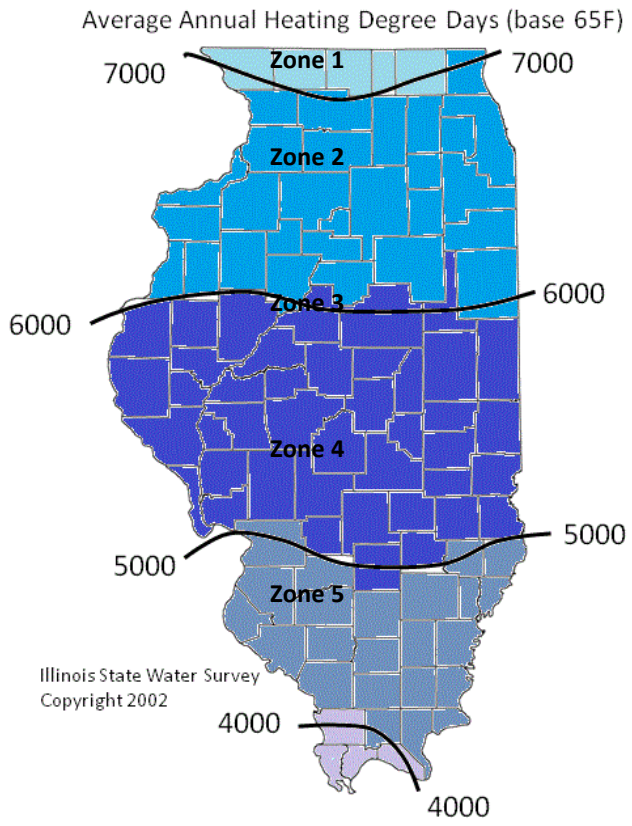

**Table 3.7: Heating Degree-Day Zones by County**

| Zone 1 | Zone 2 | Zone 3 | Zone 4 | Zone 5 |
|---|---|---|---|---|
| Boone County | Bureau County | Adams County | Clinton County | Alexander County |
| Jo Daviess County | Carroll County | Bond County | Edwards County | Massac County |
| Stephenson County | Cook County | Brown County | Franklin County | Pulaski County |
| Winnebago County | DeKalb County | Calhoun County | Gallatin County | Union County |
| | DuPage County | Cass County | Hamilton County | |
| | Grundy County | Champaign County | Hardin County | |
| | Henderson County | Christian County | Jackson County | |
| | Henry County | Clark County | Jefferson County | |
| | Iroquois County | Clay County | Johnson County | |
| | Kane County | Coles County | Lawrence County | |
| | Kankakee County | Crawford County | Madison County | |
| | Kendall County | Cumberland County | Marion County | |
| | Knox County | De Witt County | Monroe County | |
| | Lake County | Douglas County | Perry County | |
| | LaSalle County | Edgar County | Pope County | |
| | Lee County | Effingham County | Randolph County | |
| | Livingston County | Fayette County | Richland County | |
| | Marshall County | Ford County | Saline County | |
| | McHenry County | Fulton County | St. Clair County | |
| | Mercer County | Greene County | Wabash County | |
| | Ogle County | Hancock County | Washington County | |
| | Peoria County | Jasper County | Wayne County | |
| | Putnam County | Jersey County | White County | |
| | Rock Island County | Logan County | Williamson County | |
| | Stark County | Macon County | | |
| | Warren County | Macoupin County | | |
| | Whiteside County | Mason County | | |
| | Will County | McDonough County | | |
| | Woodford County | McLean County | | |
| | | Menard County | | |
| | | Montgomery | | |
| | | Morgan County | | |
| | | Moultrie County | | |
| | | Piatt County | | |
| | | Pike County | | |
| | | Sangamon County | | |
| | | Schuyler County | | |
| | | Scott County | | |
| | | Shelby County | | |
| | | Tazewell County | | |
| | | Vermilion County | | |

**Table 3.8: Cooling Degree-day Zones by County**

| Zone 1 | Zone 2 | Zone 3 | Zone 4 | Zone 5 |
|---|---|---|---|---|
| Boone County | Bureau County | Adams County | Bond County | Alexander County |
| Carroll County | Cook County | Brown County | Clay County | Hardin County |
| DeKalb County | DuPage County | Calhoun County | Clinton County | Johnson County |
| Jo Daviess County | Grundy County | Cass County | Edwards County | Massac County |
| Kane County | Henderson County | Champaign County | Fayette County | Pope County |
| Lake County | Henry County | Christian County | Franklin County | Pulaski County |
| McHenry County | Iroquois County | Clark County | Gallatin County | Randolph County |
| Ogle County | Kankakee County | Coles County | Hamilton County | Union County |
| Stephenson County | Kendall County | Crawford County | Jackson County | |
| Winnebago County | Knox County | Cumberland County | Jefferson County | |
| | LaSalle County | De Witt County | Jersey County | |
| | Lee County | Douglas County | Lawrence County | |
| | Livingston County | Edgar County | Macoupin County | |
| | Marshall County | Effingham County | Madison County | |
| | Mercer County | Ford County | Marion County | |
| | Peoria County | Fulton County | Monroe County | |
| | Putnam County | Greene County | Montgomery | |
| | Rock Island County | Hancock County | Perry County | |
| | Stark County | Jasper County | Richland County | |
| | Warren County | Logan County | Saline County | |
| | Whiteside County | Macon County | St. Clair County | |
| | Will County | Mason County | Wabash County | |
| | Woodford County | McDonough County | Washington County | |
| | | McLean County | Wayne County | |
| | | Menard County | White County | |
| | | Morgan County | Williamson County | |
| | | Moultrie County | | |
| | | Piatt County | | |
| | | Pike County | | |
| | | Sangamon County | | |
| | | Schuyler County | | |
| | | Scott County | | |
| | | Shelby County | | |
| | | Tazewell County | | |
| | | Vermilion County | | |

## 3.8 O&M Costs and the Weighted Average Cost of Capital (WACC)

Some measures specify an operations and maintenance (O&M) parameter that describes the incremental O&M cost savings that can be expected over the measure's lifetime. When estimating the cost effectiveness of these measures, it is necessary to calculate the net present value (NPV) of O&M costs over the life of the measure, which requires an appropriate discount rate. The utility's weighted average cost of capital (WACC) is the most commonly used discount rate that is used in this context.

Each utility has a unique WACC that will vary over time. As a result, the TRM does not specify the NPV of the O&M costs. Instead, the necessary information required to calculate the NPV is included. An example is provided below to demonstrate how to calculate the NPV of O&M costs.

**EXAMPLE**

Baseline Case: O&M costs equal $150 every two years.

Efficient Case: O&M costs equal $50 every five years.

Given this information, the incremental O&M costs can be determined by discounting the cash flows in the Baseline Case and the Efficient Case separately using the applicable WACC. Then the NPV of the incremental O&M costs is calculated by subtracting one NPV from the other. This value is then used in each utility's cost-effectiveness screening process.

Those measures that include baseline shifts that result in multiple component costs and lifetimes cannot be calculated by this standard method. In only these cases, the O&M costs are presented both as Annual Levelized equivalent cost (i.e., the annual payment that results in an equivalent NPV to the actual stream of O&M costs) and as NPVs using a statewide average real discount rate of 5.23%.

## 3.9 Interactive Effects

The TRM presents engineering equations for most measures. This approach is desirable because it conveys information clearly and transparently, and is widely accepted in the industry. Unlike simulation model results, engineering equations also provide flexibility and the opportunity for users to substitute local, specific information for specific input values. Furthermore, the parameters can be changed in TRM updates to be applied in future years as better information becomes available.

One limitation is that some interactive effects between measures are not automatically captured. Because we cannot know what measures will be implemented at the same time with the same customer, we cannot always capture the interactions between multiple measures within individual measure characterizations. However, interactive effects with different end-uses are included in individual measure characterizations whenever possible[31]. For instance, waste heat factors are included in the lighting characterizations to capture the interaction between more-efficient lighting measures and the amount of heating and/or cooling that is subsequently needed in the building.

By contrast, no effort is made to account for interactive effects between an efficient air conditioning measure and an efficient lighting measure, because it is impossible to know the specifics of the other measure in advance of its installation. For custom measures and projects where a bundle of measures is being implemented at the same time, these kinds of interactive effects should be estimated.

[31] For more information, please refer to the document, 'Dealing with interactive Effects During Measure Characterization" Memo to the Stakeholder Advisory Group dated 12/9/11.

# 4 Commercial and Industrial Measures

## 4.1 Agricultural End Use

### 4.1.1 Engine Block Timer for Agricultural Equipment

**DESCRIPTION**

The measure is a plug-in timer that is activated below a specific outdoor temperature to control an engine block heater in agricultural equipment. Engine block heaters are typically used during cold weather to pre-warm an engine prior to start, for convenience heaters are typically plugged in considerably longer than necessary to improve startup performance. A timer allows a user to preset the heater to come on for only the amount of time necessary to pre-warm the engine block, reducing unnecessary run time even if the baseline equipment has an engine block temperature sensor.

This measure was developed to be applicable to the following program types: RF. If applied to other program types, the measure savings should be verified.

**DEFINITION OF EFFICIENT EQUIPMENT**

The efficient measure is an engine block heater operated by an outdoor plug-in timer (15 amp or greater) that turns on the heater only when the outdoor temperature is below 25 °F.

**DEFINITION OF BASELINE EQUIPMENT**

The baseline scenario is an engine block heater that is manually plugged in by the farmer to facilitate equipment startup at a later time.

**DEEMED LIFETIME OF EFFICIENT EQUIPMENT**

The expected measure life if assumed to be 3 years[32]

**DEEMED MEASURE COST**

The incremental cost per installed plug-in timer is $10.19[33].

**COINCIDENCE FACTOR**

Engine block timers only operate in the winter so the summer peak demand savings is zero.

[32]Equipment life is expected to be longer, but measure life is more conservative to account for possible attrition in use over time.
[33]Based on bulk pricing reported by EnSave, which administers the rebate in Vermont

**Algorithm**

### CALCULATION OF SAVINGS

#### ELECTRIC ENERGY SAVINGS

$\Delta$kWh = ISR * Use Season * %Days * HrSave/Day * $kW_{heater}$ - ParaLd

= 78.39% * 87 days * 84.23% * 7.765 Hr/Day * 1.5 kW - 5.46 kWh

= 664 kWh

#### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS ENERGY SAVINGS

N/A

#### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

#### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: RS-APL-ESDH-V01-120601

## 4.1.2 High Volume Low Speed Fans

### DESCRIPTION

The measure applies to 20-24 foot diameter horizontally mounted ceiling high volume low speed (HVLS) fans that are replacing multiple non HVLS fans that have reached the end of useful life in agricultural applications.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be classified as HVLS and have a VFD[34].

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline condition is assumed to be multiple non HVLS existing fans that have reached the end of s useful life.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years[35].

### DEEMED MEASURE COST

The incremental capital cost for the fans are as follows[36]:

| Fan Diameter Size (feet) | Incremental Cost |
|---|---|
| 20 | $4150 |
| 22 | $4180 |
| 24 | $4225 |

### LOADSHAPE

Loadshape C34 - Industrial Motor

### COINCIDENCE FACTOR

The measure has deemed kW savings therefor a coincidence factor is not applied.

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS [37]

The annual electric savings from this measure are deemed values depending on fan size and apply to all building types:

[34] Act on Energy Commercial Technical Reference Manual No. 2010-4
[35] Ibid.
[36] Ibid.
[37] Ibid.

| Fan Diameter Size (feet) | kWh Savings |
|---|---|
| 20 | 6577 |
| 22 | 8543 |
| 24 | 10018 |

### SUMMER COINCIDENT PEAK DEMAND SAVINGS[38]

The annual kW savings from this measure are deemed values depending on fan size and apply to all building types:

| Fan Diameter Size (feet) | kW Savings |
|---|---|
| 20 | 2.4 |
| 22 | 3.1 |
| 24 | 3.7 |

### NATURAL GAS ENERGY SAVINGS

N/A

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-AGE-HVSF-V01-120601

[38] Ibid.

## 4.1.3 High Speed Fans

### DESCRIPTION

The measure applies to high speed exhaust, ventilation and circulation fans that are replacing an existing unit that reached the end of its useful life in agricultural applications.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be diffuser equipped and meet the following criteria[39].

| Diameter of Fan (inches) | Minimum Efficiency for Exhasut & Ventilation Fans | Minimum Efficiency for Circulation Fans |
|---|---|---|
| 24 through 35 | 14.0 cfm/W at 0.10 static pressure | 12.5 lbf/kW |
| 36 through 47 | 17.1 cfm/W at 0.10 static pressure | 18.2 lbf/kW |
| 48 through 71 | 20.3 cfm/W at 0.10 static pressure | 23.0 lbf/kW |

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline condition is assumed to be an existing fan that reached the end of its useful life.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 7 years[40].

### DEEMED MEASURE COST

The incremental capital cost for all fan sizes is $150[41].

### LOADSHAPE

Loadshape C34 - Industrial Motor

### COINCIDENCE FACTOR

The measure has deemed kW savings therefor a coincidence factor is not applied.

[39] Act on Energy Commercial Technical Reference Manual No. 2010-4
[40] Ibid.
[41] Ibid.

**Algorithm**

### CALCULATION OF SAVINGS

#### ELECTRIC ENERGY SAVINGS [42]

The annual electric savings from this measure are deemed values depending on fan size and apply to all building types:

| Diameter of Fan (inches) | kWh |
|---|---|
| 24 through 35 | 372 |
| 36 through 47 | 625 |
| 48 through 71 | 1122 |

#### SUMMER COINCIDENT PEAK DEMAND SAVINGS[43]

The annual kW savings from this measure are deemed values depending on fan size and apply to all building types:

| Diameter of Fan (inches) | kW |
|---|---|
| 24 through 35 | 0.118 |
| 36 through 47 | 0.198 |
| 48 through 71 | 0.356 |

#### NATURAL GAS ENERGY SAVINGS

N/A

#### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

#### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-AGE-HSF_-V01-120601

[42] Ibid.
[43] Ibid.

## 4.1.4 Live Stock Waterer

### DESCRIPTION

This measure applies to the replacement of electric open waterers with sinking or floating water heaters with equivalent herd size watering capacity of the old unit.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to an electrically heated thermally insulated waterer with minimum 2 inches of insulation. A thermostat is required on unit with heating element greater than or equal to 250 watts[44].

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to be an electric open waterer with sinking or floating water heaters that have reached the end of useful life.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years[45].

### DEEMED MEASURE COST

The incremental capital cost for the waters are $787.50:[46]

### LOADSHAPE

Loadshape C04 - Non-Residential Electric Heating

### COINCIDENCE FACTOR

The measure has deemed kW savings therefor a coincidence factor is not applied

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS [47]

The annual electric savings from this measure is a deemed value and assumed to be 1592.85 kWh.

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

The annual kW savings from this measure is a deemed value and assumed to be 0.525 kW. [48]

[44] Act on Energy Commercial Technical Reference Manual No. 2010-4
[45] Ibid.
[46] Ibid.
[47] Ibid.

**NATURAL GAS ENERGY SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-AGE-LSW1-V01-120601**

[48] Ibid.

# 4.2 Food Service Equipment End Use

## 4.2.1 Combination Oven

### DESCRIPTION

This measure applies to natural gas fired high efficiency combination convection and steam ovens installed in a commercial kitchen replacing existing equipment at the end of its useful life.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a new natural gas combination convection with steam oven cooking efficiency ≥ 38% and convection mode cooking efficiency ≥ 44% utilizing ASTM standard F2861 and meet idle requirements below[49]:

Idle Rate Requirements for Commercial Combination Ovens/Steamers

| Combi Oven Type | Steam Mode Idle Rate | Convection Mode Idle Rate |
|---|---|---|
| Gas Combi < 15 pan capacity | 15,000 Btu/hr | 9,000 Btu/hr |
| Gas Combi 15-28 pan capacity | 18,000 Btu/hr | 11,000 Btu/hr |
| Gas Combi > 28 pan capacity | 28,000 Btu/hr | 17,000 Btu/hr |

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is a new or existing natural gas combination convection and steam ovens that do not meet the efficient equipment criteria

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years[50]

### DEEMED MEASURE COST

The incremental capital cost for this measure is $4300[51]

### LOADSHAPE

N/A

[49] http://www.fishnick.com/saveenergy/rebates/combis.pdf
[50] Deemed values from Nicor Gas were used. Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011.
[51] Ibid.

### COINCIDENCE FACTOR

N/A

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS ENERGY SAVINGS

The annual natural gas energy savings from this measure is a deemed value equaling 644 therms.[52]

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-FSE-CBOV-V01-120601

[52] Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011.These deemed values should be compared to PY evaluation and revised as necessary.

## 4.2.2 Commercial Solid and Glass Door Refrigerators & Freezers

### DESCRIPTION

This measure relates to the installation of a new reach-in commercial refrigerator or freezer meeting ENERGY STAR efficiency standards. ENERGY STAR labeled commercial refrigerators and freezers are more energy efficient because they are designed with components such as ECM evaporator and condenser fan motors, hot gas anti-sweat heaters, or high-efficiency compressors, which will significantly reduce energy consumption.

This measure was developed to be applicable to the following program types: TOS and NC. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a new vertical solid or glass door refrigerator or freezer or vertical chest freezer meeting the minimum ENERGY STAR efficiency level standards.

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to be an existing solid or glass door refrigerator or freezer meeting the minimum federal manufacturing standards as specified by the Energy Policy Act of 2005.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years [53].

### DEEMED MEASURE COST

The incremental capital cost for this measure is provided below[54].

| Type | Refrigerator incremental Cost, per unit | Freezer Incremental Cost, per unit |
|---|---|---|
| Solid or Glass Door | | |
| 0 < V < 15 | $143 | $142 |
| 15 ≤ V < 30 | $164 | $166 |
| 30 ≤ V < 50 | $164 | $166 |
| V ≥ 50 | $249 | $407 |

[53] 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, "Effective/Remaining Useful Life Values", California Public Utilities Commission, December 16, 2008. http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf

[54] Estimates of the incremental cost of commercial refrigerators and freezers varies widely by source. Nadel, S., Packaged Commercial Refrigeration Equipment: A Briefing Report for Program Planners and Implementers, ACEEE, December 2002, indicates that incremental cost is approximately zero. Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, February, 19, 2010, assumed incremental cost ranging from $75 to $125 depending on equipment volume. ACEEE notes that incremental cost ranges from 0 to 10% of the baseline unit cost <http://www.aceee.org/ogeece/ch5_reach.htm>. For the purposes of this characterization, assume and incremental cost adder of 5% on the full unit costs presented in Goldberg et al, State of Wisconsin Public Service Commission of Wisconsin, Focus on Energy Evaluation, Business Programs: Incremental Cost Study, KEMA, October 28, 2009.

### LOADSHAPE

Loadshape C23 - Commercial Refrigeration

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 0.937.[55]

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

ΔkWh = (kWhbase – kWhee) * 365.25

Where:

kWhbase= baseline maximum daily energy consumption in kWh

= calculated using actual chilled or frozen compartment volume (V) of the efficient unit as shown in the table below.

| Type | kWhbase[56] |
|---|---|
| Solid Door Refrigerator | 0.10 * V + 2.04 |
| Glass Door Refrigerator | 0.12 * V + 3.34 |
| Solid Door Freezer | 0.40 * V + 1.38 |
| Glass Door Freezer | 0.75 * V + 4.10 |

kWhee[57] = efficient maximum daily energy consumption in kWh

= calculated using actual chilled or frozen compartment volume (V) of the efficient unit as shown in the table below.

| Type | Refrigerator kWhee | Freezer kWhee |
|---|---|---|
| | | |
| 0 < V < 15 | ≤ 0.089V + 1.411 | ≤ 0.250V + 1.250 |
| 15 ≤ V < 30 | ≤ 0.037V + 2.200 | ≤ 0.400V – 1.000 |
| 30 ≤ V < 50 | ≤ 0.056V + 1.635 | ≤ 0.163V + 6.125 |
| V ≥ 50 | ≤ 0.060V + 1.416 | ≤ 0.158V + 6.333 |
| **Glass Door** | | |
| 0 < V < 15 | ≤ 0.118V + 1.382 | ≤ 0.607V + 0.893 |
| 15 ≤ V < 30 | ≤ 0.140V + 1.050 | ≤ 0.733V – 1.000 |

[55] The CF for Commercial Refrigeration was calculated based upon the Ameren provided eShapes
[56] Energy Policy Act of 2005. Accessed on 7/7/10. <http://www.epa.gov/oust/fedlaws/publ_109-058.pdf>
[57] ENERGY STAR Program Requirements for Commercial Refrigerators and Freezers Partner Commitments Version 2.0, U.S. Environmental Protection Agency, Accessed on 7/7/10. < http://www.energystar.gov/ia/partners/product_specs/program_reqs/commer_refrig_glass_prog_req.pdf>

| Type | Refrigerator kWhee | Freezer kWhee |
|---|---|---|
| 30 ≤ V < 50 | ≤ 0.088V + 2.625 | ≤ 0.250V + 13.500 |
| V ≥ 50 | ≤ 0.110V + 1.500 | ≤ 0.450V + 3.500 |

V = the chilled or frozen compartment volume ($ft^3$) (as defined in the Association of Home Appliance Manufacturers Standard HRF1–1979)

= Actual installed

365.25 = days per year

For example a solid door refrigerator with a volume of 15 would save

ΔkWh = (3.54 – 2.76) * 365.25

= 285 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ΔkWh / HOURS * CF

Where:

HOURS = equipment is assumed to operate continuously, 24 hours per day, 365.25 days per year.

= 8766

CF = Summer Peak Coincidence Factor for measure

= 0.937

For example a solid door refrigerator with a volume of 15 would save

ΔkW = 285/ 8766 * .937

=0.030 kW

**NATURAL GAS ENERGY SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-FSE-CSDO-V01-120601**

## 4.2.3 Commercial Steam Cooker

### DESCRIPTION

To qualify for this measure the installed equipment must be an ENERGY STAR® steamer in place of a standard steamer in a commercial kitchen. Savings are presented dependent on the pan capacity and corresponding idle rate at heavy load cooking capacity and if the steamer is gas or electric.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be as follows:

| Gas | Electric |
|---|---|
| ENERGY STAR® qualified with 38% minimum cooking energy efficiency at heavy load (potato) cooking capacity for gas steam cookers. | ENERGY STAR® qualified with 50% minimum cooking energy efficiency at heavy load (potato) cooking capacity for electric steam cookers. |

### DEFINITION OF BASELINE EQUIPMENT

The baseline condition is assumed to be a non-ENERGY STAR® commercial steamer at end of life. It is assumed that the efficient equipment and baseline equipment have the same number of pans.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years[58]

### DEEMED MEASURE COST

The incremental capital cost for this measure is $998[59] for a natural gas steam cooker or $2490[60] for an electric steam cooker.

### LOADSHAPE

Loadshape C01 - Commercial Electric Cooking

### COINCIDENCE FACTOR

Summer Peak Coincidence Factor for measure is provided below for different building type[61]:

| Location | CF |
|---|---|
| Fast Food Limited Menu | 0.32 |
| Fast Food Expanded Menu | 0.41 |
| Pizza | 0.46 |

[58]California DEER 2008 which is also used by both the Food Service Technology Center and ENERGY STAR®.
[59]Source for incremental cost for efficient natural gas steamer is RSG Commercial Gas Steamer Workpaper, January 2012.
[60]Source for efficient electric steamer incremental cost is $2,490 per 2009 PG&E Workpaper - PGECOFST104.1 - Commercial Steam Cooker - Electric and Gas as reference by KEMA in the ComEd C & I TRM.
[61]Minnesota 2012 Technical Reference Manual, Electric Food Service_v03.2.xls, http://mn.gov/commerce/energy/topics/conservation/Design-Resources/Deemed-Savings.jspech. Unknown is an average of other location types

| Location | CF |
|---|---|
| Full Service Limited Menu | 0.51 |
| Full Service Expanded Menu | 0.36 |
| Cafeteria | 0.36 |
| Unknown | 0.40 |

## Algorithm

### CALCULATION OF SAVINGS

Formulas below are applicable to both gas and electric steam cookers. Please use appropriate lookup values and identified flags.

### ENERGY SAVINGS

$$\Delta Savings = (\Delta Idle\ Energy + \Delta Preheat\ Energy + \Delta Cooking\ Energy) * Z$$

For a gas cooker: $\Delta Savings = \Delta Btu * 1/100{,}000 * Z$

For an electric steam cooker: $\Delta Savings = \Delta kWh * Z$

Where:

Z = days/yr steamer operating (use 365.25 days/yr if heavy use restaurant and exact number unknown)

$\Delta$Idle Energy = $((((1- CSM_{\%Baseline})* IDLE_{BASE} + CSM_{\%Baseline} * PC_{BASE} * E_{FOOD} / EFF_{BASE})*(HOURS_{day} - (F / PC_{Base}) - ( PRE_{number} *0.25))) - (((1- CSM_{\%ENERGYSTAR}) * IDLE_{ENERGYSTAR} + CSM_{\%ENERGYSTAR} * PC_{ENERGY} * E_{FOOD} / EFF_{ENERGYSTAR}) * (HOURS_{Day} - (F l/ PC_{ENERGY} ) - (PRE_{number} * 0.25 ))))$

Where:

$CSM_{\%Baseline}$ = Baseline Steamer Time in Manual Steam Mode (% of time)

= 90%[62]

$IDLE_{Base}$ = Idle Energy Rate of Base Steamer[63]

| Number of Pans | $IDLE_{BASE}$ - Gas, Btu/hr | $IDLE_{BASE}$ - Electric, kw |
|---|---|---|
| 3 | 11,000 | 1.0 |
| 4 | 14,667 | 1.33 |
| 5 | 18,333 | 1.67 |
| 6 | 22,000 | 2.0 |

[62]Food Service Technology Center 2011 Savings Calculator
[63]Food Service Technology Center 2011 Savings Calculator

$PC_{Base}$ = Production Capacity of Base Steamer[64]

| Number of Pans | $PC_{BASE}$, gas (lbs/hr) | $PC_{BASE}$, electric (lbs/hr) |
|---|---|---|
| 3 | 65 | 70 |
| 4 | 87 | 93 |
| 5 | 108 | 117 |
| 6 | 130 | 140 |

$E_{FOOD}$= Amount of Energy Absorbed by the food during cooking known as ASTM Energy to Food (Btu/lb or kW/lb)

=105 Btu/lb[65] (gas steamers) or 0.0308[8] (electric steamers)

$EFF_{BASE}$ =Heavy Load Cooking Efficiency for Base Steamer

=15%[66] (gas steamers) or 26%[9] (electric steamers)

$HOURS_{day}$ = Average Daily Operation (hours)

| Type of Food Service | Hoursday[67] |
|---|---|
| Fast Food, limited menu | 4 |
| Fast Food, expanded menu | 5 |
| Pizza | 8 |
| Full Service, limited menu | 8 |
| Full Service, expanded menu | 7 |
| Cafeteria | 6 |
| Unknown | 6[68] |
| Custom | Varies |

F = Food cooked per day (lbs/day)

= custom or if unknown, use 100 lbs/day[69]

[64]Production capacity per Food Service Technology Center 2011 Savings Calculator of 23.3333 lb/hr per pan for electric baseline steam cookers and 21.6667 lb/hr per pan for natural gas baseline steam cookers. ENERGY STAR® savings calculator uses 23.3 lb/hr per pan for both electric and natural gas baseline steamers.
[65]Reference ENERGY STAR® savings calculator at http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COC.
[66]Reference Food Service Technology Center 2011 Savings Calculator values as used by Consortium for Energy Efficiency, Inc. for baseline electric and natural gas steamer heavy cooking load energy efficiencies.
[67]Minnesota 2012 Technical Reference Manual, Electric Food Service_v03.2.xls, http://mn.gov/commerce/energy/topics/conservation/Design-Resources/Deemed-Savings.jspech
[68]Unknown is average of other locations

$CSM_{\%ENERGYSTAR}$ = ENERGY STAR Steamer's Time in Manual Steam Mode (% of time)[70]

= 0%

$IDLE_{ENERGYSTAR}$ = Idle Energy Rate of ENERGY STAR®[71]

| Number of Pans | $IDLE_{ENERGY\ STAR}$ – gas, (Btu/hr) | $IDLE_{ENERGY\ STAR}$ – electric, (kW) |
|---|---|---|
| 3 | 6250 | 0.40 |
| 4 | 8333 | 0.53 |
| 5 | 10417 | 0.67 |
| 6 | 12500 | 0.80 |

$PC_{ENERGY}$ = Production Capacity of ENERGY STAR® Steamer[72]

| Number of Pans | $PC_{ENERGY}$ - gas(lbs/hr) | $PC_{ENERGY}$ – electric (lbs/hr) |
|---|---|---|
| 3 | 55 | 50 |
| 4 | 73 | 67 |
| 5 | 92 | 83 |
| 6 | 110 | 100 |

$EFF_{ENERGYSTAR}$ = Heavy Load Cooking Efficiency for ENERGY STAR® Steamer(%)

=38%[73] (gas steamer) or 50%[15] (electric steamer)

$PRE_{number}$ = Number of preheats per day

=1[74] (if unknown, use 1)

$\Delta Preheat\ Energy = ( PRE_{number} * \Delta Pre_{heat})$

[69]Reference amount used by both Food Service Technology Center and ENERGY STAR® savings calculator

[70]Reference information from the Food Service Technology Center siting that ENERGY STAR® steamers are not typically operated in constant steam mode, but rather are used in timed mode. Reference ENERGY STAR® savings calculator at http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COC for efficient steamer. Both baseline & efficient steamer mode values should be considered for users in Illinois market.

[71]Food Service Technology Center 2011 Savings Calculator

[72]Production capacity per Food Service Technology Center 2011 Savings Calculator of 18.3333 lb/hr per pan for gas ENERGY STAR® steam cookers and 16.6667 lb/hr per pan for electric ENERGY STAR® steam cookers. ENERGY STAR® savings calculator uses 16.7 lb/hr per pan for electric and 20 lb/hr for natural gas ENERGY STAR® steamers.

[73]Reference Food Service Technology Center 2011 Savings Calculator values as used by Consortium for Energy Efficiency, Inc. for Tier 1A and Tier 1B qualified electric and natural gas steamer heavy cooking load energy efficiencies and http://www.energystar.gov/ia/partners/product_specs/program_reqs/Commercial_Steam_Cookers_Program_Requirements.pdf?7010-36eb

[74]Reference ENERGY STAR® savings calculator at http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COC and Food

Where:

$PRE_{number}$ = Number of Preheats per Day

=1[75](if unknown, use 1)

$PRE_{heat}$ = Preheat energy savings per preheat

= 11,000 Btu/preheat[76] (gas steamer) or 0.5 kWh/preheat[77] (electric steamer)

$\Delta$Cooking Energy = ((1/ EFFBASE) - (1/ EFFENERGY STAR®)) * F * $E_{FOOD}$

Where:

$EFF_{BASE}$ =Heavy Load Cooking Efficiency for Base Steamer

=15%[78] (gas steamer) or 26%[28] (electric steamer)

$EFF_{ENERGYSTAR}$ =Heavy Load Cooking Efficiency for ENERGY STAR® Steamer

=38%[79] (gas steamer) or 50%[23] (electric steamer)

F = Food cooked per day (lbs/day)

= custom or if unknown, use 100 lbs/day[80]

$E_{FOOD}$ = Amount of Energy Absorbed by the food during cooking known as ASTM Energy to Food[81]

| $E_{FOOD}$ - gas(Btu/lb) | $E_{FOOD}$ (kWh/lb) |
|---|---|
| 105[82] | 0.0308[83] |

[75]Reference ENERGY STAR® savings calculator at http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COC and Food
[76]Ohio TRM which references 2002 Food Service Technology Center "Commercial Cooking Appliance Technology Assessment" Chapter 8: Steamers. This is time also used by ENERGY STAR® savings calculator at http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COC. 11,000 Btu/preheat is from 72,000 Btu/hr * 15 min/hr /60 min/hr for gas steamers and 0.5 kWh/preheat is from 6 kW/preheat * 15 min/hr / 60 min/hr
[77] Reference Food Service Technology Center 2011 Savings Calculator values for Baseline Preheat Energy.
[78] Reference Food Service Technology Center 2011 Savings Calculator values as used by Consortium for Energy Efficiency, Inc. for baseline electric and natural gas steamer heavy cooking load energy efficiencies.
[79] Ibid.
[80]Amount used by both Food Service Technology Center and ENERGY STAR® savings calculator
[81]Reference ENERGY STAR® savings calculator at http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COC.
[82]Ibid.
[83]Ibid.

**EXAMPLE**

For a gas steam cooker: A 3 pan steamer in a full service restaurant

| | |
|---|---|
| ΔSavings | = ΔIdle Energy + ΔPreheat Energy + ΔCooking Energy *Z * 1/100.000 |
| ΔIdle Energy | = ((((1- 0.9)* 11000 + 0.9 * 65 * 105 /0.15 )*(7 - (100 / 65)-(1*0.25))) - (((1-0) * 6250 + 0 * 55 * 105 / 0.38) * (7 - (100 / 55) - (1*0.25)))) + |
| ΔPreheat Energy | = (1 *11,000) + |
| ΔCooking Energy | = (((1/ 0.15) - (1/ 0.38)) * (100 lb/day * 105 btu/lb))) |
| | * 365.25 days)) *1/100,000 = |
| | = 883 therms |

For an electric steam cooker: A 3 pan steamer in a cafeteria:

| | |
|---|---|
| ΔSavings | = ΔIdle Energy + ΔPreheat Energy + ΔCooking Energy *Z |
| ΔIdle Energy | = ((((1- .9)* 1.0 + .9 * 70 * 0.0308 /.26 )*(6 - (100 / 70)-(1*.25))) - (((1-0) * 0.4 + 0 * 50 * .0308 / 0.50) * (6 - (100 / 50) - (1*.25)))) + |
| ΔPreheat Energy | = (1 *0.5)) + |
| ΔCooking Energy | = (((1/ 0.26) - (1/ 0.5)) * (100 * 0.0308 ))) |
| | * 365.25 days= |
| | 13,649 kWh |

#### SUMMER COINCIDENT PEAK DEMAND SAVINGS

This is only applicable to the electric steam cooker.

$\Delta kW = (\Delta kWh/(HOURSDay * DaysYear)) * CF$

Where:

CF =Summer Peak Coincidence Factor for measure is provided below for different locations[84]:

| Location | CF |
|---|---|
| Fast Food Limited Menu | 0.32 |
| Fast Food Expanded Menu | 0.41 |
| Pizza | 0.46 |
| Full Service Limited Menu | 0.51 |
| Full Service Expanded Menu | 0.36 |
| Cafeteria | 0.36 |

$Days_{Year}$ =Annual Days of Operation

=custom or 365.25 days a year

Other values as defined above

**EXAMPLE**

For 3 pan electric steam cooker located in a cafeteria:

$\Delta kW = (\Delta kWh/(HOURS_{Day} * Days_{Year})) * CF$

$= (13{,}649/(6 * 365.25)) * 0.36$

$= 2.24$ kW

#### WATER IMPACT DESCRIPTIONS AND CALCULATION

This is applicable to both gas and electric steam cookers.

$\Delta Water = [(W_{BASE} - W_{ENERGYSTAR®})* HOURS_{Day} * Days_{Year}$

Where

$W_{BASE}$ = Water Consumption Rate of Base Steamer (gal/hr)

= 40[85]

$W_{ENERGYSTAR}$ = Water Consumption Rate of ENERGY STAR® Steamer look up[86]

[84]Minnesota 2012 Technical Reference Manual, Electric Food Service_v03.2.xls, http://mn.gov/commerce/energy/topics/conservation/Design-Resources/Deemed-Savings.jspech

[85] FSTC (2002). Commercial Cooking Appliance Technology Assessment. Chapter 8: Steamers.

[86]Source Consortium for Energy Efficiency, Inc. September 2010 "Program Design Guidance for Steamers" for Tier 1A and Tier

| CEE Tier | gal/hr |
|---|---|
| Tier 1A | 15 |
| Tier 1B | 4 |
| Avg Efficient | 10 |
| Avg Most Efficient | 3 |

$Days_{Year}$ =Annual Days of Operation

=custom or 365.25 days a year[87]

**EXAMPLE**

For example, an electric 3 pan steamer with average efficiency in a full service restaurant

ΔWater = (40 -10) * 7 * 365.25

= 76,703 gallons

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-FSE-STMC-V03-150601**

1B water requirements. Ohio Technical Reference Manual 2010 for 10 gal/hr water consumption which can be used when Tier level is not known.

[87]Source for 365.25 days/yr is ENERGY STAR® savings calculator which references Food Service Technology research on average use, 2009.

## 4.2.4 Conveyor Oven

### DESCRIPTION

This measure applies to natural gas fired high efficiency conveyor ovens installed in commercial kitchens replacing existing natural gas units with conveyor width greater than 25 inches.

Conveyor ovens are available using four different heating processes: infrared, natural convection with a ceramic baking hearth, forced convection or air impingement, or a combination of infrared and forced convection. Conveyor ovens are typically used for producing a limited number of products with similar cooking requirements at high production rates. They are highly flexible and can be used to bake or roast a wide variety of products including pizza, casseroles, meats, breads, and pastries.

Some manufacturers offer an air-curtain feature at either end of the cooking chamber that helps to keep the heated air inside the conveyor oven. The air curtain operates as a virtual oven wall and helps reduce both the idle energy of the oven and the resultant heat gain to the kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a natural gas conveyor oven with a tested baking energy efficiency > 42% and an idle energy consumption rate < 57,000 Btu/hr utilizing ASTM standard F1817.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an existing pizza deck oven at end of life.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 17 years.[88]

### DEEMED MEASURE COST

The incremental capital cost for this measure is $1800[89].

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

[88] Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011
[89] Ibid.

**Algorithm**

### CALCULATION OF SAVINGS

#### ELECTRIC ENERGY SAVINGS

N/A

#### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

#### NATURAL GAS ENERGY SAVINGS

The annual natural gas energy savings from this measure is a deemed value equaling 733 Therms[90].

#### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

#### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-FSE-CVOV-V01-120601

[90]Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011.These deemed values should be compared to PY evaluation and revised as necessary

## 4.2.5 ENERGY STAR Convection Oven

### DESCRIPTION

This measure applies to natural gas fired ENERGY STAR convection ovens installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a natural gas convection oven with a cooking efficiency ≥ 44% utilizing ASTM standard 1496 and an idle energy consumption rate < 13,000 Btu/hr

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is a natural gas convection oven that is not ENERGY STAR certified and is at end of life.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years[91]

### DEEMED MEASURE COST

The incremental capital cost for this measure is $50[92]

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

[91] Lifetime from ENERGY STAR commercial griddle which cites reference as "FSTC research on available models, 2009" http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

[92] Measure cost from ENERGY STAR which cites reference as "EPA research on available models using AutoQuotes, 2010" http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

**NATURAL GAS ENERGY SAVINGS**

Custom calculation below, otherwise use deemed value of 306 therms. [93]

ΔTherms = (ΔDailyIdle Energy + ΔDailyPreheat Energy + ΔDailyCooking Energy) * Days /100000

Where:

ΔDailyIdleEnergy = (IdleBase* IdleBaseTime)- (IdleENERGYSTAR * IdleENERGYSTARTime)

ΔDailyPreheatEnergy = (PreHeatNumberBase * PreheatTimeBase / 60 * PreheatRateBase) – (PreheatNumberENERGYSTAR * PreheatTimeENERGYSTAR/60 * PreheatRateENERGYSTAR)

ΔDailyCookingEnergy = (LB * EFOOD/ EffBase) - (LB * EFOOD/ EffENERGYSTAR)

Where:

HOURSday = Average Daily Operation

= custom or if unknown, use 12 hours

Days = Annual days of operation

= custom or if unknown, use 365.25 days a year

LB = Food cooked per day

= custom or if unknown, use 100 pounds

EffENERGYSTAR = Cooking Efficiency ENERGY STAR

= custom or if unknown, use 44%

EffBase = Cooking Efficiency Baseline

= custom or if unknown, use 30%

PCENERGYSTAR = Production Capacity ENERGY STAR

= custom or if unknown, use 80 pounds/hr

PCBase = Production Capacity base

= custom or if unknown, use 70 pounds/hr

PreheatNumberENERGYSTAR = Number of preheats per day

= custom or if unknown, use 1

PreheatNumberBase = Number of preheats per day

= custom or if unknown, use 1

PreheatTimeENERGYSTAR = preheat length

= custom or if unknown, use 15 minutes

PreheatTimeBase = preheat length

= custom or if unknown, use 15 minutes

[93] Algorithms and assumptions derived from ENERGY STAR Oven Commercial Kitchen Equipment Savings Calculator.http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

PreheatRateENERGYSTAR = preheat energy rate high efficiency

= custom or if unknown, use 44000 btu/h

PreheatRateBase = preheat energy rate baseline

= custom or if unknown, use 76000 btu/h

IdleENERGYSTAR = Idle energy rate

= custom or if unknown, use 13000 btu/h

IdleBase = Idle energy rate

= custom or if unknown, use 18000 btu/h

IdleENERGYSTARTime = ENERGY STAR Idle Time

=HOURsday-LB/PCENERGYSTAR –PreHeatTimeENERGYSTAR/60

=12 – 100/80 – 15/60

=10.5 hours

IdleBaseTime = BASE Idle Time

= HOURsday-LB/PCbase –PreHeatTimeBase/60

=Custom or if unknown, use

=12 – 100/70-15/60

=10.3 hours

EFOOD = ASTM energy to food

= 250 btu/pound

**EXAMPLE**

For example, an ENERGY STAR Oven with a cooking energy efficiency of 44% and default values from above would save.

ΔTherms = (ΔIdle Energy + ΔPreheat Energy + ΔCooking Energy) * Days /100000

Where:

ΔDailyIdleEnergy =(18000*10.3)- (13000*10.5)

= 49286 btu

ΔDailyPreheatEnergy = (1 * 15 / 60 *76000) – (1 * 15 / 60 *44000)

= 8000 btu

ΔDailyCookingEnergy = (100 * 250/ .30) - (100 * 250/ .44)

=26515 btu

ΔTherms = (49286+8000+26515)* 365.25 /100000

=306 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-FSE-ESCV-V01-120**

## 4.2.6 ENERGY STAR Dishwasher

### DESCRIPTION

This measure applies to ENERGY STAR high and low temp under counter single tank door type, single tank conveyor, and multiple tank conveyor dishwashers installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be an ENERGY STAR certified dishwasher meeting idle energy rate (kW) and water consumption (gallons/rack) limits, as determined by both machine type and sanitation approach (chemical/low temp versus high temp).

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is a dishwasher that's not ENERGY STAR certified and at end of life.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be[94]

| Dishwasher type | | Equipment Life |
|---|---|---|
| Low Temp | Under Counter | 10 |
| | Door Type | 15 |
| | Single Tank Conventional | 20 |
| | Multi Tank Conventional | 20 |
| High Temp | Under Counter | 10 |
| | Door Type | 15 |
| | Single Tank Conventional | 20 |
| | Multi Tank Conventional | 20 |

[94] Lifetime from ENERGY STAR HFHC which cites reference as "FSTC research on available models, 2009" http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

### DEEMED MEASURE COST

The incremental capital cost for this measure is[95]

| Dishwasher type | | Incremental Cost |
|---|---|---|
| Low Temp | Under Counter | $530 |
| | Door Type | $530 |
| | Single Tank Conventional | $170 |
| | Multi Tank Conventional | $0 |
| High Temp | Under Counter | $1000 |
| | Door Type | $500 |
| | Single Tank Conventional | $270 |
| | Multi Tank Conventional | $0 |

### LOADSHAPE

Loadshape C01 - Commercial Electric Cooking

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

Summer Peak Coincidence Factor for measure is provided below for different restaurant types[96]:

| Location | CF |
|---|---|
| Fast Food Limited Menu | 0.32 |
| Fast Food Expanded Menu | 0.41 |
| Pizza | 0.46 |
| Full Service Limited Menu | 0.51 |
| Full Service Expanded Menu | 0.36 |
| Cafeteria | 0.36 |

## Algorithm

### ENERGY SAVINGS

ENERGY STAR dishwashers save energy in three categories, building water heating, booster water heating and idle energy. Building water heating and booster water heating could be either electric or natural gas. These deemed values are presented in a table format. Savings all water heating combinations are found in the tables below. [97]

Electric building and booster water heating

[95] Measure cost from ENERGY STAR which cites reference as "EPA research on available models using AutoQuotes, 2010" http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

[96] Minnesota 2012 Technical Reference Manual, Electric Food Service_v03.2.xls, http://mn.gov/commerce/energy/topics/conservation/Design-Resources/Deemed-Savings.jspech

[97] Algorithms and assumptions derived from ENERGY STAR Commercial Kitchen Equipment Savings Calculator.http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

| Dishwasher type | | kWh | Therms |
|---|---|---|---|
| Low Temp | Under Counter | 1,213 | 0 |
| | Door Type | 12,135 | 0 |
| | Single Tank Conventional | 11,384 | 0 |
| | Multi Tank Conventional | 17,465 | 0 |
| High Temp | Under Counter | 7471 | 0 |
| | Door Type | 14143 | 0 |
| | Single Tank Conventional | 19235 | 0 |
| | Multi Tank Conventional | 34153 | 0 |

Electric building and natural gas booster water heating

| Dishwasher type | | kWh | Therms |
|---|---|---|---|
| Low Temp | Under Counter | 9089 | 0 |
| | Door Type | 21833 | 0 |
| | Single Tank Conventional | 24470 | 0 |
| | Multi Tank Conventional | 29718 | 0 |
| High Temp | Under Counter | 7208 | 110 |
| | Door Type | 19436 | 205 |
| | Single Tank Conventional | 29792 | 258 |
| | Multi Tank Conventional | 34974 | 503 |

Natural Gas building and electric booster water heating

| Dishwasher type | | kWh | Therms |
|---|---|---|---|
| Low Temp | Under Counter | 0 | 56 |
| | Door Type | 0 | 562 |
| | Single Tank Conventional | 0 | 527 |
| | Multi Tank Conventional | 0 | 809 |
| High Temp | Under Counter | 2717 | 220 |
| | Door Type | 5269 | 441 |
| | Single Tank Conventional | 8110 | 515 |
| | Multi Tank Conventional | 12419 | 1007 |

Natural Gas building and booster water heating

| Dishwasher type | | kWh | Therms |
|---|---|---|---|
| Low Temp | Under Counter | 0 | 56 |
| | Door Type | 0 | 562 |
| | Single Tank Conventional | 0 | 527 |
| | Multi Tank Conventional | 0 | 809 |
| High Temp | Under Counter | 0 | 330 |
| | Door Type | 198 | 617 |
| | Single Tank Conventional | 1752 | 773 |
| | Multi Tank Conventional | 0 | 1510 |

**WATER SAVINGS**

Using standard assumptions water savings would be:

| Dishwasher type | | Savings (gallons) |
|---|---|---|
| Low Temp | Under Counter | 6,844 |
| | Door Type | 6,8474 |
| | Single Tank Conventional | 64,240 |
| | Multi Tank Conventional | 98,550 |
| High Temp | Under Counter | 26,828 |
| | Door Type | 50,078 |
| | Single Tank Conventional | 62,780 |
| | Multi Tank Conventional | 122,640 |

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$$\Delta kW = \Delta kWh / AnnualHours$$

Where:

AnnualHours = Hours * Days

= 365.25 * 18

= 6575 annual hours

Example:

A low temperature undercounter dishwasher with electric building and booster water heaters would save:

ΔkW = ΔkWh/ AnnualHours

= 1213/6575

= 0.184 kW

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-FSE-ESDW-V01-120601**

## 4.2.7 ENERGY STAR Fryer

### DESCRIPTION

This measure applies to natural gas fired ENERGY STAR fryer installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a natural gas fryer with a heavy load cooking efficiency ≥ 50% utilizing ASTM standard F1361 or F2144.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is a natural gas fryer that is not ENERGY STAR certified at end of life.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years.[98]

### DEEMED MEASURE COST

The incremental capital cost for this measure is $1200.[99]

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

[98]Lifetime from ENERGY STAR commercial griddle which cites reference as "FSTC research on available models, 2009" http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

[99]Measure cost from ENERGY STAR which cites reference as "EPA research on available models using AutoQuotes, 2010" http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

**NATURAL GAS ENERGY SAVINGS**[100]

Custom calculation below, otherwise use deemed value of 505 Therms.

ΔTherms = (ΔDailyIdle Energy + ΔDailyPreheat Energy + ΔDailyCooking Energy) * Days /100000

Where:

ΔDailyIdleEnergy =(IdleBase* IdleBaseTime) – (IdleENERGYSTAR * IdleENERGYSTARTime)

ΔDailyPreheatEnergy = (PreHeatNumberBase * PreheatTimeBase / 60 * PreheatRateBase) – (PreheatNumberENERGYSTAR* PreheatTimeENERGYSTAR/60 * PreheatRateENERGYSTAR)

ΔDailyCookingEnergy = (LB * EFOOD/ EffBase) - (LB * EFOOD/ EffENERGYSTAR)

Where:

| | |
|---|---|
| HOURSday | = Average Daily Operation |
| | = custom or if unknown, use 16 hours |
| Days | = Annual days of operation |
| | = custom or if unknown, use 365.25 days a year |
| LB | = Food cooked per day |
| | = custom or if unknown, use 150 pounds |
| EffENERGYSTAR | = Cooking Efficiency ENERGY STAR |
| | = custom or if unknown, use 50% |
| EffBase | = Cooking Efficiency Baseline |
| | = custom or if unknown, use 35% |
| PCENERGYSTAR | = Production Capacity ENERGY STAR |
| | = custom or if unknown, use 65 pounds/hr |
| PCBase | = Production Capacity base |
| | = custom or if unknown, use 60 pounds/hr |
| PreheatNumberENERGYSTAR | = Number of preheats per day |
| | = custom or if unknown, use 1 |
| PreheatNumberBase | = Number of preheats per day |
| | = custom or if unknown, use 1 |
| PreheatTimeENERGYSTAR | = preheat length |
| | = custom or if unknown, use 15 minutes |
| PreheatTimeBase | = preheat length |
| | = custom or if unknown, use 15 minutes |

[100] Algorithms and assumptions derived from ENERGY STAR fryer Commercial Kitchen Equipment Savings Calculator.http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

| | |
|---|---|
| PreheatRateENERGYSTAR | = preheat energy rate high efficiency |
| | = custom or if unknown, use 62000 btu/h |
| PreheatRateBase | = preheat energy rate baseline |
| | = custom or if unknown, use 64000 btu/h |
| IdleENERGYSTAR | = Idle energy rate |
| | = custom or if unknown, use 9000 btu/h |
| IdleBase | = Idle energy rate |
| | = custom or if unknown, use 14000 btu/h |
| IdleENERGYSTARTime | = ENERGY STAR Idle Time |
| | = HOURsday-LB/PCENERGYSTAR –PreHeatTimeENERGYSTAR/60 |
| | =Custom or if unknown, use |
| | =16 – 150/65-15/60 |
| | =13.44 hours |
| IdleBaseTime | = BASE Idle Time |
| | = HOURsday-LB/PCbase –PreHeatTimeBase/60 |
| | =Custom or if unknown, use |
| | =16 – 150/60-15/60 |
| | =13.25 hours |
| EFOOD | = ASTM energy to food |
| | = 570 btu/pound |

**EXAMPLE**

For example, an ENERGY STAR fryer with a tested heavy load cooking energy efficiency of 50% and an idle energy rate of 120,981 btu and an Idle Energy Consumption Rate 9000 btu would save.

ΔTherms = (ΔIdle Energy + ΔPreheat Energy + ΔCooking Energy) * Days /100000

Where:

| | |
|---|---|
| ΔDailyIdleEnergy | =(18550*13.25)- (120981 * 13.44) |
| | = 64519 btu |
| ΔDailyPreheatEnergy | = (1 * 15 / 60 *64000) – (1 * 15 / 60 *62000) |
| | = 500 btu |
| ΔDailyCookingEnergy | = (150 * 570/ .35) - (150 * 570/ .5) |
| | =73286 btu |
| ΔTherms | = (64519+500+73286)* 365.25 /100000 |
| | =508 therms |

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-FSE-ESFR-V01-120601**

## 4.2.8 ENERGY STAR Griddle

### DESCRIPTION

This measure applies to electric and natural gas fired high efficiency griddle installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be an ENERGY STAR natural gas or electric griddle with a tested heavy load cooking energy efficiency of 70 percent (electric) 38 percent (gas) or greater and an idle energy rate of 2,650 Btu/hr per square foot of cooking surface or less, utilizing ASTM F1275. The griddle must have an Idle Energy Consumption Rate < 2,600 Btu/hr per square foot of cooking surface.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an existing natural gas or electric griddle that's not ENERGY STAR certified and is at end of use.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years[101]

### DEEMED MEASURE COST

The incremental capital cost for this measure is $0 for and electric griddle and $60 for a gas griddle.[102]

### LOADSHAPE

Loadshape C01 - Commercial Electric Cooking

### COINCIDENCE FACTOR

Summer Peak Coincidence Factor for measure is provided below for different building type[103]:

| Location | CF |
|---|---|
| Fast Food Limited Menu | 0.32 |
| Fast Food Expanded Menu | 0.41 |
| Pizza | 0.46 |
| Full Service Limited Menu | 0.51 |
| Full Service Expanded Menu | 0.36 |
| Cafeteria | 0.36 |

[101] Lifetime from ENERGY STAR commercial griddle which cites reference as "FSTC research on available models, 2009" http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

[102] Measure cost from ENERGY STAR which cites reference as "EPA research on available models using AutoQuotes, 2010" http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

[103] Minnesota 2012 Technical Reference Manual, Electric Food Service_v03.2.xls, http://mn.gov/commerce/energy/topics/conservation/Design-Resources/Deemed-Savings.jspech

**Algorithm**

**CALCULATION OF SAVINGS** [104]

**ELECTRIC ENERGY SAVINGS**

ΔkWh = (ΔIdle Energy + ΔPreheat Energy + ΔCooking Energy) * Days /1000

Where:

| | |
|---|---|
| ΔDailyIdleEnergy | =[ IdleBase * Width * Length (LB/ PCBase) – (PreheatNumberBase* PreheatTimeBase/60)]- IdleENERGYSTAR * Width * Length (LB/ PCENERGYSTAR) – (PreheatNumberENERGYSTAR* PreheatTimeENERGYSTAR/60] |
| ΔDailyPreheatEnergy | = (PreHeatNumberBase * PreheatTimeBase / 60 * PreheatRateBase * Width * Depth) – (PreheatNumberENERGYSTAR* PreheatTimeENERGYSTAR/60 * PreheatRateENERGYSTAR * Width * Depth) |
| ΔDailyCookingEnergy | = (LB * EFOOD/ EffBase) - (LB * EFOOD/ EffENERGYSTAR) |

Where:

| | |
|---|---|
| HOURSday | = Average Daily Operation |
| | = custom or if unknown, use 12 hours |
| Days | = Annual days of operation |
| | = custom or if unknown, use 365.25 days a year |
| LB | = Food cooked per day |
| | = custom or if unknown, use 100 pounds |
| Width | = Griddle Width |
| | = custom or if unknown, use 3 feet |
| Depth | = Griddle Depth |
| | = custom or if unknown, use 2 feet |
| EffENERGYSTAR | = Cooking Efficiency ENERGY STAR |
| | = custom or if unknown, use 70% |
| EffBase | = Cooking Efficiency Baseline |
| | = custom or if unknown, use 65% |
| PCENERGYSTAR | = Production Capacity ENERGY STAR |
| | = custom or if unknown, use 6.67 pounds/hr/sq ft |
| PCBase | = Production Capacity base |
| | = custom or if unknown, use 5.83 pounds/hr/sq ft |

[104] Algorithms and assumptions derived from ENERGY STAR Griddle Commercial Kitchen Equipment Savings Calculator.http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

PreheatNumberENERGYSTAR = Number of preheats per day

= custom or if unknown, use 1

PreheatNumberBase = Number of preheats per day

= custom or if unknown, use 1

PreheatTimeENERGYSTAR = preheat length

= custom or if unknown, use 15 minutes

PreheatTimeBase = preheat length

= custom or if unknown, use 15 minutes

PreheatRateENERGYSTAR = preheat energy rate high efficiency

= custom or if unknown, use 1333 W/sq ft

PreheatRateBase = preheat energy rate baseline

= custom or if unknown, use 2667 W/sq ft

IdleENERGYSTAR = Idle energy rate

= custom or if unknown, use 320 W/sq ft

IdleBase = Idle energy rate

= custom or if unknown, use 400 W/sq ft

EFOOD = ASTM energy to food

= 139 w/pound

For example, an ENERGY STAR griddle with a tested heavy load cooking energy efficiency of 70 percent or greater and an idle energy rate of 320 W per square foot of cooking surface or less would save.

ΔDailyIdleEnergy =[ 400* 3 * 2 (100/5.83) – (1* 15/60)]- [320* 3 * 2 (100/6.67) – (1* 15/60]

= 3583 W

ΔDailyPreheatEnergy = (1* 15 / 60 * 2667 * 3 * 2) – (1* 15/60 * 1333 * 3 * 2)

=2000 W

ΔDailyCookingEnergy = (100 * 139/ .65) - (100 * 139/ .70)

=1527 W

ΔkWh = (2000+1527+3583) * 365.25 /1000

=2597 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

kW = ΔkWh/Hours * CF

For example, an ENERGY STAR griddle in a cafeteria with a tested heavy load cooking energy efficiency of 70 percent or greater and an idle energy rate of 320 W per square foot of cooking surface or less would save

=2595 kWh/4308 * .36

= 0.22 kW

**NATURAL GAS ENERGY SAVINGS**

Custom calculation below, otherwise use deemed value of 149 therms.

ΔTherms = (ΔIdle Energy + ΔPreheat Energy + ΔCooking Energy) * Days /100000

Where:

ΔDailyIdleEnergy =[ IdleBase * Width * Length (LB/ PCBase) – (PreheatNumberBase* PreheatTimeBase/60)]- IdleENERGYSTAR * Width * Length (LB/ PCENERGYSTAR) – (PreheatNumberENERGYSTAR* PreheatTimeENERGYSTAR/60]

ΔDailyPreheatEnergy = (PreHeatNumberBase * PreheatTimeBase / 60 * PreheatRateBase * Width * Depth) – (PreheatNumberENERGYSTAR* PreheatTimeENERGYSTAR/60 * PreheatRateENERGYSTAR * Width * Depth)

ΔDailyCookingEnergy = (LB * EFOOD/ EffBase) - (LB * EFOOD/ EffENERGYSTAR)

Where (new variables only):

EffENERGYSTAR = Cooking Efficiency ENERGY STAR

= custom or if unknown, use 38%

EffBase = Cooking Efficiency Baseline

= custom or if unknown, use 32%

PCENERGYSTAR = Production Capacity ENERGY STAR

= custom or if unknown, use 7.5 pounds/hr/sq ft

PCBase = Production Capacity base

= custom or if unknown, use 4.17 pounds/hr/sq ft

PreheatRateENERGYSTAR = preheat energy rate high efficiency

= custom or if unknown, use 10000 btu/h/sq ft

PreheatRateBase = preheat energy rate baseline

= custom or if unknown, use 14000 btu/h/sq ft

IdleENERGYSTAR = Idle energy rate

= custom or if unknown, use 2650 btu/h/sq ft

IdleBase = Idle energy rate

= custom or if unknown, use 3500 btu/h/sq ft

EFOOD = ASTM energy to food

= 475 btu/pound

For example, an ENERGY STAR griddle with a tested heavy load cooking energy efficiency of 38 percent or greater and an idle energy rate of 2,650 Btu/h per square foot of cooking surface or less and an Idle Energy Consumption Rate < 2,600 Btu/h per square foot of cooking surface would save.

ΔDailyIdleEnergy =[ 3500* 3 * 2 (100/4.17) – (1* 15/60)]- 2650* 3 * 2 (100/7.5) – (1* 15/60]

= 11258 Btu

ΔDailyPreheatEnergy = (1* 15 / 60 * 14,000 * 3 * 2) – (1* 15/60 * 10000 * 3 * 2)

=6000 btu

ΔDailyCookingEnergy = (100 * 475/ .32) - (100 * 475/ .38)

=23438 btu

ΔTherms = (11258 + 6000 + 23438) * 365.25 /100000

=149 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-FSE-ESGR-V01-120601**

## 4.2.9 ENERGY STAR Hot Food Holding Cabinets

### DESCRIPTION

This measure applies to electric ENERGY STAR hot food holding cabinets (HFHC) installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be an ENERGY STAR certified HFHC.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an electric HFHC that's not ENERGY STAR certified and at end of life.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years[105]

### DEEMED MEASURE COST

The incremental capital cost for this measure is[106]

| HFHC Size | Incremental Cost |
|---|---|
| Full Size (20 cubic feet) | $1200 |
| ¾ Size (12 cubic feet) | $1800 |
| ½ Size (8 cubic feet) | $1500 |

### LOADSHAPE

Loadshape C01 - Commercial Electric Cooking

[105] Lifetime from ENERGY STAR HFHC which cites reference as "FSTC research on available models, 2009" http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

[106] Measure cost from ENERGY STAR which cites reference as "EPA research on available models using AutoQuotes, 2010" http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

### COINCIDENCE FACTOR

Summer Peak Coincidence Factor for measure is provided below for different building type[107]:

| Location | CF |
|---|---|
| Fast Food Limited Menu | 0.32 |
| Fast Food Expanded Menu | 0.41 |
| Pizza | 0.46 |
| Full Service Limited Menu | 0.51 |
| Full Service Expanded Menu | 0.36 |
| Cafeteria | 0.36 |

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

Custom calculation below, otherwise use deemed values depending on HFHC size[108]

| Cabinet Size | Savings (kWh) |
|---|---|
| Full Size HFHC | 9308 |
| ¾ Size HFHC | 3942 |
| ½ Size HFHC | 2628 |

$$\Delta kWh = HFHCBaselinekWh - HFHCENERGYSTARkWh$$

Where:

HFHCBaselinekWh = PowerBaseline* HOURSday * Days/1000

PowerBaseline = Custom, otherwise

| Cabinet Size | Power (W) |
|---|---|
| Full Size HFHC | 2500 |
| ¾ Size HFHC | 1200 |
| ½ Size HFHC | 800 |

HOURSday = Average Daily Operation

= custom or if unknown, use 15 hours

[107] Minnesota 2012 Technical Reference Manual, Electric Food Service_v03.2.xls, http://mn.gov/commerce/energy/topics/conservation/Design-Resources/Deemed-Savings.jspech

[108] Algorithms and assumptions derived from ENERGY STAR Commercial Kitchen Equipment Savings Calculator.http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

Days = Annual days of operation

= custom or if unknown, use 365.25 days a year

HFHCENERGYSTARkWh = PowerENERGYSTAR* HOURSday * Days/1000

PowerENERGYSTAR = Custom, otherwise

| Cabinet Size | Power (W) |
|---|---|
| Full Size HFHC | 800 |
| ¾ Size HFHC | 480 |
| ½ Size HFHC | 320 |

HOURSday = Average Daily Operation

= custom or if unknown, use 15 hours

Days = Annual days of operation

= custom or if unknown, use 365.25 days a year

For example, if a full size HFHC is installed the measure would save:

ΔkWh = (PowerBaseline* HOURSday * Days)/1000 – (PowerENERGYSTAR* HOURSday * Days)/1000

= (2500*15*365.25)/1000 – (800*15*365.25)/1000

= 9,314 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ΔkWh/Hours * CF

Where:

Hours = Hoursday *Days

For example, if a full size HFHC is installed in a cafeteria the measure would save:

= 9,314 kWh / (15*365.25)* .36

=0 .61 kW

**NATURAL GAS ENERGY SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-FSE-ESHH-V01-120601**

## 4.2.10 ENERGY STAR Ice Maker

### DESCRIPTION

This measure relates to the installation of a new ENERGY STAR qualified commercial ice machine. The ENERGY STAR label applied to air-cooled, cube-type machines including ice-making head, self-contained, and remote-condensing units. This measure excludes flake and nugget type ice machines. This measure could relate to the replacing of an existing unit at the end of its useful life, or the installation of a new system in a new or existing building.

This measure was developed to be applicable to the following program types: TOS and NC. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a new commercial ice machine meeting the minimum ENERGY STAR efficiency level standards.

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to be a commercial ice machine meeting federal equipment standards established January 1, 2010.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years[109].

### DEEMED MEASURE COST

The incremental capital cost for this measure is provided below.[110]

| Harvest Rate (H) | Incremental Cost |
|---|---|
| 100-200 lb ice machine | $296 |
| 201-300 lb ice machine | $312 |
| 301-400 lb ice machine | $559 |
| 401-500 lb ice machine | $981 |
| 501-1000 lb ice machine | $1,485 |
| 1001-1500 lb ice machine | $1,821 |
| >1500 lb ice machine | $2,194 |

### LOADSHAPE

Loadshape C23 - Commercial Refrigeration

[109] DEER 2008

[110] These values are from electronic work papers prepared in support of San Diego Gas & Electric's "Application for Approval of Electric and Gas Energy Efficiency Programs and Budgets for Years 2009-2011", SDGE, March 2, 2009. Accessed on 7/7/10 <http://www.sdge.com/regulatory/documents/ee2009-2011Workpapers/SW-ComB/Food%20Service/Food%20Service%20Electic%20Measure%20Workpapers%2011-08-05.DOC>.

### COINCIDENCE FACTOR

The Summer Peak Coincidence Factor is assumed to equal 0.937

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$\Delta kWH = [(kWh_{base} - kWh_{ee}) / 100] * (DC * H) * 365.25$

Where:

$kWh_{base}$ = maximum kWh consumption per 100 pounds of ice for the baseline equipment

= calculated as shown in the table below using the actual Harvest Rate (H) of the efficient equipment.

$kWh_{ee}$ = maximum kWh consumption per 100 pounds of ice for the efficient equipment

= calculated as shown in the table below using the actual Harvest Rate (H) of the efficient equipment.

| Ice Machine Type | kWhbase[111] | kWhee[112] |
|---|---|---|
| Ice Making Head (H < 450) | 10.26 - 0.0086*H | 9.23 - 0.0077*H |
| Ice Making Head (H ≥ 450) | 6.89 – 0.0011*H | 6.20 - 0.0010*H |
| Remote Condensing Unit, without remote compressor (H < 1000) | 8.85 – 0.0038*H | 8.05 - 0.0035*H |
| Remote Condensing Unit, without remote compressor (H ≥ 1000) | 5.1 | 4.64 |
| Remote Condensing Unit, with remote compressor (H < 934) | 8.85 – 0.0038*H | 8.05 - 0.0035*H |
| Remote Condensing Unit, with remote compressor (H ≥ 934) | 5.3 | 4.82 |
| Self Contained Unit (H < 175) | 18 - 0.0469*H | 16.7 - 0.0436*H |
| Self Contained Unit (H ≥ 175) | 9.8 | 9.11 |

100 = conversion factor to convert kWhbase and kWhee into maximum kWh consumption per pound of ice.

DC = Duty Cycle of the ice machine

[111]Baseline reflects federal standards which apply to units manufactured on or after January 1, 2010 <http://ecfr.gpoaccess.gov/cgi/t/text/text-idx?c=ecfr&rgn=div6&view=text&node=10:3.0.1.4.17.8&idno=10>.

[112]ENERGY STAR Program Requirements for Commercial Ice Machines, Partner Commitments, U.S. Environmental Protection Agency, Accessed on 7/7/10 <http://www.energystar.gov/ia/partners/product_specs/program_reqs/ice_machine_prog_req.pdf>

= 0.57[113]

H = Harvest Rate (pounds of ice made per day)

= Actual installed

365.35 = days per year

For example an ice machine with an ice making head producing 450 pounds of ice would save

ΔkWH = [(6.4 – 5.8) / 100] * (0.57 * 450) * 365.25

= 562 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ΔkWh / (HOURS * DC) * CF

Where:

HOURS = annual operating hours

= 8766[114]

CF = 0.937

For example an ice machine with an ice making head producing 450 pounds of ice would save

ΔkW = 562/(8766 * 0.57) * .937

= 0.105 kW

**NATURAL GAS ENERGY SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

While the ENERGY STAR labeling criteria require that certified commercial ice machines meet certain "maximum potable water use per 100 pounds of ice made" requirements, such requirements are intended to prevent equipment manufacturers from gaining energy efficiency at the cost of water consumptions. A review of the AHRI Certification Directory[115] indicates that approximately 81% of air-cooled, cube-type machines meet the ENERGY STAR potable water use requirement. Therefore, there are no assumed water impacts for this measure.

[113]Duty cycle varies considerably from one installation to the next. TRM assumptions from Vermont, Wisconsin, and New York vary from 40 to 57%, whereas the ENERGY STAR Commercial Ice Machine Savings Calculator < http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_Ice_Machines.xls> assumes a value of 75%. A field study of eight ice machines in California indicated an average duty cycle of 57% ("A Field Study to Characterize Water and Energy Use of Commercial Ice-Cube Machines and Quantify Saving Potential", Food Service Technology Center, December 2007). Furthermore, a report prepared by ACEEE assumed a value of 40% (Nadel, S., Packaged Commercial Refrigeration Equipment: A Briefing Report for Program Planners and Implementers, ACEEE, December 2002). The value of 57% was utilized since it appears to represent a high quality data source.

[114]Unit is assumed to be connected to power 24 hours per day, 365.25 days per year.

[115]AHRI Certification Directory, Accessed on 7/7/10. <http://www.ahridirectory.org/ahridirectory/pages/home.aspx>

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-FSE-ESIM-V01-120601**

## 4.2.11 High Efficiency Pre-Rinse Spray Valve

### DESCRIPTION

Pre-rise valves use a spray of water to remove food waste from dishes prior to cleaning in a dishwasher. More efficient spray valves use less water thereby reducing water consumption, water heating cost, and waste water (sewer) charges. Pre-rinse spray valves include a nozzle, squeeze lever, and dish guard bumper. The primary impacts of this measure are water savings. Reduced hot water consumption saves either natural gas or electricity, depending on the type of energy the hot water heater uses.

This measure was developed to be applicable to the following program types: TOS, RF, and DI. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure, the new or replacement pre-rinse spray nozzle must use less than 1.6 gallons per minute with a cleanability performance of 26 seconds per plate or less.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment will vary based on the delivery method and is defined below:

| Time of Sale | Retrofit, Direct Install |
|---|---|
| The baseline equipment is assumed to be 1.6 gallons per minute. The Energy Policy Act (EPAct) of 2005 sets the maximum flow rate for pre-rinse spray valves at 1.6 gallons per minute at 60 pounds per square inch of water pressure when tested in accordance with ASTM F2324-03. This performance standard went into effect January 1, 2006. | The baseline equipment is assumed to be an existing pre-rinse spray valve with a flow rate of 1.9 gallons per minute.[116] If existing pre-rinse spray valve flow rate is unknown, then existing pre-rinse spray valve must have been installed prior to 2006. The Energy Policy Act (EPAct) of 2005 sets the maximum flow rate for pre-rinse spray valves at 1.6 gallons per minute at 60 pounds per square inch of water pressure when tested in accordance with ASTM F2324-03. This performance standard went into effect January 1, 2006. However, field data shows that not all nozzles in use have been replaced with the newer flow rate nozzle. Products predating this standard can use up to five gallons per minute |

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 5 years[117]

### DEEMED MEASURE COST

The cost of this measure is assumed to be $100[118]

### LOADSHAPE

Loadshape C01 - Commercial Electric Cooking

[116] Verification measurements taken at 195 installations showed average pre and post flowrates of 2.23 and 1.12 gallon per minute, respectively." from IMPACT AND PROCESS EVALUATION FINAL REPORT for CALIFORNIA URBAN WATER CONSERVATION COUNCIL 2004-5 PRE-RINSE SPRAY VALVE INSTALLATION PROGRAM (PHASE 2) (PG&E Program # 1198-04; SoCalGas Program 1200-04) ("CUWCC Report", Feb 2007)

[117] Reference 2010 Ohio Technical Reference Manual, Act on Energy Business Program Technical Reference Manual Rev05, and Federal Energy Management Program (2004), "How to Buy a Low-Flow Pre-Rinse Spray Valve."

[118] Costs range from $60 Chicagoland (Integrys for North Shore & People's Gas) to $150 referenced by Nicor's CLEAResultWorkpaper WPRSGCCODHW102 "Pre-Rinse Spray Valve." Act on Energy references $100.

**COINCIDENCE FACTOR**

N/A

**Algorithm**

**CALCULATION OF ENERGY SAVINGS**

**ELECTRIC ENERGY SAVINGS (NOTE WATER SAVINGS MUST FIRST BE CALCULATED)**

$$\Delta kWH = \Delta Gallons \times 8.33 \times 1 \times (Tout - Tin) \times (1/EFF\ electric) / 3{,}413 \times FLAG$$

Where:

| | |
|---|---|
| ΔGallons | = amount of water saved as calculated below |
| 8.33 lbm/gal | = specific mass in pounds of one gallon of water |
| 1 Btu/lbm°F | = Specific heat of water: 1 Btu/lbm/°F |
| Tout | = Water Heater Outlet Water Temperature |
| | = custom, otherwise assume Tin + 70°F temperature rise from Tin[119] |
| Tin | = Inlet Water Temperature |
| | = custom, otherwise assume 54.1 °F[120] |
| EFF | = Efficiency of electric water heater supplying hot water to pre-rinse spray valve |
| | =custom, otherwise assume 97%[121] |
| Flag | = 1 if electric or 0 if gas |

**EXAMPLE**

Time of Sale: For example, a new spray nozzle with 1.06 gal/min flow replacing a nozzle with 1.6 gal/min flow at a large institutional establishments with a cafeteria with 70 degree temperature rise of water used by the pre-rinse spray valve that is heated by electric hot water saves annually :

$$\Delta kWH = 30{,}326 \times 8.33 \times 1 \times ((70+54.1) - 54.1) \times (1/.97) / 3{,}413 \times 1$$

$$= 5{,}341 kWh$$

Retrofit: For example, a new spray nozzle with 1.06 gal/min flow replacing a nozzle with 1.9 gal/min flow at a large institutional establishments with a cafeteria with 70 degree temperature rise of water used by the pre-rinse spray valve that is heated by electric hot water equals:

$$\Delta kWH = 47{,}175 \times 8.33 \times 1 \times ((70+ 54.1) - 54.1) \times (1/.97) / 3{,}413 \times 1$$

$$= 8309\ kWh$$

[119] If unknown, assume a 70 degree temperature rise from Tin per Food Service Technology Center calculator assumptions to account for variations in mixing and water heater efficiencies

[120] August 31, 2011 Memo of Savings for Hot Water Savings Measures to Nicor Gas from Navigant states that 54.1°F was calculated from the weighted average of monthly water mains temperatures reported in the 2010 Building America Benchmark Study for Chicago-Waukegan, Illinois.

[121] This efficiency value is based on IECC 2012 performance requirement for electric resistant water heaters rounded without the slight adjustment allowing for reduction based on size of storage tank.

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

N/A

**NATURAL GAS ENERGY SAVINGS**

ΔTherms = ΔGallons x 8.33 x 1 x (Tout - Tin) x (1/EFF) /100,000 Btu

Where (new variables only):

EFF = Efficiency of gas water heater supplying hot water to pre-rinse spray valve

= custom, otherwise assume 75%[122]

**EXAMPLE**

Time of Sale: For example, a new spray nozzle with 1.06 gal/min flow replacing a nozzle with 1.6 gal/min flow at a large institutional establishments with a cafeteria with 70 degree temperature of water used by the pre-rinse spray valve that is heated by fossil fuel hot water saves annually:

ΔTherms = 30,326 x 8.33 x 1 x ((70+54.1) - 54.1) x (1/.75)/100,000 x 1.0

= 236 Therms

Retrofit: For example, a new spray nozzle with 1.06 gal/min flow replacing a nozzle with 1.9 gal/min flow at a busy large institutional establishments with a cafeteria with 70 degree temperature rise of water used by the pre-rinse spray valve that is heated by fossil fuel hot water saves annually:

ΔTherms = 47,175 x 8.33 x 1 x ((70+54.1) - 54.1) x (1/.75)/100,000 x (1-0)

=368 Therms

**WATER IMPACT CALCULATION[123]**

ΔGallons = (FLObase - FLOeff)gal/min x 60 min/hr x HOURSday x DAYSyear

Where:

FLObase = Base case flow in gallons per minute, or custom

| Time of Sale | Retrofit, Direct Install |
|---|---|
| 1.6 gal/min124 | 1.9 gal/min125 |

FLOeff = Efficient case flow in gallons per minute or custom

[122] IECC 2012, Table C404.2, Minimum Performance of Water-Heating Equipment

[123] In order to calculate energy savings, water savings must first be calculated

[124] The baseline equipment is assumed to be 1.6 gallons per minute. The Energy Policy Act (EPAct) of 2005 sets the maximum flow rate for pre-rinse spray valves at 1.6 gallons per minute at 60 pounds per square inch of water pressure when tested in accordance with ASTM F2324-03. This performance standard went into effect January 1, 2006. www1.eere.energy.gov/femp/pdfs/spec_prerinsesprayvavles.pdf.

[125] Verification measurements taken at 195 installations showed average pre and post flowrates of 2.23 and 1.12 gallon per minute, respectively." from IMPACT AND PROCESS EVALUATION FINAL REPORT for CALIFORNIA URBAN WATER CONSERVATION COUNCIL 2004-5 PRE-RINSE SPRAY VALVE INSTALLATION PROGRAM (PHASE 2) (PG&E Program # 1198-04; SoCalGas Program 1200-04) ("CUWCC Report", Feb 2007)

| Time of Sale | Retrofit, Direct Install |
|---|---|
| 1.06 gal/min[126] | 1.06 gal/min[127] |

HOURSday = Hours per day that the pre-rinse spray valve is used at the site, custom, otherwise[128]:

| Application | Hours/day |
|---|---|
| Small, quick- service restaurants | 1/2 |
| Medium-sized casual dining restaurants | 1.5 |
| Large institutional establishments with cafeteria | 3 |

DAYSyear = Days per year pre-rinse spray valve is used at the site, custom, otherwise 312 days/yr based on assumed 6 days/wk x 52 wk/yr = 312 day/yr.

**EXAMPLE**

Time of Sale: For example, a new spray nozzle with 1.06 gal/min flow replacing a nozzle with 1.6 gal/min flow at a large institutional establishment with a cafeteria equals

= (1.6 – 1.06) * 60 * 3 * 312

= 30,326 gal/yr

Retrofit: For example, a new spray nozzle with 106 gal/min flow replacing a nozzle with 1.9 gal/min flow at a large institutional establishments with a cafeteria equals

= (1.9 – 1.06) * 60 * 3 * 312

= 47,175 gal/yr

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-FSE-SPRY-V02-120601**

[126] 1.6 gallons per minute used to be the high efficiency flow, but more efficient spray valves are available ranging down to 0.64 gallons per minute per Federal Energy Management Program which references the Food Services Technology Center web site with the added note that even more efficient models may be available since publishing the data. The average of the nozzles listed on the FSTC website is 1.06.

[127] 1.6 gallons per minute used to be the high efficiency flow, but more efficient spray valves are available ranging down to 0.64 gallons per minute per Federal Energy Management Program which references the Food Services Technology Center web site with the added note that even more efficient models may be available since publishing the data. The average of the nozzles listed on the FSTC website is 1.06.

[128] Hours primarily based on PG& E savings estimates, algorithms, sources (2005), Food Service Pre-Rinse Spray Valves with review of 2010 Ohio Technical Reference Manual and Act on Energy Business Program Technical Resource Manual Rev05.

## 4.2.12 Infrared Charbroiler

### DESCRIPTION

This measure applies to natural gas fired charbroilers that utilize infrared burners installed in a commercial kitchen

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a new natural gas charbroiler with infrared burners.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an existing natural gas charbroiler without infrared burners.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years[129]

### DEEMED MEASURE COST

The incremental capital cost for this measure is $2200[130]

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS ENERGY SAVINGS

The annual natural gas energy savings from this measure is a deemed value equaling 661 Therms.[131]

[129]Food Service Technology Center, ENERGY STAR and CEE do not currently provide calculators for this type of equipment therefore deemed values from Nicor Gas were used. Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011

[130]Ibid.

[131] Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011.These deemed values should be compared to PY evaluation and revised as necessary.

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-FSE-IRCB-V01-120601**

## 4.2.13 Infrared Rotisserie Oven

### DESCRIPTION

This measure applies to natural gas fired high efficiency rotisserie ovens utilizing infrared burners and installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a new natural gas rotisserie oven with infrared burners.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an existing natural gas rotisserie oven without infrared burners.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years[132]

### DEEMED MEASURE COST

The incremental capital cost for this measure is $2700[133]

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS ENERGY SAVINGS

The annual natural gas energy savings from this measure is a deemed value equaling 554 Therms[134]

[132]Food Service Technology Center, ENERGY STAR and CEE do not currently provide calculators for this type of equipment therefore deemed values from Nicor Gas were used. Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011

[133]Ibid.

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-FSE-IROV-V01-120601**

[134]Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011.These deemed values should be compared to PY evaluation and revised as necessary.

## 4.2.14 Infrared Salamander Broiler

### DESCRIPTION

This measure applies to natural gas fired high efficiency salamander broilers utilizing infrared burners installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a new natural gas salamander broiler with infrared burners

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an existing natural gas salamander broiler without infrared burners

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years[135]

### DEEMED MEASURE COST

The incremental capital cost for this measure is $1000[136]

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS ENERGY SAVINGS

The annual natural gas energy savings from this measure is a deemed value equaling 239 therms[137]

[135]Food Service Technology Center, ENERGY STAR and CEE do not currently provide calculators for this type of equipment therefore deemed values from Nicor Gas were used. Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011

[136]Ibid.

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-FSE-IRBL-V01-120601**

[137] Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011.These deemed values should be compared to PY evaluation and revised as necessary.

## 4.2.15 Infrared Upright Broiler

### DESCRIPTION

This measure applies to natural gas fired high efficiency upright broilers utilizing infrared burners and installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a new natural gas upright broiler with infrared burners.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an existing natural gas upright broiler without infrared burners.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years[138]

### DEEMED MEASURE COST

The incremental capital cost for this measure is $5900[139]

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS ENERGY SAVINGS

The annual natural gas energy savings from this measure is a deemed value equaling 1089 therms[140].

[138]Food Service Technology Center, ENERGY STAR and CEE do not currently provide calculators for this type of equipment therefore deemed values from Nicor Gas were used. Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011

[139]Ibid.

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-FSE-IRUB-V01-120601**

[140] Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011.These deemed values should be compared to PY evaluation and revised as necessary

## 4.2.16 Kitchen Demand Ventilation Controls

### DESCRIPTION

Installation of commercial kitchen demand ventilation controls that vary the ventilation based on cooking load and/or time of day.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a control system that varies the exhaust rate of kitchen ventilation (exhaust and/or makeup air fans) based on the energy and effluent output from the cooking appliances (i.e., the more heat and smoke/vapors generated, the more ventilation needed). This involves installing a new temperature sensor in the hood exhaust collar and/or an optic sensor on the end of the hood that sense cooking conditions which allows the system to automatically vary the rate of exhaust to what is needed by adjusting the fan speed accordingly.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is kitchen ventilation that has constant speed ventilation motor.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years.[141]

### DEEMED MEASURE COST

The incremental capital cost for this measure is[142]

| Measure Category | Incremental Cost , $/fan |
|---|---|
| DVC Control Retrofit | $1,988 |
| DVC Control New | $1,000 |

### LOADSHAPE

Loadshape C23 - Commercial Ventilation

### COINCIDENCE FACTOR

The measure has deemed peak kW savings therefore a coincidence factor does not apply

[141] PG&E Workpaper: Commercial Kitchen Demand Ventilation Controls-Electric, 2004 - 2005
[142] Ibid.

## Algorithm

### CALCULATION OF SAVINGS

Annual energy use was based on monitoring results from five different types of sites, as summarized in PG&E Food Service Equipment work paper.

### ELECTRIC ENERGY SAVINGS

The following table provides the kWh savings

| Measure Name | Annual Energy Savings Per Unit (kWh/fan) |
|---|---|
| DVC Control Retrofit | 4,486 |
| DVC Control New | 4,486 |

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

The following table provides the kW savings

| Measure Name | Coincident Peak Demand Reduction (kW) |
|---|---|
| DVC Control Retrofit | 0.76 |
| DVC Control New | 0.76 |

### NATURAL GAS ENERGY SAVINGS

$$\Delta Therms = CFM * HP * \text{Annual Heating Load} / (Eff(heat) * 100{,}000)$$

Where:

CFM = the average airflow reduction with ventilation controls per hood

= 611 cfm/HP[143]

HP = actual if known, otherwise assume 7.75 HP

Annual Heating Load = Annual heating energy required to heat fan exhaust make-up air, Btu/cfm dependent on location[144]:

[143] PGE Workpaper, Commercial Kitchen Demand Ventilation Controls, PGECOFST116, June 1, 2009, 4,734 cfm reduction on average , with 7.75 fan horsepower on average.

[144] Food Service Technology Center Outside Air Load Calculator, http://www.fishnick.com/ventilation/oalc/oac.php, with inputs of one cfm, and hours from Commercial Kitchen Demand Ventilation Controls (Average 17.8 hours a day 4.45 am to 10.30 pm). Savings for Rockford, Chicago, and Springfield were obtained from the calculator; values for Belleview and Marion were obtained by using the average savings per HDD from the other values.

| Zone | Annual Heating Load, Btu/cfm |
|---|---|
| 1 (Rockford) | 154,000 |
| 2-(Chicago) | 144,000 |
| 3 (Springfield) | 132,000 |
| 4-(Belleville) | 102,000 |
| 5-(Marion) | 104,000 |

Eff(heat) = Heating Efficiency

= actual if known, otherwise assume 80%[145]

100,000 = conversion from Btu to Therm

**EXAMPLE**

For example, a kitchen hood in Rockford, IL with a 7.75 HP ventilation motor

ΔTherms = 611 * 7.75*154,000 / (0.80 * 100,000)

= 9,115 Therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-FSE-VENT-V02-140601**

[145]Work Paper WPRRSGNGRO301 CLEAResult"Boiler Tune-Up" which cites Focus on Energy Evaluation Business Programs: Deemed Savings Manual V1.0, PA Consulting, KEMA, March 22, 2010

## 4.2.17 Pasta Cooker

### DESCRIPTION

This measure applies to natural gas fired dedicated pasta cookers as determined by the manufacturer and installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a new natural gas fired paste cooker.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an existing natural gas fired stove where pasta is cooked in a pan.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12[146].

### DEEMED MEASURE COST

The incremental capital cost for this measure is $2400[147].

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS ENERGY SAVINGS

The annual natural gas energy savings from this measure is a deemed value equaling 1380 Therms[148].

[146]Food Service Technology Center, ENERGY STAR and CEE do not currently provide calculators for this type of equipment therefore deemed values from Nicor Gas were used. Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011
[147]Ibid.
[148]Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011.These

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-FSE-PCOK-V01-120601**

deemed values should be compared to PY evaluation and revised as necessary.

## 4.2.18 Rack Oven - Double Oven

### DESCRIPTION

This measure applies to natural gas fired high efficiency rack oven - double oven installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a new natural gas rack oven –double oven with a baking efficiency ≥ 50% utilizing ASTM standard 2093

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an existing natural gas rack oven – double oven with a baking efficiency < 50%.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years.[149]

### DEEMED MEASURE COST

The incremental capital cost for this measure is $8646.[150]

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS ENERGY SAVINGS

The annual natural gas energy savings from this measure is a deemed value equaling 2064 therms[151]

[149]Food Service Technology Center, ENERGY STAR and CEE do not currently provide calculators for this type of equipment therefore deemed values from Nicor Gas were used. Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011
[150]Ibid.
[151]Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011.These

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-FSE-RKOV-V01-120601**

deemed values should be compared to PY evaluation and revised as necessary

## 4.2.19 ENERGY STAR Electric Convection Oven

### DESCRIPTION

Commercial convection ovens that are ENERGY STAR certified have higher heavy load cooking efficiencies, and lower idle energy rates, making them on average about 20 percent more efficient than standard models. Energy savings estimates are for ovens using full size (18" x 36") sheet pans.

This measure was developed to be applicable to the following program types; TOS.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is assumed to be an ENERGY STAR qualified electric convection oven.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be a standard convection oven with a heavy load efficiency of 65%.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years.[152]

### DEEMED MEASURE COST

The incremental cost for this measure is assumed to be $800 for half size units and $1000 for full size[153]

### LOADSHAPE

Loadshape C01 - Commercial Electric Cooking

### COINCIDENCE FACTOR

Summer Peak Coincidence Factor for measure is provided below for different building type[154]:

| Location | CF |
|---|---|
| Fast Food Limited Menu | 0.32 |
| Fast Food Expanded Menu | 0.41 |
| Pizza | 0.46 |
| Full Service Limited Menu | 0.51 |

[152] Food Service Technology Center (FSTC). Default value from life cycle cost calculator. http://www.fishnick.com/saveenergy/tools/calculators/eovencalc.php

[153] Based on data from the Regional Technical Forum for the Northwest Council (http://rtf.nwcouncil.org/measures/com/ComCookingConvectionOven_v2_0.xlsm) using actual list prices for 23 units from 2012, see "ComCookingConvectionOven_v2_0.xlsm".

[154] Minnesota 2012 Technical Reference Manual, Electric Food Service_v03.2.xls, http://mn.gov/commerce/energy/topics/conservation/Design-Resources/Deemed-Savings.jspech. Unknown is an average of other location types

| | |
|---|---|
| Full Service Expanded Menu | 0.36 |
| Cafeteria | 0.36 |
| Unknown | 0.40 |

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = kWH_{base} - kWh_{eff}$$

$$kWh = [(LB * E_{FOOD}/EFF) + (IDLE * (HOURS_{DAY} - LB/PC - PRE_{TIME}/60)) + PRE_{ENERGY}] * DAYS$$

Where:

$kWH_{base}$ = the annual energy usage of the baseline equipment calculated using baseline values

$kWH_{eff}$ = the annual energy usage of the efficient equipment calculated using efficient values

$HOURS_{DAY}$ = daily operating hours

= Actual, defaults:

| Type of Food Service | $HOURS_{DAY}$ [155] |
|---|---|
| Fast Food, limited menu | 4 |
| Fast Food, expanded menu | 5 |
| Pizza | 8 |
| Full Service, limited menu | 8 |
| Full Service, expanded menu | 7 |
| Cafeteria | 6 |
| Unknown | 6[156] |
| Custom | Varies |

DAYS = Days per year of operation

= Actual, default = 365[157]

[155] Minnesota 2012 Technical Reference Manual, Electric Food Service_v03.2.xls, http://mn.gov/commerce/energy/topics/conservation/Design-Resources/Deemed-Savings.jspech

[156] Unknown is average of other locations

$PRE_{TIME}$ = Preheat time (min/day), the amount of time it takes a steamer to reach operating temperature when turned on

= 15 min/day[158]

$E_{FOOD}$ = ASTM Energy to Food (kWh/lb); the amount of energy absorbed by the food during cooking, per pound of food

= 0.0732[159]

LB = pounds of food cooked per day (lb/day)

= Actual, default = 100[160]

EFF = Heavy load cooking energy efficiency (%). See table below.

IDLE = Idle energy rate. See table below.

PC = Production capacity (lbs/hr). See table below.

$PRE_{ENERGY}$ = Preheat energy (kWh/day). See table below.

Performance Metrics: Baseline and Efficient Values

| Metric | Baseline Model [161] | Energy Efficient Model [162] |
|---|---|---|
| $PRE_{ENERGY}$ (kWh) | 1.5 | 1 |
| IDLE (kW) | 2 | Actual, default = 1.0 |
| EFF | 65% | Actual, default = 74% |
| PC (lb/hr) | 70 | Actual, default = 79 |

[157] Food Service Technology Center (FSTC). Default value from life cycle cost calculator. http://www.fishnick.com/saveenergy/tools/calculators/eovencalc.php

[158] Food Service Technology Center (2002). *Commercial Cooking Appliance Technology Assessment*. Prepared by Don Fisher. Chapter 7: Ovens

[159] American Society for Testing and Materials. Industry standard for Commercial Ovens

[160] Food Service Technology Center (FSTC). Default value from life cycle cost calculator. http://www.fishnick.com/saveenergy/tools/calculators/eovencalc.php

[161] Food Service Technology Center (FSTC). Default values from life cycle cost calculator. http://www.fishnick.com/saveenergy/tools/calculators/eovencalc.php

[162] Average ratings of units on ENERGY STAR qualified list as of 10/2014. Preheat energy is not provided so default is provided based on FSTC life cycle cost calculator.

**EXAMPLE**

Using defaults provided above, the savings for a ENERGY STAR Electric Convection Oven in unknown location are:

$kWH_{base}$ = [(100 * 0.0732/0.65) + (2 * (6 – 100/70 – 15/60)) + 1.5] * 365

= 7,813 kWh

$kWh_{eff}$ = [(100 * 0.0732/0.74) + (1 * (6 – 100/79 – 15/60)) + 1.0] * 365

= 5,612 kWh

ΔkWh = $kWH_{base}$ - $kWh_{eff}$

= 7,813 – 5,612

= 2200 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$$\Delta kW = (\Delta kWh / (HOURS_{DAY} * DAYS)) * CF$$

Where:

ΔkWh = Annual energy savings (kWh)

CF = Summer Peak Coincidence Factor for measure is provided below for different building type[163]:

| Location | CF |
|---|---|
| Fast Food Limited Menu | 0.32 |
| Fast Food Expanded Menu | 0.41 |
| Pizza | 0.46 |
| Full Service Limited Menu | 0.51 |
| Full Service Expanded Menu | 0.36 |
| Cafeteria | 0.36 |
| Unknown | 0.40 |

[163] Minnesota 2012 Technical Reference Manual, Electric Food Service_v03.2.xls, http://mn.gov/commerce/energy/topics/conservation/Design-Resources/Deemed-Savings.jspech. Unknown is an average of other location types

**EXAMPLE**

Using defaults provided above, the savings for a ENERGY STAR Electric Convection Oven in unknown location are:

$$\Delta kW = (2200 / (6 * 365)) * 0.40$$

$$= 0.40$$

**FOSSIL FUEL IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

## MEASURE CODE: CI-FSE-ECON-V01-150601

# 4.3 Hot Water

## 4.3.1 Storage Water Heater

### DESCRIPTION

This measure is for upgrading from minimum code to a storage-type water heaters. Storage water heaters are used to supply hot water for a variety of commercial building types. Storage capacities vary greatly depending on the application. Large consumers of hot water include (but not limited to) industries, hotels/motels and restaurants.

This measure was developed to be applicable to the following program types: TOS, RF, ER.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

| Gas, High Efficiency | Gas, Standard | Electric |
|---|---|---|
| In order for this characterization to apply, the efficient equipment is assumed to have heating capacity over 75,000 Btu/hr and a Thermal Efficiency (TE) greater than or equal to 88% | In order for this characterization to apply, the efficient equipment is assumed to be a gas-fired storage water heaters with 0.67 EF or better installed in a non-residential application<br><br>Primary applications would include (but not limited to) hotels/motels, small commercial spaces, offices and restaurants | In order for this characterization to apply, the efficient equipment is assumed to have[164]:<br><br>Energy factor greater than or equal to 0.95 Minimum Thermal Efficiency of 0.98<br><br>Less than 3% standby loss (standby loss is calculated as percentage of annual (energy usage)<br><br>Equivalent storage capacity to unit being replaced<br><br>Qualified units must be GAMA/AHRI efficiency rating certified |

### DEFINITION OF BASELINE EQUIPMENT

| Gas, High Efficiency | Gas, Standard | Electric |
|---|---|---|
| In order for this characterization to apply, the baseline condition is assumed to be a water heater with heating capacity over 75,000 Btu/hr and a Thermal Efficiency (TE) of 80% | In order for this characterization to apply, the baseline condition is assumed to be the minimum code compliant unit with 0.575 EF. | In order for this characterization to apply, the baseline equipment is assumed to be an electric storage water heater with 50 or more gallon capacity in input wattage between 12kW and 54kW. |

[164] Act on Energy Commercial Technical Reference Manual No. 2010-4

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

| Gas, High Efficiency | Gas, Standard | Electric |
|---|---|---|
| The expected measure life is assumed to be 15 Years[165] | The expected measure life is assumed to be 15 years[166] | The expected measure life is assumed to be 5 years[167]. |

### DEEMED MEASURE COST

| Gas, High Efficiency | Gas, Standard | Electric |
|---|---|---|
| The incremental capital cost for this measure is $209 | The deemed measure cost is assumed to be $400 | The incremental capital cost for this measure is assumed to be[168]<br>Tank Size / Incremental Cost<br>50 gallons / $1050<br>80 gallons / $1050<br>100 gallons / $1950 |

### LOADSHAPE

| Gas, High Efficiency | Gas, Standard | Electric |
|---|---|---|
| N/A | N/A | Loadshape C02 - Non-Residential Electric DHW |

### COINCIDENCE FACTOR

| Gas, High Efficiency | Gas, Standard | Electric |
|---|---|---|
| N/A | N/A | The measure has deemed kW savings therefor a coincidence factor is not applied |

[165] Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011
[166] Gas Storage Water Heater 0.67. Work Paper WPRSGNGDHW106. Resource Solutions Group. December 2010
[167] Ibid.
[168] Ibid.

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS [169]

The annual electric savings the electric water storage tank and heater is a deemed value and assumed to be:

| Tank Size | Savings (kWh) |
|---|---|
| 50 gallons | 1,781 |
| 80 gallons | 4,963 |
| 100 gallons | 8,274 |

### SUMMER COINCIDENT PEAK DEMAND SAVINGS[170]

The annual kW savings from this measure is a deemed value and assumed to be:

| Tank Size | Savings (kW) |
|---|---|
| 50 gallons | 0.20 |
| 80 gallons | 0.57 |
| 100 gallons | 0.94 |

[169] Ibid.
[170] Ibid.

**NATURAL GAS ENERGY SAVINGS**

| Gas, High Efficiency | Gas, Standard |
|---|---|
| The annual natural gas energy savings from this measure is a deemed value equaling 251[171] | Gas savings depend on building type and are based on measure case energy factor of 0.67 and a heating capacity of 75 MBtu/hr. These values are averages of qualifying units. Savings values are derived from 2008 DEER Miser, which provides MBtu/hr gas savings per MBtu/hr capacity. Savings presented here are per water heater.[172] |

| Building Type | Energy Savings (therms/unit) |
|---|---|
| Assembly | 185 |
| Education – Primary/Secondary | 124 |
| Education – Post Secondary | 178 |
| Grocery | 191 |
| Health/Medical - Hospital | 297 |
| Lodging - Hotel | 228 |
| Manufacturing - Light Industrial | 140 |
| Office – > 60,000 sq-ft | 164 |
| Office – < 60,000 sq-ft | 56 |
| Restaurant - FastFood | 109 |
| Restaurant – Sit Down | 166 |
| Retail | 105 |
| Storage | 150 |
| Multi-Family | 119 |
| Other | 148 |

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HW_-STWH-V01-120601**

[171] Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011. These deemed values should be compared to PY evaluation and revised as necessary

[172] Gas Storage Water Heater 0.67. Work Paper WPRSGNGDHW106. Resource Solutions Group. December 2010

## 4.3.2 Low Flow Faucet Aerators

### DESCRIPTION

This measure relates to the direct installation of a low flow faucet aerator in a commercial building. Expected applications include small business, office, restaurant, or motel. For multifamily or senior housing, the residential low flow faucet aerator should be used.

This measure was developed to be applicable to the following program types, DI.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be an energy efficient faucet aerator, for bathrooms rated at 1.5 gallons per minute (GPM) or less, or for kitchens rated at 2.2 GPM or less. Savings are calculated on an average savings per faucet fixture basis.

### DEFINITION OF BASELINE EQUIPMENT

The baseline condition is assumed to be a standard bathroom faucet aerator rated at 2.25 GPM or more, or a standard kitchen faucet aerator rated at 2.75 GPM or more. Note if flow rates are measured, for example through a Direct Install program, then actual baseline flow rates should be used as opposed to the deemed values.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 9 years.[173]

### DEEMED MEASURE COST

The incremental cost for this measure is $8[174] or program actual.

### LOADSHAPE

Loadshape C02 - Commercial Electric DHW

### COINCIDENCE FACTOR

The coincidence factor for this measure is dependent on building type as presented below.

[173] Table C-6, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. "http://neep.org/Assets/uploads/files/emv/emv-library/measure_life_GDS%5B1%5D.pdf"

[174] Direct-install price per faucet assumes cost of aerator and install time. (2011, Market research average of $3 and assess and install time of $5 (20min @ $15/hr)

**Algorithm**

**CALCULATION OF SAVINGS**

**ELECTRIC ENERGY SAVINGS**

**NOTE THESE SAVINGS ARE *PER* FAUCET RETROFITTED[175].**

ΔkWh = %ElectricDHW * ((GPM_base - GPM_low)/GPM_base) * Usage * EPG_electric * ISR

Where:

%ElectricDHW = proportion of water heating supplied by electric resistance heating

| DHW fuel | %Electric_DHW |
|---|---|
| Electric | 100% |
| Fossil Fuel | 0% |

GPM_base = Average flow rate, in gallons per minute, of the baseline faucet "as-used"

= 1.39[176] or custom based on metering studies[177] or if measured during DI:

= Measured full throttle flow * 0.83 throttling factor[178]

GPM_low = Average flow rate, in gallons per minute, of the low-flow faucet aerator "as-used"

= 0.94[179] or custom based on metering studies[180] or if measured during DI:

= Rated full throttle flow * 0.95 throttling factor[181]

[175] This algorithm calculates the amount of energy saved per aerator by determining the fraction of water consumption savings for the upgraded fixture. Due to the distribution of water consumption by fixture type, as well as the different number of fixtures in a building, several variables must be incorporated.

[176] Email message from Maureen Hodgins, Research Manager for Water Research Foundation, to TAC/SAG, August 26, 2014

[177] Measurement should be based on actual average flow consumed over a period of time rather than a onetime spot measurement for maximum flow. Studies have shown maximum flow rates do not correspond well to average flow rate due to occupant behavior which does not always use maximum flow.

[178] 2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings. Page 1-265. www.seattle.gov/light/Conserve/Reports/paper_10.pdf

[179] Average retrofit flow rate for kitchen and bathroom faucet aerators from sources 2, 4, 5, and 7. This accounts for all throttling and differences from rated flow rates. Assumes all kitchen aerators at 2.2 gpm or less and all bathroom aerators at 1.5 gpm or less. The most comprehensive available studies did not disaggregate kitchen use from bathroom use, but instead looked at total flow and length of use for all faucets. This makes it difficult to reliably separate kitchen water use from bathroom water use. It is possible that programs installing low flow aerators lower than the 2.2 gpm for kitchens and 1.5 gpm for bathrooms will see a lower overall average retrofit flow rate.

[180] Measurement should be based on actual average flow consumed over a period of time rather than a onetime spot measurement for maximum flow. Studies have shown maximum flow rates do not correspond well to average flow rate due to occupant behavior which does not always use maximum flow.

Usage = Estimated usage of mixed water (mixture of hot water from water heater line and cold water line) per faucet (gallons per year)

= If data is available to provide a reasonable custom estimate it should be used, if not use the following defaults (or substitute custom information in to the calculation):

| Building Type | Gallons hot water per unit per day[182] (A) | Unit | Estimated % hot water from Faucets [183] (B) | Multiplier [184] (C) | Unit | Days per year (D) | Annual gallons mixed water per faucet (A*B*C*D) |
|---|---|---|---|---|---|---|---|
| Small Office | 1 | person | 100% | 10 | employees per faucet | 250 | 2,500 |
| Large Office | 1 | person | 100% | 45 | employees per faucet | 250 | 11,250 |
| Fast Food Rest | 0.7 | meal/day | 50% | 75 | meals per faucet | 365 | 9,581 |
| Sit-Down Rest | 2.4 | meal/day | 50% | 36 | meals per faucet | 365 | 15,768 |
| Retail | 2 | employee | 100% | 5 | employees per faucet | 365 | 3,650 |
| Grocery | 2 | employee | 100% | 5 | employees per faucet | 365 | 3,650 |
| Warehouse | 2 | employee | 100% | 5 | employees per faucet | 250 | 2,500 |
| Elementary School | 0.6 | person | 50% | 50 | students per faucet | 200 | 3,000 |
| Jr High/High School | 1.8 | person | 50% | 50 | students per faucet | 200 | 9,000 |
| Health | 90 | patient | 25% | 2 | Patients per faucet | 365 | 16,425 |
| Motel | 20 | room | 25% | 1 | faucet per room | 365 | 1,825 |
| Hotel | 14 | room | 25% | 1 | faucet per room | 365 | 1,278 |
| Other | 1 | employee | 100% | 20 | employees per faucet | 250 | 5,000 |

EPG_electric = Energy per gallon of mixed water used by faucet (electric water heater)

= (8.33 * 1.0 * (WaterTemp - SupplyTemp)) / (RE_electric * 3412)

[181] 2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings. Page 1-265. www.seattle.gov/light/Conserve/Reports/paper_10.pdf

[182] Table 2-45 Chapter 49, Service Water Heating, 2007 ASHRAE Handbook, HVAC Applications.

[183] Estimated based on data provided in Appendix E; "Waste Not, Want Not: The Potential for Urban Water Conservation in California"; http://www.pacinst.org/reports/urban_usage/appendix_e.pdf

[184] Based on review of the Illinois plumbing code (Employees and students per faucet). Retail, grocery, warehouse and health are estimates. Meals per faucet estimated as 4 bathroom and 3 kitchen faucets and average meals per day of 250 (based on California study above) – 250/7 = 36. Fast food assumption estimated.

= (8.33 * 1.0 * (90 – 54.1)) / (0.98 * 3412)

= 0.0894 kWh/gal

8.33 = Specific weight of water (lbs/gallon)

1.0 = Heat Capacity of water (btu/lb-°F)

WaterTemp = Assumed temperature of mixed water

= 86F for Bath, 93F for Kitchen 91F for Unknown[185]

SupplyTemp = Assumed temperature of water entering building

= 54.1°F [186]

RE_electric = Recovery efficiency of electric water heater

= 98% [187]

3412 = Converts Btu to kWh (Btu/kWh)

ISR = In service rate of faucet aerators dependant on install method as listed in table below[188]

| Selection | ISR |
|---|---|
| Direct Install - Deemed | 0.95 |

[185] Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group. If the aerator location is unknown an average of 91% should be used which is based on the assumption that 70% of household water runs through the kitchen faucet and 30% through the bathroom (0.7*93)+(0.3*86)=0.91.

[186] US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html.

[187] Electric water heaters have recovery efficiency of 98%: http://www.ahridirectory.org/ahridirectory/pages/home.aspx

[188] ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program Table 3-8 http://ilsagfiles.org/SAG_files/Evaluation_Documents/ComEd/ComEd%20EPY2%20Evaluation%20Reports/ComEd_All_Electric_Single_Family_HEP_PY2_Evaluation_Report_Final.pdf

**EXAMPLE**

For example, a direct installed faucet in a large office with electric DHW:

ΔkWh = 1 * ((1.39 – 0.94)/1.39) * 11,250 * 0.0894 * 0.95

= 309 kWh

For example, a direct installed faucet in an Elementary School with electric DHW:

ΔkWh = 1 * ((1.39 – 0.94)/1.39) * 3,000 * 0.0894 * 0.95

= 82.5 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = (ΔkWh / Hours) * CF

Where:

ΔkWh = calculated value above on a per faucet basis

Hours = Annual electric DHW recovery hours for faucet use

= (Usage * 0.545[189] )/GPH

= Calculate if usage is custom, if using default usage use:

| Building Type | Annual Recovery Hours |
|---|---|
| Small Office | 24 |
| Large Office | 109 |
| Fast Food Rest | 93 |
| Sit-Down Rest | 153 |
| Retail | 36 |
| Grocery | 36 |
| Warehouse | 24 |
| Elementary School | 29 |
| Jr High/High School | 88 |
| Health | 160 |
| Motel | 18 |

[189] 54.5% is the proportion of hot 120F water mixed with 54.1F supply water to give 90°F mixed faucet water.

| Building Type | Annual Recovery Hours |
|---|---|
| Hotel | 12 |
| Other | 49 |

Where:

GPH = Gallons per hour recovery of electric water heater calculated for 85.9F temp rise (140-54.1), 98% recovery efficiency, and typical 12kW electric resistance storage tank.

= 56

CF = Coincidence Factor for electric load reduction

= Dependent on building type[190]

| Building Type | Coincidence Factor |
|---|---|
| Small Office | 0.0064 |
| Large Office | 0.0288 |
| Fast Food Rest | 0.0084 |
| Sit-Down Rest | 0.0184 |
| Retail | 0.0043 |
| Grocery | 0.0043 |
| Warehouse | 0.0064 |
| Elementary School | 0.0096 |
| Jr High/High School | 0.0288 |
| Health | 0.0144 |
| Motel | 0.0006 |
| Hotel | 0.0004 |
| Other | 0.0128 |

[190] Calculated as follows: Assumptions for percentage of usage during peak period (1-5pm) were made and then multiplied by 65/365 (65 being the number of days in peak period) and by the number of total annual recovery hours to give an estimate of the number of hours of recovery during peak periods. There are 260 hours in the peak period so the probability you will see savings during the peak period is calculated as the number of hours of recovery during peak divided by 260. See 'C&I Faucet Aerator.xls' for details.

**EXAMPLE**

For example, a direct installed faucet in a large office with electric DHW:

ΔkW = 309/109 * 0.0288

= 0.0816 kW

For example, a direct installed faucet in an Elementary School with electric DHW:

ΔkW = 82.5/29 * 0.0096

= 0.0273 kW

**FOSSIL FUEL IMPACT DESCRIPTIONS AND CALCULATION**

ΔTherms = %FossilDHW * ((GPM_base - GPM_low)/GPM_base) * Usage * EPG_gas * ISR

Where:

%FossilDHW = proportion of water heating supplied by fossil fuel heating

| DHW fuel | %Fossil_DHW |
|---|---|
| Electric | 0% |
| Fossil Fuel | 100% |

EPG_gas = Energy per gallon of mixed water used by faucet (gas water heater)

= (8.33 * 1.0 * (WaterTemp - SupplyTemp)) / (RE_gas * 100,000)

= 0.00446 Therm/gal

Where:

RE_gas = Recovery efficiency of gas water heater

` = 67% [191]

100,000 = Converts Btus to Therms (Btu/Therm)

Other variables as defined above.

[191] Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 75%. Commercial properties are more similar to MF homes than SF homes. MF hot water is often provided by a larger commercial boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of .59 and the .75 for single family home. An average is used for this analysis by default.

**EXAMPLE**

For example, a direct installed faucet in a large office with gas DHW:

ΔTherms = 1 * ((1.39 – 0.94)/1.39) * 11,250 * 0.00446 * 0.95

= 15.4 Therms

For example, a direct installed faucet in a Elementary School with gas DHW:

ΔTherms = 1 * ((1.39 – 0.94)/1.39) * 3,000 * 0.00446 * 0.95

= 4.12 Therms

### WATER IMPACT DESCRIPTIONS AND CALCULATION

Δgallons = ((GPM_base - GPM_low)/GPM_base) * Usage * ISR

Variables as defined above

**EXAMPLE**

For example, a direct installed faucet in a large office:

Δgallons = ((1.39 – 0.94)/1.39) * 11,250 * 0.95

= 3,640 gallons

For example, a direct installed faucet in a Elementary School:

Δgallons = ((1.39 – 0.94)/1.39) * 3,000 * 0.95

= 971 gallons

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

**SOURCES USED FOR GPM ASSUMPTIONS**

| Source ID | Reference |
|---|---|
| 1 | 2011, DeOreo, William. California Single Family Water Use Efficiency Study. April 20, 2011. |
| 2 | 2000, Mayer, Peter, William DeOreo, and David Lewis. Seattle Home Water Conservation Study. December 2000. |
| 3 | 1999, Mayer, Peter, William DeOreo. Residential End Uses of Water. Published by AWWA Research Foundation and American Water Works Association. 1999. |
| 4 | 2003, Mayer, Peter, William DeOreo. Residential Indoor Water Conservation Study. Aquacraft, Inc. Water Engineering and Management. Prepared for East Bay Municipal Utility District and the US EPA. July 2003. |
| 5 | 2011, DeOreo, William. Analysis of Water Use in New Single Family Homes. By Aquacraft. For Salt Lake City Corporation and US EPA. July 20, 2011. |
| 6 | 2011, Aquacraft. Albuquerque Single Family Water Use Efficiency and Retrofit Study. For Albuquerque Bernalillo County Water Utility Authority. December 1, 2011. |
| 7 | 2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings. |

**MEASURE CODE: CI-HWE-LFFA-V05-140601**

## 4.3.3 Low Flow Showerheads

**DESCRIPTION**

This measure relates to the direct installation of a low flow showerhead in a commercial building. Expected applications include small business, office, restaurant, or small motel. For multifamily or senior housing, the residential low flow showerhead should be used.

This measure was developed to be applicable to the following program types: DI.

If applied to other program types, the measure savings should be verified

**DEFINITION OF EFFICIENT EQUIPMENT**

To qualify for this measure the installed equipment must be an energy efficient showerhead rated at 2.0 gallons per minute (GPM) or less. Savings are calculated on a per showerhead fixture basis.

**DEFINITION OF BASELINE EQUIPMENT**

The baseline condition is assumed to be a standard showerhead rated at 2.5 GPM.

**DEEMED LIFETIME OF EFFICIENT EQUIPMENT**

The expected measure life is assumed to be 10 years.[192]

**DEEMED MEASURE COST**

The incremental cost for this measure is $12[193] or program actual.

**LOADSHAPE**

Loadshape C02 - Commercial Electric DHW

**COINCIDENCE FACTOR**

The coincidence factor for this measure is assumed to be 2.78%[194].

[192] Table C-6, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. Evaluations indicate that consumer dissatisfaction may lead to reductions in persistence, particularly in Multi-Family , "http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf"

[193] Direct-install price per showerhead assumes cost of showerhead (Market research average of $7 and assess and install time of $5 (20min @ $15/hr)

[194] Calculated as follows: Assume 11% showers take place during peak hours (based on: http://www.aquacraft.com/sites/default/files/pub/DeOreo-%282001%29-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf). There are 65 days in the summer peak period, so the percentage of total annual aerator use in peak period is 0.11*65/365 = 1.96%. The number of hours of recovery during peak periods is therefore assumed to be 1.96% * 369 = 7.23 hours of recovery during peak period. There are 260 hours in the peak period so the probability you will see savings during the peak period is 7,23/260 = 0..0278

**Algorithm**

**CALCULATION OF SAVINGS** [195]

**ELECTRIC ENERGY SAVINGS**

Note these savings are per showerhead fixture

ΔkWh =

%ElectricDHW * ((GPM_base * L_base - GPM_low * L_low) * NSPD * 365.25) * EPG_electric * ISR

Where:

%ElectricDHW = proportion of water heating supplied by electric resistance heating

= 1 if electric DHW, 0 if fuel DHW, if unknown assume 16% [196]

GPM_base = Flow rate of the baseline showerhead

= 2.67 for Direct-install programs[197]

GPM_low = As-used flow rate of the low-flow showerhead, which may, as a result of measurements of program evaulations deviate from rated flows, see table below:

| Rated Flow |
|---|
| 2.0 GPM |
| 1.75 GPM |
| 1.5 GPM |
| Custom or Actual[198] |

L_base = Shower length in minutes with baseline showerhead

= 8.20 min[199]

L_low = Shower length in minutes with low-flow showerhead

= 8.20 min[200]

365.25 = Days per year, on average.

NSPD = Estimated number of showers taken per day for one showerhead

EPG_electric = Energy per gallon of hot water supplied by electric

[195] Based on excel spreadsheet 120911.xls …on SharePoint

[196] Table HC8.9. Water Heating in U.S. Homes in Midwest Region, Divisions, and States, 2009 (RECS)

[197] Based on measured data from Ameren IL EM&V of Direct-Install program. Program targets showers that are rated 2.5 GPM or above.

[198] Note that actual values may be either a) program-specific minimum flow rate, or b)program-specific evaluation-based value of actual effective flow-rate due to increased duration or temperatures. The latter increases in likelihood as the rated flow drops and may become significant at or below rated flows of 1.5 GPM. The impact can be viewed as the inverse of the throttling described in the footnote for baseline flowrate.

[199] Representative value from sources 1, 2, 3, 4, 5, and 6 (See Source Table at end of measure section)

[200] Set equal to L_base.

= (8.33 * 1.0 * (ShowerTemp - SupplyTemp)) / (RE_electric * 3412)

= (8.33 * 1.0 * (105 – 54.1)) / (0.98 * 3412)

= 0.127 kWh/gal

8.33 = Specific weight of water (lbs/gallon)

1.0 = Heat Capacity of water (btu/lb-°F)

ShowerTemp = Assumed temperature of water

= 105°F [201]

SupplyTemp = Assumed temperature of water entering house

= 54.1°F [202]

RE_electric = Recovery efficiency of electric water heater

= 98% [203]

3412 = Converts Btu to kWh (btu/kWh)

ISR = In service rate of showerhead

= Dependant on program delivery method as listed in table below

| Selection | ISR[204] |
|---|---|
| Direct Install - Deemed | 0.98 |

**EXAMPLE**

For example, a direct-installed 1.5 GPM showerhead in an office with electric DHW where the number of showers is estimated at 3 per day:

ΔkWh = 1 * ((2.67*8.20)- (1.5*8.20)) * 3*365.25) *0.127 * 0.98

= 1308.4 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ΔkWh/Hours * CF

Where:

ΔkWh = calculated value above

Hours = Annual electric DHW recovery hours for showerhead use

[201] Shower temperature cited from SBW Consulting, Evaluation for the Bonneville Power Authority, 1994, http://www.bpa.gov/energy/n/reports/evaluation/residential/faucet_aerator.cfm

[202] US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html.

[203] Electric water heaters have recovery efficiency of 98%: http://www.ahridirectory.org/ahridirectory/pages/home.aspx

[204] Deemed values are from ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program Table 3-8. Alternative ISRs may be developed for program delivery methods based on evaluation results.

= ((GPM_base * L_base) *NSPD * 365.25 ) * 0.773[205] / GPH

Where:

GPH = Gallons per hour recovery of electric water heater calculated for 65.9F temp rise (120-54.1), 98% recovery efficiency, and typical 4.5kW electric resistance storage tank.

= 27.51

CF = Coincidence Factor for electric load reduction

= 0.0278[206]

**EXAMPLE**

For example, a direct-installed 1.5 GPM showerhead in an office with electric DHW where the number of showers is estimated at 3 per day:

ΔkW = (1308.4 / 674.1)*0.0278

= 0.054 kW

**FOSSIL FUEL IMPACT DESCRIPTIONS AND CALCULATION**

ΔTherms = %FossilDHW * ((GPM_base * L_base - GPM_low * L_low) * NSPD* 365.25) * EPG_gas * ISR

Where:

%FossilDHW = proportion of water heating supplied by fossil fuel heating

| DHW fuel | %Fossil_DHW |
|---|---|
| Electric | 0% |
| Fossil Fuel | 100% |
| Unknown | 84%[207] |

EPG_gas = Energy per gallon of Hot water supplied by gas

= (8.33 * 1.0 * (ShowerTemp - SupplyTemp)) / (RE_gas * 100,000)

= 0.0063 Therm/gal

[205] 77.3% is the proportion of hot 120F water mixed with 54.1°F supply water to give 105°F shower water

[206] Calculated as follows: Assume 11% showers take place during peak hours (based on: http://www.aquacraft.com/sites/default/files/pub/DeOreo-%282001%29-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf). There are 65 days in the summer peak period, so the percentage of total annual aerator use in peak period is 0.11*65/365.25 = 1.96%. The number of hours of recovery during peak periods is therefore assumed to be 1.96% * 369 = 7.23 hours of recovery during peak period where 369 equals the average annual electric DHW recovery hours for showerhead use including SF and MF homes with Direct Install and Retrofit/TOS measures. There are 260 hours in the peak period so the probability you will see savings during the peak period is 7.23/260 = 0.0278

[207] Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

Where:

| | |
|---|---|
| RE_gas | = Recovery efficiency of gas water heater |
| | = 67% [208] |
| 100,000 | = Converts Btus to Therms (btu/Therm) |

Other variables as defined above.

**EXAMPLE**

For example, a direct-installed 1.5 GPM showerhead in an office with gas DHW where the number of showers is estimated at 3 per day:

ΔTherms = 1.0 * (( 2.67 *8.2) – (1.5 * 8.2)) * 3 * 365.25 * 0.0063 * 0.98

= 64.9 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

Δgallons = ((GPM_base * L_base - GPM_low * L_low) * NSPD * 365.25 * ISR

Variables as defined above

**EXAMPLE**

For example, a direct-installed 1.5 GPM showerhead in an office with where the number of showers is estimated at 3 per day:

Δgallons = ((2.67 * 8.20)-(1.5 * 8.20)) * 3 * 365.25 * 0.98

= 10,302 gallons

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

[208] Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 75%. Commercial properties are more similar to MF homes than SF homes. MF hot water is often provided by a larger commercial boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of .59 and the .75 for single family home. An average is used for this analysis by default.

**SOURCES**

| Source ID | Reference |
|---|---|
| 1 | 2011, DeOreo, William. California Single Family Water Use Efficiency Study. April 20, 2011. |
| 2 | 2000, Mayer, Peter, William DeOreo, and David Lewis. Seattle Home Water Conservation Study. December 2000. |
| 3 | 1999, Mayer, Peter, William DeOreo. Residential End Uses of Water. Published by AWWA Research Foundation and American Water Works Association. 1999. |
| 4 | 2003, Mayer, Peter, William DeOreo. Residential Indoor Water Conservation Study. Aquacraft, Inc. Water Engineering and Management. Prepared for East Bay Municipal Utility District and the US EPA. July 2003. |
| 5 | 2011, DeOreo, William. Analysis of Water Use in New Single Family Homes. By Aquacraft. For Salt Lake City Corporation and US EPA. July 20, 2011. |
| 6 | 2011, Aquacraft. Albuquerque Single Family Water Use Efficiency and Retrofit Study. For Albuquerque Bernalillo County Water Utility Authority. December 1, 2011. |
| 7 | 2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings. |

**MEASURE CODE: CI-HW_-LFSH-V03-150601**

## 4.3.4 Commercial Pool Covers

### DESCRIPTION

This measure refers to the installation of covers on commercial use pools that are heated with gas-fired equipment located either indoors or outdoors. By installing pool covers, the heating load on the pool boiler will be reduced by reducing the heat loss from the water to the environment and the amount of actual water lost due to evaporation (which then requires additional heated water to make up for it).

The main source of energy loss in pools is through evaporation. This is particularly true of outdoor pools where wind plays a larger role. The point of installing pool covers is threefold. First, it will reduce convective losses due to the wind by shielding the water surface. Second, it will insulate the water from the colder surrounding air. And third, it will reduce radiative losses to the night sky. In doing so, evaporative losses will also be minimized, and the boiler will not need to work as hard in replenishing the pool with hot water to keep the desired temperature.

This measure can be used for pools that (1) currently do not have pool covers, (2) have pool covers that are past the useful life of the existing cover, or (3) have pool covers that are past their warranty period and have failed.

### DEFINITION OF EFFICIENT EQUIPMENT

For indoor pools, the efficient case is the installation of an indoor pool cover with a 5 year warranty on an indoor pool that operates all year.

For outdoor pools, the efficient case is the installation of an outdoor pool cover with a 5 year warranty on an outdoor pool that is open through the summer season.

### DEFINITION OF BASELINE EQUIPMENT

For indoor pools, the base case is an uncovered indoor pool that operates all year.

For outdoor pools, the base case is an outdoor pool that is uncovered and is open through the summer season.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The useful life of this measure is assumed to be 6 years [209]

### DEEMED MEASURE COST

The table below shows the costs for the various options and cover sizes. Since this measure covers a mix of various sizes, the average cost of these options is taken to be the incremental measure cost. [210].

[209] The effective useful life of a pool cover is typically one year longer than its warranty period. SolaPool Covers. Pool Covers Website, FAQ- "How long will my SolaPool cover blanket last?". Pool covers are typically offered with 3 and 5 year warranties with at least one company offering a 6 year warranty. Conversation with Trade Ally. Knorr Systems

[210] Pool Cover Costs: Lincoln Commercial Pool Equipment website. Accessed 8/26/11. http://www.lincolnaquatics.com/shop/catalog/Pool+and+Spa+Covers+and+Accessories/product.html?ProductID=84-010

| Cover Size | Edge Style | |
|---|---|---|
| | Hemmed (indoor) | Weighted (outdoor) |
| 1000-1,999 sq. ft. | $2.19 | $2.24 |
| 2,000-2,999 sq. ft. | $2.01 | $2.06 |
| 3,000+ sq. ft. | $1.80 | $1.83 |
| Average | $2.00 | $2.04 |

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

### NET TO GROSS RATIO

**Algorithm**

### CALCULATION OF ENERGY SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS SAVINGS

The calculations are based on modeling runs using RSPEC! Energy Smart Pools Software that was created by the U.S. Department of Energy. [211]

$$\Delta Therms = SavingFactor \times Size\ of\ Pool$$

Where

Savings factor = dependant on pool location and listed in table below[212]

| Location | Therm / sq-ft |
|---|---|
| Indoor | 2.61 |
| Outdoor | 1.01 |

Size of Pool = custom input

[211] Full method and supporting information found in reference document: IL TRM - Business Pool Covers WorkPaper.docx. Note that the savings estimates are based upon Chicago weather data.
[212] Business Pool Covers.xlsx

### WATER IMPACT DESCRIPTIONS AND CALCULATION

$$\Delta Therms = WaterSavingFactor \times Size\ of\ Pool$$

Where

WaterSavingFactor = Water savings for this measure dependant on pool location and listed in table below.[213].

| Location | Annual Savings Gal / sq-ft |
|---|---|
| Indoor | 15.28 |
| Outdoor | 8.94 |

Size of Pool = Custom input

### DEEMED O&M COST ADJUSTMENT CALCULATION

There are no O&M cost adjustments for this measure.

### MEASURE CODE: CI-HW_-PLCV-V01-130601

[213] Ibid.

## 4.3.5 Tankless Water Heater

### DESCRIPTION

This measure covers the installation of on-demand or instantaneous tankless water heaters. Tankless water heaters function similar to standard hot water heaters except they do not have a storage tank. When there is a call for hot water, the water is heated instantaneously as it passes through the heating element and then proceeds to the user or appliance calling for hot water. Tankless water heaters achieve savings by eliminating the standby losses that occur in stand-alone or tank-type water heaters and by being more efficient than the baseline storage hot water heater.

This measure was developed to be applicable to the following program types: TOS, RF, ER.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

| Electric | Gas |
|---|---|
| To qualify for this measure, the tankless water heater shall be a new electric powered tankless hot water heater with an energy factor greater than or equal to 0.98 with an output greater than or equal to 5 GPM output at 70° F temperature rise. | To qualify for this measure, the tankless water heater shall meet or exceed the efficiency requirements for tankless hot water heaters mandated by the International Energy Conservation Code (IECC) 2012, Table C404.2. |

### DEFINITION OF BASELINE EQUIPMENT

| Electric | Gas |
|---|---|
| The baseline condition is assumed to be an electric commercial-grade tanked water heater 50 or more gallon storage capacity with an energy factor less than or equal to 0.9 or the water heater is five or more years old. | The baseline condition is assumed to be a gas-fired tank-type water heater meeting the efficiency requirements mandated by the International Energy conservation Code (IECC) 2012, Table C404.2. |

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

| Electric | Gas |
|---|---|
| The expected measure life is assumed to be 5 years[214]. | The expected measure life is assumed to be 20 years[215] |

### DEEMED MEASURE COST

The incremental capital cost for an electric tankless heater this measure is assumed to be[216]

[214] Ohio Technical Reference Manual 8/2/2010 referencing CenterPoint Energy-Triennial CIP/DSM Plan 2010-2012 Report; Additional reference stating >20 years is at Energy Savers.Gov online at http://www.energysavers.gov/your_home/water_heating/index.cfm/mytopic=12820

[215] Ibid.

[216] Act on Energy Technical Reference Manual, Table 9.6.2-3

| Output (gpm) at delta T 70 | Incremental Cost |
|---|---|
| 5 | $1050 |
| 10 | $1050 |
| 15 | $1950 |

The incremental capital cost for a gas fired tankless heater is as follows:

| Program | Capital Cost, $ per unit |
|---|---|
| Retrofit | $3,255[217] |
| Time of Sale or New Construction | $2,526[218] |

### DEEMED O&M COST ADJUSTMENTS

$100[219]

### LOADSHAPE

Loadshape C02 - Commercial Electric DHW

### COINCIDENCE FACTOR

The measure has deemed kW savings therefor a coincidence factor is not applied

[217] Based on AOE historical average installation data of 42 tankless gas hot water heaters

[218] http://www.mncee.org/getattachment/7b8982e9-4d95-4bc9-8e64-f89033617f37/, Low contractor estimate used to reflect less labor required in new construction of venting.

[219] Water heaters (WH) require annual maintenance. There are different levels of effort for annual maintenance depending if the unit is gas or electric, tanked or tankless. Electric and gas tank water heater manufacturers recommend an annual tank drain to clear sediments. Also recommended are "periodic" inspections by qualified service professionals of operating controls, heating element and wiring for electric WHs and thermostat, burner, relief valve internal flue-way and venting systems for gas WHs. Tankless WH require annual maintenance by licensed professionals to clean control compartments, burners, venting system and heat exchangers. This information is from WH manufacturer product brochures including GE, Rennai, Rheem, Takagi and Kenmore. References for incremental O&M costs were not found. Therefore the incremental cost of the additional annual maintenance for tankless WH is estimated at $100.

**Algorithm**

## CALCULATION OF ENERGY SAVINGS

### ELECTRIC ENERGY SAVINGS [220]

The annual electric savings from an electric tankless heater is a deemed value and assumed to be:

| Output (gpm) at delta T 70 | Savings (kWh) |
|---|---|
| 5.0 | 2,992 |
| 10.0 | 7,905 |
| 15.0 | 12,879 |

### SUMMER COINCIDENT PEAK DEMAND SAVINGS[221]

The annual kW savings from an electric tankless heater is a deemed value and assumed to be:

| Output (gpm) at delta T 70 | Savings (kW) |
|---|---|
| 5.0 | 0.34 |
| 10.0 | 0.90 |
| 15.0 | 1.47 |

### NATURAL GAS SAVINGS

ΔTherms =[[Wgal x 8.33 x 1 x (Tout - Tin) x [(1/Eff base) - (1/Eff ee)]]/100,000] +[[(SL x 8,766)/Eff base]] / 100,000 Btu/Therms]

Where:

Wgal = Annual water use for equipment in gallons

= custom, otherwise assume 21,915 gallons [222]

8.33 lbm/gal = weight in pounds of one gallon of water

1 Btu/lbm°F = Specific heat of water: 1 Btu/lbm/°F

8,766 hr/yr = hours a year

Tout = Unmixed Outlet Water Temperature

= custom, otherwise assume 130 °F[223]

[220] Act on Energy Technical Reference Manual, Table 9.6.2-3
[221] Ibid.
[222] 21,915 gallons is an estimate of 60 gal/day for 365.25 days/yr. If building type is known, reference 2007 ASHRAE Handbook HVAC Applications p. 49.14 Table 7 Hot Water Demands and Use for Various Types of Buildings to help estimate hot water consumption.
[223] Based on 2010 Ohio Techical Reference Manual and NAHB Research Center, (2002) Performance Comparison of Residential hot Water Systems. Prepared for National Renewable Energy Laboratory, Golden, Colorado.

Tin = Inlet Water Temperature

= custom, otherwise assume 54.1 °F[224]

Eff base = Rated efficiency of baseline water heater expressed as Energy Factor (EF) or Thermal Efficiency (Et); see table below[225]

| Input Btu/hr of existing, tanked water heater | Eff base | Units |
|---|---|---|
| Size: ≤ 75,000 Btu/hr | 0.67 -0.0019*Tank Volume | Energy Factor |
| Size: >75,000 Btu/hr and ≤ 155,000 Btu/hr | 80% | Thermal Efficiency |
| Size: >155,000 Btu/hr | 80% | Thermal Efficiency |

Where:

Tank Volume = custom input, if unknown assume 60 gallons for Size: ≤ 75,000 Btu/hr

Please note: Units in base case must match units in efficient case. If Energy Factor used in base case, Energy Factor to be used in efficient case. If Themal Efficiency is used in base case, Thermal Efficiency must be used in efficient case.

Eff ee = Rated efficiency of efficient water heater expressed as Energy Factor (EF) or Thermal Efficiency (Eff t)

= custom input, if unknown assume 0.84[226]

SL = Stand-by Loss in Base Case Btu/hr

= custom input based on formula in table below, if unknown assume unit size in table below[227]

| Input Btu/h of new, tankless water heater | Standby Loss (SL) |
|---|---|
| Size: ≤ 75,000 Btu/hr | 0 |
| Size: >75,000 Btu/hr | (Input rating/800)+(110*√Tank Volume)) |

Where:

Tank Volume = custom input, if unknown assume, 60 gallons for <75,000 Btu/hr, 75 gallons for >75,000 Btu/hr and ≤ 155,000 Btu/hr and 150 for Size >155,000 Btu/hr

Input Rating = nameplate Btu/hr rating of water heater

[224] August 31, 2011 Memo of Savings for Hot Water Savings Measures to Nicor Gas from Navigant states that 54.1°F was calculated from the weighted average of monthly water mains temperatures reported in the 2010 Building America Benchmark Study for Chicago-Waukegan, Illinois.

[225] IECC 2012, Table C404.2, Minimum Performance of Water-Heating Equipment

[226] Specifications of energy efficient tankless water heater. Reference Consortium for Energy Efficiency (CEE) which maintains a list of high efficiency tankless water heaters which currently have Energy Factors up to .96. Ameren currently requires minimum .82 energy factor.

[227] Stand-by loss is provided 2012 International Energy Conservation Code (IECC2012), Table C404.2, Minimum Performance of Water-Heating Equipment

**EXAMPLE**

For example, a 75,000 Btu/hr tankless unit using 21,915 gal/yr with outlet temperature at 130.0 and inlet temperature at 54.1, replacing a baseline unit with 0.8 thermal efficiency and standby losses of 1008.3 btu/hr:

ΔTherms =[[(21,915 x 8.33x 1 x (130 – 54.1) x [(1/.8) - (1/.84)]/100,000] +[(1008.3 x 8,766)/.8]] / 100,000

=115 Therms

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

The deemed O&M cost adjustment for a gas fired tankless heater is $100

### REFERENCE TABLES

Minimum Performance Water Heating Equipment[228]

[228] International Energy Conservation Code (IECC)2012

TABLE C404.2
MINIMUM PERFORMANCE OF WATER-HEATING EQUIPMENT

| EQUIPMENT TYPE | SIZE CATEGORY (input) | SUBCATEGORY OR RATING CONDITION | PERFORMANCE REQUIRED[a, b] | TEST PROCEDURE |
|---|---|---|---|---|
| Water heaters, electric | ≤ 12 kW | Resistance | 0.97 - 0.00132 V, EF | DOE 10 CFR Part 430 |
| | > 12 kW | Resistance | 1.73 V+ 155 SL, Btu/h | ANSI Z21.10.3 |
| | ≤ 24 amps and ≤ 250 volts | Heat pump | 0.93 - 0.00132 V, EF | DOE 10 CFR Part 430 |
| Storage water heaters, gas | ≤ 75,000 Btu/h | ≥ 20 gal | 0.67 - 0.0019 V, EF | DOE 10 CFR Part 430 |
| | > 75,000 Btu/h and ≤ 155,000 Btu/h | < 4,000 Btu/h/gal | 80% $E_t$ (Q/800 + 110 $\sqrt{V}$) SL, Btu/h | ANSI Z21.10.3 |
| | > 155,000 Btu/h | < 4,000 Btu/h/gal | 80% $E_t$ (Q/800 + 110 $\sqrt{V}$) SL, Btu/h | |
| Instantaneous water heaters, gas | > 50,000 Btu/h and < 200,000 Btu/h[c] | ≥ 4,000 (Btu/h)/gal and < 2 gal | 0.62 - 0.0019 V, EF | DOE 10 CFR Part 430 |
| | ≥ 200,000 Btu/h | ≥ 4,000 Btu/h/gal and < 10 gal | 80% $E_t$ | ANSI Z21.10.3 |
| | ≥ 200,000 Btu/h | ≥ 4,000 Btu/h/gal and ≥ 10 gal | 80% $E_t$ (Q/800 + 110 $\sqrt{V}$) SL, Btu/h | |
| Storage water heaters, oil | ≤ 105,000 Btu/h | ≥ 20 gal | 0.59 - 0.0019 V, EF | DOE 10 CFR Part 430 |
| | ≥ 105,000 Btu/h | < 4,000 Btu/h/gal | 78% $E_t$ (Q/800 + 110 $\sqrt{V}$) SL, Btu/h | ANSI Z21.10.3 |
| Instantaneous water heaters, oil | ≤ 210,000 Btu/h | ≥ 4,000 Btu/h/gal and < 2 gal | 0.59 - 0.0019V, EF | DOE 10 CFR Part 430 |
| | > 210,000 Btu/h | ≥ 4,000 Btu/h/gal and < 10 gal | 80% $E_t$ | ANSI Z21.10.3 |
| | > 210,000 Btu/h | ≥ 4,000 Btu/h/gal and ≥ 10 gal | 78% $E_t$ (Q/800 + 110 $\sqrt{V}$) SL, Btu/h | |
| Hot water supply boilers, gas and oil | ≥ 300,000 Btu/h and < 12,500,000 Btu/h | ≥ 4,000 Btu/h/gal and < 10 gal | 80% $E_t$ | ANSI Z21.10.3 |
| Hot water supply boilers, gas | ≥ 300,000 Btu/h and < 12,500,000 Btu/h | ≥ 4,000 Btu/h/gal and ≥ 10 gal | 80% $E_t$ (Q/800 + 110 $\sqrt{V}$) SL, Btu/h | |
| Hot water supply boilers, oil | > 300,000 Btu/h and < 12,500,000 Btu/h | > 4,000 Btu/h/gal and > 10 gal | 78% $E_t$ (Q/800 + 110 $\sqrt{V}$) SL, Btu/h | |
| Pool heaters, gas and oil | All | — | 78% $E_t$ | ASHRAE 146 |
| Heat pump pool heaters | All | — | 4.0 COP | AHRI 1160 |
| Unfired storage tanks | All | — | Minimum insulation requirement R-12.5 (h · ft² · °F)/Btu | (none) |

For SI: °C = [(°F) - 32]/1.8, 1 British thermal unit per hour = 0.2931 W, 1 gallon = 3.785 L, 1 British thermal unit per hour per gallon = 0.078 W/L.

a. Energy factor (EF) and thermal efficiency ($E_t$) are minimum requirements. In the EF equation, $V$ is the rated volume in gallons.

b. Standby loss (SL) is the maximum Btu/h based on a nominal 70°F temperature difference between stored water and ambient requirements. In the SL equation, $Q$ is the nameplate input rate in Btu/h. In the SL equation for electric water heaters, $V$ is the rated volume in gallons. In the SL equation for oil and gas water heaters and boilers, $V$ is the rated volume in gallons.

c. Instantaneous water heaters with input rates below 200,000 Btu/h must comply with these requirements if the water heater is designed to heat water to temperatures 180°F or higher.

**MEASURE CODE: CI-HW_-TKWH-V02-120601**

## 4.3.6 Ozone Laundry

### DESCRIPTION

A new ozone laundry system(s) is added-on to new or existing commercial washing machine(s) using hot water heated with natural gas. The system generates ozone ($O_3$), a naturally occurring molecule, which helps clean fabrics by chemically reacting with soils in cold water. Adding an ozone laundry system(s) will reduce the amount of chemicals, detergents, and hot water needed to wash linens. Using ozone also reduces the total amount of water consumed, saving even more in energy.

Natural gas energy savings will be achieved at the hot water heater/boiler as they will be required to produce less hot water to wash each load of laundry. The decrease in hot water usage will increase cold water usage, but overall water usage at the facility will decrease.

Electric savings can be realized through reduced washer cycle length and reduced pumping load at the boiler feeding the commercial washers. The increased usage associated with operating the ozone system should also be accounted for when determining total kWh impact. Data reviewed for this measure characterization indicated that pumping savings should be accounted for, but washer savings and ozone generator consumption are comparatively so small that they can be ignored.

The reduced washer cycle length may decrease the dampness of the clothes when they move to the dryer. This can result in shorter runtimes which result in gas and electrical savings. However, at this time, there is inconclusive evidence that energy savings are achieved from reduced dryer runtimes so the resulting dryer effects are not included in this analysis. Additionally, there would be challenges verifying that dryer savings will be achieved throughout the life of the equipment.

This incentive only applies to the following facilities with on-premise laundry operations:

- Hotels/motels
- Fitness and recreational sports centers.
- Healthcare (excluding hospitals)
- Assisted living facilities

Ozone laundry system(s) could create significant energy savings opportunities at other larger facility types with on-premise laundry operations (such as correctional facilities, universities, and staff laundries), however, the results included in this analysis are based heavily on past project data for the applicable facility types listed above and may not apply to facilities outside of this list due to variances in number of loads and average pound (lbs.)-capacity per project site. Projects at these facilities should continue to be evaluated through custom programs and the applicable facility types and the resulting analysis should be updated based on new information.

This measure was developed to be applicable to the following program types: TOS, RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

A new ozone laundry system(s) is added-on to new or existing commercial washing machine(s) using hot water heated with natural gas. The ozone laundry system(s) must transfer ozone into the water through:

- Venturi Injection
- Bubble Diffusion
- Additional applications may be considered upon program review and approval on a case by case basis

### DEFINITION OF BASELINE EQUIPMENT

The base case equipment is a conventional washing machine system with no ozone generator installed. The washing machines are provided hot water from a gas-fired boiler.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure equipment effective useful life (EUL) is estimated at 10 years based on typical lifetime of the ozone generator's corona discharge unit.[229]

### DEEMED MEASURE COST

The actual measure costs should be used if available. If not a deemed value of $79.84 / lbs capacity should be used[230].

### LOADSHAPE

Loadshape C53 – Flat

### COINCIDENCE FACTOR

Past project documentation and data collection is not sufficient to determine a coincidence factor for this measure. Value should continue to be studied and monitored through additional studies due to limited data points used for this determination

## Algorithm

### CALCULATION OF ENERGY SAVINGS

### ELECTRIC ENERGY SAVINGS

Electric savings can be realized through reduced washer cycle length and reduced pumping load at the boiler feeding the commercial washers. There is also an increased usage associated with operating the ozone system. Data reviewed for this measure characterization indicated that while pumping savings is significant and should be accounted for, washer savings and ozone generator consumption are negligible, counter each other out and are well within the margin of error so these are not included to simplify the characterization[231].

$$\Delta kWh_{PUMP} = HP * HP_{CONVERSION} * Hours * \%water_savings$$

[229] Aligned with other national energy efficiency programs and confirmed with national vendors

[230] Average costs per unit of capacity were generated using data collected from existing ozone laundry projects that received incentives under the Non-Residential Retrofit Demand Reduction program (NRR-DR), as well as from the Nicor Custom Incentive Program, and the Nicor Emerging Technology Program (ETP). See referenced document Table 2 and RSMeans Mechanical Cost Data, 31st Annual Edition (2008)

[231] Washer savings were reviewed but were considered negligible and not included in the algorithm (0.00082 kWh / lbs-capacity, determined through site analysis through Nicor Emerging Technology Program (ETP) and confirmed with national vendors). Note that washer savings from Nicor's site analysis are smaller than those reported in a WI Focus on Energy case study (0.23kWh/100lbs, Hampton Inn Brookfield, November 2010). Electric impact of operating ozone generator (0.0021 kWh / lbs-capacity same source as washer savings) was also considered negligible and not included in calculations. Values should continue to be studied and monitored through additional studies due to limited data points used for this determination.

Where:

$\Delta kWh_{PUMP}$ = Electric savings from reduced pumping load

HP = Brake horsepower of boiler feed water pump;

= Actual or use 5 HP if unknown[232]

$HP_{CONVERSION}$ = Conversion from Horsepower to Kilowatt

= 0.746

Hours = Actual associated boiler feed water pump hours

= 800 hours if unknown[233]

%water_savings = water reduction factor: how much more efficient an ozone injection washing machine is compared to a typical conventional washing machine as a rate of hot and cold water reduction.

= 25%[234]

Using defaults above:

$\Delta kWh_{PUMP}$ = 5 * 0.746 * 800 * 0.25

= 746 kWh

Default per lb capacity: = $\Delta kWh_{PUMP}$ / lb capacity

Where:

Lbs-Capacity = Average Capacity in lbs of washer

=254.38[235]

$\Delta kWh_{PUMP}$ / lb capacity = 746/254.38

= 2.93 kWh/lb-capacity

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

Past project documentation and data collection is not sufficient to determine summer coincident peak demand savings for this measure. Value should continue to be studied and monitored through additional studies due to limited data points used for this determination. In absence of site-specific data, the summer coincident peak demand savings should be assumed to be zero.

[232] Assumed average horsepower for boilers connected to applicable washer

[233] Engineered estimate provided by CLEAResult review of Nicor custom projects. Machines spent approximately 7 minutes per hour filling with water and were in operation approximately 20 hours per day. Total pump time therefore estimated as 7/60 * 20 * 365 = 852 hours, and rounded down conservatively to 800 hours.

[234] Average water reduction factors were generated using data collected from existing ozone laundry projects that received incentives under the Non-Residential Retrofit Demand Reduction program (NRR-DR). Table 6 summarizes data gathered from several NRR-DR projects, Nicor Custom projects, and Nicor ETP projects. Nicor Savings Numbers are associated with ACEE_AWE_Ozone Laundry / From Gas Savings Calculations

[235] Average lbs-capacity per project site was generated using data collected from existing ozone laundry projects that received incentives under the Non-Residential Retrofit Demand Reduction program (NRR-DR), as well as from the Nicor Custom Incentive Program, and the Nicor Emerging Technology Program (ETP). See referenced document Table 2

$\Delta kW = 0$

**NATURAL GAS SAVINGS**

$\Delta Therm = Therm_{Baseline} * \%hot_water_savings$

Where:

$\Delta Therm$ = Gas savings resulting from a reduction in hot water use, in therm.

$Therm_{Baseline}$ = Annual Baseline Gas Consumption

= WHE * WUtiliz * WUsage_hot

Where:

WHE = water heating energy: energy required to heat the hot water used

= 0.00885 therm/gallon[236]

WUtiliz = washer utilitzation factor: the annual pounds of clothes washed per year

= actual, if unknown use 916,150 lbs laundry[237], approximately equivalent to 13 cycles/day

WUsage_hot = hot water usage factor: how much hot water a typical conventional washing machine utilizes, normalized per pounds of clothes washed

= 1.19 gallons/lbs laundry[238]

Using defaults above:

$Therm_{Baseline}$ = 0.00885 * 916,150 * 1.19

= 9,648 therms

Default per lb capacity:

$Therm_{Baseline}$ / lb capacity = 9,648 / 254.38

= 37.9 therms / lb-capacity

%hot_water_savings = hot water reduction factor: how much more efficient an ozone injection washing machine is, compared to a typical conventional washing machine, as a rate of hot water reduction

= 81%[239]

[236] Assuming boiler efficiency is the regulated minimum efficiency (80%), per Title 20 Appliance Standard of the California Energy Regulations (October 2007). The incoming municipal water temperature is assumed to be 55 °F with an average hot water supply temperature of 140°F, based on default test procedures on clothes washers set by the Department of Energy's Office of Energy Efficiency and Renewable Energy (Federal Register, Vol. 52, No. 166). Enthalpies for these temperatures (107 btu/lbs at 140F, 23.07 btu/lbs at 55F) were obtained from ASHRAE Fundamentals

[237] Average utilization factors were generated using data collected from existing ozone laundry projects that received incentives under the NRR-DR program. Table 3 summarizes data gathered from several NRR-DR projects, Nicor Custom projects, and Nicor ETP projects

[238] Average hot water usage factors were generated using data collected from existing ozone laundry projects that received incentives under the NRR-DR program. summarizes data gathered from several NRR-DR projects:

[239] Average hot water reduction factors were generated using data collected from existing ozone laundry projects that received incentives under the Non-Residential Retrofit Demand Reduction program (NRR-DR). Table 5 summarizes data gathered from

Savings using defaults above:

ΔTherm = $Therm_{Baseline}$ * %hot_water_savings

= 9648 * 0.81

= 7,815 therms

Default per lb capacity:

ΔTherm / lb-capacity = 7815 / 254.38

= 30.7 therms / lb-capacity

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

The water savings calculations listed here account for the combination of hot and cold water used. Savings calculations for this measure were based on the reduction in total water use from implementing an ozone washing system to the base case. There are three main components in obtaining this value:

Δgallons = WUsage * WUtiliz * %water_savings

Where:

Δgallons = reduction in total water use from implementing an ozone washing system to the base case

WUsage = water usage factor: how efficiently a typical conventional washing machine utilized hot and cold water normalized per unit of clothes washed

= 2.03 gallons/lbs laundry[240]

WUtiliz = washer utilitzation factor: the annual pounds of clothes washed per year

= actual, if unknown use 916,150 lbs laundry[241], approximately equivalent to 13 cycles/day

%water_savings = water reduction factor: how much more efficient an ozone injection washing machine is compared to a typical conventional washing machine as a rate of hot and cold water reduction.

= 25%[242]

Savings using defaults above:

ΔGallons = WUsage * WUtiliz * %water_savings

= 2.03 * 916,150 * 0.25

= 464,946 gallons

Default per lb capacity:

several NRR-DR projects, Nicor Custom projects, and Nicor ETP projects. Nicor Savings Numbers are associated with ACEE_AWE_Ozone Laundry / From Gas Savings Calculations

[240] Average water usage factors were generated using data collected from existing ozone laundry projects that received incentives under the NRR-DR program. summarizes data gathered from several NRR-DR projects

[241] Average utilization factors were generated using data collected from existing ozone laundry projects that received incentives under the NRR-DR program. Table 3 summarizes data gathered from several NRR-DR projects, Nicor Custom projects, and Nicor ETP projects

[242] Average water reduction factors were generated using data collected from existing ozone laundry projects that received incentives under the Non-Residential Retrofit Demand Reduction program (NRR-DR). Table 6 summarizes data gathered from several NRR-DR projects, Nicor Custom projects, and Nicor ETP projects. Nicor Savings Numbers are associated with ACEE_AWE_Ozone Laundry / From Gas Savings Calculations

Δ Gallons / lb-capacity = 464,946 / 254.38

= 1,828 gallons / lb-capacity

### DEEMED O&M COST ADJUSTMENT CALCULATION

Maintenance is required for the following components annually:[243]

- Ozone Generator: filter replacement, check valve replacement, fuse replacement, reaction chamber inspection/cleaning, reaction chamber o-ring replacement
- Air Preparation – Heat Regenerative: replacement of two medias
- Air Preparation – Oxygen Concentrators: filter replacement, pressure relief valve replacement, compressor rebuild
- Venturi Injector: check valve replacement

Maintenance is expected to cost $0.79 / lbs capacity.

### REFERENCES

1 "Lodging Report", December 2008, California Travel & Tourism Commission, http://tourism.visitcalifornia.com/media/uploads/files/editor/Research/CaliforniaTourism_200812.pdf

2 "Health, United States, 2008" Table 120, U.S. Department of Health & Human Services, Centers for Disease Control & Prevention, National Center for Health Statistics, http://www.cdc.gov/nchs/data/hus/hus08.pdf#120

3 Fourth Quarter 2008 Facts and Fictures, California Department of Corrections & Rehabilitation (CDCR), http://www.cdcr.ca.gov/Divisions_Boards/Adult_Operations/docs/Fourth_Quarter_2008_Facts_and_Figures.pdf

4 Jail Profile Survey (2008), California Department of Corrections & Rehabilitation (CDCR), http://www.cdcr.ca.gov/Divisions_Boards/CSA/FSO/Docs/2008_4th_Qtr_JPS_full_report.pdf

5 DEER2011_NTGR_2012-05-16.xls from DEER Database for Energy-Efficient Resources; Version 2011 4.01 found at :http://www.deeresources.com/index.php?option=com_content&view=article&id=68&Itemid=60

Under: DEER2011 Update Documentation linked at: DEER2011 Update Net-To-Gross table Cells: T56 and U56

6 The Benefits of Ozone in Hospitality On-Premise Laundry Operations, PG&E Emerging Technologies Program, Application Assessment Report #0802, April 2009.

7 Federal Register, Vol. 52, No. 166

8 2009 ASHRAE Handbook – Fundamentals, Thermodynamic Properties of Water at Saturation, Section 1.1 (Table 3), 2009

9 Table 2 through 6: Excel file summarizing data collected from existing ozone laundry projects that received incentives under the NRR-DR program

### MEASURE CODE CI-HW-OZLD-V01-140601

[243] Confirmed through communications with national vendors and available references E.g. http://ozonelaundry.wordpress.com/2010/11/17/the-importance-of-maintenance/

## 4.3.7 Multifamily Central Domestic Hot Water Plants

### DESCRIPTION

This measure covers multifamily central domestic hot water (DHW) plants with thermal efficiencies greater than or equal to 88%. This measure is applicable to any combination of boilers and storage tanks provided the thermal efficiency of the boilers is greater than 88%. Plants providing other than solely DHW are not applicable to this measure.

This measure was developed to be applicable to the following program types: TOS, NC, ER.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify the boiler(s) must have a Thermal Efficiency of 88% or greater and supply domestic hot water to multi-family buildings.

### DEFINITION OF BASELINE EQUIPMENT

For TOS the baseline boiler is assumed to have a Thermal Efficiency of 80%.[244]

For Early Replacement the savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit as above and efficient unit consumption for the remainder of the measure life.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for the domestic hot water boilers is 15 years.[245]

### DEEMED MEASURE COST

TOS: The actual install cost should be used for the efficient case, minus the baseline cost assumption provided below:

| Capacity Range | Baseline Installed Cost per kBtuh[246] |
|---|---|
| <300kBtuh | $65 per kBTUh |
| 300 – 2500 kBtuh | $38 per kBTUh |
| >2500 kBtuh | $32 per kBTUh |

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

[244] IECC 2012, Table C404.2, Minimum Performance of Water-Heating Equipment
[245] Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011.
[246] Baseline install costs are based on data from the W017 Itron California Measure Cost Study, accessed via http://www.energydataweb.com/cpuc/search.aspx. The data is provided in a file named "MCS Results Matrix – Volume I".

N/A

**Algorithm**

**CALCULATION OF ENERGY SAVINGS**

**ELECTRIC ENERGY SAVINGS**

There are no anticipated electrical savings from this measure.

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

N/A

**NATURAL GAS SAVINGS**

Time of Sale:

ΔTherms = Hot Water Savings + Standby Loss Savings

= [(MFHH * #Units * GPD * Days/yr * ɣWater * (Tout – Tin) * (1/Eff_base – 1/Eff_ee)) / 100,000] + [((SL * Hours/yr * (1/Eff_base – 1/Eff_ee)) / 100,000]

Early Replacment[247]:

ΔTherms for remaining life of existing unit (1st 5 years):

= [(MFHH * #Units * GPD * Days/yr * ɣWater * (Tout – Tin) * (1/Eff_exist – 1/Eff_ee)) / 100,000] + [((SL * Hours/yr * (1/Eff_exist – 1/Eff_ee)) / 100,000]

ΔTherms for remaining measure life (next 10 years):

= [(MFHH * #Units * GPD * Days/yr * ɣWater * (Tout – Tin) * (1/Eff_base – 1/Eff_ee)) / 100,000] + [((SL * Hours/yr * (1/Eff_base – 1/Eff_ee)) / 100,000]

Where:

MFHH = number of people in Multi-Family House Hold

= Actual. If unknown assume 2.1 persons/unit[248]

#Units = Number of units served by hot water boiler

[247] The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).

[248] Navigant, ComEd PY3 Multi-Family Home Energy Savings Program Evaluation Report Final, May 16, 2012.

= Actual

GPD = Gallons of hot water used per person per day

= Actual. If unknown assume 17.6 gallons per person per day[249]

Days/yr = 365.25

ɣWater = Specific Weight of Water

= 8.33 gal/lb

Tout = tank temperature of hot water

= 125°F or custom

Tin = Incoming water temperature from well or municiple system

= 54°F[250]

Eff_base = thermal efficiency of base unit

= 80%[251]

Eff_ee = thermal efficiency of efficient unit complying with this measure

= Actual. If unknown assume 88%

Eff_exist = thermal efficiency of existing unit

= Actual. If unknown assume 73%[252]

SL = Standby Loss[253]

= (Input rating / 800) + (110 * √Tank Volume)

Input rating = Name plate input capacity in Btuh

Tank Volume = Rated volume of the tank in gallons

Hours / yr = 8766 hours

[249] Email message from Maureen Hodgins, Research Manager for Water Research Foundation, to TAC/SAG, August 26, 2014
[250] US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html
[251] IECC 2012, Table C404.2, Minimum Performance of Water-Heating Equipment
[252] Based upon DCEO data provided 10/2014; average age adjusted efficiency of existing units replaced through the program. Efficiency age adjustment of 0.5% per year based upon NREL "Building America Performance Analysis Procedures for Existing Homes".
[253] Stand-by loss is provided in 2012 International Energy Conservation Code (IECC2012), Table C404.2, Minimum Performance of Water-Heating Equipment

100,000 = btu/therm

**EXAMPLES**

Time of Sale:

For example, an 88% 1000 gallon boiler with 150,000 Btuh input rating installed serving 50 units.

ΔTherms = Hot Water Savings + Standby Loss Savings

= [(MFHH * #Units * GPD * Days/yr * ɤWater * (Tout – Tin) * (1/Eff_base – 1/Eff_ee)) / 100,000] + [((SL * Hours/yr * (1/Eff_base – 1/Eff_ee)) / 100,000]

=[(2.1 * 50 * 17.6 * 8.33 * 365.25 * 1.0 * (125-54) * (1/0.8 – 1/0.88)) / 100000] + [((150000/800 + (110 * √1000)) * 8766 * (1/0.8 – 1/0.88)) / 100000]

= 454 + 37

= 490 therms

Early Replacement:

For example, an 88% 1000 gallon boiler with 150,000 Btuh input rating installed serving 50 units replaces a working unit with unknown efficiency.

ΔTherms for remaining life of existing unit (1st 5 years):

=[(2.1 * 50 * 17.6 * 8.33 * 365.25 * 1.0 * (125-54) * (1/0.73 – 1/0.88)) / 100000] + [((150000/800 + (110 * √1000)) * 8766 * (1/0.73 – 1/0.88)) / 100000]

= 932 + 75

= 1007 therms

ΔTherms for remaining measure life (next 10 years):

= 454 + 37 (as above)

= 490 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HW_-MDHW-V01-150601**

## 4.3.8 Controls for Central Domestic Hot Water

### DESCRIPTION

Demand control recirculation pumps seek to reduce inefficiency by combining control via temperature and demand inputs, whereby the controller will not activate the recirculation pump unless both (a) the recirculation loop return water has dropped below a prescribed temperature (e.g. 100°F) and (b) a CDHW demand is sensed as water flow through the CDHW system.

This measure was developed to be applicable to the following program types: TOS, RF, NC. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

Re-circulating pump shall cycle on based on (a) the recirculation loop return water dropping below a prescribed temperature (e.g. 100°F) and (b) a CDHW demand is sensed as water flow through the CDHW system.

### DEFINITION OF BASELINE EQUIPMENT

The base case for this measure category are existing, un-controlled Recirculation Pumps on gas-fired Central Domestic Hot Water Systems.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The effective useful life is 15 years[254].

### DEEMED MEASURE COST

Incremental Cost: $1,200 [255]

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

[254] Benningfield Group. (2009). *PY 2009 Monitoring Report: Demand Control for Multifamily Central Domestic Hot Water.* Folsom, CA: Prepared for Southern California Gas Company, October 30, 2009.
[255] Gas Technology Institute. (2014). *1003: Demand-based domestic hot water recirculation Public project report.* Des Plaines, IL: Prepared for Nicor Gas, January 7, 2014.

**Algorithm**

**CALCULATION OF ENERGY SAVINGS**[256]

**ELECTRIC ENERGY SAVINGS**

Deemed at 651 kWh[257].

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

N/A

**NATURAL GAS SAVINGS**

ΔTherms = 55.9[258] * number of dwelling units

**EXAMPLE**

For example, an apartment building with 53 units:

ΔTherms = 55.9 * 53

= 2,962.7 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HW_-CDHW-V01-150601**

[256] See Illinois_Statewide_TRM_Workpaper_Demand Control Central DHW for more details

[257] Based on results from the Nicor Gas Emerging Technology Program study, this value is the average kWh saved per pump. Note this value does not reflect savings from electric units but electrical savings from gas-fired units.

[258] Based on results from the Nicor Gas Emerging Technology Program study, this value is the average therms saved per dwelling unit.

## 4.4 HVAC End Use

Many of the commercial HVAC measures use equivalent full load hours (EFLH) to calculate heating and cooling savings. The tables with these values are included in this section and referenced in each measure.

To calculate the updated EFLHs by building type and climate zone provided below, a TAC Subcommittee utilized building energy models originally developed for ComEd[259], applying some adjustments and additions for new building type models and mechanical systems. Based on comparisons with available field data from Navigant, the EFLH calculation was finalized by the Subcommittee to be the annual total (heating or cooling) output (in Btu) divided by the 95th percentile hourly peak output (heating or cooling) demand (in Btu/hr). This calculation keeps EFLH independent of modeled systems efficiency (which is utilized in the TRM savings calculation) and buffers EFLH value from hourly variances in the modeling that are not representative of actual buildings. See "EFLH Description 2015-02-11.doc" for further explanation.

The building characteristics can be found in the reference table named "EFLH Building Descriptions Updated 2014-11-21.xlsx".

| Building Type | Heating EFLH | | | | |
|---|---|---|---|---|---|
| | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| Assembly | 1,787 | 1,831 | 1,635 | 1,089 | 1,669 |
| Assisted Living | 1,683 | 1,646 | 1,446 | 1,063 | 1,277 |
| College | 1,530 | 1,430 | 1,276 | 709 | 849 |
| Convenience Store | 1,481 | 1,368 | 1,214 | 871 | 973 |
| Elementary School | 1,781 | 1,736 | 1,531 | 1,057 | 1,283 |
| Garage | 985 | 969 | 852 | 680 | 752 |
| Grocery | 1,608 | 1,602 | 1,404 | 876 | 1,047 |
| Healthcare Clinic | 1,579 | 1,620 | 1,414 | 963 | 1,019 |
| High School | 1,845 | 1,857 | 1,666 | 1,187 | 1,388 |
| Hospital - CAV no econ[260] | 1,764 | 1,818 | 1,549 | 1,332 | 1,512 |
| Hospital - CAV econ[261] | 1,788 | 1,853 | 1,580 | 1,369 | 1,555 |
| Hospital - VAV econ[262] | 731 | 695 | 522 | 314 | 340 |
| Hospital - FCU | 1,325 | 1,512 | 1,232 | 1,448 | 1,946 |
| Hotel/Motel | 1,761 | 1,712 | 1,544 | 1,056 | 1,290 |

[259] A full description of the ComEd model development is found in "ComEd Portfolio Modeling Report. Energy Center of Wisconsin July 30, 2010"
[260] Based on model with single duct reheat system with a fixed outdoor air volume.
[261] Based on model with single duct reheat system with airside economizer controls, with constant volume zone reheat boxes and single speed fan motors.
[262] Based on model with single duct reheat system with airside economizer controls, zone VAV reheat boxes and VFD fan motors.

| Building Type | Heating EFLH | | | | |
|---|---|---|---|---|---|
| | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| Hotel/Motel - Common | 1,601 | 1,626 | 1,548 | 1,260 | 1,323 |
| Hotel/Motel - Guest | 1,758 | 1,702 | 1,521 | 1,018 | 1,252 |
| Manufacturing Facility | 1,048 | 1,013 | 939 | 567 | 634 |
| MF - High Rise | 1,526 | 1,506 | 1,373 | 1,169 | 1,172 |
| MF - High Rise - Common | 1,815 | 1,762 | 1,580 | 1,089 | 1,406 |
| MF - High Rise - Residential | 1,475 | 1,464 | 1,330 | 1,152 | 1,123 |
| MF - Mid Rise | 1,666 | 1,685 | 1,450 | 1,067 | 1,216 |
| Movie Theater | 1,916 | 1,905 | 1,718 | 1,288 | 1,538 |
| Office - High Rise - CAV no econ | 2,020 | 2,050 | 1,869 | 1,252 | 1,363 |
| Office - High Rise - CAV econ | 2,089 | 2,132 | 1,960 | 1,351 | 1,487 |
| Office - High Rise - VAV econ | 1,528 | 1,558 | 1,284 | 759 | 846 |
| Office - High Rise - FCU | 1,118 | 1,102 | 952 | 505 | 530 |
| Office - Low Rise | 1,428 | 1,425 | 1,132 | 692 | 793 |
| Office - Mid Rise | 1,585 | 1,587 | 1,342 | 855 | 950 |
| Religious Building | 1,603 | 1,504 | 1,440 | 1,054 | 1,205 |
| Restaurant | 1,350 | 1,354 | 1,216 | 920 | 1,091 |
| Retail - Department Store | 1,392 | 1,278 | 1,200 | 781 | 891 |
| Retail - Strip Mall | 1,332 | 1,233 | 1,090 | 751 | 810 |
| Warehouse | 1,456 | 1,357 | 1,400 | 875 | 1,078 |
| Unknown | 1,553 | 1,539 | 1,369 | 982 | 1,139 |

Equivalent Full Load Hours for Cooling ($EFLH_{cooling}$) :

| Building Type | Cooling EFLH | | | | |
|---|---|---|---|---|---|
| | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| Assembly | 725 | 796 | 937 | 1,183 | 932 |
| Assisted Living | 1,475 | 1,457 | 1,773 | 2,110 | 1,811 |
| College | 475 | 481 | 662 | 746 | 806 |
| Convenience Store | 1,088 | 1,067 | 1,368 | 1,541 | 1,371 |
| Elementary School | 725 | 764 | 905 | 1,142 | 956 |
| Garage | 934 | 974 | 1,226 | 1,582 | 1,383 |
| Grocery | 1,033 | 1,000 | 1,236 | 1,499 | 1,286 |
| Healthcare Clinic | 1,282 | 1,305 | 1,519 | 1,767 | 1,571 |
| High School | 675 | 721 | 840 | 1,060 | 920 |
| Hospital - CAV no econ | 4,166 | 4,275 | 4,319 | 4,692 | 4,445 |
| Hospital - CAV econ | 1,751 | 1,814 | 2,120 | 2,411 | 2,112 |
| Hospital - VAV econ | 1,531 | 1,592 | 1,853 | 2,163 | 1,876 |
| Hospital - FCU | 3,245 | 3,291 | 3,451 | 4,128 | 3,806 |
| Hotel/Motel | 1,233 | 1,186 | 1,436 | 1,274 | 1,616 |
| Hotel/Motel - Common | 2,186 | 2,103 | 2,344 | 1,391 | 2,651 |
| Hotel/Motel - Guest | 1,042 | 1,019 | 1,269 | 1,216 | 1,418 |
| Manufacturing Facility | 1,010 | 1,055 | 1,209 | 1,453 | 1,273 |
| MF - High Rise | 921 | 845 | 1,048 | 1,779 | 1,099 |
| MF - High Rise - Common | 914 | 839 | 1,055 | 2,893 | 1,132 |
| MF - High Rise - Residential | 899 | 831 | 1,011 | 1,569 | 1,055 |
| MF - Mid Rise | 809 | 767 | 992 | 1,119 | 993 |
| Movie Theater | 876 | 745 | 1,036 | 1,178 | 1,010 |
| Office - High Rise - CAV no econ | 1,688 | 1,708 | 1,811 | 1,865 | 1,725 |
| Office - High Rise - CAV econ | 1,454 | 1,452 | 1,551 | 1,568 | 1,416 |
| Office - High Rise - VAV econ | 875 | 919 | 1,057 | 1,275 | 1,077 |
| Office - High Rise - FCU | 1,117 | 1,170 | 1,277 | 1,642 | 1,412 |
| Office - Low Rise | 949 | 1,010 | 1,182 | 1,452 | 1,281 |
| Office - Mid Rise | 883 | 938 | 1,072 | 1,286 | 1,083 |
| Religious Building | 861 | 817 | 967 | 1,159 | 1,067 |
| Restaurant | 1,074 | 1,134 | 1,279 | 1,627 | 1,325 |

| Building Type | Cooling EFLH | | | | |
|---|---|---|---|---|---|
| | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| Retail - Department Store | 949 | 889 | 1,124 | 1,367 | 1,157 |
| Retail - Strip Mall | 950 | 919 | 1,149 | 1,351 | 1,215 |
| Warehouse | 357 | 338 | 422 | 647 | 533 |
| Unknown | 1,215 | 1,221 | 1,408 | 1,670 | 1,480 |

## 4.4.1 Air Conditioner Tune-up

### DESCRIPTION

An air conditioning system that is operating as designed saves energy and provides adequate cooling and comfort to the conditioned space

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a unitary or split system air conditioner least 3 tons and preapproved by program. The measure requires that a certified technician performs the following items:

- Check refrigerant charge
- Identify and repair leaks if refrigerant charge is low
- Measure and record refrigerant pressures
- Measure and record temperature drop at indoor coil
- Clean condensate drain line
- Clean outdoor coil and straighten fins
- Clean indoor and outdoor fan blades
- Clean indoor coil with spray-on cleaner and straighten fins
- Repair damaged insulation – suction line
- Change air filter
- Measure and record blower amp draw

A copy of contractor invoices that detail the work performed to identify tune-up items, as well as additional labor and parts to improve/repair air conditioner performance must be submitted to the program

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline condition is assumed to be an AC system that that does not have a standing maintenance contract or a tune up within in the past 36 months.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 3 years.[263]

### DEEMED MEASURE COST

The incremental capital cost for this measure is $35[264] per ton.

### LOADSHAPE

Loadshape C03 - Commercial Cooling

### COINCIDENCE FACTOR

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)

= 91.3% [265]

[263] Ibid.
[264] Ibid.

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)

= 47.8% [266]

**Algorithm**

**CALCULATION OF SAVINGS**

**ELECTRIC ENERGY SAVINGS**

For units with cooling capacities less than 65 kBtu/hr:

ΔkWH = (kBtu/hr) * [(1/SEERbefore) – (1/SEERafter)] * EFLH

For units with cooling capacities equal to or greater than 65 kBtu/hr:

ΔkWH = (kBtu/hr) * [(1/EERbefore) – (1/EERafter)] * EFLH

Where:

kBtu/hr = capacity of the cooling equipment actually installed in kBtu per hour (1 ton of cooling capacity equals 12 kBtu/hr).

=Actual

SEERbefore = Seasonal Energy Efficiency Ratio of the equipment prior to tune-up

=Actual

SEERafter = Seasonal Energy Efficiency Ratio of the equipment after to tune-up

=Actual

EERbefore = Energy Efficiency Ratio of the baseline equipment prior to tune-up

=Actual

EERafter = Energy Efficiency Ratio of the baseline equipment after to tune-up

=Actual

EFLH = Equivalent Full Load Hours for cooling are provided in section 4.4 HVAC End Use

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$\Delta kW_{SSP}$ = (kBtu/hr * (1/EERbefore - 1/EERafter)) * $CF_{SSP}$

$\Delta kW_{PJM}$ = (kBtu/hr * (1/ EERbefore - 1/EERafter)) * $CF_{PJM}$

Where:

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)

= 91.3% [267]

[265] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

[266] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

[267] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)

= 47.8%[268]

**NATURAL GAS ENERGY SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HVC-ACTU-V02-150601**

[268]Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

## 4.4.2 Space Heating Boiler Tune-up

### DESCRIPTION

This measure is for a non-residential boiler that provides space heating. The tune-up will improve boiler efficiency by cleaning and/or inspecting burners, combustion chamber, and burner nozzles. Adjust air flow and reduce excessive stack temperatures, adjust burner and gas input. Check venting, safety controls, and adequacy of combustion air intake. Combustion efficiency should be measured before and after tune-up using an electronic flue gas analyzer.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the facility must, as applicable, complete the tune-up requirements[269] listed below, by approved technician:

- Measure combustion efficiency using an electronic flue gas analyzer
- Adjust airflow and reduce excessive stack temperatures
- Adjust burner and gas input, manual or motorized draft control
- Check for proper venting
- Complete visual inspection of system piping and insulation
- Check safety controls
- Check adequacy of combustion air intake
- Clean fireside surfaces.
- Inspect all refractory. Patch and wash coat as required.
- Inspect gaskets on front and rear doors and replace as necessary.
- Seal and close front and rear doors properly.
- Clean low and auxiliary low water cut-off controls, then re-install using new gaskets.
- Clean plugs in control piping.
- Remove all hand hole and man hole plates. Flush boiler with water to remove loose scale and sediment.
- Replace all hand hole and man hole plates with new gaskets.
- Open feedwater tank manway, inspect and clean as required. Replace manway plate with new gasket.
- Clean burner and burner pilot.
- Check pilot electrode and adjust or replace.
- Clean air damper and blower assembly.
- Clean motor starter contacts and check operation.
- Make necessary adjustments to burner for proper combustion.
- Perform all flame safeguard and safety trip checks.
- Check all hand hole plates and man hole plates for leaks at normal operating temperatures and pressures.
- Troubleshoot any boiler system problems as requested by on-site personnel

### DEFINITION OF BASELINE EQUIPMENT

The baseline condition of this measure is a boiler that has not had a tune-up within the past 36 months

[269] Act on Energy Commercial Technical Reference Manual No. 2010-4, 9.2.2 Gas Boiler Tune-up

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The life of this measure is 3 years[270]

### DEEMED MEASURE COST

The cost of this measure is $0.83/MBtu/hr[271] per tune-up

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

**Algorithm**

### CALCULATION OF ENERGY SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS ENERGY SAVINGS

$$\Delta therms = (Capacity * EFLH * (((Effbefore + Ei) / Effbefore) - 1)) / 100{,}000$$

Where:

Capacity = Boiler gas input size (Btu/hr)

= custom

EFLH = Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use

Effbefore = Efficiency of the boiler before the tune-up

Ei = Efficiency Improvement of the boiler tune-up measure

[270] Act on Energy Commercial Technical Reference Manual No. 2010-4, 9.2.2 Gas Boiler Tune-up
[271] Work Paper – Tune up for Boilers serving Space Heating and Process Load by Resource Solutions Group, January 2012

100,000 = Converts Btu to therms

**EXAMPLE**

For example, a 1050 kBtu boiler in a Chicago high rise office records an efficiency prior to tune up of 82% AFUE and a 1.8% improvement in efficiency are tune up:

$$\Delta therms = (1{,}050{,}000 * 2050 * ((0.82 + 0.018)/ 0.82 - 1)) /100{,}000$$

$$= 473 \text{ Therms}$$

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HVC-BLRT-V05-150601**

## 4.4.3 Process Boiler Tune-up

### DESCRIPTION

This measure is for a non-residential boiler for process loads. For space heating, see measure 4.4.2. The tune-up will improve boiler efficiency by cleaning and/or inspecting burners, combustion chamber, and burner nozzles. Adjust air flow and reduce excessive stack temperatures, adjust burner and gas input. Check venting, safety controls, and adequacy of combustion air intake. Combustion efficiency should be measured before and after tune-up using an electronic flue gas analyzer.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the facility must, as applicable, complete the tune-up requirements[272] by approved technician, as specified below:

- Measure combustion efficiency using an electronic flue gas analyzer
- Adjust airflow and reduce excessive stack temperatures
- Adjust burner and gas input, manual or motorized draft control
- Check for proper venting
- Complete visual inspection of system piping and insulation
- Check safety controls
- Check adequacy of combustion air intake
- Clean fireside surfaces
- Inspect all refractory. Patch and wash coat as required.
- Inspect gaskets on front and rear doors and replace as necessary.
- Seal and close front and rear doors properly.
- Clean low and auxiliary low water cut-off controls, then re-install using new gaskets.
- Clean plugs in control piping.
- Remove all hand hole and man hole plates. Flush boiler with water to remove loose scale and sediment.
- Replace all hand hole and man hole plates with new gaskets.
- Open feedwater tank manway, inspect and clean as required. Replace manway plate with new gasket.
- Clean burner and burner pilot.
- Check pilot electrode and adjust or replace.
- Clean air damper and blower assembly.
- Clean motor starter contacts and check operation.
- Make necessary adjustments to burner for proper combustion.
- Perform all flame safeguard and safety trip checks.
- Check all hand hole plates and man hole plates for leaks at normal operating temperatures and pressures.
- Troubleshoot any boiler system problems as reQuested by on-site personnel

### DEFINITION OF BASELINE EQUIPMENT

The baseline condition of this measure is a boiler that has not had a tune-up within the past 36 months

[272] Act on Energy Commercial Technical Reference Manual No. 2010-4, 9.2.2 Gas Boiler Tune-up

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The life of this measure is 3 years[273]

### DEEMED MEASURE COST

The cost of this measure is $0.83/MBtu/hr[274] per tune-up

### DEEMED O&M COST ADJUSTMENTS

N/A

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

## Algorithm

### CALCULATION OF ENERGY SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS ENERGY SAVINGS

$$\Delta therms=((Ngi * 8766*UF)/100) * (1- (Eff_{pre}/Eff_{measured}))$$

Where:

Ngi = Boiler gas input size (kBtu/hr)

= custom

UF = Utilization Factor

= 41.9%[275] or custom

[273] Act on Energy Commercial Technical Reference Manual No. 2010-4, 9.2.2 Gas Boiler Tune-up
[274] Work Paper – Tune up for Boilers serving Space Heating and Process Load by Resource Solutions Group, January 2012

$Eff_{pre}$ = Boiler Combustion Efficiency Before Tune-Up

= Actual

$Eff_{measured}$ = Boiler Combustion Efficiency After Tune-Up

= Actual

100 =converstion from kBtu to therms

8766 = hours a year

**EXAMPLE**

For example, a 80% 1050 kBtu boiler is tuned-up resulting in final efficiency of 81.3%:

Δtherms =((1050 * 8766*0.419)/100) * (1- (0.80/0.813))

= 617 therms

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HVC-PBTU-V04-150601**

[275] Work Paper – Tune up for Boilers serving Space Heating and Process Load by Resource Solutions Group, January 2012

## 4.4.4 Boiler Lockout/Reset Controls

### DESCRIPTION

This measure relates to improving combustion efficiency by adding controls to non-residential building heating boilers to vary the boiler entering water temperature relative to heating load as a function of the outdoor air temperature to save energy. Energy is saved by increasing the temperature difference between the water temperature entering the boiler in the boiler's heat exchanger and the boiler's burner flame temperature. The flame temperature remains the same while the water temperature leaving the boiler decreases with the decrease in heating load due to an increase in outside air temperature. A lockout temperature is also set to prevent the boiler from turning on when it is above a certain temperature outdoors.

This measure was developed to be applicable to the following program types: RF. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

Natural gas customer adding boiler reset controls capable of resetting the boiler supply water temperature in an inverse linear fashion with outdoor air temperature. Boiler lockout temperatures should be set to 55 °F at this time as well, to turn the boiler off when the temperature goes above a certain setpoint.

### DEFINITION OF BASELINE EQUIPMENT

Existing boiler without boiler reset controls, any size with constant hot water flow.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The life of this measure is 20 years[276]

### DEEMED MEASURE COST

The cost of this measure is $612[277]

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

[276]CLEAResultreferences the Brooklyn Union Gas Company, High Efficiency Heating and Water and Controls, Gas Energy Efficiency Program Implementation Plan.
[277] Nexant. Questar DSM Market Characterization Report. August 9, 2006.

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS ENERGY SAVINGS

Therm Savings = Binput * SF * EFLH /( 100)

Where:

Binput = Boiler Input Capacity (kBtu/hr)

= custom

SF = Savings factor

= 8%[278] or custom

EFLH = Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use

100 = conversion from kBtu to therms

**EXAMPLE**

For example, a 800 kBtu/hr boiler at a restaurant in Rockford, IL

ΔTherms = 800 * 0.08 * 1,350 / (100)

= 864 Therms

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-HVC-BLRC-V03-150601

[278] Savings factor is the estimate of annual gas consumption that is saved due to adding boiler reset controls. The CLEAResultuses a boiler tuneup savings value derived from Xcel Energy "DSM Biennial Plan-Technical Assumptions," Colorado. Focus on Energy uses 8%, citing multiple sources. Vermont Energy Investment Corporation's boiler reset savings estimates for custom projects further indicate 8% savings estimate is better reflection of actual expected savings.

## 4.4.5 Condensing Unit Heaters

### DESCRIPTION

This measure applies to a gas fired condensing unit heater installed in a commercial application.

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a condensing unit heater up to 300 MBH with a Thermal Efficiency > 90% and the heater must be vented, and condensate drained per manufacturer specifications. The unit must be replacing existing natural gas equipment.

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline condition is assumed to be a non-condensing natural gas unit heater at end of life.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years[279]

### DEEMED MEASURE COST

The incremental capital cost for a unit heater is $676[280]

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

[279] DEER 2008

[280] ENERGY STAR and CEE do not currently provide calculators for this type of equipment therefore deemed values from Nicor Gas were used. Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011

### NATURAL GAS ENERGY SAVINGS

The annual natural gas energy savings from this measure is a deemed value equaling 266 Therms.

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-HVC-CUHT-V01-120601

## 4.4.6 Electric Chiller

### DESCRIPTION

This measure relates to the installation of a new electric chiller meeting the efficiency standards presented below. This measure could relate to the replacement of an existing unit at the end of its useful life, or the installation of a new system in an existing building (i.e. time of sale). Only single-chiller applications should be assessed with this methodology. The characterization is not suited for multiple chillers projects or chillers equipped with variable speed drives (VSDs).

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to exceed the efficiency requirements of the 2009 International Energy Conservation Code, Table 503.2.3(7)

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to meet the efficiency requirements of the2009 International Energy Conservation Code, Table 503.2.3(7).

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 20 years [281].

### DEEMED MEASURE COST

The incremental capital cost for this measure is provided below.

| Equipment Type | Size Category | Incremental Cost ($/ton)[282] |
|---|---|---|
| Air cooled, electrically operated | All capacities | $127/ton[283] |
| Water cooled, electrically operated, positive displacement (reciprocating) | All capacities | $22/ton |
| Water cooled, electrically operated, positive displacement (rotary screw and scroll) | < 150 tons | $128/ton |
| | >= 150 tons and < 300 tons | $70/ton |
| | >= 300 tons | $48/ton |

[281] 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, "Effective/Remaining Useful Life Values", California Public Utilities Commission, December 16, 2008 (http://deeresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xls)

[282] 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, "Cost Values and Summary Documentation", California Public Utilities Commission, December 16, 2008 (http://deeresources.com/deer0911planning/downloads/DEER2008_Costs_ValuesAndDocumentation_080530Rev1.zip)

[283] Calculated as the simple average of screw and reciprocating air-cooled chiller incremental costs from DEER2008. This assumes that baseline shift from IECC 2009 to IECC 2012 carries the same incremental costs. Values should be verified during evaluation

### LOADSHAPE

Loadshape C03 - Commercial Cooling

### COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market. Both values provided are based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)

= 91.3% [284]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)

= 47.8% [285]

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

ΔkWH = TONS * ((IPLVbase) – (IPLVee)) * EFLH

Where:

TONS = chiller nominal cooling capacity in tons (note: 1 ton = 12,000 Btu/hr)

= Actual installed

IPLVbase = efficiency of baseline equipment expressed as Integrated Part Load Value(kW/ton). Chiller units are dependent on chiller type. See Chiller Units, Convertion Values and Baseline Efficiency Values by Chiller Type and Capacity in the Reference Tables section.

IPLVee[286] = efficiency of high efficiency equipment expressed as Integrated Part Load Value (kW/ton)[287]

= Actual installed

EFLH = Equivalent Full Load Hours for cooling are provided in section 4.4 HVAC End Use.

[284] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

[285] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

[286] Integrated Part Load Value is a seasonal average efficiency rating calculated in accordance with ARI Standard 550/590. It may be calculated using any measure of efficiency (EER, kW/ton, COP), but for consistency with IECC 2012, it is expressed in terms of IPLV here.

[287] Can determine IPLV from standard testing or looking at engineering specs for design conditions. Standard data is available from AHRnetI.org. http://www.ahrinet.org/

For example, a 100 ton air-cooled electrically operated chiller in a high-rise office building with IPLV of 14 EER (0.86 kW/ton) and baseline EER of 12.5 (0.96 kW/ton) in Rockford would save:

ΔkWH = 100 * ((0.96) – (0.86)) * 923

= 9,230 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$\Delta kW_{SSP}$ = TONS * ((PEbase) – (PEee)) * $CF_{SSP}$

$\Delta kW_{PJM}$ = TONS * ((PEbase) – (PEee)) * $CF_{PJM}$

Where:

PEbase = Peak efficiency of baseline equipment expressed as Full Load (kW/ton)

PEee = Peak efficiency of high efficiency equipment expressed as Full Load (kW/ton)

= Actual installed

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)

= 91.3%

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)

= 47.8%

For example, a 100 ton air-cooled electrically operated chiller in a high-rise office building with a full load IPLV of 12 EER ( 0.86 kW/ton) with baseline full load IPLV 9.56 EER (1.3 kW/ton) in Rockford would save:

$\Delta kW_{SSP}$ = 100 * ((1.3) – (1.0)) * .913

=23 kW

**NATURAL GAS ENERGY SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**REFERENCE TABLES**

Chillers Ratings- Chillers are rated with different units depending on equipment type as shown below

| Equipment Type | Unit |
|---|---|
| Air cooled, electrically operated | EER |
| Water cooled, electrically operated, positive displacement (reciprocating) | kW/ton |

| Equipment Type | Unit |
|---|---|
| Water cooled, electrically operated, positive displacement (rotary screw and scroll) | kW/ton |

In order to convert chiller equipment ratings to IPLV the following relationships are provided

kW/ton = 12 / EER

kW/ton = 12 / (COP x 3.412)

COP = EER / 3.412

COP = 12 / (kW/ton) / 3.412

EER = 12 / kW/ton

EER = COP x 3.412

Baseline Efficiency Values by Chiller Type and Capacity[288]

**TABLE C403.2.3(7)**
**MINIMUM EFFICIENCY REQUIREMENTS:**
**WATER CHILLING PACKAGES[a]**

| EQUIPMENT TYPE | SIZE CATEGORY | UNITS | BEFORE 1/1/2010 | | AS OF 1/1/2010[b] | | | | TEST PROCEDURE[c] |
|---|---|---|---|---|---|---|---|---|---|
| | | | | | PATH A | | PATH B | | |
| | | | FULL LOAD | IPLV | FULL LOAD | IPLV | FULL LOAD | IPLV | |
| Air-cooled chillers | < 150 tons | EER | ≥ 9.562 | ≥ 10.416 | ≥ 9.562 | ≥ 12.500 | NA | NA | AHRI 550/590 |
| | ≥ 150 tons | EER | | | ≥ 9.562 | ≥ 12.750 | NA | NA | |
| Air cooled without condenser, electrical operated | All capacities | EER | ≥ 10.586 | ≥ 11.782 | Air-cooled chillers without condensers shall be rated with matching condensers and comply with the air-cooled chiller efficiency requirements | | | | |
| Water cooled, electrically operated, reciprocating | All capacities | kW/ton | ≤ 0.837 | ≤ 0.696 | Reciprocating units shall comply with water cooled positive displacement efficiency requirements | | | | |
| Water cooled, electrically operated, positive displacement | < 75 tons | kW/ton | ≤ 0.790 | ≤ 0.676 | ≤ 0.780 | ≤ 0.630 | ≤ 0.800 | ≤ 0.600 | |
| | ≥ 75 tons and < 150 tons | kW/ton | | | ≤ 0.775 | ≤ 0.615 | ≤ 0.790 | ≤ 0.586 | |
| | ≥ 150 tons and < 300 tons | kW/ton | ≤ 0.717 | ≤ 0.627 | ≤ 0.680 | ≤ 0.580 | ≤ 0.718 | ≤ 0.540 | |
| | ≥ 300 tons | kW/ton | ≤ 0.639 | ≤ 0.571 | ≤ 0.620 | ≤ 0.540 | ≤ 0.639 | ≤ 0.490 | |
| Water cooled, electrically operated, centrifugal | < 150 tons | kW/ton | ≤ 0.703 | ≤ 0.669 | ≤ 0.634 | ≤ 0.596 | ≤ 0.639 | ≤ 0.450 | |
| | ≥ 150 tons and < 300 tons | kW/ton | ≤ 0.634 | ≤ 0.596 | | | | | |
| | ≥ 300 tons and < 600 tons | kW/ton | ≤ 0.576 | ≤ 0.549 | ≤ 0.576 | ≤ 0.549 | ≤ 0.600 | ≤ 0.400 | |
| | ≥ 600 tons | kW/ton | ≤ 0.576 | ≤ 0.549 | ≤ 0.570 | ≤ 0.539 | ≤ 0.590 | ≤ 0.400 | |
| Air cooled, absorption single effect | All capacities | COP | ≥ 0.600 | NR | ≥ 0.600 | NR | NA | NA | AHRI 560 |
| Water cooled, absorption single effect | All capacities | COP | ≥ 0.700 | NR | ≥ 0.700 | NR | NA | NA | |
| Absorption double effect, indirect fired | All capacities | COP | ≥ 1.000 | ≥ 1.050 | ≥ 1.000 | ≥ 1.050 | NA | NA | |
| Absorption double effect, direct fired | All capacities | COP | ≥ 1.000 | ≥ 1.000 | ≥ 1.000 | ≥ 1.000 | NA | NA | |

For SI: 1 ton = 3517 W, 1 British thermal unit per hour = 0.2931 W, °C = [(°F) - 32]/1.8.
NA = Not applicable, not to be used for compliance; NR = No requirement.

a. The centrifugal chiller equipment requirements, after adjustment in accordance with Section C403.2.3.1 or Section C403.2.3.2, do not apply to chillers used in low-temperature applications where the design leaving fluid temperature is less than 36°F. The requirements do not apply to positive displacement chillers with leaving fluid temperatures less than or equal to 32°F. The requirements do not apply to absorption chillers with design leaving fluid temperatures less than 40°F.

b. Compliance with this standard can be obtained by meeting the minimum requirements of Path A or B. However, both the full load and IPLV shall be met to fulfill the requirements of Path A or B.

c. Chapter 6 of the referenced standard contains a complete specification of the referenced test procedure, including the referenced year version of the test procedure.

**MEASURE CODE: CI-HVC-CHIL-V03-150601**

[288] International Energy Conservation Code (IECC)2012

## 4.4.7 ENERGY STAR and CEE Tier 1 Room Air Conditioner

### DESCRIPTION

This measure relates to the purchase and installation of a room air conditioning unit that meets either the ENERGY STAR or CEE TIER 1 minimum qualifying efficiency specifications, in place of a baseline unit meeting minimum Federal Standard efficiency ratings presented below:[289]

| Product Class (Btu/H) | Federal Standard EER, with louvered sides | Federal Standard EER, without louvered sides | ENERGY STAR EER, with louvered sides | ENERGY STAR EER, without louvered sides | CEE TIER 1 EER |
|---|---|---|---|---|---|
| < 8,000 | 9.7 | 9 | 10.7 | 9.9 | 11.2 |
| 8,000 to 13,999 | 9.8 | 8.5 | 10.8 | 9.4 | 11.3 |
| 14,000 to 19,999 | 9.7 | 8.5 | 10.7 | 9.4 | 11.2 |
| >= 20,000 | 8.5 | 8.5 | 9.4 | 9.4 | 9.8 |

| Casement | Federal Standard (EER) | ENERGY STAR (EER) |
|---|---|---|
| Casement-only | 8.7 | 9.6 |
| Casement-slider | 9.5 | 10.5 |

| Reverse Cycle - Product Class (Btu/H) | Federal Standard EER, with louvered sides | Federal Standard EER, without louvered sides | ENERGY STAR EER, with louvered sides | ENERGY STAR EER, without louvered sides |
|---|---|---|---|---|
| < 14,000 | N/A | 8.5 | N/A | 9.4 |
| >= 14,000 | N/A | 8 | N/A | 8.8 |
| < 20,000 | 9 | N/A | 9.9 | N/A |
| >= 20,000 | 8.5 | N/A | 9.4 | N/A |

[289] http://www.energystar.gov/index.cfm?c=roomac.pr_crit_room_ac and http://www.cee1.org/resid/seha/rm-ac/rm-ac_specs.pdf
Side louvers that extend from a room air conditioner model in order to position the unit in a window. A model without louvered sides is placed in a built-in wall sleeve and are commonly referred to as "through-the-wall" or "built-in" models.
Casement-only refers to a room air conditioner designed for mounting in a casement window of a specific size.
Casement-slider refers to a room air conditioner with an encased assembly designed for mounting in a sliding or casement window of a specific size.
Reverse cycle refers to the heating function found in certain room air conditioner models.
http://www.energystar.gov/ia/partners/product_specs/program_reqs/room_air_conditioners_prog_req.pdf

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

**DEFINITION OF EFFICIENT EQUIPMENT**

To qualify for this measure the new room air conditioning unit must meet the ENERGY STAR efficiency standards presented above.

**DEFINITION OF BASELINE EQUIPMENT**

The baseline assumption is a new room air conditioning unit that meets the current minimum federal efficiency standards presented above.

**DEEMED LIFETIME OF EFFICIENT EQUIPMENT**

The measure life is assumed to be 9 years.[290]

**DEEMED MEASURE COST**

The incremental cost for this measure is assumed to be $40 for an ENERGY STAR unit and $80 for a CEE TIER 1 unit.[291]

**LOADSHAPE**

Loadshape C03 - Commercial Cooling

**COINCIDENCE FACTOR**

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market. Both values provided are based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)

= 91.3% [292]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)

= 47.8% [293]

[290] Energy Star Room Air Conditioner Savings Calculator,
http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=AC
http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf

[291] Based on field study conducted by Efficiency Vermont

[292] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

[293] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

**Algorithm**

## CALCULATION OF SAVINGS

### ENERGY SAVINGS

$\Delta kWh = (FLH_{RoomAC} * Btu/H * (1/EERbase - 1/EERee))/1000$

Where:

$FLH_{RoomAC}$ = Full Load Hours of room air conditioning unit

= dependent on location:[294]

| Zone | $FLH_{RoomAC}$ |
|---|---|
| 1 (Rockford) | 253 |
| 2-(Chicago) | 254 |
| 3 (Springfield) | 310 |
| 4-(Belleville) | 391 |
| 5-(Marion) | 254 |

Btu/H = Size of unit

= Actual. If unknown assume 8500 Btu/hr [295]

EERbase = Efficiency of baseline unit

= As provided in tables above

EERee = Efficiency of ENERGY STAR or CEE Tier 1 unit

= Actual. If unknown assume minimum qualifying standard as provided in tables above

For example for an 8,500 Btu/H capacity ENERGY STAR unit, with louvered sides, in Rockford:

$\Delta kWH_{ENERGY\ STAR} = (253 * 8500 * (1/9.8 – 1/10.8)) / 1000$

= 20.3 kWh

[294] Full load hours for room AC is significantly lower than for central AC. The average ratio of FLH for Room AC (provided in RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008: http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RAC.pdf) to FLH for Central Cooling for the same location (provided by AHRI: http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls) is 31%. This ratio has been applied to the FLH from the unitary and split system air conditioning measure.

[295] Based on maximum capacity average from the RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = Btu/H * ((1/EERbase - 1/EERee))/1000) * CF

Where:

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)

= 91.3% [296]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)

= 47.8%[297]

Other variable as defined above

For example for an 8,500 Btu/H capacity ENERGY STAR unit, with louvered sides, in Rockford during system peak

$\Delta kW_{ENERGY\ STAR}$ = (8500 * (1/9.8 – 1/10.8)) / 1000 * 0.913

= 0.073 kW

**FOSSIL FUEL SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HVC-ESRA-V01-120601**

[296] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

[297] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

## 4.4.8 Guest Room Energy Management (PTAC & PTHP)

### DESCRIPTION

This measure applied to the installation of a temperature setback and lighting control system for individual guest rooms. The savings are achieved based on Guest Room Energy Management's (GREM's) ability to automatically adjust the guest room's set temperatures and control the HVAC unit for various occupancy modes.

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

Guest room temperature set point must be controlled by automatic occupancy detectors or keycard that indicates the occupancy status of the room. During unoccupied periods the default setting for controlled units differs by at least 5 degrees from the operating set point. Theoretically, the control system may also be tied into other electric loads, such as lighting and plug loads to shut them off when occupancy is not sensed. This measure bases savings on improved HVAC controls. If system is connected to lighting and plug loads, additional savings would be realized. The incentive is per guestroom controlled, rather than per sensor, for multi-room suites. Replacement or upgrades of existing occupancy-based controls are not eligible for an incentive.

### DEFINITION OF BASELINE EQUIPMENT

Guest room energy management thermostats replace manual heating/cooling temperature set-point and fan On/Off/Auto thermostat controls. Two possible baselines exist based on whether housekeeping staff are directed to set-back (or turn off) thermostats when rooms are not rented.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for GREM is 15 years[298].

### DEEMED MEASURE COST

$260/unit

The IMC documented for this measure is $260 per room HVAC controller, which is the cost difference between a non-programmable thermostat and a GREM[299].

### DEEMED O&M COST ADJUSTMENTS

N/A

### LOADSHAPE

Loadshape C03 - Commercial Cooling

### COINCIDENCE FACTOR

A coincidence factor is not used in the determination of coincident peak kW savings.

[298] DEER 2008 value for energy management systems
[299] This value was extracted from Smart Ideas projects in PY1 and PY2.

**Algorithm**

### CALCULATION OF SAVINGS

Below are the annual kWh savings per installed EMS for different sizes and types of HVAC units. The savings are achieved based on GREM's ability to automatically adjust the guest room's set temperatures and control the HVAC unit to maintain set temperatures for various occupancy modes. Note that care should be taken in selecting a value consistent with actual baseline conditions (e.g. whether housekeeping staff are directed to set-back/turn-off the thermostats when rooms are unrented). Different values are provided for Motels and Hotels since significant differences in shell performance, number of external walls per room and typical heating and cooling efficiencies result in significantly different savings estimates. Energy savings estimates are derived using a prototypical EnergyPlus simulation of a motel and a hotel[300]. Model outputs are normalized to the installed capacity and reported here as kWh/Ton, coincident peak kW/Ton and Therms/Ton.

### ELECTRIC ENERGY SAVINGS

| Motel Electric Energy Savings | | | |
|---|---|---|---|
| Climate Zone (City based upon) | Heating Source | Baseline | Electric Savings (kWh/Ton) |
| 1 (Rockford) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 744 |
| | | No Housekeeping Setback | 1,786 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 63 |
| | | No Housekeeping Setback | 155 |
| | PTHP | Housekeeping Setback | 385 |
| | | No Housekeeping Setback | 986 |
| 2 (Chicago) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 506 |
| | | No Housekeeping Setback | 1,582 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 51 |
| | | No Housekeeping Setback | 163 |
| | PTHP | Housekeeping Setback | 211 |
| | | No Housekeeping Setback | 798 |
| 3 (Springfield) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 462 |
| | | No Housekeeping Setback | 1,382 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 65 |
| | | No Housekeeping Setback | 198 |

[300] For motels, see S. Keates, ADM Associates Workpaper: "Suggested Revisions to Guest Room Energy Management (PTAC & PTHP)", 11/14/2013 and spreadsheet summarizing the results: 'GREM Savings Summary_IL TRM_1_22_14.xlsx'. In 2014 the hotel models were also run to compile results, rather than by applying adjustment factors to the motel results as had been done in V3.0 of the TRM. The updated values can be found in 'GREM Savings Summary (Hotel)_IL TRM_10_16_14.xls'.

| Motel Electric Energy Savings | | | |
|---|---|---|---|
| Climate Zone (City based upon) | Heating Source | Baseline | Electric Savings (kWh/Ton) |
| | PTHP | Housekeeping Setback | 202 |
| | | No Housekeeping Setback | 736 |
| 4 (Belleville) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 559 |
| | | No Housekeeping Setback | 1,877 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 85 |
| | | No Housekeeping Setback | 287 |
| | PTHP | Housekeeping Setback | 260 |
| | | No Housekeeping Setback | 1,023 |
| 5 (Marion-Williamson) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 388 |
| | | No Housekeeping Setback | 1,339 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 81 |
| | | No Housekeeping Setback | 274 |
| | PTHP | Housekeeping Setback | 174 |
| | | No Housekeeping Setback | 682 |

| Hotel Electric Energy Savings | | | |
|---|---|---|---|
| Climate Zone (City based upon) | Heating Source | Baseline | Electric Savings (kWh/Ton) |
| 1 (Rockford) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 204 |
| | | No Housekeeping Setback | 345 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 121 |
| | | No Housekeeping Setback | 197 |
| | PTHP | Housekeeping Setback | 152 |
| | | No Housekeeping Setback | 253 |
| | Central Hot Water Fan Coil w/ Electric Resistance Heating | Housekeeping Setback | 177 |
| | | No Housekeeping Setback | 296 |
| | Central Hot Water Fan Coil w/ Gas Heating | Housekeeping Setback | 94 |
| | | No Housekeeping Setback | 148 |
| 2 (Chicago) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 188 |

| Hotel Electric Energy Savings | | | |
|---|---|---|---|
| **Climate Zone (City based upon)** | **Heating Source** | **Baseline** | **Electric Savings (kWh/Ton)** |
| | | No Housekeeping Setback | 342 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 119 |
| | | No Housekeeping Setback | 195 |
| | PTHP | Housekeeping Setback | 145 |
| | | No Housekeeping Setback | 250 |
| | Central Hot Water Fan Coil w/ Electric Resistance Heating | Housekeeping Setback | 161 |
| | | No Housekeeping Setback | 294 |
| | Central Hot Water Fan Coil w/ Gas Heating | Housekeeping Setback | 92 |
| | | No Housekeeping Setback | 147 |
| 3 (Springfield) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 182 |
| | | No Housekeeping Setback | 291 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 123 |
| | | No Housekeeping Setback | 197 |
| | PTHP | Housekeeping Setback | 145 |
| | | No Housekeeping Setback | 233 |
| | Central Hot Water Fan Coil w/ Electric Resistance Heating | Housekeeping Setback | 153 |
| | | No Housekeeping Setback | 240 |
| | Central Hot Water Fan Coil w/ Gas Heating | Housekeeping Setback | 94 |
| | | No Housekeeping Setback | 146 |
| 4 (Belleville) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 182 |
| | | No Housekeeping Setback | 308 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 125 |
| | | No Housekeeping Setback | 199 |
| | PTHP | Housekeeping Setback | 146 |
| | | No Housekeeping Setback | 240 |
| | Central Hot Water Fan Coil w/ Electric Resistance Heating | Housekeeping Setback | 152 |
| | | No Housekeeping Setback | 255 |
| | Central Hot Water Fan Coil w/ Gas Heating | Housekeeping Setback | 95 |
| | | No Housekeeping Setback | 147 |

| Hotel Electric Energy Savings | | | |
|---|---|---|---|
| **Climate Zone (City based upon)** | **Heating Source** | **Baseline** | **Electric Savings (kWh/Ton)** |
| 5 (Marion-Williamson) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 171 |
| | | No Housekeeping Setback | 295 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 122 |
| | | No Housekeeping Setback | 199 |
| | PTHP | Housekeeping Setback | 140 |
| | | No Housekeeping Setback | 235 |
| | Central Hot Water Fan Coil w/ Electric Resistance Heating | Housekeeping Setback | 141 |
| | | No Housekeeping Setback | 243 |
| | Central Hot Water Fan Coil w/ Gas Heating | Housekeeping Setback | 92 |
| | | No Housekeeping Setback | 146 |

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

| Motel Coincident Peak Demand Savings | | | |
|---|---|---|---|
| Climate Zone (City based upon) | Heating Source | Baseline | Coincident Peak Demand Savings (kW/Ton) |
| 1 (Rockford) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 0.08 |
| | | No Housekeeping Setback | 0.17 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 0.08 |
| | | No Housekeeping Setback | 0.17 |
| | PTHP | Housekeeping Setback | 0.08 |
| | | No Housekeeping Setback | 0.17 |
| 2 (Chicago) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 0.06 |
| | | No Housekeeping Setback | 0.17 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 0.06 |
| | | No Housekeeping Setback | 0.17 |
| | PTHP | Housekeeping Setback | 0.06 |
| | | No Housekeeping Setback | 0.17 |
| 3 (Springfield) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 0.07 |
| | | No Housekeeping Setback | 0.17 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 0.07 |
| | | No Housekeeping Setback | 0.17 |
| | PTHP | Housekeeping Setback | 0.07 |
| | | No Housekeeping Setback | 0.17 |
| 4 (Belleville) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 0.10 |
| | | No Housekeeping Setback | 0.28 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 0.10 |
| | | No Housekeeping Setback | 0.28 |
| | PTHP | Housekeeping Setback | 0.10 |
| | | No Housekeeping Setback | 0.28 |
| 5 (Marion-Williamson) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 0.08 |
| | | No Housekeeping Setback | 0.21 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 0.08 |
| | | No Housekeeping Setback | 0.21 |
| | PTHP | Housekeeping Setback | 0.08 |

| Motel Coincident Peak Demand Savings | | | |
|---|---|---|---|
| Climate Zone (City based upon) | Heating Source | Baseline | Coincident Peak Demand Savings (kW/Ton) |
| | | No Housekeeping Setback | 0.21 |

| Hotel Coincident Peak Demand Savings | | | |
|---|---|---|---|
| Climate Zone (City based upon) | Heating Source | Baseline | Coincident Peak Demand Savings (kW/Ton) |
| 1 (Rockford) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 0.08 |
| | | No Housekeeping Setback | 0.11 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 0.08 |
| | | No Housekeeping Setback | 0.11 |
| | PTHP | Housekeeping Setback | 0.08 |
| | | No Housekeeping Setback | 0.11 |
| | Central Hot Water Fan Coil w/ Electric Resistance Heating | Housekeeping Setback | 0.05 |
| | | No Housekeeping Setback | 0.08 |
| | Central Hot Water Fan Coil w/ Gas Heating | Housekeeping Setback | 0.05 |
| | | No Housekeeping Setback | 0.08 |
| 2 (Chicago) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 0.07 |
| | | No Housekeeping Setback | 0.11 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 0.07 |
| | | No Housekeeping Setback | 0.11 |
| | PTHP | Housekeeping Setback | 0.07 |
| | | No Housekeeping Setback | 0.11 |
| | Central Hot Water Fan Coil w/ Electric Resistance Heating | Housekeeping Setback | 0.05 |
| | | No Housekeeping Setback | 0.07 |
| | Central Hot Water Fan Coil w/ Gas Heating | Housekeeping Setback | 0.05 |
| | | No Housekeeping Setback | 0.07 |
| 3 (Springfield) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 0.08 |
| | | No Housekeeping Setback | 0.11 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 0.08 |

| Hotel Coincident Peak Demand Savings | | | |
|---|---|---|---|
| **Climate Zone (City based upon)** | **Heating Source** | **Baseline** | **Coincident Peak Demand Savings (kW/Ton)** |
| | | No Housekeeping Setback | 0.11 |
| | PTHP | Housekeeping Setback | 0.08 |
| | | No Housekeeping Setback | 0.11 |
| | Central Hot Water Fan Coil w/ Electric Resistance Heating | Housekeeping Setback | 0.05 |
| | | No Housekeeping Setback | 0.07 |
| | Central Hot Water Fan Coil w/ Gas Heating | Housekeeping Setback | 0.05 |
| | | No Housekeeping Setback | 0.07 |
| 4 (Belleville) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 0.08 |
| | | No Housekeeping Setback | 0.11 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 0.08 |
| | | No Housekeeping Setback | 0.11 |
| | PTHP | Housekeeping Setback | 0.08 |
| | | No Housekeeping Setback | 0.11 |
| | Central Hot Water Fan Coil w/ Electric Resistance Heating | Housekeeping Setback | 0.05 |
| | | No Housekeeping Setback | 0.08 |
| | Central Hot Water Fan Coil w/ Gas Heating | Housekeeping Setback | 0.05 |
| | | No Housekeeping Setback | 0.08 |
| 5 (Marion-Williamson) | PTAC w/ Electric Resistance Heating | Housekeeping Setback | 0.08 |
| | | No Housekeeping Setback | 0.11 |
| | PTAC w/ Gas Heating | Housekeeping Setback | 0.08 |
| | | No Housekeeping Setback | 0.11 |
| | PTHP | Housekeeping Setback | 0.08 |
| | | No Housekeeping Setback | 0.11 |
| | Central Hot Water Fan Coil w/ Electric Resistance Heating | Housekeeping Setback | 0.05 |
| | | No Housekeeping Setback | 0.08 |
| | Central Hot Water Fan Coil w/ Gas Heating | Housekeeping Setback | 0.05 |
| | | No Housekeeping Setback | 0.08 |

### NATURAL GAS ENERGY SAVINGS

For PTACs with gas heating:

| Motel Natural Gas Energy Savings | | |
|---|---|---|
| **Climate Zone (City based upon)** | **Baseline** | **Gas Savings (Therms/Ton)** |
| 1 (Rockford) | Housekeeping Setback | 30 |
| | No Housekeeping Setback | 71 |
| 2 (Chicago) | Housekeeping Setback | 20 |
| | No Housekeeping Setback | 62 |
| 3 (Springfield) | Housekeeping Setback | 17 |
| | No Housekeeping Setback | 52 |
| 4 (Belleville) | Housekeeping Setback | 21 |
| | No Housekeeping Setback | 70 |
| 5 (Marion-Williamson) | Housekeeping Setback | 13 |
| | No Housekeeping Setback | 47 |

| Hotel Natural Gas Energy Savings | | | |
|---|---|---|---|
| **Climate Zone (City based upon)** | **Heating Source** | **Baseline** | **Gas Savings (Therms/Ton)** |
| 1 (Rockford) | PTAC w/ Gas Heating | Housekeeping Setback | 3.6 |
| | | No Housekeeping Setback | 6.4 |
| | Central Hot Water Fan Coil w/ Gas Heating | Housekeeping Setback | 3.6 |
| | | No Housekeeping Setback | 6.4 |
| 2 (Chicago) | PTAC w/ Gas Heating | Housekeeping Setback | 3.0 |
| | | No Housekeeping Setback | 6.5 |
| | Central Hot Water Fan Coil w/ Gas Heating | Housekeeping Setback | 3.0 |
| | | No Housekeeping Setback | 6.5 |
| 3 (Springfield) | PTAC w/ Gas Heating | Housekeeping Setback | 2.6 |
| | | No Housekeeping Setback | 4.1 |
| | Central Hot Water Fan Coil w/ Gas Heating | Housekeeping Setback | 2.6 |
| | | No Housekeeping Setback | 4.1 |
| 4 (Belleville) | PTAC w/ Gas Heating | Housekeeping Setback | 2.5 |
| | | No Housekeeping Setback | 4.8 |

| Hotel Natural Gas Energy Savings | | | |
|---|---|---|---|
| **Climate Zone (City based upon)** | **Heating Source** | **Baseline** | **Gas Savings (Therms/Ton)** |
| | Central Hot Water Fan Coil w/ Gas Heating | Housekeeping Setback | 2.5 |
| | | No Housekeeping Setback | 4.8 |
| 5 (Marion-Williamson) | PTAC w/ Gas Heating | Housekeeping Setback | 2.1 |
| | | No Housekeeping Setback | 4.2 |
| | Central Hot Water Fan Coil w/ Gas Heating | Housekeeping Setback | 2.1 |
| | | No Housekeeping Setback | 4.2 |

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HVC-GREM-V05-150601**

## 4.4.9 Heat Pump Systems

### DESCRIPTION

This measure applies to the installation of high-efficiency air cooled, water source, ground water source, and ground source heat pump systems. This measure could apply to replacing an existing unit at the end of its useful life, or installation of a new unit in a new or existing building

This measure was developed to be applicable to the following program types: TOS NC. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a high-efficiency air cooled, water source, ground water source, or ground source heat pump system that exceeds the energy efficiency requirements of the International Energy Conservation Code (IECC) 2012,.

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to be a standard-efficiency air cooled, water source, ground water source, or ground source heat pump system that meets the energy efficiency requirements of the International Energy Conservation Code (IECC) 2012,. The rating conditions for the baseline and efficient equipment efficiencies must be equivalent

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years.[301]

### DEEMED MEASURE COST

For analysis purposes, the incremental capital cost for this measure is assumed as $100 per ton for air-cooled units.[302] The incremental cost for all other equipment types should be determined on a site-specific basis

### LOADSHAPE

Loadshape C05 - Commercial Electric Heating and Cooling

### COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market. Both values provided are based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)

= 91.3% [303]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)

[301] Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007.
[302] Based on a review of TRM incremental cost assumptions from Vermont, Wisconsin, and California.
[303] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

= 47.8% [304]

**Algorithm**

**CALCULATION OF SAVINGS**

**ELECTRIC ENERGY SAVINGS**

For units with cooling capacities less than 65 kBtu/hr:

$\Delta kWh$ = Annual kWh $Savings_{cool}$ + Annual kWh $Savings_{heat}$

Annual kWh $Savings_{cool}$ = $(kBtu/hr_{cool})$ * [(1/SEERbase) – (1/SEERee)] * $EFLH_{cool}$

Annual kWh $Savings_{heat}$ = $(kBtu/hr_{cool})$ * [(1/HSPFbase) – (1/HSPFee)] * $EFLH_{heat}$

For units with cooling capacities equal to or greater than 65 kBtu/hr:

$\Delta kWh$ = Annual kWh $Savings_{cool}$ + Annual kWh $Savings_{heat}$

Annual kWh $Savings_{cool}$ = $(kBtu/hr_{cool})$ * [(1/EERbase) – (1/EERee)] * $EFLH_{cool}$

Annual kWh $Savings_{heat}$ = $(kBtu/hr_{heat})$/3.412 * [(1/COPbase) – (1/COPee)] * $EFLH_{heat}$

Where:

$kBtu/hr_{cool}$ = capacity of the cooling equipment in kBtu per hour (1 ton of cooling capacity equals 12 kBtu/hr).

= Actual installed

SEERbase =Seasonal Energy Efficiency Ratio of the baseline equipment; see table below for values. [305]

[304] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

[305] International Energy Conservation Code (IECC) 2012

TABLE C403.2.3(2)
MINIMUM EFFICIENCY REQUIREMENTS:
ELECTRICALLY OPERATED UNITARY AND APPLIED HEAT PUMPS

| EQUIPMENT TYPE | SIZE CATEGORY | HEATING SECTION TYPE | SUBCATEGORY OR RATING CONDITION | MINIMUM EFFICIENCY | TEST PROCEDURE[a] |
|---|---|---|---|---|---|
| Air cooled (cooling mode) | < 65,000 Btu/h[b] | All | Split System | 13.0 SEER | AHRI 210/240 |
| | | | Single Packaged | 13.0 SEER | |
| Through-the-wall, air cooled | ≤ 30,000 Btu/h[b] | All | Split System | 13.0 SEER | |
| | | | Single Packaged | 13.0 SEER | |
| Single-duct high-velocity air cooled | < 65,000 Btu/h[b] | All | Split System | 10.0 SEER | |
| Air cooled (cooling mode) | ≥ 65,000 Btu/h and < 135,000 Btu/h | Electric Resistance (or None) | Split System and Single Package | 11.0 EER<br>11.2 IEER | AHRI 340/360 |
| | | All other | Split System and Single Package | 10.8 EER<br>11.0 IEER | |
| | ≥ 135,000 Btu/h and < 240,000 Btu/h | Electric Resistance (or None) | Split System and Single Package | 10.6 EER<br>10.7 IEER | |
| | | All other | Split System and Single Package | 10.4 EER<br>10.5 IEER | |
| | ≥ 240,000 Btu/h | Electric Resistance (or None) | Split System and Single Package | 9.5 EER<br>9.6 IEER | |
| | | All other | Split System and Single Package | 9.3 EER<br>9.4 IEER | |
| Water source (cooling mode) | < 17,000 Btu/h | All | 86°F entering water | 11.2 EER | ISO 13256-1 |
| | ≥ 17,000 Btu/h and < 65,000 Btu/h | All | 86°F entering water | 12.0 EER | |
| | ≥ 65,000 Btu/h and < 135,000 Btu/h | All | 86°F entering water | 12.0 EER | |
| Ground water source (cooling mode) | < 135,000 Btu/h | All | 59°F entering water | 16.2 EER | |
| | | All | 77°F entering water | 13.4 EER | |
| Water-source water to water (cooling mode) | < 135,000 Btu/h | All | 86°F entering water | 10.6 EER | ISO 13256-2 |
| | | | 59°F entering water | 16.3 EER | |
| Ground water source Brine to water (cooling mode) | < 135,000 Btu/h | All | 77°F entering fluid | 12.1 EER | |
| Air cooled (heating mode) | < 65,000 Btu/h[b] | — | Split System | 7.7 HSPF | AHRI 210/240 |
| | | — | Single Package | 7.7 HSPF | |
| Through-the-wall, (air cooled, heating mode) | ≤ 30,000 Btu/h[b] (cooling capacity) | — | Split System | 7.4 HSPF | |
| | | — | Single Package | 7.4 HSPF | |
| Small-duct high velocity (air cooled, heating mode) | < 65,000 Btu/h[b] | — | Split System | 6.8 HSPF | |

*(continued)*

TABLE C403.2.3(2)—continued
MINIMUM EFFICIENCY REQUIREMENTS:
ELECTRICALLY OPERATED UNITARY AND APPLIED HEAT PUMPS

| EQUIPMENT TYPE | SIZE CATEGORY | HEATING SECTION TYPE | SUB-CATEGORY OR RATING CONDITION | MINIMUM EFFICIENCY | TEST PROCEDURE[a] |
|---|---|---|---|---|---|
| Air cooled (heating mode) | ≥ 65,000 Btu/h and < 135,000 Btu/h (cooling capacity) | — | 47°F db/43°F wb Outdoor Air | 3.3 COP | AHRI 340/360 |
| | | | 17°F db/15°F wb Outdoor Air | 2.25 COP | |
| | ≥ 135,000 Btu/h (cooling capacity) | — | 47°F db/43°F wb Outdoor Air | 3.2 COP | |
| | | | 17°F db/15°F wb Outdoor Air | 2.05 COP | |
| Water source (heating mode) | < 135,000 Btu/h (cooling capacity) | — | 68°F entering water | 4.2 COP | ISO 13256-1 |
| Ground water source (heating mode) | < 135,000 Btu/h (cooling capacity) | — | 50°F entering water | 3.6 COP | |
| Ground source (heating mode) | < 135,000 Btu/h (cooling capacity) | — | 32°F entering fluid | 3.1 COP | |
| Water-source water to water (heating mode) | < 135,000 Btu/h (cooling capacity) | — | 68°F entering water | 3.7 COP | ISO 13256-2 |
| | | — | 50°F entering water | 3.1 COP | |
| Ground source brine to water (heating mode) | < 135,000 Btu/h (cooling capacity) | — | 32°F entering fluid | 2.5 COP | |

For SI: 1 British thermal unit per hour = 0.2931 W, °C = [(°F) - 32]/1.8.

a. Chapter 6 of the referenced standard contains a complete specification of the referenced test procedure, including the reference year version of the test procedure.

b. Single-phase, air-cooled air conditioners less than 65,000 Btu/h are regulated by NAECA. SEER values are those set by NAECA.

SEERee = Seasonal Energy Efficiency Ratio of the energy efficient equipment.

= Actual installed

$EFLH_{cool}$ = Equivalent Full Load Hours for cooling are provided in section 4.4 HVAC End Use.

HSPFbase = Heating Seasonal Performance Factor of the baseline equipment; see table above for values.

HSPFee = Heating Seasonal Performance Factor of the energy efficient equipment.

= Actual installed

$EFLH_{heat}$ = heating mode equivalent full load hours are provided in section 4.4 HVAC End Use.

EERbase = Energy Efficiency Ratio of the baseline equipment; see the table above for values. Since IECC 2012 does not provide EER requirements for air-cooled heat pumps < 65 kBtu/hr, assume the following conversion from SEER to EER: EER≈SEER/1.1.

EEREe = Energy Efficiency Ratio of the energy efficient equipment. For air-cooled air conditioners < 65 kBtu/hr, if the actual EEREe is unknown, assume the following conversion from SEER to EER: EER≈SEER/1.1.

= Actual installed

$kBtu/hr_{heat}$ = capacity of the heating equipment in kBtu per hour.

= Actual installed

3.412 = Btu per Wh.

COPbase = coefficient of performance of the baseline equipment; see table above for values.

COPee = coefficient of performance of the energy efficient equipment.

= Actual installed

Annual kWh $Savings_{cool}$ = $(kBtu/hr_{cool})$ * [(1/SEERbase) – (1/SEERee)] * $EFLH_{cool}$

Annual kWh $Savings_{heat}$ = $(kBtu/hr_{heat})$ * [(1/HSPFbase) – (1/HSPFee)] * $EFLH_{heat}$

For example a 5 ton cooling unit at a restaurant in Chicago with 60 kbtu heating with an efficient EER of 14 and an efficient HSPF of 9 saves

= [(60) * [(1/13) – (1/14)] * 1134] + [(60) * [(1/7.7) – (1/9)] * 1354]

= 1650 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = $(kBtu/hr_{cool})$ * [(1/EERbase) – (1/EERee)] *CF

Where CF value is chosen between:

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)

= 91.3% [306]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)

= 47.8% [307]

For example a 5 ton cooling unit with 60 kbtu heating with an efficient EER of 14 and an efficient HSPF of 9 saves

ΔkW = [(60) * [(1/13) – (1/14)] *.913

= 0.3

**NATURAL GAS ENERGY SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HVC-HPSY-V03-150601**

[306] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

[307] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

## 4.4.10 High Efficiency Boiler

### DESCRIPTION

To qualify for this measure the installed equipment must be replacement of an existing boiler at the end of its service life, in a commercial or multifamily space with a high efficiency, gas-fired steam or hot water boiler. High efficiency boilers achieve gas savings through the utilization of a sealed combustion chamber and multiple heat exchangers that remove a significant portion of the waste heat from flue gasses. Because multiple heat exchangers are used to remove waste heat from the escaping flue gasses, some of the flue gasses condense and must be drained.

This measure was developed to be applicable to the following program types: TOS, RF. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a boiler used 80% or more for space heating, not process, and boiler AFUE, TE (thermal efficiency), or Ec (combustion efficiency) rating must be rated greater than or equal to 85% for hot water boilers and 81% for steam boilers.

### DEFINITION OF BASELINE EQUIPMENT

Dependent on when the unit is installed and whether the unit is hot water or steam. The baseline efficiency source is the Energy Independence and Security Act of 2007 with technical amendments from Federal Register, volume 73, Number 145, Monday, July 28, 2008 for boilers <300,000 Btu/hr and is Final Rule, Federal Register, volume 74, Number 139, Wednesday, July 22, 2009 for boiler ≥300,000 Btu/hr.

Hot water boiler baseline:

| Year | Efficiency |
|---|---|
| Hot Water <300,000 Btu/hr < June 1, 2013[308] | 80% AFUE |
| Hot Water <300,000 Btu/hr ≥ June 1, 2013 | 82% AFUE |
| Hot Water ≥300,000 & ≤2,500,000 Btu/hr | 80% TE |
| Hot Water >2,500,000 Btu/hr | 82% Ec |

[308] The Federal baseline for boilers <300,000 btu/hr changes from 80% to 82% in September 2012. To prevent a change in baseline mid-program, the increase in efficiency is delayed until June 2013 when a new program year starts.

Steam boiler baseline:

| Year | Efficiency |
|---|---|
| Steam <300,000 Btu/hr < June 1, 2013[309] | 75% AFUE |
| Steam <300,000 Btu/hr ≥June 1, 2013 | 80% AFUE |
| Steam - all except natural draft ≥300,000 & ≤2,500,000 Btu/hr | 79% TE |
| Steam - natural draft ≥300,000 & ≤2,500,000 Btu/hr | 77% TE |
| Steam - all except natural draft >2,500,000 Btu/hr | 79% TE |
| Steam - natural draft >2,500,000 Btu/hr | 77% TE |

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 20 years[310]

### DEEMED MEASURE COST

The incremental capital cost for this measure depends on efficiency as listed below[311]

| Measure Tier | Incr. Cost, per unit |
|---|---|
| ENERGY STAR® Minimum | $1,470 |
| AFUE 90% | $2,400 |
| AFUE 95% | $3,370 |
| AFUE ≥ 96% | $4,340 |
| Boilers > 300,000 Btu/hr with TE (thermal efficiency) rating | Custom |

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

[309] Ibid.

[310] The Technical support documents for federal residential appliance standards: http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/appendix_e.pdf Note that this value is below the 20 years used by CA's DEER and the range of 20-40 year estimate made by the Consortium for Energy Efficiency in 2010

[311] Average of low and high incremental cost based on Nicor Gas program data for non-condensing and condensing boilers. Nicor Gas Energy Efficiency Plan 2011 - 2014, May 27, 2011 $1,470 for ≤ 300,000 Btu/hr for non-condensing hydronic boilers >85% AFUE & $3,365 for condensing boilers > 90% AFUE. The exception is $4,340 for AFUE ≥ 96% AFUE which was obtained from extrapolation above the size range that Nicor Gas Energy Efficiency Plan provided for incremental cost.

**Algorithm**

### CALCULATION OF ENERGY SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS ENERGY SAVINGS

ΔTherms = EFLH * Capacity * ((EfficiencyRating(actual) - EfficiencyRating(base)/ EfficiencyRating(base)) / 100,000

Where:

EFLH = Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use

Capacity = Nominal Heating Input Capacity Boiler Size (Btu/hr) for efficient unit not existing unit

= custom Boiler input capacity in Btu/hr

EfficiencyRating(base) = Baseline Boiler Efficiency Rating, dependant on year and boiler type. Baseline efficiency values by boiler type and capacity are found in the Definition of Baseline Equipment Section

EfficiencyRating(actual) = Efficent Boiler Efficiency Rating use actual value

| Measure Type | Actual AFUE |
|---|---|
| ENERGY STAR® Minimum | 85% |
| AFUE 90% | 90% |
| AFUE 95% | 95% |
| AFUE ≥ 96% | ≥ 96% |
| Custom | Value to one significant digit i.e. 95.7% |

**EXAMPLE**

For example, a 150,000 btu/hr water boiler meeting AFUE 90% in Rockford at a high rise office building , in the year 2012

ΔTherms = 2,089* 150,000 * (0.90-0.80)/0.80) / 100,000 Btu/Therm

= 392 Therms

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HVC-BOIL-V05-150601**

## 4.4.11 High Efficiency Furnace

### DESCRIPTION

This measure covers the installation of a high efficiency gas furnace in lieu of a standard efficiency gas furnace in a commercial or industrial space. High efficiency gas furnaces achieve savings through the utilization of a sealed, super insulated combustion chamber, more efficient burners, and multiple heat exchangers that remove a significant portion of the waste heat from the flue gasses. Because multiple heat exchangers are used to remove waste heat from the escaping flue gasses, most of the flue gasses condense and must be drained. Furnaces equipped with ECM fan motors can save additional electric energy

This measure was developed to be applicable to the following program types: TOS RF and EREP. If applied to other program types, the measure savings should be verified.

Time of sale:

a. The installation of a new high efficiency, gas-fired condensing furnace in a commercial location. This could relate to the replacement of an existing unit at the end of its useful life, or the installation of a new system.

Early replacement:

a. The early removal of an existing functioning AFUE 75% or less furnace from service, prior to its natural end of life, and replacement with a new high efficiency unit. Savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life. At time of writing, the DOE had rescinded the next Federal Standard change for furnaces; however it is likely that a new standard will be in effect after the assumed remaining useful life of the existing unit. For the purposes of this measure- the new baseline is assumed to be 90%.
b. The assumption of the existing unit efficiency in the Early Replacement section of this TRM is based upon the average efficiency of units that were classified in Ameren's PY3-PY4 as functioning and AFUE <=75%. Therefore it is only appropriate to use these Early Replacement assumptions where those conditions are met. The TAC defined "functioning" as the unit is fully operational – providing sufficient space conditioning (i.e. heat exchanger, compressors, pumps work effectively) and/or the cost of repair is under 20% of the new baseline replacement cost. Therefore in order to apply early replacement assumptions the programs should apply the following eligibility criteria: AFUE <=75% and cost of any repairs <$528.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a furnace with input energy less than 225,000 Btu/hr rated natural gas fired furnace with an Annual Fuel Utilization Efficiency (AFUE) rating and fan electrical efficiency exceeding the program requirements:

### DEFINITION OF BASELINE EQUIPMENT

Time of Sale: Although the current Federal Standard for gas furnaces is an AFUE rating of 78%, based upon review of available product in the AHRI database, the baseline efficiency for this characterization is assumed to be 80%. The baseline will be adjusted when the Federal Standard is updated.

Early replacement: The baseline for this measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and a new baseline unit for the remainder of the measure life. As discussed above we estimate that the new baseline unit that could be purchased in the year the existing unit would have needed replacing is 90%

### DEFINITION OF MEASURE LIFE

The expected measure life is assumed to be 16.5 years[312]

Remaining life of existing equipment is assumed to be 5.5 years[313].

### DEEMED MEASURE COST

Time of Sale: The incremental capital cost for this measure depends on efficiency as listed below[314]:

| AFUE | Installation Cost | Incremental Install Cost |
|---|---|---|
| 80% | $2011 | n/a |
| 90% | $2641 | $630 |
| 91% | $2727 | $716 |
| 92% | $2813 | $802 |
| 93% | $3049 | $1,038 |
| 94% | $3286 | $1,275 |
| 95% | $3522 | $1,511 |
| 96% | $3758 | $1,747 |

Early Replacement: The full installation cost is provided in the table above. The assumed deferred cost (after 5.5 years) of replacing existing equipment with a new baseline unit is assumed to be $2641. This cost should be discounted to present value using the utilities' discount rate.

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = \text{Heating Savings} + \text{Cooling Savings} + \text{Shoulder Season Savings}$$

Where:

[312] Average of 15-18 year lifetime estimate made by the Consortium for Energy Efficiency in 2010.
[313] Assumed to be one third of effective useful life
[314] Based on data from Appendix E of the Appliance Standards Technical Support Documents including equipment cost and installation labor (http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/appendix_e.pdf). Where efficiency ratings are not provided, the values are interpolated from those that are.

Heating Savings = Brushless DC motor or Electronically commutated motor (ECM)
= 418 kWh[315]

Cooling Savings = Brushless DC motor or electronically commutated motor (ECM) savings during cooling season

If air conditioning = 263 kWh

If no air conditioning = 175 kWh

If unknown (weighted average)= 241 kWh[316]

Shoulder Season Savings = Brushless DC motor or electronically commutated motor (ECM) savings during shoulder seasons

= 51 kWh

**EXAMPLE**

For example, a blower motor in an office building where air conditioning presence is unknown:

ΔkWh = Heating Savings + Cooling Savings + Shoulder Season Savings

= 418 +263 + 51

= 732 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

For units that have evaporator coils and condensing units and are cooling in the summer in addition to heating in the winter the summer coincident peak demand savings should be calculated. If the unit is not equipment with coils or condensing units, the summer peak demand savings will not apply.

ΔkW = (ΔkWh/HOURSyear) * CF

Where:

HOURSyear = Actual hours per year if known, otherwise use hours from Table below for building type[317].

| Building Type | Pumps and fans (h/yr) |
|---|---|
| College/University | 4216 |
| Grocery | 5840 |
| Heavy Industry | 3585 |
| Hotel/Motel | 6872 |

[315] To estimate heating, cooling and shoulder season savings for Illinois, VEIC adapted results from a 2009 Focus on Energy study of BPM blower motor savings in Wisconsin. This study included effects of behavior change based on the efficiency of new motor greatly increasing the amount of people that run the fan continuously. The savings from the Wisconsin study were adjusted to account for different run hour assumptions (average values used) for Illinois. See: FOE to IL Blower Savings.xlsx.

[316] The weighted average value is based on assumption that 75% of buildings installing BPM furnace blower motors have Central AC.

[317] ComEd Trm June 1, 2010 page 139. The Office hours is based upon occupancy from the eQuest model developed for EFLH, since it was agreed the ComEd value was too low.

| Building Type | Pumps and fans (h/yr) |
|---|---|
| Light Industry | 2465 |
| Medical | 6871 |
| Office | 2301 |
| Restaurant | 4654 |
| Retail/Service | 3438 |
| School(K-12) | 2203 |
| Warehouse | 3222 |
| Average=Miscellaneous | 4103 |

CF =Summer Peak Coincidence Factor for measure is provided below for different building types[318]:

| Location | CF |
|---|---|
| Restaurant | 0.80 |
| Office | 0.66 |
| School (K-12) | 0.22 |
| College/University | 0.56 |
| Medical | 0.75 |

**EXAMPLE**

For example, a blower motor in an office building where air conditioning presence is unknown:

ΔkW = (732 / 2301) * 0.66

= 0.21 kW

**NATURAL GAS ENERGY SAVINGS**

Time of Sale:

ΔTherms = EFLH * Capacity * ((AFUE(eff) – AFUE(base)/AFUE(base))/ 100,000 Btu/Therm

Early replacement[319]:

ΔTherms for remaining life of existing unit (1st 5.5 years):

[318] Based on DEER 2008 values

[319] The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).

ΔTherms = EFLH * Capacity * (AFUE(eff) – AFUE(exist)/ AFUE(exist)) / 100,000 Btu/Therm

ΔTherms for remaining measure life (next 11 years):

ΔTherms = EFLH * Capacity * (AFUE(eff) - AFUE(base)/AFUE(base)) / 100,000 Btu/Therm

Where:

EFLH = Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use

Capacity = Nominal Heating Input Capacity Furnace Size (Btu/hr) for efficient unit not existing unit

= custom Furnace input capacity in Btu/hr

AFUE(exist) = Existing Furnace Annual Fuel Utilization Efficiency Rating

= Use actual AFUE rating where it is possible to measure or reasonably estimate.

If unknown, assume 64.4 AFUE% [320].

AFUE(base) = Baseline Furnace Annual Fuel Utilization Efficiency Rating, dependant on year as listed below:

Dependent on program type as listed below[321]:

| Program Year | AFUE(base) |
|---|---|
| Time of Sale | 80% |
| Early Replacement | 90% |

AFUE(eff) = Efficent Furnace Annual Fuel Utilization Efficiency Rating.

= Actual. If Unknown, assume 95%[322]

**EXAMPLE**

For example, a 150,000 btu/hr 92% efficient furnace at a low rise office building in Rockford, in the year 2012

ΔTherms = 1428 * 150,000 * ((0.92-0.80)/0.80)/ 100,000

= 321 Therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

[320] Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.
[321] Though the Federal Minimum AFUE is 78%, there were only 50 models listed in the AHRI database at that level. At AFUE 79% the total rises to 308. There are 3,548 active furnace models listed with AFUE ratings between 78 and 80.
[322] Minimum ENERGY STAR efficiency after 2.1.2012.

**MEASURE CODE: CI-HVC-FRNC-V04-150601**

## 4.4.12 Infrared Heaters (all sizes), Low Intensity

### DESCRIPTION

This measure applies to natural gas fired low-intensity infrared heaters with an electric ignition that use non-conditioned air for combustion

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a natural gas heater with an electric ignition that uses non-conditioned air for combustion

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is a standard natural gas fired heater warm air heater.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years[323]

### DEEMED MEASURE COST

The incremental capital cost for this measure is $1716[324]

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

[323]ENERGY STAR and CEE do not currently provide calculators for this type of equipment therefore deemed values from Nicor Gas were used. Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011

[324]Ibid.

### NATURAL GAS ENERGY SAVINGS

The annual natural gas energy savings from this measure is a deemed value equaling 451 Therms[325]

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-HVC-IRHT-V01-120601

[325]Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011.These deemed values should be compared to PY evaluation and revised as necessary.

## 4.4.13 Package Terminal Air Conditioner (PTAC) and Package Terminal Heat Pump (PTHP)

### DESCRIPTION

A PTAC is a packaged terminal air conditioner that cools and sometimes provides heat through an electric resistance heater (heat strip). A PTHP is a packaged terminal heat pump. A PTHP uses its compressor year round to heat or cool. In warm weather, it efficiently captures heat from inside your building and pumps it outside for cooling. In cool weather, it captures heat from outdoor air and pumps it into your home, adding heat from electric heat strips as necessary to provide heat.

This measure characterizes:

a) Time of Sale: the purchase and installation of a new efficient PTAC or PTHP.
b) Early Replacement: the early removal of an existing PTAC or PTHP from service, prior to its natural end of life, and replacement with a new efficient PTAC or PTHP unit. Savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life. The measure is only valid for non-fuel switching installations – for example replacing a cooling only PTAC with a PTHP can currently not use the TRM.

This measure was developed to be applicable to the following program types: TOS NC, EREP. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be PTACs or PTHPs that exceed baseline efficiencies.

### DEFINITION OF BASELINE EQUIPMENT

Time of Sale: the baseline conditions is provided in the Federal Baseline reference table provided below.

Early Replacement: the baseline is the existing PTAC or PTHP for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years. [326]

Remaining life of existing equipment is assumed to be 5 years[327]

### DEEMED MEASURE COST

Time of Sale: The incremental capital cost for this equipment is estimated to be $84/ton.[328]

Early Replacement: The measure cost is the full cost of removing the existing unit and installing a new one. The actual program cost should be used. If unknown assume $1,047 per ton[329].

The assumed deferred cost (after 5 years) of replacing existing equipment with new baseline unit is assumed to be

[326] Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007
[327] Standard assumption of one third of effective useful life.
[328] DEER 2008. This assumes that baseline shift from IECC 2006 to IECC 2012 carries the same incremental costs. Values should be verified during evaluation
[329] Based on DCEO – IL PHA Efficient Living Program data.

$963 per ton[330]. This cost should be discounted to present value using the utilities' discount rate.

### LOADSHAPE

Loadshape C03 - Commercial Cooling

### COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market. Both values provided are based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)

= 91.3% [331]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)

= 47.8% [332]

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

Electric savings for PTACs and PTHPs should be calculated using the following algorithms

### ENERGY SAVINGS

Time of Sale:

PTAC ΔkWh[333] = Annual kWh $Savings_{cool}$

PTHP ΔkWh = Annual kWh $Savings_{cool}$ + Annual kWh $Savings_{heat}$

Annual kWh $Savings_{cool}$ = $(kBtu/hr_{cool}) * [(1/EERbase) – (1/EERee)] * EFLH_{cool}$

Annual kWh $Savings_{heat}$ = $(kBtu/hr_{heat})/3.412 * [(1/COPbase) – (1/COPee)] * EFLH_{heat}$

Early Replacement:

ΔkWh for remaining life of existing unit (1st 5years) = Annual kWh $Savings_{cool}$ + Annual kWh $Savings_{heat}$

[330] Based on subtracting TOS incremental cost from the DCEO data.

[331] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

[332] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

[333] There are no heating efficiency improvements for PTACs since although some do provide heating, it is always through electric resistance and therefore the COPbase and COPee would be 1.0.

Annual kWh $Savings_{cool}$ = $(kBtu/hr_{cool})$ * [(1/EERexist) – (1/EERee)] * $EFLH_{cool}$

Annual kWh $Savings_{heat}$ = $(kBtu/hr_{heat})$/3.412 * [(1/COPexist) – (1/COPee)] * $EFLH_{heat}$

ΔkWh for remaining measure life (next 10 years) = Annual kWh $Savings_{cool}$ + Annual kWh $Savings_{heat}$

Annual kWh $Savings_{cool}$ = $(kBtu/hr_{cool})$ * [(1/EERbase) – (1/EERee)] * $EFLH_{cool}$

Annual kWh $Savings_{heat}$ = $(kBtu/hr_{heat})$/3.412 * [(1/COPbase) – (1/COPee)] * $EFLH_{heat}$

Where:

$kBtu/hr_{cool}$ = capacity of the cooling equipment in kBtu per hour (1 ton of cooling capacity equals 12 kBtu/hr).

= Actual installed

$EFLH_{cool}$ = Equivalent Full Load Hours for cooling are provided in section 4.4 HVAC End Use:

$EFLH_{heat}$ = Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use

EERexist = Energy Efficiency Ratio of the existing equipment

= Actual. If unknown assume 8.1 EER[334]

EERbase = Energy Efficiency Ratio of the baseline equipment; see the table below for values.

Copy of Table C403.2.3(3), IECC 2012: Minimum Efficiency Reguirements: Electrically operated packaged terminal air conditioners, packaged terminal heat pumps

| Equipment Type | Minimum Efficiency as of 10/08/2012 |
|---|---|
| PTAC (Cooling mode) New Construction | 13.8 – (0.300 x Cap/1000) EER |
| PTAC (Cooling mode) Replacements | 10.9 – (0.213 x Cap/1000) EER |
| PTHP (Cooling mode) New Construction | 14.0 – (0.300 x Cap/1000) EER |
| PTHP (Cooling mode) Replacements | 10.8 – (0.213 x Cap/1000) EER |
| PTHP (Heating mode) New Construction | 3.2 – (0.026 x Cap/1000) COP |
| PTHP (Heating mode) Replacements | 2.9 – (0.026 x Cap/1000) COP |

"Cap" = The rated cooling capacity of the project in Btu/hr. If the units capacity is less than 7000 Btu/hr, use 7,000 Btu/hr in the calculation. If the unit's capacity is greater

[334] Estimated using the IECC building energy code up until year 2003 (p107; https://law.resource.org/pub/us/code/ibr/icc.iecc.2000.pdf) and assuming a 1 ton unit; EER = 10 – (0.16 * 12,000/1,000) = 8.1.

than 15,000 Btu/hr, use 15,000 Btu/hr in the calculations.

Replacement unit shall be factory labeled as follows "MANUFACTURED FOR REPLACEMENT APPLICATIONS ONLY; NOT TO BE INSTALLED IN NEW CONSTRUCTION PROJECTS", Replacement efficiencies apply only to units with existing sleeves less than 16 inches (406mm) in height and less than 42 inches (1067 mm) in width.

EEReе = Energy Efficiency Ratio of the energy efficient equipment. For air-cooled air conditioners < 65 kBtu/hr, if the actual EEReе is unknown, assume the following conversion from SEER to EER: EER≈SEER/1.1.

= Actual installed

$kBtu/hr_{heat}$ = capacity of the heating equipment in kBtu per hour.

= Actual installed

3.412 = Btu per Wh.

COPexist = coefficient of performance of the existing equipment

= Actual. If unknown assume 1.0 COP for PTAC units and 2.6 COP[335] for PTHPs.

COPbase = coefficient of performance of the baseline equipment; see table above for values.

COPee = coefficient of performance of the energy efficient equipment.

= Actual installed

Time of Sale (assuming new construction baseline):

For example a 1 ton PTAC with an efficient EER of 12 at a hotel in Rockford saves:

= [(12) * [(1/10.2) – (1/12)] * 1,042

= 184 kWh

Early Replacement (assuming replacement baseline for deferred replacement in 5 years):

For example a 1 ton PTHP with an efficient EER of 12, COP of 3.0 in Rockford replaces a PTAC unit (with electric resistance heat) with unknown efficiency.

ΔkWh for remaining life of existing unit (1st 5years)

= (12 * (1/8.1 – 1/12) * 1,042) + (12/3.412 * (1/1.0 – 1/3.0) * 1,758)

= 502 + 4,122

= 4,624 kWh

ΔkWh for remaining measure life (next 10 years)

= (12 * (1/8.3 – 1/12) * 1,042) + (12/3.412 * (1/1.0 – 1/3.0) * 1,758)

= 465 + 4,122

= 34,587 kWh

[335] Estimated using the IECC building energy code up until year 2003 (p107; https://law.resource.org/pub/us/code/ibr/icc.iecc.2000.pdf) and assuming a 1 ton unit; COP = 2.9 – (0.026 * 12,000/1,000) = 2.6

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

Time of Sale:

$\Delta kW = (kBtu/hr_{cool}) * [(1/EERbase) – (1/EERee)] *CF$

Early Replacement:

$\Delta kW$ for remaining life of existing unit (1st 5years) $= (kBtu/hr_{cool}) * [(1/EERexist) – (1/EERee)] *CF$

$\Delta kWh$ for remaining measure life (next 10 years) $= (kBtu/hr_{cool}) * [(1/EERbase) – (1/EERee)] *CF$

Where:

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)

= 91.3% [336]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)

= 47.8% [337]

Time of Sale:

For example a 1 ton replacement cooling unit with no heating with an efficient EER of 12 at a hotel in Rockford saves

$\Delta kW_{SSP} = (12 * (1/10.2 – 1/12) *0.913$

$= 0.16$ kW

For example a 1 ton PTHP with an efficient EER of 12, COP of 3.0 in Rockford replaces a PTAC unit with unknown efficiency.

$\Delta kW$ for remaining life of existing unit (1st 5years):

$\Delta kW_{SSP} = 12 * (1/8.1 – 1/12) * 0.913$

$= 0.44$ kW

$\Delta kW$ for remaining measure life (next 10 years):

$\Delta kW_{SSP} = 12 * (1/8.3 – 1/12) * 0.913$

$= 0.41$ kW

**NATURAL GAS ENERGY SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

[336] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

[337] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HVC-PTAC-V05-150601**

## 4.4.14 Pipe Insulation

### DESCRIPTION

This measure provides rebates for installation of ≥1" or ≥2" fiberglass, foam, calcium silicate or other types of insulation with similar insulating properties to existing bare pipe on straight piping as well as other pipe components such as elbows, tees, valves, and flanges for all non-residential installations.

Default per linear foot savings estimates are provided for the both exposed indoor or above ground outdoor piping distributing fluid in the following system types (natural gas fired systems only):

- Hydronic heating systems (with or without outdoor reset controls), including:
  - boiler systems that do not circulate water around a central loop and operate upon demand from a thermostat ("non-recirculation")
  - systems that recirculate during heating season only ("Recirculation – heating season only")
  - systems recirculating year round ("Recirculation – year round")
- Domestic hot water
- Low and high-pressure steam systems
  - non-recirculation
  - recirculation - heating season only
  - recirculation - year round

Process piping can also use the algorithms provided but requires custom entry of hours.

Minimum qualifying nominal pipe diameter is 1." Indoor piping must have at least 1" of insulation and outdoor piping must have at least 2" of insulation and include an all-weather protective jacket. New advanced insulating materials may be thinner and savings can be calculated with 3E Plus.

This measure was developed to be applicable to the following program types: RF, DI

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient case is installing pipe wrap insulation to a length of pipe. Indoor piping must have at least 1" of insulation (or equivalent R-value) and outdoor piping must have at least 2" of insulation (or equivalent R-value) and include an all-weather protective jacket. Minimum qualifying pipe diameter is 1." Insulation must be continuous and contiguous over fittings that directly connect to straight pipe, including elbows and tees.[338]

### DEFINITION OF BASELINE EQUIPMENT

The base case for savings estimates is a bare pipe. Pipes are required by new construction code to be insulated but are still commonly found uninsulated in older commercial buildings.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 15 years.[339]

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[338] *ASHRAE Handbook—Fundamentals*, 23.14; Hart, G., "Saving energy by insulating pipe components on steam and hot water distribution systems", *ASHRAE Journal*, October 2011

[339] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf

### DEEMED MEASURE COST

Actual costs should be used if known. Otherwise the deemed measure costs below based on RS Means[340] pricing reference materials may be used.[341] The following table summarizes the estimated costs for this measure per foot of insulation added and include installation costs:

| Insulation Thickness | | |
|---|---|---|
| | 1 Inch (Indoor) | 2 Inches (Outdoor) |
| Pipe- RS Means # | 220719.10.5170 | 220719.10.5530 |
| Jacket- RS Means # | 220719.10.0156 | 220719.10.0320 |
| Jacket Type | PVC | Aluminum |
| Insulation Cost per foot | $9.40 | $13.90 |
| Jacket Cost per foot | $4.57 | $7.30 |
| **Total Cost per foot** | **$13.97** | **$21.20** |

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS SAVINGS

$$\Delta\text{therms per foot}^{342} = [((Q_{base} - Q_{eff}) * EFLH) / (100{,}000 * \eta Boiler)] * TRF$$

$$= [\text{Provided by tables below}] * TRF$$

[340] RS Means 2008. Mechanical Cost Data, pages 106 to 119

[341] RS Means 2010: "for fittings, add 3 linear feet for each fitting plus 4 linear feet for each flange of the fitting"

[342] This value comes from the reference table "Savings Summary by Building Type and System Type." The formula and the input tables in this section document assumptions used in calculation spreadsheet "Pipe Insulation Savings 2013-11-12.xlsx"

Δtherms = $(L_{sp} + L_{oc,i})$ * Δtherms per foot

Where:

EFLH = Equivalent Full Load Hours for Heating

= Actual or defaults by building type provided in Section 4.4, HVAC end use

For year round recirculation or domestic hot water:

= 8,766

For heating season recirculation, hours with the outside air temperature below 55°F:

| Zone | Hours |
|---|---|
| Zone 1 (Rockford) | 5,039 |
| Zone 2 (Chicago) | 4,963 |
| Zone 3 (Springfield) | 4,495 |
| Zone 4 (Belleville/ | 4,021 |
| Zone 5 (Marion) | 4,150 |
| Zone 1 (Rockford) | 5,039 |

$Q_{base}$ = Heat Loss from Bare Pipe (Btu/hr/ft)

= See table below

$Q_{eff}$ = Heat Loss from Insulated Pipe (Btu/hr/ft)

= See table below

100,000 = conversion factor (1 therm = 100,000 Btu)

ηBoiler = Efficiency of the boiler being used to generate the hot water or steam in the pipe

= 81.9% for water boilers [343]

= 80.7% for steam boilers, except multifamily low-pressure [344]

= 64.8% for multifamily low-pressure steam boilers [345]

TRF = Thermal Regain Factor for space type, applied only to space heating energy and is applied to values resulting from Δtherms/ft tables below [346]

= See table below for base TRF values by pipe location

[343] Average efficiencies of units from the California Energy Commission (CEC).
[344] Ibid.
[345] Katrakis, J. and T.S. Zawacki. "Field-Measured Seasonal Efficiency of Intermediate-sized Low-Pressure Steam Boilers". ASHRAE V99, pt. 2, 1993.
[346] Thermal regain for *residential* pipe insulation measures is discussed in Home Energy Services Impact Evaluation, prepared for the Massachusetts Residential Retrofit and Low Income Program Area Evaluation, Cadmus Group, Inc., August 2012 and Andrews, John, Better Duct Systems for Home Heating and Cooling, U.S. Department of Energy, 2001. Recognizing the differences between residential and commercial heating systems, the factors have been adjusted based on professional judgment. This factor would benefit from additional study and evaluation.

May vary seasonally such as: TRF[summer] * summer hours + TRF[winter] * winter hours where TRF values reflecting summer and winter conditions are apportioned by the hours for those conditions. TRF may also be adjusted by building specific balance temperature and operating hours above and below that balance temperature.[347]

| Pipe Location | Assumed Regain | TRF, Thermal Regain Factor |
|---|---|---|
| Outdoor | 0% | 1.0 |
| Indoor, heated space | 85% | 0.15 |
| Indoor, semi- heated, (unconditioned space, with heat transfer to conditioned space. E.g.: boiler room, ceiling plenum, basement, crawlspace, wall) | 30% | 0.70 |
| Indoor, unheated, (no heat transfer to conditioned space) | 0% | 1.0 |
| Location not specified | 85% | 0.15 |
| Custom | Custom | 1 – assumed regain |

$L_{sp}$ = Length of straight pipe to be insulated (linear foot)

= actual installed ((linear foot)

$L_{oc,I}$ = Total equivalent length of the other components (valves and tees) of pipe to be insulated

= Actual installed (linear foot). See table "Equivalent Length of Other Components – Elbows and Tees" for equivalent lengths.

The heat loss estimates ($Q_{base}$ and $Q_{eff}$) were developed using the 3E Plus v4.0 software program.[348] The energy savings analysis is based on adding 1-inch (indoor) or 2-inch (outdoor) thick insulation around bare pipe. The thermal conductivity of pipe insulation varies by material and temperature rating; to obtain a typical value, a range of materials allowed for this measure were averaged. For insulation materials not in the table below, use 3E Plusv4.0 software to calculate $Q_{base}$ and $Q_{eff.}$

[347] Thermal Regain Factor_4-30-14.docx

[348] 3E Plus is a heat loss calculation software provided by the NAIMA (North American Insulation Manufacturer Association).

| Insulation Type | Conductivity (Btu.in / hr.ft$^2$.ºF @ 75F) | Max temp (ºF) |
|---|---|---|
| Polyethylene foam | 0.25 | 200 |
| Flexible polyurethane-based foam | 0.27 | 200 |
| Fiberglass | 0.31 | 250 |
| Melamine foam | 0.26 | 350 |
| Flexible silicon foam | 0.40 | 392 |
| Calcium silicate | 0.40 | 1200 |
| Cellular glass | 0.31 | 400 |
| Average conductivity of all these materials (Btu.in / hr.ft$^2$.ºF @ 75ºF) | 0.31 | |

The pipe fluid temperature assumption used depends upon both the system type and whether there is outdoor reset controls:

| System Type | Fluid temperature assumption (°F) |
|---|---|
| Hot Water space heating with outdoor reset - Non recirculation | 145 |
| Hot Water space heating without outdoor reset - Non recirculation | 170 |
| Hot Water space heating with outdoor reset – Recirculation heating season only | 145 |
| Hot Water space heating without outdoor reset – Recirculation heating season only | 170 |
| Hot Water space heating with outdoor reset – Recirculation year round | 130 |
| Hot Water space heating without outdoor reset – Recirculation year round | 170 |
| Domestic Hot Water | 125 |
| Low Pressure Steam | 225 |
| High Pressure Steam | 312 |

| | Indoor Insulation, Hot Water | Indoor Insulation, Low Pressure Steam | Indoor Insulation, High Pressure Steam | Domestic Hot Water | Outdoor Insulation, Hot Water | Outdoor Insulation, Low Pressure Steam | Outdoor Insulation, High Pressure Steam |
|---|---|---|---|---|---|---|---|
| Insulation thickness (inch) | 1 | 1 | 1 | 1 | 2 | 2 | 2 |
| Temperature, Fluid in Pipe (ºF) | 170 (w/o reset)<br>145 (w/ reset heat)<br>130 (w/reset year) | 225 | 312 | 125 | 170 (w/o reset)<br>145 (w/ reset heat)<br>130 (w/reset year) | 225 | 312 |
| Av. steam pressure (psig) | n/a | 10.9 | 82.8 | n/a | n/a | 10.9 | 82.8 |
| Operating Time (hrs/yr) | 2,746 (non-recirc)<br>5,039 (recirc heating season)<br>8,760 (recirc year round) | | | | | | |
| Ambient Temperature (ºF)[349] | 75 | 75 | 75 | 75 | 48.6 | 48.6 | 48.6 |
| Wind speed (mph)[350] | 0 | 0 | 0 | 0 | 9.4 | 9.4 | 9.4 |
| **Pipe parameters** | | | | | | | |
| Pipe material | Copper | Steel | Steel | Copper | Copper | Steel | Steel |
| Pipe size for Heat Loss Calc | 2” | 2” | 2” | 2” | 2” | 2” | 2” |
| Outer Diameter, Pipe, actual | 2.38” | 2.38” | 2.38” | 2.38” | 2.38” | 2.38” | 2.38” |
| Heat Loss, Bare Pipe (from 3EPlus) (Btu/hr.ft) | 114 (w/o reset)<br>78 (w/ reset heat)<br>58 (w/reset year) | 232 | 432 | 52 | 460 (w/o reset)<br>363 (w/ reset heat)<br>306 (w/reset year) | 710 | 1101 |
| **Insulation parameters** | | | | | | | |
| Outer diameter, insulation | 4.38” | 4.38” | 4.38” | 4.38” | 4.38” | 4.38” | 4.38” |
| Average Heat Loss, Insulation (from 3EPlus) (Btu/hr.ft) | 24 (w/o reset)<br>17 (w/ reset heat) | 40 | 70 | 13.25 | 21 (w/o reset)<br>16 (w/ reset heat) | 32 | 52 |

[349] DOE Weather Data. http://apps1.eere.energy.gov/buildings/energyplus/weatherdata/4_north_and_central_america_wmo_region_4/1_usa/USA_IL_Aurora.Muni.AP.744655_TMY3.stat Ibid.
[350] Ibid.

| | Indoor Insulation, Hot Water | Indoor Insulation, Low Pressure Steam | Indoor Insulation, High Pressure Steam | Domestic Hot Water | Outdoor Insulation, Hot Water | Outdoor Insulation, Low Pressure Steam | Outdoor Insulation, High Pressure Steam |
|---|---|---|---|---|---|---|---|
| | 13 (w/reset year) | | | | 13 (w/reset year) | | |
| **Annual Energy Savings** | | | | | | | |
| Boiler / Water Heater efficiency | 81.9% | 80.7% (64.8% for MF) | 80.7% | 67% | 81.9% | 80.7% (64.8% for MF) | 80.7% |
| Annual Gas Use, Base Case (therms/yr/ft) | 3.8 (w/o reset)<br>4.8 (w/ reset heat)<br>6.2 (w/reset year) | 7.9 (non recirc)<br>14.5 (recirc heat)<br>25.2 (recirc year) | 14.7 (non recirc)<br>27.0 (recirc heat)<br>46.9 (recirc year) | 6.76 | 15.4 (w/o reset)<br>22.5 (w/ reset heat)<br>32.7 (w/reset year) | 24.1 (non recirc)<br>44.3 (recirc heat)<br>77.0 (recirc year) | 37.5 (non recirc)<br>68.7 (recirc heat)<br>119.5 (recirc year) |
| Annual Gas Use, Measure case (therms/yr/ft) | 0.8 (w/o reset)<br>1.1 (w/ reset heat)<br>1.4 (w/reset year) | 1.4 (non recirc)<br>2.5 (recirc heat)<br>4.4 (recirc year) | 2.4 (non recirc)<br>4.4 (recirc heat)<br>7.6 (recirc year) | 1.73 | 0.7 (w/o reset)<br>1.0 (w/ reset heat)<br>1.4 (w/reset year) | 1.1 (non recirc)<br>2.0 (recirc heat)<br>3.4 (recirc year) | 1.8 (non recirc)<br>3.2 (recirc heat)<br>5.6 (recirc year) |
| Annual Gas Savings (therms/yr/ft) | 3.0 (w/o reset)<br>3.7 (w/ reset heat)<br>4.8 (w/reset year) | 6.5 (non recirc)<br>12.0 (recirc heat)<br>20.8 (recirc year) | 12.3 (non recirc)<br>22.6 (recirc heat)<br>39.3 (recirc year) | 5.0 | 14.7 (w/o reset)<br>21.4 (w/ reset heat)<br>31.3 (w/reset year) | 23.1 (non recirc)<br>42.3 (recirc heat)<br>73.6 (recirc year) | 35.7 (non recirc)<br>65.5 (recirc heat)<br>113.9 (recirc year) |

Heat = heating season only, year = year round

Values below must be multiplied by the appropriate Thermal Regain Factor (TRF). All variables were the same except for hours of operation in the calculation of the default savings per foot for the various building types and applications as presented in the table below:

**Savings Summary for Indoor pipe insulation by System Type and Building Type (Δtherms per foot) (continues for 3.5 pages)**

| | | | Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam) | | | | |
|---|---|---|---|---|---|---|---|
| Location | System Type | Building Type | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| Indoor | Hot Water Space Heating with outdoor reset – non-recirculation | Assembly | 1.32 | 1.36 | 1.21 | 0.81 | 1.24 |
| | | Assisted Living | 1.25 | 1.22 | 1.07 | 0.79 | 0.95 |
| | | College | 1.13 | 1.06 | 0.95 | 0.53 | 0.63 |
| | | Convenience Store | 1.10 | 1.01 | 0.90 | 0.65 | 0.72 |
| | | Elementary School | 1.32 | 1.29 | 1.13 | 0.78 | 0.95 |
| | | Garage | 0.73 | 0.72 | 0.63 | 0.50 | 0.56 |
| | | Grocery | 1.19 | 1.19 | 1.04 | 0.65 | 0.78 |
| | | Healthcare Clinic | 1.17 | 1.20 | 1.05 | 0.71 | 0.75 |
| | | High School | 1.37 | 1.38 | 1.23 | 0.88 | 1.03 |
| | | Hospital - CAV no econ | 1.31 | 1.35 | 1.15 | 0.99 | 1.12 |
| | | Hospital - CAV econ | 1.33 | 1.37 | 1.17 | 1.01 | 1.15 |
| | | Hospital - VAV econ | 0.54 | 0.51 | 0.39 | 0.23 | 0.25 |
| | | Hospital - FCU | 0.98 | 1.12 | 0.91 | 1.07 | 1.44 |
| | | Hotel/Motel | 1.31 | 1.27 | 1.14 | 0.78 | 0.96 |
| | | Hotel/Motel - Common | 1.19 | 1.21 | 1.15 | 0.93 | 0.98 |
| | | Hotel/Motel - Guest | 1.30 | 1.26 | 1.13 | 0.75 | 0.93 |
| | | Manufacturing Facility | 0.78 | 0.75 | 0.70 | 0.42 | 0.47 |
| | | MF - High Rise | 1.13 | 1.12 | 1.02 | 0.87 | 0.87 |
| | | MF - High Rise - Common | 1.35 | 1.31 | 1.17 | 0.81 | 1.04 |
| | | MF - High Rise - Residential | 1.09 | 1.08 | 0.99 | 0.85 | 0.83 |
| | | MF - Mid Rise | 1.23 | 1.25 | 1.07 | 0.79 | 0.90 |
| | | Movie Theater | 1.35 | 1.33 | 1.24 | 0.94 | 1.12 |
| | | Office - High Rise - CAV no econ | 1.50 | 1.52 | 1.38 | 0.93 | 1.01 |
| | | Office - High Rise - CAV econ | 1.55 | 1.58 | 1.45 | 1.00 | 1.10 |
| | | Office - High Rise - VAV econ | 1.13 | 1.15 | 0.95 | 0.56 | 0.63 |

| | | | Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam) | | | | |
|---|---|---|---|---|---|---|---|
| Location | System Type | Building Type | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| | | Office - High Rise - FCU | 0.83 | 0.82 | 0.71 | 0.37 | 0.39 |
| | | Office - Low Rise | 1.06 | 1.06 | 0.84 | 0.51 | 0.59 |
| | | Office - Mid Rise | 1.17 | 1.18 | 0.99 | 0.63 | 0.70 |
| | | Religious Building | 1.19 | 1.11 | 1.07 | 0.78 | 0.89 |
| | | Restaurant | 1.00 | 1.00 | 0.90 | 0.68 | 0.81 |
| | | Retail - Department Store | 1.03 | 0.95 | 0.89 | 0.58 | 0.66 |
| | | Retail - Strip Mall | 0.99 | 0.91 | 0.81 | 0.56 | 0.60 |
| | | Warehouse | 1.08 | 1.01 | 1.04 | 0.65 | 0.80 |
| | | Unknown | 1.15 | 1.14 | 1.01 | 0.73 | 0.84 |
| | Hot Water Space Heating without outdoor reset – non-recirculation | Assembly | 1.96 | 2.00 | 1.79 | 1.19 | 1.83 |
| | | Assisted Living | 1.84 | 1.80 | 1.58 | 1.16 | 1.40 |
| | | College | 1.67 | 1.56 | 1.40 | 0.78 | 0.93 |
| | | Convenience Store | 1.62 | 1.50 | 1.33 | 0.95 | 1.06 |
| | | Elementary School | 1.95 | 1.90 | 1.68 | 1.16 | 1.40 |
| | | Garage | 1.08 | 1.06 | 0.93 | 0.74 | 0.82 |
| | | Grocery | 1.76 | 1.75 | 1.54 | 0.96 | 1.15 |
| | | Healthcare Clinic | 1.73 | 1.77 | 1.55 | 1.05 | 1.11 |
| | | High School | 2.02 | 2.03 | 1.82 | 1.30 | 1.52 |
| | | Hospital - CAV no econ | 1.93 | 1.99 | 1.69 | 1.46 | 1.65 |
| | | Hospital - CAV econ | 1.96 | 2.03 | 1.73 | 1.50 | 1.70 |
| | | Hospital - VAV econ | 0.80 | 0.76 | 0.57 | 0.34 | 0.37 |
| | | Hospital - FCU | 1.45 | 1.65 | 1.35 | 1.58 | 2.13 |
| | | Hotel/Motel | 1.93 | 1.87 | 1.69 | 1.16 | 1.41 |
| | | Hotel/Motel - Common | 1.75 | 1.78 | 1.69 | 1.38 | 1.45 |
| | | Hotel/Motel - Guest | 1.92 | 1.86 | 1.66 | 1.11 | 1.37 |
| | | Manufacturing Facility | 1.15 | 1.11 | 1.03 | 0.62 | 0.69 |
| | | MF - High Rise | 1.67 | 1.65 | 1.50 | 1.28 | 1.28 |
| | | MF - High Rise - Common | 1.99 | 1.93 | 1.73 | 1.19 | 1.54 |
| | | MF - High Rise - Residential | 1.61 | 1.60 | 1.46 | 1.26 | 1.23 |

| | | | Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam) | | | | |
|---|---|---|---|---|---|---|---|
| Location | System Type | Building Type | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| | | MF - Mid Rise | 1.82 | 1.84 | 1.59 | 1.17 | 1.33 |
| | | Movie Theater | 1.99 | 1.96 | 1.83 | 1.39 | 1.66 |
| | | Office - High Rise - CAV no econ | 2.21 | 2.24 | 2.04 | 1.37 | 1.49 |
| | | Office - High Rise - CAV econ | 2.29 | 2.33 | 2.14 | 1.48 | 1.63 |
| | | Office - High Rise - VAV econ | 1.67 | 1.70 | 1.40 | 0.83 | 0.93 |
| | | Office - High Rise - FCU | 1.22 | 1.21 | 1.04 | 0.55 | 0.58 |
| | | Office - Low Rise | 1.56 | 1.56 | 1.24 | 0.76 | 0.87 |
| | | Office - Mid Rise | 1.73 | 1.74 | 1.47 | 0.94 | 1.04 |
| | | Religious Building | 1.75 | 1.65 | 1.58 | 1.15 | 1.32 |
| | | Restaurant | 1.48 | 1.48 | 1.33 | 1.01 | 1.19 |
| | | Retail - Department Store | 1.52 | 1.40 | 1.31 | 0.85 | 0.97 |
| | | Retail - Strip Mall | 1.46 | 1.35 | 1.19 | 0.82 | 0.89 |
| | | Warehouse | 1.59 | 1.49 | 1.53 | 0.96 | 1.18 |
| | | Unknown | 1.70 | 1.68 | 1.50 | 1.07 | 1.25 |
| | Hot Water with outdoor reset | All buildings, Recirculation heating season only (Hours below 55F) | 3.73 | 3.68 | 3.33 | 2.98 | 3.08 |
| | Hot Water w/o outdoor reset | All buildings, Recirculation heating season only (Hours below 55F) | 5.51 | 5.43 | 4.92 | 4.40 | 4.54 |
| | Hot Water with outdoor reset | All buildings, Recirculation year round (All hours) | 4.79 | 4.79 | 4.79 | 4.79 | 4.79 |
| | Hot Water w/o outdoor reset | All buildings, Recirculation year round (All hours) | 9.58 | 9.58 | 9.58 | 9.58 | 9.58 |
| | Domestic Hot Water | DHW circulation loop | 5.02 | 5.02 | 5.02 | 5.02 | 5.02 |
| | LP Steam – non-recirculation | Assembly | 4.25 | 4.36 | 3.89 | 2.59 | 3.97 |
| | | Assisted Living | 4.01 | 3.92 | 3.44 | 2.53 | 3.04 |
| | | College | 3.64 | 3.40 | 3.04 | 1.69 | 2.02 |
| | | Convenience Store | 3.52 | 3.26 | 2.89 | 2.07 | 2.32 |
| | | Elementary School | 4.24 | 4.13 | 3.64 | 2.52 | 3.05 |
| | | Garage | 2.34 | 2.31 | 2.03 | 1.62 | 1.79 |
| | | Grocery | 3.83 | 3.81 | 3.34 | 2.08 | 2.49 |
| | | Healthcare Clinic | 3.76 | 3.85 | 3.36 | 2.29 | 2.42 |

| | | | Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam) | | | | |
|---|---|---|---|---|---|---|---|
| Location | System Type | Building Type | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| | | High School | 4.39 | 4.42 | 3.96 | 2.82 | 3.30 |
| | | Hospital - CAV no econ | 4.20 | 4.33 | 3.69 | 3.17 | 3.60 |
| | | Hospital - CAV econ | 4.25 | 4.41 | 3.76 | 3.26 | 3.70 |
| | | Hospital - VAV econ | 1.74 | 1.65 | 1.24 | 0.75 | 0.81 |
| | | Hospital - FCU | 3.15 | 3.60 | 2.93 | 3.44 | 4.63 |
| | | Hotel/Motel | 4.19 | 4.07 | 3.67 | 2.51 | 3.07 |
| | | Hotel/Motel - Common | 3.81 | 3.87 | 3.68 | 3.00 | 3.15 |
| | | Hotel/Motel - Guest | 4.18 | 4.05 | 3.62 | 2.42 | 2.98 |
| | | Manufacturing Facility | 2.49 | 2.41 | 2.23 | 1.35 | 1.51 |
| | | MF - High Rise | 4.52 | 4.46 | 4.07 | 3.46 | 3.47 |
| | | MF - High Rise - Common | 5.38 | 5.22 | 4.68 | 3.23 | 4.17 |
| | | MF - High Rise - Residential | 4.37 | 4.34 | 3.94 | 3.41 | 3.33 |
| | | MF - Mid Rise | 4.94 | 4.99 | 4.30 | 3.16 | 3.60 |
| | | Movie Theater | 4.33 | 4.26 | 3.98 | 3.03 | 3.61 |
| | | Office - High Rise - CAV no econ | 4.81 | 4.88 | 4.45 | 2.98 | 3.24 |
| | | Office - High Rise - CAV econ | 4.97 | 5.07 | 4.66 | 3.21 | 3.54 |
| | | Office - High Rise - VAV econ | 3.64 | 3.71 | 3.06 | 1.81 | 2.01 |
| | | Office - High Rise - FCU | 2.66 | 2.62 | 2.27 | 1.20 | 1.26 |
| | | Office - Low Rise | 3.40 | 3.39 | 2.69 | 1.65 | 1.89 |
| | | Office - Mid Rise | 3.77 | 3.78 | 3.19 | 2.03 | 2.26 |
| | | Religious Building | 3.82 | 3.58 | 3.43 | 2.51 | 2.87 |
| | | Restaurant | 3.21 | 3.22 | 2.89 | 2.19 | 2.60 |
| | | Retail - Department Store | 3.31 | 3.04 | 2.86 | 1.86 | 2.12 |
| | | Retail - Strip Mall | 3.17 | 2.94 | 2.59 | 1.79 | 1.93 |
| | | Warehouse | 3.46 | 3.23 | 3.33 | 2.08 | 2.56 |
| | | Unknown | 3.70 | 3.66 | 3.26 | 2.34 | 2.71 |
| | LP Steam | All buildings, Recirculation heating season only (Hours below 55F) | 11.99 | 11.81 | 10.70 | 9.57 | 9.88 |
| | LP Steam | All buildings, Recirculation year round (All hours) | 20.84 | 20.84 | 20.84 | 20.84 | 20.84 |

| | | | Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam) | | | | |
|---|---|---|---|---|---|---|---|
| Location | System Type | Building Type | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| | HP Steam – non-recirculation | Assembly | 8.02 | 8.22 | 7.34 | 4.89 | 7.49 |
| | | Assisted Living | 7.56 | 7.39 | 6.49 | 4.77 | 5.73 |
| | | College | 6.87 | 6.42 | 5.73 | 3.18 | 3.81 |
| | | Convenience Store | 6.65 | 6.14 | 5.45 | 3.91 | 4.37 |
| | | Elementary School | 8.00 | 7.79 | 6.87 | 4.75 | 5.76 |
| | | Garage | 4.42 | 4.35 | 3.82 | 3.05 | 3.38 |
| | | Grocery | 7.22 | 7.19 | 6.30 | 3.93 | 4.70 |
| | | Healthcare Clinic | 7.09 | 7.27 | 6.35 | 4.32 | 4.57 |
| | | High School | 8.28 | 8.34 | 7.48 | 5.33 | 6.23 |
| | | Hospital - CAV no econ | 7.92 | 8.16 | 6.95 | 5.98 | 6.79 |
| | | Hospital - CAV econ | 8.03 | 8.32 | 7.09 | 6.14 | 6.98 |
| | | Hospital - VAV econ | 3.28 | 3.12 | 2.35 | 1.41 | 1.53 |
| | | Hospital - FCU | 5.95 | 6.79 | 5.53 | 6.50 | 8.73 |
| | | Hotel/Motel | 7.91 | 7.69 | 6.93 | 4.74 | 5.79 |
| | | Hotel/Motel - Common | 7.18 | 7.30 | 6.95 | 5.65 | 5.94 |
| | | Hotel/Motel - Guest | 7.89 | 7.64 | 6.83 | 4.57 | 5.62 |
| | | Manufacturing Facility | 4.70 | 4.55 | 4.22 | 2.55 | 2.84 |
| | | MF - High Rise | 6.85 | 6.76 | 6.16 | 5.25 | 5.26 |
| | | MF - High Rise - Common | 8.15 | 7.91 | 7.09 | 4.89 | 6.31 |
| | | MF - High Rise - Residential | 6.62 | 6.57 | 5.97 | 5.17 | 5.04 |
| | | MF - Mid Rise | 7.48 | 7.57 | 6.51 | 4.79 | 5.46 |
| | | Movie Theater | 8.16 | 8.04 | 7.52 | 5.71 | 6.80 |
| | | Office - High Rise - CAV no econ | 9.07 | 9.20 | 8.39 | 5.62 | 6.12 |
| | | Office - High Rise - CAV econ | 9.38 | 9.57 | 8.80 | 6.06 | 6.67 |
| | | Office - High Rise - VAV econ | 6.86 | 6.99 | 5.76 | 3.41 | 3.80 |
| | | Office - High Rise - FCU | 5.02 | 4.95 | 4.27 | 2.27 | 2.38 |
| | | Office - Low Rise | 6.41 | 6.40 | 5.08 | 3.11 | 3.56 |
| | | Office - Mid Rise | 7.12 | 7.12 | 6.03 | 3.84 | 4.27 |
| | | Religious Building | 7.20 | 6.75 | 6.46 | 4.73 | 5.41 |

| | | | Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam) | | | | |
|---|---|---|---|---|---|---|---|
| Location | System Type | Building Type | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| | | Restaurant | 6.06 | 6.08 | 5.46 | 4.13 | 4.90 |
| | | Retail - Department Store | 6.25 | 5.74 | 5.39 | 3.51 | 4.00 |
| | | Retail - Strip Mall | 5.98 | 5.54 | 4.89 | 3.37 | 3.63 |
| | | Warehouse | 6.53 | 6.09 | 6.29 | 3.93 | 4.84 |
| | | Unknown | 6.97 | 6.91 | 6.14 | 4.41 | 5.11 |
| | HP Steam | All buildings, Recirculation heating season only (Hours below 55F) | 22.62 | 22.28 | 20.18 | 18.05 | 18.63 |
| | HP Steam | All buildings, Recirculation year round (All hours) | 39.32 | 39.32 | 39.32 | 39.32 | 39.32 |

**Savings Summary for Outdoor pipe insulation by System Type and Building Type (Δtherms per foot) (continues for 3.5 pages)**

| | | | Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam) | | | | |
|---|---|---|---|---|---|---|---|
| Location | System Type | Building Type | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| Outdoor | Hot Water Space Heating with outdoor reset – non-recirculation | Assembly | 7.58 | 7.77 | 6.94 | 4.62 | 7.08 |
| | | Assisted Living | 7.14 | 6.98 | 6.13 | 4.51 | 5.42 |
| | | College | 6.49 | 6.07 | 5.41 | 3.01 | 3.60 |
| | | Convenience Store | 6.28 | 5.80 | 5.15 | 3.70 | 4.13 |
| | | Elementary School | 7.56 | 7.36 | 6.50 | 4.49 | 5.44 |
| | | Garage | 4.18 | 4.11 | 3.61 | 2.88 | 3.19 |
| | | Grocery | 6.82 | 6.80 | 5.96 | 3.72 | 4.44 |
| | | Healthcare Clinic | 6.70 | 6.87 | 6.00 | 4.09 | 4.32 |
| | | High School | 7.83 | 7.88 | 7.07 | 5.03 | 5.89 |
| | | Hospital - CAV no econ | 7.49 | 7.71 | 6.57 | 5.65 | 6.41 |
| | | Hospital - CAV econ | 7.59 | 7.86 | 6.70 | 5.81 | 6.60 |
| | | Hospital - VAV econ | 3.10 | 2.95 | 2.22 | 1.33 | 1.44 |
| | | Hospital - FCU | 5.62 | 6.42 | 5.23 | 6.14 | 8.26 |
| | | Hotel/Motel | 7.47 | 7.26 | 6.55 | 4.48 | 5.47 |

| | | | Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam) | | | | |
|---|---|---|---|---|---|---|---|
| Location | System Type | Building Type | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| | | Hotel/Motel - Common | 6.79 | 6.90 | 6.57 | 5.34 | 5.61 |
| | | Hotel/Motel - Guest | 7.46 | 7.22 | 6.45 | 4.32 | 5.31 |
| | | Manufacturing Facility | 4.45 | 4.30 | 3.98 | 2.41 | 2.69 |
| | | MF - High Rise | 6.48 | 6.39 | 5.83 | 4.96 | 4.97 |
| | | MF - High Rise - Common | 7.70 | 7.48 | 6.70 | 4.62 | 5.96 |
| | | MF - High Rise - Residential | 6.26 | 6.21 | 5.64 | 4.89 | 4.77 |
| | | MF - Mid Rise | 7.07 | 7.15 | 6.15 | 4.53 | 5.16 |
| | | Movie Theater | 7.71 | 7.60 | 7.10 | 5.40 | 6.43 |
| | | Office - High Rise - CAV no econ | 8.57 | 8.70 | 7.93 | 5.31 | 5.78 |
| | | Office - High Rise - CAV econ | 8.86 | 9.04 | 8.32 | 5.73 | 6.31 |
| | | Office - High Rise - VAV econ | 6.48 | 6.61 | 5.45 | 3.22 | 3.59 |
| | | Office - High Rise - FCU | 4.75 | 4.67 | 4.04 | 2.14 | 2.25 |
| | | Office - Low Rise | 6.06 | 6.05 | 4.80 | 2.94 | 3.36 |
| | | Office - Mid Rise | 6.73 | 6.73 | 5.70 | 3.63 | 4.03 |
| | | Religious Building | 6.80 | 6.38 | 6.11 | 4.47 | 5.11 |
| | | Restaurant | 5.73 | 5.75 | 5.16 | 3.90 | 4.63 |
| | | Retail - Department Store | 5.91 | 5.42 | 5.09 | 3.31 | 3.78 |
| | | Retail - Strip Mall | 5.65 | 5.23 | 4.62 | 3.19 | 3.44 |
| | | Warehouse | 6.18 | 5.76 | 5.94 | 3.71 | 4.57 |
| | | Unknown | 6.59 | 6.53 | 5.81 | 4.17 | 4.83 |
| | Hot Water Space Heating without outdoor reset – non-recirculation | Assembly | 9.59 | 9.83 | 8.77 | 5.85 | 8.96 |
| | | Assisted Living | 9.04 | 8.83 | 7.76 | 5.70 | 6.86 |
| | | College | 8.21 | 7.68 | 6.85 | 3.80 | 4.56 |
| | | Convenience Store | 7.95 | 7.34 | 6.52 | 4.68 | 5.22 |
| | | Elementary School | 9.56 | 9.32 | 8.22 | 5.68 | 6.89 |
| | | Garage | 5.28 | 5.20 | 4.57 | 3.65 | 4.04 |
| | | Grocery | 8.63 | 8.60 | 7.54 | 4.70 | 5.62 |
| | | Healthcare Clinic | 8.47 | 8.70 | 7.59 | 5.17 | 5.47 |
| | | High School | 9.90 | 9.97 | 8.94 | 6.37 | 7.45 |
| | | Hospital - CAV no econ | 9.47 | 9.76 | 8.31 | 7.15 | 8.11 |

| | | | Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam) | | | | |
|---|---|---|---|---|---|---|---|
| Location | System Type | Building Type | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| | | Hospital - CAV econ | 9.60 | 9.95 | 8.48 | 7.35 | 8.34 |
| | | Hospital - VAV econ | 3.93 | 3.73 | 2.80 | 1.68 | 1.82 |
| | | Hospital - FCU | 7.11 | 8.12 | 6.61 | 7.77 | 10.45 |
| | | Hotel/Motel | 9.45 | 9.19 | 8.29 | 5.67 | 6.92 |
| | | Hotel/Motel - Common | 8.59 | 8.73 | 8.31 | 6.76 | 7.10 |
| | | Hotel/Motel - Guest | 9.44 | 9.13 | 8.16 | 5.47 | 6.72 |
| | | Manufacturing Facility | 5.63 | 5.44 | 5.04 | 3.05 | 3.40 |
| | | MF - High Rise | 8.19 | 8.08 | 7.37 | 6.27 | 6.29 |
| | | MF - High Rise - Common | 9.74 | 9.46 | 8.48 | 5.85 | 7.54 |
| | | MF - High Rise - Residential | 7.92 | 7.86 | 7.14 | 6.18 | 6.03 |
| | | MF - Mid Rise | 8.94 | 9.05 | 7.78 | 5.73 | 6.53 |
| | | Movie Theater | 9.76 | 9.61 | 8.99 | 6.83 | 8.14 |
| | | Office - High Rise - CAV no econ | 10.84 | 11.01 | 10.03 | 6.72 | 7.32 |
| | | Office - High Rise - CAV econ | 11.21 | 11.44 | 10.52 | 7.25 | 7.98 |
| | | Office - High Rise - VAV econ | 8.20 | 8.36 | 6.89 | 4.07 | 4.54 |
| | | Office - High Rise - FCU | 6.00 | 5.91 | 5.11 | 2.71 | 2.84 |
| | | Office - Low Rise | 7.67 | 7.65 | 6.08 | 3.72 | 4.25 |
| | | Office - Mid Rise | 8.51 | 8.52 | 7.21 | 4.59 | 5.10 |
| | | Religious Building | 8.61 | 8.07 | 7.73 | 5.66 | 6.47 |
| | | Restaurant | 7.25 | 7.27 | 6.53 | 4.94 | 5.85 |
| | | Retail - Department Store | 7.47 | 6.86 | 6.44 | 4.19 | 4.78 |
| | | Retail - Strip Mall | 7.15 | 6.62 | 5.85 | 4.03 | 4.35 |
| | | Warehouse | 7.81 | 7.29 | 7.52 | 4.69 | 5.78 |
| | | Unknown | 8.34 | 8.26 | 7.35 | 5.27 | 6.11 |
| | Hot Water with outdoor reset | All buildings, Recirculation heating season only (Hours below 55F) | 21.38 | 21.06 | 19.07 | 17.06 | 17.61 |
| | Hot Water without outdoor reset | All buildings, Recirculation heating season only (Hours below 55F) | 27.05 | 26.64 | 24.13 | 21.58 | 22.28 |
| | Hot Water with outdoor reset | All buildings, Recirculation year round (All hours) | 31.30 | 31.30 | 31.30 | 31.30 | 31.30 |
| | Hot Water without outdoor reset | All buildings, Recirculation year round (All hours) | 47.02 | 47.02 | 47.02 | 47.02 | 47.02 |

| | | | Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam) | | | | |
|---|---|---|---|---|---|---|---|
| Location | System Type | Building Type | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| | LP Steam – non-recirculation | Assembly | 15.01 | 15.38 | 13.73 | 9.15 | 14.02 |
| | | Assisted Living | 14.14 | 13.82 | 12.15 | 8.93 | 10.73 |
| | | College | 12.85 | 12.01 | 10.72 | 5.95 | 7.13 |
| | | Convenience Store | 12.44 | 11.49 | 10.20 | 7.32 | 8.17 |
| | | Elementary School | 14.96 | 14.58 | 12.86 | 8.88 | 10.78 |
| | | Garage | 8.27 | 8.14 | 7.15 | 5.71 | 6.32 |
| | | Grocery | 13.51 | 13.46 | 11.80 | 7.36 | 8.79 |
| | | Healthcare Clinic | 13.26 | 13.61 | 11.88 | 8.09 | 8.56 |
| | | High School | 15.50 | 15.60 | 13.99 | 9.97 | 11.66 |
| | | Hospital - CAV no econ | 14.82 | 15.27 | 13.01 | 11.19 | 12.70 |
| | | Hospital - CAV econ | 15.02 | 15.57 | 13.27 | 11.50 | 13.06 |
| | | Hospital - VAV econ | 6.14 | 5.84 | 4.39 | 2.64 | 2.85 |
| | | Hospital - FCU | 11.13 | 12.71 | 10.35 | 12.16 | 16.35 |
| | | Hotel/Motel | 14.80 | 14.38 | 12.97 | 8.87 | 10.84 |
| | | Hotel/Motel - Common | 13.45 | 13.66 | 13.00 | 10.58 | 11.12 |
| | | Hotel/Motel - Guest | 14.77 | 14.29 | 12.78 | 8.56 | 10.52 |
| | | Manufacturing Facility | 8.80 | 8.51 | 7.89 | 4.77 | 5.32 |
| | | MF - High Rise | 15.97 | 15.76 | 14.37 | 12.23 | 12.26 |
| | | MF - High Rise - Common | 18.99 | 18.44 | 16.53 | 11.39 | 14.71 |
| | | MF - High Rise - Residential | 15.43 | 15.31 | 13.92 | 12.05 | 11.75 |
| | | MF - Mid Rise | 17.43 | 17.63 | 15.17 | 11.16 | 12.72 |
| | | Movie Theater | 15.27 | 15.05 | 14.07 | 10.69 | 12.73 |
| | | Office - High Rise - CAV no econ | 16.97 | 17.22 | 15.70 | 10.51 | 11.45 |
| | | Office - High Rise - CAV econ | 17.55 | 17.91 | 16.47 | 11.35 | 12.49 |
| | | Office - High Rise - VAV econ | 12.83 | 13.09 | 10.79 | 6.37 | 7.11 |
| | | Office - High Rise - FCU | 9.40 | 9.26 | 8.00 | 4.25 | 4.45 |
| | | Office - Low Rise | 12.00 | 11.97 | 9.51 | 5.82 | 6.66 |
| | | Office - Mid Rise | 13.32 | 13.33 | 11.28 | 7.18 | 7.98 |
| | | Religious Building | 13.47 | 12.64 | 12.10 | 8.86 | 10.13 |
| | | Restaurant | 11.34 | 11.38 | 10.21 | 7.73 | 9.16 |

| | | | Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam) | | | | |
|---|---|---|---|---|---|---|---|
| Location | System Type | Building Type | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| | | Retail - Department Store | 11.69 | 10.74 | 10.08 | 6.56 | 7.48 |
| | | Retail - Strip Mall | 11.19 | 10.36 | 9.15 | 6.31 | 6.80 |
| | | Warehouse | 12.23 | 11.40 | 11.77 | 7.35 | 9.05 |
| | | Unknown | 13.05 | 12.93 | 11.50 | 8.25 | 9.57 |
| | LP Steam | All buildings, Recirculation heating season only (Hours below 55F) | 42.33 | 41.69 | 37.76 | 33.78 | 34.86 |
| | LP Steam | All buildings, Recirculation year round (All hours) | 73.59 | 73.59 | 73.59 | 73.59 | 73.59 |
| | HP Steam – non-recirculation | Assembly | 23.24 | 23.81 | 21.26 | 14.16 | 21.70 |
| | | Assisted Living | 21.89 | 21.40 | 18.80 | 13.82 | 16.61 |
| | | College | 19.90 | 18.60 | 16.60 | 9.22 | 11.04 |
| | | Convenience Store | 19.26 | 17.79 | 15.79 | 11.33 | 12.65 |
| | | Elementary School | 23.16 | 22.57 | 19.91 | 13.75 | 16.69 |
| | | Garage | 12.80 | 12.60 | 11.08 | 8.84 | 9.78 |
| | | Grocery | 20.91 | 20.83 | 18.26 | 11.39 | 13.61 |
| | | Healthcare Clinic | 20.53 | 21.07 | 18.39 | 12.53 | 13.25 |
| | | High School | 23.99 | 24.15 | 21.66 | 15.43 | 18.05 |
| | | Hospital - CAV no econ | 22.94 | 23.64 | 20.14 | 17.32 | 19.66 |
| | | Hospital - CAV econ | 23.25 | 24.10 | 20.54 | 17.80 | 20.22 |
| | | Hospital - VAV econ | 9.51 | 9.03 | 6.79 | 4.08 | 4.42 |
| | | Hospital - FCU | 17.24 | 19.67 | 16.02 | 18.82 | 25.31 |
| | | Hotel/Motel | 22.90 | 22.27 | 20.08 | 13.74 | 16.77 |
| | | Hotel/Motel - Common | 20.81 | 21.15 | 20.13 | 16.38 | 17.21 |
| | | Hotel/Motel - Guest | 22.87 | 22.13 | 19.78 | 13.24 | 16.28 |
| | | Manufacturing Facility | 13.63 | 13.18 | 12.21 | 7.38 | 8.24 |
| | | MF - High Rise | 19.85 | 19.59 | 17.86 | 15.20 | 15.24 |
| | | MF - High Rise - Common | 23.60 | 22.92 | 20.55 | 14.16 | 18.28 |
| | | MF - High Rise - Residential | 19.18 | 19.03 | 17.30 | 14.98 | 14.61 |
| | | MF - Mid Rise | 21.67 | 21.92 | 18.86 | 13.87 | 15.81 |
| | | Movie Theater | 23.64 | 23.29 | 21.78 | 16.55 | 19.71 |
| | | Office - High Rise - CAV no econ | 26.27 | 26.66 | 24.30 | 16.28 | 17.73 |

| | | | Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam) | | | | |
|---|---|---|---|---|---|---|---|
| Location | System Type | Building Type | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| | | Office - High Rise - CAV econ | 27.16 | 27.72 | 25.49 | 17.57 | 19.33 |
| | | Office - High Rise - VAV econ | 19.87 | 20.26 | 16.70 | 9.87 | 11.00 |
| | | Office - High Rise - FCU | 14.54 | 14.33 | 12.38 | 6.57 | 6.89 |
| | | Office - Low Rise | 18.58 | 18.53 | 14.72 | 9.00 | 10.31 |
| | | Office - Mid Rise | 20.61 | 20.64 | 17.46 | 11.12 | 12.36 |
| | | Religious Building | 20.85 | 19.56 | 18.72 | 13.71 | 15.67 |
| | | Restaurant | 17.55 | 17.61 | 15.81 | 11.96 | 14.18 |
| | | Retail - Department Store | 18.10 | 16.63 | 15.61 | 10.16 | 11.58 |
| | | Retail - Strip Mall | 17.32 | 16.04 | 14.17 | 9.77 | 10.53 |
| | | Warehouse | 18.93 | 17.65 | 18.21 | 11.37 | 14.02 |
| | | Unknown | 20.20 | 20.01 | 17.80 | 12.77 | 14.81 |
| | HP Steam | All buildings, Recirculation heating season only (Hours below 55F) | 65.53 | 64.54 | 58.45 | 52.29 | 53.97 |
| | HP Steam | All buildings, Recirculation year round (All hours) | 113.92 | 113.92 | 113.92 | 113.92 | 113.92 |

For insulation covering elbows and tees that connect straight pipe, a calculated surface area will be assumed based on the dimensions for fittings given by ANSI/ASME B36.19. The surface area is then converted to an equivalent length of pipe that must be added to the total length of straight pipe in order to calculate total savings. Equivalent pipe lengths are given in 1” increments in pipe diameter for simplicity. In the case of pipe diameters in between full inch diameters, the closest equivalent length should be used. The larger pipe sizes mostly apply to steam header piping, which has the most heat loss per foot.

**Calculated Surface Areas of Elbows and Tees**

| Nominal Pipe Diameter | Calculated Surface Area (ft) | |
|---|---|---|
| | 90 Degree Elbow[351] | Straight Tee[352] |
| 1” | 0.10 | 0.13 |
| 2” | 0.41 | 0.39 |
| 3” | 0.93 | 0.77 |
| 4” | 1.64 | 1.21 |

[351] Based on the dimensions for diameter, long radius, and short radius given by ANSI/ASME 36.19
[352] Based on the center to face and diameter dimensions given by ANSI/ASME B36.19

| Nominal Pipe Diameter | Calculated Surface Area (ft) | |
|---|---|---|
| | 90 Degree Elbow[351] | Straight Tee[352] |
| 5” | 2.57 | 1.77 |
| 6” | 3.70 | 2.44 |
| 8” | 6.58 | 3.95 |
| 10” | 10.28 | 5.98 |
| 12” | 14.80 | 8.34 |

**Equivalent Length of Other Components – Elbows and Tees ($L_{oc}$)**

| Nominal Pipe Diameter | Equivalent Length of Other Components (ft) | |
|---|---|---|
| | 90 Degree Elbow | Straight Tee |
| 1” | 0.30 | 0.38 |
| 2” | 0.66 | 0.63 |
| 3” | 1.01 | 0.84 |
| 4” | 1.40 | 1.03 |
| 5” | 1.76 | 1.22 |
| 6” | 2.13 | 1.41 |
| 8” | 2.91 | 1.75 |
| 10” | 3.65 | 2.13 |
| 12” | 4.44 | 2.50 |

For insulation around valves or flanges, a surface area from ASTM standard C1129-12 will be assumed for 2” pipes. For 1” pipes, which weren’t included in the standard, a linear-trended value will be used. The surface area is then converted to an equivalent length of either 1” or 2” straight pipe that must be added to the total length of straight pipe in order to calculate total savings.

**Calculated Surface Areas of Flanges and Valves**

| Valves | | | | |
|---|---|---|---|---|
| Class (psi) | 150 | 300 | 600 | 900 |
| NPS (in) | $ft^2$ | $ft^2$ | $ft^2$ | $ft^2$ |
| 1 | 0.69 | 1.8 | 1.8 | 2.4 |
| 2 | 2.21 | 2.94 | 2.94 | 5.2 |
| 2.5 | 2.97 | 3.51 | 3.91 | 6.6 |
| 3 | 3.37 | 4.39 | 4.69 | 6.5 |
| 4 | 4.68 | 6.06 | 7.64 | 9.37 |
| 6 | 7.03 | 9.71 | 13.03 | 15.8 |

| Flanges | | | | |
|---|---|---|---|---|
| Class (psi) | 150 | 300 | 600 | 900 |
| NPS (in) | $ft^2$ | $ft^2$ | $ft^2$ | $ft^2$ |
| 1 | 0.36 | 0.36 | 0.4 | 1.23 |
| 2 | 0.71 | 0.84 | 0.88 | 1.54 |
| | | | | |
| 3 | 1.06 | 1.32 | 1.36 | 1.85 |
| 4 | 1.44 | 1.83 | 2.23 | 2.64 |
| 6 | 2.04 | 2.72 | 3.6 | 4.37 |

| Valves | | | | |
|---|---|---|---|---|
| 8 | 10.3 | 13.5 | 18.4 | 23.8 |
| 10 | 13.8 | 18 | 26.5 | 32.1 |
| 12 | 16.1 | 24.1 | 31.9 | 41.9 |

| Flanges | | | | |
|---|---|---|---|---|
| 8 | 2.92 | 3.74 | 4.89 | 6.4 |
| 10 | 3.68 | 4.8 | 6.93 | 8.47 |
| 12 | 5.01 | 6.34 | 7.97 | 10.43 |

**Equivalent Length of Other Components - Flanges and Valves ($L_{oc}$)**

| ANSI Class (psi) | Equivalent Length of Other Components (ft) | | | |
|---|---|---|---|---|
| | 1" Valve | 1" Flange | 2" Valve | 2" Flange |
| 150 | 3.56 | 1.05 | 3.56 | 1.14 |
| 300 | 4.73 | 1.05 | 4.73 | 1.35 |
| 600 | 4.73 | 1.16 | 4.73 | 1.42 |
| 900 | 8.37 | 3.57 | 8.37 | 2.48 |

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-HVC-PINS-V03-150601

## 4.4.15 Single-Package and Split System Unitary Air Conditioners

### DESCRIPTION

This measure promotes the installation of high-efficiency unitary air-, water-, and evaporatively cooled air conditioning equipment, both single-package and split systems. Air conditioning (AC) systems are a major consumer of electricity and systems that exceed baseline efficiencies can save considerable amounts of energy. This measure could apply to the replacing of an existing unit at the end of its useful life or the installation of a new unit in a new or existing building.

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a high-efficiency air-, water-, or evaporatively cooled air conditioner that exceeds the energy efficiency requirements of the International Energy Conservation Code (IECC) 2012

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to be a standard-efficiency air-, water, or evaporatively cooled air conditioner that meets the energy efficiency requirements of the International Energy Conservation Code (IECC) 2012,. The rating conditions for the baseline and efficient equipment efficiencies must be equivalent.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years.[353]

### DEEMED MEASURE COST

The incremental capital cost for this measure is assumed to be $100 per ton.[354]

### LOADSHAPE

Loadshape C03 - Commercial Cooling

### COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market. Both values provided are based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)

= 91.3% [355]

[353] Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007.
[354] Based on a review of TRM incremental cost assumptions from Vermont, Wisconsin, and California. This assumes that baseline shift from IECC 2009 to IECC 2012 carries the same incremental costs. Values should be verified during evaluation

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)

= 47.8% [356]

**Algorithm**

**CALCULATION OF SAVINGS**

**ELECTRIC ENERGY SAVINGS**

For units with cooling capacities less than 65 kBtu/hr:

ΔkWH = (kBtu/hr) * [(1/SEERbase) – (1/SEERee)] * EFLH

For units with cooling capacities equal to or greater than 65 kBtu/hr:

ΔkWH = (kBtu/hr) * [(1/EERbase) – (1/EERee)] * EFLH

Where:

kBtu/hr = capacity of the cooling equipment actually installed in kBtu per hour (1 ton of cooling capacity equals 12 kBtu/hr).

SEERbase = Seasonal Energy Efficiency Ratio of the baseline equipment; see table below for default values[357]:

[355] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

[356] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

[357] International Energy Conservation Code (IECC) 2012

**TABLE C403.2.3(1)**
**MINIMUM EFFICIENCY REQUIREMENTS:**
**ELECTRICALLY OPERATED UNITARY AIR CONDITIONERS AND CONDENSING UNITS**

| EQUIPMENT TYPE | SIZE CATEGORY | HEATING SECTION TYPE | SUBCATEGORY OR RATING CONDITION | MINIMUM EFFICIENCY | | TEST PROCEDURE[a] |
|---|---|---|---|---|---|---|
| | | | | Before 6/1/2011 | As of 6/1/2011 | |
| Air conditioners, air cooled | < 65,000 Btu/h[b] | All | Split System | 13.0 SEER | 13.0 SEER | AHRI 210/240 |
| | | | Single Package | 13.0 SEER | 13.0 SEER | |
| Through-the-wall (air cooled) | ≤ 30,000 Btu/h[b] | All | Split system | 12.0 SEER | 12.0 SEER | |
| | | | Single Package | 12.0 SEER | 12.0 SEER | |
| Small-duct high-velocity (air cooled) | < 65,000 Btu/h[b] | All | Split System | 10.0 SEER | 10.0 SEER | |
| Air conditioners, air cooled | ≥ 65,000 Btu/h and < 135,000 Btu/h | Electric Resistance (or None) | Split System and Single Package | 11.2 EER<br>11.4 IEER | 11.2 EER<br>11.4 IEER | AHRI 340/360 |
| | | All other | Split System and Single Package | 11.0 EER<br>11.2 IEER | 11.0 EER<br>11.2 IEER | |
| | ≥ 135,000 Btu/h and < 240,000 Btu/h | Electric Resistance (or None) | Split System and Single Package | 11.0 EER<br>11.2 IEER | 11.0 EER<br>11.2 IEER | |
| | | All other | Split System and Single Package | 10.8 EER<br>11.0 IEER | 10.8 EER<br>11.0 IEER | |
| | ≥ 240,000 Btu/h and < 760,000 Btu/h | Electric Resistance (or None) | Split System and Single Package | 10.0 EER<br>10.1 IEER | 10.0 EER<br>10.1 IEER | |
| | | All other | Split System and Single Package | 9.8 EER<br>9.9 IEER | 9.8 EER<br>9.9 IEER | |
| | ≥ 760,000 Btu/h | Electric Resistance (or None) | Split System and Single Package | 9.7 EER<br>9.8 IEER | 9.7 EER<br>9.8 IEER | |
| | | All other | Split System and Single Package | 9.5 EER<br>9.6 IEER | 9.5 EER<br>9.6 IEER | |
| Air conditioners, water cooled | < 65,000 Btu/h[b] | All | Split System and Single Package | 12.1 EER<br>12.3 IEER | 12.1 EER<br>12.3 IEER | AHRI 210/240 |
| | ≥ 65,000 Btu/h and < 135,000 Btu/h | Electric Resistance (or None) | Split System and Single Package | 11.5 EER<br>11.7 IEER | 12.1 EER<br>12.3 IEER | AHRI 340/360 |
| | | All other | Split System and Single Package | 11.3 EER<br>11.5 IEER | 11.9 EER<br>12.1 IEER | |
| | ≥ 135,000 Btu/h and < 240,000 Btu/h | Electric Resistance (or None) | Split System and Single Package | 11.0 EER<br>11.2 IEER | 12.5 EER<br>12.7 IEER | |
| | | All other | Split System and Single Package | 10.8 EER<br>11.0 IEER | 12.3 EER<br>12.5 IEER | |
| | ≥ 240,000 Btu/h and < 760,000 Btu/h | Electric Resistance (or None) | Split System and Single Package | 11.0 EER<br>11.1 IEER | 12.4 EER<br>12.6 IEER | |
| | | All other | Split System and Single Package | 10.8 EER<br>10.9 IEER | 12.2 EER<br>12.4 IEER | |
| | ≥ 760,000 Btu/h | Electric Resistance (or None) | Split System and Single Package | 11.0 EER<br>11.1 IEER | 12.0 EER<br>12.4 IEER | |
| | | All other | Split System and Single Package | 10.8 EER<br>10.9 IEER | 12.0 EER<br>12.2 IEER | |

*(continued)*

**TABLE C403.2.3(1)—continued**
**MINIMUM EFFICIENCY REQUIREMENTS:**
**ELECTRICALLY OPERATED UNITARY AIR CONDITIONERS AND CONDENSING UNITS**

| EQUIPMENT TYPE | SIZE CATEGORY | HEATING SECTION TYPE | SUB-CATEGORY OR RATING CONDITION | MINIMUM EFFICIENCY | | TEST PROCEDURE[a] |
|---|---|---|---|---|---|---|
| | | | | Before 6/1/2011 | As of 6/1/2011 | |
| Air conditioners, evaporatively cooled | < 65,000 Btu/h[b] | All | Split System and Single Package | 12.1 EER<br>12.3 IEER | 12.1 EER<br>12.3 IEER | AHRI 210/240 |
| | ≥ 65,000 Btu/h and < 135,000 Btu/h | Electric Resistance (or None) | Split System and Single Package | 11.5 EER<br>11.7 IEER | 12.1 EER<br>12.3 IEER | AHRI 340/360 |
| | | All other | Split System and Single Package | 11.3 EER<br>11.5 IEER | 11.9 EER<br>12.1 IEER | |
| | ≥ 135,000 Btu/h and < 240,000 Btu/h | Electric Resistance (or None) | Split System and Single Package | 11.0 EER<br>11.2 IEER | 12.0 EER<br>12.2 IEER | |
| | | All other | Split System and Single Package | 10.8 EER<br>11.0 IEER | 11.8 EER<br>12.0 IEER | |
| | ≥ 240,000 Btu/h and < 760,000 Btu/h | Electric Resistance (or None) | Split System and Single Package | 11.0 EER<br>11.1 IEER | 11.9 EER<br>12.1 IEER | |
| | | All other | Split System and Single Package | 10.8 EER<br>10.9 IEER | 12.2 EER<br>11.9 IEER | |
| | ≥ 760,000 Btu/h | Electric Resistance (or None) | Split System and Single Package | 10.0 EER<br>11.1 IEER | 11.7 EER<br>11.9 IEER | |
| | | All other | Split System and Single Package | 10.8 EER<br>10.9 IEER | 11.5 EER<br>11.7 IEER | |
| Condensing units, air cooled | ≥ 135,000 Btu/h | | | 10.1 EER<br>11.4 IEER | 10.5 EER<br>14.0 IEER | AHRI 365 |
| Condensing units, water cooled | ≥ 135,000 Btu/h | | | 13.1 EER<br>13.6 IEER | 13.5 EER<br>14.0 IEER | |
| Condensing units, evaporatively cooled | ≥ 135,000 Btu/h | | | 13.1 EER<br>13.6 IEER | 13.5 EER<br>14.0 IEER | |

For SI: 1 British thermal unit per hour = 0.2931 W.

a. Chapter 6 of the referenced standard contains a complete specification of the referenced test procedure, including the reference year version of the test procedure.

b. Single-phase, air-cooled air conditioners less than 65,000 Btu/h are regulated by NAECA. SEER values are those set by NAECA.

SEERee = Seasonal Energy Efficiency Ratio of the energy efficient equipment (actually installed).

EERbase = Energy Efficiency Ratio of the baseline equipment; see table above for default values. Since IECC 2012 does not provide EER requirements for air-cooled air conditioners < 65 kBtu/hr, assume the following conversion from SEER to EER: EER≈SEER/1.1

EERee = Energy Efficiency Ratio of the energy efficient equipment. For air-cooled air conditioners < 65 kBtu/hr, if the actual EERee is unknown, assume the following conversion from SEER to EER: EER≈SEER/1.1.

= Actual installed

EFLH = Equivalent Full Load Hours for cooling are provided in section 4.4 HVAC End Use:

For example a 5 ton air cooled split system with a SEER of 15 at a retail strip mall in Rockford would save

ΔkWH = (60) * [(1/13) – (1/15)] * 950

= 585 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$\Delta kW_{SSP}$ = (kBtu/hr * (1/EERbase - 1/EERee)) * $CF_{SSP}$

$\Delta kW_{PJM}$ = (kBtu/hr * (1/EERbase - 1/EERee)) * $CF_{PJM}$

Where:

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)

= 91.3% [358]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)

= 47.8% [359]

For example a 5 ton air cooled split system with a SEER of 15 in Rockford would save

$\Delta kW_{SSP}$ = (60) * [(1/13) – (1/15)] * .913

= 0.562

**NATURAL GAS ENERGY SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**MEASURE CODE: CI-HVC-SPUA-V03-150601**

[358] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

[359] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

## 4.4.16 Steam Trap Replacement or Repair

### DESCRIPTION

The measure is for the repair or replacement of faulty steam traps that are allowing excess steam to escape and thereby increasing steam generation. The measure is applicable to commercial applications, commercial HVAC (low pressure steam) including multifamily buildings, low pressure industrial applications, medium pressure industrial applications, applications and high pressure industrial applications. Maximum pressure for this measure is 300 psig.

This measure was developed to be applicable to the following program types: TOS, RF. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

Customers must have leaking traps to qualify for rebates. However, if a commercial customer opts to replace all traps without inspection, rebates and the savings are discounted to take into consideration the fact that some traps are being replaced that have not yet failed.

### DEFINITION OF BASELINE EQUIPMENT

The baseline criterion is a faulty steam trap in need of replacing. No minimum leak rate is required. Any leaking or blow through trap can be repaired or replaced. If a commercial customer chooses to repair or replace all the steam traps at the facility without verification, the savings are adjusted. Savings for commercial full replacement projects are reduced by the percentage of traps found to be leaking on average from the studies listed. If an audit is performed on a commercial site, then the leaking and blowdown can be adjusted.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The life of this measure is 6 years[360]

### DEEMED MEASURE COST

| Steam System | Cost per trap[361] ($) |
|---|---|
| Commercial Dry Cleaners | 77 |
| Commercial Heating (including Multifamily), low pressure steam | 77 |
| Industrial Medium Pressure >15 psig psig < 30 psig | 180 |
| Steam Trap, Industrial Medium Pressure ≥30 <75 psig | 223 |
| Steam Trap, Industrial High Pressure ≥75 <125 psig | 276 |
| Steam Trap, Industrial High Pressure ≥125 <175 psig | 322 |
| Steam Trap, Industrial High Pressure ≥175 <250 psig | 370 |

[360]Source paper is the CLEAResult"Steam Traps Revision #1" dated August 2011. Primary studies used to prepare the source paper include Enbridge Steam Trap Survey, KW Engineering Steam Trap Survey, Enbridge Steam Saver Program 2005, Armstrong Steam Trap Survey, DOE Federal Energy Management Program Steam Trap Performance Assessment, Oak Ridge National Laboratory Steam System Survey Guide, KEMA Evaluation of PG&E's Steam Trap Program, Sept. 2007. Communication with vendors suggested a inverted bucket steam trap life typically in the range of 5 - 7 years, float and thermostatic traps 4- 6 years, float and thermodynamic disc traps of 1 - 3 years. Cost does not include installation.

[361] Ibid.

| Steam System | Cost per trap[361] ($) |
|---|---|
| Steam Trap, Industrial High Pressure ≥250 psig | 418 |
| Steam Trap, Industrial Medium Pressure ≥30 <75 psig | 223 |
| Steam Trap, Industrial High Pressure ≥75 <125 psig | 276 |
| Steam Trap, Industrial High Pressure ≥125 <175 psig | 322 |
| Steam Trap, Industrial High Pressure ≥175 <250 psig | 370 |
| Steam Trap, Industrial High Pressure ≥250 psig | 418 |

**LOADSHAPE**

N/A

**COINCIDENCE FACTOR**

N/A

**Algorithm**

**CALCULATION OF SAVINGS**

**ENERGY SAVINGS**

$$\Delta therm = S * (Hv/B) * Hours * A * L / 100{,}000$$

Where:

S = Maximum theoretical steam loss per trap

| Steam System | Avg Steam Loss[362] (lb/hr/trap) |
|---|---|
| Commercial Dry Cleaners | 38.1 |
| Commercial Heating (including Multifamily)LPS | 13.8 |
| Industrial Low Pressure, <15 psig | 13.8 |
| Industrial Medium Pressure >15 psig < 30 psig | 12.7 |
| Steam Trap, Industrial Medium Pressure ≥30 <75 psig | 19.0 |
| Steam Trap, Industrial High Pressure ≥75 <125 psig | 67.9 |
| Steam Trap, Industrial High Pressure ≥125 <175 psig | 105.8 |
| Steam Trap, Industrial High Pressure ≥175 <250 psig | 143.7 |
| Steam Trap, Industrial High Pressure ≥250 psig | 200.5 |

[362] CLEAResult"Steam Traps Revision #1" dated August 2011.

Hv = Heat of vaporization of steam

| Steam System | Heat of Vaporization[363] (Btu/lb) |
|---|---|
| Commercial Dry Cleaners | 890 |
| Commercial Heating (including Multifamily) LPS | 951 |
| Industrial Low Pressure ≤15 psig | 951 |
| Industrial Medium Pressure >15 psig < 30 psig | 945 |
| Steam Trap, Industrial Medium Pressure ≥30 <75 psig | 928 |
| Steam Trap, Industrial High Pressure ≥75 <125 psig | 894 |
| Steam Trap, Industrial High Pressure ≥125 <175 psig | 868 |
| Steam Trap, Industrial High Pressure ≥175 <250 psig | 846 |
| Steam Trap, Industrial High Pressure ≥250 psig | 820 |

B = Boiler efficiency

= custom, if unknown:

= 80.7% for steam boilers, except multifamily low-pressure [364]

= 64.8% for multifamily low-pressure steam boilers [365]

Hours = Annual operating hours of steam plant

| Steam System | Zone (where applicable) | Hours/Yr[366] |
|---|---|---|
| Commercial Dry Cleaners | n/a | 2,425 |
| Industrial Low Pressure ≤15 psig | | 7,752 |
| Industrial Medium Pressure >15 psig < 30 psig | | 7,752 |
| Steam Trap, Industrial Medium Pressure ≥30 <75 psig | | 7,752 |
| Steam Trap, Industrial High Pressure ≥75 <125 psig | | 7,752 |
| Steam Trap, Industrial High Pressure ≥125 <175 psig | | 7,752 |
| Steam Trap, Industrial High Pressure ≥175 <250 psig | | 7,752 |

[363] Heat of vaporization of steam at the inlet pressure to the steam trap. Implicit assumption that the average boiler nominal pressure where the vaporization occurs, is essentially that same pressure. Reference CLEAResult"Steam Traps Revision #1" dated August 2011.

[364] Ibid.

[365] Katrakis, J. and T.S. Zawacki. “Field-Measured Seasonal Efficiency of Intermediate-sized Low-Pressure Steam Boilers”. ASHRAE V99, pt. 2, 1993.

[366] CLEAResult"Steam Traps Revision #1" dated August 2011, which references Enbridge service territory data and kW Engineering study.

| Steam System | Zone (where applicable) | Hours/Yr[366] |
|---|---|---|
| Steam Trap, Industrial High Pressure ≥250 psig | | 7,752 |
| Industrial Medium Pressure >15 psig < 30 psig | | 7,752 |
| Steam Trap, Industrial Medium Pressure ≥30 <75 psig | | 7,752 |
| Commercial Heating (including Multifamily)LPS[367] | 1 (Rockford) | 4,272 |
| | 2 (Chicago O'Hare) | 4,029 |
| | 3 (Springfield) | 3,406 |
| | 4 (Belleville) | 2,515 |
| | 5 (Marion) | 2,546 |

A = Adjustment factor

= 50%[368]

This factor is to account for reducing the maximum theortical steam flow (S) to the average steam flow (the Enbridge factor).

L = Leaking & blow-thru

L is 1.0 when applied to the replacment of an individual leaking trap. If a number of steam traps are replaced and the system has not been audited, the leaking and blow-thru is applied to reflect the assumed percentage of steam traps that were actually leaking and need to be replaced. A custom value can be utilized if a supported by an evaluation.

| Steam System | %[369] |
|---|---|
| Custom | Custom |
| Commercial Dry Cleaners | 27% |
| Industrial Low Pressure ≤15 psig | 16% |
| Industrial Medium and High Pressure >15 psig | 16% |
| Commercial Heating (including Multifamily) LPS | 27% |

[367] Since commercial LPS reflect heating systems, Hours/yr are equivalent to HDD55 zone table

[368] Enbridge adjustment factor used as referenced in CLEAResult"Steam Traps Revision #1" dated August 2011 and DOE Federal Energy Management Program Steam Trap Performance Assessment.

[369]Dry cleaners survey data as referenced in CLEAResult"Steam Traps Revision #1" dated August 2011.

**EXAMPLE**

For example, a commercial dry cleaning facility with the default hours of operation and boiler efficiency;

ΔTherms = S * (Hv/B) * Hours * A * L

= 38.1 lbs/hr/trap * (890 Btu/lb / 80%)/100,000 * 2,425 * 50% * 27% =

138.8 therms per trap

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HVC-STRE-V03-140601**

## 4.4.17 Variable Speed Drives for HVAC Pumps and Cooling Tower Fans

### DESCRIPTION

This measure is applied to variable speed drives (VSD) which are installed on the following HVAC system applications: chilled water pump, hot water pumps. There is a separate measure for HVAC supply and return fans. All other VSD applications require custom analysis by the program administrator. The VSD will modulate the speed of the motor when it does not need to run at full load. Since the power of the motor is proportional to the cube of the speed for these types of applications, significant energy savings will result.

This measure was developed to be applicable to the following program types: TOS, RF. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The VSD is applied to a motor which does not have a VSD. The application must have a variable load and installation is to include the necessary controls. Savings are based on application of VSDs to a range of baseline load conditions including no control, inlet guide vanes, outlet guide vanes and throttling valves.

### DEFINITION OF BASELINE EQUIPMENT

The time of sale baseline is a new motor installed without a VSD or other methods of control. Retrofit baseline is an existing motor operating as is. Retrofit baselines may or may not include guide vanes, throttling valves or other methods of control. This information shall be collected from the customer.

Installations of new equipment with VSDs which are required by IECC 2012 as adopted by the State of Illinois are not eligible for incentives.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life for HVAC application is 15 years;[370] measure life for process is 10 years.[371]

### DEEMED MEASURE COST

Customer provided costs will be used when available. Default measure costs[372] are noted below for up to 20 hp motors. Custom costs must be gathered from the customer for motor sizes not listed below.

| HP | Cost |
|---|---|
| 1 -5 HP | $ 1,330 |
| 7.5 HP | $ 1,622 |
| 10 HP | $ 1,898 |
| 15 HP | $ 2,518 |
| 20 HP | $ 3,059 |

[370] Efficiency Vermont TRM 10/26/11 for HVAC VSD motors
[371] DEER 2008
[372] Ohio TRM 8/6/2010 varies by motor/fan size based on equipment costs from Granger 2008 Catalog pp 286-289, average across available voltages and models. Labor costs from RS Means Data 2008 Ohio average cost adjustment applied.

### LOADSHAPE

Loadshape C42 - VFD - Boiler feedwater pumps <10 HP

Loadshape C43 - VFD - Chilled water pumps <10 HP

Loadshape C44 - VFD Boiler circulation pumps <10 HP

Loadshape C48 - VFD Boiler draft fans <10 HP

Loadshape C49 - VFD Cooling Tower Fans <10 HP

### COINCIDENCE FACTOR

The demand savings factor (DSF) is already based upon coincident savings, and thus there is no additional coincidence factor for this characterization.

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = BHP / EFFi * Hours * ESF$$

Where:

BHP = System Brake Horsepower

(Nominal motor HP * Motor load factor)

Motors are assumed to have a load factor of 65% for calculating kW if actual values cannot be determined[373]. Custom load factor may be applied if known.

EFFi = Motor efficiency, installed. Actual motor efficiency shall be used to calculate kW. If not known a default value of 93% shall be used.[374]

Hours = Default hours are provided for HVAC applications which vary by HVAC application and building type[375]. When available, actual hours should be used.

| Building Type | Pumps and fans |
|---|---|
| College/University | 4216 |
| Grocery | 5840 |
| Heavy Industry | 3585 |
| Hotel/Motel | 6872 |

[373] Del Balso, Ryan J. "Investigation into the Reliability of Energy Efficiency/Demand Side Management Savings Estimates for Variable Frequency Drives in Commercial Applications", University of Colorado, Department of Civil, Environmental and Architectural Engineering, 2013.
[374] Ohio TRM 8/6/2010 pp207-209, Com Ed TRM June 1, 2010.
[375] ComEd Trm June 1, 2010 page 139. The Office hours is based upon occupancy from the eQuest model developed for EFLH, since it was agreed the ComEd value was too low.

| Building Type | Pumps and fans |
|---|---|
| Light Industry | 2465 |
| Medical | 6871 |
| Office | 2301 |
| Restaurant | 4654 |
| Retail/Service | 3438 |
| School(K-12) | 2203 |
| Warehouse | 3222 |
| Average = Miscellaneous | 4103 |

ESF = Energy savings factor varies by VFD application. Units are kW/HP.

| Application | ESF[376] |
|---|---|
| Hot Water Pump | 0.424 |
| Chilled Water Pump | 0.411 |
| Air Foil/backward incline | 0.354 |
| Air Foil/ backward incline inlet Guide Vanes | 0.227 |
| Forward Curved Fan, with discharge dampers | 0.179 |
| Forward Curved Inlet Guide Vanes | 0.092 |

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW =BHP/EFFi * DSF

Where:

DSF = Demand Savings Factor varies by VFD application.[377] Units are kW/HP. Values listed below are based on typical peak load for the listed application.

| Application | DSF |
|---|---|
| Hot Water Pump | 0 |
| Chilled Water Pump | 0.299 |
| Air foil / backward incline | 0.260 |
| Air Foil / backward incline inlet Guide Vanes | 0.130 |
| Forward Curved Fan, with discharge dampers | 0.136 |

[376] Ibid.
[377] Ibid

| Application | DSF |
|---|---|
| Forward Curved Inlet Guide Vanes | 0.029 |

### FOSSIL FUEL IMPACT DESCRIPTIONS AND CALCULATION

There are no expected fossil fuel impacts for this measure.

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-HVC-VSDHP-V02-150601

## 4.4.18 Small Commercial Programmable Thermostats

### DESCRIPTION

This measure characterizes the energy savings from the installation of a new Programmable Thermostat for reduced heating energy consumption through temperature set-back during unoccupied or reduced demand times. This measure is limited to small businesses, as they have smaller HVAC systems that are similar to residential HVAC systems and may be controlled by a simple manual adjustment thermostat. Mid to large sized businesses will typically have a building automation system or some other form of automated HVAC controls. Therefore, it is limited to select building types, including small office, retail – strip mall, restaurants (characterized as 1, 2 or 3 meal), small manufacturing, religious facilities, and convenience stores. This measure is only appropriate for single zone heating systems. Custom calculations are required for savings for programmable thermostats installed in multi-zone systems.

This measure was developed to be applicable to the following program types: RF, DI.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The criteria for this measure are established by replacement of a manual-only temperature control, with one that has the capability to adjust temperature setpoints according to a schedule without manual intervention.

### DEFINITION OF BASELINE EQUIPMENT

For new thermostats the baseline is a non-programmable thermostat requiring manual intervention to change temperature setpoint.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life of a programmable thermostat is assumed to be 8 years[378] based upon equipment life only[379]. For the purposes of claiming savings for a new programmable thermostat, this is reduced by a 50% persistence factor to give a final measure life of 4 years.

### DEEMED MEASURE COST

Actual material and labor costs should be used if the implementation method allows. If unknown the capital and labor cost for this measure is assumed to be $181 per thermostat[380]. For the purposes of screening and planning it should be assumed that one thermostat will serve 5 tons of Cooling Capacity at a cost of $36.20 / ton or 115kBtuh of Heating Capacity at a cost of $1.57 / kBtu.

### LOADSHAPE

N/A

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[378] Table 1, HVAC Controls, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

[379] Future evaluation is strongly encouraged to inform the persistence of savings to further refine measure life assumption. As this characterization depends heavily upon a large scale but only 2-year study of the energy impacts of programmable thermostats, the longer term impacts should be assessed.

[380] Nicor Rider 30 Business EER Program Database, Paid Rebates with Programmable Thermostat Installation Costs, Program to Date as of January 11, 2013.

### COINCIDENCE FACTOR

N/A

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS[381]

ΔkWh = [Baseline Energy Use (kWh/Ton) – Proposed Energy Use (kWh/Ton)] * Cooling Capacity (Tons)

The following equations are used to calculate baseline and proposed electric energy use.The savings is the difference between the proposed and baseline calculated usage. This approach allows the savings estimate to account for the operational attributes of the baseline as well as the proposed case, yielding a better estimate than an approach that assumes a particular baseline or proposed energy use to determine savings.

**Electric Energy Use Equations (kWh / ton)**

| Building Type | Fan Mode During Occupied Period (*Fo*) | Equation |
|---|---|---|
| **Assembly** | Continuous | ***CZ***+***Fu****(0.83****Tc***+0.83****Th***+1.67****Ws***-293.018)-0.0922****Tc*******Th***+1.291****Ws*** |
| | Intermittent | ***CZ***+***Fu****(1.911-0.12****Tc***)+***Tc****(0.00311****Ws***-0.229)+0.11****Ws*** |
| **Convenience Store** | Continuous | ***CZ***+***Fu****(-28.629****Tc***-11.69****Th***+19.118****Ws***-2935.12)+0.909****Ws*** |
| | Intermittent | ***CZ***+***Tc****(0.0863****Ws***-12.688)+***Th****(0.043****Ws***-6.38)+1.669****Ws*** |
| **Office – Low Rise** | Continuous | ***CZ***+***Fu****(7.082****Tc***-41.199****Th***+18.734****Ws***-3288.55)+***Tc****(0.205****Ws***-34.929) |
| | Intermittent | ***CZ***+***Tc****(0.0806****Ws***-8.984)+***Th****(0.0864****Ws***-9.558)+1.178****Ws*** |
| **Religious** | Continuous | ***CZ***+***Fu****(-1.579****Tc***-18.14****Th***+15.01****Ws***-2417.74)+***Tc****(0.177****Ws***-26.412) |
| | Intermittent | ***CZ***+***Fu****(0.266****Tc***-2.067)+***Tc****(0.0295****Ws***-4.502)+***Th****(0.0517****Ws***-8.251)+0.735****Ws*** |
| **Restaurant – Fast Food** | Continuous | ***CZ***+***Fu****(0.678****Tc***+0.257****Th***+2.88****Ws***-494.006)+***Tc****(0.0231****Ws***-4.074)+***Th****(0.00936****Ws***-1.655)+0.918****Ws*** |
| | Intermittent | ***CZ***+***Fu****(0.377****Tc***+0.124****Th***+0.13****Ws***-24.893)+***Tc****(-0.0143****Th***+0.0166****Ws***-2.691)+0.898****Ws*** |
| **Restaurant – Full Service** | Continuous | ***CZ***+***Fu****(-8.41****Th***+11.766****Ws***-1910.81)+***Tc****(0.282****Ws***-43.851) |
| | Intermittent | ***CZ***+0.123****Fu*******Tc***+***Tc****(0.0561****Ws***-8.237)+***Th****(0.0219****Ws***-3.284)+1.038****Ws*** |
| **Retail – Department Store** | Continuous | ***CZ***+***Fu****(-1.475****Th***+0.755****Ws***-114.373)+***Th****(0.151****Ws***-24.016)+1.612****Ws*** |
| | Intermittent | ***CZ***+***Tc****(0.0173****Ws***-1.912)+***Th****(0.0249****Ws***-3.29)+0.511****Ws*** |
| **Retail – Strip** | Continuous | ***CZ***+***Fu****(1.077****Tc***-10.697****Th***+6.91****Ws***-1117.18)+***Tc****(0.0583****Ws***-7.54)+1.231****Ws*** |

[381] Savings equations and factors determined by regression of results of a series of eQuest simulations. See Programmable T-Stat Work Paper_PECI_FinalDraft_140730_Redline.docx for details.

| Building Type | Fan Mode During Occupied Period (*Fo*) | Equation |
|---|---|---|
| **Mall** | Intermittent | ***CZ***+0.0894****Fu*****Tc***+***Th****(-0.0142****Tc***+0.04****Ws***-5.278)+0.884****Ws*** |

Where:

CZ = Climate Zone Coefficient

=Depends on Building Type and Fan Mode During Occupied Period (see table below)

Tc = Degrees of Cooling Setback °F

= Must be between 0-15°F

Th = Degrees of Heating Setback °F

=Must be between 0-15°F

Fo = Fan Mode During Occupied Period (Note: Commercial mechanical code requires continuous fan operation during occupied periods to meet ventilation requirements.)

= Continuous for occupied fan that runs continuously (e.g. Fan Mode Set to 'On')

= Intermittent for occupied fan that runs intermittently (e.g. Fan Mode Set to 'Auto')

Fu = Fan Mode During Unoccupied Period

= 0 for unoccupied fan that runs continuously (e.g. Fan Mode Set to 'On')

= 1 for unoccupied fan that runs intermittently (e.g. Fan Mode Set to 'Auto')

Ws = Weekly Hours thermostat is in Occupied mode

= Minimum values depends on Building Type (see table below), maximum value of 168 (24/7)

(e.g.: Weekly occupancy schedule of Mon-Sat 8AM-5PM, Sun 9AM-2PM, Ws = 59)

**Electric Energy Use Climate Zone Coefficients and Minimum Weekly Hours Occupied**

| Building Type | Fan Mode During Occupied Period (*Fo*) | Climate Zone Coefficient (*CZ*)[382] | | | | | Minimum *Ws* |
|---|---|---|---|---|---|---|---|
| | | 1 | 2 | 3 | 4 | 5 | |
| **Assembly** | Continuous | 911.366 | 928.924 | 1152.83 | 1208.999 | 1210.173 | 98 |
| | Intermittent | 735.752 | 762.831 | 966.562 | 998.927 | 1028.906 | |
| **Convenience Store** | Continuous | 4817.094 | 4832.784 | 5139.133 | 5182.161 | 5208.608 | 108 |
| | Intermittent | 1478.133 | 1514.568 | 1784.384 | 1843.463 | 1930.47 | |
| **Office - Low Rise** | Continuous | 5047.662 | 5039.592 | 5187.924 | 5217.672 | 5177.449 | 55 |
| | Intermittent | 825.072 | 808.965 | 946.571 | 979.421 | 945.418 | |

[382] Climate Zones Refrenced in Section 3.7, Table 3.6

| Building Type | Fan Mode During Occupied Period (*Fo*) | Climate Zone Coefficient (*CZ*)382 | | | | | Minimum *Ws* |
|---|---|---|---|---|---|---|---|
| | | 1 | 2 | 3 | 4 | 5 | |
| **Religious Facility** | Continuous | 4197.117 | 4172.858 | 4380.025 | 4370.008 | 4356.054 | 133 |
| | Intermittent | 632.404 | 603.395 | 678.294 | 664.717 | 616.853 | |
| **Restaurant – Fast Food** | Continuous | 1342.988 | 1378.661 | 1664.018 | 1714.201 | 1727.841 | 108 |
| | Intermittent | 993.764 | 1039.643 | 1307.8 | 1340.544 | 1389.791 | |
| **Restaurant – Full Service** | Continuous | 4070.35 | 4094.742 | 4428.966 | 4501.829 | 4522.522 | 117 |
| | Intermittent | 1472.014 | 1516.05 | 1856.108 | 1938.441 | 2056.45 | |
| **Retail – Department Store** | Continuous | 1510.201 | 1496.47 | 1706.105 | 1716.128 | 1688.464 | 93 |
| | Intermittent | 701.27 | 702.129 | 847.735 | 875.12 | 881.677 | |
| **Retail – Strip Mall** | Continuous | 1926.294 | 1930.137 | 2156.856 | 2174.435 | 2165.03 | 93 |
| | Intermittent | 656.479 | 673.257 | 835.906 | 850.322 | 869.921 | |

EXAMPLE
A low rise office in Rockford (Climate Zone 1) is occupied Mon-Fri 7AM-6PM and has a 10 ton DX RTU controlled by a manual thermostat. The fan runs continuously during the occupied hours and building staff do not manually change the fan mode, cooling or heating setpoints during unoccupied periods.

A programmable thermostat is installed by a contractor who sets the occupied schedule to Mon-Fri 7AM-6PM with a 10°F cooling and heating unoccupied temperature setback. The contractor also programs the fan to operate continuously during the occupied periods and to intermittent "auto" during the unoccupied periods.

ΔkWh = [Baseline Energy Use (kWh/Ton) – Proposed Energy Use(kWh/Ton)] * Cooling Capacity (Tons)

Baseline Energy Use (kWh/Ton) = Equation for Office Low Rise, *Fo*=Continuous

= ***CZ***+***Fu****(7.082***Tc**-41.199***Th**+18.734***Ws**-3288.55)+***Tc****(0.205***Ws**-34.929)

= ***5047.662***+***0****(7.082***0**-41.199***0**+18.734***168**-3288.55)+***0****(0.205***168**-34.929)

= 5,047.662 kWh/Ton

Proposed Energy Use (kWh/Ton) = Equation for Office Low Rise, *Fo*=Continuous

= ***CZ***+***Fu****(7.082***Tc**-41.199***Th**+18.734***Ws**-3288.55)+***Tc****(0.205***Ws**-34.929)

= ***5047.662***+***1****(7.082***10**-41.199***10**+18.734***55**-3288.55)+***10*(***0.205***55**-34.929)

= 2,211.722 kWh/Ton

ΔkWh = [5,047.622 (kWh/Ton) – 2,211.722 (kWh/Ton)] * 10 Tons

= 2,835.89 kWh/Ton * 10 Tons

= 28,358.9 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

N/A

**NATURAL GAS ENERGY SAVINGS**

ΔTherms = [Baseline Energy Use (Therms/kBtuh) – Proposed Energy Use(Therms/kBtuh)] * Output Heating Capacity (kBtuh)

The following equations are used to calculate baseline and proposed natural gas energy use. The savings is the difference between the proposed and baseline calculated usage. This approach allows the savings estimate to account for the operational attributes of the baseline as well as the proposed case, yielding a better estimate than an approach that assumes a particular baseline or proposed energy use to determine savings.

**Natural Gas Energy Use Equations (therms / kbtu output)**

| Building Type | Fan Mode During Occupied Period (*Fo*) | Equation |
|---|---|---|
| **Assembly** | Continuous | ***CZ***+***Fu****(0.232****Th***+0.0984****Ws***-18.79)+***Th****(0.00271****Ws***-0.535)+0.0142****Ws*** |
| | Intermittent | ***CZ***+***Fu****(0.00405****Th***+0.000519****Ws***-0.11)+***Th****(0.0000689****Ws***-0.0118)+0.0022****Ws*** |
| **Convenience Store** | Continuous | ***CZ***+***Fu****(0.00545****Th***-0.00251****Ws***+0.416)+***Th****(0.000123****Ws***-0.0204)+0.00183****Ws*** |
| | Intermittent | ***CZ***+***Fu****(0.00231****Th***-0.0349)+***Th****(0.000309****Ws***-0.0494)+0.00266****Ws*** |
| **Office – Low Rise** | Continuous | ***CZ***+***Fu****(0.0205****Th***+0.364)+***Th****(0.00046****Ws***-0.0554)+0.00169****Ws*** |
| | Intermittent | ***CZ***+***Fu****(0.00745****Th***-0.142)+***Th****(0.00077****Ws***-0.111)+0.00199****Ws*** |
| **Religious** | Continuous | ***CZ***+0.00791****Fu*******Th***+***Th****(0.00096****Ws***-0.167)+0.00184****Ws*** |
| | Intermittent | ***CZ***+***Fu****(0.00143****Th***-0.0309)+***Th****(0.0008****Ws***-0.134)+0.00219****Ws*** |
| **Restaurant – Fast Food** | Continuous | ***CZ***+***Fu****(0.0431****Th***+0.0424****Ws***-7.517)+***Th****(0.00113****Ws***-0.213)+0.0119****Ws*** |
| | Intermittent | ***CZ***+***Fu****(0.0125****Th***+0.0036****Ws***-0.71)+***Th****(0.000329****Ws***-0.0615)+0.00738****Ws*** |
| **Restaurant –Full Service** | Continuous | ***CZ***+***Fu****(0.00445****Ws***-0.535)+***Th****(0.000679****Ws***-0.1)+0.00218****Ws*** |
| | Intermittent | ***CZ***+***Fu****(0.00144****Th***+0.000262****Ws***-0.0553)+***Th****(0.00018****Ws***-0.0299)+0.00166****Ws*** |
| **Retail – Department Store** | Continuous | ***CZ***+0.00203****Fu*******Th***+***Th****(0.000591****Ws***-0.0812)+0.00194****Ws*** |
| | Intermittent | ***CZ***+***Th****(0.000406****Ws***-0.0611)+0.00228****Ws*** |
| **Retail – Strip Mall** | Continuous | ***CZ***+***Fu****(0.00998****Th***+0.00207****Ws***-0.206)+***Th****(0.000665****Ws***-0.101)+0.00292****Ws*** |
| | Intermittent | ***CZ***+***Fu****(0.00383****Th***-0.0656)+***Th****(0.000575****Ws***-0.0912)+0.00249****Ws*** |

Where:

CZ = Climate Zone Coefficient

= Depends on Building Type and Fan Mode During Occupied Period (see table below)

Th = Degrees of Heating Setback °F

= Must be between 0-15°F

Fo = Fan Mode During Occupied Period (Note: Commercial mechanical code requires continuous fan operation during occupied periods to meet ventilation requirements.)

= Continuous for occupied fan that runs continuously (e.g. Fan Mode Set to 'On')

= Intermittent for occupied fan that runs intermittently (e.g. Fan Mode Set to 'Auto')

Fu = Fan Mode During Unoccupied Period

= 0 for unoccupied fan that runs continuously (e.g. Fan Mode Set to 'On')

= 1 for unoccupied fan that runs intermittently (e.g. Fan Mode Set to 'Auto')

Ws = Weekly Hours thermostat is in Occupied mode

= Minimum values depends on Building Type (see table below), maximum value of 168 (24/7)

(e.g.: Weekly occupancy schedule of Mon-Sat 8AM-5PM, Sun 9AM-2PM, Ws = 59)

**Natural Gas Energy Use Climate Zone Coefficients and Minimum Weekly Hours Occupied**

| | Fan Mode During Occupied Period (*Fo*) | Climate Zone Coefficient (*CZ*) | | | | | |
|---|---|---|---|---|---|---|---|
| **Building Type** | | **1** | **2** | **3** | **4** | **5** | **Minimum *Ws*** |
| **Assembly** | Continuous | 19.872 | 17.83 | 15.828 | 15.282 | 13.482 | 98 |
| | Intermittent | 0.237 | 0.0989 | 0.0267 | -0.0131 | -0.0871 | |
| **Convenience Store** | Continuous | 1.493 | 1.081 | 0.782 | 0.544 | 0.114 | 108 |
| | Intermittent | 1.128 | 0.854 | 0.619 | 0.437 | 0.0854 | |
| **Office - Low Rise** | Continuous | 1.718 | 1.317 | 0.971 | 0.739 | 0.319 | 55 |
| | Intermittent | 3.447 | 3.022 | 2.503 | 2.251 | 1.646 | |
| **Religious Facility** | Continuous | 6.294 | 5.55 | 4.678 | 4.202 | 3.122 | 133 |
| | Intermittent | 5.914 | 5.368 | 4.557 | 4.137 | 3.246 | |
| **Restaurant – Fast Food** | Continuous | 8.383 | 7.211 | 6.034 | 5.767 | 4.71 | 108 |
| | Intermittent | 1.227 | 0.636 | 0.302 | 0.102 | -0.262 | |
| **Restaurant – Full Service** | Continuous | 5.247 | 4.484 | 3.753 | 3.465 | 2.627 | 117 |
| | Intermittent | 0.951 | 0.704 | 0.51 | 0.381 | 0.0746 | |
| **Retail – Department Store** | Continuous | 4.385 | 3.854 | 3.192 | 2.784 | 1.858 | 93 |
| | Intermittent | 3.061 | 2.672 | 2.182 | 1.829 | 1.008 | |
| **Retail – Strip Mall** | Continuous | 3.917 | 3.394 | 2.728 | 2.394 | 1.617 | 93 |
| | Intermittent | 2.659 | 2.292 | 1.811 | 1.543 | 0.909 | |

## WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

## DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

## MEASURE CODE: CI-HVC-PROG-V02-150601

## 4.4.19 Demand Controlled Ventilation

### DESCRIPTION

Demand control ventilation (DCV) adjusts outside ventilation air based on the number of occupants and the ventilation demands that those occupants create. DCV is part of a building's ventilation system control strategy. It may include hardware, software, and controls as an integral part of a building's ventilation design. Active control of the ventilation system provides the opportunity to reduce heating and cooling energy use.

The primary component is a control sensor to communicate either directly with the economizer or with a central computer. The component is most typically a carbon dioxide ($CO_2$) sensor, occupancy sensor, or turnstile counter. This measure is applicable to multiple building types, and savings are classified by the specific building types defined in the Illinois TRM. This measure is modeled to assume night time set backs are in operation and minimum outside air is being used when the building is unoccupied. Systems that have static louvers or that are open at night will likely have greater savings by using the custom program.

This measure was developed to be applicable to the following program types: RF. If applied to other program types, the measure savings should be verified

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment condition is defined by new $CO_2$ sensors installed on return air systems where no other sensors were previously installed. For heating savings, this measure does not apply to any system with terminal reheat (constant volume or variable air volume). For terminal reheat system a custom savings calculation should be used.

### DEFINITION OF BASELINE EQUIPMENT

The base case for this measure is a space with no demand control capability. The current code minimum for outside air (OA) is 17 CFM per occupant (ASHRAE 62.1) which is the value assumed in this measure.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The deemed measure life is 10 years and based on CO2 sensor estimated life. [383]

### DEEMED MEASURE COST

The deemed measure cost is assumed to be the full cost of installation of a DCV retrofit including sensor cost ($500) and installation ($1000 labor) for a total of $1500[384].

### LOADSHAPE

Commercial ventilation C23

### COINCIDENCE FACTOR

N/A

[383] During the course of conversations with vendors and Building Automation System (BAS) contractors, it was determined that sensors have to be functional for up to 10 years. It is recommended that they are part of a normal preventive maintenance program in which calibration is an important part of extending useful life. Although they are not subject to mechanical failure, they do fall out of tolerance over time.

[384] Discussion with vendors

**Algorithm**

**CALCULATION OF ENERGY SAVINGS**

**ELECTRIC ENERGY SAVINGS**

ΔkWh = Condition Space/1000 * Savings_Factor

Where:

Conditioned Space = actual square footage of conditioned space controlled by sensor

Elec_Savings_Factor = value in table below based on building type and weather zone[385]

| Building Type | Elect_Savings_Factor (kWh/1000 sq ft) | | | | |
|---|---|---|---|---|---|
| | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| Office - Low-rise | 454 | 456 | 460 | 456 | 462 |
| Office - Mid-rise | 430 | 431 | 432 | 428 | 433 |
| Office- High-rise | 448 | 450 | 452 | 449 | 454 |
| Religious Building | 493 | 509 | 573 | 584 | 605 |
| Restaurant | 505 | 515 | 553 | 569 | 581 |
| Retail - Department Store | 620 | 625 | 630 | 638 | 642 |
| Retail - Strip Mall | 380 | 376 | 356 | 406 | 407 |
| Convenience Store | 602 | 603 | 610 | 612 | 614 |
| Elementary School | 317 | 327 | 352 | 352 | 363 |
| High School | 305 | 316 | 340 | 340 | 352 |
| College/ University | 392 | 410 | 434 | 449 | 462 |
| Healthcare Clinic | 353 | 358 | 379 | 383 | 389 |
| Lodging | 576 | 578 | 586 | 588 | 591 |
| Manufacturing | 481 | 482 | 482 | 477 | 482 |
| Special Assembly Auditorium | 410 | 427 | 479 | 494 | 514 |
| De-fault | 451 | 458 | 475 | 482 | 490 |

[385] The electric energy savings was calculated using TMY3 weather data and methodology consistent with ASHRAE standards. Savings are calculated on an annual basis for each given temperature zone in Illinois. Energy savings for DCV were developed utilizing standards, inputs and approaches as set forth by ASHRAE 62.1and 90.1, respectively. Building input parameters like square footage, equipment efficiencies and occupancy match those used in the EFLH calculations. Reference calculation found in Demand Control Ventilation 12-30-13.xls.

For example: 7,500 SqFt of low-rise office space in Chicago.

ΔkWh = 7,500 SqFt /1000 SqFt *456 kWh

= 3,420 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

NA

**NATURAL GAS SAVINGS**

Δtherms = Condition Space/1000 * Therm_Savings_Factor

Where:

Conditioned Space = actual square footage of conditioned space controlled by sensor

Therm _Savings_Factor = value in table below based on building type and weather zone[386]

| Building Type | Therm_Savings_Factor (Therm/1000 sq ft) | | | | |
|---|---|---|---|---|---|
| | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| Office - Low-rise | 30 | 26 | 23 | 22 | 19 |
| Office - Mid-rise | 20 | 17 | 15 | 15 | 13 |
| Office- High-rise | 27 | 23 | 21 | 20 | 17 |
| Religious Building | 191 | 169 | 150 | 143 | 128 |
| Restaurant | 135 | 122 | 106 | 104 | 91 |
| Retail - Department Store | 47 | 42 | 37 | 36 | 32 |
| Retail - Strip Mall | 31 | 27 | 25 | 24 | 21 |
| Convenience Store | 23 | 21 | 18 | 17 | 15 |
| Elementary School | 83 | 73 | 64 | 60 | 53 |
| High School | 81 | 71 | 63 | 59 | 52 |
| College/ University | 161 | 141 | 124 | 120 | 101 |
| Healthcare Clinic | 57 | 50 | 44 | 42 | 38 |
| Lodging | 26 | 23 | 20 | 20 | 20 |
| Manufacturing | 21 | 19 | 16 | 15 | 14 |

[386] The natural gas energy savings was calculated using TMY3 weather data and methodology consistent with ASHRAE standards. Savings are calculated on an annual basis for each given temperature zone in Illinois. Energy savings for DCV were developed utilizing standards, inputs and approaches as set forth by ASHRAE 62.1 and 90.1, respectively. Building input parameters like square footage, equipment efficiencies and occupancy match those used in the EFLH calculations. Reference calculation found in Demand Control Ventilation 12-30-13.xls.

| Building Type | Therm_Savings_Factor (Therm/1000 sq ft) | | | | |
|---|---|---|---|---|---|
| | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| Special Assembly Auditorium | 225 | 198 | 179 | 175 | 154 |
| De-fault | 77 | 68 | 60 | 58 | 51 |

For example: 7500 SqFt of low-rise office space in Chicago.

ΔTherms = 7,500 SqFt *26 Therm/1000 SqFt

= 195 Therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HVC-DCV-V02-140601**

## 4.4.20 High Turndown Burner for Space Heating Boilers

### DESCRIPTION

This measure is for a non-residential boilers equipped with linkageless controls providing space heating with burners having a turndown less than 6:1.[387] Turndown is the ratio of the high firing rate to the low firing rate. When boilers are subjected to loads below the low firing rate, the boiler must cycle on/off to meet the load requirements. A higher turndown ratio reduces burner startups, provides better load control, saves wear-and-tear on the burner, and reduces purge-air requirements, all of these benefits result in better overall efficiency.

This measure was developed to be applicable to the following program types: NC, TOS, RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify the boiler linkageless burner must operate with a turndown greater than or equal to 10:1 and be subjected to loads less than or equal to 30%[388] of the full fire input MBH for greater than 60%[389] of the operating hours.

### DEFINITION OF BASELINE EQUIPMENT

The baseline boiler utilizes a linkageless burner with a turndown ration of 6:1 or less and is used primarily for space heating. Redundant boilers do not qualify.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 21 years.[390]

### DEEMED MEASURE COST

The deemed installed measure cost including labor is approximately $2.53/MBtu/hr.[391]

### DEEMED O&M COST ADJUSTMENTS

N/A

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

[387] The standard turndown ratio for boilers is 6:1. Understanding Fuel Savings in the Boiler Room, ASHRAE Journal, David Eoff, December, 2008 p 38

[388] Focus on Energy Evaluation, Business Programs: Deemed Savings Manual V1.0, March 22, 2010. This factor implies that boilers are 30% oversized on average.

[389] FES Analysis of bin hours based upon a 30% oversizing factor.

[390] "Burner," Obtained from a nation-wide survey conducted by ASHRAE TC 1.8 (Akalin 1978). Data changed by TC 1.8 in 1986.

[391] FES review of PY2/PY3 costs for custom People's and North Shore high turndown burner projects. See High Turndown Costs.xlsx for details.

**Algorithm**

**CALCULATION OF SAVINGS**

**ELECTRIC ENERGY SAVINGS**

N/A

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

N/A

**NATURAL GAS SAVINGS**

Δtherms = Ngi * SF * EFLH / 100

Where:

Ngi = Boiler gas input size (kBtu/hr) = custom

SF = Savings Factor = Percentage of energy loss per hour

= (∑ ((EL_base – EL_eff) * H_cycling)) / H)*100

Where:

EL_base = Base Boiler Percentage of energy loss due to cycling at % of Base Boiler Load where BL_base ≤ TDR_base

= 0.003 * $(Cycles_base)^2$ – 0.001 * Cycles_base [392]

Where:

Cycles_base = Number of Cycles/hour of base boiler

= TDR_base / BL

Where:

BL = % of full boiler load at bin hours being evaluated. This is assumed to be a straight line based on 0% load at the building balance point (assumed to be 55F), and full load corrected for the oversizing (OSF) at the lowest temperature bin of -10 to -5F.

OSF = Oversizing Factor = 1.3[393] or custom

TDR_base = Turndown ratio = 0.33[394] or custom

EL_eff = Efficient Boiler Percentage of energy loss due to cycling at % of Efficient Boiler Load

= 0.003 * $(Cycles_eff)^2$ – 0.001 * Cycles_eff

[392] Release 3.0 Operations & Maintenance Best Practices A Guide to Achieving Operational Efficiency, August 2010, Federal Energy Management Program, US Department of Energy. The equation was determined by plotting the values in Table 9.2.1 – Boiler Cycling Energy Loss.

[393] PA Consulting, KEMA, Focus on Energy Evaluation, Business Programs: Deemed Savings Manual V1.0, March 22, 2010, Page 4-12.

[394] Ibid.

Where:

Cycles_eff = Number of Cycles/hour

= TDR_eff / BL

Where:

TDR_eff = Turndown ratio = 0.10[395] or custom

H_cycling = Hours base boiler is cycling at % of base boiler load

= see table below or custom

H = Total Number of Hours in Heating Season

= 4,946 or custom

100 = convert to a percentage

SF = 69.1 / 4946 *100 = 1.4% or custom (see table below for summary of values)

| Temperature | H_cycling | BL | EL_base | EL_eff | (EL_base-EL_eff)* Hours |
|---|---|---|---|---|---|
| 50 to 55 | 601 | 6.0% | 8.5% | 0.7% | 47.2 |
| 45 to 50 | 603 | 12.0% | 2.0% | 0.0% | 12.0 |
| 40 to 45 | 455 | 18.0% | 0.8% | 0.0% | 3.8 |
| 35 to 40 | 925 | 24.0% | 0.4% | 0.0% | 4.0 |
| 30 to 35 | 814 | 30.0% | 0.3% | 0.0% | 2.1 |
| | | | | Total | 69.1 |

EFLH = Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use.

100 = convert kBtu to therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HVAC-HTBC-V04-140601**

[395] 10:1 ratio used to qualify for efficient equipment.

## 4.4.21 Linkageless Boiler Controls for Space Heating

### DESCRIPTION

This measure is for a non-residential boiler providing space heating and currently having single point positioning combustion control. In single-point positioning control, the fuel valve is linked to the combustion air damper via a jackshaft mechanism to maintain correspondence between fuel and combustion air input. Most boilers with single point positioning control do not maintain low excess air levels over their entire firing range. Generally these boilers are calibrated at high fire, but due to the non-linearity required for efficient combustion, excess air levels tend to dramatically increase as the firing rate decreases. Boiler efficiency drops as the excess air levels are increased.

This measure was developed to be applicable to the following program types: TOS, RF.
If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify the boiler burner must have a linkageless control system allowing the combustion air damper position to be adjusted and set for optimal efficiency at several firing rates throughout the burner's firing range. This requires the fuel valve and combustion air damper to each be powered by a separate actuator. An alternative to the combustion air damper is a Variable Speed Drive on the combustion air fan.

### DEFINITION OF BASELINE EQUIPMENT

The baseline boiler utilizes single point positioning for the burner combustion control.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 16 years.[396]

### DEEMED MEASURE COST

The deemed measure cost is estimated at $2.50/MBtu/hr burner input.[397]

### DEEMED O&M COST ADJUSTMENTS

N/A

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

[396] Total number of hours for heating with a base temperature of 55°F for Chicago, IL as noted by National Climate Data Center

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

When a Variable Speed Drive is incorporated, electrical savings are calculated according to the Variable Speed Drive measure.

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS SAVINGS

$$\Delta therms = Ngi * SF * EFLH / 100$$

Where:

Ngi = Boiler gas input size (kBtu/hr) = custom

SF = Savings factor

Note: Savings factor is the percentage increase in efficiency as a result of the addition of linkageless burner controls. At an average boiler load of 35%, single point controls are assumed to have excess air of 91%, while linkageless controls are assumed to have 34% excess air.[398] The difference between controls types is 57% at this average operating condition. A 15% reduction in excess air is approximately a 1% increase in efficiency.[399] Therefore the nominal combustion efficiency increase is 57 / 15 * 1% = 3.8%.

= 3.8%

EFLH = Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use

100 = convert kBtu to therms

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-HVC-LBC-V04-140601

[398] Available and Emerging Technologies for Reducing Greenhouse Gas Emissions from Industrial, Commercial, and Institutional Boilers, Prepared by the Sector Policies and Programs Division Office of Air Quality Planning and Standards U.S. Environmental Protection Agency Research Triangle Park, North Carolina 27711, October 2010, Table 1. ICI Boilers – Summary of Greenhouse Gas Emission Reduction Measures, pg. 8

[399] Department of Energy (DOE). January 2012, Steam Tip Sheet #4, Improve Your Boiler's Combustion Efficiency. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. This value was determined as an appropriate average over the stack temperatures and excess air levels indicated.

## 4.4.22 Oxygen Trim Controls for Space Heating Boilers

### DESCRIPTION

This measure is for a non-residential boiler providing space heating without oxygen trim combustion controls. Oxygen trim controls limit the amount of excess oxygen provided to the burner for combustion. This oxygen level is dependent upon the amount of air provided. Oxygen trim control converts parallel positioning, linkageless controls, into a closed-loop control configuration with the addition of an exhaust gas analyzer and PID controller. Boilers with oxygen trim controls can maintain a predetermined excess air rate (generally 15% to 30% excess air) over the entire burner firing rate. Boilers without these controls typically have excess air rates around 30% over the entire firing rate. Boiler efficiency drops as the excess air levels are increased.

This measure was developed to be applicable to the following program types: NC, TOS, RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify the boiler burner must have an oxygen control system allowing the combustion air to be adjusted to maintain a predetermined excess oxygen level in the flue exhaust at all firing rates throughout the burner's firing range. This requires an oxygen sensor in the flue exhaust and linkageless fuel valve and combustion air controls.

### DEFINITION OF BASELINE EQUIPMENT

The baseline boiler utilizes single point positioning for the burner combustion control.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for the O2 Trim controls is 18 years.[400]

### DEEMED MEASURE COST

The deemed measure cost is approximately $23,250.[401]

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

[400] State of Wisconsin Public Service Commission of Wisconsin Focus on Energy Evaluation Business Programs: Measure Life Study Final Report: August 25, 2009, Table 1-2. Recommended Measure Life by WISeerts Group Description, pg. 1-4.
[401] CODES AND STANDARDS ENHANCEMENT INITIATIVE (CASE) PROCESS BOILERS, 2013 California Building Energy Efficiency Standards, California Utilities Statewide Codes and Standards Team, October 2011, pg. 22

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS ENERGY SAVINGS

Δtherms = Ngi * SF * EFLH / 100

Where:

Ngi = Boiler gas input size (kBtu/hr)

= Custom

SF = Savings factor

Note: Savings factor is the percentage reduction in gas consumption as a result of the addition of O2 trim controls. Linkageless controls have an excess air rate of 28% over the entire firing range.[402] O2 trim controls have an excess air rate of 15%.[403] The average difference is 13%. A 15% reduction in excess air is approximately a 1% increase in efficiency.[404] Therefore the nominal combustion efficiency increase is 13 / 15 * 1% = 0.87%.

= 0.87%

EFLH = Default Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use. When available, actual hours should be used.

100 = convert kBtu to therms

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

The deemed annual Operations and Maintenance cost is $800.[405]

### MEASURE CODE: CI-HVC-O2TC-V01-140601

[402] Department of Energy (DOE). 2009. Energy Matters newsletter. Fall 2009- Vol. 1, Iss. 1. Washington, DC: U.S. Department of Energy, Office of Energy Efficiency and Renewable Energy, Industrial Technologies Program.
[403] Ibid

[405] Department of Energy (DOE). January 2012, Steam Tip Sheet #4, Improving Your Boiler's Combustion Efficiency. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. This value was determined as an appropriate average over the stack temperatures and excess air levels indicated.

## 4.4.23 Shut Off Damper for Space Heating Boilers or Furnaces

### DESCRIPTION

This measure is for non-residential atmospheric boilers or furnaces providing space heating without a shut off damper. When appliances are on standby mode warm room air is drawn through the stack via the draft hood or dilution air inlet at a rate proportional to the stack height, diameter and outdoor temperature. More air is drawn through the vent immediately after the appliance shuts off and the flue is still hot. Installation of a new shut off damper can prevent heat from being drawn up the warm vent and reducing the amount of air that passes through the furnace or boiler heat exchanger. This reduction in air can slightly increase overall operating efficiency by reducing the time needed to achieve steady-state operating conditions.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify the space heating boiler or furnace must have a new electrically or thermally activated shut off damper installed on either the exhaust flue or combustion air intake. Barometric dampers do not qualify. The damper actuation shall be interlocked with the firing controls.

### DEFINITION OF BASELINE EQUIPMENT

The baseline boiler or furnace incorporates no shut off damper on the combustion air intake or flue exhaust.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for the shut off damper is 15 years.[406]

### DEEMED MEASURE COST

The deemed measure cost for this approximately $1,500.[407]

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

[406] State of Wisconsin Public Service Commission of Wisconsin Focus on Energy Evaluation Business Programs: Measure Life Study Final Report: August 25, 2009, Table 1-2. Recommended Measure Life by WISeerts Group Description, pg. 1-4.
[407] CODES AND STANDARDS ENHANCEMENT INITIATIVE (CASE) PROCESS BOILERS, 2013 California Building Energy Efficiency Standards, California Utilities Statewide Codes and Standards Team, October 2011, pg. 22

**Algorithm**

### CALCULATION OF SAVINGS

#### ELECTRIC ENERGY SAVINGS

N/A

#### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

#### NATURAL GAS ENERGY SAVINGS

Δtherms = Ngi * SF * EFLH / 100

Where:

Ngi = Boiler gas input size (kBtu/hr)

= Custom

SF = Savings factor

= 1%[408]

Note: The savings factor assumes the boiler or furnace is located in an unconditioned space. The savings factor can be higher for those units located within conditioned space.

EFLH = Default Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use. When available, actual hours should be used.

100 = convert kBtu to therms

#### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

#### DEEMED O&M COST ADJUSTMENT CALCULATION

The deemed annual Operations and Maintenance cost is $112.[409]

### MEASURE CODE: CI-HVC-SODP-V01-140601

[408] Based on internet review of savings potential;
"Up to 4%": Use of Automatic Vent Dampers for New and Existing Boilers and Furnaces, Energy Innovators Initiative Technical Fact Sheet, Office of Energy Efficiency, Canada, 2002
"Up to 1%": Page 9, The Carbon Trust, "Steam and high temperature hot water boilers" http://www.carbontrust.com/media/13332/ctv052_steam_and_high_temperature_hot_water_boilers.pdf,
"1 - 2%": Page 2, Sustainable Energy Authority of Ireland "Steam Systems Technical Guide", http://www.seai.ie/Your_Business/Technology/Buildings/Steam_Systems_Technical_Guide.pdf.

[409] CODES AND STANDARDS ENHANCEMENT INITIATIVE (CASE) PROCESS BOILERS, 2013 California Building Energy Efficiency Standards, California Utilities Statewide Codes and Standards Team, October 2011, pg. 22

## 4.4.24 Small Pipe Insulation

### DESCRIPTION

This measure provides rebates for adding insulation to bare pipes with inner diameters of ½" and ¾". Insulation must be at least one inch thick. Since new construction projects are required by code to have pipe insulation, this measure is only for retrofits of existing facilities. This covers bare straight pipe as well as all fittings.

Default savings are provided on a per linear foot basis. It is assumed that the majority of pipes less than one inch in commercial facilities are used for domestic hot water. However, this measure can cover hydronic heating systems as well as low and high pressure steam systems.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient case is a ½"or ¾" diameter pipe with at least one inch of insulation. Insulation must be protected from damage which includes moisture, sunlight, equipment maintenance and wind. Outdoor pipes should have a weather protective jacket. Insulation must be continuous over straight pipe, elbows and tees.

### DEFINITION OF BASELINE EQUIPMENT

The base case for savings estimates is a bare hot water or steam pipe with a fluid temperature of 105 degrees Fahrenheit or greater. Current new construction code requires insulation amounts similar to this measure though this base case is commonly found in older existing buildings.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 15 years.[410]

### DEEMED MEASURE COST

The incremental measure cost for insulation is the full cost of adding insulation to the pipe. Actual installation costs should be used for the measure cost. For planning purposes, the following costs can be used to estimate the full cost of materials and labor.[411]

| Insulation Thickness | ¾" pipe | ½" pipe |
|---|---|---|
| 1" | $4.45 | $4.15 |

### LOADSHAPE

N/A

[410] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf
[411] A market survey was performed to determine these costs.

### COINCIDENCE FACTOR

N/A

**Algorithm**

### CALCULATION OF ENERGY SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS SAVINGS

Δtherms per foot[412] = $[((Q_{base} – Q_{eff}) * EFLH) / (100,000 * \eta Boiler)] * TRF$

= [Provided by tables below] * TRF

Δtherms = $(L_{sp} + L_{oc,i})$ * Δtherms per foot

Where:

EFLH = Equivalent Full Load Hours for Heating

= Actual or defaults by building type provided in Section 4.4, HVAC end use

For year round recirculation or domestic hot water:

= 8,766

For heating season recirculation, hours with the outside air temperature below 55°F:

| Zone | Hours |
|---|---|
| Zone 1 (Rockford) | 5,039 |
| Zone 2 (Chicago) | 4,963 |
| Zone 3 (Springfield) | 4,495 |
| Zone 4 (Belleville/ | 4,021 |
| Zone 5 (Marion) | 4,150 |

$Q_{base}$ = Heat Loss from Bare Pipe (Btu/hr/ft)

[412]This value comes from the reference table "Savings Summary by Building Type and System Type." The formula and the input tables in this section document assumptions used in calculation spreadsheet "Pipe Insulation Savings 2013-11-12.xlsx"

= See table below

$Q_{eff}$ = Heat Loss from Insulated Pipe (Btu/hr/ft)

= See table below

100,000 = conversion factor (1 therm = 100,000 Btu)

ηBoiler = Efficiency of the boiler being used to generate the hot water or steam in the pipe

= 81.9% for water boilers [413]

= 80.7% for steam boilers, except multifamily low-pressure [414]

= 64.8% for multifamily low-pressure steam boilers [415]

TRF = Thermal Regain Factor for space type, applied only to space heating energy and is applied to values resulting from Δtherms/ft tables below [416]

= See table below for base TRF values by pipe location

May vary seasonally such as: TRF[summer] * summer hours + TRF[winter] * winter hours where TRF values reflecting summer and winter conditions are apportioned by the hours for those conditions. TRF may also be adjusted by building specific balance temperature and operating hours above and below that balance temperature.417

| Pipe Location | Assumed Regain | TRF, Thermal Regain Factor |
|---|---|---|
| Outdoor | 0% | 1.0 |
| Indoor, heated space | 85% | 0.15 |
| Indoor, semi- heated, (unconditioned space, with heat transfer to conditioned space. E.g.: boiler room, ceiling plenum, basement, crawlspace, wall) | 30% | 0.70 |
| Indoor, unheated, (no heat transfer to conditioned space) | 0% | 1.0 |
| Location not specified | 85% | 0.15 |
| Custom | Custom | 1 – assumed regain |

$L_{sp}$ = Length of straight pipe to be insulated (linear foot)

$L_{oc,i}$ = Total equivalent length of (elbows and tees) of pipe to be insulated. Use table below to

[413] Average efficiencies of units from the California Energy Commission (CEC).
[414] Ibid.
[415] Katrakis, J. and T.S. Zawacki. "Field-Measured Seasonal Efficiency of Intermediate-sized Low-Pressure Steam Boilers". ASHRAE V99, pt. 2, 1993.
[416] Thermal regain for *residential* pipe insulation measures is discussed in Home Energy Services Impact Evaluation, prepared for the Massachusetts Residential Retrofit and Low Income Program Area Evaluation, Cadmus Group, Inc., August 2012 and Andrews, John, Better Duct Systems for Home Heating and Cooling, U.S. Department of Energy, 2001. Recognizing the differences between residential and commercial heating systems, the factors have been adjusted based on professional judgment. This factor would benefit from additional study and evaluation.
[417] Thermal Regain Factor_4-30-14.docx

determine equivalent lengths.

| Nominal Pipe Diameter | Equivalent Length (ft) | |
|---|---|---|
| | 90 Degree Elbow | Straight Tee |
| 1/2" | 0.04 | 0.03 |
| 3/4" | 0.06 | 0.05 |

$Q_{base}$ = Heat Loss from Bare Pipe (Btu/hr/ft). Calculated with the 3E Plus software.

$Q_{eff}$ = Heat Loss from Insulated Pipe (Btu/hr/ft). Calculated with the 3E Plus software.

The table below shows the deemed therm savings by building type and region on a per linear foot basis for both ½" and ¾" copper pipe.

The following table provides deemed values for 1/2" copper pipe, temperatures are assumed by category below, and insulation is assumed to be one inch fiberglass.

| Piping Use | Building Type | Annual Therms Saved / Linear Foot | | | | |
|---|---|---|---|---|---|---|
| | | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| Space Heating Non-recirculating | Assembly | 0.117 | 0.120 | 0.107 | 0.071 | 0.109 |
| | Assisted Living | 0.110 | 0.107 | 0.094 | 0.069 | 0.083 |
| | College | 0.100 | 0.093 | 0.083 | 0.046 | 0.055 |
| | Convenience Store | 0.097 | 0.089 | 0.079 | 0.057 | 0.064 |
| | Elementary School | 0.116 | 0.113 | 0.100 | 0.069 | 0.084 |
| | Garage | 0.064 | 0.063 | 0.056 | 0.044 | 0.049 |
| | Grocery | 0.105 | 0.105 | 0.092 | 0.057 | 0.068 |
| | Healthcare Clinic | 0.103 | 0.106 | 0.092 | 0.063 | 0.066 |
| | High School | 0.120 | 0.121 | 0.109 | 0.077 | 0.091 |
| | Hospital - CAV no econ | 0.115 | 0.119 | 0.101 | 0.087 | 0.099 |
| | Hospital - CAV econ | 0.117 | 0.121 | 0.103 | 0.089 | 0.101 |
| | Hospital - VAV econ | 0.048 | 0.045 | 0.034 | 0.020 | 0.022 |
| | Hospital - FCU | 0.087 | 0.099 | 0.080 | 0.094 | 0.127 |
| | Hotel/Motel | 0.115 | 0.112 | 0.101 | 0.069 | 0.084 |
| | Hotel/Motel - Common | 0.104 | 0.106 | 0.101 | 0.082 | 0.086 |
| | Hotel/Motel - Guest | 0.115 | 0.111 | 0.099 | 0.066 | 0.082 |
| | Manufacturing Facility | 0.068 | 0.066 | 0.061 | 0.037 | 0.041 |
| | MF - High Rise | 0.100 | 0.098 | 0.090 | 0.076 | 0.076 |

| Piping Use | Building Type | Annual Therms Saved / Linear Foot | | | | |
|---|---|---|---|---|---|---|
| | | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| | MF - High Rise - Common | 0.118 | 0.115 | 0.103 | 0.071 | 0.092 |
| | MF - High Rise - Residential | 0.096 | 0.096 | 0.087 | 0.075 | 0.073 |
| | MF - Mid Rise | 0.109 | 0.110 | 0.095 | 0.070 | 0.079 |
| | Movie Theater | 0.119 | 0.117 | 0.109 | 0.083 | 0.099 |
| | Office - High Rise - CAV no econ | 0.132 | 0.134 | 0.122 | 0.082 | 0.089 |
| | Office - High Rise - CAV econ | 0.136 | 0.139 | 0.128 | 0.088 | 0.097 |
| | Office - High Rise - VAV econ | 0.100 | 0.102 | 0.084 | 0.050 | 0.055 |
| | Office - High Rise - FCU | 0.073 | 0.072 | 0.062 | 0.033 | 0.035 |
| | Office - Low Rise | 0.093 | 0.093 | 0.074 | 0.045 | 0.052 |
| | Office - Mid Rise | 0.103 | 0.104 | 0.088 | 0.056 | 0.062 |
| | Religious Building | 0.105 | 0.098 | 0.094 | 0.069 | 0.079 |
| | Restaurant | 0.088 | 0.088 | 0.079 | 0.060 | 0.071 |
| | Retail - Department Store | 0.091 | 0.083 | 0.078 | 0.051 | 0.058 |
| | Retail - Strip Mall | 0.087 | 0.081 | 0.071 | 0.049 | 0.053 |
| | Warehouse | 0.095 | 0.089 | 0.091 | 0.057 | 0.070 |
| | Unknown | 0.101 | 0.100 | 0.089 | 0.064 | 0.074 |
| Space Heating - recirculation heating season only | All buildings (Hours below 55°F) | 0.329 | 0.324 | 0.293 | 0.262 | 0.271 |
| Space Heating - recirculation year round | All buildings (All hours) | 0.572 | 0.572 | 0.572 | 0.572 | 0.572 |
| DHW | Recirculation loop | 0.572 | 0.572 | 0.572 | 0.572 | 0.572 |
| Process | Custom | Custom | | | | |

The following table provides deemed savings values for 3/4" copper pipe with temperatures assumed by category below, insulation is assumed to be one inch fiberglass.

| Piping Use | Building Type | Annual Therms Saved / Linear Foot | | | | |
|---|---|---|---|---|---|---|
| | | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| Space Heating Non-recirculating | Assembly | 0.142 | 0.145 | 0.129 | 0.086 | 0.132 |
| | Assisted Living | 0.133 | 0.130 | 0.115 | 0.084 | 0.101 |
| | College | 0.121 | 0.113 | 0.101 | 0.056 | 0.067 |
| | Convenience Store | 0.117 | 0.108 | 0.096 | 0.069 | 0.077 |
| | Elementary School | 0.141 | 0.137 | 0.121 | 0.084 | 0.102 |
| | Garage | 0.078 | 0.077 | 0.067 | 0.054 | 0.060 |
| | Grocery | 0.127 | 0.127 | 0.111 | 0.069 | 0.083 |
| | Healthcare Clinic | 0.125 | 0.128 | 0.112 | 0.076 | 0.081 |
| | High School | 0.146 | 0.147 | 0.132 | 0.094 | 0.110 |
| | Hospital - CAV no econ | 0.140 | 0.144 | 0.123 | 0.105 | 0.120 |
| | Hospital - CAV econ | 0.142 | 0.147 | 0.125 | 0.108 | 0.123 |
| | Hospital - VAV econ | 0.058 | 0.055 | 0.041 | 0.025 | 0.027 |
| | Hospital - FCU | 0.105 | 0.120 | 0.098 | 0.115 | 0.154 |
| | Hotel/Motel | 0.140 | 0.136 | 0.122 | 0.084 | 0.102 |
| | Hotel/Motel - Common | 0.127 | 0.129 | 0.123 | 0.100 | 0.105 |
| | Hotel/Motel - Guest | 0.139 | 0.135 | 0.120 | 0.081 | 0.099 |
| | Manufacturing Facility | 0.083 | 0.080 | 0.074 | 0.045 | 0.050 |
| | MF - High Rise | 0.121 | 0.119 | 0.109 | 0.093 | 0.093 |
| | MF - High Rise - Common | 0.144 | 0.140 | 0.125 | 0.086 | 0.111 |
| | MF - High Rise - Residential | 0.117 | 0.116 | 0.105 | 0.091 | 0.089 |
| | MF - Mid Rise | 0.132 | 0.134 | 0.115 | 0.085 | 0.096 |
| | Movie Theater | 0.144 | 0.142 | 0.133 | 0.101 | 0.120 |
| | Office - High Rise - CAV no econ | 0.160 | 0.162 | 0.148 | 0.099 | 0.108 |
| | Office - High Rise - CAV econ | 0.165 | 0.169 | 0.155 | 0.107 | 0.118 |
| | Office - High Rise - VAV econ | 0.121 | 0.123 | 0.102 | 0.060 | 0.067 |
| | Office - High Rise - FCU | 0.089 | 0.087 | 0.075 | 0.040 | 0.042 |
| | Office - Low Rise | 0.113 | 0.113 | 0.090 | 0.055 | 0.063 |
| | Office - Mid Rise | 0.126 | 0.126 | 0.106 | 0.068 | 0.075 |
| | Religious Building | 0.127 | 0.119 | 0.114 | 0.084 | 0.095 |

| Piping Use | Building Type | Annual Therms Saved / Linear Foot | | | | |
|---|---|---|---|---|---|---|
| | | Zone 1 (Rockford) | Zone 2 (Chicago) | Zone 3 (Springfield) | Zone 4 (Belleville) | Zone 5 (Marion) |
| | Restaurant | 0.107 | 0.107 | 0.096 | 0.073 | 0.086 |
| | Retail - Department Store | 0.110 | 0.101 | 0.095 | 0.062 | 0.071 |
| | Retail - Strip Mall | 0.106 | 0.098 | 0.086 | 0.059 | 0.064 |
| | Warehouse | 0.115 | 0.108 | 0.111 | 0.069 | 0.085 |
| | Unknown | 0.123 | 0.122 | 0.108 | 0.078 | 0.090 |
| Space Heating - recirculation heating season only | All buildings (Hours below 55°F) | 0.399 | 0.393 | 0.356 | 0.319 | 0.329 |
| Space Heating - recirculation year round | All buildings (All hours) | 0.694 | 0.694 | 0.694 | 0.694 | 0.694 |
| DHW | Recirculation loop | 0.694 | 0.694 | 0.694 | 0.694 | 0.694 |
| Process | Custom | Custom | | | | |

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE Code: CI-HVC-SPIN-V01-150601**

## 4.4.25 Small Commercial Programmable Thermostat Adjustments

### DESCRIPTION

This measure involves reprogramming existing commercial programmable thermostats or building automation systems for reduced energy consumption through adjustments of unoccupied heating/cooling setpoints and/or fan control. This measure is limited to packaged HVAC units that are controlled by a commercial thermostat or building automation system. The measure is limited to select building types presented below.

**Eligible Small Commercial Building Types**

| Building Type |
|---|
| Assembly |
| Convenience Store |
| Office - Low Rise |
| Restaurant - Fast Food |
| Religious Facility |
| Restaurant - Full Service |
| Retail - Strip Mall |
| Retail - Department Store |

This measure was developed to be applicable to the following program types: RF, DI.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The criteria for this measure is established by optimizing heating/cooling temperature setbacks and fan operation with a commercial programmable thermostat or building automation system, which reprogrammed to match actual facility occupancy.

### DEFINITION OF BASELINE EQUIPMENT

The baseline for this measure is a commercial programmable thermostat or building automation system that is currently operating packaged HVAC units with heating/cooling temperature setbacks and fan operation that do not align with a facilities actual occupancy.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life of a programmable thermostat is assumed to be 8 years[418] based upon equipment life only[419]. For the purposes of claiming savings for a adjustment of an existing programmable thermostat, this is

[418] Table 1, HVAC Controls, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

[419] Future evaluation is strongly encouraged to inform the persistence of savings to further refine measure life assumption.

reduced to a 25% persistence factor to give a final measure life of 2 years. It is recommended that this assumption be evaluated by future energy measurement and verification activities.

**DEEMED MEASURE COST**

Actual labor costs should be used if the implementation method allows. If unknown the labor cost for this measure is assumed to be $70.34[420] per thermostat, as summarized in the table below.

| Measure | Units | Materials | Labor | Total Cost (including O&P) | City Cost Index (Install Only)* | Total | Source |
|---|---|---|---|---|---|---|---|
| Adjust Temperature Set Points | 4 | $0.00 | $5.95 | $6.55 | 134.5% | $35.24 | RS Means 2010 (pg 255, Section 23-09-8100) |
| Adjust Fan Schedule | 2 | $0.00 | $11.86 | $13.05 | 134.5% | $35.10 | RS Means 2010 (pg 255, Section 23-09-8120) |
| **Totals** | | | | | | **$70.34** | |

* Chicago, IL - Division 23

**LOADSHAPE**

N/A

**COINCIDENCE FACTOR**

N/A

[420] RSMeans, "Instrumentation and Control for HVAC", Mechanical Cost Data , Kingston, MA: Reed Construction Data, 2010, pg. 255 & 632

**Algorithm**

**CALCULATION OF SAVINGS**

**ELECTRIC ENERGY SAVINGS**[421]

ΔkWh = [Baseline Energy Use (kWh/Ton) – Proposed Energy Use (kWh/Ton)] * Cooling Capacity (Tons)

The following equations are used to calculate baseline and proposed electric energy use. The savings is the difference between the proposed and baseline calculated usage. This approach allows the savings estimate to account for the operational attributes of the baseline as well as the proposed case, yielding a better estimate than an approach that assumes a particular baseline or proposed energy use to determine savings.

**Electric Energy Use Equations (kWh / ton)**

| Building Type | Fan Mode During Occupied Period (*Fo*) | Equation |
|---|---|---|
| **Assembly** | Continuous | ***CZ***+***Fu****(0.83****Tc***+0.83****Th***+1.67****Ws***-293.018)-0.0922****Tc*******Th***+1.291****Ws*** |
| | Intermittent | ***CZ***+***Fu****(1.911-0.12****Tc***)+***Tc****(0.00311****Ws***-0.229)+0.11****Ws*** |
| **Convenience Store** | Continuous | ***CZ***+***Fu****(-28.629****Tc***-11.69****Th***+19.118****Ws***-2935.12)+0.909****Ws*** |
| | Intermittent | ***CZ***+***Tc****(0.0863****Ws***-12.688)+***Th****(0.043****Ws***-6.38)+1.669****Ws*** |
| **Office – Low Rise** | Continuous | ***CZ***+***Fu****(7.082****Tc***-41.199****Th***+18.734****Ws***-3288.55)+***Tc****(0.205****Ws***-34.929) |
| | Intermittent | ***CZ***+***Tc****(0.0806****Ws***-8.984)+***Th****(0.0864****Ws***-9.558)+1.178****Ws*** |
| **Religious** | Continuous | ***CZ***+***Fu****(-1.579****Tc***-18.14****Th***+15.01****Ws***-2417.74)+***Tc****(0.177****Ws***-26.412) |
| | Intermittent | ***CZ***+***Fu****(0.266****Tc***-2.067)+***Tc****(0.0295****Ws***-4.502)+***Th****(0.0517****Ws***-8.251)+0.735****Ws*** |
| **Restaurant – Fast Food** | Continuous | ***CZ***+***Fu****(0.678****Tc***+0.257****Th***+2.88****Ws***-494.006)+***Tc****(0.0231****Ws***-4.074)+***Th****(0.00936****Ws***-1.655)+0.918****Ws*** |
| | Intermittent | ***CZ***+***Fu****(0.377****Tc***+0.124****Th***+0.13****Ws***-24.893)+***Tc****(-0.0143****Th***+0.0166****Ws***-2.691)+0.898****Ws*** |
| **Restaurant – Sit Down** | Continuous | ***CZ***+***Fu****(-8.41****Th***+11.766****Ws***-1910.81)+***Tc****(0.282****Ws***-43.851) |
| | Intermittent | ***CZ***+0.123****Fu*******Tc***+***Tc****(0.0561****Ws***-8.237)+***Th****(0.0219****Ws***-3.284)+1.038****Ws*** |
| **Retail – Large** | Continuous | ***CZ***+***Fu****(-1.475****Th***+0.755****Ws***-114.373)+***Th****(0.151****Ws***-24.016)+1.612****Ws*** |
| | Intermittent | ***CZ***+***Tc****(0.0173****Ws***-1.912)+***Th****(0.0249****Ws***-3.29)+0.511****Ws*** |
| **Retail – Strip Mall** | Continuous | ***CZ***+***Fu****(1.077****Tc***-10.697****Th***+6.91****Ws***-1117.18)+***Tc****(0.0583****Ws***-7.54)+1.231****Ws*** |
| | Intermittent | ***CZ***+0.0894****Fu*******Tc***+***Th****(-0.0142****Tc***+0.04****Ws***-5.278)+0.884****Ws*** |

Where:

CZ = Climate Zone Coefficient

= Depends on Building Type and Fan Mode During Occupied Period (see table below)

[421] Savings equations and factors determined by regression of results of a series of eQuest simulations. See Programmable T-Stat Work Paper_PECI_FinalDraft_140730_Redline.docx for details.

Tc = Degrees of Cooling Setback °F

= Must be between 0-15°F

Th = Degrees of Heating Setback °F

=Must be between 0-15°F

Fo = Fan Mode During Occupied Period (Note: Commercial mechanical code requires continuous fan operation during occupied periods to meet ventilation requirements.)

= Continuous for occupied fan that runs continuously (e.g. Fan Mode Set to 'On')

= Intermittent for occupied fan that runs intermittently (e.g. Fan Mode Set to 'Auto')

Fu = Fan Mode during Unoccupied Period

= 0 for unoccupied fan that runs continuously (e.g. Fan Mode Set to 'On')

= 1 for unoccupied fan that runs intermittently (e.g. Fan Mode Set to 'Auto')

Ws = Weekly Hours thermostat is in Occupied mode,

= Minimum values depend on Building Type (see table below), maximum value of 168 (24/7) ex: Weekly occupancy schedule of Mon-Sat 8AM-5PM, Sun 9AM-2PM, Ws = 59

**Electric Energy Use Climate Zone Coefficients and Minimum Weekly Hours Occupied**

| Building Type | Fan Mode During Occupied Period (*Fo*) | Climate Zone Coefficient (*CZ*) | | | | | Minimum *Ws* |
|---|---|---|---|---|---|---|---|
| | | 1 | 2 | 3 | 4 | 5 | |
| **Assembly** | Continuous | 911.366 | 928.924 | 1152.83 | 1208.999 | 1210.173 | 98 |
| | Intermittent | 735.752 | 762.831 | 966.562 | 998.927 | 1028.906 | |
| **Convenienc e Store** | Continuous | 4817.094 | 4832.784 | 5139.133 | 5182.161 | 5208.608 | 108 |
| | Intermittent | 1478.133 | 1514.568 | 1784.384 | 1843.463 | 1930.47 | |
| **Office - Low Rise** | Continuous | 5047.662 | 5039.592 | 5187.924 | 5217.672 | 5177.449 | 55 |
| | Intermittent | 825.072 | 808.965 | 946.571 | 979.421 | 945.418 | |
| **Religious Facility** | Continuous | 4197.117 | 4172.858 | 4380.025 | 4370.008 | 4356.054 | 133 |
| | Intermittent | 632.404 | 603.395 | 678.294 | 664.717 | 616.853 | |
| **Restaurant – Fast Food** | Continuous | 1342.988 | 1378.661 | 1664.018 | 1714.201 | 1727.841 | 108 |
| | Intermittent | 993.764 | 1039.643 | 1307.8 | 1340.544 | 1389.791 | |
| **Restaurant – Full Service** | Continuous | 4070.35 | 4094.742 | 4428.966 | 4501.829 | 4522.522 | 117 |
| | Intermittent | 1472.014 | 1516.05 | 1856.108 | 1938.441 | 2056.45 | |
| **Retail – Department Store** | Continuous | 1510.201 | 1496.47 | 1706.105 | 1716.128 | 1688.464 | 93 |
| | Intermittent | 701.27 | 702.129 | 847.735 | 875.12 | 881.677 | |
| **Retail – Strip Mall** | Continuous | 1926.294 | 1930.137 | 2156.856 | 2174.435 | 2165.03 | 93 |
| | Intermittent | 656.479 | 673.257 | 835.906 | 850.322 | 869.921 | |

**EXAMPLE**

A low rise office building in Rockford (Climate Zone 1) is occupied Mon-Fri 7AM-6PM and is heated and cooled with a packaged Gas (150 kBtu output) / DX (10 Ton) RTU which is controlled by a programmable thermostat. When the technician reviews the thermostat schedule they find the unoccupied schedule is programmed incorrectly. During the unoccupied periods the fan is programmed correctly, and runs in intermittent "auto" mode, although the heating and cooling temperature setpoints are not setback.

The technician adjusts the unoccupied schedule to include a 10°F cooling and heating temperature setback during the unoccupied periods.

ΔkWh = [Baseline Energy Use (kWh/Ton) – Proposed Energy Use (kWh/Ton)] * Cooling Capacity (Tons)

Baseline Energy Use (kWh/Ton) = Equation for Office Low Rise, *Fo*=Continuous

= ***CZ***+***Fu****(7.082****Tc***-41.199****Th***+18.734****Ws***-3288.55)+***Tc****(0.205****Ws***-34.929)

= ***5047.662***+1*(7.082****0***-41.199****0***+18.734****55***-3288.55)+***0****(0.205****55***-34.929)

= 2,789.482 kWh/Ton

Proposed Energy Use (kWh/Ton) = Equation for Office Low Rise, *Fo*=Continuous

= ***CZ***+***Fu****(7.082****Tc***-41.199****Th***+18.734****Ws***-3288.55)+***Tc****(0.205****Ws***-34.929)

= ***5047.662***+***1****(7.082****10***-41.199***10**+18.734***55**-3288.55)+***10*(***0.205****55***-34.929)

= 2,211.722 kWh/Ton

ΔkWh = [2,789.482 (kWh/Ton) – 2,211.722 (kWh/Ton)] * 10 Tons

= 577.71 kWh/Ton * 10 Tons

= 5777.1 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

N/A

**NATURAL GAS ENERGY SAVINGS**

ΔTherms = [Baseline Energy Use (Therms/kBtuh) – Proposed Energy Use(Therms/kBtuh)] * Output Heating Capacity (kBtuh)

The following equations are used to calculate baseline and proposed natural gas energy use. The savings is the difference between the proposed and baseline calculated usage. This approach allows the savings estimate to account for the operational attributes of the baseline as well as the proposed case, yielding a better estimate than an approach that assumes a particular baseline or proposed energy use to determine savings.

**Natural Gas Energy Use Equations (therms / kbtu)**

| Building Type | Fan Mode During Occupied Period (*Fo*) | Equation |
|---|---|---|
| **Assembly** | Continuous | CZ+Fu*(0.232*Th+0.0984*Ws-18.79)+Th*(0.00271*Ws-0.535)+0.0142*Ws |
| | Intermittent | ***CZ***+***Fu****(0.00405****Th***+0.000519****Ws***-0.11)+***Th****(0.0000689****Ws***-0.0118)+0.0022****Ws*** |
| **Convenience Store** | Continuous | ***CZ***+***Fu****(0.00545****Th***-0.00251****Ws***+0.416)+***Th****(0.000123****Ws***-0.0204)+0.00183****Ws*** |
| | Intermittent | ***CZ***+***Fu****(0.00231****Th***-0.0349)+***Th****(0.000309****Ws***-0.0494)+0.00266****Ws*** |
| **Office – Low Rise** | Continuous | ***CZ***+***Fu****(0.0205****Th***+0.364)+***Th****(0.00046****Ws***-0.0554)+0.00169****Ws*** |
| | Intermittent | ***CZ***+***Fu****(0.00745****Th***-0.142)+***Th****(0.00077****Ws***-0.111)+0.00199****Ws*** |
| **Religious** | Continuous | ***CZ***+0.00791****Fu*******Th***+***Th****(0.00096****Ws***-0.167)+0.00184****Ws*** |
| | Intermittent | ***CZ***+***Fu****(0.00143****Th***-0.0309)+***Th****(0.0008****Ws***-0.134)+0.00219****Ws*** |
| **Restaurant – Fast Food** | Continuous | ***CZ***+***Fu****(0.0431****Th***+0.0424****Ws***-7.517)+***Th****(0.00113****Ws***-0.213)+0.0119****Ws*** |
| | Intermittent | ***CZ***+***Fu****(0.0125****Th***+0.0036****Ws***-0.71)+***Th****(0.000329****Ws***-0.0615)+0.00738****Ws*** |
| **Restaurant –Sit Down** | Continuous | ***CZ***+***Fu****(0.00445****Ws***-0.535)+***Th****(0.000679****Ws***-0.1)+0.00218****Ws*** |
| | Intermittent | ***CZ***+***Fu****(0.00144****Th***+0.000262****Ws***-0.0553)+***Th****(0.00018****Ws***-0.0299)+0.00166****Ws*** |
| **Retail – Large** | Continuous | ***CZ***+0.00203****Fu*******Th***+***Th****(0.000591****Ws***-0.0812)+0.00194****Ws*** |
| | Intermittent | ***CZ***+***Th****(0.000406****Ws***-0.0611)+0.00228****Ws*** |
| **Retail – Strip Mall** | Continuous | ***CZ***+***Fu****(0.00998****Th***+0.00207****Ws***-0.206)+***Th****(0.000665****Ws***-0.101)+0.00292****Ws*** |
| | Intermittent | ***CZ***+***Fu****(0.00383****Th***-0.0656)+***Th****(0.000575****Ws***-0.0912)+0.00249****Ws*** |

Where:

CZ = Climate Zone Coefficient

= Depends on Building Type and Fan Mode During Occupied Period (see table below)

Th = Degrees of Heating Setback °F

= Must be between 0-15°F

Fo = Fan Mode During Occupied Period (Note: Commercial mechanical code requires continuous fan operation during occupied periods to meet ventilation requirements.)

= Continuous for occupied fan that runs continuously (e.g. Fan Mode Set to 'On')

= Intermittent for occupied fan that runs intermittently (e.g. Fan Mode Set to 'Auto')

Fu = Fan Mode during Unoccupied Period

= 0 for unoccupied fan that runs continuously (e.g. Fan Mode Set to 'On')

= 1 for unoccupied fan that runs intermittently (e.g. Fan Mode Set to 'Auto')

Ws = Weekly Hours thermostat is in Occupied mode,

= Minimum values depends on Building Type (see table below), maximum value of 168 (24/7) ex: Weekly occupancy schedule of Mon-Sat 8AM-5PM, Sun 9AM-2PM, Ws = 59.

**Natural Gas Energy Use Climate Zone Coefficients and Minimum Weekly Hours Occupied**

| Building Type | Fan Mode During Occupied Period (*Fo*) | Climate Zone Coefficient (*CZ*) | | | | | Minimum *Ws* |
|---|---|---|---|---|---|---|---|
| | | 1 | 2 | 3 | 4 | 5 | |
| **Assembly** | Continuous | 19.872 | 17.83 | 15.828 | 15.282 | 13.482 | 98 |
| | Intermittent | 0.237 | 0.0989 | 0.0267 | 0.0131 | 0.0871 | |
| **Convenience Store** | Continuous | 1.493 | 1.081 | 0.782 | 0.544 | 0.114 | 108 |
| | Intermittent | 1.128 | 0.854 | 0.619 | 0.437 | 0.0854 | |
| **Office - Low Rise** | Continuous | 1.718 | 1.317 | 0.971 | 0.739 | 0.319 | 55 |
| | Intermittent | 3.447 | 3.022 | 2.503 | 2.251 | 1.646 | |
| **Religious Facility** | Continuous | 6.294 | 5.55 | 4.678 | 4.202 | 3.122 | 133 |
| | Intermittent | 5.914 | 5.368 | 4.557 | 4.137 | 3.246 | |
| **Restaurant – Fast Food** | Continuous | 8.383 | 7.211 | 6.034 | 5.767 | 4.71 | 108 |
| | Intermittent | 1.227 | 0.636 | 0.302 | 0.102 | -0.262 | |
| **Restaurant – Full Service** | Continuous | 5.247 | 4.484 | 3.753 | 3.465 | 2.627 | 117 |
| | Intermittent | 0.951 | 0.704 | 0.51 | 0.381 | 0.0746 | |
| **Retail – Department Store** | Continuous | 4.385 | 3.854 | 3.192 | 2.784 | 1.858 | 93 |
| | Intermittent | 3.061 | 2.672 | 2.182 | 1.829 | 1.008 | |
| **Retail – Strip Mall** | Continuous | 3.917 | 3.394 | 2.728 | 2.394 | 1.617 | 93 |
| | Intermittent | 2.659 | 2.292 | 1.811 | 1.543 | 0.909 | |

**EXAMPLE**

A low rise office building in Rockford (Climate Zone 1) is occupied Mon-Fri 7AM-6PM and is heated and cooled with a packaged Gas (150 kBtu output) / DX (10 Ton) RTU which is controlled by a programmable thermostat. When the technician reviews the thermostat schedule they find the unoccupied schedule is programmed incorrectly. During the unoccupied periods the fan is programmed correctly, and runs in intermittent "auto" mode, although the heating and cooling temperature setpoints are not setback.

The technician adjusts the unoccupied schedule to include a 10°F cooling and heating temperature setback during the unoccupied periods.

ΔTherms = [Baseline Energy Use (Therms/kBtuh) – Proposed Energy Use(Therms/kBtuh)] * Output Heating Capacity (kBtuh)

Baseline Energy Use (Therms/kBtuh) = Equation for Office Low Rise, *Fo*=Continuous

= ***CZ***+***Fu****(0.0205****Th***+0.364)+***Th****(0.00046****Ws***-0.0554)+0.00169****Ws***

= ***1.718***+***1****(0.0205****0***+0.364)+***0****(0.00046****55***-0.0554)+0.00169****55***

= 2.17495 Therms/kBtuh output

Proposed Energy Use (Therms/kBtuh) = Equation for Office Low Rise, *Fo*=Continuous

= ***CZ***+***Fu****(0.0205****Th***+0.364)+***Th****(0.00046****Ws***-0.0554)+0.00169****Ws***

= ***1.718***+***1****(0.0205****10***+0.364)+***10****(0.00046****55***-0.0554)+0.00169****55***

= 2.07895 Therms/kBtuh output

ΔTherms = [2.17495 (Therms/kBtuh output) – 2.07895 (Therms/kBtuh output)] * 150kBtuh output

= 0.096 (Therms/kBtuh output) * 150kBtuh output

= 14.4 Therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HVC- PRGA -V01-150601**

## 4.4.26 Variable Speed Drives for HVAC Supply and Return Fans

### DESCRIPTION

This measure is applied to variable speed drives (VSD) which are installed on HVAC supply fans and return fans. There is a separate measure for HVAC pumps and cooling tower fans. All other VSD applications require custom analysis by the program administrator. The VSD will modulate the speed of the motor when it does not need to run at full load. Since the power of the motor is proportional to the cube of the speed for these types of applications, significant energy savings will result.

This measure was developed to be applicable to the following program types: TOS, RF. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The VSD is applied to a motor which does not have a VSD. The application must have a variable load and installation is to include the necessary controls. Savings are based on application of VSDs to a range of baseline load conditions including no control, inlet guide vanes, outlet guide vanes and throttling valves.

### DEFINITION OF BASELINE EQUIPMENT

The time of sale baseline is a new motor installed without a VSD or other methods of control. Retrofit baseline is an existing motor operating as is. Retrofit baselines may or may not include guide vanes, throttling valves or other methods of control. This information shall be collected from the customer.

Installations of new equipment with VSDs which are required by IECC 2012 as adopted by the State of Illinois are not eligible for incentives.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life for HVAC application is 15 years;[422] measure life for process is 10 years.[423]

### DEEMED MEASURE COST

Customer provided costs will be used when available. Default measure costs[424] are noted below for up to 20 hp motors. Custom costs must be gathered from the customer for motor sizes not listed below.

| HP | Cost |
|---|---|
| 1 -5 HP | $ 1,330 |
| 7.5 HP | $ 1,622 |
| 10 HP | $ 1,898 |
| 15 HP | $ 2,518 |
| 20 HP | $ 3,059 |

### LOADSHAPE

Loadshape C39 - VFD - Supply fans <10 HP

Loadshape C40 - VFD - Return fans <10 HP

[422] Efficiency Vermont TRM 10/26/11 for HVAC VSD motors
[423] DEER 2008
[424] Ohio TRM 8/6/2010 varies by motor/fan size based on equipment costs from Granger 2008 Catalog pp 286-289, average across available voltages and models. Labor costs from RS Means Data 2008 Ohio average cost adjustment applied.

Loadshape C41 - VFD - Exhaust fans <10 HP

### COINCIDENCE FACTOR

The demand savings factor (DSF) is already based upon coincident savings, and thus there is no additional coincidence factor for this characterization.

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS[425]

$kWh_{Base}$ =

$$\left(0.746 \times HP \times \frac{LF}{\eta_{motor}}\right) \times RHRS_{Base} \times \sum_{0\%}^{100\%} (\%FF \times PLR_{Base})$$

$kWh_{Retrofit}$ =

$$\left(0.746 \times HP \times \frac{LF}{\eta_{motor}}\right) \times RHRS_{base} \times \sum_{0\%}^{100\%} (\%FF \times PLR_{Retrofit})$$

$\Delta kWh_{fan}$ = $\mathrm{kWh_{Base}} - \mathrm{kWh_{Retrofit}}$

$\Delta kWh_{total}$ = $\Delta\mathrm{kWh_{fan}} \times (1 + \mathrm{IE_{energy}})$

Where:

$kWh_{Base}$ = Baseline annual energy consumption (kWh/yr)

$kWh_{Retrofit}$ = Retrofit annual energy consumption (kWh/yr)

$\Delta kWh_{fan}$ = Fan-only annual energy savings

$\Delta kWh_{total}$ = Total project annual energy savings

$0.746$ = Conversion factor for HP to kWh

$HP$ = Nominal horsepower of controlled motor

$LF$ = Load Factor; Motor Load at Fan Design CFM (Default = 65%)[426]

$\eta_{motor}$ = Installed nominal/nameplate motor efficiency

[425] Methodology developed and tested in Del Balso, Ryan Joseph. "Investigation into the Reliability of Energy Efficiency/Demand Side Management Savings Estimates for Variable Frequency Drives in Commercial Applications". A project report submitted to the Faculty of the Graduate School of the University of Colorado, 2013.

[426] Lawrence Berkeley National Laboratory, and Resource Dynamics Corporation. (2008). "Improving Motor and Drive System Performance; A Sourcebook for Industry". U.S. Department of Energy, Office of Energy Efficiency and Renewable Energy. Golden, CO: National Renewable Energy Laboratory.

Default motor is a NEMA Premium Efficiency, ODP, 4-pole/1800 RPM fan motor

**NEMA Premium Efficiency Motors Default Efficiencies**[427]

| Size HP | Open Drip Proof (ODP) | | | Totally Enclosed Fan-Cooled (TEFC) | | |
|---|---|---|---|---|---|---|
| | # of Poles | | | # of Poles | | |
| | 6 | 4 | 2 | 6 | 4 | 2 |
| | Speed (RPM) | | | Speed (RPM) | | |
| | 1200 | 1800 Default | 3600 | 1200 | 1800 | 3600 |
| 1 | 0.825 | 0.855 | 0.770 | 0.825 | 0.855 | 0.770 |
| 1.5 | 0.865 | 0.865 | 0.840 | 0.875 | 0.865 | 0.840 |
| 2 | 0.875 | 0.865 | 0.855 | 0.885 | 0.865 | 0.855 |
| 3 | 0.885 | 0.895 | 0.855 | 0.895 | 0.895 | 0.865 |
| 5 | 0.895 | 0.895 | 0.865 | 0.895 | 0.895 | 0.885 |
| 7.5 | 0.902 | 0.910 | 0.885 | 0.910 | 0.917 | 0.895 |
| 10 | 0.917 | 0.917 | 0.895 | 0.910 | 0.917 | 0.902 |
| 15 | 0.917 | 0.930 | 0.902 | 0.917 | 0.924 | 0.910 |
| 20 | 0.924 | 0.930 | 0.910 | 0.917 | 0.930 | 0.910 |
| 25 | 0.930 | 0.936 | 0.917 | 0.930 | 0.936 | 0.917 |
| 30 | 0.936 | 0.941 | 0.917 | 0.930 | 0.936 | 0.917 |
| 40 | 0.941 | 0.941 | 0.924 | 0.941 | 0.941 | 0.924 |
| 50 | 0.941 | 0.945 | 0.930 | 0.941 | 0.945 | 0.930 |
| 60 | 0.945 | 0.950 | 0.936 | 0.945 | 0.950 | 0.936 |
| 75 | 0.945 | 0.950 | 0.936 | 0.945 | 0.954 | 0.936 |
| 100 | 0.950 | 0.954 | 0.936 | 0.950 | 0.954 | 0.941 |
| 125 | 0.950 | 0.954 | 0.941 | 0.950 | 0.954 | 0.950 |
| 150 | 0.954 | 0.958 | 0.941 | 0.958 | 0.958 | 0.950 |
| 200 | 0.954 | 0.958 | 0.950 | 0.958 | 0.962 | 0.954 |
| 250 | 0.954 | 0.958 | 0.950 | 0.958 | 0.962 | 0.958 |
| 300 | 0.954 | 0.958 | 0.954 | 0.958 | 0.962 | 0.958 |
| 350 | 0.954 | 0.958 | 0.954 | 0.958 | 0.962 | 0.958 |

[427] Douglass, J. (2005). Induction Motor Efficiency Standards. Washington State University and the Northwest Energy Efficiency Alliance, Extension Energy Program, Olympia, WA. Retrieved October 17, 2013, from http://www1.eere.energy.gov/manufacturing/tech_assistance/pdfs/motor_efficiency_standards.pdf

| Size HP | Open Drip Proof (ODP) | | | Totally Enclosed Fan-Cooled (TEFC) | | |
|---|---|---|---|---|---|---|
| | # of Poles | | | # of Poles | | |
| | 6 | 4 | 2 | 6 | 4 | 2 |
| | Speed (RPM) | | | Speed (RPM) | | |
| | 1200 | 1800 Default | 3600 | 1200 | 1800 | 3600 |
| 400 | 0.958 | 0.958 | 0.958 | 0.958 | 0.962 | 0.958 |
| 450 | 0.962 | 0.962 | 0.958 | 0.958 | 0.962 | 0.958 |
| 500 | 0.962 | 0.962 | 0.958 | 0.958 | 0.962 | 0.958 |

$RHRS_{Base}$ = Annual operating hours for fan motor based on building type

Default hours are provided for HVAC applications which vary by HVAC application and building type[428]. When available, actual hours should be used.

| Building Type | Pumps and fans |
|---|---|
| College/University | 4216 |
| Grocery | 5840 |
| Heavy Industry | 3585 |
| Hotel/Motel | 6872 |
| Light Industry | 2465 |
| Medical | 6871 |
| Office | 2301 |
| Restaurant | 4654 |
| Retail/Service | 3438 |
| School(K-12) | 2203 |
| Warehouse | 3222 |
| Average = Miscellaneous | 4103 |

$\%FF$ = Percentage of run-time spent within a given flow fraction range

Default Fan Duty Cycle Based on 2012 ASHRAE Handbook; HVAC Systems and Equipment, page 45.11, Figure 12.

[428] ComEd Trm June 1, 2010 page 139. The Office hours is based upon occupancy from the eQuest model developed for EFLH, since it was agreed the ComEd value was too low.

| Flow Fraction (% of design cfm) | Percent of Time at Flow Fraction |
|---|---|
| 0% to 10% | 0.0% |
| 10% to 20% | 1.0% |
| 20% to 30% | 5.5% |
| 30% to 40% | 15.5% |
| 40% to 50% | 22.0% |
| 50% to 60% | 25.0% |
| 60% to 70% | 19.0% |
| 70% to 80% | 8.5% |
| 80% to 90% | 3.0% |
| 90% to 100% | 0.5% |

$PLR_{Base}$ = Part load ratio for a given flow fraction range based on the baseline flow control type

$PLR_{Retrofit}$ = Part load ratio for a given flow fraction range based on the retrofit flow control type

| Control Type | Flow Fraction | | | | | | | | | |
|---|---|---|---|---|---|---|---|---|---|---|
| | 10% | 20% | 30% | 40% | 50% | 60% | 70% | 80% | 90% | 100% |
| No Control or Bypass Damper | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 |
| Discharge Dampers | 0.46 | 0.55 | 0.63 | 0.70 | 0.77 | 0.83 | 0.88 | 0.93 | 0.97 | 1.00 |
| Outlet Damper, BI & Airfoil Fans | 0.53 | 0.53 | 0.57 | 0.64 | 0.72 | 0.80 | 0.89 | 0.96 | 1.02 | 1.05 |
| Inlet Damper Box | 0.56 | 0.60 | 0.62 | 0.64 | 0.66 | 0.69 | 0.74 | 0.81 | 0.92 | 1.07 |
| Inlet Guide Vane, BI & Airfoil Fans | 0.53 | 0.56 | 0.57 | 0.59 | 0.60 | 0.62 | 0.67 | 0.74 | 0.85 | 1.00 |
| Inlet Vane Dampers | 0.38 | 0.40 | 0.42 | 0.44 | 0.48 | 0.53 | 0.60 | 0.70 | 0.83 | 0.99 |
| Outlet Damper, FC Fans | 0.22 | 0.26 | 0.30 | 0.37 | 0.45 | 0.54 | 0.65 | 0.77 | 0.91 | 1.06 |
| Eddy Current Drives | 0.17 | 0.20 | 0.25 | 0.32 | 0.41 | 0.51 | 0.63 | 0.76 | 0.90 | 1.04 |
| Inlet Guide Vane, FC Fans | 0.21 | 0.22 | 0.23 | 0.26 | 0.31 | 0.39 | 0.49 | 0.63 | 0.81 | 1.04 |
| VFD with duct static pressure controls | 0.09 | 0.10 | 0.11 | 0.15 | 0.20 | 0.29 | 0.41 | 0.57 | 0.76 | 1.01 |
| VFD with low/no duct static pressure | 0.05 | 0.06 | 0.09 | 0.12 | 0.18 | 0.27 | 0.39 | 0.55 | 0.75 | 1.00 |

Provided below is the resultant values based upon the defaults provided above:

| Control Type | $\sum_{0\%}^{100\%}(\%\boldsymbol{FF} \times \boldsymbol{PLR}_{Base})$ |
|---|---|
| No Control or Bypass Damper | 1.00 |
| Discharge Dampers | 0.80 |
| Outlet Damper, BI & Airfoil Fans | 0.78 |
| Inlet Damper Box | 0.69 |
| Inlet Guide Vane, BI & Airfoil Fans | 0.63 |
| Inlet Vane Dampers | 0.53 |
| Outlet Damper, FC Fans | 0.53 |
| Eddy Current Drives | 0.49 |
| Inlet Guide Vane, FC Fans | 0.39 |
| VFD with duct static pressure controls | 0.30 |
| VFD with low/no duct static pressure | 0.27 |

$IE_{energy}$ = HVAC interactive effects factor for energy (default = 15.7%)

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$kW_{Base}$ = $$\left(0.746 \times HP \times \frac{LF}{\eta_{motor}}\right) \times PLR_{Base,FFpeak}$$

$kW_{Retrofit}$ = $$\left(0.746 \times HP \times \frac{LF}{\eta_{motor}}\right) \times PLR_{Retrofit,FFpeak}$$

$\Delta kW_{fan}$ = $$\mathrm{kW_{Base}} - \mathrm{kW_{Retrofit}}$$

$\Delta kW_{total}$ = $$\Delta\mathrm{kW_{fan}} \times (1 + \mathrm{IE_{demand}})$$

Where:

$kW_{Base}$ = Baseline summer coincident peak demand (kW)

$kW_{Retrofit}$ = Retrofit summer coincident peak demand (kW)

$\Delta kW_{fan}$ = Fan-only summer coincident peak demand impact

$\Delta kW_{total}$ = Total project summer coincident peak demand impact

$PLR_{Base,FFpeak}$ = The part load ratio for the average flow fraction between the peak daytime hours during the weekday peak time period based on the baseline flow control type (default average flow fraction during peak period = 90%)

$PLR_{Retrofit,FFpeak}$ = The part load ratio for the average flow fraction between the peak daytime hours during the weekday peak time period based on the retrofit flow control type (default average flow fraction during peak period = 90%)

$IE_{demand}$ = HVAC interactive effects factor for summer coincident peak demand (default = 15.7%)

### FOSSIL FUEL IMPACT DESCRIPTIONS AND CALCULATION

There are no expected fossil fuel impacts for this measure.

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-HVC-VSDF-V01-150601

## 4.4.27 Energy Recovery Ventilator

### DESCRIPTION

This measure includes the addition of energy recovery equipment on existing or new unitary equipment, where energy recovery is not required by the IECC 2012. This measure analyzes the heating savings potential from recovering energy from exhaust or relief building air. This measure assumes during unoccupied hours of the building no exhaust or relief air is available for energy recovery.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

Efficient equipment is unitary equipment that incorporates energy recovery not required by the IECC 2012.

### DEFINITION OF BASELINE EQUIPMENT

The baseline is unitary equipment not require by IECC 2012 to incorporate energy recovery.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for the domestic energy recovery equipment is 15 years.[429]

### DEEMED MEASURE COST

The incremental cost for this measure assumes cost of cabinet and controls incorporated into packaged and built up air handler units. Additionally it assumes 1 to 1 ratio of fresh and exhausted air.

| Energy Recovery Equipment Type | Incremental Cost $/CFM[430] |
|---|---|
| Fixed Plate | $6 |
| Rotary Wheel | $6 |
| Heat Pipe | $6 |

### DEEMED O&M COST ADJUSTMENTS

There are no expected O&M savings associated with this measure.

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

[429] Assumed service life limited by controls -" Demand Control Ventilation Using CO2 Sensors", pg. 19, by US Department of Energy Efficiency and Renewable Energy

[430] "Map to HVAC Solutions", by Michigan Air, Issue 3, 2006

**Algorithm**

### CALCULATION OF ENERGY SAVINGS

### ELECTRIC ENERGY SAVINGS

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

There are no anticipated electrical savings from this measure as it is assumed that the additional fan energy due to the increased static pressure drop offsets cooling energy savings. Where this is not expected to be the case, a custom calculation should be used to determine the savings.

### NATURAL GAS SAVINGS

Gas savings algorithm is derived from the following:

ΔTherms = (Design Heating Load * TE_ERV * EFLH * OccHours/24) / (100,000 * µHeat)

Where:

Design Heating Load = (1.08 * CFM * ΔT)

1.08 = A constant for sensible heat equations (BTU/h/CFM.°F)

CFM = Cubic Feet per Minute of Energy Recovery Ventilator

ΔT = T_RA – T_DD

T_RA = Temperature of the Return Air = 70°F or custom

T_DD = Temperature on design day of outside air[431]

= (see Table below) or custom

| Zone | Weather Station | T_DD, Temperature, °F |
|---|---|---|
| 1 | Greater Rockford | -5.8 |
| 2 | Chicago/O'Hare ARPT. | -1.5 |
| 3 | Springfield/Capital | 0.4 |
| 4 | Scott AFB MidAmerica | 9.0 |
| 5 | Cape Girardeau Regional | 9.7 |
| Average | - | 2.4 |

TE_ERV = Thermal Effectiveness of Energy Recovery Equipment[432]

[431] Weather Station Data, 99.6% Heating DB - 2013 Fundamentals, ASHRAE Handbook
[432] Energy Recovery Fact Sheet - Center Point Energy, MN

= (see Table below) or custom

| Heat Recovery Equipment Type | TE_ERV (%) |
|---|---|
| Fixed Plate | 0.65 |
| Rotary Equipment | 0.68 |
| Heat Pipe | 0.55 |

EFLH = Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use

OccHour = Average Hours per day facility is occupied

= (see Table 4.4.1 EQuest Modeling Inputs by Building Type) or custom

µHeat = Efficiency of heating system

= Actual

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HVC-ERVE-V01-150601**

## 4.4.28 Stack Economizer for Boilers Serving HVAC Loads

### MEASURE DESCRIPTION

Stack economizers are designed to recover heat from hot boiler flue gasses. Recovered heat is used to preheat boiler feed water. This measure describes the retrofit of HVAC boilers with stack economizers. HVAC boilers are defined as those used for space heating applications. There is another, similar measure for boilers that serve process loads.

This measure was developed to be applicable to the following program types: NC, TOS, RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify the economizer must be installed on a boiler exhaust stack. Heat captured by the economizer is to be used to pre-heat boiler feed water.

### DEFINITION OF BASELINE EQUIPMENT

The baseline boiler does not have an economizer installed.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for the boiler stack economizer is 15 years.[433]

### DEEMED MEASURE COST

The incremental and full measure cost for this measure is custom.

### DEEMED O&M COST ADJUSTMENTS

The O&M cost for this measure is custom.

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

**Algorithm**

### CALCULATION OF ENERGY SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS SAVINGS

$$\Delta therms = SF * MBH_In * EFLH / 100$$

[433] PA Consulting, Focus on Energy Evaluation, Business Programs: Measure Life Study, August 25, 2009.

Where:

SF = (T_existing – T_eff) / 40°F * TRE

= see default Savings Factor table below

Where:

T_existing = Existing Full Fire Boiler Flue Gas Temperature as it exits the Stack

= 425F[434] (water, 81.9% eff) or custom

= 480F[3] (steam, 80.7% eff) or custom

T_eff = Efficient Full Fire Boiler Flue Gas Temperature as it exits the Stack

= 338°F (conventional economizer – Water Boiler)[435] or custom

= 365°F (conventional economizer – Steam Boiler)[436] or custom

= 280°F (condensing economizer – Water Boiler)[437] or custom

= 308°F (condensing economizer – Steam Boiler)[438] or custom

TRE = % efficiency increase for 40°F of stack temperature reduction

= 1%[439] or custom

Based on defaults provided above:

[434] Cleaver Brooks. March 2012, Boiler Efficiency Guide, Pg. 7, Figure 1.

[435] The minimum stack temperature for a non-condensing economizer is 250°F from Department of Energy (DOE). January 2012, Steam Tip Sheet #26A, Consider Installing a Condensing Economizer. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. The average temperature drop is assumed to be ½ way between the existing and efficient temperature minimum, ( 425°F + 250°F ) / 2 = 338°F.

[436] The minimum stack temperature for a non-condensing economizer is 250°F from Department of Energy (DOE). January 2012, Steam Tip Sheet #26A, Consider Installing a Condensing Economizer. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. The average temperature drop is assumed to be ½ way between the existing and efficient temperature minimum, ( 480°F + 250°F ) / 2 = 365°F.

[437] The minimum stack temperature for a condensing economizer is 250°F from Department of Energy (DOE). January 2012, Steam Tip Sheet #26A, Consider Installing a Condensing Economizer. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. The average temperature drop is assumed to be ½ way between the existing and efficient temperature minimum, ( 425°F + 135°F ) / 2 = 280°F.

[438] The minimum stack temperature for a condensing economizer is 250°F from Department of Energy (DOE). January 2012, Steam Tip Sheet #26A, Consider Installing a Condensing Economizer. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. The average temperature drop is assumed to be ½ way between the existing and efficient temperature minimum, ( 480°F + 135°F ) / 2 = 308°F.

[439] United States EPA, Climate Wise: Wise Rules for Industrial Efficiency, July 1998. The Wise Rules indicate savings range of 1-2% per 40°F reduction, so utilizing 1% is a conservative approach.

| Boiler Type | SF[440] | |
|---|---|---|
| | Conventional Economizer | Condensing Economizer |
| Hot Water Boiler | 2.19% average SF or custom | 3.63% average SF or custom |
| Steam Boiler | 2.88% average SF or custom | 4.31% average SF or custom |

MBH_In = Rated boiler input capacity, in MBH

= Actual

EFLH = Equivalent Full Load Hours for heating are provided in Section 4.4 HVAC End Use

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HVC-BECO-V01-150601**

[440] These average values should be utilized in absence of actual temperature data. An economizer with a zero temperature change between the existing and the efficient temperatures would not be installed, so these average values are conservative.

## 4.4.29 Stack Economizer for Boilers Serving Process Loads

### MEASURE DESCRIPTION

Stack economizers are designed to recover heat from hot boiler flue gasses. Recovered heat is used to preheat boiler feed water. This measure describes the retrofit of process boilers with stack economizers. Process boilers are defined as those used for industrial, manufacturing, or other non-HVAC applications. There is another, similar measure for boilers that serve HVAC loads.

This measure was developed to be applicable to the following program types: NC, TOS, RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify the economizer must be installed on a boiler exhaust stack. Heat captured by the economizer is to be used to pre-heat boiler feed water.

### DEFINITION OF BASELINE EQUIPMENT

The baseline boiler does not have an economizer installed.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for the boiler stack economizer is 15 years.[441]

### DEEMED MEASURE COST

The incremental and full measure cost for this measure is custom.

### DEEMED O&M COST ADJUSTMENTS

The O&M cost for this measure is custom.

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

[441] PA Consulting, Focus on Energy Evaluation, Business Programs: Measure Life Study, August 25, 2009.

**Algorithm**

**CALCULATION OF ENERGY SAVINGS**

**ELECTRIC ENERGY SAVINGS**

N/A

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

N/A

**NATURAL GAS SAVINGS**

Δtherms = SF * MBH_In * 8766 * UF / 100

Where:

SF = (T_existing – T_eff)/40°F * TRE

= see default Savings Factor table below

T_existing = Existing Full Fire Boiler Flue Gas Temperature as it exits the Stack

= 425F[442] (water, 81.9% eff per IL TRM) or custom

= 480F[3] (steam, 80.7% eff per IL TRM) or custom

T_eff = Efficient Full Fire Boiler Flue Gas Temperature as it exits the Stack

= 338°F (conventional economizer – Water Boiler)[443] or custom

= 365°F (conventional economizer – Steam Boiler)[444] or custom

= 280°F (condensing economizer – Water Boiler)[445] or custom

= 308°F (condensing economizer – Water Boiler)[446] or custom

[442] Cleaver Brooks. March 2012, Boiler Efficiency Guide, Pg. 7, Figure 1.

[443] The minimum stack temperature for a non-condensing economizer is 250°F from Department of Energy (DOE). January 2012, Steam Tip Sheet #26A, Consider Installing a Condensing Economizer. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. The average temperature drop is assumed to be ½ way between the existing and efficient temperature minimum, ( 425°F + 250°F ) / 2 = 338°F.

[444] The minimum stack temperature for a non-condensing economizer is 250°F from Department of Energy (DOE). January 2012, Steam Tip Sheet #26A, Consider Installing a Condensing Economizer. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. The average temperature drop is assumed to be ½ way between the existing and efficient temperature minimum, ( 480°F + 250°F ) / 2 = 365°F.

[445] The minimum stack temperature for a condensing economizer is 250°F from Department of Energy (DOE). January 2012, Steam Tip Sheet #26A, Consider Installing a Condensing Economizer. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. The average temperature drop is assumed to be ½ way between the existing and efficient temperature minimum, ( 425°F + 135°F ) / 2 = 280°F.

[446] The minimum stack temperature for a condensing economizer is 250°F from Department of Energy (DOE). January 2012,

TRE = % efficiency increase for 40°F of stack temperature reduction

= 1%[447] or custom

Based on defaults provided above:

| Boiler Type | SF[448] | |
|---|---|---|
| | Conventional Economizer | Condensing Economizer |
| Hot Water Boiler | 2.19% average SF or custom | 3.63% average SF or custom |
| Steam Boiler | 2.88% average SF or custom | 4.31% average SF or custom |

MBH_In = Rated boiler input capacity, in MBH

= Actual

8766 = Hours a year

UF = Utilization Factor

= 41.9%[449] or custom

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HVC-PECO-V01-150601**

Steam Tip Sheet #26A, Consider Installing a Condensing Economizer. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. The average temperature drop is assumed to be ½ way between the existing and efficient temperature minimum, ( 480°F + 135°F ) / 2 = 308°F.

[447] United States EPA, Climate Wise: Wise Rules for Industrial Efficiency, July 1998. The Wise Rules indicate savings range of 1-2% per 40°F reduction, so utilizing 1% is a conservative approach.

[448] These average values should be utilized in absence of actual temperature data. An economizer with a zero temperature change between the existing and the efficient temperatures would not be installed, so these average values are conservative.

[449] Work Paper – Tune up for Boilers serving Space Heating and Process Load by Resource Solutions Group, January 2012

## 4.4.30 Notched V Belts for HVAC Systems

**MEASURE DESCRIPTION**

This measure is for replacement of smooth v-belts in non-residential package and split HVAC systems with notched v-belts. Typically there is a v-belt between the motor and the supply air fan and/or return air fan in larger package and split HVAC systems (RTU).

In general there are two styles of grooved v-belts, notched and synchronous. The DOE defines each as follows;

**Notched V-Belts** - A notched belt has grooves or notches that run perpendicular to the belt's length, which reduces the bending resistance of the belt. Notched belts can use the same pulleys as cross-section standard V-belts. They run cooler, last longer, and are about 2% more efficient than standard V-belts.

**Synchronous Belts** - Synchronous belts (also called cogged, timing, positive-drive, or high-torque drive belts) are toothed and require the installation of mating grooved sprockets. These belts operate with a consistent efficiency of 98% and maintain their efficiency over a wide load range.

Smooth v-belts are usually referred to in five basic groups:

- "L" belts are low end belts that are for small, fractional horsepower motors and these are not used in RTUs.
- "A" and "B" belts are the two types typically used in RTUs. The "A" belt is a ½ inch width by 5/16 inch thickness and the "B" belt is larger, 21/32 inch wide and 12/32 inch thick so it can carry more power. V-belts come in a wide variety of lengths where 20 to 100 inches is typical.
- "C" and "D" belts are primarily for industrial applications with high power transmission requirements.
- V-belts are provided by various vendors. The notched version of these belts typically have an "X" added to the designation. For this HVAC fans notched v-belt Replacement measure, only the "A" and "B" v-belts are considered. A typical "A" v-belt is replaced by a notched "AX" v-belt and a "B" is replaced by a "BX." In general, smooth v-belts have an efficiency of 90% to 98% while notched v-belts have an efficiency of 95% to 98%. Because notched v-belts are more flexible they work with smaller diameter pulleys and they have less resistance to bending. Lower bending resistance increases the power transmission efficiency, lowers the waste heat, and allows the belt to last longer than a smooth belt.

Three research papers[450] [451] [452] show that the notched v-belt efficiency is 2% to 5% better than a typical smooth v-belt. A fourth paper by USDOE's Energy Efficiency and Renewable Energy[453] group reviewed most of the earlier literature and recommended using a conservative 2% efficiency improvement for energy savings for calculations.

For this measure it is assumed that upgrading a standard smooth v-belt with a new notched v-belt will result in a fan energy reduction of 2%.

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[450] "Gates Corporation Announces New EPDM Molded Notch V-Belts," The Gates Rubber Co., June 2010 (Assumed 3% efficiency improvement) https://ww2.gates.com/news/index.cfm?id=11296&show=newsitem&location_id=753&view=Gates

[451] "Synchronous Belt Drives Offer Low Cost Energy Savings," Baldor., February 2009. (attached in Reference Documents)

[452] "Energy Savings from Synchronous Belts," The Gates Rubber Co., February 2014. (Assumed 5% efficiency improvement) http://www.gates.com/~/media/Files/Gates/Industrial/Power%20Transmission/White%20Papers/Energy%20Savings%20from%20Synchronous%20Belt%20Drives.pdf

[453] "Motor System Tip Sheet #5, Replace V-Belts with Cogged or Synchronous Belt Drives," USDOE-EERE, September 2005. (Assumed 2% efficiency improvement) http://www1.eere.energy.gov/industry/bestpractices/pdfs/replace_vbelts_motor_systemts5.pdf

**DEFINITION OF EFFICIENT EQUIPMENT**

The Efficient Equipment is HVAC RTUs that have notched v-belts installed on the supply and/or return air fans.

**DEFINITION OF BASELINE EQUIPMENT**

The Baseline Equipment is HVAC RTUs that have smooth v-belts installed on the supply and/or return air fans (i.e. RTU does not already have a notched v-belt installed).

**DEEMED LIFETIME OF EFFICIENT EQUIPMENT**

A v-belt has a life based on fan run hours which varies by building type based primarily on occupancy schedule because the fans are required by code to operate continuously during occupied hours. The supply and return fans will also run a few hours during unoccupied hours for heating and cooling as needed. For the notched v-belt EUL calculation, the default hours[454] in the following table are used for a variety of building types and HVAC applications.

EUL = Belt Life / Occupancy Hours per year

Where:

Belt Life = 24,000 hours[455]

Occupancy Hours per year = values from Table below

The notched v-belt measure EUL is summarized by building type in the following table.

**Notched v-belt Effective Useful Life (EUL)**

| Building Type | Pumps & Fans (annual Hours of operation) | EUL (Years) |
|---|---|---|
| College/University | 4216 | 5.7 |
| Grocery | 5840 | 4.1 |
| Heavy Industry | 3585 | 6.7 |
| Hotel/Motel | 6872 | 3.5 |
| Light Industry | 2465 | 9.7 |
| Medical | 6871 | 3.5 |
| Office | 2301 | 10.4 |
| Restaurant | 4654 | 5.2 |

[454] ComEd Trm June 1, 2010 page 139. The Office hours is based upon occupancy from the eQuest model developed for EFLH, since it was agreed the ComEd value was too low.

[455] "DEER2014-EUL-table-update_2014-02-05.xlsx," Database for Energy Efficiency Resources (DEER), Deer 2014. www.deerresources.com (attached in Reference Documents)

| Building Type | Pumps & Fans (annual Hours of operation) | EUL (Years) |
|---|---|---|
| Retail/Service | 3438 | 7.0 |
| School(K-12) | 2203 | 10.9 |
| Warehouse | 3222 | 7.4 |
| Average=Miscellaneous | 4103 | 5.8 |

**DEEMED MEASURE COST**

A review of the Grainger online[456] pricing for "A," "B," "AX," and "BX" v-belts showed the incremental cost to upgrade to notched v-belts would result in a 28% price increase. The notched v-belt incremental cost is summarized in the table below:

**Notched V-belt Incremental Cost Summary**

| Smooth V-Belt Industry Number | Outside Length (Inches) | Dayton Smooth V-Belt* | Notched V-belt Industry Number | Dayton Notched v-belt* | Price Increase | % Increase |
|---|---|---|---|---|---|---|
| A30 (Item # 1A095) | 32 | $12.70 | AX29 (Item # 3GWU4) | $17.65 | $4.95 | 28% |
| B29 (Item # 6L208) | 32 | $16.75 | BX29 (Item # 5TXL4) | $23.23 | $6.48 | 28% |
| * Pricing based on Dayton Belts as found on Grainger Website 10/30/14 | | | | | | |

**DEEMED O&M COST ADJUSTMENTS**

N/A

**LOADSHAPE**

Loadshape C05 - Commercial Electric Heating and Cooling

**COINCIDENCE FACTOR**

N/A

[456] Grainger catalog on-line web-site for Dayton v-belt pricing
http://www.grainger.com/Grainger/ecatalog/N-1z0r596/Ntt-v-belts

**Algorithm**

**CALCULATION OF ENERGY SAVINGS**

**Electric Energy Savings**

$$\Delta kWh = kW_{connected} * Hours * ESF$$

Where:

$kW_{Connected}$ =kW of equipment is calculated using motor efficiency[457].

= (HP * 0.746 kW/HP* Load Factor)/Motor Efficiency

Load Factor =Motors are assumed to have a load factor of 80% for calculating KW if actual values cannot be determined[458]. Custom load factor may be applied if known.

Motor Efficiency = Actual motor efficiency shall be used to calculate KW. If not known a value from the motor efficiency refrence tables below should be used[459]

| Baseline Motor Efficiencies (EPACT) | | | | | | |
|---|---|---|---|---|---|---|
| Size HP | Open Drip Proof (ODP) | | | Totally Enclosed Fan-Cooled (TEFC) | | |
| | # of Poles | | | | | |
| | *6* | *4* | *2* | *6* | *4* | *2* |
| | Speed (RPM) | | | | | |
| | *1200* | *1800* | *3600* | *1200* | *1800* | *3600* |
| 1/8 | - | 44.00% | - | - | - | - |
| 1/6 | 57.50% | 62.00% | - | - | - | - |
| 1/4 | 68.00% | 68.00% | - | 68.00% | 64.00% | - |
| 1/3 | 70.00% | 70.00% | 72.00% | 70.00% | 68.00% | 72.00% |
| 1/2 | 78.50% | 80.00% | 68.00% | 72.00% | 74.00% | 68.00% |
| 3/4 | 77.00% | 78.50% | 74.00% | 77.00% | 75.50% | 74.00% |
| 1 | 80.00% | 82.50% | 75.50% | 80.00% | 82.50% | 75.50% |
| 1.5 | 84.00% | 84.00% | 82.50% | 85.50% | 84.00% | 82.50% |
| 2 | 85.50% | 84.00% | 84.00% | 86.50% | 84.00% | 84.00% |

[457] Note that kWConnected may be determined using various methodologies. The examples provided use rated HP and assumed load factor. Other methodologies include rated voltage and full load current with assumed load factor, or actual measured voltage and current.
[458] Com Ed TRM June 1, 2010
[459] Efficiency values for motors less than one HP taken from Baldor Electric Catalog 501: http://www.baldor.com/pdf/501_Catalog/CA501.pdf

| Baseline Motor Efficiencies (EPACT) | | | | | | |
|---|---|---|---|---|---|---|
| 3 | 86.50% | 86.50% | 84.00% | 87.50% | 87.50% | 85.50% |
| 5 | 87.50% | 87.50% | 85.50% | 87.50% | 87.50% | 87.50% |
| 7.5 | 88.50% | 88.50% | 87.50% | 89.50% | 89.50% | 88.50% |
| 10 | 90.20% | 89.50% | 88.50% | 89.50% | 89.50% | 89.50% |
| 15 | 90.20% | 91.00% | 89.50% | 90.20% | 91.00% | 90.20% |
| 20 | 91.00% | 91.00% | 90.20% | 90.20% | 91.00% | 90.20% |
| 25 | 91.70% | 91.70% | 91.00% | 91.70% | 92.40% | 91.00% |

| Efficient Motor Efficiencies (NEMA Premium) | | | | | | |
|---|---|---|---|---|---|---|
| Size HP | Open Drip Proof (ODP) | | | Totally Enclosed Fan-Cooled (TEFC) | | |
| | # of Poles | | | # of Poles | | |
| | 2 | 4 | 6 | 2 | 4 | 6 |
| | Speed (RPM) | | | Speed (RPM) | | |
| | 1200 | 1800 | 3600 | 1200 | 1800 | 3600 |
| 0.125 * | - | 44.00% | - | - | - | - |
| 1/6 | 57.50% | 62.00% | - | - | - | - |
| 1/4 | 68.00% | 68.00% | - | 68.00% | 64.00% | - |
| 1/3 | 70.00% | 70.00% | 72.00% | 70.00% | 68.00% | 72.00% |
| 1/2 | 78.50% | 80.00% | 68.00% | 72.00% | 74.00% | 68.00% |
| 3/4 | 77.00% | 78.50% | 74.00% | 77.00% | 75.50% | 74.00% |
| 1 | 82.50% | 85.50% | 77.00% | 82.50% | 85.50% | 77.00% |
| 1.5 | 86.50% | 86.50% | 84.00% | 87.50% | 86.50% | 84.00% |
| 2 | 87.50% | 86.50% | 85.50% | 88.50% | 86.50% | 85.50% |
| 3 | 88.50% | 89.50% | 85.50% | 89.50% | 89.50% | 86.50% |
| 5 | 89.50% | 89.50% | 86.50% | 89.50% | 89.50% | 88.50% |
| 7.5 | 90.20% | 91.00% | 88.50% | 91.00% | 91.70% | 89.50% |
| 10 | 91.70% | 91.70% | 89.50% | 91.00% | 91.70% | 90.20% |
| 15 | 91.70% | 93.00% | 90.20% | 91.70% | 92.40% | 91.00% |
| 20 | 92.40% | 93.00% | 91.00% | 91.70% | 93.00% | 91.00% |
| 25 | 93.00% | 93.60% | 91.70% | 93.00% | 93.60% | 91.70% |

Hours = When available, actual hours should be used. If actual hours are not available default hours[460] are provided in table below for HVAC fan operation which varies by building type:

| Building Type | Pumps & Fans (annual Hours of operation) |
|---|---|
| College/University | 4216 |
| Grocery | 5840 |
| Heavy Industry | 3585 |
| Hotel/Motel | 6872 |
| Light Industry | 2465 |
| Medical | 6871 |
| Office | 2301 |
| Restaurant | 4654 |
| Retail/Service | 3438 |
| School(K-12) | 2203 |
| Warehouse | 3222 |
| Average=Miscellaneous | 4103 |

ESF = Energy Savings Factor, the ESF for notched v-belt Installation is assumed to be 2%

[460] ComEd Trm June 1, 2010 page 139. The Office hours is based upon occupancy from the eQuest model developed for EFLH, since it was agreed the ComEd value was too low.

**EXAMPLE**

For example, an office building RTU with a 5 HP NEMA premium efficiency motor using the default hours of operation, motor load and 89.5% motor efficiency;

$\Delta kWh = kW_{connected} * Hours * ESF$

= ((HP * 0.746 kW/HP* Load Factor)/Motor Efficiency) * Hours * ESF

= ((5 HP * 0.746 kW/HP* 80%) / 89.5%) * 1766 Hours * 2%

= 117.8 kWh Savings

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$\Delta kW = kW_{connected} * ESF$

Where:

$kW_{Connected}$ = kW of equipment is calculated using motor efficiency.

= (HP *0 .746 kW/HP* Load Factor)/Motor Efficiency

Variables as provided above

**EXAMPLE**

For example, an office building RTU with a 5 HP NEMA premium efficiency motor using the default motor load and 89.5% motor efficiency;

$\Delta kW = kW_{connected} * ESF$

= ((HP * 0.746 kW/HP* Load Factor)/Motor Efficiency) * ESF

= ((5 HP * 0.746 kW/HP* 80%) / 89.5%) * 2%

= 0.0667 kW Savings

**NATURAL GAS SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-MSC-NVBE-V01-150601**

## 4.4.31 Small Business Furnace Tune-Up

### DESCRIPTION

This measure is for a natural gas Small Business furnace that provides space heating. The tune-up will improve furnace performance by inspecting, cleaning and adjusting the furnace and appurtenances for correct and efficient operation. Additional savings maybe realized through a complete system tune-up.

This measure was developed to be applicable to the following program types: Small business.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure an approved technician must complete the tune-up requirements[461] listed below:

- Measure combustion efficiency using an electronic flue gas analyzer
- Check and clean blower assembly and components per manufacturer's recommendations
- Where applicable Lubricate motor and inspect and replace fan belt if required
- Inspect for gas leaks
- Clean burner per manufacturer's recommendations and adjust as needed
- Check ignition system and safety systems and clean and adjust as needed
- Check and clean heat exchanger per manufacturer's recommendations
- Inspect exhaust/flue for proper attachment and operation
- Inspect control box, wiring and controls for proper connections and performance
- Check air filter and clean or replace per manufacturer's
- Inspect duct work connected to furnace for leaks or blockages
- Measure temperature rise and adjust flow as needed
- Check for correct line and load volts/amps
- Check thermostat operation is per manufacturer's recommendations (if adjustments made, refer to 'Small Commercial Programmable Thermostat Adjustment' measure for savings estimate)
- Perform Carbon Monoxide test and adjust heating system until results are within standard industry acceptable limits

### DEFINITION OF BASELINE EQUIPMENT

The baseline is furnace assumed not to have had a tune-up in the past 2 years.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for the tune up is 2 years.[462]

### DEEMED MEASURE COST

The incremental cost for this measure should be the actual cost of tune up.

---

[461] American Standard Maintenance for Indoor Units: http://www.americanstandardair.com/owner-support/maintenance.html
[462] Act on Energy Commercial Technical Reference Manual No. 2010-4, 9.2.3 Gas Forced-Air Furnace Tune-up.

### DEEMED O&M COST ADJUSTMENTS

There are no expected O&M savings associated with this measure.

### LOADSHAPE

Loadshape C04 - Commercial Electric Heating

### COINCIDENCE FACTOR

N/A

## Algorithms

### CALCULATION OF ENERGY SAVINGS

### ELECTRIC ENERGY SAVINGS

$\Delta kWh = \Delta Therms * F_e * 29.3$

Where:

$\Delta Therms$ = as calculated below

$F_e$ = Furnace Fan energy consumption as a percentage of annual fuel consumption

= 3.14%[463]

29.3 = kWh per therm

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS SAVINGS

$\Delta therms = (Capacity * EFLH * (((Effbefore + Ei)/ Effbefore) – 1)) / 100,000$

Where:

Capacity = Furnace gas input size (Btu/hr)

[463] $F_e$ is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% $F_e$. See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

= Actual

EFLH = Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use

Effbefore = Efficiency of the furnace before the tune-up

= Actual

EI = Efficiency Improvement of the furnace tune-up measure

= Actual

100,000 = Converts Btu to therms

**EXAMPLE**

A 200 kBtu furnace in a Rockford low rise office records an efficiency prior to tune up of 82% AFUE and a 1.8% improvement in efficiency are tune up:

$$\Delta therms = (200{,}000 * 1428 * (((0.82 + 0.018)/ 0.82) - 1)) /100{,}000$$

$$= 62.3 \text{ therms}$$

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE Code: CI-HVC-FTUN-V01-150601**

## 4.4.32 Combined Heat and Power

### DESCRIPTION

The Combined Heat and Power (CHP) measure can provide energy savings within the State of Illinois through the development and operation of CHP projects. This measure is applicable for Conventional or Topping Cycle CHP systems, as well as Waste Heat-to-Power (WHP) or Bottoming Cycle CHP systems. The measure will reduce the total Btu's of energy required to meet the end use needs of the facility.

It is recognized that CHP system design and configuration may be complex, and as such the calculation of energy savings may not be reducible to the equations within this measure. In such cases a more comprehensive engineering and financial analysis may be developed that more accurately incorporates the attributes of complex CHP configurations such as variable-capacity systems, and partial combined-cycle CHP systems. Where noted, the use of values that are determined through an external engineering analysis may be substituted by agreement between the participant, the program administrator and independent evaluator. This substitution of values does not eliminate ex post evaluation risk (retroactive adjustments to savings claims) that exists when using custom inputs.

This measure was developed to be applicable to the following program types: Retrofit (RF), New Construction (NC). If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

Conventional or Topping Cycle CHP is defined as an integrated system that is located at or near the building or facility (on-site, on the customer side of the meter) that utilizes a prime mover (reciprocating engine, gas turbine, micro-turbine, fuel cell, boiler/steam turbine combination) for the purpose of generating electricity and useful thermal energy (such as steam, hot water, or chilled water) where the primary function of the facility where the CHP is located is not to generate electricity for use on the grid. An eligible system must demonstrate a minimum total system efficiency of 60% (HHV)[464] with at least 20% of the system's total useful energy output in the form of useful thermal energy on an annual basis.

*Measuring and Calculating Conventional CHP Total System Efficiency:*

CHP efficiency is calculated using the following equation:

$$CHP_{\ Efficiency}(HHV) = \frac{\left[CHP_{thermal}\ \left(\frac{kBtu}{yr}\right) + E_{CHP}\ \left(\frac{kWh}{yr}\right) * 3.412\ \left(\frac{kBtu}{kWh}\right)\right]}{F_{totalCHP}\left(\frac{kBtu}{yr}\right)}$$

Where:

$CHP_{thermal}$ = Useful annual thermal energy output from the CHP system, defined as the annual thermal energy output of the CHP system that is actually recovered and utilized in the facility/process.

$E_{CHP}$ = Useful annual electricity output produced by the CHP system, defined as the annual electric energy output of the CHP system that is actually utilized to replace purchased electricity required to meet the requirements of the facility/process.

$F_{totalCHP}$ = Total annual fuel consumed by the CHP system

[464] Higher Heating Value (HHV): refers to the heating value of the fuel and is defined as the total thermal energy available, including the heat of condensation of water vapors,resulting from complete combustion of the fuel
versus the Lower Heating Value (LHV) which assumes the heat of condensation is not available

For further definition of the terms, please see "Calculation of Energy Savings" Section below.

Waste Heat-to-Power or Bottoming Cycle CHP is defined as an integrated system that is located at or near the building or facility (on-site, on the customer side of the meter) that does one of the following:

- Utilizes exhaust heat from an industrial/commercial process to generate electricity (except for exhaust heat from a facility whose primary purpose is the generation of electricity for use on the grid); or
- Utilizes the pressure drop in an industrial/commercial facility to generate electricity through a backpressure steam turbine where the facility normally uses a pressure reducing valve (PRV) to reduce the pressure in their facility; or
- Utilizes the pressure reduction in natural gas pipelines (located at natural gas compressor stations) before the gas is distributed through the pipeline to generate electricity, provided that the conversion of energy to electricity is achieved without using additional fossil fuels.

Since these types of systems utilize waste heat as their fuel, they do not have to meet any specific total system efficiency level (assuming they use no additional fossil fuel in their operation) If additional fuel is used onsite, it should be accounted for using the following methodology:

- Treat the portion of Waste-Heat-to-Power that does not require any additional fuel using the Waste-Heat-to-Power methodology outlined in this document.
- Treat the portion of Waste-Heat-to-Power that requires additional fuel (if natural gas) using the Conventional CHP methodology outlined in this document. If the additional fuel is not natural gas, custom carbon equivalency calculations would be needed – refer to section "Calculation of Energy Savings" for more details.
- Add the energy savings together.

These systems may export power to the grid.

**DEFINITION OF BASELINE EQUIPMENT**

Electric Baseline: The baseline facility would be a facility that purchases its electric power from the grid.

Heating Baseline (for CHP applications that displace onsite heat): The baseline equipment would be the boiler/furnace operating onsite, or a boiler/furnace meeting the baseline equipment defined in the High Efficiency Boiler (Section 4.4.10)/Furnace (Section 4.4.11) measures of this TRM.

Cooling Baseline (for CHP applications that displace onsite cooling demands): The baseline equipment would be the chiller (or chillers) operating onsite, or a chiller (or chillers) meeting the definition of baseline equipment defined in the Electric Chiller (Section 4.4.6) measure of this TRM.

Facilities that use biogas or waste gas: Facilities that use (but are not purchasing) biogas or waste gas that is not otherwise used, whether they are using biogas or waste gas only or a combination of biogas or waste gas and natural gas to meet their energy demands are also eligible for this measure. If additional fuel is purchased to power the CHP system, then the additional natural gas should be taken into account using the following methodology:

- Treat the portion of CHP system that does not require any additional fuel, or that requires additional fuel that would otherwise be wasted (e.g. flared), using the Waste-Heat-to-Power methodology outlined in this document.
- Treat the portion of CHP that requires additional fuel (if natural gas) using the Conventional CHP methodology outlined in this document. If the additional fuel is not natural gas, custom carbon equivalency calculations would be needed – refer to section "Calculation of Energy Savings" for more details.
- Add the energy savings together.

Consumption of any biogas or waste gas that would not otherwise being wasted (*e.g.,* flared) will be accounted for

in the overall net BTU savings calculations the same as for purchased natural gas.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

Measure life is a custom assumption, dependent on the technology selected and the system installation.

### DEEMED MEASURE COST

Custom installation and equipment cost will be used. These costs should include the cost of the equipment and the cost of installing the equipment. Equipment costs include, but are not limited to: prime mover, heat recovery system(s), exhaust gas treatment system(s), controls, and any interconnection/electrical connection costs.

The installations costs include labor and material costs such as, but not limited to: labor costs, materials such as ductwork, piping, and wiring, project and construction management, engineering costs, commissioning costs, and other fees.

Measure costs will also include the present value of expected maintenance costs over the life of the CHP system.

### LOADSHAPE

Use Custom Loadshape. The loadshape should be obtained from the actual CHP operation strategy, based on the On-Peak and Off-Peak Energy definitions specified in Table 3.3 of "Section 3.5 Electrical Loadshapes" of the TRM.

### COINCIDENCE FACTOR

Custom coincidence factor will be used. Actual value based on the CHP operation strategy will be used.

## Algorithm

### CALCULATION OF ENERGY SAVINGS

### i) Conventional or Topping Cycle CHP Systems:

***Step 1: (Calculating Total Annual Source Fuel Savings in Btus)***

The first step is to calculate the total annual source fuel savings associated with the CHP installation, in order to ensure the CHP project produces positive total annual source fuel savings (i.e. reduction in source Btus):

$S_{FuelCHP}$ = Annual fuel savings (Btu) associated with the use of a Conventional CHP system to generate the useful electricity output (kWh, converted to Btu) and useful thermal energy output (Btu) versus the use of the equivalent electricity generated and delivered by the local grid and the equivalent thermal energy provided by the onsite boiler/furnace.

$= (F_{grid} + F_{thermalCHP}) - F_{total\ CHP}$

Where:

$F_{grid}$ = Annual fuel in Btu that would have been used to generate the useful electricity output of the CHP system if that useful electricity output was provided by the local utility grid.

$= E_{CHP} * H_{grid}$

Where:

$E_{CHP}$ = Useful annual electricity output produced by the CHP system, defined as the annual electric energy output of the CHP system that is actually utilized to replace purchased electricity required to meet the requirements of the facility/process. [465]

= ( $CHP_{capacity}$ * Hours ) - $E_{Parasitic}$

$CHP_{capacity}$ = CHP nameplate capacity

= Custom input

Hours = Annual operating hours of the system

= Custom input

$E_{parasitic}$ = The electricity required to operate the CHP system that would otherwise not be required by the facility/process

= Custom input

$H_{grid}$ = Heat rate of the grid in Btu/kWh, based on the average fossil heat rate for the EPA eGRID subregion, adjusted to take into account T&D losses.

For systems operating less than 6,500 hrs per year:

Use the Non-baseload heat rate provided by EPA eGRID for RFC West region for ComEd territory (including independent providers connected to RFC West), and SERC Midwest region for Ameren territory (including independent providers connected to SERC Midwest)[466]. Also include any line losses.

For systems operating more than 6,500 hrs per year:

Use the All Fossil Average heat rate provided by EPA eGRID for RFC West region for ComEd territory (including independent providers connected to RFC West), and SERC Midwest region for Ameren territory (including independent providers connected to SERC Midwest). Also include any line losses.

$F_{thermalCHP}$ = Annual fuel in Btu that would have been used on-site by a boiler/furnace to provide the useful thermal energy output of the CHP system. [467]

= $CHP_{thermal}$ / $Boiler_{eff}$ (or $CHP_{thermal}$ / $Furnace_{eff}$)

[465] For complex systems this value may be obtained from a CHP System design/financial analysis study.

[466] Refer to EPA eGRID data http://www.epa.gov/chp/documents/fuel_and_co2_savings.pdf, page 24 and http://www.epa.gov/cleanenergy/documents/egridzips/eGRID_9th_edition_V1-0_year_2010_Summary_Tables.pdf, page 9. Current values are:

- Non-Baseload RFC West: 9,811 Btu/kWh * (1 + Line Losses)
- Non-Baseload SERC Midwest: 10,511 Btu/kWh * (1 + Line Losses)
- All Fossil Average RFC West: 10,038 Btu/kWh * (1 + Line Losses)
- All Fossil Average SERC Midwest: 10,364 Btu/kWh * (1 + Line Losses)

[467] For complex systems this value may be obtained from a CHP System design/financial analysis study.

$CHP_{thermal}$ = Useful annual thermal energy output from the CHP system, defined as the annual thermal energy output of the CHP system that is actually recovered and utilized in the facility/process.

= Custom input

$Boiler_{eff}$ /$Furnace_{eff}$= Efficiency of the on-site Boiler/Furnace that is displaced by the CHP system or if unknown, the baseline equipment value stated in the High Efficiency Boiler (Section 4.4.10) measure or High Efficiency Furnace (Section 4.4.11) measure in this TRM. .

= Custom input

$F_{total\ CHP}$ = Total fuel in Btus consumed by the CHP system

= Custom input

***Step 2: (Savings Allocation to Program Administrators for Purposes of Assessing Compliance with Energy Savings Goals (Not for Use in Load Reduction Forecasting))***

Savings claims are a function of the electric output of the CHP system ($E_{CHP}$), the used thermal output of the CHP system ($F_{thermalCHP}$), and the CHP system efficiency ($CHP_{Eff}$(HHV)). The percentages of electric output and used thermal output that can be claimed also differ slightly depending on whether the project was included in both electric[468] and gas[469] Energy Efficiency Portfolio Standard (EEPS)[470] efficiency programs, only an electric EEPS program or only a gas EEPS program. The tables below provide the specific percentages of electric and/or thermal output that can be claimed under each of those three scenarios. These percentages apply only to cases in which natural gas is the fuel used by the CHP system. Saving estimates for systems using other fuels should be calculated on a custom basis. If the waste heat recovered from the CHP system is offsetting electric equipment, such as an absorption chiller offsetting an electric chiller, then the net change in electricity consumption associated with the electric equipment should be added to the allocated electric savings.

1) For systems participating in both electric EEPS and gas EEPS programs:

| CHP Annual System Efficiency (HHV) | Allocated Electric Savings | Allocated Gas Savings |
|---|---|---|
| **60%** | 65% of $E_{CHP}$ (kWh) | No gas savings |
| **>60% to 65%** | 65% of $E_{CHP}$ (kWh) + one percentage point increase for every one percentage point increase in CHP system efficiency (max 70% of $E_{CHP}$ | No gas Savings |

[468] 220 ILCS 5/8-103; 220 ILCS 5/16-111.5B
[469] 220 ILCS 5/8-104
[470] As used in this measure characterization, EEPS programs are defined as those energy efficiency programs implemented pursuant to Sections 8-103, 8-104, and 16-111.5B of the Illinois Public Utilities Act. Technically, EEPS programs pertain to energy efficiency programs implemented pursuant to 220 ILCS 5/8-103 and 220 ILCS 5/8-104. However, for simplicity in presentation, this measure defines EEPS programs as also including those programs implemented pursuant to 220 ILCS 5/16-111.5B (these programs are funded through the same energy efficiency riders established pursuant to Section 8-103).

| CHP Annual System Efficiency (HHV) | Allocated Electric Savings | Allocated Gas Savings |
|---|---|---|
| | in kWh) | |
| **>65%** | 70% of $E_{chp}$ (kWh) | 2.5% of $F_{thermal}$ (useful thermal output of the CHP system) for every one percentage point increase in CHP system efficiency above 65%. |

Example: System with measured annual system efficiency (HHV) of 70%: Electric savings (kWh) = 70% of $E_{CHP}$ measured over 12 months, and Gas savings (therms) = 12.5% of $F_{thermal}$ measured over 12 months (70% - 65% = 5 X 2.5% = 12.5%)

2) For systems participating in only an electric EEPS program:

| CHP Annual System Efficiency (HHV) | Allocated Electric Savings | Allocated Gas Savings |
|---|---|---|
| **60%** | 65% of $E_{CHP}$ (useful electric output of CHP system in kWh) | No gas Savings |
| **Greater than 60%** | 65% + one percentage point increase for every one percentage point increase in CHP system efficiency (no max) | No gas Savings |

Example: System with measured annual fuel use efficiency of 75%: Electric savings (kWh) = 65% + 15% = 80% of $E_{CHP}$ measured over 12 months (15% = 1% for every 1% increase in system efficiency). No gas savings (therms).

3) For systems participating in only a gas EEPS program:

| CHP Annual System Efficiency (HHV) | Allocated Electric Savings | Allocated Gas Savings |
|---|---|---|
| **60% or greater** | No electric savings | 2.5% of Fthermal (useful thermal output of the CHP system) for every one percentage point increase in CHP system efficiency above 60%. |

Example: System with measured annual system efficiency (HHV) of 70%: No Electric savings (kWh). Gas savings (therms) = 25% of $F_{thermal}$ measured over 12 months (70% - 60% = 10 X 2.5% = 25%)

Conventional or topping cycle CHP systems virtually always require an increase in the use of fuel on-site in order to produce electricity. Different jurisdictions and experts across the country have employed and/or put forward a variety of approaches[471] to address how increased on-site fuel consumption should be reflected in the attribution of electric savings to CHP systems. The approach reflected in the tables above is generally consistent – for CHP systems consuming natural gas – with approaches recently put forward by the Southwest Energy Efficiency Project (SWEEP) and Institute for Industrial Productivity (IIP) that determine reduced electric savings based on the

[471] Approaches range from ignoring the increased gas use entirely (i.e., no "penalty") to applying approximately 40-60% "penalties", depending on the CHP efficiency and based on the equivalent grid kWh that the increased gas use represents.

equivalent amount of carbon dioxide generated from the increased fuel used[472].

There are a variety of ways one could treat the potential for gas utilities to claim savings from CHP projects in their EEPS portfolios. For projects in which a natural gas EEPS program is involved, the tables above treat savings from CHP installations in two steps: (1) a fuel-switch from electricity to natural gas (i.e. using more natural gas to eliminate the need to generate as much electricity on the grid); and (2) possible increases in CHP efficiency above a "benchmark" level. When both electric EEPS and natural gas EEPS programs are involved in a project, the program administrator claims all the electricity savings associated with a fuel-switch up to a "benchmark" 65% efficient CHP system. All the savings associated with increasing CHP efficiencies above that benchmark level are allocated to natural gas (e.g. if the CHP efficiency is 75%, the natural gas savings associated with an increase in CHP efficiency from 65% to 75% are allocated to natural gas). That is consistent with the notion that CHP efficiency typically increases primarily by increasing the use of the thermal output of the system (increasing the displacement of baseline gas use). For projects that involve only a natural gas EEPS program, the "benchmark" above which the gas utility can claim savings is lowered to 60%.

## ii) Waste-Heat-to-Power CHP Systems :

**ELECTRIC ENERGY SAVINGS:**

$$\Delta kWh = E_{CHP}$$

Where:

$E_{CHP}$ = Useful annual electricity output produced by the CHP system, defined as the annual electric energy output of the CHP system that is actually utilized to replace purchased electricity required to meet the requirements of the facility/process.

= Custom input

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$$\Delta kW = CF * CHP_{capacity}$$

Where:

CF = Summer Coincidence factor. This factor should also consider any displaced chiller capacity[473]

= Custom input

[472] Consider, for example, a hypothetical CHP system that produces 5 million kWh annually, consumes 50 million kBtu of gas annual to generate that electricity (i.e. electric efficiency of approximately 34.8% HHV), reduces on-site gas use for space heating by 26 million kBtu of gas (i.e. equivalent to approximately 81.5% CHP thermal output utilization displacing gas used in a 70% efficient space heating boiler) and has a total annual CHP efficiency of 70.6% HHV. In this example, the net increase in on-site gas use is 24 million kBtu. At a carbon dioxide emission rate of 53.06 kg/MMBtu for burning natural gas, that translates to an increase in on-site carbon dioxide emissions of 1404 tons per year. At an estimated marginal emission rate of 1.098 tons of carbon dioxide per MWh in Illinois, that is equivalent to electric grid production of approximately 1.28 million kWh, or penalty of about 25.6% of the CHP system's electrical output if a precise calculation of carbon equivalency was utilitized to assign savings. In comparison, the simplified table above would entitle an electric utility to claim savings equal to 75.6% of the electric output (i.e. a penalty of 24.4% of electrical output) if it was the only utility promoting the system. In a gas and electric example, the electric savings claimed would be 70% of the production (a penalty of 30% of the CHP system's electrical output) and 12.5% of the recovered thermal output, equivalent to 2.23 million kBtu. The difference between the electric only scenario and the electric and gas, on the electric side, is 5% of the electric output or 250,000 kWh, which would require 2.45 million kBtu input at an efficiency of 34.8% HHV.

[473] If some or all of the existing electric chiller peak demand is no longer needed due to new waste heat powered chillers (e.g., absorption), the coincidence factor should be adjusted appropriately.

$CHP_{Capacity}$ = CHP nameplate capacity

= Custom input

**NATURAL GAS ENERGY SAVINGS:**

$$\Delta Therms = F_{thermalCHP} \div 100{,}000$$

Where:

$F_{thermalCHP}$ = Net savings in annual purchased fuel in Btu, if any, that would have been used on-site by a boiler/furnace to provide some or all of the useful thermal energy output of the CHP system[474].

100,000 = Conversion factor for Btu to therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

Custom estimates of maintenance costs that will be incurred for the life of the measure will be used**.** Maintenance costs vary with type and size of the prime mover. These costs include, but are not limited to:

- Maintenance labor
- Engine parts and materials such as oil filters, air filters, spark plugs, gaskets, valves, piston rings, electronic components, etc. and consumables such as oil
- Minor and major overhauls

For screening purposes, the US EPA has published resource guides that provide average maintenance costs based on CHP technology and system size[475].

**COST-EFFECTIVENESS SCREENING AND LOAD REDUCTION FORECASTING**

For the purposes of forecasting load reductions due to CHP projects per Section 16-111.5B, changes in site energy use at the customer's meter – reduced consumption of utility provided electricity – adjusted for utility line losses (at-the-busbar savings), customer switching estimates, NTG, and any other adjustment factors deemed appropriate, should be used.

For the purposes of screening a CHP measure application for cost-effectiveness, changes in site energy use – reduced consumption of utility provided electricity and the net change in consumption of fuel – should be used. In general, the benefit and cost components used in evaluating the cost-effectiveness of a CHP project would include at least the following terms:

Benefits: $E_{CHP} + \Delta kW + F_{thermal_CHP}$

Costs: $F_{total_CHP} + CHP_{COSTS} + O\&M_{COSTS}$

Where:

[474] In most cases, it is expected that waste-heat-to-power systems will not provide any new net useful thermal energy output, since the CHP system will be driven by thermal energy that was otherwise being wasted. If additional natural gas or other purchased energy is used onsite, it should be properly accounted for.

[475] "EPA Combined Heat and Power Partnership Resources" Oct 07, 2014, http://www.epa.gov/chp/resources.html in the document "Catalog of CHP technologies" http://www.epa.gov/chp/documents/catalog_chptech_full.pdf pages 2-16, 3-14, 4-14, 5-14, and 6-16.

$CHP_{Costs}$ = CHP equipment and installation costs as defined in the "Deemed Measure Costs" section

$O\&M_{Costs}$ = CHP operations and maintenance costs as defined in the "Deemed O&M Cost Adjustment Calculation" section

**MEASURE CODE: CI-HVC-CHAP-V01-150601**

## 4.5 Lighting End Use

The commercial lighting measures use a standard set of variables for hours or use, waste heat factors, coincident factors and HVAC interaction effects. This table has been developed based on information provided by the various stakeholders. For ease of review, the table is included here and referenced in each measure.

| Building Type | Fixture Annual Operating Hours[476] | Screw based bulb Annual Operating hours[477] | Waste Heat Cooling Energy WHFe[478] | Waste Heat Cooling Demand WHFd[479] | Coincid-ence Factor CF[480] | Waste Heat Gas Heating IFTherms [481] | Waste Heat Electric Resistance Heating IFkWh[482] | Waste Heat Electric Heat Pump Heating IFkWh |
|---|---|---|---|---|---|---|---|---|
| Assisted Living | 5,950 | 5,950 | 1.25 | 1.50 | 0.75 | 0.022 | 0.497 | 0.248 |
| College | 3,540 | 2,588 | 1.32 | 1.46 | 0.56 | 0.096 | 2.194 | 1.097 |
| Convenience Store | 5,802 | 3,650 | 1.34 | 1.53 | 0.69 | 0.022 | 0.504 | 0.252 |
| Elementary School | 2,422 | 2,118 | 1.31 | 1.40 | 0.22 | 0.028 | 0.634 | 0.317 |
| Garage | 3,540 | 3,540 | 1.00 | 1.00 | 1.00 | 0.000 | 0.000 | 0.000 |
| Garage, 24/7 lighting | 8,766 | 8,766 | 1.00 | 1.00 | 1.00 | 0.000 | 0.000 | 0.000 |
| Grocery | 5,802 | 3,650 | 1.14 | 1.35 | 0.69 | 0.024 | 0.553 | 0.276 |
| Healthcare Clinic | 5,095 | 4,207 | 1.34 | 1.47 | 0.75 | 0.010 | 0.218 | 0.109 |
| High School | 4,311 | 2,327 | 1.25 | 1.44 | 0.22 | 0.025 | 0.571 | 0.285 |
| Hospital - CAV no econ | 6,038 | 4,207 | 1.55 | 1.80 | 0.75 | 0.014 | 0.317 | 0.158 |

[476]Fixtures hours of use are primarily derived from the default EPY4 values developed for ComEd based on DEER 2005, DEER 2008, EPY1 and EPY2 evaluation results. 'Lighting intro wp.doc'. Values for office, grocery, light industry, restaurant, retail/service and warehouse are an average of the EPY4 values and AmerEn Missouri, March 2011 Final Report: Evaluation of Business Energy Efficiency Program Custom and Standard Incentives. Hotel/Motel common areas is the DEER 2008 average across all non-guest room spaces and guest rooms is the average of hotel and motel guest room values from DEER 2008. Elementary School is from Ameren Missouri evaluation results. Multi-family common area value based on Focus on Energy Evaluation, ACES Deemed Savings Desk Review, November 2010. Miscellaneous is an average of all indoor spaces.

[477] Hours of use for screw based bulbs are derived from DEER 2008 by building type for cfls. Garage, exterior and multi-family common area values are from the Hours of Use Table in this document. Miscellaneous is an average of interior space values. Some building types are averaged when DEER has two values: these include office, restaurant and retail. Healthcare clinic uses the hospital value.

[478] The Waste Heat Factor for Energy is developed using EQuest models for various building types averaged across 5 climate zones for Illinois. Exterior and garage values are 1, miscellaneous is an average of all indoor spaces.

[479] Waste Heat Factor for Demand are not yet complete.

[480]Coincident diversity factors are from the EPY4 values developed for ComEd based on DEER 2005, DEER 2008, EPY1 and EPY2 evaluation results. Miscellaneous value for Coincident Diversity Factor is from DEER 2008.

[481] IF Therms value is developed using EQuest models consistent with methodology for Waste Heat Factor for Energy.

[482] Electric heat penalty assumptions are based on converting the IFTherm multiplier value in to kWh and then applying relative heating system efficiencies. The gas efficiency was assumed to be 78% AFUE based upon standard TRM assumption for existing unit average efficiency, and the electric resistance is assumed to be 100%:
IFElectricHeat = IFTherms * 29.3 kWh/therm * 78% (Gas Heating Equipment Efficiency) / 100% (Electric Resistance Efficiency)

| Building Type | Fixture Annual Operating Hours[476] | Screw based bulb Annual Operating hours[477] | Waste Heat Cooling Energy WHFe[478] | Waste Heat Cooling Demand WHFd[479] | Coincid-ence Factor CF[480] | Waste Heat Gas Heating IFTherms[481] | Waste Heat Electric Resistance Heating IFkWh[482] | Waste Heat Electric Heat Pump Heating IFkWh |
|---|---|---|---|---|---|---|---|---|
| Hospital - CAV econ | 6,038 | 4,207 | 1.37 | 1.80 | 0.75 | 0.014 | 0.318 | 0.159 |
| Hospital - VAV econ | 6,038 | 4,207 | 1.47 | 1.06 | 0.75 | 0.008 | 0.173 | 0.086 |
| Hospital - FCU | 6,038 | 4,207 | 1.50 | 1.37 | 0.75 | 0.001 | 0.028 | 0.014 |
| Manufacturing Facility | 5,041 | 2,629 | 1.03 | 1.38 | 0.89 | 0.011 | 0.257 | 0.128 |
| MF - High Rise - Common | 5,950 | 5,950 | 1.37 | 1.42 | 0.75 | 0.050 | 1.153 | 0.577 |
| MF - Mid Rise | 5,950 | 5,950 | 1.15 | 1.50 | 0.75 | 0.022 | 0.505 | 0.253 |
| Hotel/Motel - Guest | 777 | 777 | 1.17 | 1.55 | 0.21 | 0.024 | 0.539 | 0.269 |
| Hotel/Motel - Common | 5,311 | 4,542 | 1.20 | 1.56 | 0.21 | 0.007 | 0.164 | 0.082 |
| Movie Theater | 5,475 | 5,475 | 1.22 | 1.59 | 0.75 | 0.033 | 0.762 | 0.381 |
| Office - High Rise - CAV no econ | 4,439 | 3,088 | 1.52 | 1.42 | 0.66 | 0.019 | 0.440 | 0.220 |
| Office - High Rise - CAV econ | 4,439 | 3,088 | 1.48 | 1.52 | 0.66 | 0.019 | 0.433 | 0.216 |
| Office - High Rise - VAV econ | 4,439 | 3,088 | 1.35 | 1.58 | 0.66 | 0.020 | 0.453 | 0.227 |
| Office - High Rise - FCU | 4,439 | 3,088 | 1.31 | 2.19 | 0.66 | 0.011 | 0.252 | 0.126 |
| Office - Low Rise | 4,439 | 3,088 | 1.46 | 1.59 | 0.66 | 0.022 | 0.494 | 0.247 |
| Office - Mid Rise | 4,439 | 3,088 | 1.34 | 1.41 | 0.66 | 0.021 | 0.489 | 0.244 |
| Religious Building | 1,664 | 1,664 | 1.48 | 1.44 | 0.66 | 0.017 | 0.396 | 0.198 |
| Restaurant | 3,673 | 4,784 | 1.36 | 1.33 | 0.80 | 0.025 | 0.567 | 0.284 |
| Retail - Department Store | 4,719 | 2,935 | 1.24 | 1.49 | 0.83 | 0.022 | 0.502 | 0.251 |
| Retail - Strip Mall | 4,719 | 2,935 | 1.24 | 1.50 | 0.83 | 0.020 | 0.463 | 0.232 |
| Warehouse | 4,746 | 4,293 | 1.09 | 1.46 | 0.70 | 0.023 | 0.535 | 0.267 |
| Unknown | 4,683 | 3,612 | 1.31 | 1.53 | 0.66 | 0.023 | 0.524 | 0.262 |
| Exterior | 4,903 | 4,903 | 1.00 | 1.00 | 0.00 | 0.000 | 0.000 | 0.000 |
| Low-Use Small Business | 2,954 | 2,954 | 1.31 | 1.53 | 0.66 | 0.023 | 0.524 | 0.262 |
| Uncooled Building | Varies | varies | 1.00 | 1.00 | 0.66 | 0.014 | 0.320 | 0.160 |
| Refrigerated Cases | 5,802 | n/a | 1.29 | 1.29 | 0.69 | 0.000 | 0.000 | 0.000 |

| Building Type | Fixture Annual Operating Hours[476] | Screw based bulb Annual Operating hours[477] | Waste Heat Cooling Energy WHFe[478] | Waste Heat Cooling Demand WHFd[479] | Coincid-ence Factor CF[480] | Waste Heat Gas Heating IFTherms[481] | Waste Heat Electric Resistance Heating IFkWh[482] | Waste Heat Electric Heat Pump Heating IFkWh |
|---|---|---|---|---|---|---|---|---|
| Freezer Cases | 5,802 | n/a | 1.50 | 1.5 | 0.69 | 0.000 | 0.000 | 0.000 |

## 4.5.1 Commercial ENERGY STAR Compact Fluorescent Lamp (CFL)

### DESCRIPTION

A low wattage ENERGY STAR qualified compact fluorescent screw-in bulb (CFL) is installed in place of a baseline screw-in bulb. This characterization assumes that the CFL is installed in a commercial location. If the implementation strategy does not allow for the installation location to be known a deemed split should be used. For Residential targeted programs (e.g. an upstream retail program), a deemed split of 96% Residential and 4% Commercial assumptions should be used[483], and for Commercial targeted programs a deemed split of 4% Residential and 96% Commercial should be used[484].

Federal legislation stemming from the Energy Independence and Security Act of 2007 (EISA) required all general-purpose light bulbs between 40W and 100W to be approximately 30% more energy efficient than current incandescent bulbs. Production of 100W, standard efficacy incandescent lamps ended in 2012 followed by restrictions on 75W in 2013 and 60W and 40W in 2014. The baseline for this measure has therefore become bulbs (improved incandescent or halogen) that meet the new standard.

Finally, a provision in the EISA regulations requires that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in essence making the baseline equivalent to a current day CFL. Therefore the measure life (number of years that savings should be claimed) should be reduced once the assumed lifetime of the bulb exceeds 2020. Due to expected delay in clearing retail inventory and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until mid-2020.

This measure was developed to be applicable to the following program types: TOS, NC, RF. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the high-efficiency equipment must be a standard ENERGY STAR qualified compact fluorescent lamp.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be an EISA qualified incandescent or halogen as provided in the table provided in the Electric Energy Savings section.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life (number of years that savings should be claimed) should be calculated by dividing the rated life of the bulb (10,000 hours[485]) by the run hours. For example using Miscellaneous at 4,589 hours would give 2.2 years. When the number of years exceeds June 2020, the number of years to that date should be used.

### DEEMED MEASURE COST

The incremental capital cost assumption for all bulbs under 2600 lumens is $1.25, from June 2014 – May 2015, $1.6 from June 2015 to May 2016 and $1.70 from June 2017 to May 2018[486].

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[483] RES v C&I split is based on a weighted (by sales volume) average of ComEd PY4-6 and Ameren PY5-6 in store intercept survey results.

[484] Based upon final weighted (by sales volume) average of the BILD program (ComEd's commercial lighting program) for PY 4 and PY5 and PY6.

[485] Energy Star bulbs have a rated life of at least 8000 hours. In commercial settings you expect significantly less on/off switching than residential and so a rated life assumption of 10,000 hours is used.

[486] Based upon pricing forecast developed by Applied Proactive Technologies Inc (APT) based on industry input and provided to

For bulbs over 2600 lumens the assumed incremental capital cost is $5.

### LOADSHAPE

Loadshape C06 - Commercial Indoor Lighting

Loadshape C07 - Grocery/Conv. Store Indoor Lighting

Loadshape C08 - Hospital Indoor Lighting

Loadshape C09 - Office Indoor Lighting

Loadshape C10 - Restaurant Indoor Lighting

Loadshape C11 - Retail Indoor Lighting

Loadshape C12 - Warehouse Indoor Lighting

Loadshape C13 - K-12 School Indoor Lighting

Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)

Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)

Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)

Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)

Loadshape C18 - Industrial Indoor Lighting

Loadshape C19 - Industrial Outdoor Lighting

Loadshape C20 - Commercial Outdoor Lighting

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the location type. Values are provided for each building type in section 4.5.

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = ((WattsBase - WattsEE)/1000) * ISR * Hours * WHFe$$

Where:

WattsBase = Actual (if retrofit measure) or based on lumens of CFL bulb and program year installed:

Ameren.

| Minimum Lumens | Maximum Lumens | Incandescent Equivalent Post-EISA 2007 (WattsBase) |
|---|---|---|
| 5280 | 6209 | 300 |
| 3000 | 5279 | 200 |
| 2601 | 2999 | 150 |
| 1490 | 2600 | 72 |
| 1050 | 1489 | 53 |
| 750 | 1049 | 43 |
| 310 | 749 | 29 |
| 250 | 309 | 25 |

WattsEE = Actual wattage of CFL purchased or installed

ISR = In Service Rate or the percentage of units rebated that get installed.

=100%[487] if application form completed with sign off that equipment is not placed into storage

If sign off form not completed assume the following 3 year ISR assumptions:

| Weighted Average 1st year In Service Rate (ISR) | 2nd year Installations | 3rd year Installations | Final Lifetime In Service Rate |
|---|---|---|---|
| 71.2%[488] | 14.5% | 12.3% | 98.0%[489] |

Hours = Average hours of use per year are provided in Reference Table in Section 4.5,

[487] Illinois evaluation of PY1 through PY3 has not found that fixtures or lamps placed into storage to be a significant enough issue to warrant including an "In-Service Rate" when commercial customers complete an application form.

[488] 1st year in service rate is based upon review of PY4-6 evaluations from ComEd's commercial lighting program (BILD) (see 'IL Commercial Lighting ISR_2014.xls' for more information. The average first year ISR was calculated weighted by the number of bulbs sold.

[489]The 98% Lifetime ISR assumption is based upon review of two evaluations:
'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings. Note that this Final Install Rate does NOT account for leakage of purchased bulbs being installed outside of the utility territory. EM&V should assess how and if data from evaluation should adjust this final installation rate to account for this impact

Screw based bulb annual operating hours, for each building type[490]. If unknown use the Miscellaneous value.

WHFe = Waste heat factor for energy to account for cooling energy savings from efficient lighting are provided below for each building type in Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, a 14W standard CFL is installed in an office in 2014 and sign off form provided:

ΔkWh = (((43 - 14)/1000)* 1.0 * 3088 * 1.25

= 111.9 kWh

**HEATING PENALTY**

If electrically heated building:

$$\Delta kWh_{heatpenalty}^{491} = (((WattsBase-WattsEE)/1000) * ISR * Hours * -IFkWh$$

Where:

IFkWh = Lighting-HVAC Interation Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficent lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, a 14W standard CFL is installed in a heat pump heated office in 2014 and sign off form provided:

$\Delta kWh_{heatpenalty}$ = (((43 - 14)/1000)* 1.0*3088*-0.183

= - 16.4 kWh

**DEFERRED INSTALLS**

As presented above, if a sign off form is not completed the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

Year 1 (Purchase Year) installs: Characterized using assumptions provided above or evaluated assumptions if available.

Year 2 and 3 installs: Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.

The NTG factor for the Purchase Year should be applied.

[490] Based on ComEd analysis taking DEER 2008 values and averaging with PY1 and PY2 evaluation results.
[491] Negative value because this is an increase in heating consumption due to the efficient lighting.

For example, for a 14W CFL (60W standard incandescent and 43W EISA qualified incandescent/halogen) purchased in 2014 and using miscellaneous hours assumption.

$\Delta kWH_{1st\ year\ installs}$ = ((43 - 14) / 1000) * 0.755 * 3198 * 1.06

= 74.2 kWh

$\Delta kWH_{2nd\ year\ installs}$ = ((43 - 14) / 1000) * 0.121 * 3198 * 1.06

= 11.9 kWh

Note: Here we assume no change in hours assumption. NTG value from Purchase year applied.

$\Delta kWH_{3rd\ year\ installs}$ = ((43 - 14) / 1000) * 0.103 * 3198 * 1.06

= 10.1 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ((WattsBase-WattsEE)/1000) * ISR * WHFd * CF

Where:

WHFd = Waste heat factor for demand to account for cooling savings from efficient lighting in cooled buildings is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value..

CF = Summer Peak Coincidence Factor for measure is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value..

Other factors as defined above

For example, a 14W standard CFLis installed in an office in 2014 and sign off form provided:

ΔkW = ((43 - 14)/1000)*1.0*1.3*0.66

= 0.025kW

**NATURAL GAS ENERGY SAVINGS**

Heating Penalty if fossil fuel heated building (or if heating fuel is unknown):

ΔTherms[492] = (((WattsBase-WattsEE)/1000) * ISR * Hours *- IFTherms

Where:

IFTherms = Lighting-HVAC Interation Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficent lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

Other factors as defined above

[492] Negative value because this is an increase in heating consumption due to the efficient lighting.

For example, a 14W standard CFL is installed in an office in 2014 and sign off form provided:

ΔTherms = (((43 - 14)/1000)* 1.0*3088*-0.016

= - 1.4 Therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

Bulb replacement costs assumed in the O&M calculations are provided below[493].

| | Std Inc. | EISA Compliant Halogen |
|---|---|---|
| 2014 | $0.34 | $1.25 |
| 2015 | $0.34 | $0.90 |
| 2016 | $0.34 | $0.80 |
| 2017 | $0.34 | $0.70 |
| 2018 | $0.34 | $0.60 |
| 2019 | $0.34 | $0.60 |
| 2020 & after | $0.34 | N/A |

In order to account for the falling EISA Qualified bulb replacement cost provided above, an equivalent annual levelized baseline replacement cost over the lifetime of the CFL bulb is calculated. Note that the measure life for these measures is capped to the number of years remaining until 2020.

The NPV for replacement lamps and annual levelized replacement costs using the statewide real discount rate of 5.23% are presented below:

| Location | Lumen Level | NPV of replacement costs for period | | | Levelized annual replacement cost savings | | |
|---|---|---|---|---|---|---|---|
| | | June 2014 - May 2015 | June 2015 - May 2016 | June 2016 - May 2017 | June 2014 - May 2015 | June 2015 - May 2016 | June 2016 - May 2017 |
| Commercial | Lumens <310 or >2600 (EISA exempt) | $2.83 | $2.83 | $2.83 | $1.40 | $1.40 | $1.40 |
| | Lumens ≥ 310 and ≤ 2600 (EISA compliant) | $8.60 | $6.91 | $6.08 | $4.26 | $3.43 | $3.02 |

[493] Based upon pricing forecast developed by Applied Proactive Technologies Inc (APT) based on industry input and provided to Ameren.

| | | | | | | | |
|---|---|---|---|---|---|---|---|
| Multi Family Common Areas | Lumens <310 or >2600 (EISA exempt) | $2.88 | $2.88 | $2.88 | $1.81 | $1.81 | $1.81 |
| | Lumens ≥ 310 and ≤ 2600 (EISA compliant) | $9.27 | $7.24 | $6.40 | $5.84 | $4.56 | $4.03 |

For halogen bulbs, we assume the same replacement cycle as incandescent bulbs.[494] The replacement cycle is based on the miscellaneous hours of use. Both incandescent and halogen lamps are assumed to last for 1,000 hours before needing replacement.

**MEASURE CODE: CI-LTG-CCFL-V05-150601**

[494] The manufacturers of the new minimally compliant EISA Halogens are using regular incandescent lamps with halogen fill gas rather than halogen infrared to meet the standard and so the component rated life is equal to the standard incandescent.

## 4.5.2 Fluorescent Delamping

### DESCRIPTION

This measure addresses the permanent removal of existing 8', 4', 3' and 2' fluorescent lamps. Unused lamps, lamp holders, and ballasts must be permanently removed from the fixture. This measure is applicable when retrofitting from T12 lamps to T8 lamps or simply removing lamps from a T8 fixture. Removing lamps from a T12 fixture that is not being retrofitted with T8 lamps are not eligible for this incentive.

Customers are responsible for determining whether or not to use reflectors in combination with lamp removal in order to maintain adequate lighting levels. Lighting levels are expected to meet the Illuminating Engineering Society of North America (IESNA) recommended light levels. Unused lamps, lamp holders, and ballasts must be permanently removed from the fixture and disposed of in accordance with local regulations. A pre-approval application is required for lamp removal projects.

This measure was developed to be applicable to the following program types: RF. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

Savings are defined on a per removed lamp basis. The retrofit wattage (efficient conditioned) is therefore assumed to be zero. The savings numbers provided below are for the straight lamp removal measures, as well as the lamp removal and install reflector measures. The lamp installed/retrofit is captured in another measure.

### DEFINITION OF BASELINE EQUIPMENT

The baseline condition is either a T12 or a T8 lamp with default wattages provided below. Note, if the program does not allow for the lamp type to be known, then a T12:T8 weighting of 80%:20% can be applied[495].

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 11 years per DEER 2005.

### DEEMED MEASURE COST

The incremental capital cost is provided in the table below:

| Measure Category | Value | Source |
|---|---|---|
| 8-Foot Lamp Removal | $16.00 | ComEd/KEMA regression[496] |
| 4-Foot Lamp Removal | $12.00 | ICF Portfolio Plan |
| 8-Foot Lamp Removal with reflector | $30.00 | KEMA Assumption |
| 4-Foot Lamp Removal with reflector | $25.00 | KEMA Assumption |
| 2-Foot or 3-Foot Removal | $12.35 | KEMA Assumption |
| 2-Foot or 3-Foot Removal with | $25.70 | KEMA Assumption |

[495] Based on ComEd's estimate of lamp type saturation.
[496] Based on the assessment of active projects in the 2008-09 ComEd Smart Ideas Program. See files "ltg costs 12-10-10.xl." and "Lighting Unit Costs 102605.doc"

| Measure Category | Value | Source |
|---|---|---|
| reflector | | |

### LOADSHAPE

Loadshape C06 - Commercial Indoor Lighting

Loadshape C07 - Grocery/Conv. Store Indoor Lighting

Loadshape C08 - Hospital Indoor Lighting

Loadshape C09 - Office Indoor Lighting

Loadshape C10 - Restaurant Indoor Lighting

Loadshape C11 - Retail Indoor Lighting

Loadshape C12 - Warehouse Indoor Lighting

Loadshape C13 - K-12 School Indoor Lighting

Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)

Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)

Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)

Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)

Loadshape C18 - Industrial Indoor Lighting

Loadshape C19 - Industrial Outdoor Lighting

Loadshape C20 - Commercial Outdoor Lighting

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the location type. Values are provided for each building type in the reference section below.

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

ΔkWh =((WattsBase-WattsEE)/1000) * ISR * Hours * WHFe

Where:

WattsBase = Assume wattage reduction of lamp removed

| | Wattage of lamp removed[497] | | Weighted average |
|---|---|---|---|
| | T8 | T12 | 80% T12, 20% T8 |
| 8-ft T8 | 38.6 | 60.3 | 56.0 |
| 4-ft T8 | 19.4 | 33.7 | 30.8 |
| 3-ft T8 | 14.6 | 40.0 | 34.9 |
| 2-ft T8 | 9.8 | 28.0 | 24.4 |

WattsEE = 0

ISR = In Service Rate or the percentage of units rebated that get installed.

=100% if application form completed with sign off that equipment permanently removed and disposed of.

Hours = Average hours of use per year are provided in Reference Table in Section 4.5. If unknown use the Miscellaneous value.

WHFe = Waste heat factor for energy to account for cooling energy savings from efficient lighting are provided below for each building type in Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, delamping a 4 ft T8 fixture in an office building:

ΔkWh =((19.4 - 0)/1000) * 1.0 * 4439 * 1.25

= 107.6 kWh

### HEATING PENALTY

If electrically heated building:

[497] Default wattage reducetion is based on averaging the savings from moving from a 2 to 1, 3 to 2 and 4 to 3 lamp fixture, as provided in the Standard Performance Contract Procedures Manual: Appendix B: Table of Standard Fixture Wattages (http://www.sce.com/NR/rdonlyres/7A3455F0-A337-439B-9607-10A016D32D4B/0/spc_B_Std_Fixture_Watts.pdf). An adjustment is made to the T8 delamped fixture to account for the significant increase in ballast factor. See 'Delamping calculation.xls' for details.

$\Delta kWh_{heatpenalty}$[498] = (((WattsBase-WattsEE)/1000) * ISR * Hours * -IFkWh

Where:

IFkWh = Lighting-HVAC Interation Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficent lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, delamping a 4 ft T8 fixture in a heat pump heated office building:

$\Delta kWh_{heatpenalty}$ =((19.4 - 0)/1000) * 1.0 * 4439 * -0.151

=-13.0 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ((WattsBase-WattsEE)/1000) * ISR * WHFd * CF

Where:

WHFd = Waste heat factor for demand to account for cooling savings from efficient lighting in cooled buildings is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value..

CF = Summer Peak Coincidence Factor for measure is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value..

Other factors as defined above

For example, delamping a 4 ft T8 fixture in an office building:

ΔkW =((19.4 - 0)/1000) * 1.0 * 1.3 * 0.66

= 0.017 kW

**NATURAL GAS ENERGY SAVINGS**

Heating Penalty if fossil fuel heated building (or if heating fuel is unknown):

ΔTherms[499] = (((WattsBase-WattsEE)/1000) * ISR * Hours *- IFTherms

Where:

IFTherms = Lighting-HVAC Interation Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficent lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

Other factors as defined above

[498] Negative value because this is an increase in heating consumption due to the efficient lighting.
[499] Negative value because this is an increase in heating consumption due to the efficient lighting.

For example, delamping a 4 ft T8 fixture in an office building:

ΔTherms =((19.4 - 0)/1000) * 1.0 * 4439 * -0.016

=-1.4 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-LTG-DLMP-V02-140601**

## 4.5.3 High Performance and Reduced Wattage T8 Fixtures and Lamps

**DESCRIPTION**

This measure applies to "High Performance T8" (HPT8) lamp/ballast systems that have higher lumens per watt than standard T8 systems. This measure applies to the installation of new equipment with efficiencies that exceed that of the equipment that would have been installed following standard market practices and is applicable to time of sale as well as retrofit measures. Retrofit measures may include new fixtures or relamp/reballast measures. In addition, options have been provided to allow for the "Reduced Wattage T8 lamps" or RWT8 lamps that result in re-lamping opportunities that produce equal or greater light levels than standard T8 lamps while using fewer watts.

If the implementation strategy does not allow for the installation location to be known, a deemed split of 99% Commercial and 1% Residential should be used[500].

This measure was developed to be applicable to the following program types: TOS, RF, DI.

If applied to other program types, the measure savings should be verified.

The measure applies to all commercial HPT8 installations excluding new construction and major renovation or change of use measures (see lighting power density measure). Lookup tables have been provided to account for the different types of installations. Whenever possible, actual costs and hours of use should be utilized for savings calculations. Default new and baseline assumptions have been provided in the reference tables. Default component costs and lifetimes have been provided for Operating and Maintenance Calculations. Please see the Definition Table to determine applicability for each program. HPT8 configurations not included in the TRM may be included in custom program design using the provided algorithms as long as energy savings is achieved. The following table defines the applicability for different programs

| Time of Sale (TOS) | Retrofit (RF) and Direct Install (DI) |
|---|---|
| This measure relates to the installation of new equipment with efficiency that exceeds that of equipment that would have been installed following standard market practices. In general, the measure will include qualifying high efficiency low ballast factor ballasts paired with high efficiency long life lamps as detailed in the attached tables. High-bay applications use this system paired with qualifying high ballast factor ballasts and high performance 32 w lamps. Custom lighting designs can use qualifying low, normal or high ballast factor ballasts and qualifying lamps in lumen equivalent applications where total system wattage is reduced when calculated using the Calculation of Savings Algorithms. | This measure relates to the replacement of existing equipment with new equipment with efficiency that exceeds that of the existing equipment. In general, the retrofit will include qualifying high efficiency low ballast factor ballasts paired with high efficiency long life lamps as detailed in the attached tables. Custom lighting designs can use qualifying low, normal or high ballast factor ballasts and qualifying lamps in lumen equivalent applications where total system wattage is reduced when calculated using the Calculation of Savings Algorithms.<br><br>High efficiency troffers (new/or retrofit) utilizing HPT8 technology can provide even greater savings. When used in a high-bay application, high-performance T8 fixtures can provide equal light to HID high-bay fixtures, while using fewer watts; these systems typically utilize high ballast factor ballasts, but qualifying low and normal |

[500] Based on weighted average of Final ComEd's BILD program data from PY5 and PY6. For Residential installations, hours of use assumptions from '5.5 Interior Hardwired Compact Fluorescent Lamp (CFL) Fixture' measure should be used.

| Time of Sale (TOS) | Retrofit (RF) and Direct Install (DI) |
|---|---|
| | ballast factor ballasts may be used when appropriate light levels are provided and overall wattage is reduced. |

### DEFINITION OF EFFICIENT EQUIPMENT

The definition of efficient equipment varies based on the program and is defined below:

| Time of Sale (TOS) | Retrofit (RF) and Direct Install (DI) |
|---|---|
| In order for this characterization to apply, new lamps and ballasts must be listed on the CEE website on the qualifying High Performance T8 lamps and ballasts list (http://www.cee1.org/com/com-lt/com-lt-main.php3).<br><br>High efficiency troffers combined with high efficiency lamps and ballasts allow for fewer lamps to be used to provide a given lumen output. High efficiency troffers must have a fixture efficiency of 80% or greater to qualify. Default values are given for a 2 lamp HPT8 fixture replacing a 3 lamp standard efficiency T8 fixture, but other configurations may qualify and the Calculation of savings algorithm used to account for base watts being replaced with EE watts.<br><br>High bay fixtures must have fixture efficiencies of 85% or greater.<br><br>RWT8 lamps: In order for this characterization to apply, new 4' and U-tube lamps must be listed on the CEE website on the qualifying Reduced Wattage High Performance T8 lamps list. (http://library.cee1.org/content/commercial-lighting-qualifying-products-lists). 2', 3' and 8' lamps must meet the wattage requirements specified in the RWT8 new and baseline assumptions table. This measure assumes a lamp only purchase. | In order for this characterization to apply, new lamps and ballasts must be listed on the CEE website on the qualifying High Performance T8 lamps and ballasts list (http://www.cee1.org/com/com-lt/com-lt-main.php3).<br><br>High efficiency troffers (new or retrofit kits) combined with high efficiency lamps and ballasts allow for fewer lamps to be used to provide a given lumen output. High efficiency troffers must have a fixture efficiency of 80% or greater to qualify. Default values are given for a 2 lamp HPT8 fixture replacing a 3 lamp standard efficiency T8 fixture, but other configurations may qualify and the Calculation of savings algorithm used to account for base watts being replaced with EE watts.<br><br>High bay fixtures will have fixture efficiencies of 85% or greater.<br><br>RWT8: in order for this characterization to apply, new 4' and U-tube lamps must be listed on the CEE website on the qualifying Reduced Wattage High Performance T8 lamps list.<br><br>(http://library.cee1.org/content/commercial-lighting-qualifying-products-lists). 2', 3' and 8' lamps must meet the wattage requirements specified in the RWT8 new and baseline assumptions table. |

### DEFINITION OF BASELINE EQUIPMENT

The definition of baseline equipment varies based on the program and is defined below:

| Time of Sale (TOS) | Retrofit (RF) and Direct Install (DI) |
|---|---|
| The baseline is standard efficiency T8 systems that would have been installed. The baseline for high-bay fixtures is pulse start metal halide fixtures, the baseline for a 2 lamp high efficiency troffer is a 3 lamp standard efficency troffer. | The baseline is the existing system.<br><br>In July 14, 2012, Federal Standards were enacted that were expected to eliminate T-12s as an option for linear fluorescent fixtures. Through v3.0 of the TRM, it was assumed that the T-12 would no longer be baseline for retrofits from 1/1/2016. However, due to significant loopholes in the legislation, T-12 compliant product is still freely available and in Illinois T-12s continue to hold a significant share of the existing and replacement lamp market. Therefore the timing of the sunsetting of T-12s as a viable baseline has been pushed back in v4.0 until 6/1/2016 and will be revisited in future update sessions.<br><br>There will be a baseline shift applied to all measures installed before 2016. See table C-1. |

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The deemed lifetime of efficient equipment varies based on the program and is defined below:

| Time of Sale (TOS) | Retrofit (RF) and Direct Install (DI) |
|---|---|
| Fixture lifetime is 15 years[501].<br><br>Fixture retrofits which utilize RWT8 lamps have a lifetime equivalent to the life of the lamp, capped at 15 years. There is no guarantee that a reduced wattage lamp will be installed at time of burnout, but if one is, savings will be captured in the RWT8 measure below.<br><br>RWT8 lifetime is the life of the product, at the reported operating hours (lamp life in hours divided by operating hours per year – see reference table "RWT8 Component Costs and Lifetime"), capped at 15 years.502 | Fixture lifetime is 15 years.<br><br>As per explanation above, for existing T12 fixtures, a mid life baseline shift should be applied in Jan 2016 as described in table C-1.<br><br>Note, since the fixture lifetime is deemed at 15 years, the replacement cost of both the lamp and ballast should be incorporated in to the O&M calculation. |

### LOADSHAPE

Loadshape C06 - Commercial Indoor Lighting

[501] 15 years from GDS Measure Life Report, June 2007
[502] ibid

Loadshape C07 - Grocery/Conv. Store Indoor Lighting

Loadshape C08 - Hospital Indoor Lighting

Loadshape C09 - Office Indoor Lighting

Loadshape C10 - Restaurant Indoor Lighting

Loadshape C11 - Retail Indoor Lighting

Loadshape C12 - Warehouse Indoor Lighting

Loadshape C13 - K-12 School Indoor Lighting

Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)

Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)

Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)

Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)

Loadshape C18 - Industrial Indoor Lighting

Loadshape C19 - Industrial Outdoor Lighting

Loadshape C20 - Commercial Outdoor Lighting

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = ((Watts_{base} - Watts_{EE})/1000) * Hours * WHF_e * ISR$$

Where:

$Watts_{base}$ = Input wattage of the existing system which depends on the baseline fixture configuration (number and type of lamp) and number of fixtures. Value can be selected from the appropriate reference table as shown below, or a custom value can be entered if the configurations in the tables is not representative of the exisitng system.

| Program | Reference Table |
|---|---|
| Time of Sale | A-1: HPT8 New and Baseline Assumptions |
| Retrofit | A-2: HPT8 New and Baseline Assumptions |
| Reduced Wattage T8, time of sale or retrofit | A-3: RWT8 New and Baseline Assumptions |

$Watts_{EE}$ = New Input wattage of EE fixture which depends on new fixture configuration (number

of lamps) and ballast factor and number of fixtures. Value can be selected from the appropriate reference table as shown below, of a custom value can be entered if the configurations in the tables is not representative of the exisitng system.

| Program | Reference Table |
|---|---|
| Time of Sale | A-1: HPT8 New and Baseline Assumptions |
| Retrofit | A-2: HPT8 New and Baseline Assumptions |
| Reduced Wattage T8, time of sale or retrofit | A-3: RWT8 New and Baseline Assumptions |

Hours = Average hours of use per year as provided by the customer or selected from the Reference Table in Section 4.5, Fixture annual operating hours. If hours or building type are unknown, use the Miscellaneous value.

$WHF_e$ = Waste heat factor for energy to account for cooling energy savings from efficient lighting is selected from the Reference Table in Section 4.5 for each building type. If building is un-cooled, the value is 1.0.

ISR = In Service Rate or the percentage of units rebated that get installed.

=100%[503] if application form completed with sign off that equipment is not placed into storage

If sign off form not completed assume the following 3 year ISR assumptions:

| Weighted Average 1st year In Service Rate (ISR) | 2nd year Installations | 3rd year Installations | Final Lifetime In Service Rate |
|---|---|---|---|
| 98%[504] | 0% | 0% | 98.0%[505] |

[503] Illinois evaluation of PY1 through PY3 has not found that fixtures or lamps placed into storage to be a significant enough issue to warrant including an "In-Service Rate" when commercial customers complete an application form.

[504] 1st year in service rate is based upon review of PY5-6 evaluations from ComEd's commercial lighting program (BILD) (see 'IL Commercial Lighting ISR_2014.xls' for more information

[505] The 98% Lifetime ISR assumption is based upon review of two evaluations:
'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings. Note that this Final Install Rate does NOT account for leakage of purchased bulbs being installed outside of the utility territory. EM&V should assess how and if data from evaluation should adjust this final installation rate to account for this impact

### HEATING PENALTY

If electrically heated building:

$$\Delta kWh_{heatpenalty}^{506} = (((WattsBase-WattsEE)/1000) * ISR * Hours * -IFkWh$$

Where:

IFkWh = Lighting-HVAC Interation Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficent lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

### SUMMER COINCIDENT DEMAND SAVINGS

$$\Delta kW = ( (Watts_{base}-Watts_{EE})/1000) * WHF_d*CF*ISR$$

Where:

$WHF_d$ = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is selected from the Reference Table in Section 4.5 for each building type. If the building is not cooled WHFd is 1.

CF = Summer Peak Coincidence Factor for measure is selected from the Reference Table in Section 4.5 for each building type. If the building type is unknown, use the Miscellaneous value of 0.66.

Other factors as defined above

### NATURAL GAS SAVINGS

$$\Delta Therms^{507} = (((WattsBase-WattsEE)/1000) * ISR * Hours *- IFTherms$$

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Please select from the Reference Table in Section 4.5 for each building type.

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

See Reference tables for Operating and Maintenance Values

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[506] Negative value because this is an increase in heating consumption due to the efficient lighting.
[507] Negative value because this is an increase in heating consumption due to the efficient lighting.

| Program | Reference Table |
|---|---|
| Time of Sale | B-1: HPT8 Component Costs and Lifetime |
| Retrofit | B-2: HPT8 Component Costs and Lifetime |
| Reduced Wattage T8, time of sale or retrofit | B-3: HPT8 Component Costs and Lifetime |

**REFERENCE TABLES**

See following page

A-1: Time of Sale: HPT8 New and Baseline Assumptions[508]

| EE Measure Description | $Watts_{EE}$ | Baseline Description | $Watts_{BASE}$ | Measure Cost | $Watts_{SAVE}$ |
|---|---|---|---|---|---|
| 4-Lamp HPT8 w/ High-BF Ballast High-Bay | 146 | 200 Watt Pulse Start Metal-Halide | 232 | $75 | 86 |
| 6-Lamp HPT8 w/ High-BF Ballast High-Bay | 221 | 320 Watt Pulse Start Metal-Halide | 350 | $75 | 129 |
| 8-Lamp HPT8 w/ High-BF Ballast High-Bay | 280 | Proportionally Adjusted according to 6-Lamp HPT8 Equivalent to 320 PSMH | 455 | $75 | 175 |
| | | | | | |
| 1-Lamp HPT8-high performance 32 w lamp | 25 | 1-Lamp Standard F32T8 w/ Elec. Ballast | 32 | $15 | 7 |
| 1-Lamp HPT8-high performance 28 w lamp | 22 | 1-Lamp Standard F32T8 w/ Elec. Ballast | 32 | $15 | 10 |
| 1-Lamp HPT8-high performance 25 w lamp | 19 | 1-Lamp Standard F32T8 w/ Elec. Ballast | 32 | $15 | 13 |
| 2-Lamp HPT8 -high performance 32 w lamp | 49 | 2-Lamp Standard F32T8 w/ Elec. Ballast | 59 | $18 | 10 |
| 2-Lamp HPT8-high performance 28 w lamp | 43 | 2-Lamp Standard F32T8 w/ Elec. Ballast | 59 | $18 | 16 |
| 2-Lamp HPT8-high performance 25 w lamp | 35 | 2-Lamp Standard F32T8 w/ Elec. Ballast | 59 | $18 | 24 |
| 3-Lamp HPT8-high performance 32 w lamp | 72 | 3-Lamp Standard F32T8 w/ Elec. Ballast | 88 | $20 | 16 |
| 3-Lamp HPT8-high performance 28 w lamp | 65 | 3-Lamp Standard F32T8 w/ Elec. Ballast | 88 | $20 | 23 |
| 3-Lamp HPT8-high performance 25 w lamp | 58 | 3-Lamp Standard F32T8 w/ Elec. Ballast | 88 | $20 | 30 |

[508] Adapted from Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, October 26, 2011.

| EE Measure Description | $Watts_{EE}$ | Baseline Description | $Watts_{BASE}$ | Measure Cost | $Watts_{SAVE}$ |
|---|---|---|---|---|---|
| 4-Lamp HPT8 -high performance 32 w lamp | 94 | 4-Lamp Standard F32T8 w/ Elec. Ballast | 114 | $23 | 20 |
| 4-Lamp HPT8-high performance 28 w lamp | 86 | 4-Lamp Standard F32T8 w/ Elec. Ballast | 114 | $23 | 28 |
| 4-Lamp HPT8-high performance 25 w lamp | 77 | 4-Lamp Standard F32T8 w/ Elec. Ballast | 114 | $23 | 37 |
| | | | | | |
| 2-lamp High-Performance HPT8 Troffer | 49 | 3-Lamp F32T8 w/ Elec. Ballast | 88 | $100 | 39 |

Table developed using a constant ballast factor of .77. Input wattages are an average of manufacturer inputs that account for ballast efficacy

A-2: Retrofit HPT8 New and Baseline Assumptions[509] (Note see definiton for validity after 2016)

| EE Measure Description | $Watts_{EE}$ | Baseline Description | $Watts_{BASE}$ | Incremental cost | $Watts_{SAVE}$ |
|---|---|---|---|---|---|
| 4-Lamp HPT8 w/ High-BF Ballast High-Bay | 146 | 200 Watt Pulse Start Metal-Halide | 232 | $200 | 86 |
| 4-Lamp HPT8 w/ High-BF Ballast High-Bay | 146 | 250 Watt Metal Halide | 295 | $200 | 149 |
| 6-Lamp HPT8 w/ High-BF Ballast High-Bay | 206 | 320 Watt Pulse Start Metal-Halide | 350 | $225 | 144 |
| 6-Lamp HPT8 w/ High-BF Ballast High-Bay | 206 | 400 Watt Metal Halide | 455 | $225 | 249 |
| 8-Lamp HPT8 w/ High-BF Ballast High-Bay | 280 | Proportionally Adjusted according to 6-Lamp HPT8 Equivalent to 320 PSMH | 476 | $250 | 196 |
| 8-Lamp HPT8 w/ High-BF Ballast High-Bay | 280 | Proportionally Adjusted according to 6-Lamp HPT8 Equivalent to 400 W Metal halide | 618 | $250 | 338 |
| | | | | | |
| 1-Lamp Relamp/Reballast T12 to HPT8 | 25 | 1-Lamp F34T12 w/ EEMag Ballast | 40 | $50 | 15 |
| 2-Lamp Relamp/Reballast T12 to HPT8 | 49 | 2-Lamp F34T12 w/ EEMag Ballast | 68 | $55 | 19 |
| 3-Lamp Relamp/Reballast T12 to HPT8 | 72 | 3-Lamp F34T12 w/ EEMag Ballast | 110 | $60 | 38 |
| 4-Lamp Relamp/Reballast T12 to HPT8 | 94 | 4-Lamp F34T12 w/ EEMag Ballast | 139 | $65 | 45 |
| | | | | | |
| 1-Lamp Relamp/Reballast T12 to HPT8 | 25 | 1-Lamp F40T12 w/ EEMag Ballast | 48 | $50 | 23 |
| 2-Lamp Relamp/Reballast T12 to HPT8 | 49 | 2-Lamp F40T12 w/ EEMag Ballast | 82 | $55 | 33 |
| 3-Lamp Relamp/Reballast T12 to HPT8 | 72 | 3-Lamp F40T12 w/ EEMag Ballast | 122 | $60 | 50 |
| 4-Lamp Relamp/Reballast T12 to HPT8 | 94 | 4-Lamp F40T12 w/ EEMag Ballast | 164 | $65 | 70 |
| | | | | | |
| 1-Lamp Relamp/Reballast T12 to HPT8 | 25 | 1-Lamp F40T12 w/ Mag Ballast | 57 | $50 | 32 |
| 2-Lamp Relamp/Reballast T12 to HPT8 | 49 | 2-Lamp F40T12 w/ Mag Ballast | 94 | $55 | 45 |
| 3-Lamp Relamp/Reballast T12 to HPT8 | 72 | 3-Lamp F40T12 w/ Mag Ballast | 147 | $60 | 75 |
| 4-Lamp Relamp/Reballast T12 to HPT8 | 94 | 4-Lamp F40T12 w/ Mag Ballast | 182 | $65 | 88 |
| | | | | | |
| 1-Lamp Relamp/Reballast T8 to HPT8 | 25 | 1-Lamp F32T8 w/ Elec. Ballast | 32 | $50 | 7 |
| 2-Lamp Relamp/Reballast T8 to HPT8 | 49 | 2-Lamp F32T8 w/ Elec. Ballast | 59 | $55 | 10 |
| 3-Lamp Relamp/Reballast T8 to HPT8 | 72 | 3-Lamp F32T8 w/ Elec. Ballast | 88 | $60 | 16 |
| 4-Lamp Relamp/Reballast T8 to HPT8 | 94 | 4-Lamp F32T8 w/ Elec. Ballast | 114 | $65 | 20 |
| | | | | | |
| 2-lamp High-Performance HPT8 Troffer or high efficiency retrofit troffer | 49 | 3-Lamp F32T8 w/ Elec. Ballast | 88 | $100 | 39 |

[509] Adapted from Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, October 26, 2011.

A– 3: RWT8 New and Baseline Assumptions

| EE Measure Description | EE Cost | System WattsEE | Baseline Description | Base Cost | System Watts Base | Measure Cost | WattsSAVE |
|---|---|---|---|---|---|---|---|
| RWT8 - F28T8 Lamp | $4.50 | 25 | F32T8 Standard Lamp | $2.50 | 28 | $2.00 | 4 |
| RWT8 - F28T8 Extra Life Lamp | $4.50 | 25 | F32T8 Standard Lamp | $2.50 | 28 | $2.00 | 4 |
| RWT8 - F32/25W T8 Lamp | $4.50 | 22 | F32T8 Standard Lamp | $2.50 | 28 | $2.00 | 6 |
| RWT8 - F32/25W T8 Lamp Extra Life | $4.50 | 22 | F32T8 Standard Lamp | $2.50 | 28 | $2.00 | 6 |
| RWT8 - F17T8 Lamp - 2 Foot | $4.80 | 14 | F17T8 Standard Lamp - 2 foot | $2.80 | 16 | $2.00 | 2 |
| RWT8 - F25T8 Lamp - 3 Foot | $5.10 | 20 | F25T8 Standard Lamp - 3 foot | $3.10 | 23 | $2.00 | 3 |
| RWT8 - F30T8 Lamp - 6" Utube | $11.31 | 26 | F32T8 Standard Utube Lamp | $9.31 | 28 | $2.00 | 2 |
| RWT8 - F29T8 Lamp - Utube | $11.31 | 26 | F32T8 Standard Utube Lamp | $9.31 | 28 | $2.00 | 3 |
| RWT8 - F96T8 Lamp - 8 Foot | $9.00 | 57 | F96T8 Standard Lamp - 8 foot | $7.00 | 62 | $2.00 | 5 |
| Notes: Wattage assumptions for Reduced-Wattage T8 based on Existing 0.88 Normal Ballast Factor. | | | | | | | |

B-1: Time of Sale T8 Component Costs and Lifetime[510]

| EE Measure Description | EE Lamp Cost | EE Lamp Life (hrs) | EE Lamp Rep. Labor Cost per lamp | EE Ballast Cost | EE Ballast Life (hrs) | EE Ballast Rep. Labor Cost | Baseline Description | Base Lamp Cost | Base Lamp Life (hrs) | Base Lamp Rep. Labor Cost | Base Ballast Cost | Base Ballast Life (hrs) | Base Ballast Rep. Labor Cost |
|---|---|---|---|---|---|---|---|---|---|---|---|---|---|
| 4-Lamp HPT8 w/ High-BF Ballast High-Bay | $5.00 | 24000 | $6.67 | $32.50 | 70000 | $15.00 | 200 Watt Pulse Start Metal-Halide | $21.00 | 10000 | $6.67 | $88 | 40000 | $22.50 |
| 6-Lamp HPT8 w/ High-BF Ballast High-Bay | $5.00 | 24000 | $6.67 | $32.50 | 70000 | $15.00 | 320 Watt Pulse Start Metal-Halide | $21.00 | 20000 | $6.67 | $109 | 40000 | $22.50 |
| 8-Lamp HPT8 w/ High-BF Ballast High-Bay | $5.00 | 24000 | $6.67 | $32.50 | 70000 | $15.00 | Lamp HPT8 Equivalent to 320 PSMH | $21.00 | 20000 | $6.67 | $109 | 40000 | $22.50 |
| | | | | | | | | | | | | | |
| 1-Lamp HPT8 - all qualifying lamps | $5.00 | 24000 | $2.67 | $32.50 | 70000 | $15.00 | 1-Lamp Standard F32T8 w/ Elec. Ballast | $2.50 | 20000 | $2.67 | $15 | 70000 | $15.00 |
| 2-Lamp HPT8 - all qualifying lamps | $5.00 | 24000 | $2.67 | $32.50 | 70000 | $15.00 | 2-Lamp Standard F32T8 w/ Elec. Ballast | $2.50 | 20000 | $2.67 | $15 | 70000 | $15.00 |
| 3-Lamp HPT8 - all qualifying lamps | $5.00 | 24000 | $2.67 | $32.50 | 70000 | $15.00 | 3-Lamp Standard F32T8 w/ Elec. Ballast | $2.50 | 20000 | $2.67 | $15 | 70000 | $15.00 |
| 4-Lamp HPT8 - all qualifying lamps | $5.00 | 24000 | $2.67 | $32.50 | 70000 | $15.00 | 4-Lamp Standard F32T8 w/ Elec. Ballast | $2.50 | 20000 | $2.67 | $15 | 70000 | $15.00 |
| | | | | | | | | | | | | | |
| 2-lamp High-Performance HPT8 Troffer | $5.00 | 24000 | $2.67 | $32.50 | 70000 | $15.00 | 3-Lamp F32T8 w/ Elec. Ballast | $2.50 | 20000 | $2.67 | $15 | 70000 | $15.00 |

[510] Ibid.

B-2: T8 Retrofit Component Costs and Lifetime[511]

| EE Measure Description | EE Lamp Cost | EE Lamp Life (hrs) | EE Lamp Rep. Labor Cost per lamp | EE Ballast Cost | EE Ballast Life (hrs) | EE Ballast Rep. Labor Cost | Baseline Description | Base Lamp Cost | Base Lamp Life (hrs) | Base Lamp Rep. Labor Cost | Base Ballast Cost | Base Ballast Life (hrs) | Base Ballast Rep. Labor Cost |
|---|---|---|---|---|---|---|---|---|---|---|---|---|---|
| 4-Lamp HPT8 w/ High-BF Ballast High-Bay | $5.00 | 24000 | $6.67 | $32.50 | 70000 | $15.00 | 200 Watt Pulse Start Metal-Halide | $29.00 | 12000 | $6.67 | $88 | 40000 | $22.50 |
| | | | | | | | 250 Watt Metal Halide | $21.00 | 10000 | $6.67 | $92 | 40000 | $22.50 |
| 6-Lamp HPT8 w/ High-BF Ballast High-Bay | $5.00 | 24000 | $6.67 | $32.50 | 70000 | $15.00 | 320 Watt Pulse Start Metal-Halide | $72.00 | 20000 | $6.67 | $109 | 40000 | $22.50 |
| | | | | | | | 400 Watt Metal Halide | $17.00 | 20000 | $6.67 | $114 | 40000 | $22.50 |
| 8-Lamp HPT8 w/ High-BF Ballast High-Bay | $5.00 | 24000 | $6.67 | $32.50 | 70000 | $15.00 | Proportionally Adjusted according to 6-Lamp HPT8 Equivalent to 320 PSMH | $72.00 | 20000 | $6.67 | $109 | 40000 | $22.50 |
| | | | | | | | Proportionally Adjusted according to 6-Lamp HPT8 Equivalent to 400 Watt Metal Halide | $17.00 | 20000 | $6.67 | $114 | 40000 | $22.50 |
| 1-Lamp Relamp/Reballast T12 to HPT8 (all lamp/ballst combinations) | $5.00 | 24000 | $2.67 | $32.50 | 70000 | $15.00 | 1-Lamp T12 all lamp/ballast combinations | $2.70 | 20000 | $2.67 | $20 | 40000 | $15.00 |
| 2-Lamp Relamp/Reballast T12 to HPT8 (all lamp/ballast combinations) | $5.00 | 24000 | $2.67 | $32.50 | 70000 | $15.00 | 2-Lamp T12 all lamp/ballast combinations | $2.70 | 20000 | $2.67 | $20 | 40000 | $15.00 |
| 3-Lamp Relamp/Reballast T12 to HPT8 (all lamp/ballast combinations) | $5.00 | 24000 | $2.67 | $32.50 | 70000 | $15.00 | 3-Lamp T12 all lamp/ballast combinations | $2.70 | 20000 | $2.67 | $20 | 40000 | $15.00 |
| 4-Lamp Relamp/Reballast T12 to HPT8 (all lamp/ballast combinations) | $5.00 | 24000 | $2.67 | $32.50 | 70000 | $15.00 | 3-Lamp T12 all lamp/ballast combinations | $2.70 | 20000 | $2.67 | $20 | 40000 | $15.00 |
| | | | | | | | | | | | | | |
| 1-Lamp Relamp/Reballast T8 to HPT8 | $5.00 | 24000 | $2.67 | $32.50 | 70000 | $15.00 | 1-Lamp F32T8 w/ Elec. Ballast | $2.70 | 20000 | $2.67 | $20 | 70000 | $15.00 |
| 2-Lamp Relamp/Reballast T8 to HPT8 | $5.00 | 24000 | $2.67 | $32.50 | 70000 | $15.00 | 2-Lamp F32T8 w/ Elec. Ballast | $2.70 | 20000 | $2.67 | $20 | 70000 | $15.00 |
| 3-Lamp Relamp/Reballast T8 to HPT8 | $5.00 | 24000 | $2.67 | $32.50 | 70000 | $15.00 | 3-Lamp F32T8 w/ Elec. Ballast | $2.70 | 20000 | $2.67 | $20 | 70000 | $15.00 |
| 4-Lamp Relamp/Reballast T8 to HPT8 | $5.00 | 24000 | $2.67 | $32.50 | 70000 | $15.00 | 4-Lamp F32T8 w/ Elec. Ballast | $2.70 | 20000 | $2.67 | $20 | 70000 | $15.00 |
| | | | | | | | | | | | | | |
| | | | | | | | | | | | | | |
| 2-lamp High-Performance HPT8 Troffer or high efficiency retrofit reflective troffer | $5.00 | 24000 | $2.67 | $32.50 | 70000 | $15.00 | 3-Lamp F32T8 w/ Elec. Ballast | $2.50 | 20000 | $2.67 | $15 | 70000 | $15.00 |
| | | | | | | | | | | | | | |

[511] Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, January 2012.

B-3: Reduced Wattage T8 Component Costs and Lifetime[512]

| EE Measure Description | EE Lamp Cost | EE Lamp Life (hrs) | Baseline Description | Base Lamp Cost | Base Lamp Life (hrs) | Base Lamp Rep. Labor Cost |
|---|---|---|---|---|---|---|
| RWT8 - F28T8 Lamp | $4.50 | 30000 | F32T8 Standard Lamp | $2.50 | 15000 | $2.67 |
| RWT8 - F28T8 Extra Life Lamp | $4.50 | 36000 | F32T8 Standard Lamp | $2.50 | 15000 | $2.67 |
| RWT8 - F32/25W T8 Lamp | $4.50 | 30000 | F32T8 Standard Lamp | $2.50 | 15000 | $2.67 |
| RWT8 - F32/25W T8 Lamp Extra Life | $4.50 | 36000 | F32T8 Standard Lamp | $2.50 | 15000 | $2.67 |
| RWT8 - F17T8 Lamp - 2 Foot | $4.80 | 18000 | F17T8 Standard Lamp - 2 foot | $2.80 | 15000 | $2.67 |
| RWT8 - F25T8 Lamp - 3 Foot | $5.10 | 18000 | F25T8 Standard Lamp - 3 foot | $3.10 | 15000 | $2.67 |
| RWT8 - F30T8 Lamp - 6" Utube | $11.31 | 24000 | F32T8 Standard Utube Lamp | $9.31 | 15000 | $2.67 |
| RWT8 - F29T8 Lamp - Utube | $11.31 | 24000 | F32T8 Standard Utube Lamp | $9.31 | 15000 | $2.67 |
| RWT8 - F96T8 Lamp - 8 Foot | $9.00 | 24000 | F96T8 Standard Lamp - 8 foot | $7.00 | 15000 | $2.67 |

[512] Adapted from EVT Technical Resource Manual, 2012-75, page 85.

C-1: T12 Baseline Adjustment:

For measures installed up to 6/1/2016, the full savings (as calculated above in the Algorithm section) will be claimed up to 6/1/2016. A savings adjustment will be applied to the annual savings for the remainder of the measure life. The adjustment to be applied for each measure is listed in the reference table below.

Savings Adjustment Factors

| EE Measure Description | Savings Adjustment T12 EEmag ballast and 34 w lamps to HPT8 | Savings Adjustment T12 EEmag ballast and 40 w lamps to HPT8 | Savings Adjustment T12 mag ballast and 40 w lamps to HPT8 |
|---|---|---|---|
| 1-Lamp Relamp/Reballast T12 to HPT8 | 47% | 30% | 20% |
| 2-Lamp Relamp/Reballast T12 to HPT8 | 53% | 30% | 22% |
| 3-Lamp Relamp/Reballast T12 to HPT8 | 42% | 38% | 21% |
| 4-Lamp Relamp/Reballast T12 to HPT8 | 44% | 29% | 23% |

Measures installed in 2012 will claim full savings for four years, 2013 for three years, 2014 two years and 2015 one year. Savings adjustment factors will be applied to the full savings for savings starting in 6/12016 and for the remainder of the measure life. The savings adjustment is equal to the ratio between wattage reduction from T8 baseline to HPT8 and wattage reduction from T12 EE ballast with 40 w lamp baseline from the table 'T8 New and Baseline Assumptions'.[513]

Example: 2 lamp T8 to 2 lamp HPT8 retrofit saves 10 watts, while the T12 EE with 40 w lamp to HPT8 saves 33 watts. Thus the ratio of wattage reduced is 30%. Thus the ratio of wattage reduced is 30%.

**MEASURE CODE: CI-LTG-T8FX-V04-150601**

[513] Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, October 26, 2011
EPE Program Downloads. Web accessed http://www.epelectricefficiency.com/downloads.asp?section=ci download Copy of LSF_2012_v4.04_250rows.xls.
Kuiken et al, Focus on Energy Evaluation. Business Programs: Deemed Savings Manual v1.0, Kema, march 22, 2010 available at http://www.focusonenergy.com/files/Document_Management_System/Evaluation/bpdeemedsavingsmanuav10_evaluationreport.pdf Based on ComEd's BILD program data from PY4 and PY5. For Residential installations, hours of use assumptions from '5.5.6 LED Downlights' should be used for LED fixtures and '5.5.8 LED Screw Based Omnidirectional Bulbs' should be used for LED bulbs.

## 4.5.4 LED Bulbs and Fixtures

### DESCRIPTION

This characterization provides savings assumptions for a variety of LED lamps including Omnidirectional (e.g. A-Type lamps), Decorative (e.g. Globes and Torpedoes) and Directional (PAR Lamps, Reflectors, MR16), and fixtures including refrigerated case, recessed and outdoor/garage fixtures.

If the implementation strategy does not allow for the installation location to be known, a deemed split of 96% Commercial and 4% Residential should be used[514].

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, new lamps must be Energy Star labeled. Lamps and fixtures should be found in the reference tables below. Fixtures must be Energy Star labeled or on the Design Lights Consortium qualifying fixture list.

### DEFINITION OF BASELINE EQUIPMENT

Refer to the baseline tables. In 2012, Federal legislation stemming from the Energy Independence and Security Act of 2007 (EIAS) required all general-purpose light bulbs between 40 watts and 100 watts to have ~30% increased efficiency, essentially phasing out standard incandescent technology. In 2012, the 100 w lamp standards apply; in 2013 the 75 w lamp standards will apply, followed by restrictions on the 60 w and 40 w lamps in 2014.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

Lifetime is the life of the product, at the reported operating hours (lamp life in hours divided by operating hours per year – see reference table "LED component Costs and Lifetime." The analysis period is the same as the lifetime, capped at 15 years. (15 years from GDS Measure Life Report, June 2007).

### DEEMED MEASURE COST

Wherever possible, actual incremental costs should be used. Refer to reference table “LED component Cost & Lifetime” for defaults.

### LOADSHAPE

Loadshape C06 - Commercial Indoor Lighting

Loadshape C07 - Grocery/Conv. Store Indoor Lighting

Loadshape C08 - Hospital Indoor Lighting

Loadshape C09 - Office Indoor Lighting

Loadshape C10 - Restaurant Indoor Lighting

Loadshape C11 - Retail Indoor Lighting

[514] Based on final ComEd’s BILD program data from PY4,PY5 and PY6. For Residential installations, hours of use assumptions from ‘5.5.6 LED Downlights’ should be used for LED fixtures and ‘5.5.8 LED Screw Based Omnidirectional Bulbs’ should be used for LED bulbs.

Loadshape C12 - Warehouse Indoor Lighting

Loadshape C13 - K-12 School Indoor Lighting

Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)

Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)

Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)

Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)

Loadshape C18 - Industrial Indoor Lighting

Loadshape C19 - Industrial Outdoor Lighting

Loadshape C20 - Commercial Outdoor Lighting

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the location type. Values are provided for each building type in the reference section below.

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = ((Watts_{base}-Watts_{EE})/1000) * Hours * WHF_e * ISR$$

Where:

$Watts_{base}$ = Input wattage of the existing or baseline system. Reference the "LED New and Baseline Assumptions" table for default values.

$Watts_{EE}$ = Actual wattage of LED purchased / installed. If unknown, use default provided below:

For ENERGY STAR rated lamps the following lumen equivalence tables should be used:

**Omnidirectional Lamps - ENERGY STAR Minimum Luminous Efficacy = 50Lm/W for <10W lamps and 55Lm/W for >=10W lamps.**

| Minimum Lumens | Maximum Lumens | Lumens used to calculate LED Wattage (midpoint) | LED Wattage[515] (WattsEE) | Baseline 2014-2019 (WattsBase) | Delta Watts 2014-2019 (WattsEE) | Baseline Post EISA 2020 requirement[516] (WattsBase) | Delta Watts Post 2020 (WattsEE) |
|---|---|---|---|---|---|---|---|
| 5280 | 6209 | 5745 | 104.4 | 300.0 | 195.6 | 300.0 | 195.6 |
| 3000 | 5279 | 4140 | 75.3 | 200.0 | 124.7 | 200.0 | 124.7 |
| 2601 | 2999 | 2800 | 50.9 | 150.0 | 99.1 | 150.0 | 99.1 |
| 1490 | 2600 | 2045 | 37.2 | 72.0 | 34.8 | 45.4 | 8.3 |
| 1050 | 1489 | 1270 | 23.1 | 53.0 | 29.9 | 28.2 | 5.1 |
| 750 | 1049 | 900 | 16.4 | 43.0 | 26.6 | 20.0 | 3.6 |
| 310 | 749 | 530 | 9.6 | 29.0 | 19.4 | 11.8 | 2.1 |
| 250 | 309 | 280 | 5.6 | 25.0 | 19.4 | 25.0 | 19.4 |

**Decorative Lamps - ENERGY STAR Minimum Luminous Efficacy = 40Lm/W for all lamps**

| Nominal wattage of lamp to be replaced ($Watts_{base}$) | Minimum initial light output of LED lamp (lumens) | LED Wattage ($Watts_{EE}$) | Delta Watts |
|---|---|---|---|
| 10 | 70 | 1.75 | 8.25 |
| 15 | 90 | 2.25 | 12.75 |
| 25 | 150 | 3.75 | 21.25 |
| 40 | 300 | 7.5 | 32.5 |
| 60 | 500 | 12.5 | 47.5 |

Decorative lamps are exempt from EISA regulations.

[515] Based on ENERGY STAR specs – minimum luminous efficacy for Omnidirectional Lamps. For LED lamp power <10W = 50lm/W and for LED lamp power >=10W = 55lm/W.
[516] Calculated as 45lm/W for all EISA non-exempt bulbs.

**Directional Lamps -** ENERGY STAR Minimum Luminous Efficacy = 40Lm/W for lamps with rated wattages less than 20Wand 50 Lm/W for lamps with rated wattages >= 20 watts[517].

For Directional R, BR, and ER lamp types[518]:

| Bulb Type | Lower Lumen Range | Upper Lumen Range | $Watts_{Base}$ | Lumens used to calculate LED Wattage (midpoint) | LED Wattage ($Watts_{EE}$) | Delta Watts |
|---|---|---|---|---|---|---|
| **R, ER, BR with medium screw bases w/ diameter >2.25" (*see exceptions below)** | 420 | 472 | 40 | 446 | 11 | 29 |
| | 473 | 524 | 45 | 499 | 12 | 33 |
| | 525 | 714 | 50 | 620 | 15 | 35 |
| | 715 | 937 | 65 | 826 | 21 | 44 |
| | 938 | 1259 | 75 | 1099 | 22 | 53 |
| | 1260 | 1399 | 90 | 1330 | 27 | 63 |
| | 1400 | 1739 | 100 | 1570 | 31 | 69 |
| | 1740 | 2174 | 120 | 1957 | 39 | 81 |
| | 2175 | 2624 | 150 | 2400 | 48 | 102 |
| | 2625 | 2999 | 175 | 2812 | 56 | 119 |
| | 3000 | 4500 | 200 | 3750 | 75 | 125 |
| ***R, BR, and ER with medium screw bases w/ diameter <=2.25"** | 400 | 449 | 40 | 425 | 11 | 29 |
| | 450 | 499 | 45 | 475 | 12 | 33 |
| | 500 | 649 | 50 | 575 | 14 | 36 |
| | 650 | 1199 | 65 | 925 | 23 | 42 |
| ***ER30, BR30, BR40, or ER40** | 400 | 449 | 40 | 425 | 11 | 29 |
| | 450 | 499 | 45 | 475 | 12 | 33 |
| | 500 | 649 | 50 | 575 | 14 | 36 |
| ***BR30, BR40, or ER40** | 650 | 1419 | 65 | 1035 | 21 | 44 |
| ***R20** | 400 | 449 | 40 | 425 | 11 | 29 |
| | 450 | 719 | 45 | 585 | 15 | 30 |
| ***All reflector lamps below lumen ranges specified above** | 200 | 299 | 20 | 250 | 6 | 14 |
| | 300 | 399 | 30 | 350 | 9 | 21 |

Directional lamps are exempt from EISA regulations.

[517] From pg 10 of the Energy Star Specification for lamps v1.1
[518] From pg 11 of the Energy Star Specification for lamps v1.1

For PAR, MR, and MRX Lamps Types:

For these highly focused directional lamp types, it is necessary to have Center Beam Candle Power (CBCP) and beam angle measurements to accurately estimate the equivalent baseline wattage. The formula below is based on the Energy Star Center Beam Candle Power tool.[519] If CBCP and beam angle information are not available, refer to the R, BR, and ER lumen based method above.

$$\text{Wattsbase} = 375.1 - 4.355(D) - \sqrt{227{,}800 - 937.9(D) - 0.9903(D^2) - 1479(BA) - 12.02(D * BA) + 14.69(BA^2) - 16{,}720 * \ln(CBCP)}$$

Where:

D = Bulb diameter (e.g. for PAR20 D = 20)

BA = Beam angle

CBCP = Center beam candle power

The result of the equation above should be rounded DOWN to the nearest wattage established by Energy Star:

| Diameter | Permitted Wattages |
|---|---|
| 16 | 20, 35, 40, 45, 50, 60, 75 |
| 20 | 50 |
| 30S | 40, 45, 50, 60, 75 |
| 30L | 50, 75 |
| 38 | 40, 45, 50, 55, 60, 65, 75, 85, 90, 100, 120, 150, 250 |

Hours = Average hours of use per year are provided in the Reference Table in Section 4.5, Screw based bulb annual operating hours, for each building type. If unknown, use the Miscellaneous value.

WHFe = Waste heat factor for energy to account for cooling energy savings from efficient lighting are provided below for each building type in the Referecne Table in Section 4.5. If unknown, use the Miscellaneous value.

ISR = In Service Rate -the percentage of units rebated that actually get installed.

=100%[520] if application form completed with sign off that equipment is not placed into storage. If sign off form not completed assume the following 3 year ISR assumptions:

[519] http://energystar.supportportal.com/link/portal/23002/23018/Article/32655/

[520] Illinois evaluation of PY1 through PY3 has not found that fixtures or lamps placed into storage to be a significant enough issue to warrant including an “In-Service Rate” when commercial customers complete an application form.

| Weighted Average 1st year In Service Rate (ISR) | 2nd year Installations | 3rd year Installations | Final Lifetime In Service Rate |
|---|---|---|---|
| 95.7%[521] | 1.2% | 1.1% | 98.0%[522] |

**HEATING PENALTY**

If electrically heated building:

$$\Delta kWh_{heatpenalty}^{523} = (((WattsBase-WattsEE)/1000) * ISR * Hours * -IFkWh$$

Where:

IFkWh = Lighting-HVAC Interation Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficent lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, For example, a 9W LED lamp, 450 lumens, is installed in a heat pump heated office in 2014 and sign off form provided:

$$\Delta kWh_{heatpenalty} = ((29-9/1000)*1.0*3088* -0.151$$

$$= - 9.3 \text{ kWh}$$

**DEFERRED INSTALLS**

As presented above, if a sign off form is not completed the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

Year 1 (Purchase Year) installs: Characterized using assumptions provided above or evaluated assumptions if available.

Year 2 and 3 installs: Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.

The NTG factor for the Purchase Year should be applied.

[521] Based on ComEd's BILD program data from PY5 and PY6, see "IL Commercial Lighting ISR_2014.xls".

[522] In the absence of any data for LEDs specifically it is assumed that the same proportion of bulbs eventually get installed as for CFLS. The 98% CFL assumption is based upon review of two evaluations:
'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings. Note that this Final Install Rate does NOT account for leakage of purchased bulbs being installed outside of the utility territory. EM&V should assess how and if data from evaluation should adjust this final installation rate to account for this impact

[523] Negative value because this is an increase in heating consumption due to the efficient lighting.

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = ( (Watts_{base}-Watts_{EE})/1000) * ISR * WHF_d * CF$$

Where:

WHFd = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is provided in Referecne Table in Section 4.5. If unknown, use the Miscellaneous value.

CF = Summer Peak Coincidence Factor for measure is provided in the Referecne Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, For example, a 9W LED lamp, 450 lumens, is installed in an office in 2014 and sign off form provided:

ΔkW = ((29-9/1000)* 1.0*1.3*0.66

= 0.002 kW

### NATURAL GAS ENERGY SAVINGS

Heating Penalty if fossil fuel heated building (or if heating fuel is unknown):

$$\Delta Therms = (((WattsBase-WattsEE)/1000) * ISR * Hours * - IFTherms$$

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Referecne Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, For example, a 9W LED lamp, 450 lumens, is installed in an office in 2014 and sign off form provided:

ΔTherms = ((29-9/1000)*1.0*3088* -0.016

= - 0.99 therms

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

For all measures except Standard Omnidirectional lamps (which have an EISA baseline shift) the individual component lifetimes and costs are provided in the reference table section below[524].

In order to account for the falling EISA Qualified bulb replacement cost provided above, an equivalent annual levelized baseline replacement cost over the lifetime of the LED bulb (assumed to be 25,000/4576 =5.5 years) is calculated (see "C&I OmniDirectional LED O&M Calc.xls"). The key assumptions used in this calculation are documented below[525]:

[524] See "LED reference tables.xls" for breakdown of component cost assumptions.

[525] Based upon pricing forecast developed by Applied Proactive Technologies Inc (APT) based on industry input and provided to

| | Std Inc. | EISA Compliant Halogen | CFL |
|---|---|---|---|
| 2014 | $0.34 | $1.25 | N/A |
| 2015 | $0.34 | $0.90 | N/A |
| 2016 | $0.34 | $0.80 | N/A |
| 2017 | $0.34 | $0.70 | N/A |
| 2018 | $0.34 | $0.60 | N/A |
| 2019 | $0.34 | $0.60 | N/A |
| 2020 & after | $0.34 | N/A | $2.50 |

The NPV for replacement lamps and annual levelized replacement costs using the statewide real discount rate of 5.23% are presented below:

| Location | Lumen Level | NPV of replacement costs for period | | | Levelized annual replacement cost savings | | |
|---|---|---|---|---|---|---|---|
| | | June 2014 - May 2015 | June 2015 - May 2016 | June 2016 - May 2017 | June 2014 - May 2015 | June 2015 - May 2016 | June 2016 - May 2017 |
| Commercial | Lumens <310 or >2600 (EISA exempt) | $6.94 | $6.94 | $6.94 | $1.49 | $1.49 | $1.49 |
| | Lumens ≥ 310 and ≤ 2600 (EISA compliant) | $16.86 | $13.90 | $11.51 | $3.63 | $2.99 | $2.48 |
| Multi Family Common Areas | Lumens <310 or >2600 (non-EISA compliant) | $7.13 | $7.13 | $7.13 | $1.93 | $1.93 | $1.93 |
| | Lumens ≥ 310 and ≤ 2600 (EISA compliant) | $18.75 | $15.57 | $13.79 | $5.09 | $4.22 | $3.74 |

For halogen bulbs, we assume the same replacement cycle as incandescent bulbs.[526] The replacement cycle is based on the miscellaneous hours of use. Both incandescent and halogen lamps are assumed to last for 1,000 hours before needing replacement and CFLs after 10,000 hours.

Ameren.

[526] The manufacturers of the new minimally compliant EISA Halogens are using regular incandescent lamps with halogen fill gas rather than halogen infrared to meet the standard and so the component rated life is equal to the standard incandescent.

LED New and Baseline Assumptions[527]

| LED Measure Description | WattsEE | Baseline Description | WattsBASE | Basis for Watt Assumptions | LED Lamp Cost | Baseline Cost (EISA 2012-2014, EISA 2020) | Incremental Cost (EISA 2012-2014, EISA 2020) | LED Minimum Lamp Life (hrs) |
|---|---|---|---|---|---|---|---|---|
| LED Screw and Pin-based Bulbs, Omnidirectional, < 10W | See tables above | | | | $30.00 | $0.34 ($1.25, $2.50) | $29.66 ($28.75, $27.50) | 25,000 |
| LED Screw and Pin-based Bulbs, Omnidirectional, >= 10W | | | | | $40.00 | $0.34 ($1.25, $2.50) | $39.66 ($38.75, $37.50) | 25,000 |
| LED Screw and Pin-based Bulbs, Decorative | | | | | $30.00 | $1.00 | $29.00 | 25,000 |
| LED Screw-based Bulbs, Directional, < 15W | | | | | $45.00 | $5.00 | $40.00 | 35,000 |
| LED Screw-based Bulbs, Directional, >= 15W | | | | | $55.00 | $5.00 | $50.00 | 35,000 |
| LED Recessed, Surface, Pendant Downlights | 17.6 | Baseline LED Recessed, Surface, Pendant Downlights | 54.3 | 2008-2010 EVT Historical Data of 947 Measures | 50,000 | | $50.00 | |
| LED Track Lighting | 12.2 | Baseline LED Track Lighting | 60.4 | 2008-2010 EVT Historical Data of 242 Measures | 50,000 | | $100.00 | |

[527] Data is based on Efficiency Vermont derived cost and actual installed wattage information.

| LED Measure Description | WattsEE | Baseline Description | WattsBASE | Basis for Watt Assumptions | LED Lamp Cost | Baseline Cost (EISA 2012-2014, EISA 2020) | Incremental Cost (EISA 2012-2014, EISA 2020) | LED Minimum Lamp Life (hrs) |
|---|---|---|---|---|---|---|---|---|
| LED Wall-Wash Fixtures | 8.3 | Baseline LED Wall-Wash Fixtures | 17.7 | 2008-2010 EVT Historical Data of 220 Measures | 50,000 | | $80.00 | |
| LED Portable Desk/Task Light Fixtures | 7.1 | Baseline LED Portable Desk/Task Light Fixtures | 36.2 | 2008-2010 EVT Historical Data of 21 Measures | 50,000 | | $50.00 | |
| LED Undercabinet Shelf-Mounted Task Light Fixtures (per foot) | 7.1 | Baseline LED Undercabinet Shelf-Mounted Task Light Fixtures | 36.2 | 2008-2010 EVT Historical Data of 21 Measures | 50,000 | | $25.00 | |
| LED Refrigerated Case Light, Horizontal or Vertical (per foot of light bar) | 7.6 | Baseline LED Refrigerated Case Light, Horizontal or Vertical (per foot of light bar) | 15.2 | PG&E Refrigerated Case Study[528] normalized to per foot of light bar. | 50,000 | | $50.00 | |
| LED Freezer Case Light, Horizontal or Vertical (per foot) | 7.7 | Baseline LED Freezer Case Light, Horizontal or Vertical (per foot) | 18.7 | PG&E Refrigerated Case Study normalized to per foot. | 50,000 | | $50.00 | |

[528] LED Refrigeration Case Ltg Workpaper 053007 rev1, May 30, 2007

| LED Measure Description | WattsEE | Baseline Description | WattsBASE | Basis for Watt Assumptions | LED Lamp Cost | Baseline Cost (EISA 2012-2014, EISA 2020) | Incremental Cost (EISA 2012-2014, EISA 2020) | LED Minimum Lamp Life (hrs) |
|---|---|---|---|---|---|---|---|---|
| LED Display Case Light Fixture (per foot) | 7.1 | Baseline LED Display Case Light Fixture | 36.2 | Modeled after LED Undercabinet Shelf-Mounted Task Light Fixtures (per foot) | 35,000 | | $25.00 | |
| LED 2x2 Recessed Light Fixture | 44.9 | T8 U-Tube 2L-FB32 w/ Elec - 2' | 61.0 | Based on average watts of DLC qualified products as of 11/21/11 | 35,000 | | $75.00 | |
| LED 2x4 Recessed Light Fixture | 53.6 | T8 3L-F32 w/ Elec - 4' | 88.0 | Based on average watts of DLC qualified products as of 11/21/11 | 35,000 | | $125.00 | |
| LED 1x4 Recessed Light Fixture | 32.2 | T8 2L-F32 w/ Elec - 4' | 59.0 | Based on average watts of DLC qualified products as of 11/21/11 | 35,000 | | $100.00 | |
| LED High- and Low-Bay Fixtures | 160.2 | MH 250 W CWA Pulse Start | 295.0 | Based on average watts of DLC qualified products as of 11/21/11 | 35,000 | | $200.00 | |
| LED Outdoor Pole/Arm Mounted Parking/Roadway, < 30W | 18.6 | Baseline LED Outdoor Pole/Arm Mounted Parking/Roadway, < 30W | 124.3 | 2008-2010 EVT Historical Data of 2,813 Measures | 50,000 | | $125.00 | |

| LED Measure Description | WattsEE | Baseline Description | WattsBASE | Basis for Watt Assumptions | LED Lamp Cost | Baseline Cost (EISA 2012-2014, EISA 2020) | Incremental Cost (EISA 2012-2014, EISA 2020) | LED Minimum Lamp Life (hrs) |
|---|---|---|---|---|---|---|---|---|
| LED Outdoor Pole/Arm Mounted Parking/Roadway, 30W - 75W | 52.5 | Baseline LED Outdoor Pole/Arm Mounted Parking/Roadway, 30W - 75W | 182.9 | 2008-2010 EVT Historical Data of 1,081 Measures | 50,000 | | $250.00 | |
| LED Outdoor Pole/Arm Mounted Parking/Roadway, >= 75W | 116.8 | Baseline LED Outdoor Pole/Arm Mounted Parking/Roadway, >= 75W | 361.4 | 2008-2010 EVT Historical Data of 806 Measures | 50,000 | | $375.00 | |
| LED Outdoor Pole/Arm Mounted Decorative Parking/Roadway, < 30W | 18.6 | Baseline LED Outdoor Pole/Arm Mounted Decorative Parking/Roadway, < 30W | 124.3 | 2008-2010 EVT Historical Data of 2,813 Measures | 50,000 | | $125.00 | |
| LED Outdoor Pole/Arm Mounted Decorative Parking/Roadway, 30W - 75W | 52.5 | Baseline LED Outdoor Pole/Arm Mounted Decorative Parking/Roadway, 30W - 75W | 182.9 | 2008-2010 EVT Historical Data of 1,081 Measures | 50,000 | | $250.00 | |
| LED Outdoor Pole/Arm Mounted Decorative Parking/Roadway, >= 75W | 116.8 | Baseline LED Outdoor Pole/Arm Mounted Decorative Parking/Roadway, >= 75W | 361.4 | 2008-2010 EVT Historical Data of 806 Measures | 50,000 | | $375.00 | |
| LED Parking Garage/Canopy, < 30W | 18.6 | Baseline LED Parking Garage/Canopy, < 30W | 124.3 | 2008-2010 EVT Historical Data of | 50,000 | | $125.00 | |

| LED Measure Description | WattsEE | Baseline Description | WattsBASE | Basis for Watt Assumptions | LED Lamp Cost | Baseline Cost (EISA 2012-2014, EISA 2020) | Incremental Cost (EISA 2012-2014, EISA 2020) | LED Minimum Lamp Life (hrs) |
|---|---|---|---|---|---|---|---|---|
| | | | | 2,813 Measures | | | | |
| LED Parking Garage/Canopy, 30W - 75W | 52.5 | Baseline LED Parking Garage/Canopy, 30W - 75W | 182.9 | 2008-2010 EVT Historical Data of 1,081 Measures | 50,000 | | $250.00 | |
| LED Parking Garage/Canopy, >= 75W | 116.8 | Baseline LED Parking Garage/Canopy, >= 75W | 361.4 | 2008-2010 EVT Historical Data of 806 Measures | 50,000 | | $375.00 | |
| LED Wall-Mounted Area Lights, < 30W | 18.6 | Baseline LED Wall-Mounted Area Lights, < 30W | 124.3 | 2008-2010 EVT Historical Data of 2,813 Measures | 50,000 | | $125.00 | |
| LED Wall-Mounted Area Lights, 30W - 75W | 52.5 | Baseline LED Wall-Mounted Area Lights, 30W - 75W | 182.9 | 2008-2010 EVT Historical Data of 1,081 Measures | 50,000 | | $250.00 | |
| LED Wall-Mounted Area Lights, >= 75W | 116.8 | Baseline LED Wall-Mounted Area Lights, >= 75W | 361.4 | 2008-2010 EVT Historical Data of 806 Measures | 50,000 | | $375.00 | |
| LED Bollard, < 30W | 13.9 | Baseline LED Bollard, < 30W | 54.3 | 2008-2010 EVT Historical Data of 33 Measures | 50,000 | | $150.00 | |
| LED Bollard, >= 30W | 41.0 | Baseline LED Bollard, >= 30W | 78.0 | 2008-2010 EVT Historical Data of 15 Measures | 50,000 | | $250.00 | |

| LED Measure Description | WattsEE | Baseline Description | WattsBASE | Basis for Watt Assumptions | LED Lamp Cost | Baseline Cost (EISA 2012-2014, EISA 2020) | Incremental Cost (EISA 2012-2014, EISA 2020) | LED Minimum Lamp Life (hrs) |
|---|---|---|---|---|---|---|---|---|
| LED Flood Light, < 15W | 8.7 | Baseline LED Flood Light, < 15W | 51.7 | Consistent with LED Screw-base Directional | 50,000 | | $35.00 | |
| LED Flood Light, >= 15W | 16.2 | Baseline LED Flood Light, >= 15W | 64.4 | Consistent with LED Screw-base Directional | 50,000 | | $45.00 | |

LED Component Costs & Lifetime[529]

| LED Measure Description | LED Minimum Lamp Life (hrs) | LED Lamp Cost Total | LED Driver Life (hrs) | LED Driver Cost Total | Baseline Technology (1) | Lamp (1) Life (hrs) | Lamp (1) Total Cost | Ballast (1) Life (hrs) | Ballast (1) Total Cost | Baseline Technology (2) | Lamp (2) Life (hrs) | Lamp (2) Total Cost |
|---|---|---|---|---|---|---|---|---|---|---|---|---|
| LED Screw and Pin-based Bulbs, Decorative | 25,000 | N/A | N/A | N/A | 53W EISA Halogen | 2,000 | $4.67 | N/A | N/A | N/A | N/A | N/A |
| LED Screw-based Bulbs, Directional, < 15W | 35,000 | N/A | N/A | N/A | 15% CFL 18W Pin Base | 10,000 | $11.62 | 40,000 | $36.00 | 85% Halogen PAR20 | 2,500 | $12.67 |
| LED Screw-based Bulbs, Directional, >= 15W | 35,000 | N/A | N/A | N/A | 15% CFL 26W Pin Base | 10,000 | $12.62 | 40,000 | $36.00 | 85% Halogen PAR30/38 | 2,500 | $12.67 |
| LED Recessed, Surface, Pendant Downlights | 50,000 | $47.50 | 70,000 | $47.50 | 40% CFL 26W Pin Base | 10,000 | $12.62 | 40,000 | $36.00 | 60% Halogen PAR30/38 | 2,500 | $12.67 |
| LED Track Lighting | 50,000 | $47.50 | 70,000 | $47.50 | 10% CMH PAR38 | 12,000 | $62.92 | 40,000 | $110.00 | 90% Halogen PAR38 | 2,500 | $12.67 |
| LED Wall-Wash Fixtures | 50,000 | $47.50 | 70,000 | $47.50 | 40% CFL 42W Pin Base | 10,000 | $15.72 | 40,000 | $67.50 | 60% Halogen PAR38 | 2,500 | $12.67 |
| LED Portable Desk/Task Light Fixtures | 50,000 | $47.50 | 70,000 | $47.50 | 50% 13W CFL Pin Base | 10,000 | $5.52 | 40,000 | $25.00 | 50% 50W Halogen | 2,500 | $12.67 |
| LED Undercabinet Shelf-Mounted Task Light Fixtures (per foot) | 50,000 | $47.50 | 70,000 | $47.50 | 50% 2' T5 Linear | 7,500 | $9.92 | 40,000 | $45.00 | 50% 50W Halogen | 2,500 | $12.67 |

[529] Note some measures have blended baselines. All values are provided to enable calculation of appropriate O&M impacts. Total costs include lamp, labor and disposal cost assumptions where applicable, see "LED reference tables.xls" for more information.

| LED Measure Description | LED Minimum Lamp Life (hrs) | LED Lamp Cost Total | LED Driver Life (hrs) | LED Driver Cost Total | Baseline Technology (1) | Lamp (1) Life (hrs) | Lamp (1) Total Cost | Ballast (1) Life (hrs) | Ballast (1) Total Cost | Baseline Technology (2) | Lamp (2) Life (hrs) | Lamp (2) Total Cost |
|---|---|---|---|---|---|---|---|---|---|---|---|---|
| LED Refrigerated Case Light, Horizontal or Vertical (per foot) | 50,000 | $9.50 | 70,000 | $9.50 | 5' T8 | 15,000 | $2.77 | 40,000 | $9.50 | N/A | N/A | N/A |
| LED Freezer Case Light, Horizontal or Vertical (per foot) | 50,000 | $8.75 | 70,000 | $7.92 | 6' T12HO | 12,000 | $11.03 | 40,000 | $59.58 | N/A | N/A | N/A |
| LED Display Case Light Fixture (per foot) | 35,000 | $47.50 | 70,000 | $28.75 | 50% 2' T5 Linear | 7,500 | $9.92 | 40,000 | $45.00 | 50% 50W Halogen | 2,500 | $12.67 |
| LED 2x2 Recessed Light Fixture | 35,000 | $47.50 | 70,000 | $47.50 | T8 U-Tube 2L-FB32 w/ Elec - 2' | 15,000 | $24.95 | 40,000 | $52.00 | N/A | N/A | N/A |
| LED 2x4 Recessed Light Fixture | 35,000 | $72.50 | 70,000 | $47.50 | T8 3L-F32 w/ Elec - 4' | 15,000 | $17.00 | 40,000 | $35.00 | N/A | N/A | N/A |
| LED 1x4 Recessed Light Fixture | 35,000 | $47.50 | 70,000 | $47.50 | T8 2L-F32 w/ Elec - 4' | 15,000 | $11.33 | 40,000 | $35.00 | N/A | N/A | N/A |
| LED High- and Low-Bay Fixtures | 35,000 | $112.50 | 70,000 | $62.50 | 250W MH | 10,000 | $41.25 | 40,000 | $130.25 | N/A | N/A | N/A |
| LED Outdoor Pole/Arm Mounted Parking/Roadway, < 30W | 50,000 | $62.50 | 70,000 | $62.50 | 100W MH | 10,000 | $54.25 | 40,000 | $166.70 | N/A | N/A | N/A |
| LED Outdoor Pole/Arm Mounted Parking/Roadway, 30W - | 50,000 | $87.50 | 70,000 | $62.50 | 175W MH | 10,000 | $48.25 | 40,000 | $110.00 | N/A | N/A | N/A |

| LED Measure Description | LED Minimum Lamp Life (hrs) | LED Lamp Cost Total | LED Driver Life (hrs) | LED Driver Cost Total | Baseline Technology (1) | Lamp (1) Life (hrs) | Lamp (1) Total Cost | Ballast (1) Life (hrs) | Ballast (1) Total Cost | Baseline Technology (2) | Lamp (2) Life (hrs) | Lamp (2) Total Cost |
|---|---|---|---|---|---|---|---|---|---|---|---|---|
| 75W | | | | | | | | | | | | |
| LED Outdoor Pole/Arm Mounted Parking/Roadway, >= 75W | 50,000 | $112.50 | 70,000 | $62.50 | 250W MH | 10,000 | $41.25 | 40,000 | $130.25 | N/A | N/A | N/A |
| LED Outdoor Pole/Arm Mounted Decorative Parking/Roadway, < 30W | 50,000 | $62.50 | 70,000 | $62.50 | 100W MH | 10,000 | $54.25 | 40,000 | $166.70 | N/A | N/A | N/A |
| LED Outdoor Pole/Arm Mounted Decorative Parking/Roadway, 30W - 75W | 50,000 | $87.50 | 70,000 | $62.50 | 175W MH | 10,000 | $48.25 | 40,000 | $110.00 | N/A | N/A | N/A |
| LED Outdoor Pole/Arm Mounted Decorative Parking/Roadway, >= 75W | 50,000 | $112.50 | 70,000 | $62.50 | 250W MH | 10,000 | $41.25 | 40,000 | $130.25 | N/A | N/A | N/A |
| LED Parking Garage/Canopy, < 30W | 50,000 | $47.50 | 70,000 | $47.50 | 100W MH | 10,000 | $36.92 | 40,000 | $151.70 | N/A | N/A | N/A |
| LED Parking Garage/Canopy, 30W - 75W | 50,000 | $72.50 | 70,000 | $47.50 | 175W MH | 10,000 | $30.92 | 40,000 | $95.00 | N/A | N/A | N/A |
| LED Parking Garage/Canopy, >= 75W | 50,000 | $97.50 | 70,000 | $47.50 | 250W MH | 10,000 | $23.92 | 40,000 | $115.25 | N/A | N/A | N/A |
| LED Wall-Mounted Area | 50,000 | $47.50 | 70,000 | $47.50 | 100W MH | 10,000 | $36.92 | 40,000 | $151.70 | N/A | N/A | N/A |

| LED Measure Description | LED Minimum Lamp Life (hrs) | LED Lamp Cost Total | LED Driver Life (hrs) | LED Driver Cost Total | Baseline Technology (1) | Lamp (1) Life (hrs) | Lamp (1) Total Cost | Ballast (1) Life (hrs) | Ballast (1) Total Cost | Baseline Technology (2) | Lamp (2) Life (hrs) | Lamp (2) Total Cost |
|---|---|---|---|---|---|---|---|---|---|---|---|---|
| Lights, < 30W | | | | | | | | | | | | |
| LED Wall-Mounted Area Lights, 30W - 75W | 50,000 | $72.50 | 70,000 | $47.50 | 175W MH | 10,000 | $30.92 | 40,000 | $95.00 | N/A | N/A | N/A |
| LED Wall-Mounted Area Lights, >= 75W | 50,000 | $97.50 | 70,000 | $47.50 | 250W MH | 10,000 | $23.92 | 40,000 | $115.25 | N/A | N/A | N/A |
| LED Bollard, < 30W | 50,000 | $47.50 | 70,000 | $47.50 | 50W MH | 10,000 | $36.92 | 40,000 | $135.50 | N/A | N/A | N/A |
| LED Bollard, >= 30W | 50,000 | $72.50 | 70,000 | $47.50 | 70W MH | 10,000 | $36.92 | 40,000 | $142.50 | N/A | N/A | N/A |
| LED Flood Light, < 15W | 50,000 | $47.50 | 70,000 | $47.50 | 25% 50W MH | 10,000 | $36.92 | 40,000 | $135.50 | 75% Halogen PAR20 | 2,500 | $12.67 |
| LED Flood Light, >= 15W | 50,000 | $47.50 | 70,000 | $47.50 | 50% 50W MH | 10,000 | $36.92 | 40,000 | $135.50 | 50% Halogen PAR30/38 | 2,500 | $12.67 |

**MEASURE CODE: CI-LTG-LEDB-V04-150601**

## 4.5.5 Commercial LED Exit Signs

### DESCRIPTION

This measure characterizes the savings associated with installing a Light Emitting Diode (LED) exit sign in place of a fluorescent or incandescent exit sign in a Commercial building. Light Emitting Diode exit signs have a string of very small, typically red or green, glowing LEDs arranged in a circle or oval. The LEDs may also be arranged in a line on the side, top or bottom of the exit sign. LED exit signs provide the best balance of safety, low maintenance, and very low energy usage compared to other exit sign technologies.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is assumed to be an exit sign illuminated by LEDs.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be a fluorescent or incandescent model.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 16 years[530].

### DEEMED MEASURE COST

The incremental cost for this measure is assumed to be $30[531].

### LOADSHAPE

Loadshape C53 - Flat

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 100%[532].

[530] 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, "Effective/Remaining Useful Life Values", California Public Utilities Commission, December 16, 2008.
[531] NYSERDA Deemed Savings Database, Labor cost assumes 25 minutes @ $18/hr.
[532] Assuming continuous operation of an LED exit sign, the Summer Peak Coincidence Factor is assumed to equal 1.0.

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = ((WattsBase - WattsEE) / 1000) * HOURS * WHF_e$$

Where:

WattsBase = Actual wattage if known, if unknown assume the following:

| Baseline Type | WattsBase |
|---|---|
| Incandescent | 35W[533] |
| Fluorescent | 11W[534] |
| Unknown (e.g. time of sale) | 23W[535] |

WattsEE = Actual wattage if known, if unknown assume 2W[536]

HOURS = Annual operating hours

= 8766

$WHF_e$ = Waste heat factor for energy to account for cooling energy savings from efficient lighting are provided for each building type in the Referecne Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, replacing incandescent fixture in an office

$\Delta kWH$ = (35 – 2)/1000 * 8766 * 1.25

= 362 kWh

For example, replacing fluorescent fixture in a hospital

$\Delta kWH$ = (11 – 2)/1000 * 8766 * 1.35

= 106.5 kWh

### HEATING PENALTY

If electrically heated building:

$$\Delta kWh_{heatpenalty}^{537} = (((WattsBase-WattsEE)/1000) * Hours * -IFkWh$$

[533] Based on review of available product.
[534] Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, February, 19, 2010
[535] ComEd has been using a weighted baseline of 70 percent incandescent and 30 percent compact fluorescent, reflecting program experience and a limited sample of evaluation verification findings that we consider to be reasonable (Navigant, through comment period February 2013)
[536] Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, February, 19, 2010

Where:

IFkWh = Lighting-HVAC Interation Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficent lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, replacing incandescent fixture in a heat pump heated office

$\Delta kWh_{heatpenalty}$ = (35 – 2)/1000 * 8766 * -0.151

= -43.7 kWh

For example, replacing fluorescent fixture in a heat pump heated hospital

$\Delta kWh_{heatpenalty}$ = (11 – 2)/1000 * 8766 * -0.104

= -8.2 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$\Delta kW$ = ((WattsBase - WattsEE) / 1000) * $WHF_d$ * CF

Where:

$WHF_d$ = Waste heat factor for demand to account for cooling savings from efficient lighting in cooled buildings is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value..

CF = Summer Peak Coincidence Factor for measure

= 1.0

For example, replacing incandescent fixture in an office

$\Delta kW$ = (35 – 2)/1000 * 1.3 * 1.0

= 0.043 kW

For example, replacing fluorescent fixture in a hospital

$\Delta kW$ = (11 – 2)/1000 * 1.69 * 1.0

= 0.015 kW

**NATURAL GAS SAVINGS**

Heating Penalty if natural gas heated building (or if heating fuel is unknown):

Δtherms = (((WattsBase-WattsEE)/1000) * Hours *- IFTherms

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected

[537]Negative value because this is an increase in heating consumption due to the efficient lighting.

by the efficient lighting. Values are provided in the Referecne Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, replacing incandescent fixture in an office

ΔTherms = (35 – 2)/1000 * 8766 * -0.016

= -4.63 Therms

For example, replacing fluorescent fixture in a hospital

ΔTherms = (11 – 2)/1000 * 8766 * -0.011

= - 0.87 Therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

The annual O&M Cost Adjustment savings should be calculated using the following component costs and lifetimes.

| | Baseline Measures | |
|---|---|---|
| Component | Cost | Life (yrs) |
| Lamp | $7.00[538] | 1.37 years[539] |

**MEASURE CODE: CI-LTG-LEDE-V02-140601**

[538] Consistent with assumption for a Standard CFL bulb with an estimated labor cost of $4.50 (assuming $18/hour and a task time of 15 minutes).
[539] Assumes a lamp life of 12,000 hours and 8766 run hours 12000/8766 = 1.37 years.

## 4.5.6 LED Traffic and Pedestrian Signals

### DESCRIPTION

Traffic and pedestrian signals are retrofitted to be illuminated with light emitting diodes (LED) instead of incandescent lamps. Incentive applies for the replacement or retrofit of existing incandescent traffic signals with new LED traffic and pedestrian signal lamps. Each lamp can have no more than a maximum LED module wattage of 25. Incentives are not available for spare lights. Lights must be hardwired and single lamp replacements are not eligible, with the exception of pedestrian hand signals. Eligible lamps must meet the Energy Star Traffic Signal Specification and the Institute for Transportation Engineers specification for traffic signals.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

Refer to the Table titled 'Traffic Signals Technology Equivalencies' for efficient technology wattage and savings assumptions.

### DEFINITION OF BASELINE EQUIPMENT

Refer to the Table titled 'Traffic Signals Technology Equivalencies' for baseline efficiencies and savings assumptions.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of an LED traffic signal is 100,000 hours (manufacturer's estimate), capped at 10 years.[540] The life in years is calculated by dividing 100,000 hrs by the annual operating hours for the particular signal type.

### DEEMED MEASURE COST

The actual measure installation cost should be used (including material and labor).

### LOADSHAPE

Loadshape C24 - Traffic Signal - Red Balls, always changing or flashing

Loadshape C25 - Traffic Signal - Red Balls, changing day, off night

Loadshape C26 - Traffic Signal - Green Balls, always changing

Loadshape C27 - Traffic Signal - Green Balls, changing day, off night

Loadshape C28 - Traffic Signal - Red Arrows

Loadshape C29 - Traffic Signal - Green Arrows

Loadshape C30 - Traffic Signal - Flashing Yellows

Loadshape C31 - Traffic Signal - "Hand" Don't Walk Signal

Loadshape C32 - Traffic Signal - "Man" Walk Signal

[540] ACEEE, (1998) A Market Transformation Opportunity Assessment for LED Traffic Signals, http://www.cee1.org/gov/led/led-ace3/ace3led.pdf

Loadshape C33 - Traffic Signal - Bi-Modal Walk/Don't Walk

### COINCIDENCE FACTOR[541]

The summer peak coincidence factor (CF) for this measure is dependent on lamp type as below:

| Lamp Type | CF |
|---|---|
| Red Round, always changing or flashing | 0.55 |
| Red Arrows | 0.90 |
| Green Arrows | 0.10 |
| Yellow Arrows | 0.03 |
| Green Round, always changing or flashing | 0.43 |
| Flashing Yellow | 0.50 |
| Yellow Round, always changing | 0.02 |
| "Hand" Don't Walk Signal | 0.75 |
| "Man" Walk Signal | 0.21 |

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = (W_{base} - W_{eff}) \times HOURS / 1000$$

Where:

Wbase =The connected load of the baseline equipment

= see Table 'Traffic Signals Technology Equivalencies'

Weff =The connected load of the baseline equipment

= see Table 'Traffic Signals Technology Equivalencies'

EFLH = annual operating hours of the lamp

= see Table 'Traffic Signals Technology Equivalencies'

1000 = conversion factor (W/kW)

[541] Ibid

**EXAMPLE**

For example, an 8 inch red, round signal:

ΔkWh = ((69 - 7) x 4818) / 1000

= 299 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = (Wbase– Weff) x CF / 1000

Where:

Wbase =The connected load of the baseline equipment

= see Table 'Traffic Signals Technology Equivalencies'

Weff =The connected load of the efficient equipment

= see Table 'Traffic Signals Technology Equivalencies'

CF = Summer Peak Coincidence Factor for measure

**EXAMPLE**

For example, an 8 inch red, round signal:

ΔkW = ((69 – 7) x 0.55) / 1000

= 0.0341 kW

**NATURAL GAS ENERGY SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**REFERENCE TABLES**

Traffic Signals Technology Equivalencies[542]

| Traffic Fixture Type | Fixture Size and Color | Efficient Lamps | Baseline Lamps | HOURS | Efficient Fixture Wattage | Baseline Fixture Wattage | Energy Savings (in kWh) |
|---|---|---|---|---|---|---|---|
| Round Signals | 8" Red | LED | Incandescent | 4818 | 7 | 69 | 299 |
| Round Signals | 12" Red | LED | Incandescent | 4818 | 6 | 150 | 694 |
| Flashing Signal[543] | 8" Red | LED | Incandescent | 4380 | 7 | 69 | 272 |
| Flashing Signal | 12" Red | LED | Incandescent | 4380 | 6 | 150 | 631 |
| Flashing Signal | 8" Yellow | LED | Incandescent | 4380 | 10 | 69 | 258 |
| Flashing Signal | 12" Yellow | LED | Incandescent | 4380 | 13 | 150 | 600 |
| Round Signals | 8" Yellow | LED | Incandescent | 175 | 10 | 69 | 10 |
| Round Signals | 12" Yellow | LED | Incandescent | 175 | 13 | 150 | 24 |
| Round Signals | 8" Green | LED | Incandescent | 3767 | 9 | 69 | 266 |
| Round Signals | 12" Green | LED | Incandescent | 3767 | 12 | 150 | 520 |
| Turn Arrows | 8" Yellow | LED | Incandescent | 701 | 7 | 116 | 76 |
| Turn Arrows | 12" Yellow | LED | Incandescent | 701 | 9 | 116 | 75 |
| Turn Arrows | 8" Green | LED | Incandescent | 701 | 7 | 116 | 76 |
| Turn Arrows | 12" Green | LED | Incandescent | 701 | 7 | 116 | 76 |
| Pedestrian Sign | 12" Hand/Man | LED | Incandescent | 8766 | 8 | 116 | 946 |

Reference specifications for above traffic signal wattages are from the following manufacturers:

1. 8" Incandescent traffic signal bulb: General Electric Traffic Signal Model 17325-69A21/TS
2. 12" Incandescent traffic signal bulb: General Electric Signal Model 35327-150PAR46/TS
3. Incandescent Arrows & Hand/Man Pedestrian Signs: General Electric Traffic Signal Model 19010-116A21/TS
4. 8" and 12" LED traffic signals: Leotek Models TSL-ES08 and TSL-ES12
5. 8" LED Yellow Arrow: General Electric Model DR4-YTA2-01A
6. 8" LED Green Arrow: General Electric Model DR4-GCA2-01A
7. 12" LED Yellow Arrow: Dialight Model 431-3334-001X
8. 12: LED Green Arrow: Dialight Model 432-2324-001X
9. LED Hand/Man Pedestrian Sign: Dialight 430-6450-001X

[542] Technical Reference Manual for Pennsylvania Act 129 Energy Efficiency and Conservation Program and Act 213 Alternative Energy Portfolio Standards. Pennsylvania Public Utility Commission. May 2009
[543] Technical Reference Manual for Ohio, August 6, 2010

**MEASURE CODE: CI-LTG-LEDT-V01-120601**

## 4.5.7 Lighting Power Density

### DESCRIPTION

This measure relates to installation of efficient lighting systems in new construction or substantial renovation of commercial buildings excluding low rise (three stories or less) residential buildings. Substantial renovation is when two or more building systems are renovated, such as shell and heating, heating and lighting, etc. State Energy Code specifies a lighting power density level by building type for both the interior and the exterior. Either the Building Area Method or Space by Space method as defined in IECC 2012 can be used for calculating the Interior Lighting Power Density[544]. The measure consists of a design that is more efficient (has a lower lighting power density in watts/square foot) than code requires. The IECC 2012, which is adopted in Illinois, applies to both new construction and renovation.

This measure was developed to be applicable to the following program types: NC.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the lighting system must be more efficient than the baseline Energy Code lighting power density in watts/square foot for either the interior space or exterior space.

### DEFINITION OF BASELINE EQUIPMENT

The baseline is assumed to be a lighting power density that meets IECC 2012, the State of Illinois Energy Code requirements.

### DEEMED CALCULATION FOR THIS MEASURE

Annual kWh Savings

$$\Delta kWh = (WSFbase-WSFeffic)/1000 * SF * Hours * WHF_e$$

Summer Coincident Peak kW Savings

$$\Delta kW = (WSFbase-WSFeffic)/1000 * SF * CF * WHF_d$$

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years[545]

### DEEMED MEASURE COST

The actual incremental cost over a baseline system will be collected from the customer if possible or developed on a fixture by fixture basis.

### LOADSHAPE

Loadshape C06 - Commercial Indoor Lighting

[544] Refer to the referenced code documents for specifics on calculating lighting power density using either the whole building method (IECC) or the Space by Space method (ASHRAE 90.1).

[545] Measure Life Report, Residential and Commercial/Industrial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

Loadshape C07 - Grocery/Conv. Store Indoor Lighting

Loadshape C08 - Hospital Indoor Lighting

Loadshape C09 - Office Indoor Lighting

Loadshape C10 - Restaurant Indoor Lighting

Loadshape C11 - Retail Indoor Lighting

Loadshape C12 - Warehouse Indoor Lighting

Loadshape C13 - K-12 School Indoor Lighting

Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)

Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)

Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)

Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)

Loadshape C18 - Industrial Indoor Lighting

Loadshape C19 - Industrial Outdoor Lighting

Loadshape C20 - Commercial Outdoor Lighting

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the building type.

**Algorithm**

### CALCULATION OF SAVINGS

### ENERGY SAVINGS

$$\Delta kWh = (WSF_{base}-WSF_{effic})/1000 * SF * Hours * WHF_e$$

Where:

| | |
|---|---|
| $WSF_{base}$ | = Baseline lighting watts per square foot or linear foot as determined by building or space type. Whole building analysis values are presented in the Reference Tables below.[546] |
| $WSF_{effic}$ | = The actual installed lighting watts per square foot or linear foot. |
| SF | = Provided by customer based on square footage of the building area applicable to the lighting design for new building. |
| Hours | = Annual site-specific hours of operation of the lighting equipment collected from the customer. If not available, use building area type as provided in the Reference Table in Section 4.5, Fixture annual operating hours. |
| $WHF_e$ | = Waste Heat Factor for Energy to account for cooling savings from efficient lighting is as |

[546] IECC 2012 - Reference Code documentation for additional information.

provided in the Reference Table in Section 4.5 by buidling type. If building is not cooled $WHF_e$ is 1.

**HEATING PENALTY**

If electrically heated building:

$$\Delta kWh_{heatpenalty}^{547} = (WSF_{base}-WSF_{effic})/1000 * SF * Hours * -IFkWh$$

Where:

IFkWh = Lighting-HVAC Interation Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficent lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$$\Delta kW = (WSF_{base}-WSF_{effic})/1000 * SF * CF * WHF_d$$

Where:

$WHF_d$ = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is as provided in the Reference Table in Section 4.5 by buidling type. If building is not cooled WHFd is 1.

CF = Summer Peak Coincidence Factor for measure is as provided in the Reference Table in Section 4.5 by buidling type. If the building type is unknown, use the Miscellaneous value of 0.66.

Other factors as defined above

**NATURAL GAS ENERGY SAVINGS**

$$\Delta Therms = (WSF_{base}-WSF_{effic})/1000 * SF * Hours * - IFTherms$$

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. This value is provided in the Reference Table in Section 4.5 by buidling type.

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

[547] Negative value because this is an increase in heating consumption due to the efficient lighting.

**REFERENCE TABLES**

Lighting Power Density Values from IECC 2012 for Interior Commercial New Construction and Substantial Renovation Building Area Method:

| Building Area Type [548] | Lighting Power Density (w/ft$^2$) |
|---|---|
| Automotive Facility | 0.9 |
| Convention Center | 1.2 |
| Court House | 1.2 |
| Dining: Bar Lounge/Leisure | 1.3 |
| Dining: Cafeteria/Fast Food | 1.4 |
| Dining: Family | 1.6 |
| Dormitory | 1.0 |
| Exercise Center | 1.0 |
| Fire station | 0.8 |
| Gymnasium | 1.1 |
| Healthcare – clinic | 1.0 |
| Hospital | 1.2 |
| Hotel | 1.0 |
| Library | 1.3 |
| Manufacturing Facility | 1.3 |
| Motel | 1.0 |
| Motion Picture Theater | 1.2 |
| Multifamily | 0.7 |
| Museum | 1.1 |
| Office | 0.9 |
| Parking Garage | 0.3 |
| Penitentiary | 1.0 |
| Performing Arts Theater | 1.6 |
| Police Station | 1.0 |
| Post Office | 1.1 |
| Religious Building | 1.3 |

[548] IECC 2012 in cases where both a general building area type and a more specific building area type are listed, the more specific building area type shall apply.

| Building Area Type [548] | Lighting Power Density (w/ft$^2$) |
|---|---|
| Retail[549] | 1.4 |
| School/University | 1.2 |
| Sports Arena | 1.1 |
| Town Hall | 1.1 |
| Transportation | 1.0 |
| Warehouse | 0.6 |
| Workshop | 1.4 |

Lighting Power Density Values from IECC 2012 for Interior Commercial New Construction and Substantial Renovation Space by Space Method:

[549] Where lighting equipment is specified to be installed to highlight specific merchandise in addition to lighting equipment specified for general lighting and is switched or dimmed on circuits different from the circuits for general lighting, the small of the actual wattage of the lighting equipment installed specifically for merchandise, or additional lighting power as determined below shall be added to the interior lighting power determined in accordance with this line item.

**TABLE C405.5.2(2)**
**INTERIOR LIGHTING POWER ALLOWANCES: SPACE-BY-SPACE METHOD**

| COMMON SPACE-BY-SPACE TYPES | LPD (w/ft²) |
|---|---|
| Atrium – First 40 feet in height | 0.03 per ft. ht. |
| Atrium – Above 40 feet in height | 0.02 per ft. ht. |
| Audience/seating area – permanent | |
| For auditorium | 0.9 |
| For performing arts theater | 2.6 |
| For motion picture theater | 1.2 |
| Classroom/lecture/training | 1.30 |
| Conference/meeting/multipurpose | 1.2 |
| Corridor/transition | 0.7 |
| Dining area | |
| Bar/lounge/leisure dining | 1.40 |
| Family dining area | 1.40 |
| Dressing/fitting room performing arts theater | 1.1 |
| Electrical/mechanical | 1.10 |
| Food preparation | 1.20 |
| Laboratory for classrooms | 1.3 |
| Laboratory for medical/industrial/research | 1.8 |
| Lobby | 1.10 |
| Lobby for performing arts theater | 3.3 |
| Lobby for motion picture theater | 1.0 |
| Locker room | 0.80 |
| Lounge recreation | 0.8 |
| Office – enclosed | 1.1 |
| Office – open plan | 1.0 |
| Restroom | 1.0 |
| Sales area | 1.6[a] |
| Stairway | 0.70 |
| Storage | 0.8 |
| Workshop | 1.60 |
| Courthouse/police station/penetentiary | |
| Courtroom | 1.90 |
| Confinement cells | 1.1 |
| Judge chambers | 1.30 |
| Penitentiary audience seating | 0.5 |
| Penitentiary classroom | 1.3 |
| Penitentiary dining | 1.1 |
| **BUILDING SPECIFIC SPACE-BY-SPACE TYPES** | |
| Automotive – service/repair | 0.70 |
| Bank/office – banking activity area | 1.5 |
| Dormitory living quarters | 1.10 |
| Gymnasium/fitness center | |
| Fitness area | 0.9 |
| Gymnasium audience/seating | 0.40 |
| Playing area | 1.40 |

*(continued)*

**TABLE C405.5.2(2)—continued**
**INTERIOR LIGHTING POWER ALLOWANCES: SPACE-BY-SPACE METHOD**

| COMMON SPACE-BY-SPACE TYPES | LPD (w/ft²) |
|---|---|
| Healthcare clinic/hospital | |
| Corridors/transition | 1.00 |
| Exam/treatment | 1.70 |
| Emergency | 2.70 |
| Public and staff lounge | 0.80 |
| Medical supplies | 1.40 |
| Nursery | 0.9 |
| Nurse station | 1.00 |
| Physical therapy | 0.90 |
| Patient room | 0.70 |
| Pharmacy | 1.20 |
| Radiology/imaging | 1.3 |
| Operating room | 2.20 |
| Recovery | 1.2 |
| Lounge/recreation | 0.8 |
| Laundry – washing | 0.60 |
| Hotel | |
| Dining area | 1.30 |
| Guest rooms | 1.10 |
| Hotel lobby | 2.10 |
| Highway lodging dining | 1.20 |
| Highway lodging guest rooms | 1.10 |
| Library | |
| Stacks | 1.70 |
| Card file and cataloguing | 1.10 |
| Reading area | 1.20 |
| Manufacturing | |
| Corridors/transition | 0.40 |
| Detailed manufacturing | 1.3 |
| Equipment room | 1.0 |
| Extra high bay (> 50-foot floor-ceiling height) | 1.1 |
| High bay (25- – 50-foot floor-ceiling height) | 1.20 |
| Low bay (< 25-foot floor-ceiling height) | 1.2 |
| Museum | |
| General exhibition | 1.00 |
| Restoration | 1.70 |
| Parking garage – garage areas | 0.2 |
| Convention center | |
| Exhibit space | 1.50 |
| Audience/seating area | 0.90 |
| Fire stations | |
| Engine room | 0.80 |
| Sleeping quarters | 0.30 |
| Post office | 0.9 |
| Sorting area | |
| Religious building | |
| Fellowship hall | 0.60 |
| Audience seating | 2.40 |
| Worship pulpit/choir | 2.40 |
| Retail | |
| Dressing/fitting area | 0.9 |
| Mall concourse | 1.6 |
| Sales area | 1.6[a] |

*(continued)*

**TABLE C405.5.2(2)—continued**
**INTERIOR LIGHTING POWER ALLOWANCES: SPACE-BY-SPACE METHOD**

| BUILDING SPECIFIC SPACE-BY-SPACE TYPES | LPD (w/ft²) |
|---|---|
| Sports arena | |
| Audience seating | 0.4 |
| Court sports area – Class 4 | 0.7 |
| Court sports area – Class 3 | 1.2 |
| Court sports area – Class 2 | 1.9 |
| Court sports area – Class 1 | 3.0 |
| Ring sports area | 2.7 |
| Transportation | |
| Air/train/bus baggage area | 1.00 |
| Airport concourse | 0.60 |
| Terminal – ticket counter | 1.50 |
| Warehouse | |
| Fine material storage | 1.40 |
| Medium/bulky material | 0.60 |

The exterior lighting design will be based on the building location and the applicable "Lighting Zone" as defined in IECC 2012 Table C405.6.2(1) which follows.

**TABLE C405.6.2(1)**
**EXTERIOR LIGHTING ZONES**

| LIGHTING ZONE | DESCRIPTION |
|---|---|
| 1 | Developed areas of national parks, state parks, forest land, and rural areas |
| 2 | Areas predominantly consisting of residential zoning, neighborhood business districts, light industrial with limited nighttime use and residential mixed use areas |
| 3 | All other areas |
| 4 | High-activity commercial districts in major metropolitan areas as designated by the local land use planning authority |

The lighting power density savings will be based on reductions below the allowable design levels as specified in IECC 2012 Table 405.6.2(2) which follows.

**TABLE C405.6.2(2)**
**INDIVIDUAL LIGHTING POWER ALLOWANCES FOR BUILDING EXTERIORS**

| | | LIGHTING ZONES | | | |
|---|---|---|---|---|---|
| | | Zone 1 | Zone 2 | Zone 3 | Zone 4 |
| Base Site Allowance (Base allowance is usable in tradable or nontradable surfaces.) | | 500 W | 600 W | 750 W | 1300 W |
| Tradable Surfaces (Lighting power densities for uncovered parking areas, building grounds, building entrances and exits, canopies and overhangs and outdoor sales areas are tradable.) | **Uncovered Parking Areas** | | | | |
| | Parking areas and drives | 0.04 W/ft$^2$ | 0.06 W/ft$^2$ | 0.10 W/ft$^2$ | 0.13 W/ft$^2$ |
| | Building Grounds | | | | |
| | Walkways less than 10 feet wide | 0.7 W/linear foot | 0.7 W/linear foot | 0.8 W/linear foot | 1.0 W/linear foot |
| | Walkways 10 feet wide or greater, plaza areas special feature areas | 0.14 W/ft$^2$ | 0.14 W/ft$^2$ | 0.16 W/ft$^2$ | 0.2 W/ft$^2$ |
| | Stairways | 0.75 W/ft$^2$ | 1.0 W/ft$^2$ | 1.0 W/ft$^2$ | 1.0 W/ft$^2$ |
| | Pedestrian tunnels | 0.15 W/ft$^2$ | 0.15 W/ft$^2$ | 0.2 W/ft$^2$ | 0.3 W/ft$^2$ |
| | **Building Entrances and Exits** | | | | |
| | Main entries | 20 W/linear foot of door width | 20 W/linear foot of door width | 30 W/linear foot of door width | 30 W/linear foot of door width |
| | Other doors | 20 W/linear foot of door width | 20 W/linear foot of door width | 20 W/linear foot of door width | 20 W/linear foot of door width |
| | Entry canopies | 0.25 W/ft$^2$ | 0.25 W/ft$^2$ | 0.4 W/ft$^2$ | 0.4 W/ft$^2$ |
| | **Sales Canopies** | | | | |
| | Free-standing and attached | 0.6 W/ft$^2$ | 0.6 W/ft$^2$ | 0.8 W/ft$^2$ | 1.0 W/ft$^2$ |
| | **Outdoor Sales** | | | | |
| | Open areas (including vehicle sales lots) | 0.25 W/ft$^2$ | 0.25 W/ft$^2$ | 0.5 W/ft$^2$ | 0.7 W/ft$^2$ |
| | Street frontage for vehicle sales lots in addition to "open area" allowance | No allowance | 10 W/linear foot | 10 W/linear foot | 30 W/linear foot |
| Nontradable Surfaces (Lighting power density calculations for the following applications can be used only for the specific application and cannot be traded between surfaces or with other exterior lighting. The following allowances are in addition to any allowance otherwise permitted in the "Tradable Surfaces" section of this table.) | Building facades | No allowance | 0.1 W/ft$^2$ for each illuminated wall or surface or 2.5 W/linear foot for each illuminated wall or surface length | 0.15 W/ft$^2$ for each illuminated wall or surface or 3.75 W/linear foot for each illuminated wall or surface length | 0.2 W/ft$^2$ for each illuminated wall or surface or 5.0 W/linear foot for each illuminated wall or surface length |
| | Automated teller machines and night depositories | 270 W per location plus 90 W per additional ATM per location | 270 W per location plus 90 W per additional ATM per location | 270 W per location plus 90 W per additional ATM per location | 270 W per location plus 90 W per additional ATM per location |
| | Entrances and gatehouse inspection stations at guarded facilities | 0.75 W/ft$^2$ of covered and uncovered area | 0.75 W/ft$^2$ of covered and uncovered area | 0.75 W/ft$^2$ of covered and uncovered area | 0.75 W/ft$^2$ of covered and uncovered area |
| | Loading areas for law enforcement, fire, ambulance and other emergency service vehicles | 0.5 W/ft$^2$ of covered and uncovered area | 0.5 W/ft$^2$ of covered and uncovered area | 0.5 W/ft$^2$ of covered and uncovered area | 0.5 W/ft$^2$ of covered and uncovered area |
| | Drive-up windows/doors | 400 W per drive-through | 400 W per drive-through | 400 W per drive-through | 400 W per drive-through |
| | Parking near 24-hour retail entrances | 800 W per main entry | 800 W per main entry | 800 W per main entry | 800 W per main entry |

For SI: 1 foot = 304.8 mm, 1 watt per square foot = W/0.0929 m$^2$.

**MEASURE CODE: CI-LTG-LPDE-V02-1406**

## 4.5.8 Miscellaneous Commercial/Industrial Lighting

### DESCRIPTION

This measure is designed to calculate savings from energy efficient lighting upgrades that are not captured in other measures within the TRM. If a lighting project fits the measure description in sections 4.5.1-4.5.4, then those criteria, definitions, and calculations should be used.

Unlike other lighting measures this one applies only to RF applications (because there is no defined baseline for TOS or NC applications).

### DEFINITION OF EFFICIENT EQUIPMENT

A lighting fixture that replaces an existing fixture to provide the same or greater lumen output at a lower kW consumption.

### DEFINITION OF BASELINE EQUIPMENT

The definition of baseline equipment is the existing lighting fixture.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The deemed lifetime of the efficient equipment fixture, regardless of program type is 15 years[550].

### DEEMED MEASURE COST

The actual cost of the efficient light fixture should be used.

### LOADSHAPE

Loadshape C06 - Commercial Indoor Lighting

Loadshape C07 - Grocery/Conv. Store Indoor Lighting

Loadshape C08 - Hospital Indoor Lighting

Loadshape C09 - Office Indoor Lighting

Loadshape C10 - Restaurant Indoor Lighting

Loadshape C11 - Retail Indoor Lighting

Loadshape C12 - Warehouse Indoor Lighting

Loadshape C13 - K-12 School Indoor Lighting

Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)

Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)

Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)

Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)

Loadshape C18 - Industrial Indoor Lighting

[550] 15 years from GDS Measure Life Report, June 2007

Loadshape C19 - Industrial Outdoor Lighting

Loadshape C20 - Commercial Outdoor Lighting

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the location type. Values are provided for each building type in section 4.5.

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = ((Watts_{base}-Watts_{EE})/1000) * Hours * WHF_e * ISR$$

Where:

$Watts_{base}$ = Input wattage of the existing system which depends on the baseline fixture configuration (number and type of lamp) and ballast factor (if applicable) and number of fixtures.

=Actual

$Watts_{EE}$ = New Input wattage of EE fixture which depends on new fixture configuration (number of lamps) and ballast factor (if applicable) (if applicable) and number of fixtures.

= Actual

Hours = Average hours of use per year as provided by the customer or selected from the Reference Table in Section 4.5, Fixture annual operating hours, by building type. If hours or building type are unknown, use the Miscellaneous value.

$WHF_e$ = Waste heat factor for energy to account for cooling energy savings from efficient lighting is selected from the Reference Table in Section 4.5 for each building type. If building is un-cooled, the value is 1.0.

ISR = In Service Rate or the percentage of units rebated that get installed.

=100%[551] if application form completed with sign off that equipment is not placed into storage. If sign off form not completed assume the following 3 year ISR assumptions:

[551] Illinois evaluation of PY1 through PY3 has not found that fixtures or lamps placed into storage to be a significant enough issue to warrant including an "In-Service Rate" when commercial customers complete an application form.

| Weigted Average 1st year In Service Rate (ISR) | 2nd year Installations | 3rd year Installations | Final Lifetime In Service Rate |
|---|---|---|---|
| 75.5%[552] | 12.1% | 10.3% | 98.0%[553] |

**HEATING PENALTY**

If electrically heated building:

$$\Delta kWh_{heatpenalty}^{554} = (((WattsBase-WattsEE)/1000) * ISR * Hours * -IFkWh$$

Where:

IFkWh = Lighting-HVAC Interation Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficent lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

**DEFERRED INSTALLS**

As presented above, if a sign off form is not completed the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

Year 1 (Purchase Year) installs: Characterized using assumptions provided above or evaluated assumptions if available.

Year 2 and 3 installs: Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.

The NTG factor for the Purchase Year should be applied.

**SUMMER COINCIDENT DEMAND SAVINGS**

$$\Delta kW = ((Watts_{base}-Watts_{EE})/1000) * WHF_d * CF * ISR$$

Where:

WHFd = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is selected from the Reference Table in Section 4.5 for each building type. If the building is not cooled WHFd is 1.

[552] 1st year in service rate is based upon review of PY4-5 evaluations from ComEd's commercial lighting program (BILD) (see 'IL Commercial Lighting ISR.xls' for more information. The average first year ISR was calculated weighted by the number of bulbs sold.

[553] The 98% Lifetime ISR assumption is based upon review of two evaluations:
'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings.

[554] Negative value because this is an increase in heating consumption due to the efficient lighting.

CF = Summer Peak Coincidence Factor for measure is selected from the Reference able in Section 4.5 for each building type. If the building type is unknown, use the Miscellaneous value of 0.66.

Other factors as defined above

**NATURAL GAS ENERGY SAVINGS**

$\Delta Therms$[555] = (((WattsBase-WattsEE)/1000) * ISR * Hours * - IFTherms

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. This value is selected from the Reference Table in Section 6.5 for each building type.

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

If there are differences between the maintenance of the efficient and baseline lighting system then they should be evaluated on a project-by-project basis.

**MEASURE CODE: CI-LTG-MSCI-V02-140601**

[555]Negative value because this is an increase in heating consumption due to the efficient lighting.

## 4.5.9 Multi-Level Lighting Switch

### DESCRIPTION

This measure relates to the installation new multi-level lighting switches on an existing lighting system.

This measure can only relate to the adding of a new control in an existing building, since multi-level switching is required in the Commercial new construction building energy code (IECC 2012).

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient system is assumed to be a lighting system controlled by multi-level lighting controls.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be an uncontrolled lighting system where all lights in a given area are on the same circuit or all circuits come on at the same time.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life for all lighting controls is assumed to be 8 years[556].

### DEEMED MEASURE COST

When available, the actual cost of the measure shall be used. When not available, the incremental capital cost for this measure is assumed to be $274[557].

### LOADSHAPE

Loadshape C06 - Commercial Indoor Lighting

Loadshape C07 - Grocery/Conv. Store Indoor Lighting

Loadshape C08 - Hospital Indoor Lighting

Loadshape C09 - Office Indoor Lighting

Loadshape C10 - Restaurant Indoor Lighting

Loadshape C11 - Retail Indoor Lighting

Loadshape C12 - Warehouse Indoor Lighting

Loadshape C13 - K-12 School Indoor Lighting

Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)

Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)

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[556] Consistent with Occupancy Sensor control measure.

[557] Goldberg et al, State of Wisconsin Public Service Commission of Wisconsin, Focus on Energy Evaluation, Business Programs: Incremental Cost Study, KEMA, October 28, 2009.

Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)

Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)

Loadshape C18 - Industrial Indoor Lighting

Loadshape C19 - Industrial Outdoor Lighting

Loadshape C20 - Commercial Outdoor Lighting

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the location type. Values are provided for each building type in the reference section below.

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = KW_{Controlled} * Hours * ESF * WHF_e$$

Where:

$KW_{Controlled}$ = Total lighting load connected to the control in kilowatts.

= Actual

Hours = total operating hours of the controlled lighting circuit before the lighting controls are installed. This number should be collected from the customer. Average hours of use per year are provided in the Reference Table in Section 4.5, Fixture annual operating hours, for each building type if customer specific information is not collected. If unknown buidling type, use the Miscellaneous value.

ESF = Energy Savings factor (represents the percentage reduction to the KWcontrolled due to the use of multi-level switching).

= Dependent on building type[558]:

| Building Type | Energy Savings Factor (ESF) |
|---|---|
| Private Office | 21.6% |
| Open Office | 16.0% |
| Retail | 14.8% |
| Classrooms | 8.3% |

[558] Based on results from "Lighting Controls Effectiveness Assessment: Final Report on Bi-Level Lighting Study" published by the California Public Utilities Commission (CPUC), prepared by ADM Associates. http://lightingcontrolsassociation.org/bi-level-switching-study-demonstrates-energy-savings/

| Building Type | Energy Savings Factor (ESF) |
|---|---|
| Unknown, average | 15% |

$WHF_e$ = Waste heat factor for energy to account for cooling energy savings from efficient lighting is provided in the Reference Table in Section 4.5 for each building type. If building is un-cooled, the value is 1.0.

**HEATING PENALTY**

If electrically heated building:

$$\Delta kWh_{heatpenalty}^{559} = KW_{Controlled} * Hours * ESF * -IFkWh$$

Where:

IFkWh = Lighting-HVAC Interation Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficent lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$$\Delta kW = KW_{controlled} * ESF * WHF_d * CF$$

Where:

$WHF_d$ = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is provided in the Reference Table in Section 4.5. If the building is un-cooled WHFd is 1.

CF = Summer Peak Coincidence Factor for measure is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value of 0.66[560].

**NATURAL GAS ENERGY SAVINGS**

$$\Delta therms = KW_{Controlled} * Hours * ESF * - IFTherms$$

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting and provided in the Reference Table in Section 4.5 by buidling type.

[559] Negative value because this is an increase in heating consumption due to the efficient lighting.

[560] By applying the ESF and the same coincidence factor for general lighting savings we are in essence assuming that the savings from multi-level switching are as likely during peak periods as any other time. In the absence of better information this seems like a reasonable assumption and if anything may be on the conservative side since you might expect the peak periods to be generally sunnier and therefore more likely to have lower light levels. It is also consistent with the control type reducing the wattage lighting load, the same as the general lighting measures.

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-LTG-MLLC-V02-140601

## 4.5.10 Occupancy Sensor Lighting Controls

### DESCRIPTION

This measure relates to the installation of new occupancy sensors on a new or existing lighting system. Lighting control types covered by this measure include wall, ceiling or fixture mounted occupancy sensors. Passive infrared, ultrasonic detectors and fixture-mounted sensors or sensors with a combination thereof are eligible. Lighting controls required by state energy codes are not eligible. This must be a new installation and may not replace an existing lighting occupancy sensor control.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the existing system is assumed to be manually controlled or an uncontrolled lighting system which is being controlled by one of the lighting controls systems listed above. All sensors must be hard wired and control interior lighting.

A subset of occupancy sensors are those that are programmed as "vacancy" sensors. To qualify as a vacancy sensor, the control must be configured such that manual input is required to turn on the controlled lighting and the control automatically turns the lighting off. Additional savings are achieved compared to standard occupancy sensors because lighting does not automatically turn on and occupants may decide to not turn it on. Note that vacancy sensors are not a viable option for many applications where standard occupancy sensors should be used instead.

### DEFINITION OF BASELINE EQUIPMENT

The baseline is assumed to be a lighting system uncontrolled by occupancy.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life for all lighting controls is assumed to be 8 years[561].

### DEEMED MEASURE COST

When available, the actual cost of the measure shall be used. When not available, the following default values are provided:

| Lighting control type | Cost |
|---|---|
| Full cost of wall mounted occupancy sensor | $42[562] |
| Full cost mounted occupancy sensor | $66[563] |
| Full cost of fixture-mounted occupancy sensor | $125[564] |

[561] DEER 2008
[562] Goldberg et al, State of Wisconsin, Public Service Commission of Wisconsin, Focus on Energy Evaluation Business programs Incremental Cost Study, KEMA, October 28, 2009
[563] Ibid
[564] Efficiency Vermont TRM, October 26, 2011.

### LOADSHAPE

Loadshape C06 - Commercial Indoor Lighting

Loadshape C07 - Grocery/Conv. Store Indoor Lighting

Loadshape C08 - Hospital Indoor Lighting

Loadshape C09 - Office Indoor Lighting

Loadshape C10 - Restaurant Indoor Lighting

Loadshape C11 - Retail Indoor Lighting

Loadshape C12 - Warehouse Indoor Lighting

Loadshape C13 - K-12 School Indoor Lighting

Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)

Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)

Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)

Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)

Loadshape C18 - Industrial Indoor Lighting

Loadshape C19 - Industrial Outdoor Lighting

Loadshape C20 - Commercial Outdoor Lighting

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on location.

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = KW_{Controlled} * Hours * ESF * WHF_e$$

Where:

$Kw_{Controlled}$ = Total lighting load connected to the control in kilowatts. Savings is per control. The total connected load per control should be collected from the customer or the default values presented below used;

| Lighting Control Type | Default kw controlled |
|---|---|
| Wall mounted occupancy sensor | 0.350[565] |

[565] Goldberg et al, State of Wisconsin Public Service Commission of Wisconsin, Focus on Energy Evaluation, Business Programs, Incremental Cost Study, KEMA, October 28, 2009

| Lighting Control Type | Default kw controlled |
|---|---|
| Remote mounted occupancy sensor | 0.587[566] |
| Fixture mounted sensor | 0.073[567] |

Hours = total operating hours of the controlled lighting circuit before the lighting controls are installed. This number should be collected from the customer. Average hours of use per year are provided in the Reference Table in Section 4.5, Fixture annual operating hours, for each building type if customer specific information is not collected. If unknown buidling type, use the Miscellaneous value.

ESF = Energy Savings factor (represents the percentage reduction to the operating Hours from the non-controlled baseline lighting system).

| Lighting Control Type | Energy Savings Factor[568] |
|---|---|
| Wall or Ceiling-Mounted Occupancy Sensors | 41% or custom |
| Fixture Mounted Occupancy Sensors | 30% or custom |
| Wall-Mounted Occupancy Sensors Configured as "Vacancy Sensors" | 53% or custom[569] |

$WHF_e$ = Waste heat factor for energy to account for cooling energy savings from efficient lighting is provided in the Reference Table in Section 4.5 for each building type. If building is un-cooled, the value is 1.0.

**HEATING PENALTY**

If electrically heated building:

$$\Delta kWh_{heatpenalty}^{570} = KW_{Controlled} * Hours * ESF * -IFkWh$$

Where:

IFkWh = Lighting-HVAC Interation Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficent lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$$\Delta kW = KW_{controlled} * WHF_d * (CFbaseline - CFos)$$

[566] Ibid
[567] Efficiency Vermont TRM 2/19/2010
[568] Kuiken, Tammy eta al, State of Wisconsin/Public Service Commission of Wisconsin, Focus on Energy Evaluation, Business Programs, Deemed Savings Manual V1.0, PA Consulting Group and KEMA, March 22, 2010 pp 4-192-194.
[569] Papamichael, Konstantions, Bi-Level Switching in Office Spaces, California Lighting Technology Center, February 1,2010. Note: See Figure 8 on page 10 for relevant study results. The study shows a 30% extra savings above a typical occupancy sensor; 41% * 1.3 = 53%..
[570] Negative value because this is an increase in heating consumption due to the efficient lighting.

Where:

$WHF_d$ = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is provided in the Reference Table in Section 4.5. If the building is un-cooled WHFd is 1.

CFbaseline = Baseline Summer Peak Coincidence Factor for the lighting system without Occupancy Sensors installed selected from the Reference Table in Section 4.5 for each building type. If the building type is unknown, use the Miscellaneous value of 0.66

CFos = Retrofit Summer Peak Coincidence Factor the lighting system with Occupancy Sensors installed is 0.15 regardless of building type.[571]

**NATURAL GAS ENERGY SAVINGS**

$$\Delta therms = KW_{Controlled} * Hours * ESF * - IFTherms$$

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting and provided in the Reference Table in Section 4.5 by buidling type.

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-LTG-OSLC-V02-140601**

[571] Coincidence Factor Study Residential and Commercial Industrial Lighting Measures, RLW Analytics, Spring 2007. Note, the connected load used in the calculation of the CF for occupancy sensor lights includes the average ESF.

## 4.5.11 Solar Light Tubes

### DESCRIPTION

A tubular skylight which is 10" to 21" in diameter with a prismatic or translucent lens is installed on the roof of a commercial facility. The lens reflects light captured from the roof opening through a highly specular reflective tube down to the mounted fixture height. When in use, a light tube fixture resembles a metal halide fixture. Uses include grocery, school, retail and other single story commercial buildings.

In order that the savings characterized below apply, the electric illumination in the space must be automatically controlled to turn off or down when the tube is providing enough light.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is assumed to be a tubular skylight that concentrates and directs light from the roof to an area inside the facility.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment for this measure is a fixture with comparable luminosity. The specifications for the baseline lamp depend on the size of the Light Tube being installed.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The estimated useful life for a light tube commercial skylight is 10 years[572].

### DEEMED MEASURE COST

If available, the actual incremental cost should be used. For analysis purposes, assume an incremental cost for a light tube commercial skylight is $500[2].

### LOADSHAPE

Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)[573]

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on location.

[572] Equal to the manufacturers standard warranty

[573] The savings from solar light tubes are only realized during the sunlight hours. It is therefore appropriate to apply the single shift (8/5) loadshape to this measure.

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = kW_f * HOURS * WHFe$$

Where:

$kW_f$ = Connected load of the fixture the solar tube replaces

| Size of Tube | Average Lumen output for Chicago Illinois (minimum)[574] | Equivalent fixture | kW |
|---|---|---|---|
| 21" | 9,775 (4,179) | 50% 3 x 2 32W lamp CFL (207W, 9915 lumens)<br>50% 4 lamp F32 w/Elec 4' T8 (114W, 8895 lumens) | 0.161 |
| 14" | 4,392 (1,887) | 50% 2 42W lamp CFL (94W, 4406 lumens)<br>50% 2 lamp F32 w/Elec 4' T8 (59W, 4448 lumens) | 0.077 |
| 10" | 2,157 (911) | 50% 1 42W lamp CFL (46W, 2203 lumens)<br>50% 1 lamp F32 w/Elec 4' T8 (32W, 2224 lumens) | 0.039 |
| | | **AVERAGE** | **0.092** |

HOURS = Equivalent full load hours

= 2400 [575]

$WHF_e$ = Waste heat factor for energy to account for cooling energy savings from efficient lighting is selected from the Reference Table in Section 4.5 for each building type. If building is un-cooled, the value is 1.0.

### HEATING PENALTY

If electrically heated building:

$$\Delta kWh_{heatpenalty}^{576} = kW_f * HOURS * -IFkWh$$

Where:

IFkWh = Lighting-HVAC Interation Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficent lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

[574] Solatube Test Report (2005). http://www.mainegreenbuilding.com/files/file/solatube/stb_lumens_datasheet.pdf
[575] Ibid. The lumen values presented in the kW table represent the average of the lightest 2400 hours.
[576] Negative value because this is an increase in heating consumption due to the efficient lighting.

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$$\Delta kW = \Delta kW_f * WHFd * CF$$

Where:

WHFd = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is selected from the Reference Table in Section 4.5 for each building type. If the building is not cooled WHFd is 1.

CF = Summer Peak Coincidence Factor for measure is selected from the Reference Table in Section 4.5 for each building type. If the building type is unknown, use the Miscellaneous value of 0.66.

**NATURAL GAS SAVINGS**

$$\Delta Therms^{577} = \Delta kW_f * HOURS * - IFTherms$$

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Please select from the Reference Table in Section 4.5 for each building type.

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-LTG-STUB-V02-140601**

[577] Negative value because this is an increase in heating consumption due to the efficient lighting.

## 4.5.12 T5 Fixtures and Lamps

### DESCRIPTION

T5 Lamp/ballast systems have higher lumens per watt than a standard T8 or an existing T8 or T12 system. The smaller lamp diameter allows for better optical systems, and more precise control of lighting. These characteristics result in light fixtures that produce equal or greater light than standard T8 or T12 fixtures, while using fewer watts.

This measure applies to the installation of new equipment with efficiencies that exceed that of the equipment that would have been installed following standard market practices and is applicable to time of sale as well as retrofit measures.

If the implementation strategy does not allow for the installation location to be known, a deemed split of 99% Commercial and 1% Residential should be used[578].

This measure was developed to be applicable to the following program types: TOS, RF, DI.

If applied to other program types, the measure savings should be verified.

The measure applies to all commercial T5 installations excluding new construction and substantial renovation or change of use measures (see lighting power density measure). Lookup tables have been provided to account for various installations. Actual existing equipment wattages should be compared to new fixture wattages whenever possible while maintaining lumen equivalent designs. Default new and baseline assumptions are provided if existing equipment cannot be determined. Actual costs and hours of use should be utilized when available. Default component costs and lifetimes have been provided for Operating and Maintenance Calculations. Please see the Definition Table to determine applicability for each program. Configurations not included in the TRM may be included in custom program design using the provided algorithms as long as energy savings is achieved. The following table defines the applicability for different programs:

| Time of Sale (TOS) | Retrofit (RF) and DI |
|---|---|
| This program applies to installations where customer and location of equipment is not known, or at time of burnout of existing equipment. T5 Lamp/ballast systems have higher lumens per watt than a standard T8 system. The smaller lamp diameter allows for better optical systems, and more precise control of lighting. These characteristics result in light fixtures that produce equal or greater light than standard T8 fixtures, while using fewer watts. | For installations that upgrade installations before the end of their useful life. T5 Lamp/ballast systems have higher lumens per watt than a standard T8 or T12 system. The smaller lamp diameter allows for better optical systems, and more precise control of lighting. These characteristics result in light fixtures that produce equal or greater light than standard T8 or T12 fixtures, while using fewer watts and having longer life. |

### DEFINITION OF EFFICIENT EQUIPMENT

The definition of efficient equipment varies based on the program and is defined below:

[578] Based on weighted average of Final ComEd's BILD program data from PY5 and PY6. For Residential installations, hours of use assumptions from '5.5 Interior Hardwired Compact Fluorescent Lamp (CFL) Fixture' measure should be used.

| Time of Sale (TOS) | Retrofit (RF) and DI |
|---|---|
| 4' fixtures must use a T5 lamp and ballast configuration. 1' and 3' lamps are not eligible. High Performance Troffers must be 85% efficient or greater. T5 HO high bay fixtures must be 3, 4 or 6 lamps and 90% efficient or better. | 4' fixtures must use a T5 lamp and ballast configuration. 1' and 3' lamps are not eligible. High Performance Troffers must be 85% efficient or greater. T5 HO high bay fixtures must be 3, 4 or 6 lamps and 90% efficient or better. |

**DEFINITION OF BASELINE EQUIPMENT**

The definition of baseline equipment varies based on the program and is defined below:

| Time of Sale (TOS) | Retrofit (RF) and DI |
|---|---|
| The baseline is T8 with equivalent lumen output. In high-bay applications, the baseline is pulse start metal halide systems. | The baseline is the existing system.<br><br>In July 14, 2012, Federal Standards were enacted that were expected to eliminate T-12s as an option for linear fluorescent fixtures. Through v3.0 of the TRM, it was assumed that the T-12 would no longer be baseline for retrofits from 1/1/2016. However, due to significant loopholes in the legislation, T-12 compliant product is still freely available and in Illinois T-12s continue to hold a significant share of the existing and replacement lamp market. Therefore the timing of the sunsetting of T-12s as a viable baseline has been pushed back in v4.0 until 6/1/2016 and will be revisited in future update sessions.<br><br>There will be a baseline shift applied to all measures installed before 2016 in 2016 in years remaining in the measure life. See table C-1. |

**DEEMED LIFETIME OF EFFICIENT EQUIPMENT**

The deemed lifetime of the efficient equipment fixture, regardless of program type is Fixture lifetime is 15 years[579].

**LOADSHAPE**

Loadshape C06 - Commercial Indoor Lighting

Loadshape C07 - Grocery/Conv. Store Indoor Lighting

Loadshape C08 - Hospital Indoor Lighting

Loadshape C09 - Office Indoor Lighting

Loadshape C10 - Restaurant Indoor Lighting

Loadshape C11 - Retail Indoor Lighting

[579] 15 years from GDS Measure Life Report, June 2007

Loadshape C12 - Warehouse Indoor Lighting

Loadshape C13 - K-12 School Indoor Lighting

Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)

Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)

Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)

Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)

Loadshape C18 - Industrial Indoor Lighting

Loadshape C19 - Industrial Outdoor Lighting

Loadshape C20 - Commercial Outdoor Lighting

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = ((Watts_{base} - Watts_{EE})/1000) * Hours * WHF_e * ISR$$

Where:

$Watts_{base}$ = Input wattage of the existing system which depends on the baseline fixture configuration (number and type of lamp) and number of fixtures. Value can be selected from the appropriate reference table as shown below, of a custom value can be entered if the configurations in the tables is not representative of the exisitng system.

$Watts_{EE}$ = New Input wattage of EE fixture which depends on new fixture configuration (number of lamps) and ballast factor and number of fixtures. Value can be selected from the appropriate reference table as shown below, of a custom value can be entered if the configurations in the tables is not representative of the exisitng system.

| Program | Reference Table |
|---|---|
| Time of Sale | A-1: T5 New and Baseline Assumptions |
| Retrofit, DI | A-2: T5 New and Baseline Assumptions |

Hours = Average hours of use per year as provided by the customer or selected from the Reference Table in Section 4.5, Fixture annual operating hours, by building type. If hours or building type are unknown, use the Miscellaneous value.

$WHF_e$ = Waste heat factor for energy to account for cooling energy savings from efficient lighting is selected from the Reference Table in Section 4.5 for each building type. If building is un-cooled, the value is 1.0.

ISR = In Service Rate or the percentage of units rebated that get installed.

=100%[580] if application form completed with sign off that equipment is not placed into storage. If sign off form not completed assume the following 3 year ISR assumptions:

| Weigted Average 1st year In Service Rate (ISR) | 2nd year Installations | 3rd year Installations | Final Lifetime In Service Rate |
|---|---|---|---|
| 98%[581] | 0% | 0% | 98.0%[582] |

**HEATING PENALTY**

If electrically heated building:

$$\Delta kWh_{heatpenalty}^{583} = (((WattsBase-WattsEE)/1000) * ISR * Hours * -IFkWh$$

Where:

IFkWh = Lighting-HVAC Interation Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficent lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

**SUMMER COINCIDENT DEMAND SAVINGS**

$$\Delta kW = ((Watts_{base}-Watts_{EE})/1000) * WHF_d*CF*ISR$$

Where:

WHFd = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is selected from the Reference Table in Section 4.5 for each building type. If the building is not cooled WHFd is 1.

CF = Summer Peak Coincidence Factor for measure is selected from the Reference Table in Section 4.5 for each building type. If the building type is unknown, use the Miscellaneous value.

[580] Illinois evaluation of PY1 through PY3 has not found that fixtures or lamps placed into storage to be a significant enough issue to warrant including an "In-Service Rate" when commercial customers complete an application form.

[581] 1st year in service rate is based upon review of PY5-6 evaluations from ComEd's commercial lighting program (BILD) (see 'IL Commercial Lighting ISR_2014.xls' for more information

[582] The 98% Lifetime ISR assumption is based upon review of two evaluations:
'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings.

[583] Negative value because this is an increase in heating consumption due to the efficient lighting.

### NATURAL GAS ENERGY SAVINGS

$$\Delta Therms^{584} = (((WattsBase-WattsEE)/1000) * ISR * Hours *- IFTherms$$

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. This value is selected from the Reference Table in Section 4.5 for each building type.

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

See Reference tables for Operating and Maintenance Values

| Program | Reference Table |
|---|---|
| Time of Sale | B-1: T5 Component Costs and Lifetime |
| Retrofit, DI | B-2: T5 Component Costs and Lifetime |

### REFERENCE TABLES

See following page

[584] Negative value because this is an increase in heating consumption due to the efficient lighting.

A-1: Time of Sale: T5 New and Baseline Assumptions[585]

| EE Measure Description | EE Cost | $Watts_{EE}$ | Baseline Description | Base Cost | $Watts_{BASE}$ | Measure Cost | $Watts_{SAVE}$ |
|---|---|---|---|---|---|---|---|
| 2-Lamp T5 High-Bay | $200.00 | 180 | 200 Watt Pulse Start Metal-Halide | $100.00 | 232 | $100.00 | 52 |
| 3-Lamp T5 High-Bay | $200.00 | 180 | 200 Watt Pulse Start Metal-Halide | $100.00 | 232 | $100.00 | 52 |
| 4-Lamp T5 High-Bay | $225.00 | 240 | 320 Watt Pulse Start Metal-Halide | $125.00 | 350 | $100.00 | 110 |
| 6-Lamp T5 High-Bay | $250.00 | 360 | Proportionally Adjusted according to 6-Lamp HPT8 Equivalent to 320 PSMH | $150.00 | 476 | $100.00 | 116 |
| | | | | | | | |
| 1-Lamp T5 Troffer/Wrap | $100.00 | 32 | Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8 | $60.00 | 44 | $40.00 | 12 |
| 2-Lamp T5 Troffer/Wrap | $100.00 | 64 | 3-Lamp F32T8 Equivalent w/ Elec. Ballast | $60.00 | 88 | $40.00 | 24 |
| | | | | | | | |
| 1-Lamp T5 Industrial/Strip | $70.00 | 32 | Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8 | $40.00 | 44 | $30.00 | 12 |
| 2-Lamp T5 Industrial/Strip | $70.00 | 64 | 3-Lamp F32T8 Equivalent w/ Elec. Ballast | $40.00 | 88 | $30.00 | 24 |
| 3-Lamp T5 Industrial/Strip | $70.00 | 96 | Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8 | $40.00 | 132 | $30.00 | 36 |
| 4-Lamp T5 Industrial/Strip | $70.00 | 128 | Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8 | $40.00 | 178 | $30.00 | 50 |
| | | | | | | | |
| 1-Lamp T5 Indirect | $175.00 | 32 | Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8 | $145.00 | 44 | $30.00 | 12 |
| 2-Lamp T5 Indirect | $175.00 | 64 | 3-Lamp F32T8 Equivalent w/ Elec. Ballast | $145.00 | 88 | $30.00 | 24 |

[585] Adapted from Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, October 26, 2011.

A-2: Retrofit T5 New and Baseline Assumptions[586]

| EE Measure Description | EE Cost | $Watts_{EE}$ |
|---|---|---|
| 3-Lamp T5 High-Bay | $ 200 | 180 |
| 4-Lamp T5 High-Bay | $ 225 | 240 |
| 6-Lamp T5 High-Bay | $ 250 | 360 |
| | | |
| 1-Lamp T5 Troffer/Wrap | $ 100 | 32 |
| 2-Lamp T5 Troffer/Wrap | $ 100 | 64 |
| | | |
| 1-Lamp T5 Industrial/Strip | $ 70 | 32 |
| 2-Lamp T5 Industrial/Strip | $ 70 | 64 |
| 3-Lamp T5 Industrial/Strip | $ 70 | 96 |
| 4-Lamp T5 Industrial/Strip | $ 70 | 128 |
| | | |
| 1-Lamp T5 Indirect | $ 175 | 32 |
| 2-Lamp T5 Indirect | $ 175 | 64 |
| | | |

| Baseline Description | $Watts_{BASE}$ |
|---|---|
| 200 Watt Pulse Start Metal-Halide | 232 |
| 250 Watt Metal-Halide | 295 |
| 320 Watt Pulse Start Metal-Halide | 350 |
| 400 Watt Metal halide | 455 |
| 400 Watt Pulse Start Metal-halide | 476 |
| | |
| 1-Lamp F34T12 w/ EEMag Ballast | 40 |
| 2-Lamp F34T12 w/ EEMag Ballast | 68 |
| 3-Lamp F34T12 w/ EEMag Ballast | 110 |
| 4-Lamp F34T12 w/ EEMag Ballast | 139 |
| | |
| 1-Lamp F40T12 w/ EEMag Ballast | 48 |
| 2-Lamp F40T12 w/ EEMag Ballast | 82 |
| 3-Lamp F40T12 w/ EEMag Ballast | 122 |
| 4-Lamp F40T12 w/ EEMag Ballast | 164 |
| | |
| 1-Lamp F40T12 w/ Mag Ballast | 57 |
| 2-Lamp F40T12 w/ Mag Ballast | 94 |
| 3-Lamp F40T12 w/ Mag Ballast | 147 |
| 4-Lamp F40T12 w/ Mag Ballast | 182 |
| | |
| 1-Lamp F32 T8 | 32 |
| 2-Lamp F32 T8 | 59 |
| 3-Lamp F32 T18 | 88 |
| 4-Lamp F32 T8 | 114 |

[586] Ibid.

## B-1: Time of Sale T5 Component Costs and Lifetime[587]

| EE Measure Description | EE Lamp Cost | EE Lamp Life (hrs) | EE Lamp Rep. Labor Cost per lamp | EE Ballast Cost | EE Ballast Life (hrs) | EE Ballast Rep. Labor Cost | Baseline Description | # Base Lamps | Base Lamp Cost | Base Lamp Life (hrs) | Base Lamp Rep. Labor Cost | # Base Ballasts | Base Ballast Cost | Base Ballast Life (hrs) | Base Ballast Rep. Labor Cost |
|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|
| 3-Lamp T5 High-Bay | $12.00 | 20000 | $6.67 | $52.00 | 70000 | $22.50 | 200 Watt Pulse Start Metal-Halide | 1.00 | $21.00 | 10000 | $6.67 | 1.00 | $87.75 | 40000 | $22.50 |
| 4-Lamp T5 High-Bay | $12.00 | 20000 | $6.67 | $52.00 | 70000 | $22.50 | 320 Watt Pulse Start Metal-Halide | 1.00 | $21.00 | 20000 | $6.67 | 1.00 | $109.35 | 40000 | $22.50 |
| 6-Lamp T5 High-Bay | $12.00 | 20000 | $6.67 | $52.00 | 70000 | $22.50 | Adjusted according to 6-Lamp HPT8 Equivalent to 320 | 1.36 | $21.00 | 20000 | $6.67 | 1.50 | $109.35 | 40000 | $22.50 |
| | | | | | | | | | | | | | | | |
| 1-Lamp T5 Troffer/Wrap | $12.00 | 20000 | $2.67 | $52.00 | 70000 | $15.00 | Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8 | 1.50 | $2.50 | 20000 | $2.67 | 0.50 | $15.00 | 70000 | $15.00 |
| 2-Lamp T5 Troffer/Wrap | $12.00 | 20000 | $2.67 | $52.00 | 70000 | $15.00 | 3-Lamp F32T8 Equivalent w/ Elec. Ballast | 3.00 | $2.50 | 20000 | $2.67 | 1.00 | $15.00 | 70000 | $15.00 |
| | | | | | | | | | | | | | | | |
| 1-Lamp T5 Industrial/Strip | $12.00 | 20000 | $2.67 | $52.00 | 70000 | $15.00 | Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8 | 1.50 | $2.50 | 20000 | $2.67 | 0.50 | $15.00 | 70000 | $15.00 |
| 2-Lamp T5 Industrial/Strip | $12.00 | 20000 | $2.67 | $52.00 | 70000 | $15.00 | 3-Lamp F32T8 Equivalent w/ Elec. Ballast | 3.00 | $2.50 | 20000 | $2.67 | 1.00 | $15.00 | 70000 | $15.00 |
| 3-Lamp T5 Industrial/Strip | $12.00 | 20000 | $2.67 | $52.00 | 70000 | $15.00 | Proportionally adjusted according to 2-Lamp T5 Equivalent | 4.50 | $2.50 | 20000 | $2.67 | 1.50 | $15.00 | 70000 | $15.00 |
| 4-Lamp T5 Industrial/Strip | $12.00 | 20000 | $2.67 | $52.00 | 70000 | $15.00 | Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8 | 6.00 | $2.50 | 20000 | $2.67 | 2.00 | $15.00 | 70000 | $15.00 |
| | | | | | | | | | | | | | | | |
| 1-Lamp T5 Indirect | $12.00 | 20000 | $2.67 | $52.00 | 70000 | $15.00 | Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8 | 1.50 | $2.50 | 20000 | $2.67 | 0.50 | $15.00 | 70000 | $15.00 |
| 2-Lamp T5 Indirect | $12.00 | 20000 | $2.67 | $52.00 | 70000 | $15.00 | 3-Lamp F32T8 Equivalent w/ Elec. Ballast | 3.00 | $2.50 | 20000 | $2.67 | 1.00 | $15.00 | 70000 | $15.00 |

[587] Adapted from Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, October 26, 2011.

B-2: T5 Retrofit Component Costs and Lifetime[588]

| EE Measure Description | EE Lamp Cost | EE Lamp Life (hrs) | EE Lamp Rep. Labor Cost per lamp | EE Ballast Cost | EE Ballast Life (hrs) | EE Ballast Rep. Labor Cost | Baseline Description | # Base Lamps | Base Lamp Cost | Base Lamp Life (hrs) | Base Lamp Rep. Labor Cost | # Base Ballasts | Base Ballast Cost | Base Ballast Life (hrs) | Base Ballast Rep. Labor Cost |
|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|---|
| 3-Lamp T5 High-Bay | $12.00 | 20000 | $6.67 | $52.00 | 70000 | $22.50 | 200 Watt Pulse Start Metal-Halide | 1.00 | $21.00 | 10000 | $6.67 | 1.00 | $ 88 | 40000 | $22.50 |
| | | | | | | | 250 Watt Metal Halide | 1.00 | $21.00 | 10000 | $6.67 | 1.00 | $ 92 | 40000 | $22.50 |
| 4-Lamp T5 High-Bay | $12.00 | 20000 | $6.67 | $52.00 | 70000 | $22.50 | 320 Watt Pulse Start Metal-Halide | 1.00 | $72.00 | 20000 | $6.67 | 1.00 | $ 109 | 40000 | $22.50 |
| | | | | | | | 400 Watt Metal Halide | 1.00 | $17.00 | 20000 | $6.67 | 1.00 | $ 114 | 40000 | $22.50 |
| 6-Lamp T5 High-Bay | $12.00 | 20000 | $6.67 | $52.00 | 70000 | $22.50 | Proportionally Adjusted according to 6-Lamp HPT8 Equivalent to 320 PSMH | 1.36 | $72.00 | 20000 | $6.67 | 1.50 | $ 109 | 40000 | $22.50 |
| 1-Lamp T5 Troffer/Wrap | $12.00 | 20000 | $2.67 | $52.00 | 70000 | $15.00 | Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8 | 1.50 | $2.50 | 20000 | $2.67 | 0.50 | $ 15 | 70000 | $15.00 |
| 2-Lamp T5 Troffer/Wrap | $12.00 | 20000 | $2.67 | $52.00 | 70000 | $15.00 | 3-Lamp F32T8 Equivalent w/ Elec. Ballast | 3.00 | $2.50 | 20000 | $2.67 | 1.00 | $ 15 | 70000 | $15.00 |
| 1-Lamp T5 Industrial/Strip | $12.00 | 20000 | $2.67 | $52.00 | 70000 | $15.00 | Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8 | 1.50 | $2.50 | 20000 | $2.67 | 0.50 | $ 15 | 70000 | $15.00 |
| 2-Lamp T5 Industrial/Strip | $12.00 | 20000 | $2.67 | $52.00 | 70000 | $15.00 | 3-Lamp F32T8 Equivalent w/ Elec. Ballast | 3.00 | $2.50 | 20000 | $2.67 | 1.00 | $ 15 | 70000 | $15.00 |
| 3-Lamp T5 Industrial/Strip | $12.00 | 20000 | $2.67 | $52.00 | 70000 | $15.00 | Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8 | 4.50 | $2.50 | 20000 | $2.67 | 1.50 | $ 15 | 70000 | $15.00 |
| 4-Lamp T5 Industrial/Strip | $12.00 | 20000 | $2.67 | $52.00 | 70000 | $15.00 | Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8 | 6.00 | $2.50 | 20000 | $2.67 | 2.00 | $ 15 | 70000 | $15.00 |
| 1-Lamp T5 Indirect | $12.00 | 20000 | $2.67 | $52.00 | 70000 | $15.00 | Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8 | 1.50 | $2.50 | 20000 | $2.67 | 0.50 | $ 15 | 70000 | $15.00 |
| 2-Lamp T5 Indirect | $12.00 | 20000 | $2.67 | $52.00 | 70000 | $15.00 | 3-Lamp F32T8 Equivalent w/ Elec. Ballast | 3.00 | $2.50 | 20000 | $2.67 | 1.00 | $ 15 | 70000 | $15.00 |

[588] Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, October 26, 2011
EPE Program Downloads. Web accessed http://www.epelectricefficiency.com/downloads.asp?section=ci download Copy of LSF_2012_v4.04_250rows.xls.
Kuiken et al, Focus on Energy Evaluation. Business Programs: Deemed Savings Manual v1.0, Kema, march 22, 2010 available at http://www.focusonenergy.com/files/Document_Management_System/Evaluation/bpdeemedsavingsmanuav10_evaluationreport.pdf

C-1: T12 Baseline Adjustment:

Savings Adjustment Factors

| | watts | Equivalent T12 watts adjusted for lumen equivalency-34 w and 40 w with EEMag ballast | Equivalent T12 watts adjusted for lumen equivalency-40 w with EEMag ballast | Equivalent T12 watts adjusted for lumen equivalency-40 w with Mag ballast | Prportionally Adjusted for Lumens wattage for T8 equivalent |
|---|---|---|---|---|---|
| 1-Lamp T5 Industrial/Strip | 32 | 61 | 73 | 82 | 44 |
| 2-Lamp T5 Industrial/Strip | 64 | 103 | 125 | 135 | 88 |
| 3-Lamp T5 Industrial/Strip | 96 | 167 | 185 | 211 | 132 |
| 4-Lamp T5 Industrial/Strip | 128 | 211 | 249 | 226 | 178 |
| | | Savings Factor Adjustment to the T8 baseline | Savings Factor Adjustment to the T8 baseline | Savings Factor Adjustment to the T8 baseline | |
| 1-Lamp T5 Industrial/Strip | | 42% | 29% | 24% | |
| 2-Lamp T5 Industrial/Strip | | 61% | 40% | 34% | |
| 3-Lamp T5 Industrial/Strip | | 51% | 40% | 31% | |
| 4-Lamp T5 Industrial/Strip | | 60% | 41% | 51% | |

Measures installed in 2012 will claim full savings for four years, 2013 for three years, 2014 two years and 2015 one year. Savings adjustment factors based on a T8 baseline will be applied to the full savings for savings starting in 2016 and for the remainder of the measure life. The adjustment to be applied for each measure is listed in the reference table above and is based on equivalent lumens.

**MEASURE CODE: CI-LTG-T5FX-V02-140601**

## 4.5.13 Occupancy Controlled Bi-Level Lighting Fixtures

### DESCRIPTION

This measure relates to replacing existing uncontrolled continuous lighting fixtures with new bi-level lighting fixtures. This measure can only relate to replacement in an existing building, since multi-level switching is required in the Commercial new construction building energy code (IECC 2012).

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient system is assumed to be an occupancy controlled lighting fixture that reduces light level during unoccupied periods.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be an uncontrolled lighting system on continuously, e.g. in stairwells and corridors for health and safety reasons.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life for all lighting controls is assumed to be 8 years[589].

### DEEMED MEASURE COST

When available, the actual cost of the measure shall be used. When not available, the assumed measure cost is $274[590].

### LOADSHAPE

Loadshape C06 - Commercial Indoor Lighting

Loadshape C07 - Grocery/Conv. Store Indoor Lighting

Loadshape C08 - Hospital Indoor Lighting

Loadshape C09 - Office Indoor Lighting

Loadshape C10 - Restaurant Indoor Lighting

Loadshape C11 - Retail Indoor Lighting

Loadshape C12 - Warehouse Indoor Lighting

Loadshape C13 - K-12 School Indoor Lighting

Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)

Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)

Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)

---

[589] DEER 2008.

[590] Consistent with the Multi-level Fixture measure with reference to Goldberg et al, State of Wisconsin Public Service Commission of Wisconsin, Focus on Energy Evaluation, Business Programs: Incremental Cost Study, KEMA, October 28, 2009. Also consistent with field experience of about $250 per fixture and $25 install labor.

Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)

Loadshape C18 - Industrial Indoor Lighting

Loadshape C19 - Industrial Outdoor Lighting

Loadshape C20 - Commercial Outdoor Lighting

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the location type. Values are provided for each building type in the reference section below.

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = (KW_{Baseline} - (KW_{Controlled} * (1 - ESF))) * Hours * WHF_e$$

Where:

$KW_{Baseline}$ = Total baseline lighting load of the existing/baseline fixture

= Actual

Note that if the existing fixture is only being retrofit with bi-level occpuancy controls and not being replaced $KW_{Baseline}$ will equal $KW_{Controlled}$ .

$KW_{Controlled}$ = Total contolled lighting load at full light output of the new bi-level fixture

= Actual

Hours = Number of hours lighting is on. This measure is limited to 24/7 operation.

= 8,766

ESF = Energy Savings factor (represents the percentage reduction to the $KW_{Controlled}$ due to the occupancy control).

= % Standby Mode * (1 - % Full Light at Standby Mode)

% Standby Mode = Represents the percentage of the time the fixture is operating in standby (i.e. low-wattage) mode.

% Full Light at Standby Mode = Represents the assumed wattage consumption during standby mode relative to the full wattage consumption. Can be achieved either through dimming or a stepped control strategy.

= Dependent on application. If customer provided or metered data is available for both or either of these inputs a custom savings factor should be calculated. If not defaults are provided below:

| Application | % Standby Mode | % Full Light at Standby Mode | Energy Savings Factor (ESF) |
|---|---|---|---|

| Application | % Standby Mode | % Full Light at Standby Mode | Energy Savings Factor (ESF) |
|---|---|---|---|
| Stairwells | 78.5%[591] | 50% | 39.3% |
| | | 33% | 52.6% |
| | | 10% | 70.7% |
| | | 5% | 74.6% |
| Corridors | 50.0%[592] | 50% | 25.0% |
| | | 33% | 33.5% |
| | | 10% | 45.0% |
| | | 5% | 47.5% |
| Other 24/7 Space Type | 50.0%[593] | 50% | 25.0% |
| | | 33% | 33.5% |
| | | 10% | 45.0% |
| | | 5% | 47.5% |

$WHF_e$ = Waste heat factor for energy to account for cooling energy savings from efficient lighting is provided in the Reference Table in Section 4.5 for each building type. If building is un-cooled, the value is 1.0.

**HEATING PENALTY**

If electrically heated building:

$$\Delta kWh_{heatpenalty}{}^{594} = (KW_{Baseline} - (KW_{Controlled} * (1 - ESF))) * Hours * -IFkWh$$

Where:

IFkWh = Lighting-HVAC Interation Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficent lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$$\Delta kW = (KW_{Baseline} - (KW_{Controlled} * (1 - ESF))) * WHF_d * (CF_{baseline} - CF_{os})$$

[591] Average found from the four buildings in the State of California Energy Commission Lighting Research Program Bi-Level Stairwell Fixture Performance Final Report: http://www.archenergy.com/lrp/lightingperf_standards/project_5_1_reports.htm

[592] Value determined from the Pacific Gas and Electric Company: Bi-Level Lighting Control Credits study for Interior Corridors of Hotels, Motels and High Rise Residential. http://www.energy.ca.gov/title24/2005standards/archive/documents/2002-07-18_workshop/2002-07-18_BILEVEL_LIGHTING.PDF

[593] Conservative estimate.

[594] Negative value because this is an increase in heating consumption due to the efficient lighting.

Where:

$WHF_d$ = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is provided in the Reference Table in Section 4.5. If the building is un-cooled WHFd is 1.

$CF_{baseline}$ = Baseline Summer Peak Coincidence Factor for the lighting system without Occupancy Sensors installed selected from the Reference Table in Section 4.5 for each building type. If the building type is unknown, use the Miscellaneous value of 0.66

$CF_{os}$ = Retrofit Summer Peak Coincidence Factor the lighting system with Occupancy Sensors installed is 0.15 regardless of building type.[595]

**NATURAL GAS HEATING PENALTY**

If natural gas heating:

$$\Delta therms = (KW_{Baseline} - (KW_{Controlled} *(1 – ESF))) * Hours * - IFTherms$$

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting and provided in the Reference Table in Section 4.5 by buidling type.

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-LTG-OCBL-V01-140601**

[595] Coincidence Factor Study Residential and Commercial Industrial Lighting Measures, RLW Analytics, Spring 2007. Note, the connected load used in the calculation of the CF for occupancy sensor lights includes the average ESF.

## 4.5.14 Commercial ENERGY STAR Specialty Compact Fluorescent Lamp (CFL)

### DESCRIPTION

An ENERGY STAR qualified specialty compact fluorescent bulb is installed in place of an incandescent specialty bulb in a commercial location. If the implementation strategy does not allow for the installation location to be known a deemed split should be used. For Residential targeted programs (e.g. an upstream retail program), a deemed split of 96% Residential and 4% Commercial assumptions should be used[596], and for Commercial targeted programs a deemed split of 4% Residential and 96% Commercial should be used[597].

This measure was developed to be applicable to the following program types: TOS, NC, RF.
If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

Energy Star qualified specialty CFL bulb based upon the draft ENERGY STAR specification for lamps (http://energystar.gov/products/specs/sites/products/files/ENERGY_STAR_Lamps_V1_0_Draft%203.pdf).

### DEFINITION OF BASELINE EQUIPMENT

The baseline is a specialty incandescent light bulb including those exempt of the EISA 2007 standard: three-way, plant light, daylight bulb, bug light, post light, globes G40 (<40W), candelabra base (<60W), vibration service bulb, decorative candle with medium or intermediate base (<40W), shatter resistant and reflector bulbs and standard bulbs greater than 2601 lumens, and those non-exempt from EISA 2007: dimmable, globes (less than 5" diameter and >40W), candle (shapes B, BA, CA >40W, candelabra base lamps (>60W) and intermediate base lamps (>40W).

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life (number of years that savings should be claimed) should be calculated by dividing the rated life of the bulb (10,000 hours[598]) by the run hours. For example using Miscellaneous at 4,589 hours would give 2.2 years. When the number of years exceeds June 2020, the number of years to that date should be used.

### DEEMED MEASURE COST

For the Retail (Time of Sale) measure, the incremental capital cost for this measure is $5[599].

For the Refrtofit measures, the full cost of $8.50 should be used plus $5 labor[600] for a total of $13.50. However actual program delivery costs should be utilized if available.

### LOADSHAPE

Loadshape C06 - Commercial Indoor Lighting

Loadshape C07 - Grocery/Conv. Store Indoor Lighting

Loadshape C08 - Hospital Indoor Lighting

Loadshape C09 - Office Indoor Lighting

[596] RES v C&I split is based on a weighted (by sales volume) average of ComEd PY4-6 and Ameren PY5-6 in store intercept survey results.
[597] Based upon final weighted (by sales volume) average of the BILD program (ComEd's commercial lighting program) for PY 4 and PY5 and PY6.
[598] Energy Star bulbs have a rated life of at least 8000 hours. In commercial settings you expect significantly less on/off switching than residential and so a rated life assumption of 10,000 hours is used.
[599] NEEP Residential Lighting Survey, 2011
[600] Based on 15 minutes at $20 per hour.

Loadshape C10 - Restaurant Indoor Lighting
Loadshape C11 - Retail Indoor Lighting
Loadshape C12 - Warehouse Indoor Lighting
Loadshape C13 - K-12 School Indoor Lighting
Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)
Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)
Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)
Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)
Loadshape C18 - Industrial Indoor Lighting
Loadshape C19 - Industrial Outdoor Lighting
Loadshape C20 - Commercial Outdoor Lighting

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the location type. Values are provided for each building type in section 4.5.

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

ΔkWh = ((WattsBase - WattsEE) / 1000) * ISR * Hours * WHFe

Where:

WattsBase = Actual wattage equivalent of incandescent specialty bulb, use the tables below to obtain the incandescent bulb equivalent wattage[601]; use 60W if unknown[602]

EISA exempt bulb types:

| Bulb Type | Lower Lumen Range | Upper Lumen Range | WattsBase |
|---|---|---|---|
| **Standard Spirals >=2601** | 2601 | 2999 | 150 |
| | 3000 | 5279 | 200 |
| | 5280 | 6209 | 300 |
| **3-Way** | 250 | 449 | 25 |
| | 450 | 799 | 40 |
| | 800 | 1099 | 60 |
| | 1100 | 1599 | 75 |

[601] Based upon the draft ENERGY STAR specification for lamps (http://energystar.gov/products/specs/sites/products/files/ENERGY_STAR_Lamps_V1_0_Draft%203.pdf) and the Energy Policy and Conservation Act of 2012.

[602] A 2006-2008 California Upstream Lighting Evaluation found an average incandescent wattage of 61.7 Watts (KEMA, Inc, The Cadmus Group, Itron, Inc, PA Consulting Group, Jai J. Mitchell Analytics, Draft Evaluation Report: Upstream Lighting Program. Prepared for the California Public Utilities Commission, Energy Division. December 10, 2009)

| Bulb Type | Lower Lumen Range | Upper Lumen Range | WattsBase |
|---|---|---|---|
| | 1600 | 1999 | 100 |
| | 2000 | 2549 | 125 |
| | 2550 | 2999 | 150 |
| **Globe (medium and intermediate bases less than 750 lumens)** | 90 | 179 | 10 |
| | 180 | 249 | 15 |
| | 250 | 349 | 25 |
| | 350 | 749 | 40 |
| **Decorative (Shapes B, BA, C, CA, DC, F, G, medium and intermediate bases less than 750 lumens)** | 70 | 89 | 10 |
| | 90 | 149 | 15 |
| | 150 | 299 | 25 |
| | 300 | 749 | 40 |
| **Globe (candelabra bases less than 1050 lumens)** | 90 | 179 | 10 |
| | 180 | 249 | 15 |
| | 250 | 349 | 25 |
| | 350 | 499 | 40 |
| | 500 | 1049 | 60 |
| **Decorative (Shapes B, BA, C, CA, DC, F, G, candelabra bases less than 1050 lumens)** | 70 | 89 | 10 |
| | 90 | 149 | 15 |
| | 150 | 299 | 25 |
| | 300 | 499 | 40 |
| | 500 | 1049 | 60 |

EISA non-exempt bulb types:

| Bulb Type | Lower Lumen Range | Upper Lumen Range | Incandescent Equivalent Post-EISA 2007 (WattsBase) |
|---|---|---|---|
| **Dimmable Twist, Globe (less than 5" in diameter and > 749 lumens), candle (shapes B, BA, CA > 749 lumens), Candelabra Base Lamps (>1049 lumens), Intermediate Base Lamps (>749 lumens)** | 310 | 749 | 29 |
| | 750 | 1049 | 43 |
| | 1050 | 1489 | 53 |
| | 1490 | 2600 | 72 |

**Directional Lamps -** ENERGY STAR Minimum Luminous Efficacy = 40Lm/W for lamps with rated wattages less than 20Wand 50 Lm/W for lamps with rated wattages >= 20 watts[603].

[603] From pg 10 of the Energy Star Specification for lamps v1.1

For Directional R, BR, and ER lamp types[604]:

| Bulb Type | Lower Lumen Range | Upper Lumen Range | $Watts_{Base}$ |
|---|---|---|---|
| **R, ER, BR with medium screw bases w/ diameter >2.25" (*see exceptions below)** | 420 | 472 | 40 |
| | 473 | 524 | 45 |
| | 525 | 714 | 50 |
| | 715 | 937 | 65 |
| | 938 | 1259 | 75 |
| | 1260 | 1399 | 90 |
| | 1400 | 1739 | 100 |
| | 1740 | 2174 | 120 |
| | 2175 | 2624 | 150 |
| | 2625 | 2999 | 175 |
| | 3000 | 4500 | 200 |
| ***R, BR, and ER with medium screw bases w/ diameter <=2.25"** | 400 | 449 | 40 |
| | 450 | 499 | 45 |
| | 500 | 649 | 50 |
| | 650 | 1199 | 65 |
| ***ER30, BR30, BR40, or ER40** | 400 | 449 | 40 |
| | 450 | 499 | 45 |
| | 500 | 649 | 50 |
| ***BR30, BR40, or ER40** | 650 | 1419 | 65 |
| ***R20** | 400 | 449 | 40 |
| | 450 | 719 | 45 |
| ***All reflector lamps below lumen ranges specified above** | 200 | 299 | 20 |
| | 300 | 399 | 30 |

Directional lamps are exempt from EISA regulations.

For PAR, MR, and MRX Lamps Types:

For these highly focused directional lamp types, it is necessary to have Center Beam Candle Power (CBCP) and beam angle measurements to accurately estimate the equivalent baseline wattage. The formula below is based on the Energy Star Center Beam Candle Power tool.[605] If CBCP and beam angle information are not available, refer to the R, BR, and ER lumen based method above.

[604] From pg 11 of the Energy Star Specification for lamps v1.1

[605] http://energystar.supportportal.com/link/portal/23002/23018/Article/32655/

$$\text{Wattsbase} = 375.1 - 4.355(D) - \sqrt{227{,}800 - 937.9(D) - 0.9903(D^2) - 1479(BA) - 12.02(D * BA) + 14.69(BA^2) - 16{,}720 * \ln(CBCP)}$$

Where:

D = Bulb diameter (e.g. for PAR20 D = 20)

BA = Beam angle

CBCP = Center beam candle power

The result of the equation above should be rounded DOWN to the nearest wattage established by Energy Star:

| Diameter | Permitted Wattages |
|---|---|
| 16 | 20, 35, 40, 45, 50, 60, 75 |
| 20 | 50 |
| 30S | 40, 45, 50, 60, 75 |
| 30L | 50, 75 |
| 38 | 40, 45, 50, 55, 60, 65, 75, 85, 90, 100, 120, 150, 250 |

EISA non-exempt bulb types:

| Bulb Type | Lower Lumen Range | Upper Lumen Range | Incandescent Equivalent Post-EISA 2007 (WattsBase) |
|---|---|---|---|
| **Dimmable Twist, Globe (less than 5" in diameter and > 749 lumens), candle (shapes B, BA, CA > 749 lumens), Candelabra Base Lamps (>1049 lumens), Intermediate Base Lamps (>749 lumens)** | 310 | 749 | 29 |
| | 750 | 1049 | 43 |
| | 1050 | 1489 | 53 |
| | 1490 | 2600 | 72 |

WattsEE = Actual wattage of energy efficient specialty bulb purchased, use 15W if unknown[606]

ISR = In Service Rate or the percentage of units rebated that get installed.

[606] An evaluation (Energy Efficiency / Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: Residential Energy Star ® Lighting http://ilsag.org/yahoo_site_admin/assets/docs/ComEd_Res_Lighting_PY2_Evaluation_Report_2010-12-21_Final.12113928.pdf ) reported 13-17W as the most common specialty CFL wattage (69% of program bulbs). 2009 California data also reported an average CFL wattage of 15.5 Watts (KEMA, Inc, The Cadmus Group, Itron, Inc, PA Consulting Group, Jai J. Mitchell Analytics, Draft Evaluation Report: Upstream Lighting Program, Prepared for the California Public Utilities Commission, Energy Division. December 10, 2009).

=100%[607] if application form completed with sign off that equipment is not placed into storage

If sign off form not completed assume the following 3 year ISR assumptions:

| Weigted Average 1st year In Service Rate (ISR) | 2nd year Installations | 3rd year Installations | Final Lifetime In Service Rate |
|---|---|---|---|
| 71.2%[608] | 14.5% | 12.3% | 98.0%[609] |

Hours = Average hours of use per year are provided in Reference Table in Section 4.5, Screw based bulb annual operating hours, for each building type[610]. If unknown use the Miscellaneous value.

WHFe = Waste heat factor for energy to account for cooling energy savings from efficient lighting are provided below for each building type in Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

**DEFERRED INSTALLS**

As presented above, the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

Year 1 (Purchase Year) installs: Characterized using assumptions provided above or evaluated assumptions if available.

Year 2 and 3 installs: Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.

The NTG factor for the Purchase Year should be applied.

[607] Illinois evaluation of PY1 through PY3 has not found that fixtures or lamps placed into storage to be a significant enough issue to warrant including an "In-Service Rate" when commercial customers complete an application form.

[608] 1st year in service rate is based upon review of PY4-6 evaluations from ComEd's commercial lighting program (BILD) (see 'IL Commercial Lighting ISR_2014.xls' for more information. The average first year ISR was calculated weighted by the number of bulbs sold.

[609] The 98% Lifetime ISR assumption is based upon review of two evaluations:
'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings. Note that this Final Install Rate does NOT account for leakage of purchased bulbs being installed outside of the utility territory. EM&V should assess how and if data from evaluation should adjust this final installation rate to account for this impact

[610] Based on ComEd analysis taking DEER 2008 values and averaging with PY1 and PY2 evaluation results.

**EXAMPLE**

For example, for a 14W 500 lumen R20 reflector lamp is installed in an office and sign off form provided.

$\Delta kWh$ = (((45 - 14)/1000) * 1.0 * 3088 * 1.25

= 119.7 kWh

### HEATING PENALTY

If electrically heated building:

$\Delta kWh_{heatpenalty}$[611] = (((WattsBase-WattsEE)/1000) * ISR * Hours * -IFkWh

Where:

IFkWh = Lighting-HVAC Interation Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficent lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

**EXAMPLE**

For example, for a 14W 500 lumen R20 reflector lamp is installed in a heat pump heated office and sign off form provided.

$\Delta kWh_{heatpenalty}$ = (((45 - 14)/1000) * 1.0 * 3088 * -0.183

= - 17.5 kWh

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW$ = ((WattsBase-WattsEE)/1000) * ISR * WHFd * CF

Where:

WHFd = Waste heat factor for demand to account for cooling savings from efficient lighting in cooled buildings is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value..

CF = Summer Peak Coincidence Factor for measure is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value..

Other factors as defined above

[611] Negative value because this is an increase in heating consumption due to the efficient lighting.

**EXAMPLE**

For example, for a 14W 500 lumen R20 reflector lamp is installed in an office and sign off form provided.

ΔkW = ((45 - 14)/1000) * 1.0 * 1.3 * 0.66

= 0.027kW

**NATURAL GAS SAVINGS**

Heating Penalty if fossil fuel heated building (or if heating fuel is unknown):

ΔTherms[612] = (((WattsBase-WattsEE)/1000) * ISR * Hours *- IFTherms

Where:

IFTherms = Lighting-HVAC Interation Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficent lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

Other factors as defined above

**EXAMPLE**

For example, for a 14W 500 lumen R20 reflector lamp is installed in a gas heated office and sign off form provided.

ΔTherms = (((45 - 14)/1000) * 1.0 * 3088 * -0.016

= - 1.5 Therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

The following O&M assumptions should be used: Life of the baseline bulb is assumed to be (1000/HOURS) year; baseline replacement cost is assumed to be $3.5 for those bulbs types exempt from EISA and $5 for non-exempt EISA bulb types defined above[613].

Measure Code: CI-LTG-SCFL-V01-150601

[612] Negative value because this is an increase in heating consumption due to the efficient lighting.
[613] NEEP Residential Lighting Survey, 2011

# 4.6 Refrigeration End Use

## 4.6.1 Automatic Door Closer for Walk-In Coolers and Freezers

### DESCRIPTION

This measure is for installing an auto-closer to the main insulated opaque door(s) of a walk-in cooler or freezer. The auto-closer must firmly close the door when it is within 1 inch of full closure.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

This measure consists of the installation of an automatic, hydraulic-type door closer on main walk-in cooler or freezer doors. These closers save energy by reducing the infiltration of warm outside air into the refrigeration itself.

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline condition is assumed to be a walk in cooler or freezer without an automatic closure.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The deemed measure life is 8 years.[614]

### DEEMED MEASURE COST

The deemed measure cost is $156.82 for a walk-in cooler or freezer.[615]

### LOADSHAPE

Loadshape C22 - Commercial Refrigeration

### COINCIDENCE FACTOR

The measure has deemed kW savings therefore a coincidence factor does not apply.

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

Savings calculations are based on values from through PG&E's Workpaper PGECOREF110.1 – Auto-Closers for Main Cooler or Freezer Doors. Savings are averaged across all California climate zones and vintages[616].

[614] Source: DEER 2008
[615] Ibid.
[616] Measure savings from ComEd TRM developed by KEMA. June 1, 2010

| Annual Savings | kWh |
|---|---|
| Walk in Cooler | 943 |
| Walk in Freezer | 2307 |

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

| Annual Savings | kW |
|---|---|
| Walk in Cooler | 0.137 |
| Walk in Freezer | 0.309 |

### NATURAL GAS ENERGY SAVINGS

N/A

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-RFG-ATDC-V01-120601

## 4.6.2 Beverage and Snack Machine Controls

### DESCRIPTION

This measure relates to the installation of new controls on refrigerated beverage vending machines, non-refrigerated snack vending machines, and glass front refrigerated coolers. Controls can significantly reduce the energy consumption of vending machine and refrigeration systems. Qualifying controls must power down these systems during periods of inactivity but, in the case of refrigerated machines, must always maintain a cool product that meets customer expectations. This measure relates to the installation of a new control on a new or existing unit. This measure should **not** be applied to ENERGY STAR qualified vending machines, as they already have built-in controls.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a standard efficiency refrigerated beverage vending machine, non-refrigerated snack vending machine, or glass front refrigerated cooler with a control system capable of powering down lighting and refrigeration systems during periods of inactivity.

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to be a standard efficiency refrigerated beverage vending machine, non-refrigerated snack vending machine, or glass front refrigerated cooler without a control system capable of powering down lighting and refrigeration systems during periods of inactivity

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 5 years [617].

### DEEMED MEASURE COST

The actual measure installation cost should be used (including material and labor), but the following can be assumed for analysis purposes[618]:

Refrigerated Vending Machine and Glass Front Cooler: $180.00

Non-Refrigerated Vending Machine: $80.00

### LOADSHAPE

Loadshape C52 - Beverage and Snack Machine Controls

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 0[619].

[617] Measure Life Study, prepared for the Massachusetts Joint Utilities, Energy & Resource Solutions, November 2005.
[618] ComEd workpapers, 8—15-11.pdf
[619] Assumed that the peak period is coincident with periods of high traffic diminishing the demand reduction potential of occupancy based controls.

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

ΔkWh = WATTSbase / 1000 * HOURS * ESF

Where:

WATTSbase = connected W of the controlled equipment; see table below for default values by connected equipment type:

| Equipment Type | WATTSbase[620] |
|---|---|
| Refrigerated Beverage Vending Machines | 400 |
| Non-Refrigerated Snack Vending Machines | 85 |
| Glass Front Refrigerated Coolers | 460 |

1000 = conversion factor (W/kW)

HOURS = operating hours of the connected equipment; assumed that the equipment operates 24 hours per day, 365.25 days per year

= 8766

ESF = Energy Savings Factor; represents the percent reduction in annual kWh consumption of the equipment controlled; see table below for default values:

| Equipment Type | Energy Savings Factor (ESF)[621] |
|---|---|
| Refrigerated Beverage Vending Machines | 46% |
| Non-Refrigerated Snack Vending Machines | 46% |
| Glass Front Refrigerated Coolers | 30% |

**EXAMPLE**

For example, adding controls to a refrigerated beverage vending machine:

ΔkWh = WATTSbase / 1000 * HOURS * ESF

=400/1000* 8766* 0.46

= 1613 kWh

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

[620] USA Technologies Energy Management Product Sheets, July 2006; cited September 2009. <http:// http://www.usatech.com/energy_management/energy_productsheets.php>
[621] Ibid.

### NATURAL GAS ENERGY SAVINGS

N/A

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-RFG-BEVM-V02-150601

## 4.6.3 Door Heater Controls for Cooler or Freezer

### DESCRIPTION

By installing a control device to turn off door heaters when there is little or no risk of condensation, one can realize significant energy savings. There are two commercially available control strategies that achieve "on-off" control of door heaters based on either (1) the relative humidity of the air in the store or (2) the "conductivity" of the door (which drops when condensation appears). In the first strategy, the system activates your door heaters when the relative humidity in your store rises above a specific setpoint, and turns them off when the relative humidity falls below that setpoint. In the second strategy, the sensor activates the door heaters when the door conductivity falls below a certain setpoint, and turns them off when the conductivity rises above that setpoint.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a door heater control on a commercial glass door cooler or refrigerator utilizing humidity or conductivity control.

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline condition is assumed to be a commercial glass door cooler or refrigerator with a standard heated door with no controls installed.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years [622].

### DEEMED MEASURE COST

The incremental capital cost for a humidity-based control is $300 per circuit regardless of the number of doors controlled. The incremental cost for conductivity-based controls is $200[623].

### LOADSHAPE

Loadshape C51 - Door Heater Control

### COINCIDENCE FACTOR[624]

The summer peak coincidence factor for this measure is assumed to be 0%[625].

[622] 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, "Effective/Remaining Useful Life Values", California Public Utilities Commission, December 16, 2008.
[623] Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, February, 19, 2010
[624] Source partial list from DEER 2008
[625] Based on the assumption that humidity levels will most likely be relatively high during the peak period, reducing the likelihood of demand savings from door heater controls.

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

ΔkWH = kWbase * NUMdoors * ESF * BF * 8766

Where:

kWbase[626] = connected load kW for typical reach-in refrigerator or freezer door and frame with a heater.

= If actual kWbase is unknown, assume 0.195 kW for freezers and 0.092 kW for coolers.

NUMdoors = number of reach-in refrigerator or freezer doors controlled by sensor

= Actual installed

ESF[627] = Energy Savings Factor; represents the percentage of hours annually that the door heater is powered off due to the controls.

= assume 55% for humidity-based controls, 70% for conductivity-based controls

BF[628] = Bonus Factor; represents the increased savings due to reduction in cooling load inside the cases, and the increase in cooling load in the building space to cool the additional heat generated by the door heaters.

| Definition | Representative Evaporator Temperature Range, °F[629] | Typical Uses | BF |
|---|---|---|---|
| Low | -35 to 0 | Freezers for times such as frozen pizza, ice cream, etc. | 1.36 |
| Medium | 0 – 20 | Coolers for items such as meat, milk, dairy, etc | 1.22 |
| High | 20 – 45 | Coolers for items such as floral, produce and meat preperation rooms | 1.15 |

8766 = annual hours of operation

[626] A review of TRM methodologies from Vermont, New York, Wisconsin, and Connecticut reveals several different sources for this factor. Connecticut requires site-specific information, whereas New York's characterization does not explicitly identify the kWbase. Connecticut and Vermont provide values that are very consistent, and the simple average of these two values has been used for the purposes of this characterization.

[627] A review of TRM methodologies from Vermont, New York, Wisconsin, and Connecticut reveals several different estimates of ESF. Vermont is the only TRM that provides savings estimates dependent on the control type. Additionally, these estimates are the most conservative of all TRMs reviewed. These values have been adopted for the purposes of this characterization.

[628] Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, February, 19, 2010

[629] Energy Efficiency Supermarket Refrigeration, Wisconsin Electric Power Company, July 23, 1993

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

## NATURAL GAS ENERGY SAVINGS

N/A

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-RFG-DHCT-V01-120601

## 4.6.4 Electronically Commutated Motors (ECM) for Walk-in and Reach-in Coolers / Freezers

### DESCRIPTION

This measure is applicable to the replacement of an existing standard-efficiency shaded-pole evaporator fan motor in refrigerated display cases or fan coil in walk-ins.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

This measure applies to the replacement of an existing standard-efficiency shaded-pole evaporator fan motor in refrigerated display cases or fan coil in walk-ins. The replacement unit must be an electronically commutated motor (ECM). This measure cannot be used in conjunction with the evaporator fan controller measure

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to be a shaded pole motor

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years[630]

### DEEMED MEASURE COST

The measure cost is assumed to be $50 for a walk in cooler and walk in freezer. [631]

### LOADSHAPE

Loadshape C22 - Commercial Refrigeration

### COINCIDENCE FACTOR

The measure has deemed peak kW savings therefore a coincidence factor does not apply.

**Algorithm**

### CALCULATION OF SAVINGS [632]

Savings values are obtained from the SCE workpaper for efficient evaporator fan motors, which covers all 16 California climate zones. SCE savings values were determined using a set of assumed conditions for restaurants and grocery stores. We have used only PG&E climate zones in calculating our averages and have taken out the drier, warmer climates of southern California. SCE's savings approach calculates refrigeration demand, by taking into consideration temperature, compressor efficiency, and various loads involved for both walk-in and reach-in refrigerators. Details on cooling load calculations, including refrigeration conditions, can be found in the SCE workpaper. The baseline for this measure assumes that the refrigeration unit has a shaded-pole motor. The

[630] DEER
[631] Act on Energy Commercial Technical Reference Manual No. 2010-4
[632] "Efficient Evaporator Fan Motors (Shaded Pole to ECM)," Workpaper WPSCNRRN0011. Southern California Edison Company. 2007.

following tables are values calculated within the SCE workpaper.

Table 156 SCE Restaurant Savings Walk-In

| | Restaurant | | | |
|---|---|---|---|---|
| SCE Workpaper Values | Cooler | | Freezer | |
| Northern California Climate Zones | kWh Savings Per Motor | Peak kW Savings Per Motor | kWh Savings Per Motor | Peak kW Savings Per Motor |
| 1 | 318 | 0.0286 | 507 | 0.03 |
| 2 | 253 | 0.033 | 263 | 0.037 |
| 3 | 364 | 0.0315 | 649 | 0.034 |
| 4 | 365 | 0.0313 | 652 | 0.034 |
| 5 | 350 | 0.0305 | 605 | 0.033 |
| 11 | 410 | 0.0351 | 780 | 0.04 |
| 12 | 399 | 0.034 | 748 | 0.039 |
| 13 | 407 | 0.0342 | 771 | 0.039 |
| 16 | 354 | 0.0315 | 620 | 0.034 |
| Average | 358 | 0.0322 | 622 | 0.036 |

Table 157: SCE Grocery Savings Walk-In

| | Grocery | | | |
|---|---|---|---|---|
| SCE Workpaper Values | Cooler | | Freezer | |
| Northern California Climate Zones | kWh Savings Per Motor | Peak kW Savings Per Motor | kWh Savings Per Motor | Peak kW Savings Per Motor |
| 1 | 318 | 0.0284 | 438 | 0.03 |
| 2 | 252 | 0.0534 | 263 | 0.064 |
| 3 | 364 | 0.0486 | 552 | 0.056 |
| 4 | 365 | 0.048 | 553 | 0.055 |
| 5 | 349 | 0.0452 | 516 | 0.051 |
| 11 | 410 | 0.0601 | 656 | 0.074 |
| 12 | 398 | 0.0566 | 631 | 0.069 |
| 13 | 406 | 0.0574 | 649 | 0.07 |
| 16 | 354 | 0.0486 | 528 | 0.056 |
| Average | 357 | 0.0496 | 532 | 0.058 |

Table 158: SCE Grocery Savings Reach-In

| | Grocery | | | |
|---|---|---|---|---|
| SCE Workpaper Values | Cooler | | Freezer | |
| Northern California Climate Zones | kWh Savings Per Motor | Peak kW Savings Per Motor | kWh Savings Per Motor | Peak kW Savings Per Motor |
| 1 | 306 | 0.031 | 362 | 0.031 |
| 2 | 269 | 0.033 | 273 | 0.035 |
| 3 | 331 | 0.032 | 421 | 0.034 |
| 4 | 332 | 0.032 | 422 | 0.034 |
| 5 | 323 | 0.032 | 402 | 0.033 |
| 11 | 357 | 0.034 | 476 | 0.037 |
| 12 | 350 | 0.034 | 462 | 0.036 |
| 13 | 355 | 0.034 | 472 | 0.037 |
| 16 | 325 | 0.032 | 409 | 0.034 |
| Average | 328 | 0.033 | 411 | 0.035 |

Savings values in the following table are an average of walk-in cooler (80 percent) and freezer (20 percent) applications. The workpapers for the 2006-2008 program years include this distribution of coolers and freezers in their refrigeration measure savings analyses.

### ELECTRIC ENERGY SAVINGS

The following table provides the kWh savings.

| Building type | kWh Savings/motor |
|---|---|
| Restaurant | 411 |
| Grocery | 392 |
| Average | 401 |

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

The following table provides the kW savings

| Building Type | Peak kW Savings/motor |
|---|---|
| Restaurant | 0.033 |
| Grocery | 0.051 |
| Average | 0.042 |

## NATURAL GAS ENERGY SAVINGS

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-RFG-ECMF-V01-120601**

## 4.6.5 ENERGY STAR Refrigerated Beverage Vending Machine

### DESCRIPTION

ENERGY STAR qualified new and rebuilt vending machines incorporate more efficient compressors, fan motors, and lighting systems as well as low power mode option that allows the machine to be placed in low-energy lighting and/or low-energy refrigeration states during times of inactivity.

This measure was developed to be applicable to the following program types: TOS, NC .

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The refrigerated vending machine can be new or rebuilt but must meet the ENERGY STAR specifications which include low power mode.

### DEFINITION OF BASELINE EQUIPMENT

The baseline vending machine is a standard unit

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The deemed lifetime of this measure is 14 years[633]

### DEEMED MEASURE COST

The incremental cost of this measure is $500[634]

### LOADSHAPE

Loadshape C22 - Commercial Refrigeration

### COINCIDENCE FACTOR

It is assumed that controls are only effective during off-peak hours and so have no peak-kW savings.

**Algorithm**

### CALCULATION OF SAVINGS

Beverage machine savings are taken from the ENERGY STAR savings calculator and summarized in the following table. ENERGY STAR provides savings numbers for machines with and without control software. The average savings are calculated here.

### ELECTRIC ENERGY SAVINGS

ENERGY STAR Vending Machine Savings[635]

[633] ENERGY STAR
[634] ENERGY STAR
[635] Savings from Vending Machine Calculator:
http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=VMC

| Vending Machine Capacity (cans) | kWh Savings Per Machine w/o software | kWh Savings Per Machine w/ software |
|---|---|---|
| <500 | 1,099 | 1,659 |
| 500 - 599 | 1,754 | 2,231 |
| 600 - 699 | 1,242 | 1,751 |
| 700 - 799 | 1,741 | 2,283 |
| 800+ | 713 | 1,288 |
| Average | 1,310 | 1,842 |

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

N/A

**NATURAL GAS ENERGY SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-RFG-ESVE-V02-150601**

## 4.6.6 Evaporator Fan Control

### DESCRIPTION

This measure is for the installation of controls in existing medium temperature walk-in coolers. The controller reduces airflow of the evaporator fans when there is no refrigerant flow.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The measure must control a minimum of 1/20 HP where fans operate continuously at full speed. The measure also must reduce fan motor power by at least 75% during the off cycle. This measure is not applicable if any of the following conditions apply:

- The compressor runs all the time with high duty cycle
- The evaporator fan does not run at full speed all the time
- The evaporator fan motor runs on poly-phase power
- Evaporator does not use off-cycle or time-off defrost.

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline measure is assumed to be a cooler with continuously running evaporator fan. An ECM can also be updated with controls.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 16 years[636]

### DEEMED MEASURE COST

The measure cost is assumed to be $291[637]

### LOADSHAPE

Loadshape C46 - Evaporator Fan Control

### COINCIDENCE FACTOR

The measure has deemed kW savings therefore a coincidence factor does not apply.

**Algorithm**

### CALCULATION OF SAVINGS

Savings for this measure were obtained from the DEER database. The baseline is assumed to be evaporator fans that run continuously with either a permanent split capacitor or shaded-pole motors. In the energy-efficient case

[636] Source: DEER
[637] Source: DEER

the fan is still assumed to operate even with the evaporator inactive.[638]

### ELECTRIC ENERGY SAVINGS

DEER provides savings numbers for building vintages and grocery only. The numbers are averages of these vintages. We are assuming that this measure will be applicable for all building types. The DEER savings vary by climate zone between 476 and 483 kWh/motor. Climate zone most closely remembling IL are 1, 3, and 16. The simple average of the savings in those zones is given below.[639]

$$\Delta kWh = \text{Savings per motor} * \text{motors}$$

Where:

| | |
|---|---|
| Savings per motor | = 481 kWh |
| motors | = number of fan motors controlled |

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

Using the same source and methodology as for ΔkWh:

$$\Delta kW = 0.060\ kW$$

## NATURAL GAS ENERGY SAVINGS

N/A

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-RFG-EVPF-V02-140601

[638] 2005 Database for Energy Efficiency Resources (DEER) Update Study Final Report
[639] See “Ca Climate Zone Translation.docx” and “CDD Base 80 zone comparison.xlsx”

## 4.6.7 Strip Curtain for Walk-in Coolers and Freezers

**DESCRIPTION**

This commercial measure pertains to the installation of infiltration barriers (strip curtains) on walk-in coolers or freezers. Strip curtains impede heat transfer from adjacent warm and humid spaces into walk-ins when the main door is opened, thereby reducing the cooling load. As a result, compressor run time and energy consumption are reduced. The engineering assumption is that the walk-in door is open 72 minutes per day every day, and the strip curtain covers the entire door frame.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

**DEFINITION OF EFFICIENT EQUIPMENT**

The efficient equipment is a polyethylene strip curtain added to a walk-in cooler or freezer

**DEFINITION OF BASELINE EQUIPMENT**

The baseline assumption is a walk-in cooler or freezer that previously had either no strip curtain installed or an old, ineffective strip curtain installed.

**DEEMED LIFETIME OF EFFICIENT EQUIPMENT**

The expected measure life is assumed to be 6 years[640].

**DEEMED MEASURE COST**

The incremental capital cost for this measure is $286.16 [641]

**LOADSHAPE**

Loadshape C22 - Commercial Refrigeration

**COINCIDENCE FACTOR**

The summer peak coincidence factor for this measure is 100%[642].

[640] M. Goldberg, J. Ryan Barry, B. Dunn, M. Ackley, J. Robinson, and D. Deangelo-Woolsey, KEMA. "Focus on Energy: Business Programs – Measure Life Study", August 2009.

[641] Assume average walk in door size is 3.5 feet wide and 8 feet tall or 28 square feet. The reference for incremental cost is $10.22 per square foot of door opening (includes material and labor). 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, "Cost Values and Summary Documentation", California Public Utilities Commission, December 16, 2008, Therefore incremental cost per door is $286.16

[642] The summer coincident peak demand reduction is assumed as the total annual savings divided by the total number of hours per year, effectively assuming the average demand reduction is realized during the peak period. This is a reasonable assumption for refrigeration savings.

**Algorithm**

**CALCULATION OF SAVINGS**

**ELECTRIC ENERGY SAVINGS**[643]

ΔkWh = 2,974 per freezer with curtains installed

= 422 per cooler with curtains installed

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ΔkWh / 8766 * CF

= 0.34 for freezers

= 0.05 for coolers

Where:

8766 = hours per year

CF = Summer Peak Coincidence Factor for the measure

= 1.0

**NATURAL GAS ENERGY SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-RFG-CRTN-V03-150601**

[643] Values based on analysis prepared by ADM for FirstEnergy utilities in Pennsylvania, provided via personal communication with Diane Rapp of FirstEnergy on June 4, 2010. Based on a review of deemed savings assumptions and methodologies from Oregon and California, the values from Pennsylvania appear reasonable and are the most applicable.

## 4.6.8 Refrigeration Economizers

**DESCRIPTION**

This measure applies to commercial walk in refrigeration systems and includes two components, outside air economizers and evaporator fan controllers. Economizers save energy by bringing in outside air when weather conditions allow, rather than operating the compressor. Walk-in refrigeration systems evaporator fans run almost all the time; 24 hrs/day, 365 days/yr. This is because they must run constantly to provide cooling when the compressor is running, and to provide air circulation when the compressor is not running. However, evaporator fans are a very inefficient method of providing air circulation. Installing an evaporator fan control system will turn off evaporator fans while the compressor is not running, and instead turn on an energy-efficient 35 watt fan to provide air circulation, resulting in significant energy savings. This measure allows for economizer systems with evaporator fan controls plus a circulation fan and without a circulation fan.

**DEFINITION OF EFFICIENT EQUIPMENT**

To qualify for this measure an economizer is installed on a walk in refrigeration system.

**DEFINITION OF BASELINE EQUIPMENT**

The baseline condition is a walk-in refrigeration system without an economizer

**DEEMED LIFETIME OF EFFICIENT EQUIPMENT**

The estimated life of this measure is 15 years[644].

**DEEMED MEASURE COST**

The installation cost for an economizer is $2,558.[645]

**LOADSHAPE**

Loadshape C22 - Commercial Refrigeration

**COINCIDENCE FACTOR**

The summer peak coincidence factor for this measure is assumed to be 0%[646].

[644] Estimated life from Efficiency Vermont TRM
[645] Based on average of costs from Freeaire, Natural Cool, and Cooltrol economizer systems.
[646] Based on the assumption that humidity levels will most likely be relatively high during the peak period, reducing the likelihood of demand savings.

**Algorithm**

**CALCULATION OF ENERGY SAVINGS**

**ELECTRIC ENERGY SAVINGS**

Electric energy savings is calculated based on whether evaporator fans run all

With Fan Control Installed

ΔkWh = [HP * kWhCond] + [((kWEvap * nFans) – kWCirc) * Hours * DCComp * BF] – [kWEcon * DCEcon * Hours]

Without Fan Control Installed

ΔkWh = [HP * kWhCond] – [kWEcon * DCEcon * Hours]

Where:

HP = Horsepower of Compressor

= actual installed

kWhCond = Condensing unit savings, per hp. (value from savings table) [647]

| | Hermetic / Semi-Hermetic | Scroll | Discus |
|---|---|---|---|
| kWh/HP | 1,256 | 1,108 | 1,051 |

Hours = Number of annual hours that economizer operates [648].

| Region (city) | Hours |
|---|---|
| 1 (Rockford) | 2,376 |
| 2 (Chicago/O'Hare) | 1,968 |
| 3 (Springfield) | 1,728 |
| 4 (Belleview) | 1,488 |
| 5 (Marion) | 1,224 |

DCComp = Duty cycle of the compressor

= 50% [649]

[647] Savings table uses Economizer Calc.xls. Assume 5HP compressor size used to develop kWh/Hp value. No floating head pressure controls and compressor is located outdoors

[648] In the source TRM (VT) this value was 2,996 hrs based on 38° F cooler setpoint, Burlington VT weather data, and 5 degree economizer deadband. The IL numbers were calculated by using weather bin data for each location (number of hours < 38F at each location is the Hours value).

kWEvap = Connected load kW of each evaporator fan,

= If known, actual installed. Otherwise assume 0.123 kW[650]

kWCirc = Connected load kW of the circulating fan

= If known, actual installed. Otherwise assume 0.035 kW[651]

nFans = Number of evaporator fans

= actual number of evaporator fans

DCEcon = Duty cycle of the economizer fan on days that are cool enough for the economizer to be working

= If known, actual installed. Otherwise assume 63%[652]

BF = Bonus factor for reduced cooling load from running the evaporator fan less or (1.3)[653]

kWEcon = Connected load kW of the economizer fan

= If known, actual installed. Otherwise assume 0.227 kW.[654]

**EXAMPLE**

For example, adding an outdoor air economizer and fan controls in Rockford to a 5 hp walk in refrigeration unit with 3 evaporator fans would save:

ΔkWh = [HP * kWhCond] + [((kWEvap * nFans) – kWCirc) * Hours * DCComp * BF] – [kWEcon * DCEcon * Hours]

= [5 * 1256] + [((0.123 * 3) – 0.035) * 2376 *0.5 * 1.3] – [0.227 * 0.63 * 2376]

= 6456 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$$\Delta kW = \Delta kWh / Hours$$

**NATURAL GAS SAVINGS**

N/A

[649] A 50% duty cycle is assumed based on examination of duty cycle assumptions from Richard Travers (35%-65%), Cooltrol (35%-65%), Natural Cool (70%), Pacific Gas & Electric (58%). Also, manufacturers typically size equipment with a built-in 67% duty factor and contractors typically add another 25% safety factor, which results in a 50% overall duty factor. (as referenced by the Efficiency Vermont, Technical Reference User Manual)

[650] Based on an a weighted average of 80% shaded pole motors at 132 watts and 20% PSC motors at 88 watts

[651] Wattage of fan used by Freeaire and Cooltrol. This fan is used to circulate air in the cooler when the evaporator fan is turned off. As such, it is not used when fan control is not present

[652] Average of two manufacturer estimates of 50% and 75%.

[653] Bonus factor (1+ 1/3.5) assumes COP of 3.5, based on the average of standard reciprocating and discus compressor efficiencies with a Saturated Suction Temperature of 20°F and a condensing temperature of 90°F

[654] The 227 watts for an economizer is calculated from the average of three manufacturers: Freeaire (186 Watts), Cooltrol (285 Watts), and Natural Cool (218 Watts).

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-RFG-ECON-V05-150601**

## 4.6.9 Night Covers for Open Refrigerated Display Cases

### DESCRIPTION

This measure is the installation of fitted covers on existing open-type refrigerated and freezer display cases that are deployed during the facility unoccupied hours. Night covers are designed to reduce refrigeration energy consumption by reducing the work done by the compressor. Night covers reduce the heat and moisture entry into the refrigerated space through various heat transfer mechanisms. By fully or partially covering the case opening, night covers reduce the convective heat transfer into the case through reduced air infiltration. Additionally, they provide a measure of insulation, reducing conduction into the case, and also decrease radiation into the case by blocking radiated heat from entering the refrigerated space.

### DEFINITION OF EFFICIENT EQUIPMENT

Curtains or covers on top of open refrigerated or freezer display cases that are applied at least six hours (during off-hours) in a 24-hour period.

### DEFINITION OF BASELINE EQUIPMENT

Refrigerated and freezer, open-type display case in vertical, semi-vertical, and horizontal displays, with no night cover.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is 5 years, based on DEER 2014.[655]

### DEEMED MEASURE COST

The incremental capital cost for this measure is $42 per linear foot of cover installed including material and labor.[656]

### LOADSHAPE

Loadshape 22: Commercial Refrigeration

### COINCIDENCE FACTOR

N/A – savings occur at night only.

[655] 2014 Database for Energy-Efficiency Resources (DEER), Version 2014, "Cost Values and Summary Documentation", California Public Utilities Commission, January, 2014.
[656] 2014 Database for Energy-Efficiency Resources (DEER), Version 2014, "Cost Values and Summary Documentation", California Public Utilities Commission, January, 2014.

**Algorithm**

**CALCULATION OF ENERGY SAVINGS**

**ELECTRIC ENERGY SAVINGS**

$$\Delta kWh = ES * L$$

Where:

ES = the energy savings (ΔkWh/ft) found in table below:

| Display Case Description | Case Temperature Range (°F) | Annual Electricity Use kWh/ft[657] | ES ΔkWh/ft reduction (= 9% reduction of electricity use[658,659]) |
|---|---|---|---|
| Vertical Open, Remote Condensing, Medium Temperature | 35°F to 55°F | 1453 | 131 |
| Vertical Open, Remote Condensing, Low Temperature | 0°F to 30°F | 3292 | 296 |
| Vertical Open, Self-Contained Medium Temperature | 35°F to 55°F | 2800 | 252 |
| Horizontal Open, Remote Condensing, Medium Temperature | 35°F to 55°F | 439 | 40 |
| Horizontal Open, Remote Condensing, Low Temperature | 0°F to 30°F | 1007 | 91 |
| Horizontal Open, Self-Contained, Medium Temperature | 35°F to 55°F | 1350 | 121 |
| Horizontal Open, Self-Contained, Low Temperature | 0°F to 30°F | 2749 | 247 |

L = the length of the refrigerated case in linear feet

[657] Energy Conservation Standards for Commercial Refrigeration Equipment: Technical Support Document, U.S. Department of Energy, September 2013. The information required to estimate annual energy savings for refrigerated display cases is taken from the 2013-2014 U.S. Department of Energy (DOE) energy conservation standard rulemaking for Commercial Refrigerated Equipment. During the rulemaking process, DOE estimates the energy savings specific to night covers through extensive simulation and energy models that are validated by both manufacturers of night covers and refrigerated cases. The information is also referenced from a study done by Southern California Edison and testing by Technischer Uberwachungs-Verein Rheinland, which are used by DOE for the rulemaking process.

[658] Southern California Edison Refrigeration Technology and Test Center. Effects of the Low Emissivity Shields on Performance and Power Use of a Refrigerated Display Case. 1997. Southern California Edison, Rancho Cucamonga, CA.

[659] Technischer Uberwachungs-Verein Rheinland E.V. Laboratory test results for energy savings on refrigerated dairy case, conducted for Econofrost.

= Actual

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

Peak savings are null because savings occur at night only.

**NATURAL GAS SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-RFG-NCOV-V01-150601**

# 4.7 Miscellaneous End Use

## 4.7.1 VSD Air Compressor

### DESCRIPTION

This measure relates to the installation of an air compressor with a variable frequency drive, load/no load controls or variable displacement control. The baseline compressors defined choke off the inlet air to modulate the compressor output, which is not efficient. Efficient compressors use a variable speed drive on the motor to match output to the load. Savings are calculated using representative baseline and efficient demand numbers for compressor capacities according to the facility's load shape, and the number of hours the compressor runs at that capacity. Demand curves are as per DOE data for a Variable Speed compressor versus a Modulating compressor. This measure applies only to an individual compressor ≤ 40 hp

This measure was developed to be applicable to the following program types: TOS.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The high efficiency equipment is a compressor ≤ 40 hp with variable speed control.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is a modulating compressor with blow down ≤ 40 hp

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

10 years.

### DEEMED MEASURE COST

$$\text{IncrementalCost (\$)} = (127 \times hp_{compressor}) + 1446$$

Where:

127 and 1446[660] = compressor motor nominal hp to incremental cost conversion factor and offset

$hp_{compressor}$ = compressor motor nominal

### DEEMED O&M COST ADJUSTMENTS

N/A

### LOADSHAPE

Loadshape C35 - Industrial Process

### COINCIDENCE FACTOR

The coincidence factor equals 0.95

[660] Conversion factor and offset based on a linear regression analysis of the relationship between air compressor motor nominal horsepower and incremental cost. Several Vermont vendors were surveyed to determine the cost of equipment. See "Compressed Air Analysis.xls" and "Compiled Data ReQuest Results.xls" for incremental cost details.

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = 0.9 \times hp_{compressor} \times HOURS \times (CF_b - CF_e)$$

Where:

$\Delta kWh$ = gross customer annual kWh savings for the measure

$hp_{compressor}$ = compressor motor nominal hp

0.9[661] = compressor motor nominal hp to full load kW conversion factor

HOURS = compressor total hours of operation below depending on shift

| Shift | Hours |
|---|---|
| Single shift (8/5) | 1976 hours<br>7 AM – 3 PM, weekdays, minus some holidays and scheduled down time |
| 2-shift (16/5) | 3952 hours<br>7AM – 11 PM, weekdays, minus some holidays and scheduled down time |
| 3-shift (24/5) | 5928 hours<br>24 hours per day, weekdays, minus some holidays and scheduled down time |
| 4-shift (24/7) | 8320 hours<br>24 hours per day, 7 days a week minus some holidays and scheduled down time |

$CF_b$ = baseline compressor factor[662]

=0.890

$CF_e$ = efficient compressor [663]

=0.705

[661] Conversion factor based on a linear regression analysis of the relationship between air compressor motor nominal horsepower and full load kW from power measurements of 72 compressors at 50 facilities on Long Island. See "BHP Weighted Compressed Air Load Profiles v2.xls".

[662] Compressor factors were developed using DOE part load data for different compressor control types as well as load profiles from 50 facilities employing air compressors less than or equal to 40 hp. "See "BHP Weighted Compressed Air Load Profiles.xls" for source data and calculations (The "variable speed drive" compressor factor has been adjusted up from the 0.675 presented in the analysis to 0.705 to account for the additional power draw of the VSD).

[663] Ibid.

**EXAMPLE**

For example a VFD compressor with 10 HP operating in a 1 shift facility would save

$\Delta$kWh = 0.9 x 10 x 1976 x (0.890 – 0.705)

= 3290 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$\Delta$kW = $\Delta$kWh / HOURS * CF

**EXAMPLE**

For example a VFD compressor with 10 HP operating in a 1 shift facility would save

$\Delta$kW = 3290/1976*.95

= 1.58 kW

**NATURAL GAS ENERGY SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-MSC-VSDA-V01-120601**

## 4.7.2 Compressed Air Low Pressure Drop Filters

### DESCRIPTION

Low pressure drop filters remove solids and aerosols from compressed air systems with a longer life and lower pressure drop than standard coalescing filters, resulting in better efficiencies.

This measure was developed to be applicable to the following program types: RF. If applied to other program types, the measure savings should be verified

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is a low pressure drop filter with pressure drop not exceeding 1 psid when new and 3 psid at element change.

### DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a standard coalescing filter with a pressure drop of 3 psid when new and 5 or more at element change

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

5 years

### DEEMED MEASURE COST

The incremental cost for this measure is estimated to be $1000 Incremental cost per filter[664]

### LOADSHAPE

Loadshape C35 - Industrial Process

### COINCIDENCE FACTOR

The coincidence factor equals 0.95

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = (kW_{typical} \times \Delta P \times SF \times Hours / HP_{typical}) \times HP_{real}$$

Where:

$kW_{typical}$ = Adjusted compressor power (kW) based on typical compressor loading and operating profile. Use Use actual compressor control type if known:

[664] Incremental cost research found in LPDF Costs. xlsx

Compressor $kW_{typical}$

| Control Type | $kW_{typical}$[665] |
|---|---|
| Reciprocating - On/off Control | 70.2 |
| Reciprocating - Load/Unload | 74.8 |
| Screw - Load/Unload | 82.3 |
| Screw - Inlet Modulation | 82.5 |
| Screw - Inlet Modulation w/ Unloading | 82.5 |
| Screw - Variable Displacement | 73.2 |
| Screw - VFD | 70.8 |

= If the actual compressor control type is not known, then use a weighted average based on the following market assumptions:

| Control Type | Share % | $kW_{typical}$[666] |
|---|---|---|
| Market share estimation for load/unload control compressors | 40% | 74.8 |
| Market share estimation for modulation w/unloading control compressors | 40% | 82.5 |
| Market share estimation for variable displacement control compressors | 20% | 73.2 |
| | Weighted Average | 77.6 |

ΔP = Reduced filter loss (psi)

=2 psi[667]

SF =1% reduction in power per 2 psi reduction in system pressure is equal to 0.5% reduction per 1 psi, or a Savings Factor of 0.005[668]

Hours = depending on shifts

Single shift (8/5) – 1976 hours (7 AM – 3 PM, weekdays, minus some holidays and scheduled down time) + 500 hrs maintenance = 2476 hrs

2-shift (16/5) – 3952 hours (7AM – 11 PM, weekdays, minus some holidays and scheduled down time) + 500 hrs maintenance = 4452 hrs

[665] See "Industrial System Standard Deemed Saving Analysis.xls"
[666] See "Industrial System Standard Deemed Saving Analysis.xls"
[667] Assumed pressure will be reduced from a roughly 3 psi pressure drop through a filter to less than 1 psi, for a 2 psi savings
[668] "Optimizing pneumatic systems for extra savings," 10, 2010, http://www.compressedairchallenge.org/library/articles/2010-10-CABP.pdf

3-shift (24/5) – 5928 hours (24 hours per day, weekdays, minus some holidays and scheduled down time) + 500 hrs maintenance = 6428 hrs

4-shift (24/7) – 8320 hours (24 hours per day, 7 days a week minus some holidays and scheduled down time)

$HP_{typical}$ = Nominal HP for typical compressor = 100 hp[669]

$HP_{real}$ = Total HP of real compressors distibuting air through filter. This should include the total horsepower of the compressors that normally run through the filter, but not backup compressors

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$\Delta kW = \Delta kWh / HOURS * CF$

**NATURAL GAS ENERGY SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-MSC-CALPDF-V01-140601**

[669] Industrial System Standard Deemed Saving Analysis.xls

## 4.7.3 Compressed Air No-Loss Condensate Drains

### DESCRIPTION

No-loss condensate drains remove condensate as needed without venting compressed air, resulting in less air demand and consequently better efficiency. Replacement or upgrades of existing no-loss drains are not eligible for the incentive.

This measure was developed to be applicable to the following program types: RF. If applied to other program types, the measure savings should be verified

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is installation of no-loss condensate drains.

### DEFINITION OF BASELINE EQUIPMENT

The baseline condition is installation of standard condensate drains (open valve, timer, or both)

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

10 years

### DEEMED MEASURE COST

$700 per drain [670]

### LOADSHAPE

Loadshape C35 - Industrial Process

### COINCIDENCE FACTOR

The coincidence factor equals 0.95

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = CFM_{reduced} \times kW_{CFM} \times Hours$$

Where:

$CFM_{reduced}$ = Reduced air consumption (CFM) per drain

= 3 CFM[671]

$kW_{CFM}$ = System power reduction per reduced air demand (kw/CFM) depending on the type of

[670] Based on empirical project data from ComEd Comprehensive Compressed Air Study program and VEIC review of pricing data found in CAS Cost Data.xls

[671] Reduced CFM consumption is based on an a timer drain opening for 10 seconds every 300 seconds as the baseline. See "Industrial System Standard Deemed Saving Analysis.xls"

compressor control:

System Power Reduction per Reduced Air Demand[672]

| Control Type | kW / CFM |
|---|---|
| Reciprocating - On/off Control | 0.184 |
| Reciprocating - Load/Unload | 0.136 |
| Screw - Load/Unload | 0.152 |
| Screw - Inlet Modulation | 0.055 |
| Screw - Inlet Modulation w/ Unloading | 0.055 |
| Screw - Variable Displacement | 0.153 |
| Screw - VFD | 0.178 |

Or if compressor control type is unknow, then a weighted average based on market share can be used:

| Control Type | Share % | kW / CFM |
|---|---|---|
| Market share estimation for load/unload control compressors | 40% | 0.136 |
| Market share estimation for modulation w/unloading control compressors | 40% | 0.055 |
| Market share estimation for variable displacement control compressors | 20% | 0.153 |
| | Weighted Average | 0.107 |

Hours = Compressed air system pressurized hours

=6136 hours[673]

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$\Delta$kW = $\Delta$kWh / HOURS * CF

**NATURAL GAS ENERGY SAVINGS**

N/A

[672] Calculated based on the type of compressor control. This assumes the compressor will be between 40% and 100% capacity before and after the changes to the system demand. See "Industrial System Standard Deemed Saving Analysis.xls"

[673] US DOE, Evaluation of the Compressed Air Challenge® Training Program, Page 19

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-MSC-CANLCD-V01-140601**

## 4.7.4 Pump Optimization

### DESCRIPTION

Pump improvements can be done to optimize the design and control of centrifugal water pumping systems, including water solutions with freeze protection up to 15% concentration by volume. Other fluid and gas pumps cannot use this measure calculation. The measurement of energy and demand savings for commercial and industrial applications will vary with the type of pumping technology, operating hours, efficiency, and existing and proposed controls. Depending on the specific application slowing the pump, trimming or replacing the impeller may be suitable options for improving pumping efficiency. Pumps up to 40 HP are allowed to use this energy savings calculation. Larger motors should use a custom calculation (which may result in larger savings that this measure would claim).

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is proven to be an optimized centrifugal pumping system meeting the applicable program efficiency requirements:

- Pump balancing valves no more than 15% throttled
- Balancing valves on at least one load 100% open.

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to be the existing pumping system including existing controls and sequence of operations.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years[674]

### DEEMED MEASURE COST

The incremental capital cost for this measure can vary considerably depending upon the strategy employed to achieve the required efficiency levels and should be determined on a site-specific basis.

### DEEMED O&M COST ADJUSTMENTS

N/A

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 38%[675]

[674] Martin, N. et al., Emerging Energy-Efficient Industrial Technologies: New York State Edition, American Council for an Energy Efficient Economy (ACEEE), March 2001 (as stated in the OH State TRM, page 269)

[675] Summer Peak Coincidence Factor has been preserved from the "Technical Reference Manual" (TRM) for Ohio Senate Bill 221 Energy Efficiency and Conservation Program and 09-512-GE-UNC," October 15, 2009. This is likely a conservative estimate, but is recommended for further study (as stated in the OH State TRM, page 269)

**Algorithm**

**CALCULATION OF SAVINGS**

**ELECTRIC ENERGY SAVINGS**

$$\Delta kWh = (HP_{motor} * 0.746 * LF / \eta_{motor}) * HOURS * ESF$$

Where:

$HP_{motor}$ = Installed nameplate motor horsepower

= Actual

0.746 = Conversion factor from horse-power to kW (kW/hp)

$LF / \eta_{motor}$ = Combined as a single factor since efficiency is a function of load

= 0.65 [676]

Where:

LF = Load Factor; Ratio of the peak running load to the nameplate rating of the motor

$\eta_{motor}$ = Motor efficiency at pump operating conditions

HOURS = Annual operating hours of the pump

= Actual

ESF = Energy Savings Factor; assume a value of 15%[677].

[676] "Measured Loading of Energy Efficient Motors - the Missing Link in Engineering Estimates of Savings," ACEEE 1994 Summer Study Conference, Asilomar, CA.

[677] Published estimates of typical pumping efficiency improvements range from 5 to 40%. For analysis purposes, assume 15%. United States Industrial Electric Motor Systems Market Opportunities Assessment December 2002, Table E-7, Page 18, https://www1.eere.energy.gov/manufacturing/tech_assistance/pdfs/mtrmkt.pdf

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = (HP_{motor} * 0.746 *( LF / \eta_{motor})) * (ESF) * CF$$

Where:

CF = Summer Coincident Peak Factor for measure

= 0.38[678]

## NATURAL GAS ENERGY SAVINGS

N/A

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: CI-MSC-PMPO-V01-150601

[678] Summer Peak Coincidence Factor has been preserved from the "Technical Reference Manual" (TRM) for Ohio Senate Bill 221 Energy Efficiency and Conservation Program and 09-512-GE-UNC," October 15, 2009. This is likely a conservative estimate, but is recommended for further study (as stated in the OH State TRM, page 269)

## 4.7.5 Efficient Compressed Air Nozzles

### DESCRIPTION

This measure is for the replacement of standard air nozzle with high-efficiency air nozzle used in a compressed air system. High-efficiency air nozzles reduce the amount of air required to blow off parts or for drying. These nozzles utilize the Coandă effect to pull in free air to accomplish tasks with significantly less compressed air. High-efficiency nozzles often replace simple copper tubes. These nozzles have the added benefits of noise reduction and improved safety in systems with greater than 30 psig.

### DEFINITION OF EFFICIENT EQUIPMENT

The high-efficiency air nozzle must meet the following specifications:

1. High-efficiency air nozzle must replace continuous open blow-offs
2. High-efficiency air nozzle must meet SCFM rating at 80psig less than or equal to: 1/8” 11 SCFM, 1/4" 29 SCFM, 5/16” 56 SCFM, 1/2" 140 SCFM.
3. Manufacturer’s specification sheet of the high-efficiency air nozzle must be provided along with the make and model

### DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a standard air nozzle

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is 15 years[679]

### DEEMED MEASURE COST

The estimated incremental measure costs are presented in the following table[680]

| Nozzle Diameter | 1/8” | 1/4" | 5/16” | 1/2" |
|---|---|---|---|---|
| Average IMC | $42 | $57 | $87 | $121 |

### LOADSHAPE

Loadshape C35 - Industrial Process

### COINCIDENCE FACTOR

The coincidence factor equals 0.95

[679] PA Consulting Group (2009). Business Programs: Measure Life Study. Prepared for State of Wisconsin Public Service Commission.

[680] Costs are from EXAIR’s website and are an average of nozzles that meet the flow requirements. Models include Atto Super, Pico Super, Nano Super, Micro Super, Mini Super, Super and Large Super nozzles. www.exair.com. Accessed March 20, 2014

**Algorithm**

**CALCULATION OF ENERGY SAVINGS**

**ELECTRIC ENERGY SAVINGS**

ΔkWh = (SCFM * SCFM%Reduced) * kW/CFM * %USE * HOURS

Where:

SCFM = Air flow through standard nozzle. Use actual rated flow at 80 psi if known. If unknown, the table below includes the CFM by orifice diameter[681, 682].

| Orifice Diameter | SCFM |
|---|---|
| 1/8” | 21 |
| 1/4” | 58 |
| 5/16” | 113 |
| 1/2" | 280 |

SCFM%Reduced = Percent in reduction of air loss per nozzle. Estimated at 50%[683]

kW/CFM = System power reduction per air demand (kW/CFM) depending on the type of air compressor found in table below[684]

| Air Compressor Type | ΔkW/CFM |
|---|---|
| Reciprocating – On/off Control | 0.18 |
| Reciprocating – Load/Unload | 0.14 |
| Screw – Load/Unload | 0.15 |
| Screw – Inlet Modulation | 0.06 |
| Screw – Inlet Modulation w/ Unloading | 0.06 |
| Screw – Variable Displacement | 0.15 |
| Screw - VFD | 0.18 |

%USE = percent of the compressor total operating hours that the nozzle is in use

[681] Review of manufacturer’s information
[682] Technical Reference Manual (TRM) for Ohio Senate Bill 221”Energy Efficiency and Conservation Program” and 09-512-GE-UNC, October 15,
2009. Pgs 170-171
[683] Conservative estimate based on average values provided by the Compressed Air Challenge Training Program, Machinery’s Handbook 25th
Edition, and manufacturers’ catalog.
[684] Calculated based on the type of compressor control. This assumes the compressor will be between 40% and 100% capacity before and after the changes to the system demand. See “Industrial System Standard Deemed Saving Analysis.xls”

= Custom, if unknown assume 5%[685]

Hours = Compressed air system pressurized hours.

= Use actual hours if known, otherwise assume values in table below:

| Shift | Hours |
|---|---|
| Single Shift | 1976 |
| Two Shifts | 3952 |
| Three Shifts | 5928 |
| Four Shifts or Continual Operation | 8320 |
| Unknown / Weighted average[686] | 5702 |

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$$\Delta kW = \Delta kWh/HOURS * CF$$

Where:

ΔkWh = As calculated above

CF = 0.95

**NATURAL GAS SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-MSC-CNOZ-V01-150601**

[685] Assumes 50% handheld air guns and 50% stationary air nozzles. Manual air guns tend to be used less than stationary air nozzles, and a conservative estimate of 1 second of blow-off per minute of compressor run time is assumed. Stationary air nozzles are commonly more wasteful as they are often mounted on machine tools and can be manually operated resulting in the possibility of a long term open blow situation. An assumption of 5 seconds of blow-off per minute of compressor run time is used.

[686] Weighting of 16% single shift, 23% two shift, 25% three shift and 36% continual based on DOE evaluation of the Compressed Air Challenge, section 2.1.5 Facility Operating Schedules

## 4.7.6 Roof Insulation for C&I Facilities

### DESCRIPTION

Energy and demand saving are realized through reductions in the building cooling and heating loads. This measure was developed to be applicable to the following program types: RF and NC.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is above code and should be determined by the program. An example definition could be ASHRAE 90.1 – 2013 (see tables below):

| | IL TRM Zones 1, 2, & 3 [ASHRAE/IECC Climate Zone 5 (A, B, C)] | | | |
|---|---|---|---|---|
| | Nonresidential | | Semiheated | |
| | Assembly Maximum | Insulation Min. R-Value | Assembly Maximum | Insulation Min. R-Value |
| Insulation Entirely Above Deck | 0.032 | R-30.0 c.i. | 0.063 | R-15 c.i. |
| Metal Building (Roof) | 0.037 | R-19 + R-11 Ls or R-25 + R-8 Ls | 0.082 | R-19 |
| Attic and Other | 0.021 | R-49 | 0.034 | R-30 |

| | IL TRM Zones 4 & 5 [ASHRAE/IECC Climate Zone 4 (A, B, C)] | | | |
|---|---|---|---|---|
| | Nonresidential | | Semiheated | |
| | Assembly Maximum | Insulation Min. R-Value | Assembly Maximum | Insulation Min. R-Value |
| Insulation Entirely Above Deck | 0.032 | R-30.0 c.i. | 0.093 | R-10 c.i. |
| Metal Building (Roof) | 0.037 | R-19 + R-11 Ls or R-25 + R-8 Ls | 0.082 | R-19 |
| Attic and Other | 0.021 | R-49 | 0.034 | R-30 |

Table Notes

c.i. = continuous insulation

Ls = linear system, a continuous vapor barrier liner installed below the purlins and uninterrupted by framing members

### DEFINITION OF BASELINE EQUIPMENT

The retrofit baseline condition is adopted from Ohio Energy Technical Reference Manual and expanded to cover all type of commercial buildings in the state of Illinois as follows.

For retrofits, the R-value for the entire assembly:

| Building Type | Retrofit Assembly R-Value |
|---|---|
| Assembly | 13.5 |
| Assisted Living | 13.5 |
| College | 13.5 |
| Convenience Store | 13.5 |
| Elementary School | 13.5 |
| Garage | 13.5 |
| Grocery | 13.5 |
| Healthcare Clinic | 13.5 |
| High School | 13.5 |
| Hospital | 13.5 |
| Hotel/Motel | 13.5 |
| Manufacturing Facility | 12 |
| MF - High Rise | 13.5 |
| MF - Mid Rise | 13.5 |
| Movie Theater | 13.5 |
| Office - High Rise | 13.5 |
| Office - Low Rise | 13.5 |
| Office - Mid Rise | 13.5 |
| Religious Building | 13.5 |
| Restaurant | 13.5 |
| Retail - Department Store | 13.5 |
| Retail - Strip Mall | 13.5 |
| Warehouse | 12 |
| Unknown | 13.5 |

For new construction use IECC 2012 or ASHRAE – 90.1 – 2010 (listed below)

| | IL TRM Zones 1, 2, & 3 [ASHRAE/IECC Climate Zone 5 (A, B, C)] | | | |
|---|---|---|---|---|
| | Nonresidential | | Semiheated | |
| | Assembly Maximum | Assembly Maximum | Assembly Maximum | Assembly Maximum |
| Insulation Entirely Above Deck | 0.048 | R-20 c.i. | U-0.119 | R-7.6 c.i. |
| Metal Building (Roof) | 0.055 | R-13.0 + R-13.0 | U-0.083 | R-13.0 |
| Attic and Other | 0.027 | R-38.0 | U-0.053 | R-19.0 |

| | IL TRM Zones 4 & 5 [ASHRAE/IECC Climate Zone 4 (A, B, C)] | | | |
|---|---|---|---|---|
| | Nonresidential | | Semiheated | |
| | Assembly Maximum | Insulation Min. R-Value | Assembly Maximum | Insulation Min. R-Value |
| Insulation Entirely Above Deck | 0.048 | R-20.0 c.i. | 0.173 | R-5.0 c.i. |
| Metal Building (Roof) | 0.055 | R-13.0 + R-13.0 | 0.097 | R-10.0 |
| Attic and Other | 0.027 | R-38.0 | 0.053 | R-19.0 |

Table Notes

c.i. = continuous insulation

Ls = linear system, a continuous vapor barrier liner installed below the purlins and uninterrupted by framing members

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure expected useful life (EUL) is assumed to be 20 years per DEER 2008. This is consistent with SDG&E's 9th Year Measure Retrofit Study (1996 & 1997 Residential Weatherization Programs), CPUC's Energy Efficiency Policy Manual v.2, and GDS's Measure Life Report Residential and Commercial/Industrial Lighting and HVAC Measures (June 2007).

### DEEMED MEASURE COST

Per the 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, "Cost Values and Summary Documentation", the material cost for R-30 insulation is $0.75 per square foot. The installation cost is $0.61 per square foot. The total measure cost, therefore, is $1.36 per square foot of insulation installed. However, the actual cost should be used when available.

### LOADSHAPE

Loadshape C03: Commercial Cooling

### COINCIDENCE FACTOR

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)

= 91.3% [687]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)

= 47.8% [688]

**Algorithm**

### CALCULATION OF ENERGY SAVINGS

### ELECTRIC ENERGY SAVINGS

Electric energy savings is calculated as the sum of energy saved when cooling the building and energy saved when heating the building

$$\Delta kWh = \Delta kWh_cooling + \Delta kWh_heating$$

If central cooling, the electric energy saved in annual cooling due to the added insulation is

$$\Delta kWh_cooling = ((1/R_existing) - (1/R_new)) * Area * EFLH_{cooling} * \Delta T_{AVG,cooling} / 1{,}000 / \eta_cooling$$

Where:

R_existing = Roof heat loss coefficient with existing insulation [(hr-$^{o}$F-ft$^{2}$)/Btu]

R_new = Roof heat loss coefficienty with new insulation [(hr-$^{o}$F-ft$^{2}$)/Btu]

Area = Area of the roof surface in square feet. Assume 1000 sq ft for planning.

$EFLH_{cooling}$ = Equivalent Full Load Hours for Cooling [hr] are provided in Section 4.4, HVAC end use

$\Delta T_{AVG,cooling}$ = Average temperature difference [$^{o}$F] during cooling season between outdoor air temperature and assumed 75$^{o}$F indoor air temperature

[687] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

[688] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

| Climate Zone (City based upon) | $OA_{AVG,cooling}$ [°F][689] | $\Delta T_{AVG,cooling}$ [°F] |
|---|---|---|
| 1 (Rockford) | 81 | 6 |
| 2 (Chicago) | 81 | 6 |
| 3 (Springfield) | 81 | 6 |
| 4 (Belleville) | 82 | 7 |
| 5 (Marion) | 82 | 7 |

1,000 = Conversion from Btu to kBtu

η_cooling = Seasonal energy efficiency ratio (SEER) of cooling system (kBtu/kWh). Use actual if possible, if unknown and for planning purposes assume the following:

| Year Equipment was Installed | SEER estimate |
|---|---|
| Before 2006 | 10 |
| After 2006 | 13 |

If the building is heated with electric heat (resistance or heat pump), the electric energy saved in annual heating due to the added insulation is

$$\Delta kWh_heating = [(1/R_existing) - (1/R_new)] * Area * EFLH_{heating} * \Delta T_{AVG,heating} / 3{,}412 / \eta_heating$$

Where:

$EFLH_{heating}$ = Equivalent Full Load Hours for Heating [hr] are provided in Section 4.4, HVAC end use

$\Delta T_{AVG,heating}$ = Average temperature difference [°F] during heating season between outdoor air temperature and assumed 55°F heating base temperature

| Climate Zone (City based upon) | $OA_{AVG,heating}$ [°F][690] | $\Delta T_{AVG,heating}$ [°F] |
|---|---|---|
| 1 (Rockford) | 32 | 23 |
| 2 (Chicago) | 34 | 21 |
| 3 (Springfield) | 35 | 20 |

[689] National Solar Radiation Data Base -- 1991- 2005 Update: Typical Meteorological Year 3 http://rredc.nrel.gov/solar/old_data/nsrdb/1991-2005/tmy3/by_state_and_city.html
[690] National Solar Radiation Data Base -- 1991- 2005 Update: Typical Meteorological Year 3 http://rredc.nrel.gov/solar/old_data/nsrdb/1991-2005/tmy3/by_state_and_city.html

| Climate Zone (City based upon) | $OA_{AVG,heating}$ [°F][690] | $\Delta T_{AVG,heating}$ [°F] |
|---|---|---|
| 4 (Belleville) | 36 | 19 |
| 5 (Marion) | 39 | 16 |

3,142 = Conversion from Btu to kWh.

η_heating = Efficiency of heating system. Use actual efficiency. If not available refer to default table below.

| System Type | Age of Equipment | HSPF Estimate | ηHeat (Effective COP Estimate) (HSPF/3.413)*0.85 |
|---|---|---|---|
| Heat Pump | Before 2006 | 6.8 | 1.7 |
| | After 2006 | 7.7 | 1.92 |
| Resistance | N/A | N/A | 1 |

If the building is heated with a gas furnace, there will be some electric savings in heating the building attributed to extra insulation since the furnace fans will run less.

ΔkWh_heating = ΔTherms * Fe * 29.3

Where:

ΔTherms = Gas savings calculated with equation below.

Fe = Percentage of heating energy consumed by fans, assume 3.14%

29.3 = Conversion from therms to kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = (ΔkWh_cooling / EFLH_cooling) * CF

Where:

$EFLH_{cooling}$ = Equivalent full load hours of air conditioning are provided in Section 4.4, HVAC end use

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)

= 91.3% [691]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)

= 47.8% [692]

**NATURAL GAS SAVINGS**

If building uses a gas furnace, the savings resulting from the insulation is calculated with the following formula.

$$\Delta Therms = ((1/R_existing) - (1/R_new)) * Area * EFLH_{heating} * \Delta T_{AVG,heating} / 100{,}000 / \eta_heat$$

Where:

R_existing = Roof heat loss coefficient with existing insulation [(hr-$^{o}$F-ft$^{2}$)/Btu]

R_new = Roof heat loss coefficienty with new insulation [(hr-$^{o}$F-ft$^{2}$)/Btu]

Area = Area of the roof surface in square feet. Assume 1000 sq ft for planning.

$EFLH_{heating}$ = Equvalent Full Load Hours for Heating are provided in Section 4.4, HVAC end use

$\Delta T_{AVG,heating}$ = Average temperature difference [$^{o}$F] during heating season (see above)

100,000 = Conversion from BTUs to Therms

η_heat = Efficiency of existing furnace. Assume 0.78 for planning purposes.

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: CI-HVC-RINS-V01-150601**

[691] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

[692] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

## 4.7.7 Computer Power Management Software

### DESCRIPTION

Computer power management software is installed on a network of computers. This is software which monitors and records computer and monitor usage, as well as allows centralized control of computer power management settings.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is defined by the requirements listed below:

- Allow centralized control and override of computer power management settings of workstations which include both a computer monitor and CPU (i.e. a desktop or laptop computer on a distributed network)
- Be able to control on/off/sleep states on both the CPU and monitor according to the Network Administrator-defined schedules and apply power management policies to network groups
- Have capability to allow networked workstations to be remotely wakened from power-saving mode (e.g. for system maintenance or power/setting adjustments)
- Have capability to detect and monitor power management performance and generate energy savings reports
- Have capability to produce system reports to confirm the inventory and performance of equipment on which the software is installed.

This measure was developed to be applicable to the following program types: Retrofit. If applied to other program types, the measure savings should be verified.

### DEFINITION OF BASELINE EQUIPMENT

Baseline is defined as a computer network without software enforcing the power management capabilities in existing computers and monitors.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is five years.[693]

### DEEMED MEASURE COST

The deemed measure cost is $29 per networked computer, including labor.[694]

### LOADSHAPE

Loadshape C21: Commercial Office Equipment.

[693] The following reference uses 10 years, however, given the rapid changes in the technology industry, there is quite a lot of uncertainty about the measure life and a more conservative value was used (i.e. half the published measure life): Table VI.1: Dimetrosky, S., Luedtke, J. S., & Seiden, K. (2005). Surveyor Network Energy Manager: Market Progress Evaluation Report, No. 2 (Northwest Energy Efficiency Alliance report #E05-136). Portland, OR: Quantec LLC;).

[694] Work Paper WPSCNROE0003 Revision 1, Power Management Software for Networked Computers. Southern California Edison

### COINCIDENCE FACTOR

NA

**Algorithm**

### CALCULATION OF ENERGY SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = Wsavings * W$$

Where:

Wsavings = annual energy savings per workstation

= 200 kWh[695] for desktops, 50 kWh for laptops[696]

= If unknown assume 161 kWh (based on 74% desktop and 26% laptop[697])

W = number of desktop or laptop workstations controlled by the power management software

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

NA

### NATURAL GAS SAVING

NA

### WATER IMPACT DESCRIPTIONS AND CALCULATION

NA

### DEEMED O&M COST ADJUSTMENT CALCULATION

Assumed to be $2/unit [698]

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[695] Based on average energy savings/computer from the following sources:
South California Edison, Work Paper WPSCNROE0003 (200k Wh)
Surveyor Network Energy Manager Evaluation Report , NEEA (68, 100, and 128kWh)
Regional Technical Forum http://rtf.nwcouncil.org/measures/measure.asp?id=95 (200 kWh)
EnergySTAR Computer Power Management Savings Calculator (~190 kWh for a mix of laptop/desktop and assuming 30% are already turned off at night)
http://www.energystar.gov/ia/products/power_mgt/LowCarbonITSavingsCalc.xlsx?78c1-120e&78c1-120e
Power Management for Networked Computers: A Review of Utility Incentive Programs J. Michael Walker, Beacon Consultants Network Inc., 2009 ACEEE Summer Study on Energy Efficiency in Industry (330 kWh)

[696] Power Management for Networked Computers: A Review of Utility Incentive Programs J. Michael Walker, Beacon Consultants Network Inc., 2009 ACEEE Summer Study on Energy Efficiency in Industry

[697] Based on PY6 ComEd Computer Software Program data showing a split of 74% desktop to 26% laptop.

[698] Based on Dimetrosky, S., Luedtke, J. S., & Seiden, K. (2005). Surveyor Network Energy Manager: Market Progress Evaluation

**MEASURE CODE: CI-MSC-CPMS-V01-150601**

Report, No. 2 (Northwest Energy Efficiency Alliance report #E05-136). Portland, OR: Quantec LLC and review of CLEARResult document providing Qualifying Software Providers for ComEd program and their licensing fees; "Qualifying Vendor Software Comparison.pdf".

# 5 Residential Measures

## 5.1 Appliances End Use

### 5.1.1 ENERGY STAR Air Purifier/Cleaner

#### DESCRIPTION

An air purifier (cleaner) meeting the efficiency specifications of ENERGY STAR is purchased and installed in place of a model meeting the current federal standard.

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

#### DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is defined as an air purifier meeting the efficiency specifications of ENERGY STAR as provided below.

- Must produce a minimum 50 Clean Air Delivery Rate (CADR) for Dust[699] to be considered under this specification.
- Minimum Performance Requirement: = 2.0 CADR/Watt (Dust)
- Standby Power Requirement: = 2.0 Watts Qualifying models that perform secondary consumer functions (e.g. clock, remote control) must meet the standby power requirement.
- UL Safety Requirement: Models that emit ozone as a byproduct of air cleaning must meet UL Standard 867 (ozone production must not exceed 50ppb)

#### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be a conventional unit[700].

#### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 9 years[701].

#### DEEMED MEASURE COST

The incremental cost for this measure is $70.[702]

#### LOADSHAPE

Loadshape C53 - Flat

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[699] Measured according to the latest ANSI/AHAM AC-1 (AC-1) Standard
[700] As defined as the average of non-ENERGY STAR products found in EPA research, 2008, ENERGY STAR Qualified Room Air Cleaner Calculator, http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/CalculatorRoomAirCleaner.xls?8ed7-275b.
[701] ENERGY STAR Qualified Room Air Cleaner Calculator, http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/CalculatorRoomAirCleaner.xls?8ed7-275b.
[702] Ibid

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 100 % (the unit is assumed to be always on).

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = kWh_{Base} - kWh_{ESTAR}$$

Where:

$kWh_{BASE}$ = Baseline kWh consumption per year[703]

= see table below

$kWh_{ESTAR}$ = ENERGY STAR kWh consumption per year[704]

= see table below

| Clean Air Delivery Rate | Baseline Unit Energy Consumption (kWh/year) | ENERGY STAR Unit Energy Consumption (kWh/year) | ΔkWH |
|---|---|---|---|
| CADR 51-100 | 596 | 329 | 268 |
| CADR 101-150 | 1,072 | 548 | 525 |
| CADR 151-200 | 1,480 | 767 | 714 |
| CADR 201-250 | 1,887 | 986 | 902 |
| CADR Over 250 | 1,641 | 1205 | 437 |

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh/Hours * CF$$

Where:

ΔkWh = Gross customer annual kWh savings for the measure

Hours = Average hours of use per year

= 8766 hours[705]

[703] ENERGY STAR Qualified Room Air Cleaner Calculator, http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/CalculatorRoomAirCleaner.xls?8ed7-275b
[704] Ibid.
[705] Consistent with ENERGY STAR Qualified Room Air Cleaner Calculator.

CF = Summer Peak Coincidence Factor for measure

= 1.0

| Clean Air Delivery Rate | ΔkW |
|---|---|
| CADR 51-100 | 0.031 |
| CADR 101-150 | 0.060 |
| CADR 151-200 | 0.081 |
| CADR 201-250 | 0.103 |
| CADR Over 250 | 0.050 |

**NATURAL GAS SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

There are no operation and maintenance cost adjustments for this measure.[706]

**MEASURE CODE: RS-APL-ESAP-V01-120601**

[706] Some types of room air cleaners require filter replacement or periodic cleaning, but this is likely to be true for both efficient and baseline units and so no difference in cost is assumed.

## 5.1.2 ENERGY STAR and ENERGY STAR Most Efficient Clothes Washers

### DESCRIPTION

This measure relates to the installation of a clothes washer meeting the ENERGY STAR, or ENERGY STAR Most Efficient minimum qualifications. Note if the DHW and dryer fuels of the installations are unknown (for example through a retail program) savings should be based on a weighted blend using RECS data (the resultant values (kWh, therms and gallons of water) are provided). The algorithms can also be used to calculate site specific savings where DHW and dryer fuels are known.

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

Clothes washer must meet the ENERGY STAR or ENERGY STAR Most Efficient minimum qualifications, as required by the program.

### DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a standard sized clothes washer meeting the minimum federal baseline as of March 2015[707].

| Efficiency Level | Top loading >2.5 Cu ft | Front Loading >2.5 Cu ft |
|---|---|---|
| Federal Standard | 1.29 IMEF, 8.4 IWF | 1.84 IMEF, 4.7 IWF |
| ENERGY STAR | 2.06 IMEF, 4.3 IWF | 2.38 IMEF, 3.7 IWF |
| ENERGY STAR Most Efficient | 2.76 IMEF, 3.5 IWF | 2.74 IMEF, 3.2IWF |

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 14 years[708].

### DEEMED MEASURE COST

The incremental cost for an ENERGY STAR unit is assumed to be $65 and for an ENERGY STAR Most Efficient unit it is $210[709].

[707] See http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/39.
[708] Based on DOE Life-Cycle Cost and Payback Period Excel-based analytical tool, available online at: http://www1.eere.energy.gov/buildings/appliance_standards/residential/clothes_washers_support_stakeholder_negotiations.html
[709] Cost estimates are based on Navigant analysis for the Department of Energy (see CW Analysis_09092014.xls). This analysis looked at incremental cost and shipment data from manufacturers and the Association of Home Appliance Manufacturers and

### DEEMED O&M COST ADJUSTMENTS

N/A

### LOADSHAPE

Loadshape R01 - Residential Clothes Washer

### COINCIDENCE FACTOR

The coincidence factor for this measure is 3.8%[710].

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

1. Calculate clothes washer savings based on Modified Energy Factor (MEF).

   The Modified Energy Factor (MEF) includes unit operation, water heating and drying energy use: *"MEF is the quotient of the capacity of the clothes container, C, divided by the total clothes washer energy consumption per cycle, with such energy consumption expressed as the sum of the machine electrical energy consumption, M, the hot water energy consumption, E, and the energy required for removal of the remaining moisture in the wash load, D"* [711].

   The hot water and dryer savings calculated here assumes electric DHW and Dryer (this will be separated in Step 2).

IMEFsavings[712] = Capacity * (1/IMEFbase - 1/IMEFeff) * Ncycles

Where

Capacity = Clothes Washer capacity (cubic feet)

= Actual. If capacity is unknown assume 3.45 cubic feet [713]

IMEFbase = Integrated Modified Energy Factor of baseline unit

= 1.66[714]

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attempts to find the costs associated only with the efficiency improvements. The ENERGY STAR level in this analysis was made the baseline (as it is now equivalent), the CEE Tier 3 level was made ENERGY STAR and ENERGY STAR Most efficient was extrapolated based on equal rates. Note these assumptions should be reviewed as qualifying product becomes available.

[710] Calculated from Itron eShapes, 8760 hourly data by end use for Missouri, as provided by Ameren.

[711] Definition provided on the Energy star website.

[712] Tsavings represents total kWh only when water heating and drying are 100% electric.

[713] Based on the average clothes washer volume of all units that pass the new Federal Standard on the California Energy Commission (CEC) database of Clothes Washer products accessed on 08/28/2014. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.

[714] Weighted average IMEF of Federal Standard rating for Front Loading and Top Loading units. Weighting is based upon the relative top v front loading percentage of available non-ENERGY STAR product in the CEC database.

IMEFeff = Integrated Modified Energy Factor of efficient unit

= Actual. If unknown assume average values provided below.

Ncycles = Number of Cycles per year

= 295[715]

IMEFsavings is provided below based on deemed values[716]:

| Efficiency Level | IMEF | IMEFSavings (kWh) |
|---|---|---|
| Federal Standard | 1.66 | 0.0 |
| Energy StarENERGY STAR | 2.26 | 163 |
| ENERGY STAR Most Efficient | 2.74 | 242 |

2. Break out savings calculated in Step 1 for electric DHW and electric dryer

ΔkWh = [(Capacity * 1/IMEFbase * Ncycles) * (%CWbase + (%DHWbase * %Electric_DHW) + (%Dryerbase * %Electric_Dryer)] - [(Capacity * 1/IMEFeff * Ncycles) * (%CWeff + (%DHWeff * %Electric_DHW) + (%Dryereff * %Electric_Dryer)]

Where:

%CW = Percentage of total energy consumption for Clothes Washer operation (different for baseline and efficient unit – see table below)

%DHW = Percentage of total energy consumption used for water heating (different for baseline and efficient unit – see table below)

%Dryer = Percentage of total energy consumption for dryer operation (different for baseline and efficient unit – see table below)

| Percentage of Total Energy Consumption717 |
|---|

[715] Weighted average of 295 clothes washer cycles per year (based on 2009 Residential Energy Consumption Survey (RECS) national sample survey of housing appliances section, state of IL: http://www.eia.gov/consumption/residential/data/2009/
If utilities have specific evaluation results providing a more appropriate assumption for single-family or multi-family homes, in a particular market, or geographical area then that should be used.

[716] IMEF values are the weighted average of the new ENERGY STAR specifications. Weighting is based upon the relative top v front loading percentage of available ENERGY STAR and ENERGY STAR Most Efficient product in the CEC database. See "CW Analysis_09092014.xls" for the calculation.

[717] The percentage of total energy consumption that is used for the machine, heating the hot water or by the dryer is different

| | %CW | %DHW | %Dryer |
|---|---|---|---|
| Baseline | 8% | 31% | 61% |
| ENERGY STAR | 8% | 23% | 69% |
| ENERGY STAR Most Efficient | 14% | 10% | 76% |

%Electric_DHW = Percentage of DHW savings assumed to be electric

| DHW fuel | %Electric_DHW |
|---|---|
| Electric | 100% |
| Natural Gas | 0% |
| Unknown | 16%[718] |

%Electric_Dryer = Percentage of dryer savings assumed to be electric

| Dryer fuel | %Electric_DHW |
|---|---|
| Electric | 100% |
| Natural Gas | 0% |
| Unknown | 27%[719] |

Using the default assumptions provided above, the prescriptive savings for each configuration are presented below:

| | ΔkWH | | | |
|---|---|---|---|---|
| | Electric DHW Electric Dryer | Gas DHW Electric Dryer | Electric DHW Gas Dryer | Gas DHW Gas Dryer |
| ENERGY STAR | 162.7 | 77.0 | 96.0 | 10.2 |
| ENERGY STAR Most Efficient | 242.1 | 88.2 | 149.9 | -4.0 |

depending on the efficiency of the unit. Values are based on a weighted average of top loading and front loading units based on data from Life-Cycle Cost and Payback Period Excel-based analytical tool, available online at: http://www1.eere.energy.gov/buildings/appliance_standards/residential/clothes_washers_support_stakeholder_negotiations.html. See "CW Analysis_09092014.xls" for the calculation.

[718] Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

[719] Default assumption for unknown is based on percentage of homes with electric dryer from EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.

If the DHW and dryer fuel is unknown the prescriptive kWH savings based on defaults provided above should be:

| | ΔkWH |
|---|---|
| ENERGY STAR | 42.0 |
| ENERGY STAR Most Efficient | 45.5 |

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ΔkWh/Hours * CF

Where:

ΔkWh = Energy Savings as calculated above

Hours = Assumed Run hours of Clothes Washer

= 295 hours[720]

CF = Summer Peak Coincidence Factor for measure.

= 0.038[721]

Using the default assumptions provided above, the prescriptive savings for each configuration are presented below:

| | ΔkW | | | |
|---|---|---|---|---|
| | Electric DHW Electric Dryer | Gas DHW Electric Dryer | Electric DHW Gas Dryer | Gas DHW Gas Dryer |
| ENERGY STAR | 0.021 | 0.010 | 0.012 | 0.0013 |
| ENERGY STAR Most Efficient | 0.031 | 0.011 | 0.019 | -0.001 |

If the DHW and dryer fuel is unknown the prescriptive kW savings should be:

| ΔkW |
|---|

[720] Based on a weighted average of 295 clothes washer cycles per year assuming an average load runs for one hour (2009 Residential Energy Consumption Survey (RECS) national sample survey of housing appliances section: http://www.eia.gov/consumption/residential/data/2009/)

[721] Calculated from Itron eShapes, 8760 hourly data by end use for Missouri, as provided by Ameren.

| | ΔkW |
|---|---|
| ENERGY STAR | 0.005 |
| ENERGY STAR Most Efficient | 0.006 |

### NATURAL GAS SAVINGS

Break out savings calculated in Step 1 of electric energy savings (MEF savings) and extract Natural Gas DHW and Natural Gas dryer savings from total savings:

ΔTherm = [(Capacity * 1/IMEFbase * Ncycles) * ((%DHWbase * %Natural Gas_DHW * R_eff) + (%Dryerbase * %Gas _Dryer)] - [(Capacity * 1/IMEFeff * Ncycles) * ((%DHWeff * %Natural Gas_DHW * R_eff) + (%Dryereff * %Gas_Dryer)] * Therm_convert

Where:

Therm_convert = Convertion factor from kWh to Therm

= 0.03413

R_eff = Recovery efficiency factor

= 1.26[722]

%Natural Gas_DHW = Percentage of DHW savings assumed to be Natural Gas

| DHW fuel | %Natural Gas_DHW |
|---|---|
| Electric | 0% |
| Natural Gas | 100% |
| Unknown | 84%[723] |

%Gas_Dryer = Percentage of dryer savings assumed to be Natural Gas

| Dryer fuel | %Gas_Dryer |
|---|---|
| Electric | 100% |
| Natural Gas | 0% |

[722] To account for the different efficiency of electric and Natural Gas hot water heaters (gas water heater: recovery efficiencies ranging from 0.74 to 0.85 (0.78 used), and electric water heater with 0.98 recovery efficiency (http://www.energystar.gov/ia/partners/bldrs_lenders_raters/downloads/Waste_Water_Heat_Recovery_Guidelines.pdf ). Therefore a factor of 0.98/0.78 (1.26) is applied.

[723] Default assumption for unknown fuel is based on percentage of homes with gas dryer from EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

| Dryer fuel | %Gas_Dryer |
|---|---|
| Unknown | 44%[724] |

Other factors as defined above

Using the default assumptions provided above, the prescriptive savings for each configuration are presented below:

| | ΔTherms | | | |
|---|---|---|---|---|
| | Electric DHW Electric Dryer | Gas DHW Electric Dryer | Electric DHW Gas Dryer | Gas DHW Gas Dryer |
| ENERGY STAR | 0.00 | 3.7 | 2.3 | 6.0 |
| ENERGY STAR Most Efficient | 0.00 | 6.6 | 3.1 | 9.8 |

If the DHW and dryer fuel is unknown the prescriptive Therm savings should be:

| | ΔTherms |
|---|---|
| ENERGY STAR | 4.10 |
| ENERGY STAR Most Efficient | 6.94 |

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

ΔWater (gallons) = (Capacity * (IWFbase - IWFeff)) * Ncycles

Where

IWFbase = Integrated Water Factor of baseline clothes washer

= 5.92[725]

IWFeff = Water Factor of efficient clothes washer

= Actual. If unknown assume average values provided below.

Using the default assumptions provided above, the prescriptive water savings for each efficiency level are presented below:

[724] Ibid.
[725] Weighted average IWF of Federal Standard rating for Front Loading and Top Loading units. Weighting is based upon the relative top v front loading percentage of available non-ENERGY STAR product in the CEC database.

| Efficiency Level | IWF[726] | ΔWater (gallons per year) |
|---|---|---|
| Federal Standard | 5.92 | 0.0 |
| ENERGY STAR | 3.93 | 2024 |
| ENERGY STAR Most Efficient | 3.21 | 2760 |

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-APL-ESCL-V03-150601**

[726] IWF values are the weighted average of the new ENERGY STAR specifications. Weighting is based upon the relative top v front loading percentage of available ENERGY STAR and ENERGY STAR Most Efficient product in the CEC database. See "CW Analysis_09092014.xls" for the calculation.

## 5.1.3 ENERGY STAR Dehumidifier

### DESCRIPTION

A dehumidifier meeting the minimum qualifying efficiency standard established by the current ENERGY STAR (Version 2.1 or 3.0)[727] is purchased and installed in a residential setting in place of a unit that meets the minimum federal standard efficiency.

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure, the new dehumidifier must meet the ENERGY STAR standards as defined below:

Until 9/30/2012:

| Capacity (pints/day) | ENERGY STAR Criteria (L/kWh) |
|---|---|
| ≤25 | ≥1.20 |
| > 25 to ≤35 | ≥1.40 |
| > 35 to ≤45 | ≥1.50 |
| > 45 to ≤ 54 | ≥1.60 |
| > 54 to ≤ 75 | ≥1.80 |
| > 75 to ≤ 185 | ≥2.50 |

After 10/1/2012[728]:

| Capacity (pints/day) | ENERGY STAR Criteria (L/kWh) |
|---|---|
| <75 | ≥1.85 |
| 75 to ≤185 | ≥2.80 |

Qualifying units shall be equipped with an adjustable humidistat control or shall require a remote humidistat control to operate.

### DEFINITION OF BASELINE EQUIPMENT

The baseline for this measure is defined as a new dehumidifier that meets the Federal Standard efficiency standards. The Federal Standard for Dehumidifiers changed as of October 2012 as defined below:

[727] Energy Star Version 3.0 will become effective 10/1/12

[728] http://www.energystar.gov/ia/partners/prod_development/revisions/downloads/dehumid/ES_Dehumidifiers_Final_V3.0_Eligibility_Criteria.pdf?d70c-99b0

Until 9/30/2012:

| Capacity (pints/day) | Federal Standard Criteria (L/kWh) |
|---|---|
| ≤25 | ≥1.0 |
| > 25 to ≤35 | ≥1.20 |
| > 35 to ≤45 | ≥1.30 |
| > 45 to ≤ 54 | ≥1.30 |
| > 54 to ≤ 75 | ≥1.50 |
| > 75 to ≤ 185 | ≥2.25 |

Post 10/1/2013

| Capacity (pints/day) | Federal Standard Criteria (L/kWh) [729] |
|---|---|
| Up to 35 | ≥1.35 |
| > 35 to ≤45 | ≥1.50 |
| > 45 to ≤ 54 | ≥1.60 |
| > 54 to ≤ 75 | ≥1.70 |
| > 75 to ≤ 185 | ≥2.50 |

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of the measure is 12 years[730].

### DEEMED MEASURE COST

The assumed incremental capital cost for this measure is $40 for units purchased prior to 10/1/2012 and $60 for units purchased after 10/1/2012[731].

[729] The Federal Standard for Dehumidifiers changed as of October 2012; https://www.federalregister.gov/articles/2010/12/02/2010-29756/energy-conservation-program-for-consumer-products-test-procedures-for-residential-dishwashers#h-11

[730] ENERGY STAR Dehumidifier Calculator http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/CalculatorConsumerDehumidifier.xls

[731] Based on extrapolating available data from the Department of Energy's Life Cycle Cost analysis spreadsheet and weighting based on volume of units available: http://www1.eere.energy.gov/buildings/appliance_standards/residential/docs/lcc_dehumidifier.xls See 'DOE life cycle cost_dehumidifier.xls' for calculation.

### LOADSHAPE

Loadshape R12 - Residential - Dehumidifier

### COINCIDENCE FACTOR

The coincidence factor is assumed to be 37% [732].

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = (((Avg\ Capacity * 0.473) / 24) * Hours) * (1 / (L/kWh_Base) - 1 / (L/kWh_Eff))$$

Where:

Avg Capacity = Average capacity of the unit (pints/day)

0.473 = Constant to convert Pints to Liters

24 = Constant to convert Liters/day to Liters/hour

Hours = Run hours per year

= 1620 [733]

L/kWh = Liters of water per kWh consumed, as provided in tables above

Annual kWh results for each capacity class are presented below:

Until 9/30/2012 (V 2.1):

| | | | | Annual kWh | | |
|---|---|---|---|---|---|---|
| Capacity (pints/day) Range | Capacity Used | Federal Standard Criteria (≥ L/kWh) | ENERGY STAR Criteria (≥ L/kWh) | Federal Standard | ENERGY STAR | Savings |
| ≤25 | 20 | 1.0 | 1.2 | 643 | 536 | 107 |
| > 25 to ≤35 | 30 | 1.2 | 1.4 | 804 | 689 | 115 |
| > 35 to ≤45 | 40 | 1.3 | 1.5 | 990 | 858 | 132 |
| > 45 to ≤ 54 | 50 | 1.3 | 1.6 | 1237 | 1005 | 232 |
| > 54 to ≤ 75 | 65 | 1.5 | 1.8 | 1394 | 1161 | 232 |

[732] Assume usage is evenly distributed day vs. night, weekend vs. weekday and is used between April through the end of September (4392 possible hours). 1620 operating hours from ENERGY STAR Dehumidifier Calculator. Coincidence peak during summer peak is therefore 1620/4392 = 36.9%

[733] ENERGY STAR Dehumidifier Calculator http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/CalculatorConsumerDehumidifier.xls

| Capacity (pints/day) Range | Capacity Used | Federal Standard Criteria (≥ L/kWh) | ENERGY STAR Criteria (≥ L/kWh) | Annual kWh Federal Standard | Annual kWh ENERGY STAR | Annual kWh Savings |
|---|---|---|---|---|---|---|
| > 75 to ≤ 185 | 130 | 2.25 | 2.5 | 1858 | 1673 | 186 |
| Average | 46 | 1.31 | 1.55 | 1129 | 953 | 176 |

After 10/1/2012 (V 3.0):

| Capacity (pints/day) Range | Capacity Used | Federal Standard Criteria (≥ L/kWh) | ENERGY STAR Criteria (≥ L/kWh) | Annual kWh Federal Standard | Annual kWh ENERGY STAR | Annual kWh Savings |
|---|---|---|---|---|---|---|
| ≤25 | 20 | 1.35 | 1.85 | 477 | 348 | 129 |
| > 25 to ≤35 | 30 | 1.35 | 1.85 | 715 | 522 | 193 |
| > 35 to ≤45 | 40 | 1.5 | 1.85 | 858 | 695 | 162 |
| > 45 to ≤ 54 | 50 | 1.6 | 1.85 | 1005 | 869 | 136 |
| > 54 to ≤ 75 | 65 | 1.7 | 1.85 | 1230 | 1130 | 100 |
| > 75 to ≤ 185 | 130 | 2.5 | 2.8 | 1673 | 1493 | 179 |
| Average | 46 | 1.51 | 1.85 | 983 | 800 | 183 |

**Summer Coincident Peak Demand Savings**

$$\Delta kW = \Delta kWh/Hours * CF$$

Where:

Hours = Annual operating hours

= 1632 hours [734]

CF = Summer Peak Coincidence Factor for measure

= 0.37 [735]

Summer coincident peak demand results for each capacity class are presented below:

[734] Based on 68 days of 24 hour operation; ENERGY STAR Dehumidifier Calculator http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/appliance_calculator.xlsx?f3f7-6a8b&f3f7-6a8b

[735] Assume usage is evenly distributed day vs. night, weekend vs. weekday and is used between April through the end of September (4392 possible hours). 1620 operating hours from ENERGY STAR Dehumidifier Calculator. Coincidence peak during summer peak is therefore 1620/4392 = 36.9%

Until 9/30/2012 (V 2.1):

| Capacity (pints/day) Range | Annual Summer peak kW Savings |
|---|---|
| ≤25 | 0.024 |
| > 25 to ≤35 | 0.026 |
| > 35 to ≤45 | 0.030 |
| > 45 to ≤ 54 | 0.053 |
| > 54 to ≤ 75 | 0.053 |
| > 75 to ≤ 185 | 0.042 |
| Average | 0.040 |

After 10/1/2012 (V 3.0):

| Capacity (pints/day) Range | Annual Summer peak kW Savings |
|---|---|
| ≤25 | 0.029 |
| > 25 to ≤35 | 0.044 |
| > 35 to ≤45 | 0.037 |
| > 45 to ≤ 54 | 0.031 |
| > 54 to ≤ 75 | 0.023 |
| > 75 to ≤ 185 | 0.041 |
| Average | 0.042 |

**NATURAL GAS SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-APL-ESDH-V02-130601**

## 5.1.4 ENERGY STAR Dishwasher

### DESCRIPTION

A dishwasher meeting the efficiency specifications of ENERGY STAR is installed in place of a model meeting the federal standard. This measure is only for standard dishwashers, not compact dishwashers. A compact dishwasher is a unit that holds less than eight place settings with six serving pieces.

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is defined as a dishwasher meeting the efficiency specifications of ENERGY STAR (for standard dishwashers[736]). The Energy Star standard is presented in the table below:

| Dishwasher Type | Maximum kWh/year | Maximum gallons/cycle |
|---|---|---|
| Standard | 295 | 4.25 |

### DEFINITION OF BASELINE EQUIPMENT

The Baseline reflects the minimum federal efficiency standards for dishwashers effective May 30, 2013, as presented in the table below[737].

| Dishwasher Type | Maximum kWh/year | Maximum gallons/cycle |
|---|---|---|
| Standard | 307 | 5.0 |

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of the measure is 13 years[738].

### DEEMED MEASURE COST

The incremental cost for this measure is $50[739].

### LOADSHAPE

Loadshape R02 - Residential Dish Washer

### COINCIDENCE FACTOR

The coincidence factor is assumed to be 2.6%[740].

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[736] As of May 30, 2013 the Federal Baseline specification for compact units is equal to the ENERGY STAR specification.
[737] http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/67
[738] Koomey, Jonathan et al. (Lawrence Berkeley National Lab), Projected Regional Impacts of Appliance Efficiency Standards for the U.S. Residential Sector, February 1998.
[739] Estimate based on review of Energy Star stakeholder documents

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh^{741} = ((kWh_{Base} - kWh_{ESTAR}) * (\%kWh_op + (\%kWh_heat * \%Electric_DHW)))$$

Where:

$kWh_{BASE}$ = Baseline kWh consumption per year

= 307 kWh

$kWh_{ESTAR}$ = ENERGY STAR kWh annual consumption

= 295 kWh

%kWh_op = Percentage of dishwasher energy consumption used for unit operation

= 1 - 56%[742]

= 44%

%kWh_heat = Percentage of dishwasher energy consumption used for water heating

= 56%[743]

%Electric_DHW = Percentage of DHW savings assumed to be electric

| DHW fuel | %Electric_DHW |
|---|---|
| Electric | 100% |
| Natural Gas | 0% |
| Unknown | 16%[744] |

An Energy Star standard dishwasher installed in place of a baseline unit with unknown DHW fuel:

ΔkWh = ((307 - 295) * (0.44 + (0.56*0.16)))

= 6.4 kWh

An Energy Star standard dishwasher installed in place of a baseline unit with electric DHW:

ΔkWh = ((307 - 295) * (0.44 + (0.56*1.0)))

= 12 kWh

[740] Calculated from Itron eShapes, 8760 hourly data by end use for Missouri, as provided by Ameren.
[741] The Federal Standard and ENERGY STAR annual consumption values include electric consumption for both the operation of the machine and for heating the water that is used by the machine.
[742] ENERGY STAR Dishwasher Calculator (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/CalculatorConsumerDishwasher.xls)
[743] Ibid.
[744] Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.

**Summer Coincident Peak Demand Savings**

$$\Delta kW = \Delta kWh/Hours * CF$$

Where:

Hours = Annual operating hours[745]

= 252 hours

CF = Summer Peak Coincidence Factor

= 2.6% [746]

An Energy Star standard dishwasher installed in place of a baseline unit with unknown DHW fuel:

$\Delta kWh$ = 6.4/252 * 0.026

= 0.0007 kW

An Energy Star standard dishwasher installed in place of a baseline unit with electric DHW:

$\Delta kWh$ = 12/252 * 0.026

= 0.001 kWh

**NATURAL GAS SAVINGS**

$$\Delta \text{ Therm} = (kWh_{Base} - kWh_{ESTAR}) * \%kWh_heat * \%Natural\ Gas_DHW * R_eff * 0.03413$$

Where

%kWh_heat = % of dishwasher energy used for water heating

= 56%

%Natural Gas_DHW = Percentage of DHW savings assumed to be Natural Gas

| DHW fuel | %Natural Gas_DHW |
|---|---|
| Electric | 0% |
| Natural Gas | 100% |
| Unknown | 84%[747] |

R_eff = Recovery efficiency factor

= 1.26[748]

[745] Assuming one and a half hours per cycle and 168 cycles per year therefore 252 operating hours per year; 168 cycles per year is based on a weighted average of dishwasher usage in Illinois derived from the 2009 RECs data; http://205.254.135.7/consumption/residential/data/2009/

[746] End use data from Ameren representing the average DW load during peak hours/peak load.

[747] Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.

[748] To account for the different efficiency of electric and Natural Gas hot water heaters (gas water heater: recovery efficiencies ranging from 0.74 to 0.85 (0.78 used), and electric water heater with 0.98 recovery efficiency (http://www.energystar.gov/ia/partners/bldrs_lenders_raters/downloads/Waste_Water_Heat_Recovery_Guidelines.pdf).

0.03413 = factor to convert from kWh to Therm

An Energy Star standard dishwasher installed in place of a baseline unit with unknown DHW fuel:

Δ Therm = (307 - 295) * 0.56 * 0.84* 1.26 * 0.03413

= 0.24 therm

An Energy Star standard dishwasher installed in place of a baseline unit with gas DHW:

Δ Therm = (307 - 295) * 0.56 * 1.0* 1.26 * 0.03413

= 0.29 Therm

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

$\Delta Water = Water_{Base} - Water_{EFF}$

Where

$Water_{Base}$ = water consumption of conventional unit

= 840 gallons[749]

$Water_{EFF}$ = annual water consumption of efficient unit:

= 714 gallons[750]

Δ Water = 840 – 714

= 126 gallons

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-APL-ESDI-V02-130601**

Therefore a factor of 0.98/0.78 (1.26) is applied.

[749] Assuming 5 gallons/cycle (maximum allowed) and 168 cycles per year based on a weighted average of dishwasher usage in Illinois derived from the 2009 RECs data; http://205.254.135.7/consumption/residential/data/2009/

[750] Assuming 4.25gallons/cycle (maximum allowed) and 168 cycles per year based on a weighted average of dishwasher usage in Illinois derived from the 2009 RECs data; http://205.254.135.7/consumption/residential/data/2009/

## 5.1.5 ENERGY STAR Freezer

### DESCRIPTION

A freezer meeting the efficiency specifications of ENERGY STAR is installed in place of a model meeting the federal standard (NAECA). Energy usage specifications are defined in the table below (note, AV is the freezer Adjusted Volume and is calculated as 1.73*Total Volume):

| Product Category | Volume (cubic feet) | Assumptions up to September 2014 | | Assumptions after September 2014 | |
|---|---|---|---|---|---|
| | | Federal Baseline Maximum Energy Usage in kWh/year[751] | ENERGY STAR Maximum Energy Usage in kWh/year[752] | Federal Baseline Maximum Energy Usage in kWh/year[753] | ENERGY STAR Maximum Energy Usage in kWh/year[754] |
| Upright Freezers with Manual Defrost | 7.75 or greater | 7.55*AV+258.3 | 6.795*AV+232.47 | 5.57*AV + 193.7 | 5.01*AV + 174.3 |
| Upright Freezers with Automatic Defrost | 7.75 or greater | 12.43*AV+326.1 | 11.187*AV+293.49 | 8.62*AV + 228.3 | 7.76*AV + 205.5 |
| Chest Freezers and all other Freezers except Compact Freezers | 7.75 or greater | 9.88*AV+143.7 | 8.892*AV+129.33 | 7.29*AV + 107.8 | 6.56*AV + 97.0 |
| Compact Upright Freezers with Manual Defrost | < 7.75 and 36 inches or less in height | 9.78*AV+250.8 | 7.824*AV+200.64 | 8.65*AV + 225.7 | 7.79*AV + 203.1 |
| Compact Upright Freezers with Automatic Defrost | < 7.75 and 36 inches or less in height | 11.40*AV+391 | 9.12*AV+312.8 | 10.17*AV + 351.9 | 9.15*AV + 316.7 |
| Compact Chest Freezers | <7.75 and 36 inches or less in height | 10.45*AV+152 | 8.36*AV+121.6 | 9.25*AV + 136.8 | 8.33*AV + 123.1 |

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is defined as a freezer meeting the efficiency specifications of ENERGY STAR, as defined below and calculated above:

[751] http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/43
[752] http://www.energystar.gov/ia/products/appliances/refrig/NAECA_calculation.xls?c827-f746
[753] http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/43
[754] http://www.energystar.gov/products/specs/sites/products/files/ENERGY%20STAR%20Final%20Version%205.0%20Residential%20Refrigerators%20and%20Freezers%20Specification.pdf

| Equipment | Volume | Criteria |
|---|---|---|
| Full Size Freezer | 7.75 cubic feet or greater | At least 10% more energy efficient than the minimum federal government standard (NAECA). |
| Compact Freezer | Less than 7.75 cubic feet and 36 inches or less in height | At least 20% more energy efficient than the minimum federal government standard (NAECA). |

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be a model that meets the federal minimum standard for energy efficiency. The standard varies depending on the size and configuration of the freezer (chest freezer or upright freezer, automatic or manual defrost) and is defined in the table above.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 11 years[755].

### DEEMED MEASURE COST

The incremental cost for this measure is $35[756].

### LOADSHAPE

Loadshape R04 - Residential Freezer

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 95%[757].

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS:

$$\Delta kWh = kWh_{Base} - kWh_{ESTAR}$$

Where:

$kWh_{BASE}$ = Baseline kWh consumption per year as calculated in algorithm provided in table above.

$kWh_{ESTAR}$ = ENERGY STAR kWh consumption per year as calculated in algorithm provided in table

[755] Energy Star Freezer Calculator; http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Consumer_Residential_Freezer_Sav_Calc.xls?570a-f000

[756] Based on review of data from the Northeast Regional ENERGY STAR Consumer Products Initiative; "2009 ENERGY STAR Appliances Practices Report", submitted by Lockheed Martin, December 2009.

[757] Based on eShapes Residential Freezer load data as provided by Ameren.

above.

For example for a 7.75 cubic foot Upright Freezers with Manual Defrost purchased after September 2014:

ΔkWh =(5.57*(7.75* 1.73)+193.7) – (5.01*(7.75* 1.73)+174.3)

= 268.4 – 241.5

= 26.9 kWh

If volume is unknown, use the following default values:

| Product Category | Volume Used[758] | Assumptions up to September 2014 | | | Assumptions after September 2014 | | |
|---|---|---|---|---|---|---|---|
| | | $kWh_{BASE}$ | $kWh_{ESTAR}$ | kWh Savings | $kWh_{BASE}$ | $kWh_{ESTAR}$ | kWh Savings |
| Upright Freezers with Manual Defrost | 27.9 | 469.1 | 422.2 | 46.9 | 349.2 | 314.2 | 35.0 |
| Upright Freezers with Automatic Defrost | 27.9 | 673.2 | 605.9 | 67.3 | 469.0 | 422.2 | 46.8 |
| Chest Freezers and all other Freezers except Compact Freezers | 27.9 | 419.6 | 377.6 | 42.0 | 311.4 | 280.2 | 31.2 |
| Compact Upright Freezers with Manual Defrost | 10.4 | 352.3 | 281.9 | 70.5 | 467.2 | 420.6 | 46.6 |
| Compact Upright Freezers with Automatic Defrost | 10.4 | 509.3 | 407.5 | 101.9 | 635.9 | 572.2 | 63.7 |
| Compact Chest Freezers | 10.4 | 260.5 | 208.4 | 52.1 | 395.1 | 355.7 | 39.4 |

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ΔkWh/ Hours * CF

Where:

ΔkWh = Gross customer annual kWh savings for the measure

Hours = Full Load hours per year

= 5890[759]

CF = Summer Peak Coincident Factor

= 0.95 [760]

[758] Volume is based on ENERGY STAR Calculator assumption of 16.14 $ft^3$ average volume, converted to Adjusted volume by multiplying by 1.73.

[759] Calculated from eShapes Residential Freezer load data as provided by Ameren by dividing total annual load by the maximum kW in any one hour.

[760] Based on eShapes Residential Freezer load data as provided by Ameren.

For example for a 7.75 cubic foot Upright Freezers with Manual Defrost:

$$\Delta kW = 26.9/5890 * 0.95$$

$$= 0.0043 \text{ kW}$$

If volume is unknown, use the following default values:

| Product Category | Assumptions up to September 2014 | Assumptions after September 2014 |
|---|---|---|
| | kW Savings | kW Savings |
| Upright Freezers with Manual Defrost | 0.0076 | 0.0057 |
| Upright Freezers with Automatic Defrost | 0.0109 | 0.0076 |
| Chest Freezers and all other Freezers except Compact Freezers | 0.0068 | 0.0050 |
| Compact Upright Freezers with Manual Defrost | 0.0114 | 0.0075 |
| Compact Upright Freezers with Automatic Defrost | 0.0164 | 0.0103 |
| Compact Chest Freezers | 0.0084 | 0.0064 |

**NATURAL GAS SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-APL-ESFR-V02-140601**

## 5.1.6 ENERGY STAR and CEE Tier 2 Refrigerator

### DESCRIPTION

This measure relates to:

a) Time of Sale: the purchase and installation of a new refrigerator meeting either ENERGY STAR or CEE TIER 2 specifications.

b) Early Replacement: the early removal of an existing residential inefficient Refrigerator from service, prior to its natural end of life, and replacement with a new ENERGY STAR or CEE Tier 2 qualifying unit. Savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life.

Energy usage specifications are defined in the table below (note, Adjusted Volume is calculated as the fresh volume + (1.63 * Freezer Volume):

| Product Category | Existing Unit | Assumptions up to September 2014 | | Assumptions after September 2014 | |
|---|---|---|---|---|---|
| | Based on Refrigerator Recycling algorithm | Federal Baseline Maximum Energy Usage in kWh/year[761] | ENERGY STAR Maximum Energy Usage in kWh/year[762] | Federal Baseline Maximum Energy Usage in kWh/year[763] | ENERGY STAR Maximum Energy Usage in kWh/year[764] |
| 1. Refrigerators and Refrigerator-freezers with manual defrost | Use Algorithm in 5.1.8 Refrigerator and Freezer Recycling measure to estimate existing unit consumption | 8.82*AV+248.4 | 7.056*AV+198.72 | 6.79AV + 193.6 | 6.11 * AV + 174.2 |
| 2. Refrigerator-Freezer--partial automatic defrost | | 8.82*AV+248.4 | 7.056*AV+198.72 | 7.99AV + 225.0 | 7.19 * AV + 202.5 |
| 3. Refrigerator-Freezers--automatic defrost with top-mounted freezer without through-the-door ice service and all-refrigerators--automatic defrost | | 9.80*AV+276 | 7.84*AV+220.8 | 8.07AV + 233.7 | 7.26 * AV + 210.3 |
| 4. Refrigerator-Freezers--automatic defrost with side- | | 4.91*AV+507.5 | 3.928*AV+406 | 8.51AV + 297.8 | 7.66 * AV + 268.0 |

[761] http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/43
[762] http://www.energystar.gov/ia/products/appliances/refrig/NAECA_calculation.xls?c827-f746
[763] http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/43
[764] http://www.energystar.gov/products/specs/sites/products/files/ENERGY%20STAR%20Final%20Version%205.0%20Residential%20Refrigerators%20and%20Freezers%20Specification.pdf

| Product Category | Existing Unit | Assumptions up to September 2014 | | Assumptions after September 2014 | |
|---|---|---|---|---|---|
| | Based on Refrigerator Recycling algorithm | Federal Baseline Maximum Energy Usage in kWh/year[761] | ENERGY STAR Maximum Energy Usage in kWh/year[762] | Federal Baseline Maximum Energy Usage in kWh/year[763] | ENERGY STAR Maximum Energy Usage in kWh/year[764] |
| mounted freezer without through-the-door ice service | | | | | |
| 5. Refrigerator-Freezers--automatic defrost with bottom-mounted freezer without through-the-door ice service | | 4.60*AV+459 | 3.68*AV+367.2 | 8.85AV + 317.0 | 7.97 * AV + 285.3 |
| 5A Refrigerator-freezer—automatic defrost with bottom-mounted freezer with through-the-door ice service | | N/A | N/A | 9.25AV + 475.4 | 8.33 * AV * 436.3 |
| 6. Refrigerator-Freezers--automatic defrost with top-mounted freezer with through-the-door ice service | | 10.20*AV+356 | 8.16*AV+284.8 | 8.40AV + 385.4 | 7.56 * AV + 355.3 |
| 7. Refrigerator-Freezers--automatic defrost with side-mounted freezer with through-the-door ice service | | 10.10*AV+406 | 8.08*AV+324.8 | 8.54AV + 432.8 | 7.69 * AV + 397.9 |

Note CEE Tier 2 standard criteria is 25% less consumption than a new baseline unit. It is assumed that after September 2014 when the Federal Standard and ENERGY STAR specifications change, the CEE Tier 2 will remain set at 25% less that the new baseline assumption.

This measure was developed to be applicable to the following program types: TOS, NC, EREP.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is defined as a refrigerator meeting the efficiency specifications of ENERGY STAR or CEE Tier 2 (defined as requiring >= 20% or >= 25% less energy consumption than an equivalent unit meeting federal standard requirements respectively). The ENERGY STAR standard varies according to the size and configuration of the unit, as shown in table above.

### DEFINITION OF BASELINE EQUIPMENT

Time of Sale: baseline is a new refrigerator meeting the minimum federal efficiency standard for refrigerator efficiency. The current federal minimum standard varies according to the size and configuration of the unit, as shown in table above. Note also that this federal standard will be increased for units manufactured after September 1, 2014.

Early Replacement: the baseline is the existing refrigerator for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 12 years.[765]

Remaining life of existing equipment is assumed to be 4 years[766]

### DEEMED MEASURE COST

Time of Sale: The incremental cost for this measure is assumed to be $40[767] for an ENERGY STAR unit and $140[768] for a CEE Tier 2 unit.

Early Replacement: The measure cost is the full cost of removing the existing unit and installing a new one. The actual program cost should be used. If unavailable assume $451 for ENERGY STAR unit and $551 for CEE Tier 2 unit[769].

The avoided replacement cost (after 4 years) of a baseline replacement refrigerator is $390[770].

### LOADSHAPE

Loadshape R05 - Residential Refrigerator

### COINCIDENCE FACTOR

A coincidence factor is not used to calculate peak demand savings for this measure, see below.

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[765] From ENERGY STAR calculator: http://www.energystar.gov/buildings/sites/default/uploads/files/appliance_calculator.xlsx?7224-046c=&7224-__046ceiling_fan_calculator_xlsx=&f7d8-39dd&f7d8-39dd

[766] Standard assumption of one third of effective useful life.

[767] From ENERGY STAR calculator linked above.

[768] Based on weighted average of units participating in Efficiency Vermont program and retail cost data provided in Department of Energy, "TECHNICAL REPORT: Analysis of Amended Energy Conservation Standards for Residential Refrigerator-Freezers", October 2005; http://www1.eere.energy.gov/buildings/appliance_standards/pdfs/refrigerator_report_1.pdf

[769] ENERGY STAR full cost is based upon IL PHA Efficient Living Program data on sample size of 910 replaced units finding average cost of $430 plus an average recycling/removal cost of $21. The CEE Tier 2 estimate uses the delta from the Time of Sale estimate.

[770] Calculated using incremental cost from Time of Sale measure.

**Algorithm**

**CALCULATION OF SAVINGS**

**ELECTRIC ENERGY SAVINGS:**

Time of Sale: $\Delta kWh = UEC_{BASE} - UEC_{EE}$

Early Replacement:

$\Delta kWh$ for remaining life of existing unit (1st 4 years) $= UEC_{EXIST} - UEC_{EE}$

$\Delta kWh$ for remaining measure life (next 8 years) $= UEC_{BASE} - UEC_{EE}$

Where:

$UEC_{EXIST}$ = Annual Unit Energy Consumption of existing unit as calculated in algorithm from 5.1.8 Refrigerator and Freezer Recycling measure.

$UEC_{BASE}$ = Annual Unit Energy Consumption of baseline unit as calculated in algorithm provided in table above.

$UEC_{EE}$ = Annual Unit Energy Consumption of ENERGY STAR unit as calculated in algorithm provided in table above.

For CEE Tier 2, unit consumption is calculated as 25% lower than baseline.

If volume is unknown, use the following defaults, based on an assumed Adjusted Volume of 25.8[771]:

Assumptions prior to standard changes on September 1st, 2014:

| Product Category | Existing Unit $UEC_{EXIST}$ [772] | New Baseline $UEC_{BASE}$ | New Efficient $UEC_{EE}$ | | Early Replacement (1st 4 years) ΔkWh | | Time of Sale and Early Replacement (last 8 years) ΔkWh | |
|---|---|---|---|---|---|---|---|---|
| | | | ENERGY STAR | CEE T2 | ENERGY STAR | CEE T2 | ENERGY STAR | CEE T2 |
| 1. Refrigerators and Refrigerator-freezers with manual defrost | 1027.7 | 475.7 | 380.5 | 356.8 | 647.2 | 671.0 | 95.1 | 118.9 |
| 2. Refrigerator-Freezer--partial automatic defrost | 1027.7 | 475.7 | 380.5 | 356.8 | 647.2 | 671.0 | 95.1 | 118.9 |
| 3. Refrigerator-Freezers--automatic defrost with top-mounted freezer without through-the-door ice service and | 814.5 | 528.5 | 422.8 | 396.4 | 391.7 | 418.1 | 105.7 | 132.1 |

[771] Volume is based on the ENERGY STAR calculator average assumption of 14.75 $ft^3$ fresh volume and 6.76 $ft^3$ freezer volume.
[772] Estimates of existing unit consumption are based on using the 5.1.8 Refrigerator and Freezer Recycling algorithm and the inputs described here: Age = 10 years, Pre-1990 = 0, Size = 21.5 ft3 (from ENERGY STAR calc and consistent with AV of 25.8), Single Door = 0, Side by side = 1 for classifications stating side by side, 0 for classifications stating top/bottom, and 0.5 for classifications that do not distinguish, Primary appliances = 1, unconditioned = 0, Part use factor = 0.

| Product Category | Existing Unit $UEC_{EXIST}$ [772] | New Baseline $UEC_{BASE}$ | New Efficient $UEC_{EE}$ | | Early Replacement (1st 4 years) ΔkWh | | Time of Sale and Early Replacement (last 8 years) ΔkWh | |
|---|---|---|---|---|---|---|---|---|
| | | | ENERGY STAR | CEE T2 | ENERGY STAR | CEE T2 | ENERGY STAR | CEE T2 |
| all-refrigerators--automatic defrost | | | | | | | | |
| 4. Refrigerator-Freezers--automatic defrost with side-mounted freezer without through-the-door ice service | 1241.0 | 634.0 | 507.2 | 475.5 | 733.7 | 765.4 | 126.8 | 158.5 |
| 5. Refrigerator-Freezers--automatic defrost with bottom-mounted freezer without through-the-door ice service | 814.5 | 577.5 | 462.0 | 433.2 | 352.5 | 381.4 | 115.5 | 144.4 |
| 6. Refrigerator-Freezers--automatic defrost with top-mounted freezer with through-the-door ice service | 814.5 | 618.8 | 495.1 | 464.1 | 319.5 | 350.4 | 123.8 | 154.7 |
| 7. Refrigerator-Freezers--automatic defrost with side-mounted freezer with through-the-door ice service | 1241.0 | 666.3 | 533.0 | 499.7 | 707.9 | 741.3 | 133.3 | 166.6 |

Assumptions after standard changes on September 1st, 2014:

| Product Category | Existing Unit $UEC_{EXIST}$ [773] | New Baseline $UEC_{BASE}$ | New Efficient $UEC_{EE}$ | | Early Replacement (1st 4 years) ΔkWh | | Time of Sale and Early Replacement (last 8 years) ΔkWh | |
|---|---|---|---|---|---|---|---|---|
| | | | ENERGY STAR | CEE T2 | ENERGY STAR | CEE T2 | ENERGY STAR | CEE T2 |
| 1. Refrigerators and Refrigerator-freezers with manual defrost | 1027.7 | 368.6 | 331.6 | 276.4 | 696.1 | 751.3 | 36.9 | 92.1 |
| 2. Refrigerator-Freezer--partial automatic defrost | 1027.7 | 430.9 | 387.8 | 323.2 | 640.0 | 704.6 | 43.1 | 107.7 |

[773] Estimates of existing unit consumption are based on using the 5.1.8 Refrigerator and Freezer Recycling algorithm and the inputs described here: Age = 10 years, Pre-1990 = 0, Size = 21.5 ft3 (from ENERGY STAR calc and consistent with AV of 25.8), Single Door = 0, Side by side = 1 for classifications stating side by side, 0 for classifications stating top/bottom, and 0.5 for classifications that do not distinguish, Primary appliances = 1, unconditioned = 0, Part use factor = 0.

| Product Category | Existing Unit $UEC_{EXIST}$ [773] | New Baseline $UEC_{BASE}$ | New Efficient $UEC_{EE}$ | | Early Replacement (1st 4 years) ΔkWh | | Time of Sale and Early Replacement (last 8 years) ΔkWh | |
|---|---|---|---|---|---|---|---|---|
| | | | ENERGY STAR | CEE T2 | ENERGY STAR | CEE T2 | ENERGY STAR | CEE T2 |
| 3. Refrigerator-Freezers--automatic defrost with top-mounted freezer without through-the-door ice service and all-refrigerators--automatic defrost | 814.5 | 441.7 | 397.4 | 331.2 | 417.2 | 483.3 | 44.3 | 110.4 |
| 4. Refrigerator-Freezers--automatic defrost with side-mounted freezer without through-the-door ice service | 1241.0 | 517.1 | 465.4 | 387.8 | 775.6 | 853.1 | 51.7 | 129.3 |
| 5. Refrigerator-Freezers--automatic defrost with bottom-mounted freezer without through-the-door ice service | 814.5 | 545.1 | 490.7 | 408.8 | 323.9 | 405.8 | 54.4 | 136.3 |
| 5A Refrigerator-freezer—automatic defrost with bottom-mounted freezer with through-the-door ice service | 814.5 | 713.8 | 651.0 | 535.3 | 163.6 | 279.2 | 62.8 | 178.4 |
| 6. Refrigerator-Freezers--automatic defrost with top-mounted freezer with through-the-door ice service | 814.5 | 601.9 | 550.1 | 451.4 | 264.4 | 363.2 | 51.7 | 150.5 |
| 7. Refrigerator-Freezers--automatic defrost with side-mounted freezer with through-the-door ice service | 1241.0 | 652.9 | 596.1 | 489.6 | 644.9 | 751.3 | 56.8 | 163.2 |

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = (ΔkWh/8766) * TAF * LSAF

Where:

TAF = Temperature Adjustment Factor

= 1.25[774]

[774] Average temperature adjustment factor (to account for temperature conditions during peak period as compared to year as a whole) based on Blasnik, Michael, "Measurement and Verification of Residential Refrigerator Energy Use, Final Report, 2003-2004 Metering Study", July 29, 2004 (p. 47). It assumes 90 °F average outside temperature during peak period, 71°F average temperature in kitchens and 65°F average temperature in basement, and uses assumption that 66% of homes in Illinois having

LSAF = Load Shape Adjustment Factor

= 1.057 [775]

If volume is unknown, use the following defaults:

central cooling (CAC saturation: "Table HC7.9 Air Conditioning in Homes in Midwest Region, Divisions, and States, 2009 from Energy Information Administration", 2009 Residential Energy Consumption Survey; http://www.eia.gov/consumption/residential/data/2009/xls/HC7.9%20Air%20Conditioning%20in%20Midwest%20Region.xls )

[775] Daily load shape adjustment factor (average load in peak period /average daily load) also based on Blasnik, Michael, "Measurement and Verification of Residential Refrigerator Energy Use, Final Report, 2003-2004 Metering Study", July 29, 2004 (p. 48, using the average Existing Units Summer Profile for hours 13 through 17)

| Product Category | Assumptions prior to September 2014 standard change ΔkW | | | | Assumptions after September 2014 standard change ΔkW | | | |
|---|---|---|---|---|---|---|---|---|
| | Early Replacement (1st 4 years) | | Time of Sale and Early Replacement (last 8 years) | | Early Replacement (1st 4 years) | | Time of Sale and Early Replacement (last 8 years) | |
| | ENERGY STAR | CEE T2 | ENERGY STAR | CEE T2 | ENERGY STAR | CEE T2 | ENERGY STAR | CEE T2 |
| 1. Refrigerators and Refrigerator-freezers with manual defrost | 0.098 | 0.101 | 0.014 | 0.018 | 0.105 | 0.113 | 0.006 | 0.014 |
| 2. Refrigerator-Freezer--partial automatic defrost | 0.098 | 0.101 | 0.014 | 0.018 | 0.096 | 0.106 | 0.006 | 0.016 |
| 3. Refrigerator-Freezers--automatic defrost with top-mounted freezer without through-the-door ice service and all-refrigerators--automatic defrost | 0.059 | 0.063 | 0.016 | 0.020 | 0.063 | 0.073 | 0.007 | 0.017 |
| 4. Refrigerator-Freezers--automatic defrost with side-mounted freezer without through-the-door ice service | 0.111 | 0.115 | 0.019 | 0.024 | 0.117 | 0.129 | 0.008 | 0.019 |
| 5. Refrigerator-Freezers--automatic defrost with bottom-mounted freezer without through-the-door ice service | 0.053 | 0.057 | 0.017 | 0.022 | 0.049 | 0.061 | 0.008 | 0.021 |
| 5A Refrigerator-freezer—automatic defrost with bottom-mounted freezer with through-the-door ice service | n/a | n/a | n/a | n/a | 0.025 | 0.042 | 0.009 | 0.027 |
| 6. Refrigerator-Freezers--automatic defrost with top-mounted freezer with through-the-door ice service | 0.048 | 0.053 | 0.019 | 0.023 | 0.040 | 0.055 | 0.008 | 0.023 |
| 7. Refrigerator-Freezers--automatic defrost with side-mounted freezer with through-the-door ice service | 0.107 | 0.112 | 0.020 | 0.025 | 0.097 | 0.113 | 0.009 | 0.025 |

### NATURAL GAS SAVINGS

N/A

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: RS-APL-ESRE-V02-140601

## 5.1.7 ENERGY STAR and CEE Tier 1 Room Air Conditioner

**DESCRIPTION**

This measure relates to:

a) Time of Sale the purchase and installation of a room air conditioning unit that meets CEE TIER 1 (equivalent to ENERGY STAR version 3.0 which is effective October 1st 2013) or CEE Tier 2 minimum qualifying efficiency specifications, in place of a baseline unit. The baseline is equivalent to ENERGY STAR Version 2.0 efficiency ratings presented below since according to ENERGY STAR Shipment Data the estimated market penetration of ENERGY STAR Room AC went from 33%[776] in 2010 to 62%[777] in 2011 and a 2012 Illinois program evaluation found a net-to-gross ratio of just 1% for a Version 2.0 ENERGY STAR unit.

| Product Type and Class (Btu/hr) | | ENERGY STAR v2.0 with louvered sides (EER)[778] | ENERGY STAR v2.0 without louvered sides (EER) | ENERGY STAR v3.0 / CEE Tier 1 with louvered sides (EER)[779] | ENERGY STAR v3.0 / CEE Tier 1 without louvered sides (EER) | CEE TIER 2 (EER)[780] |
|---|---|---|---|---|---|---|
| Without Reverse Cycle | < 8,000 | 10.7 | 9.9 | 11.2 | 10.4 | 11.6 |
| | 8,000 to 10,999 | 10.8 | 9.9 | 11.3 | 9.8 | 11.8 |
| | 11,000 to 13,999 | 10.8 | 9.4 | 11.3 | 9.8 | 11.8 |
| | 14,000 to 19,999 | 10.7 | 9.4 | 11.2 | 9.8 | 11.6 |
| | 20,000 to 24,999 | 9.4 | 9.4 | 9.8 | 9.8 | 10.2 |
| | >=25,000 | 9.4 | 9.4 | 9.8 | 9.8 | 10.2 |
| With Reverse Cycle | <14,000 | 9.9 | 9.4 | 10.4 | 9.8 | 11.8 |
| | 14,000 to 19,999 | 9.9 | 8.8 | 10.4 | 9.2 | 11.6 |
| | >=20,000 | 9.4 | 8.8 | 9.8 | 9.2 | 10.2 |
| Casement only | | 9.6 | | 10.0 | | |
| Casement-Slider | | 10.5 | | 10.9 | | |

Side louvers extend from a room air conditioner model in order to position the unit in a window. A model without louvered sides is placed in a built-in wall sleeve and are commonly referred to as "through-the-wall" or "built-in" models.

Casement-only refers to a room air conditioner designed for mounting in a casement window of a specific size.

Casement-slider refers to a room air conditioner with an encased assembly designed for mounting in a sliding or casement window of a specific size.

[776] http://www.energystar.gov/ia/partners/downloads/unit_shipment_data/2010_USD_Summary_Report.pdf?3193-51e7
[777] http://www.energystar.gov/ia/partners/downloads/unit_shipment_data/2011_USD_Summary_Report.pdf?3193-51e7
[778] http://www.energystar.gov/ia/partners/prod_development/revisions/downloads/roomac/RAC_ProgramRequirements_1105.pdf?c2df-6034
[779] http://www.energystar.gov/index.cfm?c=roomac.pr_crit_room_ac
[780] http://library.cee1.org/sites/default/files/library/9296/CEE_ResApp_RoomAirConditionerSpecification_2003_Updated.pdf

Reverse cycle refers to the heating function found in certain room air conditioner models.

b) Early Replacement: the early removal of an existing residential inefficient Room AC unit from service, prior to its natural end of life, and replacement with a new ENERGY STAR or CEE Tier 1 qualifying unit. Savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life.

This measure was developed to be applicable to the following program types: TOS, NC, EREP.

If applied to other program types, the measure savings should be verified.

**DEFINITION OF EFFICIENT EQUIPMENT**

To qualify for this measure the new room air conditioning unit must meet the CEE TIER 1 (equivalent to ENERGY STAR version 3.0 which is effective October 1st 2013) efficiency standards presented above.

**DEFINITION OF BASELINE EQUIPMENT**

Time of Sale: the baseline assumption is a new room air conditioning unit that meets the ENERGY STAR Version 2.0 efficiency standards as presented above.

Early Replacement: the baseline is the existing Room AC for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

**DEEMED LIFETIME OF EFFICIENT EQUIPMENT**

The measure life is assumed to be 12 years[781].

Remaining life of existing equipment is assumed to be 4 years[782]

**DEEMED MEASURE COST**

Time of Sale: The incremental cost for this measure is assumed to be $40 for a CEE TIER 1 unit and $100 for a CEE Tier 2 unit[783].

Early Replacement: The measure cost is the full cost of removing the existing unit and installing a new one. The actual program cost should be used. If unavailable assume $448 for CEE Tier 1 unit and $548 for CEE Tier 2 unit[784].

The avoided replacement cost (after 4 years) of a baseline replacement refrigerator is $376.[785]

**LOADSHAPE**

Loadshape R08 - Residential Cooling

[781] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf
[782] Standard assumption of one third of effective useful life.
[783] CEE Tier 1 based on field study conducted by Efficiency Vermont and Tier 2 based on professional judgement.
[784] Based on IL PHA Efficient Living Prgroam Data for 810 replaced units showing $416 per unit plus $32 average recycling/removal cost.
[785] Estimate based upon Time of Sale incremental costs.

### COINCIDENCE FACTOR

The coincidence factor for this measure is assumed to be 0.3[786].

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

Time of Sale: $\Delta kWh = (FLH_{RoomAC} * Btu/H * (1/EERbase - 1/EERee))/1000$

Early Replacment:

$\Delta kWh$ for remaining life of existing unit (1st 4 years) $= (FLH_{RoomAC} * Btu/H * (1/EERexist - 1/EERee))/1000$

$\Delta kWh$ for remaining measure life (next 8 years) $= (FLH_{RoomAC} * Btu/H * (1/EERbase - 1/EERee))/1000$

Where:

$FLH_{RoomAC}$ = Full Load Hours of room air conditioning unit

= dependent on location[787]:

| Climate Zone (City based upon) | $FLH_{RoomAC}$ |
|---|---|
| 1 (Rockford) | 220 |
| 2 (Chicago) | 210 |
| 3 (Springfield) | 319 |
| 4 (Belleville) | 428 |
| 5 (Marion) | 374 |
| Weighted Average[788] | 248 |

[786] Consistent with coincidence factors found in: RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008 (http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RAC.pdf)

[787] Full load hours for room AC is significantly lower than for central AC. The average ratio of FLH for Room AC (provided in RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008: http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RAC.pdf) to FLH for Central Cooling for the same location (provided by AHRI: http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls) is 31%. This ratio is applied to those IL cities that have FLH for Central Cooling provided in the Energy Star calculator. For other cities this is extrapolated using the FLH assumptions VEIC have developed for Central AC. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.

[788] Weighted based on number of residential occupied housing units in each zone.

Btu/H = Size of rebated unit

= Actual. If unknown assume 8500 Btu/hr[789]

EERexist = Efficiency of existing unit

= Actual. If unknown assume 7.7[790]

EERbase = Efficiency of baseline unit

= As provided in tables above

EERee = Efficiency of CEE Tier 1 (or ENERGY STAR Version 3.0) unit

= Actual. If unknown assume minimum qualifying standard as provided in tables above

Time of Sale:

For example for an 8,500 Btu/H capacity unit, with louvered sides, in an unknown location:

$\Delta kWH_{CEE\ TIER\ 1}$ = (248 * 8500 * (1/10.8 – 1/11.3)) / 1000

= 8.6 kWh

Early Replacement:

A 7.7EER, 9000Btu/h unit is removed from a home in Springfield and replaced with a CEE T1 unit with louvered sides:

ΔkWh for remaining life of existing unit (1st 4 years) = (319 * 9000 * (1/7.7 - 1/11.3))/1000

= 118.8 kWh

ΔkWh for remaining measure life (next 8 years) = (319 * 9000 * (1/10.8 - 1/11.3))/1000

= 11.8 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = Btu/H * ((1/EERbase - 1/EERee))/1000) * CF

Where:

CF = Summer Peak Coincidence Factor for measure

= 0.3[791]

Other variable as defined above

[789] Based on maximum capacity average from the RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008

[790] Based on Nexus Market Research Inc, RLW Analytics, December 2005; "Impact, Process, and Market Study of the Connecticut Appliance Retirement Program: Overall Report."

[791] Consistent with coincidence factors found in: RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008 (http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RAC.pdf)

Time of Sale:

For example for an 8,500 Btu/H capacity unit, with louvered sides, for an unknown location:

$\Delta kW_{CEE\ TIER\ 1}$ = (8500 * (1/10.8 – 1/11.3)) / 1000 * 0.3

= 0.010 kW

Early Replacement:

A 7.7EER, 9000Btu/h unit is removed from a home in Springfield and replaced with a CEE T1 unit with louvered sides:

ΔkW for remaining life of existing unit (1st 4 years) = (9000 * (1/7.7 - 1/11.3))/1000 * 0.3

= 0.11 kW

ΔkW for remaining measure life (next 8 years) = (9000 * (1/10.8 - 1/11.3))/1000 * 0.3

= 0.011 kW

**NATURAL GAS SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-APL-ESRA-V03-140601**

## 5.1.8 Refrigerator and Freezer Recycling

### DESCRIPTION

This measure describes savings from the retirement and recycling of inefficient but operational refrigerators and freezers. Savings are provided based on a 2013 workpaper provided by Cadmus that used data from a 2012 ComEd metering study and metering data from a Michigan study, to develop a regression equation that uses key inputs describing the retired unit. The savings are equivalent to the Unit Energy Consumption of the retired unit and should be claimed for the assumed remaining useful life of that unit. A part use factor is applied to account for those secondary units that are not in use throughout the entire year. The reader should note that the regression algorithm is designed to provide an accurate portrayal of savings for the population as a whole and includes those parameters that have a significant effect on the consumption. The precision of savings for individual units will vary.

The Net to Gross factor applied to these units should incorporate adjustments that account for:

- Those participants who would have removed the unit from the grid anyway (e.g. customers replacing their refrigerator via a big box store and using the pick-up option, customers taking their unit to the landfill or recycling station);
- Those participants who decided, based on the incentive provided by the Appliance Recycling program alone, to replace their existing inefficient unit with a new unit. This segment of participants is expected to be very small and documentation of their intentions will be gathered via telephone surveys (i.e., primary data sources). For such customers, the consumption of the new unit should be subtracted from the retired unit consumption and savings claimed for the remaining life of the existing unit. Note that participants who were already planning to replace their unit, and the incentive just ensured that the retired unit was recycled and not placed on the secondary market, should not be included in this adjustment.

This measure was developed to be applicable to the following program types: ERET.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

n/a

### DEFINITION OF BASELINE EQUIPMENT

The existing inefficient unit must be operational and have a capacity of between 10 and 30 cubic feet.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The estimated remaining useful life of the recycling units is 8 years [792].

### DEEMED MEASURE COST

Measure cost includes the cost of pickup and recycling of the refrigerator and should be based on actual costs of running the program. If unknown assume $120[793] per unit.

[792] KEMA "Residential refrigerator recycling ninth year retention study", 2004
[793] Based on similar Efficiency Vermont program.

### LOADSHAPE

Loadshape R05 - Residential Refrigerator

### COINCIDENCE FACTOR

The coincidence factor is assumed to be 0.00012.

## Algorithm

### CALCULATION OF SAVINGS

### ENERGY SAVINGS[794]

Refrigerators:

Energy savings for refrigerators are based upon a linear regression model using the following coefficients[795]:

| Independent Variable Description | Estimate Coefficient |
|---|---|
| Intercept | 83.324 |
| Age (years) | 3.678 |
| Pre-1990 (=1 if manufactured pre-1990) | 485.037 |
| Size (cubic feet) | 27.149 |
| Dummy: Side-by-Side (= 1 if side-by-side) | 406.779 |
| Dummy: Primary Usage Type (in absence of the program)<br>(= 1 if primary unit) | 161.857 |
| Interaction: Located in Unconditioned Space x CDD/365.25 | 15.366 |
| Interaction: Located in Unconditioned Space x HDD/365.25 | -11.067 |

ΔkWh = [83.32 + (Age * 3.68) + (Pre-1990 * 485.04) + (Size * 27.15) + (Side-by-side * 406.78) + (Proportion of Primary Appliances * 161.86) + (CDD/365.25 * unconditioned * 15.37) + (HDD/365.25 *unconditioned *-11.07)] * Part Use Factor

Where:

Age = Age of retired unit

[794] Based on the specified regression, a small number of units may have negative energy and demand consumption. These are a function of the unit size and age, and should comprise a very small fraction of the population. While on an individual basis this result is counterintuitive it is important that these negative results remain such that as a population the average savings is appropriate.

[795] Energy savings are based on an average 30-year TMY temperature of 51.1 degrees. Coefficients provided in July 30, 2014 memo from Cadmus: "Appliance Recycling Update no single door July 30 2014".

Pre-1990 = Pre-1990 dummy (=1 if manufactured pre-1990, else 0)

Size = Capacity (cubic feet) of retired unit

Side-by-side = Side-by-side dummy (= 1 if side-by-side, else 0)

Single-Door = Single-Door dummy (= 1 if Single-Door, else 0)

Primary Usage = Primary Usage Type (in absence of the program) dummy (= 1 if Primary, else 0)

Interaction: Located in Unconditioned Space x CDD/365.25

(=1 * CDD/365.25 if in unconditioned space)

CDD = Cooling Degree Days

= Dependent on location[796]:

| Climate Zone (City based upon) | CDD 65 | CDD/365.25 |
|---|---|---|
| 1 (Rockford) | 820 | 2.25 |
| 2 (Chicago) | 842 | 2.31 |
| 3 (Springfield) | 1,108 | 3.03 |
| 4 (Belleville) | 1,570 | 4.30 |
| 5 (Marion) | 1,370 | 3.75 |

Interaction: Located in Unconditioned Space x HDD/365.25

(=1 * HDD/365.25 if in unconditioned space)

HDD = Heating Degree Days

= Dependent on location:[797]

| Climate Zone (City based upon) | HDD 65 | HDD/365.25 |
|---|---|---|
| 1 (Rockford) | 6,569 | 17.98 |
| 2 (Chicago) | 6,339 | 17.36 |
| 3 (Springfield) | 5,497 | 15.05 |
| 4 (Belleville) | 4,379 | 11.99 |
| 5 (Marion) | 4,476 | 12.25 |

[796] National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 65°F.
[797] National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 65°F.

Part Use Factor = To account for those units that are not running throughout the entire year. The most recent part-use factor participant survey results available at the start of the current program year shall be used[798]. For illustration purposes, this example uses 0.93.[799]

For example, the program averages for AIC's ARP in PY4 produce the following equation:

ΔkWh = [83.32 + (22.81 * 3.68) + (0.45 * 485.04) + (18.82 * 27.15) + (0.17 * 406.78) + (0.34 * 161.86) + (1.29 * 15.37) + (6.49 * -11.07)] * 0.93

= 969 * 0.93

= 900.9 kWh

Freezers:

Energy savings for freezers are based upon a linear regression model using the following coefficients[800]:

| Independent Variable Description | Estimate Coefficient |
|---|---|
| Intercept | 132.122 |
| Age (years) | 12.130 |
| Pre-1990 (=1 if manufactured pre-1990) | 156.181 |
| Size (cubic feet) | 31.839 |
| Chest Freezer Configuration (=1 if chest freezer) | -19.709 |
| Interaction: Located in Unconditioned Space x CDD/365.25 | 9.778 |
| Interaction: Located in Unconditioned Space x HDD/365.25 | -12.755 |

ΔkWh = [132.12 + (Age * 12.13) + (Pre-1990 * 156.18) + (Size * 31.84) + (Chest Freezer * -19.71) + (CDDs* unconditioned *9.78) + (HDDs*unconditioned *-12.75)] * Part Use Factor

Where:

Age = Age of retired unit

Pre-1990 = Pre-1990 dummy (=1 if manufactured pre-1990, else 0)

Size = Capacity (cubic feet) of retired unit

Chest Freezer = Chest Freezer dummy (= 1 if chest freezer, else 0)

[798] For example, the part-use factor that shall be applied to the current program year t (PYt) for savings verification purposes should be determined through the PYt-2 participant surveys conducted in the respective utility's service territory, if available. If an evaluation was not performed in PYt-2 the latest available evaluation should be used.

[799] Most recent refrigerator part-use factor from Ameren Illinois PY5 evaluation.

[800] Energy savings are based on an average 30-year TMY temperature of 51.1 degrees. Coefficients provided in January 31, 2013 memo from Cadmus: "Appliance Recycling Update".

Interaction: Located in Unconditioned Space x CDD/365.25

(=1 * CDD/365.25 if in unconditioned space)

CDD = Cooling Degree Days (see table above)

Interaction: Located in Unconditioned Space x HDD/365.25

(=1 * HDD/365.25 if in unconditioned space)

HDD = Heating Degree Days (see table above)

Part Use Factor = To account for those units that are not running throughout the entire year. The most recent part-use factor participant survey results available at the start of the current program year shall be used[801]. . For illustration purposes, the example uses 0.85.[802]

The program averages for AIC's ARP PY4 program are used as an example.

ΔkWh = [132.12 + (26.92 * 12.13) + (0.6 * 156.18) + (15.9 * 31.84) + (0.48 * -19.71) + (6.61 * 9.78) + (1.3 * -12.75)] * 0.825

= 977 * 0.825

= 905 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = kWh/8766 * CF

Where:

kWh = Savings provided in algorithm above

CF = Coincident factor defined as summer kW/average kW

= 1.081 for Refrigerators

= 1.028 for Freezers[803]

For example, the program averages for AIC's ARP in PY4 produce the following equation:

ΔkW = 806/8766 * 1.081

= 0.099 kW

[801] For example, the part-use factor that shall be applied to the current program year t (PYt) for savings verification purposes should be determined through the PYt-2 participant surveys conducted in the respective utility's service territory, if available. If an evaluation was not performed in PYt-2 the latest available evaluation should be used.
[802] Most recent freezer part-use factor from Ameren Illnois Company PY5 evaluation.
[803] Cadmus memo, February 12, 2013; "Appliance Recycling Update"

### NATURAL GAS SAVINGS

N/A

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: RS-APL-RFRC-V05-150601

## 5.1.9 Room Air Conditioner Recycling

### DESCRIPTION

This measure describes the savings resulting from running a drop off service taking existing residential, inefficient Room Air Conditioner units from service, prior to their natural end of life. This measure assumes that though a percentage of these units will be replaced this is not captured in the savings algorithm since it is unlikely that the incentive made someone retire a unit that they weren't already planning to retire. The savings therefore relate to the unit being taken off the grid as opposed to entering the secondary market. The Net to Gross factor applied to these units should incorporate adjustments that account for those participants who would have removed the unit from the grid anyway.

This measure was developed to be applicable to the following program types: ERET.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

N/A. This measure relates to the retiring of an existing inefficient unit.

### DEFINITION OF BASELINE EQUIPMENT

The baseline condition is the existing inefficient room air conditioning unit.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed remaining useful life of the existing room air conditioning unit being retired is 4 years[804].

### DEEMED MEASURE COST

The actual implementation cost for recycling the existing unit should be used.

### LOADSHAPE

Loadshape R08 - Residential Cooling

### COINCIDENCE FACTOR

The coincidence factor for this measure is assumed to be 30%[805].

[804] A third of assumed measure life for Room AC.

[805] Consistent with coincidence factors found in: RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008 (http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RAC.pdf)

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = ((FLH_{RoomAC} * Btu/hr * (1/EERexist))/1000)$$

Where:

$FLH_{RoomAC}$ = Full Load Hours of room air conditioning unit

= dependent on location[806]:

| Climate Zone (City based upon) | $FLH_{RoomAC}$ |
|---|---|
| 1 (Rockford) | 220 |
| 2 (Chicago) | 210 |
| 3 (Springfield) | 319 |
| 4 (Belleville) | 428 |
| 5 (Marion) | 374 |
| Weighted Average[807] | 248 |

Btu/H = Size of retired unit

= Actual. If unknown assume 8500 Btu/hr [808]

EERexist = Efficiency of existing unit

= 7.7[809]

[806] The average ratio of FLH for Room AC (provided in RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008: http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RAC.pdf) to FLH for Central Cooling for the same location (provided by AHRI: http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls) is 31%. This ratio is applied to those IL cities that have FLH for Central Cooling provided in the Energy Star calculator. For other cities this is extrapolated using the FLH assumptions VEIC have developed for Central AC. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.

[807] Weighted based on number of residential occupied housing units in each zone.

[808] Based on maximum capacity average from the RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008

[809] Based on Nexus Market Research Inc, RLW Analytics, December 2005; "Impact, Process, and Market Study of the Connecticut Appliance Retirement Program: Overall Report."

For example for an 8500 Btu/h unit in Springfield:

$\Delta kWh$ = ((319 * 8500 * (1/7.7)) / 1000)

= 352 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$\Delta kW$ = (Btu/hr * (1/EERexist))/1000) * CF

Where:

CF = Summer Peak Coincidence Factor for measure

= 0.3[810]

For example an 8500 Btu/h unit:

$\Delta kW$ = (8500 * (1/7.7)) / 1000) * 0.3

= 0.33 kW

**NATURAL GAS SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-APL-RARC-V01-120601**

[810] Consistent with coincidence factors found in:
RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008 (http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RAC.pdf)

## 5.1.10 Residential ENERGY STAR Clothes Dryer

### DESCRIPTION

This measure relates to the installation of a residential clothes dryer meeting the ENERGY STAR criteria. ENERGY STAR qualified clothes dryers save energy through a combination of more efficient drying and reduced runtime of the drying cycle. More efficient drying is achieved through increased insulation, modifying operating conditions such as air flow and/or heat input rate, improving air circulation through better drum design or booster fans, and improving efficiency of motors. Reducing the runtime of dryers through automatic termination by temperature and moisture sensors is believed to have the greatest potential for reducing energy use in clothes dryers[811]. ENERGY STAR provides criteria for both gas and electric clothes dryers.

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

Clothes dryer must meet the ENERGY STAR criteria, as required by the program.

### DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a clothes dryer meeting the minimum federal requirements for units manufactured on or after January 1, 2015.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 14 years[812].

### DEEMED MEASURE COST

The incremental cost for an ENERGY STAR clothes dryer is assumed to be $152[813]

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

The coincidence factor for this measure is 3.8%[814].

[811] ENERGY STAR Market & Industry Scoping Report. Residential Clothes Dryers. Table 8. November 2011. http://www.energystar.gov/ia/products/downloads/ENERGY_STAR_Scoping_Report_Residential_Clothes_Dryers.pdf

[812] Based on an average estimated range of 12-16 years. ENERGY STAR Market & Industry Scoping Report. Residential Clothes Dryers. November 2011. http://www.energystar.gov/ia/products/downloads/ENERGY_STAR_Scoping_Report_Residential_Clothes_Dryers.pdf

[813] Based on the difference in installed cost for an efficient dryer ($716) and standard dryer ($564). http://www.aceee.org/files/proceedings/2012/data/papers/0193-000286.pdf

[814] Based on coincidence factor of 3.8% for clothes washers

## Algorithm

### CALCULATION OF ENERGY SAVINGS

#### ELECTRIC ENERGY SAVINGS

ΔkWh = (Load/CEFbase – Load/CEFeff) * Ncycles * %Electric

Where:

Load = The average total weight (lbs) of clothes per drying cycle. If dryer size is unknown, assume standard.

| Dryer Size | Load (lbs)[815] |
|---|---|
| Standard | 8.45 |
| Compact | 3 |

CEFbase = Combined energy factor (CEF) (lbs/kWh) of the baseline unit is based on existing federal standards energy factor and adjusted to CEF as performed in the ENERGY STAR analysis[816]. If product class unknown, assume electric, standard.

| Product Class | CEF (lbs/kWh) |
|---|---|
| Vented Electric, Standard (≥ 4.4 $ft^3$) | 3.11 |
| Vented Electric, Compact (120V) (< 4.4 $ft^3$) | 3.01 |
| Vented Electric, Compact (240V) (<4.4 $ft^3$) | 2.73 |
| Ventless Electric, Compact (240V) (<4.4 $ft^3$) | 2.13 |
| Vented Gas | 2.84[817] |

CEFeff = CEF (lbs/kWh) of the ENERGY STAR unit based on ENERGY STAR requirements.[818] If product class unknown, assume electric, standard.

| Product Class | CEF (lbs/kWh) |
|---|---|
| Vented or Ventless Electric, Standard (≥ 4.4 $ft^3$) | 3.93 |
| Vented or Ventless Electric, Compact (120V) (< 4.4 $ft^3$) | 3.80 |
| Vented Electric, Compact (240V) (< 4.4 $ft^3$) | 3.45 |
| Ventless Electric, Compact (240V) (< 4.4 $ft^3$) | 2.68 |
| Vented Gas | 3.48[819] |

[815] Based on ENERGY STAR test procedures. https://www.energystar.gov/index.cfm?c=clothesdry.pr_crit_clothes_dryers
[816] ENERGY STAR Draft 2 Version 1.0 Clothes Dryers Data and Analysis
[817] Federal standards report CEF for gas clothes dryers in terms of lbs/kWh. To determine gas savings, this number is later converted to therms.
[818] ENERGY STAR Clothes Dryers Key Product Criteria. https://www.energystar.gov/index.cfm?c=clothesdry.pr_crit_clothes_dryers
[819] Federal standards report CEF for gas clothes dryers in terms of lbs/kWh. To determine gas savings, this number is later converted to therms.

Ncycles = Number of dryer cycles per year. Use actual data if available. If unknown, use 283 cycles per year.[820]

%Electric = The percent of overall savings coming from electricity

= 100% for electric dryers, 16% for gas dryers[821]

**EXAMPLE**

Time of Sale: For example, a standard, vented, electric clothes dryer:

ΔkWh = ((8.45/3.11 – 8.45/3.93) * 283 * 100%)

= 160 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ΔkWh/Hours * CF

Where:

ΔkWh = Energy Savings as calculated above

Hours = Annual run hours of clothes dryer. Use actual data if available. If unknown, use 283 hours per year.[822]

CF = Summer Peak Coincidence Factor for measure

= 3.8%[823]

**EXAMPLE**

Time of Sale: For example, a standard, vented, electric clothes dryer:

ΔkW = 160/283 * 3.8%

= 0.0215 kW

**NATURAL GAS SAVINGS**

Natural gas savings only apply to ENERGY STAR vented gas clothes dryers.

[820] Appendix D to Subpart B of Part 430 – Uniform Test Method for Measuring the Energy Consumption of Dryers.
[821] %Electric accounts for the fact that some of the savings on gas dryers comes from electricity (motors, controls, etc). 16% was determined using a ratio of the electric to total savings from gas dryers given by ENERGY STAR Draft 2 Version 1.0 Clothes Dryers Data and Analysis.
[822] ENERGY STAR qualified dryers have a maximum test cycle time of 80 minutes. Assume one hour per dryer cycle.
[823] Based on coincidence factor of 3.8% for clothes washers.

ΔTherm = (Load/EFbase – Load/CEFeff) * Ncycles * Therm_convert * %Gas

Where:

Therm_convert = Conversion factor from kWh to Therm

= 0.03413

%Gas = Percent of overall savings coming from gas

= 0% for electric units and 84% for gas units[824]

**EXAMPLE**

Time of Sale: For example, a standard, vented, gas clothes dryer:

ΔTherm = (8.45/2.84 – 8.45/3.48) * 283 * 0.03413 * 0.84

= 4.44 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

## MEASURE CODE: RS-APL-ESDR-V01-150601

[824] %Gas accounts for the fact that some of the savings on gas dryers comes from electricity (motors, controls, etc). 84% was determined using a ratio of the gas to total savings from gas dryers given by ENERGY STAR Draft 2 Version 1.0 Clothes Dryers Data and Analysis.

# 5.2 Consumer Electronics End Use

## 5.2.1 Smart Strip

### DESCRIPTION

This measure relates to Controlled Power Strips (or Smart Strips) which are multi-plug power strips with the ability to automatically disconnect specific connected loads depending upon the power draw of a control load, also plugged into the strip. Power is disconnected from the switched (controlled) outlets when the control load power draw is reduced below a certain adjustable threshold, thus turning off the appliances plugged into the switched outlets. By disconnecting, the standby load of the controlled devices, the overall load of a centralized group of equipment (i.e. entertainment centers and home office) can be reduced. Uncontrolled outlets are also provided that are not affected by the control device and so are always providing power to any device plugged into it. This measure characterization provides savings for a 5-plug strip and a 7-plug strip.

This measure was developed to be applicable to the following program types: TOS, NC, DI.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient case is the use of a 5 or 7-plug smart strip.

### DEFINITION OF BASELINE EQUIPMENT

The assumed baseline is a standard power strip that does not control connected loads.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of the smart strip is 4 years[825].

### DEEMED MEASURE COST

The incremental cost of a smart strip over a standard power strip with surge protection is assumed to be $16 for a 5-plug and $26 for a 7-plug[826].

### LOADSHAPE

Loadshape R13 - Residential Standby Losses – Entertainment

Loadshape R14 - Residential Standby Losses - Home Office

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 80%[827].

[825] David Rogers, Power Smart Engineering, October 2008; "Smart Strip electrical savings and usability", p22.
[826] Price survey performed in NYSERDA Measure Characterization for Advanced Power Strips, p4
[827] Efficiency Vermont coincidence factor for smart strip measure –in the absence of empirical evaluation data, this was based on assumptions of the typical run pattern for televisions and computers in homes.

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$\Delta kWh_{5\text{-}Plug}$ = 56.5 kWh [828]

$\Delta kWh_{7\text{-}Plug}$ = 103 kWh [829]

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW$ = $\Delta kWh$ / Hours * CF

Where:

Hours = Annual number of hours during which the controlled standby loads are turned off by the Smart Strip.

= 7,129 [830]

CF = Summer Peak Coincidence Factor for measure

= 0.8 [831]

$\Delta kW_{5\text{-}Plug}$ = 56.5 / 7129 * 0.8

= 0.00634 kW

$\Delta kW_{7\text{-}Plug}$ = 102.8 / 7129 * 0.8

= 0.0115 kW

### NATURAL GAS SAVINGS

N/A

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

[828] NYSERDA Measure Characterization for Advanced Power Strips. Study based on review of:
Smart Strip Electrical Savings and Usability, Power Smart Engineering, October 27, 2008.
Final Field Research Report, Ecos Consulting, October 31, 2006. Prepared for California Energy Commission's PIER Program.
Developing and Testing Low Power Mode Measurement Methods, Lawrence Berkeley National Laboratory (LBNL), September 2004. Prepared for California Energy Commission's Public Interest Energy Research (PIER) Program.
2005 Intrusive Residential Standby Survey Report, Energy Efficient Strategies, March, 2006.
Smart Strip Portfolio of the Future, Navigant Consulting for San Diego G&E, March 31, 2009.
[829] Ibid.
[830] Average of hours for controlled TV and computer from; NYSERDA Measure Characterization for Advanced Power Strips
[831] Efficiency Vermont coincidence factor for smart strip measure –in the absence of empirical evaluation data, this was based on assumptions of the typical run pattern for televisions and computers in homes.

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-CEL-SSTR-V01-120601**

# 5.3 HVAC End Use

## 5.3.1 Air Source Heat Pump

### DESCRIPTION

A heat pump provides heating or cooling by moving heat between indoor and outdoor air.

This measure characterizes:

a) Time of Sale:
   a. The installation of a new residential sized (<= 65,000 Btu/hr) air source heat pump that is more efficient than required by federal standards. This could relate to the replacement of an existing unit at the end of its useful life, or the installation of a new system in a new home.

b) Early Replacement:
   a. The early removal of functioning electric heating and cooling (SEER 10 or under if present) systems from service, prior to its natural end of life, and replacement with a new high efficiency air source heat pump unit. Savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life.
   b. The assumption of the existing unit efficiency in the Early Replacement section of this TRM is based upon the average efficiency of units that were classified in Ameren's PY3-PY4 as functioning and SEER <=10. Therefore it is only appropriate to use these Early Replacement assumptions where those conditions are met. The TAC defined "functioning" as the unit is fully operational – providing sufficient space conditioning (i.e. heat exchanger, compressors, pumps work effectively) and/or the cost of repair is under 20% of the new baseline replacement cost. Therefore in order to apply early replacement assumptions the programs should apply the following eligibility criteria: SEER <=10 and cost of any repairs <$249 per ton.
   c. A weighted average early replacement rate is provided for use when the actual baseline early replacement rates are unknown[832].

**Deemed Early Replacement Rates For ASHP**

| | Deemed Early Replacement Rate |
|---|---|
| Early Replacement Rate for ASHP participants | 7% |

This measure was developed to be applicable to the following program types: TOS, NC, EREP. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

A new residential sized (<= 65,000 Btu/hr) air source heat pump with specifications to be determined by program.

[832] Based upon research from "Home Energy Efficiency Rebate Program GPY2 Evaluation Report" which outlines early replacement rates for both primary and secondary central air cooling (CAC) and residential furnaces. This is used as a reasonable proxy for ASHP installations since ASHP specific data is not available. Report presented to Nicor Gas Company February 27, 2014, available at http://www.ilsag.info/evaluation-documents.html.

### DEFINITION OF BASELINE EQUIPMENT

A new residential sized (<= 65,000 Btu/hr) air source heat pump meeting federal standards.

The baseline for the Time of Sale measure is based on the current Federal Standard efficiency level as of January 1st 2015; 14 SEER and 8.2HSPF.

The baseline for the early replacement measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 18 years[833].

Remaining life of existing equipment is assumed to be 6 years[834].

### DEEMED MEASURE COST

Time of sale: The incremental capital cost for this measure is dependent on the efficiency and capacity of the new unit[835]. Note these costs are per ton of unit capacity:

| Efficiency (SEER) | Incremental Cost per Ton of Capacity ($/ton) |
|---|---|
| 15 | $137 |
| 16 | $274 |
| 17 | $411 |
| 18 | $548 |

Early replacement: The full install cost for this measure is the actual cost of removing the existing unit and installing the new one. If this is unknown, assume the following (note these costs are per ton of unit capacity)[836]:

| Efficiency (SEER) | Full Retrofit Cost (including labor) per Ton of Capacity ($/ton) |
|---|---|
| 15 | $1,518 |
| 16 | $1,655 |
| 17 | $1,792 |
| 18 | $1,929 |

[833] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007, http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf
[834] Assumed to be one third of effective useful life
[835] Based on costs derived from DEER 2008 Database Technology and Measure Cost Data (www.deeresources.com).
[836] Ibid. See 'ASHP_Revised DEER Measure Cost Summary.xls' for calculation.

Assumed deferred cost (after 6 years) of replacing existing equipment with new baseline unit is assumed to be $1,381 per ton of capacity[837]. This cost should be discounted to present value using the utilities' discount rate.

### LOADSHAPE

Loadshape R10 - Residential Electric Heating and Cooling

### COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Heat Pumps (during utility peak hour)

= 72%%[838]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Heat Pumps (average during PJM peak period)

= 46.6%[839]

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

Time of sale:

ΔkWh = ((FLH_cooling * Capacity_cooling * (1/SEER_base - 1/SEER_ee)) / 1000) + ((FLH_heat * Capacity_heating * (1/HSPF_base - 1/HSFP_ee)) / 1000)

Early replacement[840]:

ΔkWH for remaining life of existing unit (1st 6 years):

= ((FLH_cooling * Capacity_cooling * (1/SEER_exist - 1/SEER_ee)) / 1000) + ((FLH_heat * Capacity_heating * (1/HSPF_exist - 1/HSFP_ee)) / 1000)

ΔkWH for remaining measure life (next 12 years):

= ((FLH_cooling * Capacity_cooling * (1/SEER_base - 1/SEER_ee)) / 1000) + ((FLH_heat * Capacity_heating * (1/HSPF_base - 1/HSFP_ee)) / 1000)

[837] Ibid.

[838] Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

[839] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

[840] The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).

Where:

FLH_cooling = Full load hours of air conditioning

= dependent on location[841]:

| Climate Zone (City based upon) | FLH_cooling (single family) | FLH_cooling (multi family) |
|---|---|---|
| 1 (Rockford) | 512 | 467 |
| 2 (Chicago) | 570 | 506 |
| 3 (Springfield) | 730 | 663 |
| 4 (Belleville) | 1,035 | 940 |
| 5 (Marion) | 903 | 820 |
| Weighted Average[842] | 629 | 564 |

Capacity_cooling = Cooling Capacity of Air Source Heat Pump (Btu/hr)

= Actual (1 ton = 12,000Btu/hr)

SEER_exist = Seasonal Energy Efficiency Ratio of existing cooling system (kBtu/kWh)

= Use actual SEER rating where it is possible to measure or reasonably estimate.

| Existing Cooling System | SEER_exist[843] |
|---|---|
| Air Source Heat Pump | 9.12 |
| Central AC | 8.60 |
| No central cooling[844] | Make '1/SEER_exist' = 0 |

SEER_base = Seasonal Energy Efficiency Ratio of baseline Air Source Heat Pump (kBtu/kWh)

= 14 [845]

[841] Full load hours for Chicago, Moline and Rockford are provided in "Final Evaluation Report: Central Air Conditioning Efficiency Services (CACES), 2010, Navigant Consulting", http://ilsag.org/yahoo_site_admin/assets/docs/ComEd_PY2_CACES_Evaluation_Report_2010-10-18.299122020.pdf, p.33. An average FLH/Cooling Degree Day (from NCDC) ratio was calculated for these locations and applied to the CDD of the other locations in order to estimate FLH. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.

[842] Weighted based on number of occupied residential housing units in each zone.

[843] Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

[844] If there is no central cooling in place but the incentive encourages installation of a new ASHP with cooling, the added cooling load should be subtracted from any heating benefit.

[845] Based on Minimum Federal Standard effective 1/1/2015;

SEER_ee = Seasonal Energy Efficiency Ratio of efficient Air Source Heat Pump (kBtu/kWh)

= Actual

FLH_heat = Full load hours of heating

= Dependent on location[846]:

| Climate Zone (City based upon) | FLH_heat |
|---|---|
| 1 (Rockford) | 1,969 |
| 2 (Chicago) | 1,840 |
| 3 (Springfield) | 1,754 |
| 4 (Belleville) | 1,266 |
| 5 (Marion) | 1,288 |
| Weighted Average[847] | 1,821 |

Capacity_heating = Heating Capacity of Air Source Heat Pump (Btu/hr)

= Actual (1 ton = 12,000Btu/hr)

HSPF_exist =Heating System Performance Factor[848] of existing heating system (kBtu/kWh)

= Use actual HSPF rating where it is possible to measure or reasonably estimate. If not available use:

| Existing Heating System | HSPF_exist |
|---|---|
| Air Source Heat Pump | 5.44 [849] |
| Electric Resistance | 3.41[850] |

http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf.

[846] Full load heating hours for heat pumps are provided for Rockford, Chicago and Springfield in the Energy Star Calculator. Estimates for the other locations were calculated based on the FLH to Heating Degree Day (from NCDC) ratio. VEIC consider Energy Star estimates to be high due to oversizing not being adequately addressed. Using average Illinois billing data (from http://www.icc.illinois.gov/ags/consumereducation.aspx) VEIC estimated the average gas heating load and used this to estimate the average home heating output (using 83% average gas heat efficiency). Dividing this by a typical 36,000 Btu/hr ASHP gives an estimate of average ASHP FLH_heat of 1821 hours. We used the ratio of this value to the average of the locations using the Energy Star data (1994 hours) to scale down the Energy Star estimates. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.

[847] Weighted based on number of occupied residential housing units in each zone.

[848] HSPF ratings for Heat Pumps account for the seasonal average efficiency of the units and are based on testing within zone 4 which encompasses most of Illinois. Furthermore, a recent Cadmus/Opinion Dynamics metering study, "Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)", found no significant variance between metered performance and that presented in the TRM

[849] This is estimated based on finding the average HSPF/SEER ratio from the AHRI directory data (using the least efficient models – SEER 12 and SEER 13) – 0.596, and applying to the average nameplate SEER rating of all Early Replacement qualifying equipment in Ameren PY3-PY4. This estimation methodology appears to provide a result within 10% of actual HSPF.

[850] Electric resistance has a COP of 1.0 which equals 1/0.293 = 3.41 HSPF.

HSPF_base =Heating System Performance Factor of baseline Air Source Heat Pump (kBtu/kWh)

= 8.2 [851]

HSFP_ee =Heating System Performance Factor of efficient Air Source Heat Pump (kBtu/kWh)

= Actual

Time of Sale:

For example, a three ton, 15 SEER, 12EER, 9 HSPF Air Source Heat Pump installed in Marion:

ΔkWh = ((903 * 36,000 * (1/14 - 1/15)) / 1000) + ((1,288 * 36,000 * (1/8.2 - 1/9)) / 1000)

= 657 kWh

Early Replacement:

For example, a three ton, 15 SEER, 12EER, 9 HSPF Air Source Heat Pump replaces an existing working Air Source Heat Pump with unknown efficiency ratings in Marion:

ΔkWH for remaining life of existing unit (1st 6 years):

= ((903 * 36,000 * (1/9.12 - 1/15)) / 1000) + ((1,288 * 36,000 * (1/5.44 - 1/9)) / 1000)

= 4769 kWh

ΔkWH for remaining measure life (next 12 years):

= ((903 * 36,000 * (1/14 - 1/15)) / 1000) + ((1,288 * 36,000 * (1/8.2 - 1/9)) / 1000)

= 657 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

Time of sale:

ΔkW = (Capacity_cooling * (1/EER_base - 1/EER_ee)) / 1000) * CF

Early replacement[852]:

ΔkW for remaining life of existing unit (1st 6 years):

= ((Capacity_cooling * (1/EERexist - 1/EERee))/1000 * CF);

ΔkW for remaining measure life (next 12 years):

= ((Capacity_cooling * (1/EERbase - 1/EERee))/1000 * CF)

[851] Based on Minimum Federal Standard effective 1/1/2015; http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf

[852] The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).

Where:

EER_exist = Energy Efficiency Ratio of existing cooling system (kBtu/hr / kW)

= Use actual EER rating where it is possible to measure or reasonably estimate. If EER unknown but SEER available convert using the equation:

EER_base = (-0.02 * SEER_base$^2$) + (1.12 * SEER) [853]

If SEER rating unavailable use:

| Existing Cooling System | EER_exist[854] |
|---|---|
| Air Source Heat Pump | 8.55 |
| Central AC | 8.15 |
| No central cooling[855] | Make '1/EER_exist' = 0 |

EER_base = Energy Efficiency Ratio of baseline Air Source Heat Pump (kBtu/hr / kW)

= 11.8 [856]

EER_ee = Energy Efficiency Ratio of baseline Air Source Heat Pump (kBtu/hr / kW)

= Actual, If not provided convert SEER to EER using this formula:[857]

= (-0.02 * SEER$^2$) + (1.12 * SEER)

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)

= 72%%[858]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Heat Pumps (average during peak period)

= 46.6%[859]

[853] From Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder.
[854] Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.
[855] If there is no central cooling in place but the incentive encourages installation of a new ASHP with cooling, the added cooling load should be subtracted from any heating benefit.
[856] The Federal Standard does not include an EER requirement, so it is approximated with this formula: (-0.02 * SEER2) + (1.12 * SEER) Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder. Note this is appropriate for single speed units only.
[857] Based on Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder. Note this is appropriate for single speed units only.
[858] Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.
[859] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

Time of Sale:

For example, a three ton, 15 SEER, 12EER, 9 HSPF Air Source Heat Pump installed in Marion:

$\Delta kW_{SSP}$ = ((36,000 * (1/11.8 – 1/12)) / 1000) * 0.72

= 0.037 kW

$\Delta kW_{PJM}$ = ((36,000 * (1/11.8 – 1/12)) / 1000) * 0.466

= 0.024 kW

Early Replacement:

For example, a three ton, 15 SEER, 12EER, 9 HSPF Air Source Heat Pump replaces an existing working Air Source Heat Pump with unknown efficiency ratings in Marion:

$\Delta kW_{SSP}$ for remaining life of existing unit (1st 6 years):

= ((36,000 * (1/8.55 - 1/12)) / 1000) * 0.72

= 0.872 kW

$\Delta kW_{SSP}$ for remaining measure life (next 12 years):

= ((36,000 * (1/11.8 – 1/12)) / 1000) * 0.72

= 0.037 kW

**NATURAL GAS SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-HVC-ASHP-V04-150601**

## 5.3.2 Boiler Pipe Insulation

### DESCRIPTION

This measure describes adding insulation to un-insulated boiler pipes in un-conditioned basements or crawlspaces.

This measure was developed to be applicable to the following program types: TOS, RNC, RF, DI.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient case is installing pipe wrap insulation to a length of boiler pipe.

### DEFINITION OF BASELINE EQUIPMENT

The baseline is an un-insulated boiler pipe.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 15 years[860].

### DEEMED MEASURE COST

The measure cost including material and installation is assumed to be $3 per linear foot[861].

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

### NATURAL GAS SAVINGS

$$\Delta Therm = (((1/R_{exist} * C_{exist}) - (1/R_{new} * C_{new})) * FLH_heat * L * \Delta T) / \eta Boiler / 100{,}000$$

[860] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf

[861] Consistent with DEER 2008 Database Technology and Measure Cost Data (www.deeresources.com).

Where:

$R_{exist}$ = Pipe heat loss coefficient of uninsulated pipe (existing) [(hr-°F-ft$^2$)/Btu]

= 0.5[862]

$R_{new}$ = Pipe heat loss coefficient of insulated pipe (new) [(hr-°F-ft$^2$)/Btu]

= Actual (0.5 + R value of insulation)

FLH_heat = Full load hours of heating

= Dependent on location[863]:

| Climate Zone (City based upon) | FLH_heat |
|---|---|
| 1 (Rockford) | 1,969 |
| 2 (Chicago) | 1,840 |
| 3 (Springfield) | 1,754 |
| 4 (Belleville) | 1,266 |
| 5 (Marion) | 1,288 |
| Weighted Average[864] | 1,821 |

L = Length of boiler pipe in unconditioned space covered by pipe wrap (ft)

= Actual

$C_{exist}$ = Circumference of bare pipe (ft) (Diameter (in) * π/12)

= Actual (0.5" pipe = 0.131ft, 0.75" pipe = 0.196ft)

$C_{new}$ = Circumference of pipe with insulation (ft) (Diameter (in) * π/12)

= Actual

ΔT = Average temperature difference between circulated heated water and unconditioned space air temperature (°F) [865]

[862] Assumption based on data obtained from the 3E Plus heat loss calculation software provided by the NAIMA (North American Insulation Manufacturer Association) and derived from Table 15 and Table 16 of 2009 ASHRAE Fundamentals Handbook, Chapter 23 Insulation for Mechanical Systems, page 23.17.

[863] Full load heating hours for heat pumps are provided for Rockford, Chicago and Springfield in the Energy Star Calculator. Estimates for the other locations were calculated based on the FLH to Heating Degree Day (from NCDC) ratio. VEIC consider Energy Star estimates to be high due to oversizing not being adequately addressed. Using average Illinois billing data (from http://www.icc.illinois.gov/ags/consumereducation.aspx) VEIC estimated the average gas heating load and used this to estimate the average home heating output (using 83% average gas heat efficiency). Dividing this by a typical 36,000 Btu/hr ASHP gives an estimate of average ASHP FLH_heat of 1821 hours. We used the ratio of this value to the average of the locations using the Energy Star data (1994 hours) to scale down the Energy Star estimates. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.

[864] Weighted based on number of occupied residential housing units in each zone.

[865] Assumes 160°F water temp for a boiler without reset control, 120°F for a boiler with reset control, and 50°F air temperature

Pipes in unconditioned basement:

| Outdoor reset controls | ΔT (°F) |
|---|---|
| Boiler without reset control | 110 |
| Boiler with reset control | 70 |

Pipes in crawl space:

| Climate Zone (City based upon) | ΔT (°F) | |
|---|---|---|
| | Boiler without reset control | Boiler with reset control |
| 1 (Rockford) | 127 | 87 |
| 2 (Chicago) | 126 | 86 |
| 3 (Springfield) | 122 | 82 |
| 4 (Belleville) | 120 | 80 |
| 5 (Marion) | 120 | 80 |
| Weighted Average[866] | 125 | 85 |

ηBoiler = Efficiency of boiler

= 0.819 [867]

For example, insulating 10 feet of 0.75" pipe with R-3 wrap (0.75" thickness) in a crawl space of a Marion home with a boiler without reset control:

ΔTherm = (((1/0.5 * 0.196) – (1/3.5 * 0.589)) * 10 * 120 * 1288) / 0.819 / 100,000

= 4.2 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

for pipes in unconditioned basements and the following average heating season outdoor temperatures as the air temperature in crawl spaces: Zone 1 – 33.1, Zone 2 – 34.4, Zone 3 – 37.7, Zone 4 – 40.0, Zone 5 – 39.8, Weighted Average – 35.3 (NCDC 1881-2010 Normals, average of monthly averages Nov – Apr for zones 1-3 and Nov-March for zones 4 and 5).

[866] Weighted based on number of occupied residential housing units in each zone.

[867] Average efficiency of boiler units found in Ameren PY3-PY4 data.

**MEASURE CODE: RS-HVC-PINS-V01-130601**

## 5.3.3 Central Air Conditioning > 14.5 SEER

**DESCRIPTION**

This measure characterizes:

a) Time of Sale:
   a. The installation of a new residential sized (<= 65,000 Btu/hr) Central Air Conditioning ducted split system meeting ENERGY STAR efficiency standards presented below. This could relate to the replacement of an existing unit at the end of its useful life, or the installation of a new system in a new home.

b) Early Replacement:
   a. The early removal of an existing residential sized (<= 65,000 Btu/hr) inefficient Central Air Conditioning unit from service, prior to its natural end of life, and replacement with a new ENERGY STAR qualifying unit. Savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life.
   b. The assumption of the existing unit efficiency in the Early Replacement section of this TRM is based upon the average efficiency of units that were classified in Ameren's PY3-PY4 as functioning and SEER <=10. Therefore it is only appropriate to use these Early Replacement assumptions where those conditions are met. The TAC defined "functioning" as the unit is fully operational – providing sufficient space conditioning (i.e. heat exchanger, compressors, pumps work effectively) and/or the cost of repair is under 20% of the new baseline replacement cost. Therefore in order to apply early replacement assumptions the programs should apply the following eligibility criteria: SEER <=10 and cost of any repairs <$190 per ton.
   c. A weighted average early replacement rate is provided for use when the actual baseline early replacement rate is unknown[868].

**Deemed Early Replacement Rates For CAC Units in Combined System Replacement (CSR) Projects**

| Replacement Scenario for the CAC Unit | Deemed Early Replacement Rate |
|---|---|
| Early Replacement Rate for a CAC unit when the CAC unit is the Primary unit in a CSR project | 14% |
| Early Replacement Rate for a CAC unit when the CAC unit is the Secondary unit in a CSR project | 40% |

This measure was developed to be applicable to the following program types: TOS, NC, EREP. If applied to other program types, the measure savings should be verified.

[868] Based upon research from "Home Energy Efficiency Rebate Program GPY2 Evaluation Report" which outlines early replacement rates for both primary and secondary central air cooling (CAC) and residential funaces. The unit (furnace or CAC unit) that initially caused the customer to contact a trade ally is defined as the "primary unit". The furnace or CAC unit that was also replaced but did not initially prompt the customer to contact a trade ally is defined as the "secondary unit". This evaluation used different criteria for early replacement due to the availability of data after the fact; cost of any repairs < $550 and age of unit < 20 years. Report presented to Nicor Gas Company February 27, 2014, available at http://www.ilsag.info/evaluation-documents.html.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a ducted split central air conditioning unit meeting the minimum ENERGY STAR efficiency level standards; 14.5 SEER and 12 EER.

### DEFINITION OF BASELINE EQUIPMENT

The baseline for the Time of Sale measure is based on the current Federal Standard efficiency level; 13 SEER and 11 EER.

The baseline for the early replacement measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and the new baseline as defined above[869] for the remainder of the measure life.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 18 years [870].

Remaining life of existing equipment is assumed to be 6 years[871].

### DEEMED MEASURE COST

Time of sale: The incremental capital cost for this measure is dependent on equipment size and efficiency. Assumed costs per ton of cooling capacity are provided below[872]:

| Efficiency Level | Cost per Ton |
|---|---|
| SEER 14 | $119 |
| SEER 15 | $238 |
| SEER 16 | $357 |
| SEER 17 | $476 |
| SEER 18 | $596 |
| SEER 19 | $715 |
| SEER 20 | $834 |
| SEER 21 | $908 |
| Average | $530 |

Early replacement: The full install cost for this measure is the actual cost of removing the existing unit and

[869] Baseline SEER and EER should be updated when new minimum federal standards become effective.
[870] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf
The "lifespan" of a central air conditioner is about 15 to 20 years (US DOE: http://www.energysavers.gov/your_home/space_heating_cooling/index.cfm/mytopic=12440).
[871] Assumed to be one third of effective useful life
[872] DEER 2008 Database Technology and Measure Cost Data (www.deeresources.com)

installing the new one. If this is unknown, assume $3,413[873].

Assumed deferred cost (after 6 years) of replacing existing equipment with new baseline unit is assumed to be $2,857[874]. This cost should be discounted to present value using the utilities' discount rate.

**LOADSHAPE**

Loadshape R08 - Residential Cooling

**COINCIDENCE FACTOR**

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)
= 68%[875]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)
= 46.6%[876]

[873] Based on 3 ton initial cost estimate for an ENERGY STAR unit from ENERGY STAR Central AC calculator (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls).
[874] Based on 3 ton initial cost estimate for a conventional unit from ENERGY STAR Central AC calculator (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls). While baselines are likely to shift in the future, there is currently no good indication of what the cost of a new baseline unit will be in 6 years. In the absence of this information, assuming a constant federal baseline cost is within the range of error for this prescriptive measure.
[875] Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.
[876] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

Time of sale:

ΔkWH = (FLHcool * Btu/hr * (1/SEERbase - 1/SEERee))/1000

Early replacement[877]:

ΔkWH for remaining life of existing unit (1st 6 years):

=((FLHcool * Capacity * (1/SEERexist - 1/SEERee))/1000);

ΔkWH for remaining measure life (next 12 years):

= ((FLHcool * Capacity * (1/SEERbase - 1/SEERee))/1000)

Where:

FLHcool = Full load cooling hours

= dependent on location and building type[878]:

| Climate Zone (City based upon) | FLHcool (single family) | FLHcool (multi family) |
|---|---|---|
| 1 (Rockford) | 512 | 467 |
| 2 (Chicago) | 570 | 506 |
| 3 (Springfield) | 730 | 663 |
| 4 (Belleville) | 1035 | 940 |
| 5 (Marion) | 903 | 820 |
| Weighted Average[879] | 629 | 564 |

[877] The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).

[878] Full load hours for Chicago, Moline and Rockford are provided in "Final Evaluation Report: Central Air Conditioning Efficiency Services (CACES), 2010, Navigant Consulting", http://ilsag.org/yahoo_site_admin/assets/docs/ComEd_PY2_CACES_Evaluation_Report_2010-10-18.299122020.pdf, p.33. An average FLH/Cooling Degree Day (from NCDC) ratio was calculated for these locations and applied to the CDD of the other locations in order to estimate FLH. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.

Capacity = Size of new equipment in Btu/hr (note 1 ton = 12,000Btu/hr)

= Actual installed, or if actual size unknown 33,600Btu/hr for single-family buildings[880]

SEERbase = Seasonal Energy Efficiency Ratio of baseline unit (kBtu/kWh)

= 13[881]

SEERexist = Seasonal Energy Efficiency Ratio of existing unit (kBtu/kWh)

= Use actual SEER rating where it is possible to measure or reasonably estimate. If unknown assume 10.0[882].

SEERee = Seasonal Energy Efficiency Ratio of ENERGY STAR unit (kBtu/kWh)

= Actual installed or 14.5 if unknown

Time of sale example: a 3 ton unit with SEER rating of 14.5, in unknown location:

ΔkWH = (629 * 36,000 * (1/13 – 1/14.5)) / 1000

= 180 kWh

Early replacement example: a 3 ton unit, with SEER rating of 14.5 replaces an existing unit in unknown location:

ΔkWH(for first 6 years) = (629 * 36,000 * (1/10 – 1/14.5)) / 1000

= 702 kWh

ΔkWH(for next 12 years) = (629 * 36,000 * (1/13 – 1/14.5)) / 1000

= 180 kWh

Therefore savings adjustment of 26% (180/702) after 6 years.

[879] Weighted based on number of residential occupied housing units in each zone.

[880] Actual unit size required for multi-family building, no size assumption provided because the unit size and resulting savings can vary greatly depending on the number of units.

[881] Based on Minimum Federal Standard; http://www1.eere.energy.gov/buildings/appliance_standards/residential/residential_cac_hp.html.

[882] VEIC estimate based on Department of Energy Federal Standard between 1992 and 2006. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

Time of sale:

ΔkW = (Capacity * (1/EERbase - 1/EERee))/1000 * CF

Early replacement[883]:

ΔkW for remaining life of existing unit (1st 6 years):

= ((Capacity * (1/EERexist - 1/EERee))/1000 * CF);

ΔkW for remaining measure life (next 12 years):

= ((Capacity * (1/EERbase - 1/EERee))/1000 * CF)

Where:

EERbase = EER Efficiency of baseline unit

= 11.2 [884]

EERexist = EER Efficiency of existing unit

= Actual EER of unit should be used, if EER is unknown, use 9.2[885]

EERee = EER Efficiency of ENERGY STAR unit

= Actual installed or 12 if unknown

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)

= 68%[886]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)

= 46.6%[887]

[883] The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).

[884] The federal Standard does not currently include an EER component. The value is approximated based on the SEER standard (13) and equals EER 11.2. To perform this calculation we are using this formula: (-0.02 * SEER2) + (1.12 * SEER) (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder).

[885] Based on SEER of 10,0, using formula above to give 9.2 EER.

[886] Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

[887] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

Time of sale example: a 3 ton unit with EER rating of 12:

$\Delta kW_{SSP}$ = (36,000 * (1/11.2– 1/12)) / 1000 * 0.68

= 0.146 kW

$\Delta kW_{PJM}$ = (36,000 * (1/11.2– 1/12)) / 1000 * 0.466

= 0.100 kW

Early replacement example: a 3 ton unit with EER rating of 12 replaces an existing unit:

$\Delta kW_{SSP}$ (for first 6 years) = (36,000 * (1/9.2– 1/12)) / 1000 * 0.68

= 0.621 kW

$\Delta kW_{SSP}$ (for next 12 years) = (36,000 * (1/11.2– 1/12)) / 1000 * 0.68

= 0.146 kW

$\Delta kW_{PJM}$ (for first 6 years) = (36,000 * (1/9.2– 1/12)) / 1000 * 0.466

= 0.425 kW

$\Delta kW_{PJM}$ (for next 12 years)= (36,000 * (1/11.2– 1/12)) / 1000 * 0.466

= 0.100 kW

### NATURAL GAS SAVINGS

N/A

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: RS-HVC-CAC1-V04-150601

## 5.3.4 Duct Insulation and Sealing

### DESCRIPTION

This measure describes evaluating the savings associated with performing duct sealing using mastic sealant or metal tape to the distribution system of homes with either central air conditioning or a ducted heating system.

Two methodologies for estimating the savings associate from sealing the ducts are provided. The first preferred method requires the use of a blower door and the second requires careful inspection of the duct work.

1. **Modified Blower Door Subtraction** – this technique is described in detail on p.44 of the Energy Conservatory Blower Door Manual; which can be found on the Energy Conservatory website (As of Oct 2014: http://www.energyconservatory.com/sites/default/files/documents/mod_3-4_dg700_-_new_flow_rings_-_cr_-_tpt_-_no_fr_switch_manual_ce_0.pdf)

2. **Evaluation of Distribution Efficiency** – this methodology requires the evaluation of three duct characteristics below, and use of the Building Performance Institutes 'Distribution Efficiency Look-Up Table';

   http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf

   a. Percentage of duct work found within the conditioned space

   b. Duct leakage evaluation

   c. Duct insulation evaluation

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is sealed duct work throughout the unconditioned space in the home.

### DEFINITION OF BASELINE EQUIPMENT

The existing baseline condition is leaky duct work within the unconditioned space in the home.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of this measure is 20 years[888].

### DEEMED MEASURE COST

The actual duct sealing measure cost should be used.

[888] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf

### LOADSHAPE

Loadshape R08 - Residential Cooling

Loadshape R09 - Residential Electric Space Heat

Loadshape R10 - Residential Electric Heating and Cooling (Shell Measures)

### COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during utility peak hour)

= 68%[889]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)

= 46.6%[890]

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

***Methodology 1: Modified Blower Door Subtraction***

a) Determine Duct Leakage rate before and after performing duct sealing:
Duct Leakage ($CFM50_{DL}$) = ($CFM50_{Whole\ House}$ – $CFM50_{Envelope\ Only}$) * SCF

Where:

$CFM50_{Whole\ House}$ = Standard Blower Door test result finding Cubic Feet per Minute at 50 Pascal pressure differential

$CFM50_{Envelope\ Only}$ = Blower Door test result finding Cubic Feet per Minute at 50 Pascal pressure differential with all supply and return registers sealed.

SCF = Subtraction Correction Factor to account for underestimation of duct leakage due to connections between the duct system and the home. Determined by measuring pressure in duct system with registers sealed and using look up table provided by Energy Conservatory.

[889] Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.
[890] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

b) Calculate duct leakage reduction, convert to $CFM25_{DL}$ and factor in Supply and Return Loss Factors
Duct Leakage Reduction ($\Delta CFM25_{DL}$) = (Pre $CFM50_{DL}$ – Post $CFM50_{DL}$) * 0.64 * (SLF + RLF)

Where:

0.64 = Converts CFM50 to CFM25[891]

SLF = Supply Loss Factor

= % leaks sealed located in Supply ducts * 1 [892]

Default = 0.5[893]

RLF = Return Loss Factor

= % leaks sealed located in Return ducts * 0.5[894]

Default = 0.25[895]

c) Calculate Electric Energy Savings:

$\Delta kWh = \Delta kWh_{cooling} + \Delta kWh_{Fan}$

$\Delta kWh_{cooling}$ = (($\Delta CFM25_{DL}$)/ ((CapacityCool/12,000) * 400)) * FLHcool * CapacityCool) / 1000 / ηCool

$\Delta kWh_{Fan}$ = (ΔTherms * $F_e$ * 29.3)

Where:

$\Delta CFM25_{DL}$ = Duct leakage reduction in CFM25

= calculated above

[891] 25 Pascals is the standard assumption for typical pressures experienced in the duct system under normal operating conditions. To convert CFM50 to CFM25 you multiply by 0.64 (inverse of the "Can't Reach Fifty" factor for CFM25; see Energy Conservatory Blower Door Manual).

[892] Assumes that for each percent of supply air loss there is one percent annual energy penalty. This assumes supply side leaks are direct losses to the outside and are not recaptured back to the house. This could be adjusted downward to reflect regain of usable energy to the house from duct leaks. For example, during the winter some of the energy lost from supply leaks in a crawlspace will probably be regained back to the house (sometimes 1/2 or more may be regained). More information provided in "Appendix E Estimating HVAC System Loss From Duct Airtightness Measurements" from http://www.energyconservatory.com/download/dbmanual.pdf

[893] Assumes 50% of leaks are in supply ducts.

[894] Assumes that for each percent of return air loss there is a half percent annual energy penalty. Note that this assumes that return leaks contribute less to energy losses than do supply leaks. This value could be adjusted upward if there was reason to suspect that the return leaks contribute significantly more energy loss than "average" (e.g. pulling return air from a super heated attic), or can be adjusted downward to represent significantly less energy loss (e.g. pulling return air from a moderate temperature crawl space) . More information provided in "Appendix E Estimating HVAC System Loss From Duct Airtightness Measurements" from http://www.energyconservatory.com/download/dbmanual.pdf

[895] Assumes 50% of leaks are in return ducts.

CapacityCool = Capacity of Air Cooling system (Btu/hr)

=Actual

12,000 = Converts Btu/H capacity to tons

400 = Converts capacity in tons to CFM (400CFM / ton)[896]

FLHcool = Full load cooling hours

= Dependent on location as below[897]:

| Climate Zone (City based upon) | FLHcool Single Family | FLHcool Multifamily |
|---|---|---|
| 1 (Rockford) | 512 | 467 |
| 2 (Chicago) | 570 | 506 |
| 3 (Springfield) | 730 | 663 |
| 4 (Belleville) | 1,035 | 940 |
| 5 (Marion) | 903 | 820 |
| Weighted Average898 | 629 | 564 |

1000 = Converts Btu to kBtu

ηCool = Efficiency (SEER) of Air Conditioning equipment (kBtu/kWh)

= Actual. If unknown assume the following[899]:

| Age of Equipment | SEER Estimate |
|---|---|
| Before 2006 | 10 |
| After 2006 - 2014 | 13 |
| Central AC After 1/1/2015 | 13 |
| Heat Pump After 1/1/2015 | 14 |

[896] This conversion is an industry rule of thumb; e.g. see http://www.hvacsalesandsupply.com/Linked%20Documents/Tech%20Tips/61-Why%20400%20CFM%20per%20ton.pdf

[897] Based on Full Load Hours from ENERGY Star with adjustments made in a Navigant Evaluation, other cities were scaled using those results and CDD. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.

[898] Weighted based on number of occupied residential housing units in each zone.

[899] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

ΔTherms = Therm savings as calculated in Natural Gas Savings

$F_e$ = Furnace Fan energy consumption as a percentage of annual fuel consumption

= 3.14%[900]

29.3 = kWh per therm

For example, duct sealing in a single family house in Springfield with a 36,000 Btu/H, SEER 11 central air conditioning, an 80% AFUE, 105,000 Btu/H natural gas furnace and the following blower door test results:

Before: $CFM50_{Whole\ House}$ = 4800 CFM50

$CFM50_{Envelope\ Only}$ = 4500 CFM50

House to duct pressure of 45 Pascals. = 1.29 SCF (Energy Conservatory look up table)

After: $CFM50_{Whole\ House}$ = 4600 CFM50

$CFM50_{Envelope\ Only}$ = 4500 CFM50

House to duct pressure of 43 Pascals = 1.39 SCF (Energy Conservatory look up table)

Duct Leakage:

$CFM50_{DL\ before}$ = (4800 – 4500) * 1.29

= 387 CFM

$CFM50_{DL\ after}$ = (4600 – 4500) * 1.39

= 139 CFM

Duct Leakage reduction at CFM25:

$\Delta CFM25_{DL}$ = (387 – 139) * 0.64 * (0.5 + 0.25)

= 119 CFM25

Energy Savings:

$\Delta kWh_{cooling}$ = [((119 / ((36,000/12,000) * 400)) * 730 * 36,000) / 1000 / 11] + (212 * 0.0314 * 29.3)

= 237 + 195

= 432 kWh

[900] $F_e$ is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% $F_e$. See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

Heating savings for homes with electric heat (Heat Pump):

$\Delta kWh_{heating}$ = (((ΔCFM25$_{DL}$ /((OutputCapacityHeat/12,000) * 400)) * FLHheat * OutputCapacityHeat) / ηHeat / 3412

Where:

OutputCapacityHeat = Heating output capacity (Btu/hr) of electric heat

=Actual

FLHheat = Full load heating hours

= Dependent on location as below[901]:

| Climate Zone (City based upon) | FLH_heat |
|---|---|
| 1 (Rockford) | 1,969 |
| 2 (Chicago) | 1,840 |
| 3 (Springfield) | 1,754 |
| 4 (Belleville) | 1,266 |
| 5 (Marion) | 1,288 |
| Weighted Average[902] | 1,821 |

ηHeat = Efficiency in COP of Heating equipment

= Actual. If not available use[903]:

| System Type | Age of Equipment | HSPF Estimate | COP Estimate |
|---|---|---|---|
| Heat Pump | Before 2006 | 6.8 | 2.00 |
| | After 2006 - 2014 | 7.7 | 2.26 |
| | 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1.00 |

[901] Heating EFLH based on ENERGY Star EFLH for Rockford, Chicago, and Springfield and on NCDC/NOAA HDD for the other two cities. In all cases, the hours were adjusted based on average natural gas heating consumption in IL.
[902] Weighted based on number of occupied residential housing units in each zone.
[903] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

3412 = Converts Btu to kWh

For example, duct sealing in a 36,000 Btu/H 2.5 COP heat pump heated single family house in Springfield with the blower door results described above:

$\Delta kWh_{heating}$ = (((119 / ((36,000/12,000) * 400)) * 1,754 * 36,000) / 2.5 / 3412

= 734 kWh

***Methodology 2: Evaluation of Distribution Efficiency***

Determine Distribution Efficiency by evaluating duct system before and after duct sealing using Building Performance Institute "Distribution Efficiency Look-Up Table"

$\Delta kWh = (((DE_{after} - DE_{before})/ DE_{after})) * FLHcool * CapacityCool)/1000 / \eta Cool) + (\Delta Therms * F_e * 29.3)$

Where:

$DE_{after}$ = Distribution Efficiency after duct sealing

$DE_{before}$ = Distribution Efficiency before duct sealing

FLHcool = Full load cooling hours

= Dependent on location as below[904]:

| Climate Zone (City based upon) | FLHcool Single Family | FLHcool Multifamily |
|---|---|---|
| 1 (Rockford) | 512 | 467 |
| 2 (Chicago) | 570 | 506 |
| 3 (Springfield) | 730 | 663 |
| 4 (Belleville) | 1,035 | 940 |
| 5 (Marion) | 903 | 820 |
| Weighted Average[905] | 629 | 564 |

CapacityCool = Capacity of Air Cooling system (Btu/hr)

[904] Based on Full Load Hours from ENERGY Star with adjustments made in a Navigant Evaluation, other cities were scaled using those results and CDD. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.

[905] Weighted based on number of occupied residential housing units in each zone.

=Actual

1000 = Converts Btu to kBtu

ηCool = Efficiency (SEER) of Air Conditioning equipment (kBtu/kWh)

= Actual. If unknown assume[906]:

| Age of Equipment | SEER Estimate |
|---|---|
| Before 2006 | 10 |
| After 2006 - 2014 | 13 |
| Central AC After 1/1/2015 | 13 |
| Heat Pump After 1/1/2015 | 14 |

For example, duct sealing in a single family house in Springfield, with 36,000 Btu/H SEER 11 central air conditioning, an 80% AFUE, 105,000 Btu/H natural gas furnace and the following duct evaluation results:

$DE_{before}$ = 0.85

$DE_{after}$ = 0.92

Energy Savings:

$\Delta kWh_{cooling}$ = (((0.92 – 0.85)/0.92) * 730 * 36,000) / 1000 / 11) + (212 * 0.0314 * 29.3)

= 182 + 195

= 377 kWh

<u>Heating savings for homes with electric heat (Heat Pump):</u>

$\Delta kWh_{heating}$ = (($DE_{after}$ – $DE_{before}$)/ $DE_{after}$)) * FLHheat * OutputCapacityHeat) / ηHeat / 3412

Where:

OutputCapacityHeat = Heating output capacity (Btu/hr) of the electric heat

=Actual

FLHheat = Full load heating hours

= Dependent on location as below[907]:

[906] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

[907] Heating EFLH based on ENERGY Star EFLH for Rockford, Chicago, and Springfield and on NCDC/NOAA HDD for the other two

| Climate Zone (City based upon) | FLH_heat |
|---|---|
| 1 (Rockford) | 1,969 |
| 2 (Chicago) | 1,840 |
| 3 (Springfield) | 1,754 |
| 4 (Belleville) | 1,266 |
| 5 (Marion) | 1,288 |
| Weighted Average[908] | 1,821 |

COP = Coefficient of Performance of electric heating system[909]

= Actual. If not available use[910]:

| System Type | Age of Equipment | HSPF Estimate | COP Estimate |
|---|---|---|---|
| Heat Pump | Before 2006 | 6.8 | 2.00 |
| | After 2006 - 2014 | 7.7 | 2.26 |
| | 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1.00 |

For example, duct sealing in a 36,000 Btu/H, 2.5 COP heat pump heated single family house in Springfield with the following duct evaluation results:

$DE_{after}$ = 0.92

$DE_{before}$ = 0.85

Energy Savings:

$\Delta kWh_{heating}$ = ((0.92 – 0.85)/0.92) * 1,967 * 36,000) / 2.5) / 3412

= 632 kWh

cities. In all cases, the hours were adjusted based on average natural gas heating consumption in IL.

[908] Weighted based on number of occupied residential housing units in each zone.

[909] Note that the HSPF of a heat pump is equal to the COP * 3.413.

[910] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$\Delta kW$ = $\Delta kWh_{cooling}$/ FLHcool * CF

Where:

FLHcool = Full load cooling hours:

= Dependent on location as below[911]:

| Climate Zone (City based upon) | FLHcool Single Family | FLHcool Multifamily |
|---|---|---|
| 1 (Rockford) | 512 | 467 |
| 2 (Chicago) | 570 | 506 |
| 3 (Springfield) | 730 | 663 |
| 4 (Belleville) | 1,035 | 940 |
| 5 (Marion) | 903 | 820 |
| Weighted Average[912] | 629 | 564 |

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)

= 68%[913]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)

= 46.6%[914]

**NATURAL GAS SAVINGS**

<u>For homes with Natural Gas Heating:</u>

***Methodology 1: Modified Blower Door Subtraction***

$\Delta Therm$ = $(((\Delta CFM25_{DL} / (InputCapacityHeat * 0.0123)) * FLHheat * InputCapacityHeat * (\eta Equipment / \eta System)) / 100{,}000$

Where:

[911] Based on Full Load Hours from ENERGY Star with adjustments made in a Navigant Evaluation, other cities were scaled using those results and CDD. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.
[912] Weighted based on number of occupied residential housing units in each zone.
[913] Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.
[914] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

$\Delta CFM25_{DL}$ = Duct leakage reduction in CFM25

InputCapacityHeat = Heating input capacity (Btu/hr)

=Actual

0.0123 = Conversion of Capacity to CFM (0.0123CFM / Btu/hr)[915]

FLHheat = Full load heating hours

=Dependent on location as below[916]:

| Climate Zone (City based upon) | FLH_heat |
|---|---|
| 1 (Rockford) | 1,969 |
| 2 (Chicago) | 1,840 |
| 3 (Springfield) | 1,754 |
| 4 (Belleville) | 1,266 |
| 5 (Marion) | 1,288 |
| Weighted Average[917] | 1,821 |

100,000 = Converts Btu to therms

ηEquipment = Heating Equipment Efficiency

= Actual[918]. If not available use 83%[919]

[915] Based on Natural Draft Furnaces requiring 100 CFM per 10,000 Btu, Induced Draft Furnaces requiring 130CFM per 10,000Btu and Condensing Furnaces requiring 150 CFM per 10,000 Btu (rule of thumb from http://contractingbusiness.com/enewsletters/cb_imp_43580/). Data provided by GAMA during the federal rule-making process for furnace efficiency standards, suggested that in 2000, 24% of furnaces purchased in Illinois were condensing units. Therefore a weighted average required airflow rate is calculated assuming a 50:50 split of natural v induced draft non-condensing furnaces, as 123 per 10,000Btu or 0.0123/Btu.

[916] Heating EFLH based on ENERGY Star EFLH for Rockford, Chicago, and Springfield and on NCDC/NOAA HDD for the other two cities. In all cases, the hours were adjusted based on average natural gas heating consumption in IL.

[917] Weighted based on number of occupied residential housing units in each zone.

[918] The Equipment Efficiency can be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test.
If there are more than one heating systems, the weighted (by consumption) average efficiency should be used.
If the heating system or distribution is being upgraded within a package of measures together with the insulation upgrade, the new average heating system efficiency should be used.

[919] This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls)) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to

ηSystem = Pre duct sealing Heating System Efficiency (Equipment Efficiency * Pre Distribution Efficiency)[920]

= Actual. If not available use 70%[921]

For example, duct sealing in a house in Springfield with an 80% AFUE, 105,000 Btu/H (input capacity) natural gas furnace and the following blower door test results:

Before: $CFM50_{Whole\ House}$ = 4800 CFM50

$CFM50_{Envelope\ Only}$ = 4500CFM50

House to duct pressure of 45 Pascals = 1.29 SCF (Energy Conservatory look up table)

After: $CFM50_{Whole\ House}$ = 4600 CFM50

$CFM50_{Envelope\ Only}$ = 4500CFM50

House to duct pressure of 43 Pascals = 1.39 SCF (Energy Conservatory look up table)

Duct Leakage:

$CFM50_{DL\ before}$ = (4800 – 4500) * 1.29

= 387 CFM

$CFM50_{DL\ after}$ = (4600 – 4500) * 1.39

= 119 CFM

Duct Leakage reduction at CFM25:

$\Delta CFM25_{DL}$ = (387 – 139) * 0.64 * (0.5 + 0.25)

= 119 CFM25

Energy Savings:

Pre Distribution Efficiency = 1 – (387/4800) = 92%

ηSystem = 80% * 92% = 74%

ΔTherm = ((119/ (105,000 * 0.0123)) * 1,754 * 105,000 * (0.8/0.74)) / 100,000

= 183 therms

last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:
(0.24*0.92) + (0.76*0.8) = 0.829

[920] The Distribution Efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute: (http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf) or by performing duct blaster testing.

[921] Estimated as follows: 0.829 * (1-0.15) = 0.70

***Methodology 2: Evaluation of Distribution Efficiency***

$$\Delta Therm = ((DE_{after} - DE_{before}) / DE_{after})) * FLHheat * InputCapacityHeat\ (\eta Equipment / \eta System)) / 100{,}000$$

Where:

$DE_{after}$ = Distribution Efficiency after duct sealing

$DE_{before}$ = Distribution Efficiency before duct sealing

Other variables as defined above

For example, duct sealing in a house in Springfield an 80% AFUE, 105,000 Btu/H (input capacity) natural gas furnace and the following duct evaluation results:

$DE_{after}$ = 0.92

$DE_{before}$ = 0.85

Energy Savings:

ηSystem = 80% * 85% = 68%

ΔTherm = ((0.92 – 0.85)/0.92) * 1,754 * 105,000 * (0.8/0.68)) / 100,000

= 164 therm

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-HVC-DINS-V05-150601**

## 5.3.5 Furnace Blower Motor

### DESCRIPTION

A new furnace with a brushless permanent magnet (BPM) blower motor is installed instead of a new furnace with a lower efficiency motor. This measure characterizes only the electric savings associated with the fan and could be coupled with gas savings associated with a more efficient furnace. Savings decrease sharply with static pressure so duct improvements, and clean, low pressure drop filters can maximize savings. Savings improve when the blower is used for cooling as well and when it is used for continuous ventilation, but only if the non-BPM motor would have been used for continuous ventilation too. If the resident runs the BPM blower continuously because it is a more efficient motor and would not run a non-BPM motor that way, savings are near zero and possibly negative. This characterization uses a 2009 Focus on Energy study of BPM blower motor savings in Wisconsin, which accounted for the effects of this behavioral impact.

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

A furnace with a brushless permanent magnet (BPM) blower motor, also known by the trademark ECM, BLDC, and other names.

### DEFINITION OF BASELINE EQUIPMENT

A furnace with a non-BPM blower motor.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 20 years[922].

### DEEMED MEASURE COST

The capital cost for this measure is assumed to be $97[923].

### LOADSHAPE

Loadshape R08 - Residential Cooling

Loadshape R09 - Residential Electric Space Heat

Loadshape R10 - Residential Electric Heating and Cooling

### COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid

[922] Consistent with assumed life of a new gas furnace. Table 8.3.3 The Technical support documents for federal residential appliance standards: http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/chapter_8.pdf
[923] Adapted from Tables 8.2.3 and 8.2.13 in http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/hvac_ch_08_lcc_2011-06-24.pdf

into PJM's Forward Capacity Market.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during utility peak hour)

= 68%[924]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)

= 46.6%[925]

**Algorithm**

**CALCULATION OF SAVINGS**

**ELECTRIC ENERGY SAVINGS**

ΔkWh = Heating Savings + Cooling Savings + Shoulder Season Savings

Where:

Heating Savings = Blower motor savings during heating season

= 418 kWh[926]

Cooling Savings = Blower motor savings during cooling season

If Central AC = 263 kWh

If No Central AC = 175 kWh

If unknown (weighted average)

= 241 kWh[927]

Shoulder Season Savings = Blower motor savings during shoulder seasons

= 51 kWh

For example, a blower motor in a home where Central AC presence is unknown:

ΔkWh = Heating Savings + Cooling Savings + Shoulder Season Savings

= 418 +263 + 51

= 732 kWh

[924] Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

[925] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

[926] To estimate heating, cooling and shoulder season savings for Illinois, VEIC adapted results from a 2009 Focus on Energy study of BPM blower motor savings in Wisconsin. This study included effects of behavior change based on the efficiency of new motor greatly increasing the amount of people that run the fan continuously. The savings from the Wisconsin study were adjusted to account for different run hour assumptions (average values used) for Illinois. See: FOE to IL Blower Savings.xlsx.

[927] The weighted average value is based on assumption that 75% of homes installing BPM furnace blower motors have Central AC. 66% of IL housing units have CAC and 66% have gas furnaces. It is logical these two groups overlap to a large extent (like the 95% in the FOE study above).

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = Cooling Savings / FLH_cooling * CF

Where:

FLH_cooling = Full load hours of air conditioning

= Dependent on location[928]:

| Climate Zone (City based upon) | FLH_cooling |
|---|---|
| 1 (Rockford) | 512 |
| 2 (Chicago) | 570 |
| 3 (Springfield) | 730 |
| 4 (Belleville) | 1,035 |
| 5 (Marion) | 903 |
| Weighted Average[929] | 629 |

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)

= 68%[930]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)

= 46.6%[931]

For example, a blower motor in a home of unknown location where Central AC prevalence is unknown:

$\Delta kW_{SSP}$ = 251 / 629 * 0.68

= 0.271 kW

$\Delta kW_{SSP}$ = 251 / 629 * 0.466

= 0.186 kW

[928] Full load hours for Chicago, Moline and Rockford are provided in "Final Evaluation Report: Central Air Conditioning Efficiency Services (CACES), 2010, Navigant Consulting", http://ilsag.org/yahoo_site_admin/assets/docs/ComEd_PY2_CACES_Evaluation_Report_2010-10-18.299122020.pdf, p.33. An average FLH/Cooling Degree Day (from NCDC) ratio was calculated for these locations and applied to the CDD of the other locations in order to estimate FLH. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.

[929] Weighted based on number of occupied residential housing units in each zone.

[930] Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

[931] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

### NATURAL GAS SAVINGS

Δtherms[932] = - Heating Savings * 0.03412/ AFUE

Where:

0.03412 = Converts kWh to therms

AFUE = Efficiency of the Furnace

= Actual. If unknown assume 95%[933] if in new furnace or 64.4 AFUE% [934] if in existing furnace

Using defaults:

For new Furnace = - (418 * 0.03412) / 0.95

= - 15.0 therms

For existing Furnace = - (418 * 0.03412) / 0.644

= - 22.1 therms

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: RS-HVC-FBMT-V03-150601

[932] The blower fan is in the heating duct so all, or very nearly all, of its waste heat is delivered to the conditioned space. Negative value since this measure will increase the heating load due to reduced waste heat.
[933] Minimum ENERGY STAR efficiency after 2.1.2012.
[934] Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

## 5.3.6 Gas High Efficiency Boiler

### DESCRIPTION

High efficiency boilers achieve most gas savings through the utilization of a sealed combustion chamber and multiple heat exchangers that remove a significant portion of the waste heat from flue gasses. Because multiple heat exchangers are used to remove waste heat from the escaping flue gasses, some of the flue gasses condense and must be drained.

This measure characterizes:

a) Time of Sale:
   a. The installation of a new high efficiency, gas-fired hot water boiler in a residential location. This could relate to the replacement of an existing unit at the end of its useful life, or the installation of a new system in a new home.

b) Early Replacement:
   a. The early removal of an existing functional AFUE 75% or less boiler from service, prior to its natural end of life, and replacement with a new high efficiency unit. Savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life.

   b. The assumption of the existing unit efficiency in the Early Replacement section of this TRM is based upon the average efficiency of units that were classified in Ameren's PY3-PY4 as functioning and AFUE <=75%. Therefore it is only appropriate to use these Early Replacement assumptions where those conditions are met. The TAC defined "functioning" as the unit is fully operational – providing sufficient space conditioning (i.e. heat exchanger, compressors, pumps work effectively) and/or the cost of repair is under 20% of the new baseline replacement cost. Therefore in order to apply early replacement assumptions the programs should apply the following eligibility criteria: AFUE <=75% and cost of any repairs <$709.
   c. A weighted average early replacement rate is provided for use when the actual baseline early replacement rates are unknown[935].

**Deemed Early Replacement Rates For Boilers**

| | Deemed Early Replacement Rate |
|---|---|
| Early Replacement Rate for Boiler participants | 7% |

This measure was developed to be applicable to the following program types: TOS, NC, EREP. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed Boiler must be ENERGY STAR qualified (AFUE rated at or greater than 85%

[935] Based upon research from "Home Energy Efficiency Rebate Program GPY2 Evaluation Report" which outlines early replacement rates for both primary and secondary central air cooling (CAC) and residential furnaces. This is used as a reasonable proxy for boiler installations since boiler specific data is not available. Report presented to Nicor Gas Company February 27, 2014, available at http://www.ilsag.info/evaluation-documents.html.

and input capacity less than 300,000 Btu/hr).

### DEFINITION OF BASELINE EQUIPMENT

Time of sale: The baseline equipment for this measure is a new, gas-fired, standard-efficiency water boiler. The current Federal Standard minimum AFUE rating is 80%. For boilers manufactured after September 2012 the Federal Standards is raised to 82% AFUE. Baseline assumptions are therefore provided below:

| Program Year | AFUE |
|---|---|
| June 2012 – May 2013[936] | 80% |
| June 2013 on | 82% |

Early replacement: The baseline for this measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 25 years[937].

Early replacement: Remaining life of existing equipment is assumed to be 8 years[938].

### DEEMED MEASURE COST

Time of sale: The incremental install cost for this measure is dependent on tier[939]:

| Measure Type | Installation Cost | Incremental Install Cost (June 2012 – May 2013) | Incremental Install Cost (June 2013 on) |
|---|---|---|---|
| AFUE 80% | $3334 | n/a | |
| AFUE 82% | $3543 | | |
| AFUE 85% (Energy Star Minimum) | $4268 | $934 | $725 |
| AFUE 90% | $4815 | $1,481 | $1,272 |
| AFUE 95% | $5328 | $1,994 | $1,785 |

[936] There will be some delay to the baseline shift while existing stocks of lower efficiency equipment is sold.
[937] Table 8.3.3 The Technical support documents for federal residential appliance standards: http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/chapter_8.pdf
[938] Assumed to be one third of effective useful life
[939] Based on data provided in Appendix E of the Appliance Standards Technical Support Documents including equipment cost and installation labor (http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/appendix_e.pdf). Where efficiency ratings are not provided, the values are interpolated from those that are.

Early Replacement: The full installation cost is provided in the table above. The assumed deferred cost (after 8 years) of replacing existing equipment with a new baseline unit is assumed to be $3543. This cost should be discounted to present value using the utilities' discount rate.

**LOADSHAPE**

N/A

**COINCIDENCE FACTOR**

N/A

**Algorithm**

**CALCULATION OF SAVINGS**

**ELECTRIC ENERGY SAVINGS**

N/A

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

N/A

**NATURAL GAS SAVINGS**

Time of Sale:

ΔTherms = Gas_Boiler_Load * (1/AFUE(base) - 1/AFUE(eff))

Early replacement[940]:

ΔTherms for remaining life of existing unit (1st 8 years):

= Gas_Boiler_Load * (1/AFUE(exist) - 1/AFUE(eff)))

ΔTherms for remaining measure life (next 17 years):

= Gas_Boiler_Load * (1/AFUE(base) - 1/AFUE(eff)))

Where:

Gas_Boiler_Load[941] = Estimate of annual household Load for gas boiler heated single-family homes.

[940] The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).

[941] Boiler consumption values are informed by an evaluation which did not identify any fraction of heating load due to domestic hot water (DHW) provided by the boiler. Thus these values are an average of both homes with boilers only providing heat, and homes with boilers that also provide DHW. Heating load is used to describe the household heating need, which is equal to (gas heating consumption * AFUE )

If location is unknown, assume the average below[942].

= or Actual if informed by site-specific load calculations, ACCA Manual J or equivalent[943].

| Climate Zone (City based upon) | Gas_Boiler Load (therms) |
|---|---|
| 1 (Rockford) | 1275 |
| 2 (Chicago) | 1218 |
| 3 (Springfield) | 1043 |
| 4 (Belleville) | 805 |
| 5 (Marion) | 819 |
| Average | 1158 |

AFUE(exist) = Existing Boiler Annual Fuel Utilization Efficiency Rating

= Use actual AFUE rating where it is possible to measure or reasonably estimate.

If unknown, assume 61.6 AFUE% [944].

AFUE(base) = Baseline Boiler Annual Fuel Utilization Efficiency Rating

= Dependent on year as listed below:

| Program Year | AFUE(base) |
|---|---|
| June 2012 – May 2013 | 80% |
| June 2013 on | 82% |

AFUE(eff) = Efficent Boiler Annual Fuel Utilization Efficiency Rating

= Actual. If unknown, use defaults dependent[945] on tier as listed below:

| Measure Type | AFUE(eff) |
|---|---|
| ENERGY STAR® | 87.5% |
| AFUE 90% | 92.5% |

[942] Values are based on household heating consumption values and inferred average AFUE results from Table 3-4, Program Sample Analysis, *Nicor R29 Res Rebate Evaluation Report 092611_REV FINAL to Nicor*). Adjusting to a statewide average using relative HDD values to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

[943] The Air Conditioning Contractors of America Manual J, Residential Load Calculation 8th Edition produces equipment sizing loads for Single Family, Multi-single, and Condominiums using input characteristics of the home. A best practice for equipment selection and installation of Heating and Air Conditioning, load calculations should be completed by contractors during the selection process and may be readily available for program data purposes.

[944] Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

[945] Default values per tier selected based upon the average AFUE value for the tier range except for the top tier where the minimum is used due to proximity to the maximum possible.

| Measure Type | AFUE(eff) |
|---|---|
| AFUE 95% | 95% |

Time of Sale:

For example, a default sized ENERGY STAR boiler purchased and installed near Springfield in the year 2012

ΔTherms = (1043) * (1/0.8) - 1/0.875)

= 112 Therms

Early Replacement:

For example, an existing function boiler with unknown efficiency is replaced with an ENERGY STAR boiler purchased and installed in Springfield in 2013.

ΔTherms for remaining life of existing unit (1st 8 years):

= 1043 * (1/0.616 – 1/0.875)

= 501 Therms

ΔTherms for remaining measure life (next 17 years):

= (1043) * (1/0.82 - 1/0.875)

= 80.0 Therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-HVC-GHEB-V03-150601**

## 5.3.7 Gas High Efficiency Furnace

**DESCRIPTION**

High efficiency furnace features may include improved heat exchangers and modulating multi-stage burners.

This measure characterizes:

a) Time of sale:
   a. The installation of a new high efficiency, gas-fired condensing furnace in a residential location. This could relate to the replacement of an existing unit at the end of its useful life, or the installation of a new system in a new home.

b) Early Replacement:
   a. The early removal of an existing functioning AFUE 75% or less furnace from service, prior to its natural end of life, and replacement with a new high efficiency unit. Savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life. At time of writing, the DOE had rescinded the next Federal Standard change for furnaces, however it is likely that a new standard will be in effect after the assumed remaining useful life of the existing unit. For the purposes of this measure- the new baseline is assumed to be 90%.
   b. The assumption of the existing unit efficiency in the Early Replacement section of this TRM is based upon the average efficiency of units that were classified in Ameren's PY3-PY4 as functioning and AFUE <=75%. Therefore it is only appropriate to use these Early Replacement assumptions where those conditions are met. The TAC defined "functioning" as the unit is fully operational – providing sufficient space conditioning (i.e. heat exchanger, compressors, pumps work effectively) and/or the cost of repair is under 20% of the new baseline replacement cost. Therefore in order to apply early replacement assumptions the programs should apply the following eligibility criteria: AFUE <=75% and cost of any repairs <$528.
   c. A weighted average early replacement rate is provided for use when the actual baseline early replacement rate is unknown[946].

**Deemed Early Replacement Rates For Furnaces**

| Replacement Scenario for the Furnace | Deemed Early Replacement Rate |
|---|---|
| Early Replacement Rate for Furnace-only participants | 7% |
| Early Replacement Rate for a furnace when the furnace is the Primary unit in a Combined System Replacement (CSR) project | 14% |
| Early Replacement Rate for a furnace when the furnace is the | 46% |

[946] Based upon research from "Home Energy Efficiency Rebate Program GPY2 Evaluation Report" which outlines early replacement rates for both primary and secondary central air cooling (CAC) and residential funaces. The unit (furnace or CAC unit) that initially caused the customer to contact a trade ally is defined as the "primary unit". The furnace or CAC unit that was also replaced but did not initially prompt the customer to contact a trade ally is defined as the "secondary unit". This evaluation used different criteria for early replacement due to the availability of data after the fact; cost of any repairs < $550 and age of unit < 20 years. Report presented to Nicor Gas Company February 27, 2014, available at http://www.ilsag.info/evaluation-documents.html.

| Replacement Scenario for the Furnace | Deemed Early Replacement Rate |
|---|---|
| Secondary unit in a CSR project | |

This measure was developed to be applicable to the following program types: TOS, NC, EREP. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a residential sized (input energy less than 225,000 Btu/hr) natural gas fired furnace with an Annual Fuel Utilization Efficiency (AFUE) rating exceeding the program requirements.

### DEFINITION OF BASELINE EQUIPMENT

Time of Sale: Although the current Federal Standard for gas furnaces is an AFUE rating of 78%, based upon review of available product in the AHRI database, the baseline efficiency for this characterization is assumed to be 80%. The baseline will be adjusted when the Federal Standard is updated.

Early replacement: The baseline for this measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and a new baseline unit for the remainder of the measure life. As discussed above we estimate that the new baseline unit that could be purchased in the year the existing unit would have needed replacing is 90%.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 20 years[947].

For early replacement: Remaining life of existing equipment is assumed to be 6 years[948].

### DEEMED MEASURE COST

Time of sale: The incremental installed cost (retail equipment cost plus installation cost) for this measure depends on efficiency as listed below[949]:

| AFUE | Installed Cost | Incremental Installed Cost |
|---|---|---|
| 80% | $2011 | n/a |
| 90% | $2641 | $630 |
| 91% | $2727 | $716 |
| 92% | $2813 | $802 |

[947] Table 8.3.3 The Technical support documents for federal residential appliance standards: http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/chapter_8.pdf

[948] Assumed to be one third of effective useful life

[949] Based on data from Table E.1.1 of Appendix E of the Appliance Standards Technical Support Documents including equipment cost and installation labor.(http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/appendix_e.pdf). Where efficiency ratings are not provided, the values are interpolated from those that are. Note that ECM furnace fan cost (refer to other measure in TRM) has been deducted from the 93%-96% AFUE values to avoid double counting.

| AFUE | Installed Cost | Incremental Installed Cost |
|---|---|---|
| 93% | $3025 | $1014 |
| 94% | $3237 | $1226 |
| 95% | $3449 | $1438 |
| 96% | $3661 | $1650 |

Early Replacement: The full installed cost is provided in the table above. The assumed deferred cost (after 6 years) of replacing existing equipment with a new baseline unit is assumed to be $2641. This cost should be discounted to present value using the utility's discount rate.

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

Electrical energy savings from the more fan-efficient (typically using brushless permanent magnet (BPM) blower motor) should also be claimed, please refer to "Furnace Blower Motor" characterization for details.

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

If the blower motor is also used for cooling, coincident peak demand savings should also be claimed, please refer to "Furnace Blower Motor" characterization for savings details.

### NATURAL GAS SAVINGS

Time of Sale:

ΔTherms = Gas_Furnace_Heating_Load * (1/AFUE(base) - 1/AFUE(eff))

Early replacement[950]:

ΔTherms for remaining life of existing unit (1st 6 years):

= Gas_Furnace_Heating_Load * (1/AFUE(exist) - 1/AFUE(eff)))

ΔTherms for remaining measure life (next 14 years):

[950] The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).

= Gas_Furnace_Heating_Load * (1/AFUE(base) - 1/AFUE(eff)))

Where:

Gas_Furnace_Heating_Load

= Estimate of annual household heating load [951] for gas furnace heated single-family homes. If location is unknown, assume the average below[952].

= Actual if informed by site-specific load calculations, ACCA Manual J or equivalent[953].

| Climate Zone (City based upon) | Gas_Furnace_Heating_Load (therms) |
|---|---|
| 1 (Rockford) | 873 |
| 2 (Chicago) | 834 |
| 3 (Springfield) | 714 |
| 4 (Belleville) | 551 |
| 5 (Marion) | 561 |
| Average | 793 |

HF = Household factor, to adjust heating consumption for non-single-family households.

| Household Type | HF |
|---|---|
| Single-Family | 100% |
| Multi-Family | 65%[954] |
| Actual | Custom[955] |

AFUE(exist) = Existing Furnace Annual Fuel Utilization Efficiency Rating

= Use actual AFUE rating where it is possible to measure or reasonably estimate.

[951] Heating load is used to describe the household heating need, which is equal to (gas consumption * AFUE )

[952] Values are based on household heating consumption values and inferred average AFUE results from Table 2-1, *Energy Efficiency / Demand Response Nicor Gas Plan Year 1 (6/1/2011-5/31/2012) Research Report: Furnace Metering Study* (August 1, 2013) (prepared by Navigant Consulting, Inc.) and adjusting to a statewide average using relative HDD values to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

[953] The Air Conditioning Contractors of America Manual J, Residential Load Calculation 8th Edition produces equipment sizing loads for Single Family, Multi-single, and Condominiums using input characteristics of the home. A best practice for equipment selection and installation of Heating and Air Conditioning, load calculations are commonly completed by contractors during the selection process and may be readily available for program data purposes.

[954] Multifamily household heating consumption relative to single-family households is affected by overall household square footage and exposure to the exterior. This 65% reduction factor is applied to MF homes with electric resistance, based on professional judgment that average household size, and heat loads of MF households are smaller than single-family homes

[955] Program-specific household factors may be utilized on the basis of sufficiently validated program evaluations.

If unknown, assume 64.4 AFUE% [956].

AFUE(base) = Baseline Furnace Annual Fuel Utilization Efficiency Rating

= Dependent on program type as listed below[957]:

| Program Year | AFUE(base) |
|---|---|
| Time of Sale | 80% |
| Early Replacement | 90% |

AFUE(eff) = Efficent Furnace Annual Fuel Utilization Efficiency Rating

= Actual. If unknown, assume 95%[958]

Time of Sale:

For example, a 95% AFUE furnace near Rockford and purchased in the year 2014

ΔTherms = 873 * (1/0.8 - 1/0.95)

=172 therms

Early Replacement:

For example, an existing functioning furnace with unknown efficiency is replaced with an 95% furnace purchased and installed in Rockford in 2014.

ΔTherms for remaining life of existing unit (1st 6 years):

= 873 * (1/0.644 – 1/0.95)

= 437 therms

ΔTherms for remaining measure life (next 14 years):

= 873 * (1/0.9 - 1/0.95)

=51.1 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-HVC-GHEF-V04-150601**

[956] Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

[957] Though the Federal Minimum AFUE is 78%, there were only 50 models listed in the AHRI database at that level. At AFUE 79% the total rises to 308. There are 3,548 active furnace models listed with AFUE ratings between 78 and 80.

[958] Minimum ENERGY STAR efficiency after 2.1.2012.

## 5.3.8 Ground Source Heat Pump

**DESCRIPTION**

This measure characterizes the installation of a Ground Source Heat Pump under the following scenarios:

a) New Construction:
   a. The installation of a new residential sized Ground Source Heat Pump system meeting ENERGY STAR efficiency standards presented below in a new home.
   b. Note the baseline in this case should be determined via EM&V and the algorithms are provided to allow savings to be calculated from any baseline condition.

b) Time of Sale:
   a. The planned installation of a new residential sized Ground Source Heat Pump system meeting ENERGY STAR efficiency standards presented below to replace an existing system(s) that does not meet the criteria for early replacement described in section c below.
   b. Note the baseline in this case is an equivalent replacement system to that which exists currently in the home. The calculation of savings is dependent on whether an incentive for the installation has been provided by both a gas and electric utility, just an electric utility or just a gas utility.
   c. Additional DHW savings are calculated based upon the fuel and efficiency of the existing unit.

c) Early Replacement/Retrofit:
   a. The early removal of functioning either electric or gas space heating and/or cooling systems from service, prior to the natural end of life, and replacement with a new high efficiency Ground Source Heat Pump system.
   b. Note the baseline in this case is the existing equipment being replaced. The calculation of savings is dependent on whether an incentive for the installation has been provided by both a gas and electric utility, just an electric utility or just a gas utility.
   c. Additional DHW savings are calculated based upon the fuel and efficiency of the existing unit.
   d. The definitions for when an installation can be claimed as an early replacement are provided below. Note if one system (heating or cooling) has failed or does not meet the criteria below but the other system does, then the appropriate new baseline replacement should be used for the unit not meeting early replacement criteria and the existing system efficiency for the unit that does should be used in the algorithm:

| Existing System | Early Replacement Criteria |
|---|---|
| Air Source Heat Pump | SEER <=10 and cost of any repairs <$249 per ton |
| Central Air Conditioner | SEER <=10 and cost of any repairs <$190 per ton |
| Boiler | AFUE <= 75% and cost of any repairs <$709 |
| Furnace | AFUE <= 75% and cost of any repairs <$528 |
| Ground Source Heat Pump | SEER <=10 and cost of any repairs <$249 per ton |

The ENERGY STAR efficiency standards are presented below.

ENERGY STAR Requirements (Effective January 1, 2012)

| Product Type | Cooling EER | Heating COP |
|---|---|---|
| **Water-to-air** | | |
| Closed Loop | 17.1 | 3.6 |
| Open Loop | 21.1 | 4.1 |
| **Water-to-Water** | | |
| Closed Loop | 16.1 | 3.1 |
| Open Loop | 20.1 | 3.5 |
| DGX | 16 | 3.6 |

This measure was developed to be applicable to the following program types: TOS, NC, EREP. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment must be a Ground Source Heat Pump unit meeting the minimum ENERGY STAR efficiency level standards effective at the time of installation as detailed above.

### DEFINITION OF BASELINE EQUIPMENT

For these products, baseline equipment includes Air Conditioning, Space Heating and Water Heating.

New Construction:

To calculate savings with an electric baseline, the baseline equipment is assumed to be an Air Source Heat Pump meeting the Federal Standard efficiency level; 14 SEER, 8.2 HSPF and 11.8[959] EER and a Federal Standard electric hot water heater.

To calculate savings with a furnace/central AC baseline, the baseline equipment is assumed to be an 80% AFUE Furnace and central AC meeting the Federal Standard efficiency level; 13 SEER, 11 EER. If a gas water heater, the Federal Standard baseline is calculated as follows[960]; for <=55 gallon tanks = 0.675 – (0.0015 * storage size in gallons) and for tanks >55 gallon = 0.8012 – (0.00078 * storage size in gallons). For a 40-gallon storage water heater this would be 0.615 EF.

[959] The Federal Standard does not include an EER requirement, so it is approximated with this formula: (-0.02 * SEER2) + (1.12 * SEER) Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder.

[960] Minimum Federal Standard as of 4/1/2015; http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf

Time of Sale: The baseline for this measure is a new replacement unit of the same system type as the existing unit, meeting the baselines provided below.

| Unit Type | Efficiency Standard |
|---|---|
| ASHP | 14 SEER, 11.8 EER, 8.2 HSPF |
| Gas Furnace | 80% AFUE |
| Gas Boiler | 82% AFUE |
| Central AC | 13 SEER, 11 EER |

Early replacement / Retrofit: The baseline for this measure is the efficiency of the *existing* heating, cooling and hot water equipment for the assumed remaining useful life of the existing unit and a new baseline heating and cooling system for the remainder of the measure life (as provided in table above except for Gas Furnace where new baseline assumption is 90% due to pending standard change).

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 25 years[961].

For early replacement, the remaining life of existing equipment is assumed to be 8 years[962].

### DEEMED MEASURE COST

New Construction and Time of Sale: The actual installed cost of the Ground Source Heat Pump should be used (default of $3957 per ton[963]), minus the assumed installation cost of the baseline equipment ($1936 per ton for ASHP[964] or $2011 for a new baseline 80% AFUE furnace or $3543 for a new 82% AFUE boiler[965] and $2,857[966] for new baseline Central AC replacement).

Early Replacement: The full installation cost of the Ground Source Heat Pump should be used (default provided above). The assumed deferred cost (after 8 years) of replacing existing equipment with a new baseline unit is assumed to be $1936 per ton for a new baseline Air Source Heat Pump, or $2641[967] for a new baseline 90% AFUE furnace or $3543 for a new 82% AFUE boiler and $2,857 for new baseline Central AC replacement. This future cost

[961] System life of indoor components as per DOE estimate http://energy.gov/energysaver/articles/geothermal-heat-pumps. The ground loop has a much longer life, but the compressor and other mechanical components are the same as an ASHP. http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf

[962] Assumed to be one third of effective useful life

[963] Based on data provided in 'Results of HomE geothermal and air source heat pump rebate incentives documented by IL electric cooperatives'.

[964] Based on data provided on Home Advisor website, providing national average ASHP cost based on 2465 cost submittals. http://www.homeadvisor.com/cost/heating-and-cooling/install-a-heat-pump/

[965] Furnace and boiler costs are based on data provided in Appendix E of the Appliance Standards Technical Support Documents including equipment cost and installation labor (http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/appendix_e.pdf). Where efficiency ratings are not provided, the values are interpolated from those that are.

[966] Based on 3 ton initial cost estimate for a conventional unit from ENERGY STAR Central AC calculator (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls). While baselines are likely to shift in the future, there is currently no good indication of what the cost of a new baseline unit will be in 6 years. In the absence of this information, assuming a constant federal baseline cost is within the range of error for this prescriptive measure.

[967] Based on data from Table E.1.1 of Appendix E of the Appliance Standards Technical Support Documents including equipment cost and installation labor. (http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/appendix_e.pdf).

should be discounted to present value using the utilities' discount rate.

### LOADSHAPE

| | |
|---|---|
| Loadshape R08 - Residential Cooling | (if replacing gas heat and central AC) |
| Loadshape R09 - Residential Electric Space Heat | (if replacing electric heat with no cooling) |
| Loadshape R10 - Residential Electric Heating and Cooling | (if replacing ASHP) |

### COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Heat Pumps (during utility peak hour)

= 72%%[968]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Heat Pumps (average during PJM peak period)

= 46.6%[969]

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

New Construction and Time of Sale (non-fuel switch only):

ΔkWh = [Cooling savings] + [Heating savings] + [DHW savings]

= [(FLHcool * Capacity_cooling * (1/$SEER_{base}$– (1/$EER_{PL}$)/1000] + [Elecheat * FLHheat * Capacity_heating * (1/$HSPF_{base}$ – (1/$COP_{PL}$ * 3.412)))/1000] + [ElecDHW * %DHWDisplaced * (((1/$EF_{ELEC}$) * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 3412)]

New Construction and Time of Sale (fuel switch only):

If measure is supported by gas utility only, ΔkWH = 0

If measure is supported by gas and electric utility or electric utility only, electric utility claim savings calculated below:

ΔkWh = [Cooling savings] + [Heating savings from base ASHP to GSHP] + [DHW savings]

= [(FLHcool * Capacity_cooling * (1/$SEER_{base}$– (1/$EER_{PL}$)/1000] + [FLHheat *

[968] Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'. http://www.icc.illinois.gov/downloads/public/edocket/368522.pdf

[969] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

Capacity_heating * (1/$HSPF_{ASHP}$ – (1/$COP_{PL}$ * 3.412)))/1000] + [ElecDHW * %DHWDisplaced * (((1/$EF_{ELEC}$) * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 3412)]

Early replacement (non-fuel switch only)[970]:

ΔkWH for remaining life of existing unit (1st 8 years):

= [Cooling savings] + [Heating savings] + [DHW savings]

= [(FLHcool * Capacity_cooling * (1/SEERexist – (1/$EER_{PL}$)/1000] + [ElecHeat * (FLHheat * Capacity_heating * (1/HSPFexist) – (1/$COP_{PL}$ * 3.412)))/1000] + [ElecDHW * %DHWDisplaced * (((1/ $EF_{ELEC}$) * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 3412)]

ΔkWH for remaining measure life (next 17 years):

= [(FLHcool * Capacity_cooling * (1/SEERbase – (1/$EER_{PL}$)/1000] + [ElecHeat * (FLHheat * Capacity_heating * (1/HSPFbase) – (1/$COP_{PL}$ * 3.412)))/1000] + [ElecDHW * %DHWDisplaced * (((1/ $EF_{ELEC}$) * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 3412)]

Early replacement - fuel switch only (see illustrative examples after Natural Gas section):

If measure is supported by gas utility only, ΔkWH = 0

If measure is supported by gas and electric utility or electric utility only, electric utility claim savings calculated below:

ΔkWh for remaining life of existing unit (1st 8 years):

= [Cooling savings] + [Heating savings from base ASHP to GSHP] + [DHW savings]

= [(FLHcool * Capacity_cooling * (1/SEERexist – (1/$EER_{PL}$)/1000] + [(FLHheat * Capacity_heating * (1/$HSPF_{ASHP}$ – (1/$COP_{PL}$ * 3.412)))/1000] + [ElecDHW * %DHWDisplaced * (((1/ $EF_{ELEC}$) * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 3412)]

ΔkWh for remaining measure life (next 17 years):

= [Cooling savings] + [Heating savings from base ASHP to GSHP] + [DHW savings]

= [(FLHcool * Capacity_cooling * (1/$SEER_{base}$ – (1/$EER_{PL}$)/1000] + [(FLHheat * Capacity_heating * (1/$HSPF_{ASHP}$ – (1/$COP_{PL}$ * 3.412)))/1000] + [ElecDHW * %DHWDisplaced * (((1/ $EF_{ELEC}$) * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 3412)]

Where:

FLHcool = Full load cooling hours

Dependent on location as below[971]:

[970] The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).

[971] Based on Full Load Hours from ENERGY STAR with adjustments made in a Navigant Evaluation, other cities were scaled using

| Climate Zone (City based upon) | FLHcool Single Family | FLHcool Multifamily |
|---|---|---|
| 1 (Rockford) | 512 | 467 |
| 2 (Chicago) | 570 | 506 |
| 3 (Springfield) | 730 | 663 |
| 4 (Belleville) | 1,035 | 940 |
| 5 (Marion) | 903 | 820 |
| Weighted Average[972] | 629 | 564 |

Capacity_cooling = Cooling Capacity of Ground Source Heat Pump (Btu/hr)

= Actual (1 ton = 12,000Btu/hr)

SEERbase = SEER Efficiency of new replacement baseline unit

| Existing Cooling System | SEERbase |
|---|---|
| Air Source Heat Pump | 14[973] |
| Central AC | 13[974] |
| No central cooling | 13[975] |

SEERexist = SEER Efficiency of existing cooling unit

= Use actual SEER rating where it is possible to measure or reasonably estimate, if unknown assume default provided below:

| Existing Cooling System | SEER_exist |
|---|---|
| Air Source Heat Pump | 9.12[976] |
| Central AC | 8.60[977] |
| No central cooling | 13 [978] |

$SEER_{ASHP}$ = SEER Efficiency of new baseline Air Source Heat Pump unit (for fuel switch)

those results and CDD. There is a county mapping table in the Section 3.7 of the TRM providing the appropriate city to use for each county of Illinois.

[972] Weighted based on number of occupied residential housing units in each zone.

[973] Minimum Federal Standard as of 1/1/2015; http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf

[974] Minimum Federal Standard; Federal Register, Vol. 66, No. 14, Monday, January 22, 2001/Rules and Regulations, p. 7170-7200.

[975] Assumes that the decision to replace existing systems includes desire to add cooling.

[976] Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

[977] Ibid.

[978] Assumes that the decision to replace existing systems includes desire to add cooling.

= 14 [979]

$EER_{PL}$ = Part Load EER Efficiency of efficient GSHP unit[980]

= Actual installed

ElecHeat = 1 if existing building is electrically heated

= 0 if existing building is not electrically heated

FLHheat = Full load heating hours

Dependent on location as below[981]:

| Climate Zone (City based upon) | FLH_heat |
|---|---|
| 1 (Rockford) | 1,969 |
| 2 (Chicago) | 1,840 |
| 3 (Springfield) | 1,754 |
| 4 (Belleville) | 1,266 |
| 5 (Marion) | 1,288 |
| Weighted Average[982] | 1,821 |

Capacity_heating = Heating Capacity of Ground Source Heat Pump (Btu/hr)

= Actual (1 ton = 12,000Btu/hr)

$HSPF_{base}$ =Heating System Performance Factor of new replacement baseline heating system (kBtu/kWh)

| Existing Heating System | HSPF_base |
|---|---|
| Air Source Heat Pump | 8.2 |
| Electric Resistance | 3.41[983] |

HSPF_exist =Heating System Performance Factor of existing heating system (kBtu/kWh)

= Use actual HSPF rating where it is possible to measure or reasonably estimate. If unknown assume default:

[979] Minimum Federal Standard as of 1/1/2015; http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf

[980] As per conversations with David Buss territory manager for Connor Co, the SEER and COP ratings of an ASHP equate most appropriately with the part load EER and COP of a GSHP.

[981] Heating EFLH based on ENERGY STAR EFLH for Rockford, Chicago, and Springfield and on NCDC/NOAA HDD for the other two cities. In all cases, the hours were adjusted based on average natural gas heating consumption in IL. There is a county mapping table in Section 3.7 of the TRM providing the appropriate city to use for each county of Illinois.

[982] Weighted based on number of occupied residential housing units in each zone.

[983] Electric resistance has a COP of 1.0 which equals 1/0.293 = 3.41 HSPF.

| Existing Heating System | HSPF_exist |
|---|---|
| Air Source Heat Pump | 5.44 |
| Electric Resistance | 3.41 |

$HSPF_{ASHP}$ =Heating Season Performance Factor for new ASHP baseline unit (for fuel switch)

=8.2 [984]

$COP_{PL}$ = Part Load Coefficient of Performance of efficient unit[985]

= Actual Installed

3.412 = Constant to convert the COP of the unit to the Heating Season Performance Factor (HSPF).

ElecDHW = 1 if existing DHW is electrically heated

= 0 if existing DHW is not electrically heated

%DHWDisplaced = Percentage of total DHW load that the GSHP will provide

= Actual if known

= If unknown and if desuperheater installed assume 44%[986]

= 0% if no desuperheater installed

$EF_{ELEC}$ = Energy Factor (efficiency) of electric water heater

= Actual. If unknown or for new construction assume federal standard[987]:

For <=55 gallons: 0.96 – (0.0003 * rated volume in gallons)

For >55 gallons: 2.057 – (0.00113 * rated volume in gallons)

GPD = Gallons Per Day of hot water use per person

= 45.5 gallons hot water per day per household/2.59 people per household[988]

= 17.6

Household = Average number of people per household

[984] Minimum Federal Standard as of 1/1/2015; http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf

[985] As per conversations with David Buss territory manager for Connor Co, the SEER and COP ratings of an ASHP equate most appropriately with the part load EER and COP of a GSHP.

[986] Assumes that the desuperheater can provide two thirds of hot water needs for eight months of the year (2/3 * 2/3 = 44%). Based on input from Doug Dougherty, Geothermal Exchange Organization.

[987] Minimum Federal Standard as of 4/1/2015; http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf

[988] Email message from Maureen Hodgins, Research Manager for Water Research Foundation, to TAC/SAG, August 26, 2014

| Household Unit Type | Household |
|---|---|
| Single-Family - Deemed | 2.56[989] |
| Custom | Actual Occupancy or Number of Bedrooms[990] |

365.25 = Days per year

γWater = Specific weight of water

= 8.33 pounds per gallon

$T_{OUT}$ = Tank temperature

= 125°F

$T_{IN}$ = Incoming water temperature from well or municiplal system

= 54°F[991]

1.0 = Heat Capacity of water (1 Btu/lb*°F)

3412 = Conversion from Btu to kWh

[989] ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program citing 2006-2008 American Community Survey data from the US Census Bureau for Illinois cited on p. 17 of the PY2 Evaluation report. 2.75 * 93% evaluation adjustment

[990] Bedrooms are suitable proxies for household occupancy, and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.

[991] US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html

Illustrative Examples

New Construction using ASHP baseline:

For example, a 3 ton unit with Part Load EER rating of 19 and Part Load COP of 4.4 with desuperheater is installed with a 50 gallon electric water heater in single family house in Springfield:

ΔkWh = [(FLHcool * Capacity_cooling * (1/$SEER_{base}$ – (1/$EER_{PL}$)/1000] + [(FLHheat * Capacity_heating * (1/HSPFbase – (1/$COP_{PL}$ * 3.412)))/1000] + [ElecDHW * %DHWDisplaced * (((1/ $EF_{ELEC\ EXIST}$) * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 3412)]

ΔkWh = [(730 * 36,000 * (1/14 – 1/19)) / 1000] + [(1754* 36,000 * (1/8.2 – 1/ (4.4*3.412))) / 1000] + [1 * 0.44 * (((1/0.945) * 17.6 * 2.56 *365.25 * 8.33 * (125-54) * 1)/3412)]

= 494 + 3494 + 1328

= 5316 kWh

Early Replacement – non-fuel switch (see example after Natural gas section for Fuel switch):

For example, a 3 ton unit with Part Load EER rating of 19 and Part Load COP of 4.4 with desuperheater is installed in single family house in Springfield with a 50 gallon electric water heater replacing an existing working Air Source Heat Pump with unknown efficiency ratings:

ΔkWH for remaining life of existing unit (1st 8 years):

= [(730 * 36,000 * (1/9.12 - 1/19)) / 1000] + [(1754 * 36,000 * (1/5.44 - 1/(4.4 * 3.412))) / 1000] + [0.44 * 1 * (((1/0.945) * 17.6 * 2.56 *365.25 * 8.33 * (125-54) * 1)/3412)]

= 1498 + 7401 + 1328

= 10,227 kWh

ΔkWH for remaining measure life (next 17 years):

= [(730 * 36,000 * (1/14 – 1/28)) / 1000] + [(1967 * 36,000 * (1/8.2 – 1/ (4.4 * 3.412)) / 1000] + [0.44 * 1 * (((1/0.945) * 17.6 * 2.56 *365.25 * 8.33 * (125-54) * 1)/3412)]

= 494 + 3494 + 1328

= 5316 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

New Construction and Time of Sale:

ΔkW = (Capacity_cooling * (1/EERbase - 1/$EER_{FL}$))/1000) * CF

Early replacement:

ΔkW for remaining life of existing unit (1st 8 years):

= (Capacity_cooling * (1/EERexist - 1/$EER_{FL}$))/1000) * CF

ΔkW for remaining measure life (next 17 years):

= (Capacity_cooling * (1/EERbase - 1/$EER_{FL}$))/1000) * CF

Where:

EERbase = EER Efficiency of new replacement unit

| Existing Cooling System | EER_base |
|---|---|
| Air Source Heat Pump | 11.8[992] |
| Central AC | 11 [993] |
| No central cooling | 11[994] |

EERexist = Energy Efficiency Ratio of existing cooling unit (kBtu/hr / kW)

= Use actual EER rating where it is possible to measure or reasonably estimate. If EER unknown but SEER available convert using the equation:

EERexist = (-0.02 * SEERexist$^2$) + (1.12 * SEERexist) [995]

If SEER rating unavailable use:

| Existing Cooling System | EER_exist |
|---|---|
| Air Source Heat Pump | 8.55 |
| Central AC | 8.15[997] |
| No central cooling | 11 [998] |

$EER_{FL}$ = Full Load EER Efficiency of ENERGY STAR GSHP unit [999]

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)

= 72%%[1000]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)

= 46.6%[1001]

[992] The Federal Standard does not include an EER requirement, so it is approximated with the conversion formula from Wassmer, M. 2003 thesis refererenced below.
[993] Minimum Federal Standard; Federal Register, Vol. 66, No. 14, Monday, January 22, 2001/Rules and Regulations, p. 7170-7200.
[994] Assumes that the decision to replace existing systems includes desire to add cooling.
[995] From Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder.
[996] Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.
[997] Ibid.
[998] Assumes that the decision to replace existing systems includes desire to add cooling.
[999] As per conversations with David Buss territory manager for Connor Co, the EER rating of an ASHP equate most appropriately with the full load EER of a GSHP.
[1000] Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'. http://www.icc.illinois.gov/downloads/public/edocket/368522.pdf
[1001] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

New Construction or Time of Sale:

For example, a 3 ton unit with Full Load EER rating of 19:

$\Delta kW_{SSP}$ = ((36,000 * (1/11.8 – 1/19))/1000) * 0.72

= 0.83 kW

$\Delta kW_{PJM}$ = ((36,000 * (1/11 – 1/19))/1000) * 0.466

= 0.54 kW

Early Replacement:

For example, a 3 ton Full Load 19 EER replaces an existing working Air Source Heat Pump with unknown efficiency ratings in Marion:

$\Delta kW_{SSP}$ for remaining life of existing unit (1st 8 years):

= ((36,000 * (1/8.55 – 1/19))/1000) * 0.72

= 1.67 kW

$\Delta kW_{SSP}$ for remaining measure life (next 17 years):

= ((36,000 * (1/11.8 – 1/19))/1000) * 0.72

= 0.83 kW

$\Delta kW_{PJM}$ for remaining life of existing unit (1st 8 years):

= ((36,000 * (1/8.55 – 1/19))/1000) * 0.466

= 1.08 kW

$\Delta kW_{PJM}$ for remaining measure life (next 17 years):

= ((36,000 * (1/11.8 – 1/19))/1000) * 0.466

= 0.54 kW

**NATURAL GAS SAVINGS**

New Construction and Time of Sale with baseline gas heat and/or hot water:

If measure is supported by gas utility only, gas utility claim savings calculated below:

ΔTherms = [Heating Savings] + [DHW Savings]

= [Replaced gas consumption – therm equivalent of GSHP source kWh] + [DHW Savings]

= [(1 – ElecHeat) * ((Gas_Heating_Load/AFUEbase) – (kWhtoTherm * FLHheat * Capacity_heating * 1/$COP_{PL}$)/1000)] + [(1 – ElecDHW) * %DHWDisplaced * (1/ $EF_{GAS\ EXIST}$ * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 100,000)]

If measure is supported by electric utility only, ΔTherms = 0

If measure is supported by gas and electric utility, gas utility claim savings calculated below, (electric savings is provided in Electric Energy Savings section):

ΔTherms = [Heating Savings] + [DHW Savings]

= [Replaced gas consumption – therm equivalent of base ASHP source kWh] + [DHW Savings]

= [(1 – ElecHeat) * ((Gas_Heating_Load/AFUEbase) – (kWhtoTherm * FLHheat * Capacity_heating * 1/($HSPF_{ASHP}$/3.412))/1000)] + [(1 – ElecDHW) * %DHWDisplaced * (1/ $EF_{GAS\ EXIST}$ * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 100,000)]

Early replacement for homes with existing gas heat and/or hot water:

If measure is supported by gas utility only, gas utility claim savings calculated below:

ΔTherms for remaining life of existing unit (1st 8 years):

= [Heating Savings] + [DHW Savings]

= [Replaced gas consumption – therm equivalent of GSHP source kWh] + [DHW Savings]

= [(1 – ElecHeat) * ((Gas_Heating_Load/AFUEexist) – (kWhtoTherm * FLHheat * Capacity_heating * 1/($COP_{PL}$ * 3.412))/1000)] + [(1 – ElecDHW) * %DHWDisplaced * (1/ $EF_{GAS\ EXIST}$ * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 100,000)]

ΔTherms for remaining measure life (next 17 years):

= [(1 – ElecHeat) * ((Gas_Heating_Load/AFUEbaseER) – (kWhtoTherm * FLHheat * Capacity_heating * 1//($COP_{PL}$ * 3.412))/1000)] + [(1 – ElecDHW) * %DHWDisplaced * (1/ $EF_{GAS\ EXIST}$ * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 100,000)]

If measure is supported by electric utility only, ΔTherms = 0

If measure is supported by gas and electric utility, gas utility claim savings calculated below:

ΔTherms for remaining life of existing unit (1st 8 years):

ΔTherms = [Heating Savings] + [DHW Savings]

= [Replaced gas consumption – therm equivalent of base ASHP source kWh] + [DHW Savings]

= [(1 – ElecHeat) * ((Gas_Heating_Load/AFUEexist) – (kWhtoTherm * FLHheat * Capacity_heating * 1/$HSPF_{ASHP}$)/1000)] + [(1 – ElecDHW) * %DHWDisplaced * (1/ $EF_{GAS\ EXIST}$ * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 100,000)]

ΔTherms for remaining measure life (next 17 years):

= [(1 – ElecHeat) * ((Gas_Heating_Load/AFUEbaseER) – (kWhtoTherm * FLHheat * Capacity_heating * 1/$HSPF_{ASHP}$)/1000)] + [(1 – ElecDHW) * %DHWDisplaced * (1/ $EF_{GAS\ EXIST}$ * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 100,000)]

Where:

ElecHeat = 1 if existing building is electrically heated

= 0 if existing building is not electrically heated

Gas_Heating_Load

= Estimate of annual household heating load [1002] for gas furnace heated single-family homes. If location is unknown, assume the average below.

= Actual if informed by site-specific load calculations, ACCA Manual J or equivalent[1003].

| Climate Zone (City based upon) | Gas_Heating_Load if Furnace (therms) [1004] | Gas_Heating_Load if Boiler (therms) [1005] |
|---|---|---|
| 1 (Rockford) | 873 | 1275 |
| 2 (Chicago) | 834 | 1218 |
| 3 (Springfield) | 714 | 1043 |
| 4 (Belleville) | 551 | 805 |
| 5 (Marion) | 561 | 819 |
| Average | 793 | 1158 |

AFUEbase = Baseline Annual Fuel Utilization Efficiency Rating

= 80% if furnace and 82% if boiler.

AFUEexist = Existing Annual Fuel Utilization Efficiency Rating

[1002] Heating load is used to describe the household heating need, which is equal to (gas consumption * AFUE )

[1003] The Air Conditioning Contractors of America Manual J, Residential Load Calculation 8th Edition produces equipment sizing loads for Single Family, Multi-single, and Condominiums using input characteristics of the home. A best practice for equipment selection and installation of Heating and Air Conditioning, load calculations are commonly completed by contractors during the selection process and may be readily available for program data purposes.

[1004] Values are based on household heating consumption values and inferred average AFUE results from Table 2-1, *Energy Efficiency / Demand Response Nicor Gas Plan Year 1 (6/1/2011-5/31/2012) Research Report: Furnace Metering Study* (August 1, 2013) (prepared by Navigant Consulting, Inc.) and adjusting to a statewide average using relative HDD values to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

[1005] Boiler consumption values are informed by an evaluation which did not identify any fraction of heating load due to domestic hot water (DHW) provided by the boiler. Thus these values are an average of both homes with boilers only providing heat, and homes with boilers that also provide DHW. Values are based on household heating consumption values and inferred average AFUE results from Table 3-4, Program Sample Analysis, *Nicor R29 Res Rebate Evaluation Report 092611_REV FINAL to Nicor*). Adjusting to a statewide average using relative HDD values to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

= Use actual AFUE rating where it is possible to measure or reasonably estimate.

If unknown, assume 64.4% if furnace and 61.6% [1006] if boiler.

AFUEbaseER = Baseline Annual Fuel Utilization Efficiency Rating for early replacement measure

= 90%[1007] if furnace and 82% if boiler.

kWhtoTherm = Converts source kWh to Therms

= $H_{grid}$ / 100000

$H_{grid}$ = Heat rate of the grid in btu/kWh based on the average fossil heat rate for the EPA eGRID subregion and includes a factor that takes into account T&D losses.

For systems operating less than 6,500 hrs per year:

Use the Non-baseload heat rate provided by EPA eGRID for RFC West region for ComEd territory (including independent providers connected to RFC West), and SERC Midwest region for Ameren territory (including independent providers connected to SERC Midwest)[1008]. Also include any line losses.

For systems operating more than 6,500 hrs per year:

Use the All Fossil Average heat rate provided by EPA eGRID for RFC West region for ComEd territory, and SERC Midwest region for Ameren territory. Also include any line losses.

3.412 = Converts HSPF to COP

$EF_{GAS\ EXIST}$ = Energy Factor (efficiency) of existing gas water heater

= Actual. If unknown assume federal standard[1009]:

For <=55 gallons: 0.675 – (0.0015 * tank_size)

For > 55 gallons 0.8012 – (0.00078 * tank size)

= If tank size unknown assume 40 gallons and EF_Baseline of 0.615

All other variables provided above

[1006] Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

[1007] Assumes that Federal Standard will have been increased to 90% by the time the existing unit would have to have been replaced.

[1008] Refer to EPA eGRID data http://www.epa.gov/chp/documents/fuel_and_co2_savings.pdf, page 24 and http://www.epa.gov/cleanenergy/documents/egridzips/eGRID_9th_edition_V1-0_year_2010_Summary_Tables.pdf, page 9. Current values are:

- Non-Baseload RFC West: 9,811 Btu/kWh * (1 + Line Losses)
- Non-Baseload SERC Midwest: 10,511 Btu/kWh * (1 + Line Losses)
- All Fossil Average RFC West: 10,038 Btu/kWh * (1 + Line Losses)
- All Fossil Average SERC Midwest: 10,364 Btu/kWh * (1 + Line Losses)

[1009] Minimum Federal Standard as of 4/1/2015; http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf

Illustrative Examples *[for illustrative purposes a Heat Rate of 10,000 Btu/kWh is used]*

New construction using gas furnace and central AC baseline, *supported by Gas utility only*:

For example, a 3 ton unit with Part Load EER rating of 19 and Part Load COP of 4.4 in single family house in Springfield with a 40 gallon gas water heater is installed in place of a natural gas furnace and 3 ton Central AC unit:

ΔkWH = 0

ΔTherms = [Heating Savings] + [DHW Savings]

= [Replaced gas consumption – therm equivalent of GSHP source kWh] + [DHW Savings]

= [(1 – ElecHeat) * ((Gas_Heating_Load/AFUEbase) – (kWhtoTherm * FLHheat * Capacity_heating * 1/($COP_{PL}$ * 3.412)/1000)] + [(1 – ElecDHW) * %DHWDisplaced * (1/ $EF_{GAS\ EXIST}$ * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 100,000)]

= [(1-0) * ((714/0.80) – (10000/100000 * 1754 * 36,000 * 1/(4.4 * 3.412))/1000)] + [(1 – 0) * (0.44 * (1/ 0.615 * 17.6 * 2.56 *365.25 * 8.33 * (125-54) * 1) / 100,000)]

= 472 + 70

= 542 therms

Early Replacement fuel switch, *supported by gas and electric utility*:

For example, a 3 ton unit with Part Load EER rating of 19 and Part Load COP of 4.4 in single family house in Springfield with a 40 gallon gas water heater replaces an existing working natural gas furnace and 3 ton Central AC unit with unknown efficiency ratings:

ΔkWh for remaining life of existing unit (1st 8 years):

= [Cooling savings] + [Heating savings from base ASHP to GSHP] + [DHW savings]

= [(FLHcool * Capacity_cooling * (1/SEERexist – (1/$EER_{PL}$)/1000] + [(FLHheat * Capacity_heating * (1/$HSPF_{ASHP}$ – (1/$COP_{PL}$ * 3.412)))/1000] + [ElecDHW * %DHWDisplaced * (((1/ $EF_{ELEC}$) * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 3412)]

= [(730* 36,000 * (1/8.6 - 1/19)) / 1000] + [(1754 * 36,000 * (1/8.2 - 1/(4.4 * 3.412))) / 1000] + [0 * 0.44 * (((1/0.904) * 17.6 * 2.56 *365.25 * 8.33 * (125-54) * 1)/3412)]

= 1673 + 3494 + 0

= 5167 kWh

Continued on next page.

Illustrative Example continued

ΔkWh for remaining measure life (next 17 years):

= [Cooling savings] + [Heating savings] + [DHW savings]

= [(FLHcool * Capacity_cooling * (1/$SEER_{base}$ – (1/$EER_{PL}$)/1000] + [(FLHheat * Capacity_heating * (1/$HSPF_{ASHP}$ – (1/$COP_{PL}$ * 3.412)))/1000] + [ElecDHW * %DHWDisplaced * (((1/ $EF_{ELEC}$) * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) /3412)]

= [(730 * 36,000 * (1/13 – 1/19)) / 1000] + [1754 * 36,000 * (1/8.2 – 1/ (4.4 *3.412)) / 1000] + [0 * 0.44 * (((1/0.904) * 17.6 * 2.56 *365.25 * 8.33 * (125-54) *1)/3412)]

= 638 + 3494 + 0

= 4132 kWh

ΔTherms for remaining life of existing unit (1st 8 years):

= [Heating Savings] + [DHW Savings]

= [Replaced gas consumption – therm equivalent of base ASHP source kWh] + [DHW Savings]

= [(1 – ElecHeat) * ((Gas_Heating_Load/AFUEexist) – (kWhtoTherm * FLHheat * Capacity_heating * 1/$HSPF_{ASHP}$)/1000)] + [(1 – ElecDHW) * %DHWDisplaced * (1/ $EF_{GAS\ EXIST}$ * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 100,000)]

= [(1-0) * ((714/0.644) – (10000/100000 * 1754 * 36,000 * 1/8.2)/1000)] + [(1 – 0) * (0.44 * (1/ 0.615 * 17.6 * 2.56 *365.25 * 8.33 * (125-54) * 1) / 100,000)]

= 339 + 70

= 408 therms

ΔTherms for remaining measure life (next 17 years):

= [(1 – ElecHeat) * ((Gas_Heating_Load/AFUEbaseER) – (kWhtoTherm * FLHheat * Capacity_heating * 1/$HSPF_{ASHP}$)/1000)] + [(1 – ElecDHW) * %DHWDisplaced * (1/ $EF_{GAS\ EXIST}$ * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 100,000)]

= [(1-0) * ((714/0.9) – (10000/100000 * 1754 * 36,000 * 1/8.2)/1000)] + [(1 – 0) * (0.44 * (1/ 0.615 * 17.6 * 2.56 *365.25 * 8.33 * (125-54) * 1) / 100,000)]

= 23 + 70

= 93 therms

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### COST EFFECTIVENESS SCREENING AND LOAD REDUCTION FORECASTING WHEN FUEL SWITCHING

This measure can involve fuel switching from gas to electric.

For the purposes of forecasting load reductions due to fuel switch GSHP projects per Section 16-111.5B, changes in site energy use at the customer's meter (using ΔkWh algorithm below) adjusted for utility line losses (at-the-busbar savings), customer switching estimates, NTG, and any other adjustment factors deemed appropriate, should be used.

The inputs to cost effectiveness screening should reflect the actual impacts on the electric and fuel consumption at the customer meter and, for fuel switching measures, this will not match the output of the calculation/allocation methodology presented in the "Electric Energy Savings" and "Natural Gas Savings" sections above. Therefore in addition to the calculation of savings claimed, the following values should be used to assess the cost effectiveness of the measure.

ΔTherms = [Heating Consumption Replaced[1010]] + [DHW Savings if gas]

= [(1 – ElecHeat) * ((Gas_Heating_Load/AFUEbase)] + [(1 – ElecDHW) * %DHWDisplaced * (1/ $EF_{GAS\ EXIST}$ * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 100,000)]

ΔkWh = - [GSHP heating consumption] + [Cooling savings[1011]] + [DHW savings if electric]

= - [(FLHheat * Capacity_heating * (1/$COP_{PL}$ * 3.412))/1000] + [(FLHcool * Capacity_cooling * (1/SEERbase - 1/$EER_{PL}$))/1000] + [ElecDHW * %DHWDisplaced * ((1/$EF_{ELEC}$ * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 3412)]

[1010] Note AFUEbase in the algorithm should be replaced with AFUEexist for early replacement measures.
[1011] Note SEERbase in the algorithm should be replaced with SEERexist for early replacement measures.

Illustrative Example of Cost Effectiveness Inputs for Fuel Switching

For example, a 3 ton unit with Part Load EER rating of 19 and Part Load COP of 4.4 in single family house in Springfield with a 40 gallon gas water heater replaces an existing working natural gas furnace and 3 ton Central AC unit with unknown efficiency ratings. [Note the calculation provides the annual savings for the first 8 years of the measure life, an additional calculation (not shown) would be required to calculated the annual savings for the remaining life (years 9-25)]:

ΔTherms = [(1 – ElecHeat) * ((Gas_Heating_Load/AFUEexist)] + [(1 – ElecDHW) * %DHWDisplaced * (1/ $EF_{GAS\ EXIST}$ * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 100,000)]

= [(1-0) * (714/0.644)] + [((1 – 0) * 0.44 * (1/ 0.615 * 17.6 * 2.56 *365.25 * 8.33 * (125-54) * 1) / 100,000)]

= 1109 + 70

= 1179 therms

ΔkWh = - [(FLHheat * Capacity_heating * (1/$COP_{PL}$ * 3.412))/1000] + [(FLHcool * Capacity_cooling * (1/SEERexist - 1/$EER_{PL}$))/1000] + [ElecDHW * %DHWDisplaced * (((1/$EF_{ELEC}$) * GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 3412)]

= - [(1754 * 36,000 * (1/(4.4 * 3.412)))/ 1000] + [(730 * 36,000 * (1/8.6 - 1/19))/ 1000)] + [0 * 0.44 * (((1/0.904) * 17.6 * 2.56 *365.25 * 8.33 * (125-54) * 1)/3412)]

= -4206 + 1673 + 0

= -2533 kWh

**MEASURE CODE: RS-HVC-GSHP-V04-150601**

## 5.3.9 High Efficiency Bathroom Exhaust Fan

### DESCRIPTION

This market opportunity is defined by the need for continuous mechanical ventilation due to reduced air-infiltration from a tighter building shell. In retrofit projects, existing fans may be too loud, or insufficient in other ways, to be operated as required for proper ventilation. This measure assumes a fan capacity of 50 CFM rated at a sound level of less than 2.0 sones at 0.1 inches of water column static pressure. This measure may be applied to larger capacity, up to 130 CFM, efficient fans with bi-level controls because the savings and incremental costs are very similar. All eligible installations shall be sized to provide the mechanical ventilation rate indicated by ASHRAE 62.2.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

New efficient (average CFM/watt of 8.3 [1012]) exhaust-only ventilation fan, quiet (< 2.0 sones) Continuous operation in accordance with recommended ventilation rate indicated by ASHRAE 62.2[1013]

### DEFINITION OF BASELINE EQUIPMENT

New standard efficiency (average CFM/Watt of 3.1[1014]) exhaust-only ventilation fan, quiet (< 2.0 sones) operating in accordance with recommended ventilation rate indicated by ASHRAE 62.2 [1015]

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 19 years[1016].

### DEEMED MEASURE COST

Incremental cost per installed fan is $43.50 for quiet, efficient fans[1017].

### LOADSHAPE

Loadshape R11 - Residential Ventilation

### COINCIDENCE FACTOR

The summer Peak Coincidence Factor is assumed to be 100% because the fan runs continuously.

---

[1012] VEIC analysis looking at average efficient fan (i.e. Brushless Permanent Magnet) efficacies at static pressures of 0.1 and 0.25 inches of water column for quiet fans rated for 50 CFM.
[1013] Bi-level controls may be used by efficient fans larger than 50 CFM
[1014] VEIC analysis looking at average baseline fan (i.e. non-Brushless Permanent Magnet) efficacies at static pressures of 0.1 and 0.25 inches of water column for quiet fans rated for 50 CFM.
[1015] On/off cycling controls may be required of baseline fans larger than 50CFM.
[1016] Conservative estimate based upon GDS Associates Measure Life Report "Residential and C&I Lighting and HVAC measures" 25 years for whole-house fans, and 19 for thermostatically-controlled attic fans.
[1017] VEIC analysis using cost data collected from wholesale vendor; http://www.westsidewholesale.com/ .

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = (CFM * (1/\eta_{,BASELINE} - 1/\eta_{EFFICIENT})/1000) * Hours$$

Where:

CFM = Nominal Capacity of the exhaust fan

= 50 CFM[1018]

$\eta_{BASELINE}$ = Average efficacy for baseline fan

= 3.1 CFM/Watt[1019]

$\eta_{EFFCIENT}$ = Average efficacy for efficient fan

= 8.3 CFM/Watt[1020]

Hours = assumed annual run hours,

= 8766 for continuous ventilation.

$$\Delta kWh = (50 * (1/3.1 – 1/8.3)/1000) * 8766$$

$$= 88.6 \text{ kWh}$$

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = (CFM * (1/\eta_{BASELINE} - 1/\eta_{EFFICIENT})/1000) * CF$$

Where:

CF = Summer Peak Coincidence Factor

= 1.0 (continuous operation)

Other variables as defined above

$$\Delta kW = (50 * (1/3.1 – 1/8.3)/1000) * 1.0$$

$$= 0.0101 \text{ kW}$$

### NATURAL GAS SAVINGS

N/A

[1018] 50CFM is the closest available fan size to ASHRAE 62.2 Section 4.1 Whole House Ventilation rates based upon typical square footage and bedrooms.

[1019] VEIC analysis looking at average baseline fan (i.e. non-Brushless Permanent Magnet) efficacies at static pressures of 0.1 and 0.25 inches of water column for quiet fans rated for 50 CFM.

[1020] VEIC analysis looking at average efficient fan (i.e. Brushless Permanent Magnet) efficacies at static pressures of 0.1 and 0.25 inches of water column for quiet fans rated for 50 CFM.

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: RS-HVC-BAFA-V01-120601

## 5.3.10 HVAC Tune Up (Central Air Conditioning or Air Source Heat Pump)

### DESCRIPTION

This measure involves the measurement of refrigerant charge levels and airflow over the central air conditioning or heat pump unit coil, correction of any problems found and post-treatment re-measurement. Measurements must be performed with standard industry tools and the results tracked by the efficiency program.

Savings from this measure are developed using a reputable Wisconsin study. It is recommended that future evaluation be conducted in Illinois to generate a more locally appropriate characterization.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

N/A

### DEFINITION OF BASELINE EQUIPMENT

This measure assumes that the existing unit being maintained is either a residential central air conditioning unit or an air source heat pump that has not been serviced for at least 3 years.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 2 years[1021].

### DEEMED MEASURE COST

If the implementation mechanism involves delivering and paying for the tune up service, the actual cost should be used. If however the customer is provided a rebate and the program relies on private contractors performing the work, the measure cost should be assumed to be $175[1022].

### LOADSHAPE

Loadshape R08 - Residential Cooling

### COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during utility peak hour)

= 68%[1023]

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)

[1021] Based on VEIC professional judgment.

[1022] Based on personal communication with HVAC efficiency program consultant Buck Taylor or Roltay Inc., 6/21/10, who estimated the cost of tune up at $125 to $225, depending on the market and the implementation details.

[1023] Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

= 72%%[1024]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)

= 46.6%[1025]

**Algorithm**

**CALCULATION OF SAVINGS**

**ELECTRIC ENERGY SAVINGS**

$\Delta kWh_{Central\ AC}$ = (FLHcool * Capacity_cooling* (1/$SEER_{CAC}$))/1000 * MFe

$\Delta kWh_{Air\ Source\ Heat\ Pump}$ = ((FLHcool * Capacity_cooling * (1/$SEER_{ASHP}$))/1000 * MFe) + (FLHheat * Capacity_heating * (1/$HSPF_{ASHP}$))/1000 * MFe)

Where:

FLHcool = Full load cooling hours

Dependent on location as below:[1026]

| Climate Zone (City based upon) | FLHcool Single Family | FLHcool Multifamily |
|---|---|---|
| 1 (Rockford) | 512 | 467 |
| 2 (Chicago) | 570 | 506 |
| 3 (Springfield) | 730 | 663 |
| 4 (Belleville) | 1,035 | 940 |
| 5 (Marion) | 903 | 820 |
| Weighted Average[1027] | 629 | 564 |

Capacity_cooling = Cooling cpacity of equipment in Btu/hr (note 1 ton = 12,000 Btu/hr)

= Actual

$SEER_{CAC}$ = SEER Efficiency of existing central air conditioning unit receiving maintenence

[1024] Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

[1025] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

[1026] Based on Full Load Hours from ENERGY Star with adjustments made in a Navigant Evaluation, other cities were scaled using those results and CDD. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.

[1027] Weighted based on number of occupied residential housing units in each zone.

= Actual. If unknown assume 10 SEER [1028]

MFe = Maintenance energy savings factor

= 0.05[1029]

$SEER_{ASHP}$ = SEER Efficiency of existing air source heat pump unit receiving maintenence

= Actual. If unknown assume 10 SEER [1030]

FLHheat = Full load heating hours

Dependent on location:[1031]

| Climate Zone (City based upon) | FLHheat |
|---|---|
| 1 (Rockford) | 2208 |
| 2 (Chicago) | 2064 |
| 3 (Springfield) | 1967 |
| 4 (Belleville) | 1420 |
| 5 (Marion) | 1445 |
| Weighted Average[1032] | 1821 |

Capacity_heating = Heating cpacity of equipment in Btu/hr (note 1 ton = 12,000 Btu/hr)

= Actual

HSPFbase = Heating Season Performance Factor of existing air source heat pump unit receiving maintenence

= Actual. If unknown assume 6.8 HSPF [1033]

[1028] Use actual SEER rating where it is possible to measure or reasonably estimate. Unknown default of 10 SEER is a VEIC estimate of existing unit efficiency, based on minimum federal standard between the years of 1992 and 2006.
[1029] Energy Center of Wisconsin, May 2008; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research."
[1030] Use actual SEER rating where it is possible to measure or reasonably estimate. Unknown default of 10 SEER is a VEIC estimate of existing unit efficiency, based on minimum federal standard between the years of 1992 and 2006.
[1031] Full load heating hours for heat pumps are provided for Rockford, Chicago and Springfield in the Energy Star Calculator. Estimates for the other locations were calculated based on the FLH to Heating Degree Day (from NCDC) ratio. VEIC consider Energy Star estimates to be high due to oversizing not being adequately addressed. Using average Illinois billing data (from http://www.icc.illinois.gov/ags/consumereducation.aspx) VEIC estimated the average gas heating load and used this to estimate the average home heating output (using 83% average gas heat efficiency). Dividing this by a typical 36,000 Btu/hr ASHP gives an estimate of average ASHP FLH_heat of 1821 hours. We used the ratio of this value to the average of the locations using the Energy Star data (1994 hours) to scale down the Energy Star estimates. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.
[1032] Weighted based on number of occupied residential housing units in each zone.
[1033] Use actual HSPF rating where it is possible to measure or reasonably estimate. Unknown default of 6.8 HSPF is a VEIC estimate based on minimum Federal Standard between 1992 and 2006.

For example, maintenance of a 3-ton, SEER 10 air conditioning unit in a single family house in Springfield:

$\Delta kWh_{CAC}$ = (730 * 36,000 * (1/10))/1000 * 0.05

= 131 kWh

For example, maintenance of a 3-ton, SEER 10, HSPF 6.8 air source heat pump unit in a single family house in Springfield:

$\Delta kWh_{ASHP}$ = ((730 * 36,000 * (1/10))/1000 * 0.05) + (1967 * 36,000 * (1/6.8))/1000 * 0.05)

= 652 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = Capacity_cooling * (1/EER)/1000 * MFd * CF

Where:

EER = EER Efficiency of existing unit receiving maintenance in Btu/H/Watts

= Calculate using Actual SEER

= - $0.02*SEER^2 + 1.12*SEER$ [1034]

MFd = Maintenance demand savings factor

= 0.02 [1035]

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)

= 68%[1036]

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)

= 72%%[1037]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)

= 46.6%[1038]

[1034] Based on Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder. Note this is appropriate for single speed units only.

[1035] Based on June 2010 personal conversation with Scott Pigg, author of Energy Center of Wisconsin, May 2008; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research" suggesting the average WI unit system draw of 2.8kW under peak conditions, and average peak savings of 50W.

[1036] Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

[1037] Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

[1038] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

For example, maintenance of 3-ton, SEER 10 (equals EER 9.2) CAC unit:

$\Delta kW_{SSP}$ = 36,000 * 1/(9.2)/1000 * 0.02 * 0.68

= 0.0532 kW

$\Delta kW_{PJM}$ = 36,000 * 1/(9.2)/1000 * 0.02 * 0.466

= 0.0365 kW

**NATURAL GAS SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

Conservatively not included.

**MEASURE CODE: RS-HVC-TUNE-V02-140601**

## 5.3.11 Programmable Thermostats

### DESCRIPTION

This measure characterizes the household energy savings from the installation of a new or reprogramming of an existing Programmable Thermostat for reduced heating energy consumption through temperature set-back during unoccupied or reduced demand times. Because a literature review was not conclusive in providing a defensible source of prescriptive cooling savings from programmable thermostats, cooling savings from programmable thermostats are assumed to be zero for this version of the measure. It is not appropriate to assume a similar pattern of savings from setting a thermostat down during the heating season and up during the cooling season. Note that the EPA's EnergyStar program is developing a new specification for this project category, and if/when evaluation results demonstrate consistent cooling savings, subsequent versions of this measure will revisit this assumption[1039]. Energy savings are applicable at the household level; all thermostats controlling household heat should be programmable and installation of multiple programmable thermostats per home does not accrue additional savings.

This measure was developed to be applicable to the following program types: TOS, NC, RF, DI.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The criteria for this measure are established by replacement of a manual-only temperature control, with one that has the capability to adjust temperature setpoints according to a schedule without manual intervention. This category of equipment is broad and rapidly advancing in regards to the capability, and usability of the controls and their sophistication in setpoint adjustment and information display, but for the purposes of this characterization, eligibility is perhaps most simply defined by what it isn't: a manual only temperature control.

For the thermostat reprogramming measure, the auditor consults with the homeowner to determine an appropriate set back schedule, reprograms the thermostat and educates the homeowner on its appropriate use.

### DEFINITION OF BASELINE EQUIPMENT

For new thermostats the baseline is a non-programmable thermostat requiring manual intervention to change temperature setpoint.

For the purpose of thermostat reprogramming, an existing programmable thermostat that an auditor determines is being used in override mode or otherwise effectively being operated like a manual thermostat.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life of a programmable thermostat is assumed to be 10 years[1040] based upon equipment life only[1041]. For the purposes of claiming savings for a new programmable thermostat, this is reduced by a 50% persistence factor to give final measures life of 5 years. For reprogramming, this is reduced further to give a measure life of 2 years.

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[1039] The EnergyStar program discontinued its support for this measure category effective 12/31/09, and is presently developing a new specification for 'Residential Climate Controls'.

[1040] Table 1, HVAC Controls, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

[1041] Future evaluation is strongly encouraged to inform the persistence of savings to further refine measure life assumption. As this characterization depends heavily upon a large scale but only 2-year study of the energy impacts of programmable thermostats, the longer term impacts should be assessed.

### DEEMED MEASURE COST

Actual material and labor costs should be used if the implementation method allows. If unknown (e.g. through a retail program) the capital cost for the new installation measure is assumed to be $30[1042]. The cost for reprogramming is assumed to be $10 to account for the auditors time to reprogram and educate the homeowner.

### LOADSHAPE

Loadshape R09 - Residential Electric Space Heat

### COINCIDENCE FACTOR

N/A due to no savings attributable to cooling during the summer peak period.

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

ΔkWh[1043] = %ElectricHeat * Elec_Heating_Consumption * Heating_Reduction * HF * Eff_ISR + (ΔTherms * $F_e$ * 29.3)

Where:

%ElectricHeat = Percentage of heating savings assumed to be electric

| Heating fuel | %ElectricHeat |
|---|---|
| Electric | 100% |
| Natural Gas | 0% |
| Unknown | 13%[1044] |

Elec_Heating_ Consumption

= Estimate of annual household heating consumption for electrically heated single-family homes[1045]. If location and heating type is unknown, assume 15,678 kWh[1046]

[1042] Market prices vary significantly in this category, generally increasing with thermostat capability and sophistication. The basic functions required by this measure's eligibility criteria are available on units readily available in the market for the listed price.

[1043] Note the second part of the algorithm relates to furnace fan savings if the heating system is Natural Gas.

[1044] Average (default) value of 13% electric space heating from 2010 Residential Energy Consumption Survey for Illinois. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.

[1045] Values in table are based on converting an average household heating load (834 therms) for Chicago based on 'Table E-1, Energy Efficiency/Demand Response Nicor Gas Plan Year 1: Research Report: Furnace Metering Study, Draft, Navigant, August 1 2013 to an electric heat load (divide by 0.03413) to electric resistance and ASHP heat load (resistance load reduced by 15% to account for distribution losses that occur in furnace heating but not in electric resistance while ASHP heat is assumed to suffer

| Climate Zone (City based upon) | Electric Resistance Elec_Heating_ Consumption (kWh) | Electric Heat Pump Elec_Heating_ Consumption (kWh) |
|---|---|---|
| 1 (Rockford) | 21,741 | 12,789 |
| 2 (Chicago) | 20,771 | 12,218 |
| 3 (Springfield) | 17,789 | 10,464 |
| 4 (Belleville) | 13,722 | 8,072 |
| 5 (Marion) | 13,966 | 8,215 |
| Average | 19,743 | 11,613 |

Heating_Reduction = Assumed percentage reduction in total household heating energy consumption due to programmable thermostat

= 6.2%[1047]

HF = Household factor, to adjust heating consumption for non-single-family households.

| Household Type | HF |
|---|---|
| Single-Family | 100% |
| Multi-Family | 65%[1048] |
| Actual | Custom[1049] |

Eff_ISR = Effective In-Service Rate, the percentage of thermostats installed and programmed effectively

| Program Delivery | Eff_ISR |
|---|---|

from similar distribution losses) and then to electric consumption assuming efficiencies of 100% for resistance and 200% for HP (see 'Household Heating Load Summary Calculations_11062013.xls'). Finally these values were adjusted to a statewide average using relative HDD assumptions to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

[1046] Assumption that 1/2 of electrically heated homes have electric resistance and 1/2 have Heat Pump, based on 2010 Residential Energy Consumption Survey for Illinois.

[1047] The savings from programmable thermostats are highly susceptible to many factors best addressed, so far for this category, by a study that controlled for the most significant issues with a very large sample size. To the extent that the treatment group is representative of the program participants for IL, this value is suitable. Higher and lower values would be justified based upon clear dissimilarities due to program and product attributes. Future evaluation work should assess program specific impacts associated with penetration rates, baseline levels, persistence, and other factors which this value represents.

[1048] Multifamily household heating consumption relative to single-family households is affected by overall household square footage and exposure to the exterior. This 65% reduction factor is applied to MF homes based on professional judgment that average household size, and heat loads of MF households are smaller than single-family homes

[1049] Program-specific household factors may be utilized on the basis of sufficiently validated program evaluations.

| Program Delivery | Eff_ISR |
|---|---|
| Direct Install | 100% |
| Other, or unknown | 56%[1050] |

ΔTherms = Therm savings if Natural Gas heating system

= See calculation in Natural Gas section below

$F_e$ = Furnace Fan energy consumption as a percentage of annual fuel consumption

= 3.14%[1051]

29.3 = kWh per therm

For example, a programmable thermostat directly installed in an electric resistance heated, single-family home in Springfield:

ΔkWH = 1 * 17,789 * 0.062 * 100% * 100% + (0 * 0.0314 * 29.3)

= 1,103 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

N/A due to no savings from cooling during the summer peak period.

**NATURAL GAS ENERGY SAVINGS**

ΔTherms = %FossilHeat * Gas_Heating_Consumption * Heating_Reduction * HF * Eff_ISR

Where:

%FossilHeat = Percentage of heating savings assumed to be Natural Gas

| Heating fuel | %FossilHeat |
|---|---|
| Electric | 0% |
| Natural Gas | 100% |
| Unknown | 87%[1052] |

Gas_Heating_Consumption

= Estimate of annual household heating consumption for gas heated single-family

[1050] "Programmable Thermostats. Report to KeySpan Energy Delivery on Energy Savings and Cost Effectiveness," GDS Associates, Marietta, GA. 2002GDS

[1051] $F_e$ is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% $F_e$. See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

[1052] Average (default) value of 87% electric space heating from 2010 Residential Energy Consumption Survey for Illinois. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.

homes. If location is unknown, assume the average below[1053].

| Climate Zone (City based upon) | Gas_Heating_ Consumption (therms) |
|---|---|
| 1 (Rockford) | 1,052 |
| 2 (Chicago) | 1,005 |
| 3 (Springfield) | 861 |
| 4 (Belleville) | 664 |
| 5 (Marion) | 676 |
| Average | 955 |

For example, a programmable thermostat directly-installed in a gas heated single-family home in Chicago:

ΔTherms = 1.0 * 1005 * 0.062 * 100% * 100%

= 62.3 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-HVC-PROG-V03-140601**

[1053] Values are based on adjusting the average household heating load (834 therms) for Chicago based on 'Table E-1, Energy Efficiency / Demand Response Nicor Gas Plan Year 1, Research Report: Furnace Metering Study', divided by standard assumption of existing unit efficiency of 83% (estimate based on 24% of furnaces purchased in Illinois were condensing in 2000 (based on data from GAMA, provided to Department of Energy), assuming typical efficiencies: (0.24*0.92) + (0.76*0.8) = 0.83) to give 1005 therms. This Chicago value was then adjusted to a statewide average using relative HDD assumptions to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

## 5.3.12 Ductless Heat Pumps

### DESCRIPTION

This measure is designed to calculate electric savings for supplementing existing electric HVAC systems with ductless heat pumps. Existing systems can include: electric resistance heating or ducted air-source heat pumps. For ducted air source heat pumps, cooling savings are also possible if there is an existing air conditioning system.

Savings are achieved by displacing some of the heating or cooling load currently provided by the existing system and meeting that load with the more efficient ductless heat pump instead. The offset of the home's heating load is likely for the milder heating periods. The limitations on heating offset increase as the outdoor temperature drops, because the DHP capacity decreases, and the point-source nature of the heater is less able to satisfy heating loads in remote rooms.

For cooling, the proposed savings calculations are aligned with those of typical replacement systems. In most cases, the DHP is expected to replace (rather than offset) a comparable amount of cooling in homes with electric resistance heat—at a much higher efficiency than the previously used cooling.

In order for this measure to apply, the control strategy for the heat pump is assumed to be chosen to maximize savings per installer recommendation.[1054]

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the new equipment must be a high-efficiency, variable-capacity (typically "inverter-driven" DC motor) ductless heat pump system that exceeds the current Federal Standard. This means the unit must meet or exceed 8.2 HSPF (heating mode) and 14 SEER (cooling mode)[1055].

This measure only applies to the *first* ductless heat pump installed in a residence[1056].

### DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, baseline equipment must include a permanent electric resistance heating source or a ducted air-source heat pump. For multifamily buildings, each residence must have existing individual heating equipment. Multifamily residences with central heating do not qualify for this characterization. Existing cooling equipment is assumed to be standard efficiency. Note that in order to claim cooling savings, there must be an existing air conditioning system.

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[1054] The whole purpose of installing ductless heat pumps is to conserve energy, so the installer can be assumed to be capable of recommending an appropriate controls strategy. For most applications, the heating setpoint for the ductless heat pump should be at least 2F higher than any remaining existing system and the cooling setpoint for the ductless heat pump should be at least 2F cooler than the existing system (this should apply to all periods of a programmable schedule, if applicable). This helps ensure that the ductless heat pump will be used to meet as much of the load as possible before the existing system operates to meet the remaining load. Ideally, the new ductless heat pump controls should be set to the current comfort settings, while the existing system setpoints should be adjusted down (heating) and up (cooling) to capture savings.

[1055] Minimum Federal Standard as of 1/1/2015; http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf

[1056] Additional heat pumps will achieve additional savings, but not as much as the first one.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 18 years[1057].

### DEEMED MEASURE COST

The incremental cost for this measure is provided below:

| Unit Size | Incremental Cost[1058] |
|---|---|
| 1-Ton | $3,000 |
| 1.5-Ton | $3750 |
| 2-Ton | $4,500 |

### LOADSHAPE

Loadshape R10 - Residential Electric Heating and Cooling

### COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market. Both values provided are based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for ASHP (during utility peak hour)

= 72%%[1059]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)

= 46.6%[1060]

[1057] Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007

[1058] *Ductless Heat Pumps for Residential Customers in Connecticut*, Swift, Joseph R and Rebecca A. Meyer, The Connecticut Light & Power Company, 2010 ACEEE Summer Study on Energy Efficiency in Buildings (2-292)

[1059] Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

[1060] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

## Algorithms

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

Electric savings

$$\Delta kWh = \Delta kWh_{heat} + \Delta kWh_{cool}$$

$$\Delta kWh_{heat} = PLD*AHHL*HF*(1/HSPF_{exist}-1/HSPF_{ee})*3.413$$

$$\Delta kWh_{cool} = Capacity_{cool}*HF*(1/SEER_{exist}-1/SEER_{ee})*EFLH_{cool}$$

Where:

PLD = Percent Load Displaced. The average total annual heating load displaced from the existing heating system and now provided by the ductless heat pump[1061]

For a first DHP installed in a given home.

| | PLD5 | | |
|---|---|---|---|
| Climate zone | 1-ton unit | 1.5-ton unit | 2-ton unit |
| Rockford | 26% | 39% | 39% |
| Chicago | 27% | 40% | 42% |
| Springfield | 31% | 47% | 48% |
| Belleville | 30% | 45% | 48% |
| Marion | 31% | 46% | 50% |

AHHL = Annual Household Heating Load in kWh[1062]

[1061] PLD values calculated in "DHP Savings Model 12-31-13.xls". To verify that the proposed algorithm generates reasonable savings, we compared the results to metering studies done to measure ductless heat pump savings.
Ecotope Study, prepared for Bonneville Power Administration, "Residential Ductless Mini-Split Heat Pump Retrofit Monitoring," Monmouth, Oregon, June, 2009.
Ecotope Study, Prepared for Bonneville Power Administration, "Ductless Heat Pump Retrofits in Multifamily and Small Commercial Buildings," December, 2012.
KEMA Study, Prepared for NSTAR Electric and Gas Corporation et al. "Ductless Mini Pilot Study," Middletown, Connecticut, June, 2009

[1062] Values in table are based on converting an average household heating load (834 therms) for Chicago based on 'Table E-1, Energy Efficiency/Demand Response Nicor Gas Plan Year 1: Research Report: Furnace Metering Study, Draft, Navigant, August 1 2013 to an electric heat load (divide by 0.03413) to electric resistance and ASHP heat load (resistance load reduced by 15% to account for distribution losses that occur in furnace heating but not in electric resistance while ASHP heat is assumed to suffer from similar distribution losses) (see 'Household Heating Load Summary Calculations_11062013.xls'). Finally these values were adjusted to a statewide average using relative HDD assumptions to adjust for the evaluation results focus on northern region.

| Climate Zone | Annual Household Heating Load Resistance (kWh) | Annual Household Heating Load ASHP (kWh) |
|---|---|---|
| 1 (Rockford) | 21,741 | 25,578 |
| 2 (Chicago) | 20,771 | 24,436 |
| 3 (Springfield) | 17,789 | 20,928 |
| 4 (Belleville) | 13,722 | 16,144 |
| 5 (Marion) | 13,966 | 16,431 |
| Average | 19,743 | 23,227 |

HF = Household factor, to adjust heating consumption for non-single-family households.

| Household Type | HF |
|---|---|
| Single-Family | 100% |
| Multi-Family | 65%[1063] |
| Actual | Custom[1064] |

$Capacity_{cool}$ = the cooling capacity of the ductless heat pump unit in kBtu/hr[1065].

= Actual installed

$HSPF_{ee}$ = HSPF rating of new equipment

= Actual installed

$HSPF_{exist}$ = HSPF rating of existing equipment

| Existing Equipment Type | HSPFbase |
|---|---|
| Electric resistance heating | 3.41[1066] |
| Air Source Heat Pump | 5.44[1067] |

Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

[1063] Multifamily household heating consumption relative to single-family households is affected by overall household square footage and exposure to the exterior. This 65% reduction factor is applied to MF homes with electric resistance, based on professional judgment that average household size, and heat loads of MF households are smaller than single-family homes

[1064] Program-specific household factors may be utilized on the basis of sufficiently validated program evaluations.

[1065] 1 Ton = 12 kBtu/hr

[1066] Electric resistance has a COP of 1.0 which equals 1/0.293 = 3.41 HSPF.

$SEER_{ee}$ = SEER rating of new equipment

= Actual installed[1068]

$SEER_{exist}$ = SEER rating of existing equipment

= Use actual value. If unknown, see table below

| Equipment Type | SEERexist[1069] |
|---|---|
| PTAC | 7.4 SEER |
| PTHP | 7.4 SEER |
| SPVAC < 65kBtu/hr | 9.0 SEER |
| SPVHP < 65 kBtu/hr | 9.0 SEER |
| Room AC | 7.0 SEER |
| Ducted ASHP | 13.0 SEER |
| No existing system | No cooling savings. |

$EFLH_{cool}$ = Equivalent Full Load Hours for cooling. Depends on location. See table below[1070].

| Climate Zone (City based upon) | FLHRoomAC |
|---|---|
| 1 (Rockford) | 220 |
| 2 (Chicago) | 210 |
| 3 (Springfield) | 319 |
| 4 (Belleville) | 428 |
| 5 (Marion) | 374 |
| Weighted Average[1071] | 248 |

[1067] This is from the ASHP measure which estimated HSPF based on finding the average HSPF/SEER ratio from the AHRI directory data (using the least efficient models – SEER 12 and SEER 13) – 0.596, and applying to the average nameplate SEER rating of all Early Replacement qualifying equipment in Ameren PY3-PY4. This estimation methodology appears to provide a result within 10% of actual HSPF.
[1068] Note that if only an EER rating is available, a conversion factor of SEER=1.1*EER can be used
[1069] Converted from EER using formula EER = 1.1 SEER
[1070] Residential EFLH for room AC
[1071] Weighted based on number of residential occupied housing units in each zone.

For example, installing a 1.5-ton (heating and cooling capacity) ductless heat pump unit rated at 8 HSPF and 14 SEER in a single-family home in Chicago to displace electric baseboard heat and replace a window air conditioner, savings are:

$\Delta kWh_{heat}$ = 40% x 20,771kWh x 100% x (1/3.41 – 1/8) x 3.413 = 4,771kWh

$\Delta kWh_{cool}$ = 18 x 100% x (1/7 – 1/14) x 210 = 270kWh

ΔkWh = 4,771 + 270 = 5,041kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = (Capacity_cooling *HF* (1/EER_exist - 1/EER_ee)) / 1000) * CF

Where:

EER_exist = Energy Efficiency Ratio of existing cooling system (kBtu/hr / kW)

= Use actual EER rating otherwise:

| Equipment Type | EERexist |
|---|---|
| PTAC | 8.1EER[1072] |
| PTHP | 8.1EER[1073] |
| SPVAC < 65kBtu/hr | 9.9 EER [1074] |
| SPVHP < 65 kBtu/hr | 9.9 EER[1075] |
| Room AC | 7.7 EER[1076] |
| Ducted ASHP | 11.2 EER [1077] |
| No existing system | |

EER_ee = Energy Efficiency Ratio of new ductless Air Source Heat Pump (kBtu/hr / kW)

= Actual, If not provided convert SEER to EER using this formula:

[1072] Same EER as PTAC recycling. Estimated using the IECC building energy code up until year 2003 (p107; https://law.resource.org/pub/us/code/ibr/icc.iecc.2000.pdf) and assuming a 1 ton unit; EER = 10 – (0.16 * 12,000/1,000) = 8.1.
[1073] Same method to calculate EER as PTAC recycling. Estimated using the IECC building energy code up until year 2003 (p107; https://law.resource.org/pub/us/code/ibr/icc.iecc.2000.pdf) and assuming a 1 ton unit; EER = 10 – (0.16 * 12,000/1,000) = 8.1.
[1074] The quoted efficiency rating in the IECC was given in EER and was translated to SEER using a conversion factor of SEER=1.1*EER.
[1075] Ibid.
[1076] Same EER as Window AC recycling. Based on Nexus Market Research Inc, RLW Analytics, December 2005; "Impact, Process, and Market Study of the Connecticut Appliance Retirement Program: Overall Report."
[1077] The Federal Standard does not include an EER requirement, so it is approximated with this formula: (-0.02 * SEER2) + (1.12 * SEER) Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder. Note this is appropriate for single speed units only.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)

= 72%%[1078]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)

= 46.6%[1079]

**NATURAL GAS SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-HVC-DHP-V02-150601**

[1078] Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

[1079] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

## 5.3.13 Residential Furnace Tune-Up

**DESCRIPTION**

This measure is for a natural gas Residential furnace that provides space heating. The tune-up will improve furnace performance by inspecting, cleaning and adjusting the furnace and appurtenances for correct and efficient operation. Additional savings maybe realized through a complete system tune-up.

This measure was developed to be applicable to the following program types: Residential.

If applied to other program types, the measure savings should be verified.

**DEFINITION OF EFFICIENT EQUIPMENT**

To qualify for this measure an approved technician must complete the tune-up requirements[1080] listed below:

- Measure combustion efficiency using an electronic flue gas analyzer
- Check and clean blower assembly and components per manufacturer's recommendations
- Where applicable Lubricate motor and inspect and replace fan belt if required
- Inspect for gas leaks
- Clean burner per manufacturer's recommendations and adjust as needed
- Check ignition system and safety systems and clean and adjust as needed
- Check and clean heat exchanger per manufacturer's recommendations
- Inspect exhaust/flue for proper attachment and operation
- Inspect control box, wiring and controls for proper connections and performance
- Check air filter and clean or replace per manufacturer's
- Inspect duct work connected to furnace for leaks or blockages
- Measure temperature rise and adjust flow as needed
- Check for correct line and load volts/amps
- Check thermostat operation is per manufacturer's recommendations(if adjustments made, refer to 'Residential Programmable Thermostat' measure for savings estimate)
- Perform Carbon Monoxide test and adjust heating system until results are within standard industry acceptable limits

**DEFINITION OF BASELINE EQUIPMENT**

The baseline is furnace assumed not to have had a tune-up in the past 2 years.

**DEEMED LIFETIME OF EFFICIENT EQUIPMENT**

The measure life for the tune up is 2 years.[1081]

[1080] American Standard Maintenance for Indoor Units: http://www.americanstandardair.com/owner-support/maintenance.html
[1081] Act on Energy Commercial Technical Reference Manual No. 2010-4, 9.2.3 Gas Forced-Air Furnace Tune-up.

**DEEMED MEASURE COST**

The incremental cost for this measure should be the actual cost of tune up.

**DEEMED O&M COST ADJUSTMENTS**

There are no expected O&M savings associated with this measure.

**LOADSHAPE**

Loadshape R09 - Residential Electric Space Heat

**COINCIDENCE FACTOR**

N/A

## Algorithms

**CALCULATION OF ENERGY SAVINGS**

**ELECTRIC ENERGY SAVINGS**

$$\Delta kWh = \Delta Therms * F_e * 29.3$$

Where:

| | |
|---|---|
| ΔTherms | = as calculated below |
| $F_e$ | = Furnace Fan energy consumption as a percentage of annual fuel consumption |
| | = 3.14%[1082] |
| 29.3 | = kWh per therm |

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

N/A

[1082] $F_e$ is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% $F_e$. See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

**NATURAL GAS SAVINGS**

Δtherms =( Gas_Furnace_Heating_Load *HF * (1/ Effbefore – 1/ (Effbefore + Ei)))

Where:

Gas_Furnace_Heating_Load = Estimate of annual household heating load[1083] for gas furnace heated single-family homes. If location is unknown, assume the average below[1084].

= Actual if informed by site-specific load calculations, ACCA Manual J or equivalent[1085].

| Climate Zone (City based upon) | Gas_Furnace_Heating_Load (therms) |
|---|---|
| 1 (Rockford) | 873 |
| 2 (Chicago) | 834 |
| 3 (Springfield) | 714 |
| 4 (Belleville) | 551 |
| 5 (Marion) | 561 |
| Average | 793 |

HF = Household factor, to adjust heating consumption for non-single-family households.

| Household Type | HF |
|---|---|
| Single-Family | 100% |
| Multi-Family | 65%[1086] |
| Actual | Custom[1087] |

Effbefore = Efficiency of the furnace before the tune-up

[1083] Heating load is used to describe the household heating need, which is equal to (gas consumption * AFUE )

[1084] Values are based on household heating consumption values and inferred average AFUE results from Table 2-1, Energy Efficiency / Demand Response Nicor Gas Plan Year 1 (6/1/2011-5/31/2012) Research Report: Furnace Metering Study (August 1, 2013) (prepared by Navigant Consulting, Inc.) and adjusting to a statewide average using relative HDD values to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

[1085] The Air Conditioning Contractors of America Manual J, Residential Load Calculation 8th Edition produces equipment sizing loads for Single Family, Multi-single, and Condominiums using input characteristics of the home.

[1086] Multifamily household heating consumption relative to single-family households is affected by overall household square footage and exposure to the exterior. This 65% reduction factor is applied to MF homes with electric resistance, based on professional judgment that average household size, and heat loads of MF households are smaller than single-family homes

[1087] Program-specific household factors may be utilized on the basis of sufficiently validated program evaluations.

= Actual

EI = Efficiency Improvement of the furnace tune-up measure

= Actual

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-HVC-FTUN-V01-150601**

## 5.3.14 Boiler Reset Controls

**DESCRIPTION**

This measure relates to improving system efficiency by adding controls to residential heating boilers to vary the boiler entering water temperature relative to heating load as a function of the outdoor air temperature to save energy. The water can be run a little cooler during fall and spring, and a little hotter during the coldest parts of the winter. A boiler reset control has two temperature sensors - one outside the house and one in the boiler water. As the outdoor temperature goes up and down, the control adjusts the water temperature setting to the lowest setting that is meeting the house heating demand. There are also limits in the controls to keep a boiler from operating outside of its safe performance range.[1088]

This measure was developed to be applicable to the following program types: RF.

**DEFINITION OF EFFICIENT EQUIPMENT**

Natural gas single family residential customer adding boiler reset controls capable of resetting the boiler supply water temperature in an inverse fashion with outdoor air temperature. The system must be set so that the minimum temperature is not more than 10 degrees above manufacturer's recommended minimum return temperature. This boiler reset measure is limited to existing condensing boilers serving a single family residence. Boiler reset controls for non-condensing boilers in single family residences should be implemented as a custom measure, and the cost-effectiveness should be confirmed.

**DEFINITION OF BASELINE EQUIPMENT**

Existing condensing boiler in a single family residential setting without boiler reset controls.

**DEEMED LIFETIME OF EFFICIENT EQUIPMENT**

The life of this measure is 20 years[1089]

**DEEMED MEASURE COST**

The cost of this measure is $612[1090]

**LOADSHAPE**

NA

**COINCIDENCE FACTOR**

NA

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[1088] Energy Solutions Center, a consortium of natural gas utilities, equipment manufacturers and vendors. Boiler Reset Control, accessed at http://naturalgasefficiency.org/residential/Boiler_Reset_Control.htm

[1089] CLEAResultreferences the Brooklyn Union Gas Company, High Efficiency Heating and Water and Controls, Gas Energy Efficiency Program Implementation Plan.

[1090] Nexant. Questar DSM Market Characterization Report. August 9, 2006.

## Algorithm

### CALCULATION OF ENERGY SAVINGS

### ELECTRIC ENERGY SAVINGS

NA

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

NA

### NATURAL GAS SAVINGS

ΔTherms = Gas_Boiler_Load * (1/AFUE) * Savings Factor

Where:

Gas_Boiler_Load[1091]

= Estimate of annual household Load for gas boiler heated single-family homes. If location is unknown, assume the average below[1092].

= or Actual if informed by site-specific load calculations, ACCA Manual J or equivalent[1093].

| Climate Zone (City based upon) | Gas_Boiler Load (therms) |
|---|---|
| 1 (Rockford) | 1275 |
| 2 (Chicago) | 1218 |
| 3 (Springfield) | 1043 |
| 4 (Belleville) | 805 |

[1091] Boiler consumption values are informed by an evaluation which did not identify any fraction of heating load due to domestic hot water (DHW) provided by the boiler. Thus these values are an average of both homes with boilers only providing heat, and homes with boilers that also provide DHW. Heating load is used to describe the household heating need, which is equal to (gas heating consumption * AFUE )

[1092] Values are based on household heating consumption values and inferred average AFUE results from Table 3-4, Program Sample Analysis, *Nicor R29 Res Rebate Evaluation Report 092611_REV FINAL to Nicor*). Adjusting to a statewide average using relative HDD values to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

[1093] The Air Conditioning Contractors of America Manual J, Residential Load Calculation 8th Edition produces equipment sizing loads for Single Family, Multi-single, and Condominiums using input characteristics of the home. A best practice for equipment selection and installation of Heating and Air Conditioning, load calculations should be completed by contractors during the selection process and may be readily available for program data purposes.

| Climate Zone (City based upon) | Gas_Boiler Load (therms) |
|---|---|
| 5 (Marion) | 819 |
| Average | 1158 |

AFUE = Existing Condensing Boiler Annual Fuel Utilization Efficiency Rating

= Actual.

SF = Savings Factor, 5%[1094]

**EXAMPLE**

For example, boiler reset controls on a 92.5 AFUE boiler at a household in Rockford, IL

ΔTherms = 1275 * (1/0.925) * 0.05

= 69 Therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

NA

**DEEMED O&M COST ADJUSTMENT CALCULATION**

NA

**MEASURE CODE: RS-HVC-BREC-V01-150601**

[1094] Energy Solutions Center, a consortium of natural gas utilities, equipment manufacturers and vendors. Boiler Reset Control, accessed at http://naturalgasefficiency.org/residential/Boiler_Reset_Control.htm

## 5.3.15 ENERGY STAR Ceiling Fan

### DESCRIPTION

A ceiling fan/light unit meeting the efficiency specifications of ENERGY STAR is installed in place of a model meeting the federal standard. ENERGY STAR qualified ceiling fan/light combination units are over 60% more efficient than conventional fan/light units, and use improved motors and blade designs[1095].

Due to the savings from this measure being derived from more efficient ventilation and more efficient lighting, and the loadshape and measure life for each component being very different, the savings are split in to the component parts and should be claimed together. Lighting savings should be estimated utilizing the 5.5.1 ENERGY STAR Compact Fluorescent Lamp measure.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is defined as an ENERGY STAR certified ceiling fan with integral CFL bulbs.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be a standard fan with efficient incandescent or halogen light bulbs. Production of 100W, standard efficacy incandescent lamps ended in 2012 followed by restrictions on 75W in 2013 and 60W and 40W in 2014, due to the Energy Independence and Security Act of 2007 (EISA). Finally, a provision in the EISA regulations requires that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in essence making the baseline equivalent to a current day CFL. Therefore the measure life (number of years that savings should be claimed) for the lighting portion of the savings should be reduced once the assumed lifetime of the bulb exceeds 2020. Due to expected delay in clearing retail inventory and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until mid-2020.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The fan savings measure life is assumed to be 10 years.[2]

The lighting savings measure life is assumed to be 5 years for lighting savings, see 5.5.1 ENERGY STAR Compact Fluorescent Lamp measure.

### DEEMED MEASURE COST

Incremental cost of unit is $46.[1096]

### LOADSHAPE

R06 - Residential Indoor Lighting

R11 - Residential Ventilation

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[1095] http://www.energystar.gov/products/certified-products/detail/ceiling-fans

[1096] ENERGY STAR Ceiling Fan Savings Calculator
http://www.energystar.gov/buildings/sites/default/uploads/files/light_fixture_ceiling_fan_calculator.xlsx?8178-e52c

#### COINCIDENCE FACTOR

The summer peak coincidence factor for the ventilation savings is assumed to be 30%.[1097]

For lighting savings, see 5.5.1 ENERGY STAR Compact Fluorescent Lamp measure.

## Algorithm

#### CALCULATION OF SAVINGS

#### ELECTRIC ENERGY SAVINGS

$\Delta kWh = \Delta kWh_{fan} + \Delta kWh_{Light}$

$\Delta kWh_{fan}$ = [Days * FanHours * ((%$Low_{base}$ * $WattsLow_{base}$) + (%$Med_{base}$ * $WattsMed_{base}$) + (%$High_{base}$ * $WattsHigh_{base}$))/1000 ] - [Days * FanHours * ((%$Low_{ES}$ * $WattsLow_{ES}$) + (%$Med_{ES}$ * $WattsMed_{ES}$) + (%$High_{ES}$ * $WattsHigh_{ES}$))/1000]

$\Delta kWh_{light}$ = see 5.5.1 ENERGY STAR Compact Fluorescent Lamp measure.

Where[1098]:

Days = Days used per year

= Actual. If unknown use 365.25 days/year

FanHours = Daily Fan "On Hours"

= Actual. If unknown use 3 hours

%$Low_{base}$ = Percent of time spent at Low speed of baseline

= 40%

$WattsLow_{base}$ = Fan wattage at Low speed of baseline

= Actual. If unknown use 15 watts

%$Med_{base}$ = Percent of time spent at Medium speed of baseline

= 40%

$WattsMed_{base}$ = Fan wattage at Medium speed of baseline

= Actual. If unknown use 34 watts

[1097] Assuming that the CF same as a Room AC. Consistent with coincidence factors found in: RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008 (http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RAC.pdf)

[1098] All fan default assumptions are based upon assumptions provided in the ENERGY STAR Ceiling Fan Savings Calculator; http://www.energystar.gov/buildings/sites/default/uploads/files/light_fixture_ceiling_fan_calculator.xlsx?8178-e52c

$\%High_{base}$ = Percent of time spent at High speed of baseline

= 20%

$WattsHigh_{base}$ = Fan wattage at High speed of baseline

= Actual. If unknown use 67 watts

%LowES = Percent of time spent at Low speed of ENERGY STAR

= 40%

$WattsLow_{ES}$ = Fan wattage at Low speed of ENERGY STAR

= Actual. If unknown use 6 watts

$\%Med_{ES}$ = Percent of time spent at Medium speed of ENERGY STAR

= 40%

$WattsMed_{ES}$ = Fan wattage at Medium speed of ENERGY STAR

= Actual. If unknown use 23 watts

$\%High_{ES}$ = Percent of time spent at High speed of ENERGY STAR

= 20%

$WattsHigh_{ES}$ = Fan wattage at High speed of ENERGY STAR

= Actual. If unknown use 56 watts

For ease of reference, the fan assumptions are provided below in table form:

| | Low Speed | Medium Speed | High Speed |
|---|---|---|---|
| Percent of Time at Given Speed | 40% | 40% | 20% |
| Conventional Unit Wattage | 15 | 34 | 67 |
| ENERGY STAR Unit Wattage | 6 | 23 | 56 |
| ΔW | 9 | 11 | 11 |

If the lighting WattsBase and WattsEE is unknown, assume the following

WattsBase = 3 x 43 = 129 W

WattsEE = 1 x 42 = 42 W

**EXAMPLE**

For example, a ceiling fan with three 43W bulb light fixtures, replaced with an ES ceiling fan with one 42W bulb light fixture, the savings are:

$\Delta kWh_{fan}$ = [365.25*3*((0.4*15)+(0.4*34)+(0.2*67))/1000] – [365.25*3*((0.4*6)+(0.4*23)+(0.2*56))/1000]

= 36.2 – 25.0 = 11.2 kWh

$\Delta kWh_{light}$ =((129 – 42)/1000) *759 * 1.06

= 70.0 kWh

$\Delta kWh$ = 11.2 + 70

=81.2 kWh

Using the default assumptions provided above, the deemed savings is 81.2 kWh.

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$\Delta kW = \Delta kW_{Fan} + \Delta kW_{light}$

$\Delta kW_{Fan} = ((WattsHigh_{base} - WattsHigh_{ES})/1000) * CF_{fan}$

$\Delta kW_{Light}$ = see 5.5.1 ENERGY STAR Compact Fluorescent Lamp measure.

Where:

$CF_{fan}$ = Summer Peak coincidence factor for ventilation savings

= 30%[1099]

$CF_{light}$ = Summer Peak coincidence factor for lighting savings

= 7.1%[1100]

[1099] Assuming that the CF same as a Room AC. Consistent with coincidence factors found in: RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008 (http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RAC.pdf)

[1100] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

**EXAMPLE**

For example a ceiling fan with three 43W bulb light fixtures, replaced with an ES ceiling fan with one 42W bulb light fixture, the savings are:

$\Delta kW_{fan}$ = ((67-56)/1000) * 0.3

=0.0033 kW

$\Delta kW_{light}$ =((129 – 42)/1000) * 1.11 * 0.071

= 0.0068 kW

$\Delta kW$ = 0.0033 + 0.0068

= 0.010 kW

Using the default assumptions provided above, the deemed savings is 0.010kW.

**NATURAL GAS SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

See 5.5.1 ENERGY STAR Compact Fluorescent Lamp measure for bulb replacement costs.

**MEASURE CODE: RS-HVC-CFAN-V01-150601**

# 5.4 Hot Water End Use

## 5.4.1 Domestic Hot Water Pipe Insulation

### DESCRIPTION

This measure describes adding insulation to un-insulated domestic hot water pipes. The measure assumes the pipe wrap is installed to the first length of both the hot and cold pipe up to the first elbow. This is the most cost effective section to insulate since the water pipes act as an extension of the hot water tank up to the first elbow which acts as a heat trap. Insulating this length therefore helps reduce standby losses. Default savings are provided per 3ft length and are appropriate up to 6ft of the hot water pipe and 3ft of the cold.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient case is installing pipe wrap insulation to a length of hot water pipe.

### DEFINITION OF BASELINE EQUIPMENT

The baseline is an un-insulated hot water pipe.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 15 years[1101].

### DEEMED MEASURE COST

The measure cost including material and installation is assumed to be $3 per linear foot[1102].

### LOADSHAPE

Loadshape C53 - Flat

### COINCIDENCE FACTOR

This measure assumes a flat loadshape since savings relate to reducing standby losses and as such the coincidence factor is 1.

[1101] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf

[1102] Consistent with DEER 2008 Database Technology and Measure Cost Data (www.deeresources.com).

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

For electric DHW systems:

$$\Delta kWh = ((1/Rexist - 1/Rnew) * (L * C) * \Delta T * 8{,}766)/ \eta DHW / 3413$$

Where:

Rexist = Pipe heat loss coefficient of uninsulated pipe (existing) [(hr-°F-ft)/Btu]

= 1.0[1103]

Rnew = Pipe heat loss coefficient of insulated pipe (new) [(hr-°F-ft)/Btu]

= Actual (1.0 + R value of insulation)

L = Length of pipe from water heating source covered by pipe wrap (ft)

= Actual

C = Circumference of pipe (ft) (Diameter (in) * π/12)

= Actual (0.5" pipe = 0.131ft, 0.75" pipe = 0.196ft)

ΔT = Average temperature difference between supplied water and outside air temperature (°F)

= 60°F [1104]

8,766 = Hours per year

ηDHW = Recovery efficiency of electric hot water heater

= 0.98 [1105]

3412 = Conversion from Btu to kWh

For example, insulating 5 feet of 0.75" pipe with R-5 wrap:

$$\Delta kWh = ((1/Rexist - 1/Rnew) * (L * C) * \Delta T * 8{,}766) / \eta DHW / 3412$$

$$= ((1/1- 1/(1+5) * (5 * 0.196) * 60 * 8766) / 0.98 /3412$$

$$= 128 \text{ kWh}$$

If inputs above are not available the following default per 3ft R-5 length can be used for up to 6 ft length on the hot pipe and 3 ft on the cold pipe.

$$\Delta kWh = ((1/Rexist - 1/Rnew) * (L * C) * \Delta T * 8{,}766) / \eta DHW / 3412$$

$$= ((1/1- 1/(1+5)) * (3 * 0.196) * 60 * 8766) / 0.98 /3412$$

[1103] Navigant Consulting Inc., April 2009; "Measures and Assumptions for Demand Side Management (DSM) Planning; Appendix C Substantiation Sheets", p77.
[1104] Assumes 125°F water leaving the hot water tank and average temperature of basement of 65°F.
[1105] Electric water heaters have recovery efficiency of 98%: http://www.ahridirectory.org/ahridirectory/pages/home.aspx

= 77.1 kWh per 3ft length

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ΔkWh / 8766

Where:

ΔkWh = kWh savings from pipe wrap installation

8766 = Number of hours in a year (since savings are assumed to be constant over year).

For example, insulating 5 feet of 0.75" pipe with R-5 wrap:

ΔkW = 128/8766

= 0.015kW

If inputs above are not available the following default per 3ft R-4 length can be used for up to 6 ft length on the hot pipe and 3 ft on the cold pipe.

ΔkW = 77.1/8766

= 0.0088 kW

**NATURAL GAS SAVINGS**

For Natural Gas DHW systems:

ΔTherm = ((1/Rexist – 1/Rnew) * (L * C) * ΔT * 8,766) / ηDHW /100,000

Where:

ηDHW = Recovery efficiency of gas hot water heater

= 0.78 [1106]

Other variables as defined above

For example, insulating 5 feet of 0.75" pipe with R-5 wrap:

ΔTherm = ((1/1– 1/(1+5)) * (5 * 0.196) * 60 * 8766) / 0.78 / 100,000

= 5.51 therms

If inputs above are not available the following default per 3ft R-4 length can be used for up to 6ft length on the hot pipe and 3ft on the cold pipe.

ΔTherm = ((1/Rexist – 1/Rnew) * (L * C) * ΔT * 8,766) / ηDHW / 100,000

= ((1/1– 1/(1+5)) * (3 * 0.196) * 60 * 8766) / 0.78 /100,000

= 3.30 therms per 3ft length

[1106] Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 78%

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: RS-HWE-PINS-V02-150601

## 5.4.2 Gas Water Heater

### DESCRIPTION

This measure characterizes:

a) Time of sale or new construction:
The purchase and installation of a new efficient gas-fired water heater, in place of a Federal Standard unit in a residential setting. Savings are provided for power-vented, condensing storage, and whole-house tankless units meeting specific EF criteria.

b) Early replacement:
The early removal of an existing functioning natural gas water heater from service, prior to its natural end of life, and replacement with a new high efficiency unit. Savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life.

This measure was developed to be applicable to the following program types: TOS, NC, EREP.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the efficient equipment must be a water heater rated with the following minimum efficiency ratings:

| Water Heater Type | Minimum Energy Factor |
|---|---|
| Gas Storage | 0.67 |
| Condensing gas storage | 0.80 |
| Tankless whole-house unit | 0.82 |

### DEFINITION OF BASELINE EQUIPMENT

Time of Sale or New Construction: The baseline condition is assumed to be a standard gas storage water heater of the same capacity as the efficient unit, rated at the federal minimum. For 20 to 55 gallon tanks the Federal Standard is calculated as 0.675 – (0.0015 * storage size in gallons) and for tanks 55 - 100 gallon 0.8012 – (0.00078 * storage size in gallons)[1107]. For a 40-gallon storage water heater this would be 0.615 EF.

Early replacement: The baseline for this measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and a new baseline unit for the remainder of the measure life.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 13 years.[1108]

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[1107] Minimum Federal Standard as of 4/1/2015;
http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf
[1108] DOE, 2010 Residential Heating Products Final Rule Technical Support Document, Table 8.2.14
http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/htgp_finalrule_ch8.pdf Note: This source is

For early replacement: Remaining life of existing equipment is assumed to be 4 years[1109].

### DEEMED MEASURE COST

Time of Sale or New Construction:

The incremental capital cost for this measure is dependent on the type of water heater as listed below[1110].

Early Replacement: The full installed cost is provided in the table below. The assumed deferred cost (after 4 years) of replacing existing equipment with a new baseline unit is assumed to be $614[1111]. This cost should be discounted to present value using the utility's discount rate.

| Water heater Type | Incremental Cost | Full Install Cost |
|---|---|---|
| Gas Storage | $400 | $1014 |
| Condensing gas storage | $685 | $1299 |
| Tankless whole-house unit | $605 | $1219 |

### LOADSHAPE

N/A

### COINCIDENCE FACTOR

N/A

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

N/A

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

used to support this category in aggregate. For all water heaters, life expectancy will depend on local variables such as water chemistry and homeowner maintenance. Some categories, including condensing storage and tankless water heaters do not yet have sufficient field data to support separate values. Preliminary data show lifetimes may exceed 20 years, though this has yet to be sufficiently demonstrated.

[1109] Assumed to be one third of effective useful life

[1110] Source for cost info; DOE, 2010 Residential Heating Products Final Rule Technical Support Document, Table 8.2.14 (http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/htgp_finalrule_ch8.pdf)

[1111] The deemed install cost of a Gas Storage heater is based upon DCEO Efficient Living Program Data for a sample size of 157 gas water heaters.

**NATURAL GAS ENERGY SAVINGS**

Time of Sale or New Construction:

$\Delta Therms = (1/ EF_{BASE} - 1/EF_{EFFICIENT}) * (GPD * Household * 365.25 * \gamma Water * (T_{OUT} – T_{IN}) * 1.0 )/100,000$

Early replacement[1112]:

ΔTherms for remaining life of existing unit (1st 4 years):

$= (1/ EF_{EXISTING} - 1/EF_{EFFICIENT}) * (GPD * Household * 365.25 * \gamma Water * (T_{OUT} – T_{IN}) * 1.0 )/100,000$

ΔTherms for remaining measure life (next 9 years):

$= (1/ EF_{BASE} - 1/EF_{EFFICIENT}) * (GPD * Household * 365.25 * \gamma Water * (T_{OUT} – T_{IN}) * 1.0 )/100,000$

Where:

EF_Baseline = Energy Factor rating for baseline equipment

For <=55 gallons: 0.675 – (0.0015 * tank_size)

For > 55 gallons: 0.8012 – (0.00078 * tank size)

= If tank size unknown assume 40 gallons and EF_Baseline of 0.615

EF_Efficient = Energy Factor Rating for efficient equipment

= Actual. If Tankless whole-house multiply rated efficiency by 0.91[1113]. If unknown assume values in look up in table below

| Water Heater Type | EF_Efficient |
|---|---|
| Condensing Gas Storage | 0.80 |
| Gas Storage | 0.67 |
| Tankless whole-house | 0.82 * 0.91 = 0.75 |

EF_Existing = Energy Factor rating for existing equipment

= Use actual EF rating where it is possible to measure or reasonably estimate.

= if unknown assume 0.52 [1114]

GPD = Gallons Per Day of hot water use per person

[1112] The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).

[1113] The disconnect between rated energy factor and in-situ energy consumption is markedly different for tankless units due to significantly higher contributions to overall household hot water usage from short draws. In tankless units the large burner and unit heat exchanger must fire and heat up for each draw. The additional energy losses incurred when the mass of the unit cools to the surrounding space in-between shorter draws was found to be 9% in a study prepared for Lawrence Berkeley National Laboratory by Davis Energy Group, 2006. "Field and Laboratory Testing of Tankless Gas Water Heater Performance" Due to the similarity (storage) between the other categories and the baseline, this derating factor is applied only to the tankless category.

[1114] Based on DCEO Efficient Living Program Data for a sample size of 157 gas water heaters.

= 45.5 gallons hot water per day per household/2.59 people per household[1115]

= 17.6

Household = Average number of people per household

| Household Unit Type | Household |
|---|---|
| Single-Family - Deemed | 2.56[1116] |
| Multi-Family - Deemed | 2.1[1117] |
| Custom | Actual Occupancy or Number of Bedrooms[1118] |

365.25 = Days per year, on average

γWater = Specific Weight of water

= 8.33 pounds per gallon

$T_{OUT}$ = Tank temperature

= 125°F

$T_{IN}$ = Incoming water temperature from well or municipal system

= 54°F[1119]

1.0 = Heat Capacity of water (1 Btu/lb*°F)

For example, a 40 gallon condensing gas storage water heater, with an energy factor of 0.80 in a single family house:

ΔTherms = (1/0.615 - 1/0.8) * (17.6 * 2.56 * 365.25* 8.33 * (125 – 54) * 1) / 100,000

= 36.6 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-HWE-GWHT-V04-150601**

[1115] Email message from Maureen Hodgins, Research Manager for Water Research Foundation, to TAC/SAG, August 26, 2014

[1116] ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program citing 2006-2008 American Community Survey data from the US Census Bureau for Illinois cited on p. 17 of the PY2 Evaluation report. 2.75 * 93% evaluation adjustment

[1117] Navigant, ComEd PY3 Multi-Family Home Energy Savings Program Evaluation Report Final, May 16, 2012.

[1118] Bedrooms are suitable proxies for household occupancy, and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.

[1119] US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html

## 5.4.3 Heat Pump Water Heaters

**DESCRIPTION**

The installation of a heat pump domestic hot water heater in place of a standard electric water heater in a home. Savings are presented dependent on the heating system installed in the home due to the impact of the heat pump water heater on the heating loads.

This measure was developed to be applicable to the following program types: TOS, NC, RF.
If applied to other program types, the measure savings should be verified.

**DEFINITION OF EFFICIENT EQUIPMENT**

To qualify for this measure the installed equipment must be a Heat Pump domestic water heater.

**DEFINITION OF BASELINE EQUIPMENT**

The baseline condition is assumed to be a new electric water heater meeting federal minimum efficiency standards[1120]:

For <=55 gallons: 0.96 – (0.0003 * rated volume in gallons)

For >55 gallons: 2.057 – (0.00113 * rated volume in gallons)

**DEEMED LIFETIME OF EFFICIENT EQUIPMENT**

The expected measure life is assumed to be 13 years.[1121]

**DEEMED MEASURE COST**

The incremental capital cost for this measure is $1,000, for a HPWH with an energy factor of 2.0. The full cost, applicable in a retrofit, is $1,575. For a HPWH with an energy factor of 2.35, these costs are $1,134 and $1,703 respectively.[1122]

**LOADSHAPE**

Loadshape R03 - Residential Electric DHW

**COINCIDENCE FACTOR**

The summer Peak Coincidence Factor is assumed to be 12%.[1123]

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[1120] Minimum Federal Standard as of 4/1/2015; http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf

[1121] DOE, 2010 Residential Heating Products Final Rule Technical Support Document, Page 8-52 http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/htgp_finalrule_ch8.pdf

[1122] DOE, 2010 Residential Heating Products Final Rule Technical Support Document, Table 8.2.14 http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/htgp_finalrule_ch8.pdf

[1123] Calculated from Figure 8 "Combined six-unit summer weekday average electrical demand" in FEMP study; Field Testing of Pre-Production Prototype Residential Heat Pump Water Heaters http://www1.eere.energy.gov/femp/pdfs/tir_heatpump.pdf as (average kW usage during peak period * hours in peak period) / [(annual kWh savings / FLH) * hours in peak period] = (0.1 kW * 5 hours) / [(2100 kWh (default assumptions) / 2533 hours) * 5 hours] = 0.12

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$\Delta kWh$ = $(((1/EF_{BASE} - 1/EF_{EFFICIENT}) * GPD * Household * 365.25 * \gamma Water * (T_{OUT} - T_{IN}) * 1.0) / 3412) + kWh_cooling - kWh_heating$

Where:

$EF_{BASE}$ = Energy Factor (efficiency) of standard electric water heater according to federal standards[1124]:

For <=55 gallons: 0.96 – (0.0003 * rated volume in gallons)

For >55 gallons: 2.057 – (0.00113 * rated volume in gallons)

= 0.945 for a 50 gallon tank, the most common size for HPWH

$EF_{EFFICIENT}$ = Energy Factor (efficiency) of Heat Pump water heater

= Actual

GPD = Gallons Per Day of hot water use per person

= 45.5 gallons hot water per day per household/2.59 people per household[1125]

= 17.6

Household = Average number of people per household

| Household Unit Type | Household |
|---|---|
| Single-Family - Deemed | 2.56[1126] |
| Multi-Family - Deemed | 2.1[1127] |
| Custom | Actual Occupancy or Number of Bedrooms[1128] |

365.25 = Days per year

[1124] Minimum Federal Standard as of 1/1/2015; http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf

[1125] Email message from Maureen Hodgins, Research Manager for Water Research Foundation, to TAC/SAG, August 26, 2014

[1126] ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program citing 2006-2008 American Community Survey data from the US Census Bureau for Illinois cited on p. 17 of the PY2 Evaluation report. 2.75 * 93% evaluation adjustment

[1127] Navigant, ComEd PY3 Multi-Family Home Energy Savings Program Evaluation Report Final, May 16, 2012.

[1128] Bedrooms are suitable proxies for household occupancy, and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.

γWater = Specific weight of water

= 8.33 pounds per gallon

$T_{OUT}$ = Tank temperature

= 125°F

$T_{IN}$ = Incoming water temperature from well or municiple system

= 54°F[1129]

1.0 = Heat Capacity of water (1 Btu/lb*°F)

3412 = Conversion from Btu to kWh

kWh_cooling[1130] = Cooling savings from conversion of heat in home to water heat

$$=(((((GPD * Household * 365.25 * \gamma Water * (T_{OUT} - T_{IN}) * 1.0) / 3412) - ((1/ EF_{NEW} * GPD * Household * 365.25 * \gamma Water * (T_{OUT} - T_{IN}) * 1.0) / 3412)) * LF * 27\%) / COP_{COOL}) * LM$$

Where:

LF = Location Factor

= 1.0 for HPWH installation in a conditioned space

= 0.5 for HPWH installation in an unknown location

= 0.0 for installation in an unconditioned space

27% = Portion of reduced waste heat that results in cooling savings[1131]

$COP_{COOL}$ = COP of central air conditioning

= Actual, if unknown, assume 3.08 (10.5 SEER / 3.412)

LM = Latent multiplier to account for latent cooling demand

= 1.33 [1132]

[1129] US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html

[1130] This algorithm calculates the heat removed from the air by subtracting the HPWH electric consumption from the total water heating energy delivered. This is then adjusted to account for location of the HP unit and the coincidence of the waste heat with cooling requirements, the efficiency of the central cooling and latent cooling demands.

[1131] REMRate determined percentage (27%) of lighting savings that result in reduced cooling loads (lighting is used as a proxy for hot water heating since load shapes suggest their seasonal usage patterns are similar).

kWh_heating = Heating cost from conversion of heat in home to water heat (dependent on heating fuel)

For Natural Gas heating, kWh_heating = 0

For electric heating:

$$= ((((GPD * Household * 365.25 * \gamma Water * (T_{OUT} - T_{IN}) * 1.0) / 3412) - ((1/ EF_{NEW} * GPD * Household * 365.25 * \gamma Water * (T_{OUT} - T_{IN}) * 1.0) / 3412)) * LF * 49\%) / COP_{HEAT}$$

Where:

49% = Portion of reduced waste heat that results in increased heating load[1133]

$COP_{HEAT}$ = COP of electric heating system

= actual. If not available use[1134]:

| System Type | Age of Equipment | HSPF Estimate | $COP_{HEAT}$ (COP Estimate) |
|---|---|---|---|
| Heat Pump | Before 2006 | 6.8 | 2.00 |
| | After 2006 - 2014 | 7.7 | 2.26 |
| | 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1.00 |

For example, a 2.0 EF heat pump water heater, in a conditioned space in a single family home with gas space heat and central air conditioning (SEER 10.5) in Belleville:

$$\Delta kWh = [(1 / 0.945 - 1 / 2.0) * 17.6 * 2.56 * 365.25* 8.33 * (125 - 54)] / 3412 + 166.3 - 0$$

$$= 1759 \text{ kWh}$$

[1132] A sensible heat ratio (SHR) of 0.75 corresponds to a latent multiplier of 4/3 or 1.33. SHR of 0.75 for typical split system from page 10 of "Controlling Indoor Humidity Using Variable-Speed Compressors and Blowers" by M. A. Andrade and C. W. Bullard, 1999: www.ideals.illinois.edu/bitstream/handle/2142/11894/TR151.pdf

[1133] REMRate determined percentage (49%) of lighting savings that result in increased heating loads (lighting is used as a proxy for hot water heating since load shapes suggest their seasonal usage patterns are similar).

[1134] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ΔkWh / Hours * CF

Where:

Hours = Full load hours of water heater

= 2533 [1135]

CF = Summer Peak Coincidence Factor for measure

= 0.12 [1136]

For example, a 2.0 COP heat pump water heater, in a conditioned space in a single family home with gas space heat and central air conditioning in Belleville:

kW = 1759 / 2533 * 0.12

= 0.083 kW

**NATURAL GAS SAVINGS**

ΔTherms = - ((((GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 3412) – (((GPD * Household * 365.25 * γWater * ($T_{OUT}$ – $T_{IN}$) * 1.0) / 3412) / $EF_{EFFICIENT}$)) * LF * 49% * 0.03412) / (ηHeat * % Natural Gas)

Where:

ΔTherms = Heating cost from conversion of heat in home to water heat for homes with Natural Gas heat.[1137]

0.03412 = conversion factor (therms per kWh)

ηHeat = Efficiency of heating system

= Actual.[1138] If not available use 70%.[1139]

[1135] Full load hours assumption based on Efficiency Vermont analysis of Itron eShapes.

[1136] Calculated from Figure 8 "Combined six-unit summer weekday average electrical demand" in FEMP study; Field Testing of Pre-Production Prototype Residential Heat Pump Water Heaters http://www1.eere.energy.gov/femp/pdfs/tir_heatpump.pdf as (average kW usage during peak period * hours in peak period) / [(annual kWh savings / FLH) * hours in peak period] = (0.1 kW * 5 hours) / [(2100 kWh / 2533 hours) * 5 hours] = 0.12

[1137] This is the additional energy consumption required to replace the heat removed from the home during the heating season by the heat pump water heater. kWh_heating (electric resistance) is that additional heating energy for a home with electric resistance heat (COP 1.0). This formula converts the additional heating kWh for an electric resistance home to the MMBtu required in a Natural Gas heated home, applying the relative efficiencies.

[1138] Ideally, the System Efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The Distribution Efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute: (http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf ) or by performing duct blaster testing.

% Natural Gas = Factor dependent on heating fuel:

| Heating System | %Natural Gas |
|---|---|
| Electric resistance or heat pump | 0% |
| Natural Gas | 100% |
| Unknown heating fuel[1140] | 87% |

Other factors as defined above

For example, a 2.0 COP heat pump water heater in conditioned space, in a single family home with gas space heat (70% system efficiency):

ΔTherms = -((((17.6 * 2.56 * 365.25* 8.33 * (125 – 54) * 1.0) / 3412) – (17.6 * 2.56 * 365.25* 8.33 * (125 – 54) * 1.0 / 3412 / 2.0)) * 1 * 0.49 * 0.03412) / (0.7 * 1)

= - 34.1 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-HWE-HPWH-V04-150601**

[1139] This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls )) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:
(0.24*0.92) + (0.76*0.8) * (1-0.15) = 0.70

[1140] 2010 American Community Survey.

## 5.4.4 Low Flow Faucet Aerators

This measure relates to the installation of a low flow faucet aerator in a household kitchen or bath faucet fixture.

This measure may be used for units provided through Efficiency Kit's however the in service rate for such measures should be derived through evaluation results specifically for this implementation methodology.

This measure was developed to be applicable to the following program types: TOS, NC, RF, DI, KITS.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a low flow faucet aerator, for bathrooms rated at 1.5 gallons per minute (GPM) or less, or for kitchens rated at 2.2 GPM or less. Savings are calculated on an average savings per faucet fixture basis.

### DEFINITION OF BASELINE EQUIPMENT

The baseline condition is assumed to be a standard bathroom faucet aerator rated at 2.25 GPM or greater, or a standard kitchen faucet aerator rated at 2.75 GPM or greater. Average measured flow rates are used in the algorithm and are lower, reflecting the penetration of previously installed low flow fixtures (and therefore the freerider rate for this measure should be 0), use of the faucet at less than full flow, debris buildup, and lower water system pressure than fixtures are rated at.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 9 years.[1141]

### DEEMED MEASURE COST

The incremental cost for this measure is $8[1142] or program actual.

For faucet aerators provided in Efficiency Kits, the actual program delivery costs should be utilized.

### LOADSHAPE

Loadshape R03 - Residential Electric DHW

### COINCIDENCE FACTOR

The coincidence factor for this measure is assumed to be 2.2%.[1143]

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[1141] Table C-6, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. "http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf"

[1142] Direct-install price per faucet assumes cost of aerator and install time. (2011, Market research average of $3 and assess and install time of $5 (20min @ $15/hr)

[1143] Calculated as follows: Assume 18% aerator use takes place during peak hours (based on: http://www.aquacraft.com/sites/default/files/pub/DeOreo-%282001%29-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf) There are 65 days in the summer peak period, so the percentage of total annual aerator use in peak period is 0.18*65/365 = 3.21%. The number of hours of recovery during peak periods is therefore assumed to be 3.21% *180 = 5.8 hours of recovery during peak period where 180 equals the average annual electric DHW recovery hours for faucet use including SF and MF homes. There are 260 hours in the peak period so the probability you will see savings during the peak period is 5.8/260 = 0.022

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

Note these savings are *per* faucet retrofitted[1144] (unless faucet type is unknown, then it is per household).

ΔkWh = %ElectricDHW * ((GPM_base * L_base - GPM_low * L_low) * Household * 365.25 *DF / FPH) * EPG_electric * ISR

Where:

%ElectricDHW = proportion of water heating supplied by electric resistance heating

| DHW fuel | %ElectricDHW |
|---|---|
| Electric | 100% |
| Natural Gas | 0% |
| Unknown | 16%[1145] |

GPM_base = Average flow rate, in gallons per minute, of the baseline faucet "as-used." This includes the effect of existing low flow fixtures and therefore the freerider rate for this measure should be 0.

= 1.39[1146] or custom based on metering studies[1147] or if measured during DI:

= Measured full throttle flow * 0.83 throttling factor[1148]

GPM_low = Average flow rate, in gallons per minute, of the low-flow faucet aerator "as-used"

= 0.94[1149] or custom based on metering studies[1150] or if measured during DI:

[1144] This algorithm calculates the amount of energy saved per aerator by determining the fraction of water consumption savings for the upgraded fixture.

[1145] Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

[1146] Email message from Maureen Hodgins, Research Manager for Water Research Foundation, to TAC/SAG, August 26, 2014

[1147] Measurement should be based on actual average flow consumed over a period of time rather than a onetime spot measurement for maximum flow. Studies have shown maximum flow rates do not correspond well to average flow rate due to occupant behavior which does not always use maximum flow.

[1148] 2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings. Page 1-265. www.seattle.gov/light/Conserve/Reports/paper_10.pdf

[1149] Average retrofit flow rate for kitchen and bathroom faucet aerators from sources 2, 4, 5, and 7(see source table at end of characterization). This accounts for all throttling and differences from rated flow rates. Assumes all kitchen aerators at 2.2 gpm or less and all bathroom aerators at 1.5 gpm or less. The most comprehensive available studies did not disaggregate kitchen use from bathroom use, but instead looked at total flow and length of use for all faucets. This makes it difficult to reliably separate kitchen water use from bathroom water use. It is possible that programs installing low flow aerators lower than the 2.2 gpm for kitchens and 1.5 gpm for bathrooms will see a lower overall average retrofit flow rate.

[1150] Measurement should be based on actual average flow consumed over a period of time rather than a onetime spot measurement for maximum flow. Studies have shown maximum flow rates do not correspond well to average flow rate due to

= Rated full throttle flow * 0.95 throttling factor[1151]

L_base = Average baseline daily length faucet use per capita for faucet of interest in minutes

= if available custom based on metering studies, if not use:

| Faucet Type | L_base (min/person/day) |
|---|---|
| Kitchen | 4.5[1152] |
| Bathroom | 1.6[1153] |
| If location unknown (total for household): Single-Family | 9.0[1154] |
| If location unknown (total for household): Multi-Family | 6.9[1155] |

L_low = Average retrofit daily length faucet use per capita for faucet of interest in minutes

= if available custom based on metering studies, if not use:

| Faucet Type | L_low (min/person/day) |
|---|---|
| Kitchen | 4.5[1156] |
| Bathroom | 1.6[1157] |
| If location unknown (total for household): Single-Family | 9.0[1158] |
| If location unknown (total for household): Multi-Family | 6.9[1159] |

Household = Average number of people per household

occupant behavior which does not always use maximum flow.

[1151] 2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings. Page 1-265. www.seattle.gov/light/Conserve/Reports/paper_10.pdf

[1152] Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group. This study of 135 single and multi-family homes in Michigan metered energy parameters for efficient showerhead and faucet aerators.

[1153] Ibid.

[1154] One kitchen faucet plus 2.83 bathroom faucets. Based on findings from a 2009 ComEd residential survey of 140 sites, provided by Cadmus.

[1155] One kitchen faucet plus 1.5 bathroom faucets. Based on findings from a 2009 ComEd residential survey of 140 sites, provided by Cadmus.

[1156] Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group.

[1157] Ibid.

[1158] One kitchen faucet plus 2.83 bathroom faucets. Based on findings from a 2009 ComEd residential survey of 140 sites, provided by Cadmus.

[1159] One kitchen faucet plus 1.5 bathroom faucets. Based on findings from a 2009 ComEd residential survey of 140 sites, provided by Cadmus.

| Household Unit Type | Household |
|---|---|
| Single-Family - Deemed | 2.56[1160] |
| Multi-Family - Deemed | 2.1[1161] |
| Custom | Actual Occupancy or Number of Bedrooms[1162] |

365.25 = Days in a year, on average.

DF = Drain Factor

| Faucet Type | Drain Factor[1163] |
|---|---|
| Kitchen | 75% |
| Bath | 90% |
| Unknown | 79.5% |

FPH = Faucets Per Household

| Faucet Type | FPH |
|---|---|
| Kitchen Faucets Per Home (KFPH) | 1 |
| Bathroom Faucets Per Home (BFPH): Single-Family | 2.83[1164] |
| Bathroom Faucets Per Home (BFPH): Multi-Family | 1.5[1165] |
| If location unknown (total for household): Single-Family | 3.83 |
| If location unknown (total for household): Multi-Family | 2.5 |

[1160] ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program citing 2006-2008 American Community Survey data from the US Census Bureau for Illinois cited on p. 17 of the PY2 Evaluation report. 2.75 * 93% evaluation adjustment

[1161] Navigant, ComEd PY3 Multi-Family Home Energy Savings Program Evaluation Report Final, May 16, 2012.

[1162] Bedrooms are suitable proxies for household occupancy, and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.

[1163] Because faucet usages are at times dictated by volume, only usage of the sort that would go straight down the drain will provide savings. VEIC is unaware of any metering study that has determined this specific factor and so through consensus with the Illinois Technical Advisory Group have deemed these values to be 75% for the kitchen and 90% for the bathroom. If the aerator location is unknown an average of 79.5% should be used which is based on the assumption that 70% of household water runs through the kitchen faucet and 30% through the bathroom (0.7*0.75)+(0.3*0.9)=0.795.

[1164] Based on findings from a 2009 ComEd residential survey of 140 sites, provided by Cadmus.

[1165] Ibid.

EPG_electric = Energy per gallon of water used by faucet supplied by electric water heater

= (8.33 * 1.0 * (WaterTemp - SupplyTemp)) / (RE_electric * 3412)

= (8.33 * 1.0 * (86 – 54.1)) / (0.98 * 3412)

= 0.0795 kWh/gal (Bath), 0.0969 kWh/gal (Kitchen), 0.0919 kWh/gal (Unknown)

8.33 = Specific weight of water (lbs/gallon)

1.0 = Heat Capacity of water (btu/lb-°F)

WaterTemp = Assumed temperature of mixed water

= 86F for Bath, 93F for Kitchen 91F for Unknown[1166]

SupplyTemp = Assumed temperature of water entering house

= 54.1F [1167]

RE_electric = Recovery efficiency of electric water heater

= 98% [1168]

3412 = Converts Btu to kWh (btu/kWh)

ISR = In service rate of faucet aerators dependant on install method as listed in table below

| Selection | ISR |
|---|---|
| Direct Install - Single Family | 0.95[1169] |
| Direct Install – Multi Family Kitchen | 0.91[1170] |
| Direct Install – Multi Family Bathroom | 0.95[1171] |
| Efficiency Kits | To be determined through evaluation |

[1166] Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group. If the aerator location is unknown an average of 91% should be used which is based on the assumption that 70% of household water runs through the kitchen faucet and 30% through the bathroom (0.7*93)+(0.3*86)=0.91.

[1167] US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html.

[1168] Electric water heaters have recovery efficiency of 98%: http://www.ahridirectory.org/ahridirectory/pages/home.aspx

[1169] ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program Table 3-8

[1170] Navigant, ComEd-Nicor Gas EPY4/GPY1 Multi-Family Home Energy Savings Program Evaluation Report DRAFT 2013-01-28

[1171] Ibid.

For example, a direct installed kitchen low flow faucet aerator in a single-family electric DHW home:

ΔkWh = 1.0 * (((1.39 * 4.5 – 0.94 * 4.5) * 2.56 * 365.25 *0.75) / 1) * 0.0969 * 0.95

= 131 kWh

For example, a direct installed bath low flow faucet aerator in a multi-family electric DHW home:

ΔkWh = 1.0 * (((1.39 * 1.6 – 0.94 * 1.6) * 2.1 * 365.25 * 0.90) /1.5) * 0.0795 * 0.95

= 25.0 kWh

For example, a direct installed low flow faucet aerator in unknown faucet in a single-family electric DHW home:

ΔkWh = 1.0 * (((1.39 * 9.0 – 0.94 * 9.0) * 2.56 * 365.25 * 0.795) /3.83) * 0.0919 * 0.95

= 68.6 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ΔkWh / Hours * CF

Where:

ΔkWh = calculated value above

Hours = Annual electric DHW recovery hours for faucet use per faucet

= ((GPM_base * L_base) * Household/FPH * 365.25 * DF ) * 0.545[1172] / GPH

| Building Type | Faucet location | Calculation | Hours per faucet |
|---|---|---|---|
| Single Family | Kitchen | ((1.39 * 4.5) * 2.56/1 * 365.25 * 0.75) * 0.545 / 25.5 | 94 |
| | Bathroom | ((1. 39 * 1.6) * 2.56/2.83 * 365.25 * 0.9) * 0.545 / 25.5 | 14 |
| | Unknown | ((1. 39 * 9.0) * 2.56/3.83 * 365.25 * 0.795) * 0.545 / 25.5 | 52 |
| Multi Family | Kitchen | ((1. 39 * 4.5) * 2.1/1 * 365.25 * 0.75) * 0.545 / 25.5 | 77 |
| | Bathroom | ((1. 39 * 1.6) * 2.1/1.5 * 365.25 * 0.9) * 0.545 / 25.5 | 22 |
| | Unknown | ((1. 39 * 6.9) * 2.1/2.5 * 365.25 * 0.795) * 0.545 / 25.5 | 50 |

GPH = Gallons per hour recovery of electric water heater calculated for 70.9F temp rise (125-54.1), 98% recovery efficiency, and typical 4.5kW electric resistance storage tank.

= 25.5

CF = Coincidence Factor for electric load reduction

[1172] 54.5% is the proportion of hot 120F water mixed with 54.1F supply water to give 90F mixed faucet water.

= 0.022[1173]

For example, a direct installed kitchen low flow faucet aerator in a single family electric DHW home:

ΔkW = 131/94 * 0.022

= 0.0306 kW

**NATURAL GAS SAVINGS**

ΔTherms = %FossilDHW * ((GPM_base * L_base - GPM_low * L_low) * Household * 365.25 *DF / FPH) * EPG_gas * ISR

Where:

%FossilDHW = proportion of water heating supplied by Natural Gas heating

| DHW fuel | %Fossil_DHW |
|---|---|
| Electric | 0% |
| Natural Gas | 100% |
| Unknown | 84%[1174] |

EPG_gas = Energy per gallon of Hot water supplied by gas

= (8.33 * 1.0 * (WaterTemp - SupplyTemp)) / (RE_gas * 100,000)

= 0.00341 Therm/gal for SF homes (Bath), 0.00415 Therm/gal for SF homes (Kitchen), 0.00394 Therm/gal for SF homes (Unknown)

= 0.00397 Therm/gal for MF homes (Bath), 0.00484 Therm/gal for MF homes (Kitchen), 0.00459 Therm/gal for MF homes (Unknown)

RE_gas = Recovery efficiency of gas water heater

= 78% For SF homes[1175]

= 67% For MF homes[1176]

[1173] Calculated as follows: Assume 18% aerator use takes place during peak hours (based on: http://www.aquacraft.com/sites/default/files/pub/DeOreo-%282001%29-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf) There are 65 days in the summer peak period, so the percentage of total annual aerator use in peak period is 0.18*65/365 = 3.21%. The number of hours of recovery during peak periods is therefore assumed to be 3.21% *180 = 5.8 hours of recovery during peak period where 180 equals the average annual electric DHW recovery hours for faucet use including SF and MF homes. There are 260 hours in the peak period so the probability you will see savings during the peak period is 5.8/260 = 0.022

[1174] Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

[1175] DOE Final Rule discusses Recovery Efficiency with an average around 0.76 for Gas Fired Storage Water heaters and 0.78 for standard efficiency gas fired tankless water heaters up to 0.95 for the highest efficiency gas fired condensing tankless water heaters. These numbers represent the range of new units however, not the range of existing units in stock. Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 78%.

100,000 = Converts Btus to Therms (btu/Therm)

Other variables as defined above.

For example, a direct-installed kitchen low flow faucet aerator in a fuel DHW single-family home:

ΔTherms = 1.0 * (((1.39 * 4.5 – 0.94 * 4.5) * 2.56 * 365.25 *0.75) / 1) * 0.00415 * 0.95

= 5.60 Therms

For example, a direct installed bath low flow faucet aerator in a fuel DHW multi-family home:

ΔTherms = 1.0 * (((1.39 * 1.6 – 0.94 * 1.6) * 2.1 * 365.25 * 0.90) /1.5) * 0.003974 * 0.95

= 1.25 Therms

For example, a direct installed low flow faucet aerator in unknown faucet in a fuel DHW single-family home:

ΔTherms = 1.0 * (((1.39 * 9.0 – 0.94 * 9.0) * 2.56 * 365.25 * 0.795) /3.83) * 0.00394 * 0.95

= 2.94 Therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

Δgallons = ((GPM_base * L_base - GPM_low * L_low) * Household * 365.25 *DF / FPH) * ISR

Variables as defined above

For example, a direct-installed kitchen low flow aerator in a single family home

Δgallons = (((1.39 * 4.5 – 0.94 * 4.5) * 2.56 * 365.25 *0.75) / 1) * 0.95

= 1350 gallons

For example, a direct installed bath low flow faucet aerator in a multi-family home:

Δgallons = (((1.39 * 1.6 – 0.94 * 1.6) * 2.1 * 365.25 * 0.90) /1.5) * 0.95

= 314 gallons

For example, a direct installed low flow faucet aerator in unknown faucet in a single-family home:

Δgallons = (((1.39 * 9.0 – 0.94 * 9.0) * 2.56 * 365.25 * 0.795) /3.83) * 0.95

= 747 gallons

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

[1176] Water heating in multi-family buildings is often provided by a larger central boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of 0.59 and the 0.75 for single family homes. An average efficiency of 0.67 is used for this analysis as a default for multi-family buildings.

**SOURCES**

| Source ID | Reference |
|---|---|
| 1 | 2011, DeOreo, William. California Single Family Water Use Efficiency Study. April 20, 2011. |
| 2 | 2000, Mayer, Peter, William DeOreo, and David Lewis. Seattle Home Water Conservation Study. December 2000. |
| 3 | 1999, Mayer, Peter, William DeOreo. Residential End Uses of Water. Published by AWWA Research Foundation and American Water Works Association. 1999. |
| 4 | 2003, Mayer, Peter, William DeOreo. Residential Indoor Water Conservation Study. Aquacraft, Inc. Water Engineering and Management. Prepared for East Bay Municipal Utility District and the US EPA. July 2003. |
| 5 | 2011, DeOreo, William. Analysis of Water Use in New Single Family Homes. By Aquacraft. For Salt Lake City Corporation and US EPA. July 20, 2011. |
| 6 | 2011, Aquacraft. Albuquerque Single Family Water Use Efficiency and Retrofit Study. For Albuquerque Bernalillo County Water Utility Authority. December 1, 2011. |
| 7 | 2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings. |

**MEASURE CODE: RS-HWE-LFFA-V04-150601**

## 5.4.5 Low Flow Showerheads

### DESCRIPTION

This measure relates to the installation of a low flow showerhead in a single or multi-family household.

This measure may be used for units provided through Efficiency Kit's however the in service rate for such measures should be derived through evaluation results specifically for this implementation methodology.

This measure was developed to be applicable to the following program types: TOS, RF, NC, DI, KITS.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a low flow showerhead rated at 2.0 gallons per minute (GPM) or less. Savings are calculated on a per showerhead fixture basis.

### DEFINITION OF BASELINE EQUIPMENT

For Direct-install programs, the baseline condition is assumed to be a standard showerhead rated at 2.5 GPM or greater.

For retrofit and time-of-sale programs, the baseline condition is assumed to be a representative average of existing showerhead flow rates of participating customers including a range of low flow showerheads, standard-flow showerheads, and high-flow showerheads.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years.[1177]

### DEEMED MEASURE COST

The incremental cost for this measure is $12[1178] or program actual.

For low flow showerheads provided in Efficiency Kits, the actual program delivery costs should be utilized.

### LOADSHAPE

Loadshape R03 - Residential Electric DHW

### COINCIDENCE FACTOR

The coincidence factor for this measure is assumed to be 2.78%.[1179]

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[1177] Table C-6, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. Evaluations indicate that consumer dissatisfaction may lead to reductions in persistence, particularly in Multi-Family , "http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf"

[1178] Direct-install price per showerhead assumes cost of showerhead (Market research average of $7 and assess and install time of $5 (20min @ $15/hr)

[1179] Calculated as follows: Assume 11% showers take place during peak hours (based on: http://www.aquacraft.com/sites/default/files/pub/DeOreo-%282001%29-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf). There are 65 days in the summer peak period, so the percentage of total annual aerator use in peak period is 0.11*65/365 = 1.96%. The number of hours of recovery during peak periods is therefore assumed to be 1.96% * 369 = 7.23 hours of recovery during peak period, where 369 equals the average annual electric DHW recovery hours for

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

Note these savings are per showerhead fixture

ΔkWh = %ElectricDHW * ((GPM_base * L_base - GPM_low * L_low) * Household * SPCD * 365.25 / SPH) * EPG_electric * ISR

Where:

%ElectricDHW = proportion of water heating supplied by electric resistance heating

| DHW fuel | %ElectricDHW |
|---|---|
| Electric | 100% |
| Natural Gas | 0% |
| Unknown | 16%[1180] |

GPM_base = Flow rate of the baseline showerhead

| Program | GPM_base |
|---|---|
| Direct-install | 2.67[1181] |
| Retrofit, Efficiency Kits, NC or TOS | 2.35[1182] |

GPM_low = As-used flow rate of the low-flow showerhead, which may, as a result of measurements of program evaulations deviate from rated flows, see table below:

| Rated Flow |
|---|
| 2.0 GPM |
| 1.75 GPM |
| 1.5 GPM |

showerhead use including SF and MF homes with Direct Install and Retrofit/TOS measures. There are 260 hours in the peak period so the probability you will see savings during the peak period is 7.23/260 = 0.0278

[1180] Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

[1181] Based on measured data from Ameren IL EM&V of Direct-Install program. Program targets showers that are rated 2.5 GPM or above.

[1182] Representative value from sources 1, 2, 4, 5, 6 and 7 (See Source Table at end of measure section) adjusted slightly upward to account for program participation which is expected to target customers with existing higher flow devices rather than those with existing low flow devices.

| Rated Flow |
|---|
| Custom or Actual[1183] |

L_base = Shower length in minutes with baseline showerhead

= 7.8 min[1184]

L_low = Shower length in minutes with low-flow showerhead

= 7.8 min[1185]

Household = Average number of people per household

| Household Unit Type[1186] | Household |
|---|---|
| Single-Family - Deemed | 2.56[1187] |
| Multi-Family - Deemed | 2.1[1188] |
| Custom | Actual Occupancy or Number of Bedrooms[1189] |

SPCD = Showers Per Capita Per Day

= 0.6[1190]

365.25 = Days per year, on average.

SPH = Showerheads Per Household so that per-showerhead savings fractions can be determined

| Household Type | SPH |
|---|---|
| Single-Family | 1.79[1191] |

[1183] Note that actual values may be either a) program-specific minimum flow rate, or b) program-specific evaluation-based value of actual effective flow-rate due to increased duration or temperatures. The latter increases in likelihood as the rated flow drops and may become significant at or below rated flows of 1.5 GPM. The impact can be viewed as the inverse of the throttling described in the footnote for baseline flowrate.

[1184] Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group. This study of 135 single and multi-family homes in Michigan metered energy parameters for efficient showerhead and faucet aerators.

[1185] Ibid.

[1186] If household type is unknown, as may be the case for time of sale measures, then single family deemed value shall be used.

[1187] ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program citing 2006-2008 American Community Survey data from the US Census Bureau for Illinois cited on p. 17 of the PY2 Evaluation report. 2.75 * 93% evaluation adjustment

[1188] ComEd PY3 Multi-Family Evaluation Report REVISED DRAFT v5 2011-12-08.docx

[1189] Bedrooms are suitable proxies for household occupancy, and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.

[1190] Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group.

[1191] Based on findings from a 2009 ComEd residential survey of 140 sites, provided by Cadmus.

| Household Type | SPH |
|---|---|
| Multi-Family | 1.3[1192] |
| Custom | Actual |

EPG_electric = Energy per gallon of hot water supplied by electric

= (8.33 * 1.0 * (ShowerTemp - SupplyTemp)) / (RE_electric * 3412)

= (8.33 * 1.0 * (101 – 54.1)) / (0.98 * 3412)

= 0.117 kWh/gal

8.33 = Specific weight of water (lbs/gallon)

1.0 = Heat Capacity of water (btu/lb-°)

ShowerTemp = Assumed temperature of water

= 101F [1193]

SupplyTemp = Assumed temperature of water entering house

= 54.1F [1194]

RE_electric = Recovery efficiency of electric water heater

= 98% [1195]

3412 = Converts Btu to kWh (btu/kWh)

ISR = In service rate of showerhead

= Dependant on program delivery method as listed in table below

| Selection | ISR |
|---|---|
| Direct Install - Single Family | 0.98[1196] |
| Direct Install – Multi Family | 0.95[1197] |
| Efficiency Kits | To be determined through evaluation |

[1192] Ibid.
[1193] Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group.
[1194] US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html.
[1195] Electric water heaters have recovery efficiency of 98%: http://www.ahridirectory.org/ahridirectory/pages/home.aspx
[1196] Deemed values are from ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program Table 3-8. Alternative ISRs may be developed for program delivery methods based on evaluation results.
[1197] Navigant, ComEd-Nicor Gas EPY4/GPY1 Multi-Family Home Energy Savings Program Evaluation Report FINAL 2013-06-05

For example, a direct-installed 1.5 GPM low flow showerhead in a single family home with electric DHW where the number of showers is not known:

ΔkWh = 1.0 * ((2.67 * 7.8 – 1.5 * 7.8) * 2.56 * 0.6 * 365.25 / 1.79) * 0.117 * 0.98

= 328 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ΔkWh/Hours * CF

Where:

ΔkWh = calculated value above

Hours = Annual electric DHW recovery hours for showerhead use

= ((GPM_base * L_base) * Household * SPCD * 365.25 ) * 0.712[1198] / GPH

= 302 for SF Direct Install; 248 for MF Direct Install

= 266 for SF Retrofit, Efficiency Kits, NC and TOS; 218 for MF Retrofit, Efficiency Kits, NC and TOS

GPH = Gallons per hour recovery of electric water heater calculated for 65.9F temp rise (120-54.1), 98% recovery efficiency, and typical 4.5kW electric resistance storage tank.

= 27.51

CF = Coincidence Factor for electric load reduction

= 0.0278[1199]

For example, a direct installed 1.5 GPM low flow showerhead in a single family home with electric DHW where the number of showers is not known:

ΔkW = 328/302 * 0.0278

= 0.0302 kW

**NATURAL GAS SAVINGS**

ΔTherms = %FossilDHW * ((GPM_base * L_base - GPM_low * L_low) * Household * SPCD * 365.25 / SPH) * EPG_gas * ISR

Where:

%FossilDHW = proportion of water heating supplied by Natural Gas heating

[1198] 71.2% is the proportion of hot 120F water mixed with 54.1F supply water to give 101F shower water.

[1199] Calculated as follows: Assume 11% showers take place during peak hours (based on: http://www.aquacraft.com/sites/default/files/pub/DeOreo-%282001%29-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf). There are 65 days in the summer peak period, so the percentage of total annual aerator use in peak period is 0.11*65/365 = 1.96%. The number of hours of recovery during peak periods is therefore assumed to be 1.96% * 369 = 7.23 hours of recovery during peak period where 369 equals the average annual electric DHW recovery hours for showerhead use including SF and MF homes with Direct Install and Retrofit/TOS measures. There are 260 hours in the peak period so the probability you will see savings during the peak period is 7.23/260 = 0.0278

| DHW fuel | %Fossil_DHW |
|---|---|
| Electric | 0% |
| Natural Gas | 100% |
| Unknown | 84%[1200] |

EPG_gas = Energy per gallon of Hot water supplied by gas

= (8.33 * 1.0 * (ShowerTemp - SupplyTemp)) / (RE_gas * 100,000)

= 0.00501 Therm/gal for SF homes

= 0.00583 Therm/gal for MF homes

RE_gas = Recovery efficiency of gas water heater

= 78% For SF homes[1201]

= 67% For MF homes[1202]

100,000 = Converts Btus to Therms (btu/Therm)

Other variables as defined above.

For example, a direct installed 1.5 GPM low flow showerhead in a gas fired DHW single family home where the number of showers is not known:

ΔTherms = 1.0 * ((2.67 * 7.8 – 1.5 * 7.8) * 2.56 * 0.6 * 365.25 / 1.79) * 0.00501 * 0.98

= 14.0 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

Δgallons = ((GPM_base * L_base - GPM_low * L_low) * Household * SPCD * 365.25 / SPH) * ISR

Variables as defined above

[1200] Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

[1201] DOE Final Rule discusses Recovery Efficiency with an average around 0.76 for Gas Fired Storage Water heaters and 0.78 for standard efficiency gas fired tankless water heaters up to 0.95 for the highest efficiency gas fired condensing tankless water heaters. These numbers represent the range of new units however, not the range of existing units in stock. Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 78%.

[1202] Water heating in multi-family buildings is often provided by a larger central boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of 0.59 and the 0.75 for single family homes. An average efficiency of 0.67 is used for this analysis as a default for multi-family buildings.

For example, a direct installed 1.5 GPM low flow showerhead in a single family home where the number of showers is not known:

Δgallons = ((2.67 * 7.8 – 1.5 * 7.8) * 2.56 * 0.6 * 365.25 / 1.79) * 0.98

= 2803 gallons

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**SOURCES**

| Source ID | Reference |
|---|---|
| 1 | 2011, DeOreo, William. California Single Family Water Use Efficiency Study. April 20, 2011. |
| 2 | 2000, Mayer, Peter, William DeOreo, and David Lewis. Seattle Home Water Conservation Study. December 2000. |
| 3 | 1999, Mayer, Peter, William DeOreo. Residential End Uses of Water. Published by AWWA Research Foundation and American Water Works Association. 1999. |
| 4 | 2003, Mayer, Peter, William DeOreo. Residential Indoor Water Conservation Study. Aquacraft, Inc. Water Engineering and Management. Prepared for East Bay Municipal Utility District and the US EPA. July 2003. |
| 5 | 2011, DeOreo, William. Analysis of Water Use in New Single Family Homes. By Aquacraft. For Salt Lake City Corporation and US EPA. July 20, 2011. |
| 6 | 2011, Aquacraft. Albuquerque Single Family Water Use Efficiency and Retrofit Study. For Albuquerque Bernalillo County Water Utility Authority. December 1, 2011. |
| 7 | 2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings. |

**MEASURE CODE: RS-HWE-LFSH-V03-140601**

## 5.4.6 Water Heater Temperature Setback

### DESCRIPTION

This measure was developed to be applicable to the following program types: NC, RF, DI, KITS.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

High efficiency is a hot water tank with the thermostat reduced to no lower than 120 degrees.

### DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a hot water tank with a thermostat setting that is higher than 120 degrees, typically systems with settings of 130 degrees or higher. Note if there are more than one DHW tanks in the home at or higher than 130 degrees and they are all turned down, then the savings per tank can be multiplied by the number of tanks.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of the measure is 2 years.

### DEEMED MEASURE COST

The incremental cost of a setback is assumed to be $5 for contractor time, or no cost if the measure is self-installed.

### LOADSHAPE

Loadshape R03 - Residential Electric DHW

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 1.

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

For homes with electric DHW tanks:

$$\Delta kWh^{1203} = (UA * (Tpre - Tpost) * Hours) / (3412 * RE_electric)$$

Where:

U = Overall heat transfer coefficient of tank (Btu/Hr-°F-ft$^2$).

= Actual if known. If unknown assume R-12, U = 0.083

A = Surface area of storage tank (square feet)

= Actual if known. If unknown use table below based on capacity of tank. If capacity unknown assume 50 gal tank; A = 24.99ft$^2$

| Capacity (gal) | A (ft$^2$)[1204] |
|---|---|
| 30 | 19.16 |
| 40 | 23.18 |
| 50 | 24.99 |
| 80 | 31.84 |

Tpre = Actual hot water setpoint prior to adjustment

Tpost = Actual new hot water setpoint, which may not be lower than 120 degrees

| Default Hot Water Temperature Inputs | |
|---|---|
| Tpre | 135 |
| Tpost | 120 |

[1203] Note this algorithm provides savings only from reduction in standby losses. The TAC considered avoided energy from not heating the water to the higher temperature but determined that dishwashers are likely to boost the temperature within the unit (roughly canceling out any savings), faucet and shower use is likely to be at the same temperature so there would need to be more lower temperature hot water being used (cancelling any savings) and clothes washers will only see savings if the water from the tank is taken without any temperature control. It was felt the potential impact was too small to be characterized.

[1204] Assumptions from PA TRM. Area values were calculated from average dimensions of several commercially available units, with radius values measured to the center of the insulation.

Hours = Number of hours in a year (since savings are assumed to be constant over year).

= 8766

3412 = Conversion from Btu to kWh

RE_electric = Recovery efficiency of electric hot water heater

= 0.98 [1205]

A deemed savings assumption, where site specific assumptions are not available would be as follows:

ΔkWh = (UA * (Tpre – Tpost) * Hours) / (3412 * RE_electric)

= (((0.083 * 24.99) * (135 – 120) * 8766) / (3412 * 0.98)

= 81.6 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ΔkWh / Hours * CF

Where:

Hours = 8766

CF = Summer Peak Coincidence Factor for measure

= 1

A deemed savings assumption, where site specific assumptions are not available would be as follows:

ΔkW = (81.6/ 8766) * 1

ΔkW default = 0.00931 kW

**NATURAL GAS SAVINGS**

For homes with gas water heaters:

[1205] Electric water heaters have recovery efficiency of 98%: http://www.ahridirectory.org/ahridirectory/pages/home.aspx

ΔTherms = (UA * (Tpre – Tpost) * Hours) / (100,000 * RE_gas)

Where

100,000 = Converts Btus to Therms (btu/Therm)

RE_gas = Recovery efficiency of gas water heater

= 78% For SF homes[1206]

= 67% For MF homes[1207]

A deemed savings assumption, where site specific assumptions are not available would be as follows:

For Single Family homes:

ΔTherms = (UA * (Tpre – Tpost) * Hours) / (RE_gas)

= (((0.083 * 24.99) * (135 – 120) * 8766) / (100,000 * 0.78)

= 3.5 Therms

For Multi Family homes:

ΔTherms = (UA * (Tpre – Tpost) * Hours) / (RE_gas)

= (((0.083 * 24.99) * (135 – 120) * 8766) / (100,000 * 0.67)

= 4.1 Therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-HWE-TMPS-V04-150601**

[1206] DOE Final Rule discusses Recovery Efficiency with an average around 0.76 for Gas Fired Storage Water heaters and 0.78 for standard efficiency gas fired tankless water heaters up to 0.95 for the highest efficiency gas fired condensing tankless water heaters. These numbers represent the range of new units however, not the range of existing units in stock. Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 78%.

[1207] Water heating in multi-family buildings is often provided by a larger central boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of 0.59 and the 0.75 for single family homes. An average efficiency of 0.67 is used for this analysis as a default for multi-family buildings.

## 5.4.7 Water Heater Wrap

### DESCRIPTION

This measure relates to a Tank Wrap or insulation "blanket" that is wrapped around the outside of a hot water tank to reduce stand-by losses. This measure applies only for homes that have an electric water heater that is not already well insulated. Generally this can be determined based upon the appearance of the tank.[1208]

This measure was developed to be applicable to the following program types: RF, DI.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The measure is a properly installed, R-8 or greater insulating tank wrap to reduce standby energy losses from the tank to the surrounding ambient area.

### DEFINITION OF BASELINE EQUIPMENT

The baseline is a standard electric domestic hot water tank without an additional tank wrap. Gas storage water heaters are excluded due to the limitations of retrofit wrapping and the associated impacts on reduced savings and safety.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 5 years[1209].

### DEEMED MEASURE COST

The incremental cost for this measure will be the actual material cost of procuring and labor cost of installing the tank wrap.

### LOADSHAPE

Loadshape R03 - Residential Electric DHW

### COINCIDENCE FACTOR

This measure assumes a flat loadshape and as such the coincidence factor is 1.

[1208] Visually determine whether it is insulated by foam (newer, rigid, and more effective) or fiberglass (older, gives to gently pressure, and not as effective)

[1209] This estimate assumes the tank wrap is installed on an existing unit with 5 years remaining life.

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

For electric DHW systems:

$$\Delta kWh = ((A_{base} / Rbase - A_{insul} / R_{insul}) * \Delta T * Hours) / (3412 * \eta DHW)$$

Where:

| | |
|---|---|
| $R_{base}$ | = Overall thermal resistance coefficient prior to adding tank wrap (Hr-°F-ft$^2$/BTU). |
| $R_{insul}$ | = Overall thermal resistance coefficient after addition of tank wrap (Hr-°F-ft$^2$/BTU). |
| $A_{base}$ | = Surface area of storage tank prior to adding tank wrap (square feet)[1210] |
| $A_{insul}$ | = Surface area of storage tank after addition of tank wrap (square feet)[1211] |
| ΔT | = Average temperature difference between tank water and outside air temperature (°F) |
| | = 60°F [1212] |
| Hours | = Number of hours in a year (since savings are assumed to be constant over year). |
| | = 8766 |
| 3412 | = Conversion from Btu to kWh |
| ηDHW | = Recovery efficiency of electric hot water heater |
| | = 0.98 [1213] |

[1210] Area includes tank sides and top to account for typical wrap coverage.
[1211] Ibid.
[1212] Assumes 125°F water leaving the hot water tank and average temperature of basement of 65°F.
[1213] Electric water heaters have recovery efficiency of 98%: http://www.ahridirectory.org/ahridirectory/pages/home.aspx

The following table has default savings for various tank capacity and pre and post R-VALUES.

| Capacity (gal) | Rbase | Rinsul | Abase (ft2)[1214] | Ainsul (ft2)[1215] | ΔkWh | ΔkW |
|---|---|---|---|---|---|---|
| 30 | 8 | 16 | 19.16 | 20.94 | 171 | 0.0195 |
| 30 | 10 | 18 | 19.16 | 20.94 | 118 | 0.0135 |
| 30 | 12 | 20 | 19.16 | 20.94 | 86 | 0.0099 |
| 30 | 8 | 18 | 19.16 | 20.94 | 194 | 0.0221 |
| 30 | 10 | 20 | 19.16 | 20.94 | 137 | 0.0156 |
| 30 | 12 | 22 | 19.16 | 20.94 | 101 | 0.0116 |
| 40 | 8 | 16 | 23.18 | 25.31 | 207 | 0.0236 |
| 40 | 10 | 18 | 23.18 | 25.31 | 143 | 0.0164 |
| 40 | 12 | 20 | 23.18 | 25.31 | 105 | 0.0120 |
| 40 | 8 | 18 | 23.18 | 25.31 | 234 | 0.0268 |
| 40 | 10 | 20 | 23.18 | 25.31 | 165 | 0.0189 |
| 40 | 12 | 22 | 23.18 | 25.31 | 123 | 0.0140 |
| 50 | 8 | 16 | 24.99 | 27.06 | 225 | 0.0257 |
| 50 | 10 | 18 | 24.99 | 27.06 | 157 | 0.0179 |
| 50 | 12 | 20 | 24.99 | 27.06 | 115 | 0.0131 |
| 50 | 8 | 18 | 24.99 | 27.06 | 255 | 0.0291 |
| 50 | 10 | 20 | 24.99 | 27.06 | 180 | 0.0206 |
| 50 | 12 | 22 | 24.99 | 27.06 | 134 | 0.0153 |
| 80 | 8 | 16 | 31.84 | 34.14 | 290 | 0.0331 |
| 80 | 10 | 18 | 31.84 | 34.14 | 202 | 0.0231 |
| 80 | 12 | 20 | 31.84 | 34.14 | 149 | 0.0170 |
| 80 | 8 | 18 | 31.84 | 34.14 | 328 | 0.0374 |
| 80 | 10 | 20 | 31.84 | 34.14 | 232 | 0.0265 |
| 80 | 12 | 22 | 31.84 | 34.14 | 173 | 0.0198 |

[1214] Assumptions from PA TRM. Area values were calculated from average dimensions of several commercially available units, with radius values measured to the center of the insulation. Area includes tank sides and top to account for typical wrap coverage.

[1215] Assumptions from PA TRM. $A_{insul}$ was calculated by assuming that the water heater wrap is a 2" thick fiberglass material.

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh / 8766 * CF$$

Where:

| | |
|---|---|
| ΔkWh | = kWh savings from tank wrap installation |
| 8766 | = Number of hours in a year (since savings are assumed to be constant over year). |
| CF | = Summer Coincidence Factor for this measure |
| | = 1.0 |

The table above has default kW savings for various tank capacity and pre and post R-values.

### NATURAL GAS SAVINGS

N/A

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: RS-HWE-WRAP-V02-150601

## 5.4.8 Thermostatic Restrictor Shower Valve

### DESCRIPTION

The measure is the installation of a thermostatic restrictor shower valve in a single or multi-family household. This is a valve attached to a residential showerhead which restricts hot water flow through the showerhead once the water reaches a set point (generally 95F or lower).

This measure was developed to be applicable to the following program types: RF, NC, DI. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a thermostatic restrictor shower valve installed on a residential showerhead.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is the residential showerhead without the restrictor valve installed.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years. [1216]

### DEEMED MEASURE COST

The incremental cost of the measure should be the actual program cost or $30[1217] if not available.

### LOADSHAPE

Loadshape R03 - Residential Electric DHW

### COINCIDENCE FACTOR

The coincidence factor for this measure is assumed to be 0.22%.[1218]

[1216] Assumptions based on NY TRM, Pacific Gas and Electric Company Work Paper PGECODHW113, and measure life of low-flow showerhead

[1217] Based on actual cost of the SS-1002CP-SB Ladybug Water-Saving Shower-Head adapter from Evolve showerheads

[1218] Calculated as follows: Assume 11% showers take place during peak hours (based on: http://www.aquacraft.com/sites/default/files/pub/DeOreo-%282001%29-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf). There are 65 days in the summer peak period, so the percentage of total annual use in peak period is 0.11*65/365 = 1.96%. The number of hours of recovery during peak periods is therefore assumed to be 1.96% * 29.5 = 0.577 hours of recovery during peak period, where 29.5 equals the average annual electric DHW recovery hours for showerhead use prevented by the device including SF and MF homes with Direct Install and Retrofit/TOS measures. There are 260 hours in the peak period so the probability you will see savings during the peak period is 0.577/260 = 0.0022

## Algorithm

### CALCULATION OF ENERGY SAVINGS

#### ELECTRIC ENERGY SAVINGS

ΔkWh = %ElectricDHW * ((GPM_base_S * L_showerdevice) * Household * SPCD * 365.25 / SPH) * ISR

Where:

%ElectricDHW = proportion of water heating supplied by electric resistance heating

| DHW fuel | %ElectricDHW |
|---|---|
| Electric | 100% |
| Natural Gas | 0% |
| Unknown | 16%[1219] |

GPM_base_S = Flow rate of the basecase showerhead, or actual if available

| Program | GPM |
|---|---|
| Direct-install, device only | 2.67[1220] |
| New Construction or direct install of device and low flow showerhead | Rated or actual flow of program-installed showerhead |
| Retrofit or TOS | 2.35[1221] |

L_showerdevice = Hot water waste time avoided due to thermostatic restrictor valve

= 0.89 minutes[1222]

Household = Average number of people per household

[1219] Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

[1220] Based on measured data from Ameren IL EM&V of Direct-Install program. Program targets showers that are rated 2.5 GPM or above. Assumes low flow showerhead not included in direct installation.

[1221] Representative value from sources 1, 2, 4, 5, 6 and 7 (See Source Table at end of measure section) adjusted slightly upward to account for program participation which is expected to target customers with existing higher flow devices rather than those with existing low flow devices.

[1222] Average of the following sources: ShowerStart LLC survey; "Identifying, Quantifying and Reducing Behavioral Waste in the Shower: Exploring the Savings Potential of ShowerStart", City of San Diego Water Department survey; "Water Conservation Program: ShowerStart Pilot Project White Paper", and PG&E Work Paper PGECODHW113.

| Household Unit Type[1223] | Household |
|---|---|
| Single-Family - Deemed | 2.56[1224] |
| Multi-Family - Deemed | 2.1[1225] |
| Custom | Actual Occupancy or Number of Bedrooms[1226] |

SPCD = Showers Per Capita Per Day

= 0.6[1227]

365.25 = Days per year, on average.

SPH = Showerheads Per Household so that per-showerhead savings fractions can be determined

| Household Type | SPH |
|---|---|
| Single-Family | 1.79[1228] |
| Multi-Family | 1.3[1229] |
| Custom | Actual |

EPG_electric = Energy per gallon of hot water supplied by electric

= (8.33 * 1.0 * (ShowerTemp - SupplyTemp)) / (RE_electric * 3412)

= (8.33 * 1.0 * (101 – 54.1)) / (0.98 * 3412)

= 0.117 kWh/gal

8.33 = Specific weight of water (lbs/gallon)

1.0 = Heat Capacity of water (btu/lb-°)

ShowerTemp = Assumed temperature of water

[1223] If household type is unknown, as may be the case for time of sale measures, then single family deemed value shall be used.
[1224] ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program citing 2006-2008 American Community Survey data from the US Census Bureau for Illinois cited on p. 17 of the PY2 Evaluation report. 2.75 * 93% evaluation adjustment
[1225] ComEd PY3 Multi-Family Evaluation Report REVISED DRAFT v5 2011-12-08.docx
[1226] Bedrooms are suitable proxies for household occupancy, and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.
[1227] Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group.
[1228] Based on findings from a 2009 ComEd residential survey of 140 sites, provided by Cadmus.
[1229] Ibid.

= 101F [1230]

SupplyTemp = Assumed temperature of water entering house

= 54.1F [1231]

RE_electric = Recovery efficiency of electric water heater

= 98% [1232]

3412 = Converts Btu to kWh (btu/kWh)

ISR = In service rate of showerhead

= Dependent on program delivery method as listed in table below

| Selection | ISR |
|---|---|
| Direct Install - Single Family | 0.98[1233] |
| Direct Install – Multi Family | 0.95[1234] |
| Efficiency Kits | To be determined through evaluation |

**EXAMPLE**

For example, a direct installed valve in a single-family home with electric DHW:

ΔkWh = 1.0 * (2.67 * 0.89 * 2.56 * 0.6 * 365.25 / 1.79) * 0.117 * 0.98

= 85 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ΔkWh/Hours * CF

Where:

ΔkWh = calculated value above

[1230] Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group.
[1231] US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html.
[1232] Electric water heaters have recovery efficiency of 98%: http://www.ahridirectory.org/ahridirectory/pages/home.aspx
[1233] Deemed values are from ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program Table 3-8. Alternative ISRs may be developed for program delivery methods based on evaluation results.
[1234] Navigant, ComEd-Nicor Gas EPY4/GPY1 Multi-Family Home Energy Savings Program Evaluation Report FINAL 2013-06-05

Hours = Annual electric DHW recovery hours for wasted showerhead use prevented by device

= ((GPM_base_S * L_showerdevice) * Household * SPCD * 365.25 ) * 0.712[1235] / GPH

GPH = Gallons per hour recovery of electric water heater calculated for 65.9F temp rise (120-54.1), 98% recovery efficiency, and typical 4.5kW electric resistance storage tank.

= 27.51

= 34.4 for SF Direct Install; 28.3 for MF Direct Install

= 30.3 for SF Retrofit and TOS; 24.8 for MF Retrofit and TOS

CF = Coincidence Factor for electric load reduction

= 0.0022[1236]

**EXAMPLE**

For example, a direct installed thermostatic restrictor device in a single family home with electric DHW where the number of showers is not known.

ΔkW = 85.3/34.4 * 0.0022

= 0.0055 kW

**NATURAL GAS SAVINGS**

ΔTherms = %FossilDHW * ((GPM_base_S * L_showerdevice)* Household * SPCD * 365.25 / SPH) * EPG_gas * ISR

Where:

%FossilDHW = proportion of water heating supplied by Natural Gas heating

| DHW fuel | %Fossil_DHW |
|---|---|
| Electric | 0% |
| Natural Gas | 100% |

[1235] 71.2% is the proportion of hot 120F water mixed with 54.1F supply water to give 101F shower water.

[1236] Calculated as follows: Assume 11% showers take place during peak hours (based on: http://www.aquacraft.com/sites/default/files/pub/DeOreo-%282001%29-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf). There are 65 days in the summer peak period, so the percentage of total annual use in peak period is 0.11*65/365 = 1.96%. The number of hours of recovery during peak periods is therefore assumed to be 1.96% * 29.5 = 0.577 hours of recovery during peak period, where 29.5 equals the average annual electric DHW recovery hours for showerhead use prevented by the device including SF and MF homes with Direct Install and Retrofit/TOS measures. There are 260 hours in the peak period so the probability you will see savings during the peak period is 0.577/260 = 0.0022

| DHW fuel | %Fossil_DHW |
|---|---|
| Unknown | 84%[1237] |

EPG_gas = Energy per gallon of Hot water supplied by gas

= (8.33 * 1.0 * (ShowerTemp - SupplyTemp)) / (RE_gas * 100,000)

= 0.00501 Therm/gal for SF homes

= 0.00583 Therm/gal for MF homes

RE_gas = Recovery efficiency of gas water heater

= 78% For SF homes[1238]

= 67% For MF homes[1239]

100,000 = Converts Btus to Therms (btu/Therm)

Other variables as defined above.

**EXAMPLE**

For example, a direct installed thermostatic restrictor device in a gas fired DHW single family home where the number of showers is not known:

ΔTherms = 1.0 * ((2.67 * 0.89) * 2.56 * 0.6 * 365.25 / 1.79) * 0.00501 * 0.98

= 3.7 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

Δgallons = ((GPM_base_S * L_showerdevice) * Household * SPCD * 365.25 / SPH) * ISR

Variables as defined above

[1237] Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

[1238] DOE Final Rule discusses Recovery Efficiency with an average around 0.76 for Gas Fired Storage Water heaters and 0.78 for standard efficiency gas fired tankless water heaters up to 0.95 for the highest efficiency gas fired condensing tankless water heaters. These numbers represent the range of new units however, not the range of existing units in stock. Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 78%.

[1239] Water heating in multi-family buildings is often provided by a larger central boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of 0.59 and the 0.75 for single family homes. An average efficiency of 0.67 is used for this analysis as a default for multi-family buildings.

**EXAMPLE**

For example, a direct installed thermostatic restrictor device in a single family home where the number of showers is not known:

Δgallons = ((2.67 * 0.89) * 2.56 * 0.6 * 365.25 / 1.79) * 0.98

= 730 gallons

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**SOURCES**

| Source ID | Reference |
|---|---|
| 1 | 2011, DeOreo, William. California Single Family Water Use Efficiency Study. April 20, 2011. |
| 2 | 2000, Mayer, Peter, William DeOreo, and David Lewis. Seattle Home Water Conservation Study. December 2000. |
| 3 | 1999, Mayer, Peter, William DeOreo. Residential End Uses of Water. Published by AWWA Research Foundation and American Water Works Association. 1999. |
| 4 | 2003, Mayer, Peter, William DeOreo. Residential Indoor Water Conservation Study. Aquacraft, Inc. Water Engineering and Management. Prepared for East Bay Municipal Utility District and the US EPA. July 2003. |
| 5 | 2011, DeOreo, William. Analysis of Water Use in New Single Family Homes. By Aquacraft. For Salt Lake City Corporation and US EPA. July 20, 2011. |
| 6 | 2011, Aquacraft. Albuquerque Single Family Water Use Efficiency and Retrofit Study. For Albuquerque Bernalillo County Water Utility Authority. December 1, 2011. |
| 7 | 2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings. |
| 8 | 2011, Lutz, Jim. "Water and Energy Wasted During Residential Shower Events: Findings from a Pilot Field Study of Hot Water Distribution Systems", Energy Analysis Department Lawrence Berkeley National Laboratory, September 2011. |
| 9 | 2008, Water Conservation Program: ShowerStart Pilot Project White Paper, City of San Diego, CA. |
| 10 | 2012, Pacific Gas and Electric Company, Work Paper PGECODHW113, Low Flow Showerhead and Thermostatic Shower Restriction Valve, Revision # 4, August 2012. |
| 11 | 2008, "Simply & Cost Effectively Reducing Shower Based Warm-Up Waste: Increasing Convenience & Conservation by Attaching ShowerStart to Existing Showerheads", ShowerStart LLC. |
| 12 | 2014, New York State Record of Revision to the TRM, Case 07-M-0548, June 19, 2014. |

**MEASURE CODE: RS-HWE-TRVA-V01-150601**

# 5.5 Lighting End Use

## 5.5.1 ENERGY STAR Compact Fluorescent Lamp (CFL)

### DESCRIPTION

A low wattage ENERGY STAR qualified compact fluorescent screw-in bulb (CFL) is installed in place of a baseline screw-in bulb.

This characterization assumes that the CFL is installed in a residential location. If the implementation strategy does not allow for the installation location to be known (e.g. an upstream retail program), a deemed split of 96% Residential and 4% Commercial assumptions should be used[1240].

Federal legislation stemming from the Energy Independence and Security Act of 2007 (EISA) required all general-purpose light bulbs between 40W and 100W to be approximately 30% more energy efficient than current incandescent bulbs. Production of 100W, standard efficacy incandescent lamps ended in 2012, followed by restrictions on 75W in 2013 and 60W and 40W in 2014. The baseline for this measure has therefore become bulbs (improved incandescent or halogen) that meet the new standard.

A provision in the EISA regulations requires that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in essence making the baseline equivalent to a current day CFL. Therefore the measure life (number of years that savings should be claimed) should be reduced once the assumed lifetime of the bulb exceeds 2020. Due to expected delay in clearing retail inventory and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until mid-2020.

This measure was developed to be applicable to the following program types: TOS, NC, DI, KITS. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the high-efficiency equipment must be a standard ENERGY STAR qualified compact fluorescent lamp.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be an EISA qualified incandescent or halogen as provided in the table provided in the Electric Energy Savings section.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

Residential, Multi Family In unit bulbs and Unknown: The expected measure life (number of years that savings should be claimed) for bulbs installed June 2012 – May 2015 is assumed to be 5.2 years[1241]. For bulbs installed June 2015 – May 2016, this would be reduced to 5 years and then for every subsequent year should be reduced by one year[1242].

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[1240] RES v C&I split is based on a weighted (by sales volume) average of ComEd PY4, PY5 and PY6 and Ameren PY5 and PY6 in store intercept survey results. See 'RESvCI Split_122014.xls'.

[1241] Jump et al 2008: "Welcome to the Dark Side: The Effect of Switching on CFL Measure Life" indicates that the "observed life" of CFLs with an average rated life of 8000 hours (8000 hours is the average rated life of ENERGY STAR bulbs (http://www.energystar.gov/index.cfm?c=cfls.pr_crit_cfls) is 5.2 years.

[1242] Since the replacement baseline bulb from 2020 on will be equivalent to a CFL, no additional savings should be claimed from that point. Due to expected delay in clearing stock from retail outlets and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until mid-2020.

Exterior bulbs: The expected measure life is 3.2 years[1243] for bulbs installed June 2012 – May 2016. For bulbs installed June 2017-May 2018 this would be reduced to 3 years.

### DEEMED MEASURE COST

For the Retail (Time of Sale) measure, the incremental capital cost is $1.25 from June 2014 – May 2015, $1.6 from June 2015 to May 2016 and $1.70 from June 2017 to May 2018[1244].

For the Direct Install measure, the full cost of $2.50 per bulb should be used, plus $5 labor cost[1245] for a total of $7.50 per bulb. However actual program delivery costs should be utilized if available.

For bulbs provided in Efficiency Kits, the actual program delivery costs should be utilized.

### LOADSHAPE

Loadshape R06 - Residential Indoor Lighting

Loadshape R07 - Residential Outdoor Lighting

### COINCIDENCE FACTOR

The summer peak coincidence factor is assumed to be 7.1% for Time of Sale Residential and in-unit Multi Family bulbs, 27.3% for exterior bulbs and 8.1% for unknown[1246] and 7.4% for Residential Direct Install[1247].

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = ((WattsBase - WattsEE) / 1000) * ISR * (1-Leakage) * Hours * WHFe$$

Where:

WattsBase = Based on lumens of CFL bulb and program year installed:

| Minimum Lumens | Maximum Lumens | Incandescent Equivalent Post-EISA 2007 (WattsBase) |
|---|---|---|
| 5280 | 6209 | 300 |
| 3000 | 5279 | 200 |

[1243] Based on using 8,000 hour rated life assumption since more switching and use outdoors. 8,000/2475 = 3.2years
[1244] Based upon pricing forecast developed by Applied Proactive Technologies Inc (APT) based on industry input and provided to Ameren.
[1245] Based on 15 minutes at $20 an hour. Includes some portion of travel time to site.
[1246] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.
[1247] Based on lighting logger study conducted as part of the PY5/PY6 ComEd Residential Lighting Program evaluation and excluding all logged bulbs installed in closets.

| Minimum Lumens | Maximum Lumens | Incandescent Equivalent Post-EISA 2007 (WattsBase) |
|---|---|---|
| 2601 | 2999 | 150 |
| 1490 | 2600 | 72 |
| 1050 | 1489 | 53 |
| 750 | 1049 | 43 |
| 310 | 749 | 29 |
| 250 | 309 | 25 |

WattsEE = Actual wattage of CFL purchased / installed

ISR = In Service Rate, the percentage of units rebated that are actually in service.

| Program | | Weighted Average 1st Year In Service Rate (ISR) | 2nd year Installations | 3rd year Installations | Final Lifetime In Service Rate |
|---|---|---|---|---|---|
| Retail (Time of Sale) | | 73.2%[1248] | 13.4% | 11.4% | 98.0%[1249] |
| Direct Install | | 96.9%[1250] | | | |
| Efficiency Kits[1251] | CFL Distribution[1252] | 59% | 13% | 11% | 83% |
| | School Kits[1253] | 61% | 13% | 11% | 86% |

[1248] 1st year in service rate is based upon review of PY4-6 evaluations from ComEd and PY5-6 for Ameren (see 'IL RES Lighting ISR_122014.xls' for more information). The average first year ISR for each utility was calculated weighted by the number of bulbs in the each year's survey. This was then weighted by annual sales to give a statewide assumption.

[1249] The 98% Lifetime ISR assumption is based upon review of two evaluations:
'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings.

[1250] Based upon review of the PY2 and PY3 ComEd Direct Install program surveys. This value includes bulb failures in the 1st year to be consistent with the Commission approval of annualization of savings for first year savings claims. ComEd PY2 All Electric Single Family Home Energy Performance Tune-Up Program Evaluation, Navigant Consulting, December 21, 2010. http://www.icc.illinois.gov/downloads/public/edocket/287090.pdf.

[1251] In Service Rates provided are for the CFL bulb within a kit only. Given the significant differences in program design and the level of education provided through Efficiency Kits programs, the evaluators should apply the ISR estimated through evaluations (either past evaluations or the current program year evaluation) of the specific Efficiency Kits program. In cases where program-specific evaluation results for an ISR are unavailable, the default ISR values for Efficiency Kits provided may be used.

[1252] Free bulbs provided without request, with little or no education. Based on 'Impact and Process Evaluation of 2013 (PY6) Ameren Illinois Company Residential CFL Distribution Program', Report Table 11 and Appendix B.

[1253] Kits provided free to students through school, with education program. Based on 'Impact and Process Evaluation of 2013

| Program | | Weighted Average 1st Year In Service Rate (ISR) | 2nd year Installations | 3rd year Installations | Final Lifetime In Service Rate |
|---|---|---|---|---|---|
| | Direct Mail Kits[1254] | 66% | 14% | 12% | 93% |

Leakage = Adjustment to account for the percentage of bulbs purchased that move out (and in if deemed appropriate) of the Utility Jurisdiction.

Upstream (TOS) Lighting programs = Determined through evaluation[1255].

All other programs = 0

Hours = Average hours of use per year

| Program Delivery | Installation Location | Hours[1256] |
|---|---|---|
| Retail (Time of Sale) and Efficiency Kits | Residential Interior and in-unit Multi Family | 759 |
| | Exterior | 2,475 1257 |
| | Unknown | 847 1258 |
| Direct Install | Residential Interior and in-unit Multi Family | 793 |
| | Exterior | 2,475 |

WHFe = Waste heat factor for energy to account for cooling energy savings from efficient lighting

| Bulb Location | WHFe |
|---|---|
| Interior single family or unknown location | 1.06 [1259] |

(PY6) Ameren Illinois Company Residential Efficiency Kits Program', table 10. Final ISR assumptions are based upon comparing with CFL Distribution First year ISR and multiplying by the CFL Distribution Final ISR value, and second and third year estimates based on same proportion of future installs.

[1254] Opt-in program to receive kits via mail, with little or no education. Based on 'Impact and Process Evaluation of 2013 (PY6) Ameren Illinois Company Residential Efficiency Kits Program', table 10, as above.

[1255] Using a leakage estimate from the current program year evaluation, from past evaluation results, or a rolling average of leakage estimates from previous years.

[1256] Except where noted, based on lighting logger study conducted as part of the PY5/PY6 ComEd Residential Lighting Program evaluation. Direct Install value excludes all logged bulbs installed in closets.

[1257] Based on secondary research conducted as part of the PY5/PY6 ComEd Residential Lighting Program evaluation.

[1258] Assumes 5% exterior lighting, based on PYPY5/PY6 ComEd Residential Lighting Program evaluation.

[1259] The value is estimated at 1.06 (calculated as 1 + (0.66*(0.27 / 2.8)). Based on cooling loads decreasing by 27% of the lighting savings (average result from REMRate modeling of several different configurations and IL locations of homes), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm (-0.02 * SEER2) + (1.12 * SEER) (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted

| Bulb Location | WHFe |
|---|---|
| Multi family in unit | 1.04 [1260] |
| Exterior or uncooled location | 1.0 |

### DEFERRED INSTALLS

As presented above, the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

Year 1 (Purchase Year) installs: Characterized using assumptions provided above or evaluated assumptions if available.

Year 2 and 3 installs: Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.

The NTG factor for the Purchase Year should be applied.

For example, for a 14W CFL (60W standard incandescent and 43W EISA qualified incandescent/halogen) purchased in 2014.

$\Delta kWH_{1st\ year\ installs}$ = ((43 - 14) / 1000) * 0.722 * 847 * 1.06

= 18.8 kWh

$\Delta kWH_{2nd\ year\ installs}$ = ((43 - 14) / 1000) * 0.139 * 847 * 1.06

= 3.6 kWh

Note: Here we assume no change in hours assumption. NTG value from Purchase year applied.

$\Delta kWH_{3rd\ year\ installs}$ = ((43 - 14) / 1000) * 0.119 * 847 * 1.06

= 3.1 kWh

### HEATING PENALTY

If electric heated home (if heating fuel is unknown assume gas, see Natural Gas section):

$\Delta kWh$[1261] = - (((WattsBase - WattsEE) / 1000) * ISR * Hours * HF) / ηHeat

to COP = EER/3.412 = 2.8COP) and 66% of homes in Illinois having central cooling ("Table HC7.9 Air Conditioning in Homes in Midwest Region, Divisions, and States, 2009 from Energy Information Administration", 2009 Residential Energy Consumption Survey; http://www.eia.gov/consumption/residential/data/2009/xls/HC7.9%20Air%20Conditioning%20in%20Midwest%20Region.xls )

[1260] As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average); http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls

[1261] Negative value because this is an increase in heating consumption due to the efficient lighting.

Where:

HF = Heating Factor or percentage of light savings that must be heated

= 49%[1262] for interior or unknown location

= 0% for exterior or unheated location

ηHeat = Efficiency in COP of Heating equipment

= actual. If not available use[1263]:

| System Type | Age of Equipment | HSPF Estimate | ηHeat (COP Estimate) |
|---|---|---|---|
| Heat Pump | Before 2006 | 6.8 | 2.00 |
| | 2006 - 2014 | 7.7 | 2.26 |
| | 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1.00 |

For example, a 14W standard CFL is purchased in 2014 and installed in home with 2.0 COP Heat Pump:

$\Delta kWh_{1st\ year}$ = - (((43 - 14) / 1000) * 0.722 * 759 * 0.49) / 2.0

= - 3.9 kWh

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year.

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ((WattsBase - WattsEE) / 1 000) * ISR * WHFd * CF

Where:

WHFd = Waste heat factor for demand to account for cooling savings from efficient lighting.

| Bulb Location | WHFd |
|---|---|
| Interior single family or unknown location | 1.11[1264] |
| Multi family in unit | 1.07[1265] |

[1262] This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

[1263] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

[1264] The value is estimated at 1.11 (calculated as 1 + (0.66 * 0.466 / 2.8)). See footnote relating to WHFe for details. Note the 46.6% factor represents the average Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load.

[1265] As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table

| Bulb Location | WHFd |
|---|---|
| Exterior or uncooled location | 1.0 |

CF = Summer Peak Coincidence Factor for measure.

| Program Delivery | Bulb Location | CF[1266] |
|---|---|---|
| Retail(Time of Sale) | Interior single family or Multi Family in unit | 7.1% |
| | Exterior | 27.3% |
| | Unknown location | 8.1% |
| Direct Install | Residential | 7.4% |

Other factors as defined above

For example, a 14W standard CFL is purchased and installed in a single family interior location in 2014:

ΔkW = ((43 - 14) / 1000) * 0.722 * 1.11 * 0.071

= 0.0017 kW

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year.

### NATURAL GAS SAVINGS

Heating Penalty if Natural Gas heated home (or if heating fuel is unknown):

ΔTherms[1267] = - (((WattsBase - WattsEE) / 1000) * ISR * Hours * HF * 0.03412) / ηHeat

Where:

HF = Heating Factor or percentage of light savings that must be heated

= 49%[1268] for interior or unknown location

= 0% for exterior or unheated location

0.03412 =Converts kWh to Therms

ηHeat = Efficiency of heating system

HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average); http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls.

[1266] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation. Direct Install value is based on resut excluding all logged bulbs installed in closets.

[1267] Negative value because this is an increase in heating consumption due to the efficient lighting.

[1268] This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

=70%[1269]

For example, a14 standard CFL is purchased and installed in a home in 2014:

ΔTherms = - (((43 - 14) / 1000) * 0.722 * 759 * 0.49 * 0.03412) / 0.7

= - 0.38 Therms

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year.

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

Bulb replacement costs assumed in the O&M calculations are provided below[1270].

| | Std Inc. | EISA Compliant Halogen |
|---|---|---|
| 2014 | $0.34 | $1.25 |
| 2015 | $0.34 | $0.90 |
| 2016 | $0.34 | $0.80 |
| 2017 | $0.34 | $0.70 |
| 2018 | $0.34 | $0.60 |
| 2019 | $0.34 | $0.60 |
| 2020 & after | $0.34 | N/A |

In order to account for the falling EISA Qualified bulb replacement cost provided above, an equivalent annual levelized baseline replacement cost over the lifetime of the CFL bulb is calculated. Note that the measure life for these measures is capped to the number of years remaining until 2020.

The NPV for replacement lamps and annual levelized replacement costs using the statewide real discount rate of 5.23% are presented below:

[1269] This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls)) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:
(0.24*0.92) + (0.76*0.8) * (1-0.15) = 0.70

[1270] Based upon pricing forecast developed by Applied Proactive Technologies Inc (APT) based on industry input and provided to Ameren.

| Location | Lumen Level | NPV of replacement costs for period | | | Levelized annual replacement cost savings | | |
|---|---|---|---|---|---|---|---|
| | | June 2015 - May 2016 | June 2016 - May 2017 | June 2017 - May 2018 | June 2015 - May 2016 | June 2016 - May 2017 | June 2017 - May 2018 |
| Residential and in-unit Multi Family | Lumens <310 or >2600 (EISA exempt) | $0.86 | $0.66 | $0.45 | $0.19 | $0.15 | $0.13 |
| | Lumens ≥ 310 and ≤ 2600 (EISA compliant) | $1.73 | $1.24 | $0.80 | $0.39 | $0.29 | $0.23 |
| Exterior | Lumens <310 or >2600 (EISA exempt) | $2.91 | $2.64 | $1.96 | $1.00 | $0.91 | $0.56 |
| | Lumens ≥ 310 and ≤ 2600 (EISA compliant) | $6.21 | $5.31 | $3.60 | $2.14 | $1.83 | $1.02 |
| Unknown | Lumens <310 or >2600 (EISA exempt) | $0.97 | $0.74 | $0.51 | $0.22 | $0.17 | $0.14 |
| | Lumens ≥ 310 and ≤ 2600 (EISA compliant) | $1.93 | $1.39 | $0.90 | $0.43 | $0.32 | $0.25 |

Note incandescent lamps in lumen range <310 and >2600 are exempt from EISA. For halogen bulbs, we assume the same replacement cycle as incandescent bulbs.[1271] The replacement cycle is based on the location of the lamp and varies based on the hours of use for that location. Both incandescent and halogen lamps are assumed to last for 1,000 hours before needing replacement.

**MEASURE CODE: RS-LTG-ESCF-V04-150601**

[1271] The manufacturers of the new minimally compliant EISA Halogens are using regular incandescent lamps with halogen fill gas rather than halogen infrared to meet the standard and so the component rated life is equal to the standard incandescent.

## 5.5.2 ENERGY STAR Specialty Compact Fluorescent Lamp (CFL)

### DESCRIPTION

An ENERGY STAR qualified specialty compact fluorescent bulb is installed in place of an incandescent specialty bulb.

This characterization assumes that the specialty CFL is installed in a residential location. If the implementation strategy does not allow for the installation location to be known (e.g. an upstream retail program) a deemed split of 96% Residential and 4% Commercial assumptions should be used[1272].

This measure was developed to be applicable to the following program types: TOS, NC, DI, KITS.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

Energy Star qualified specialty CFL bulb based upon the draft ENERGY STAR specification for lamps (http://energystar.gov/products/specs/sites/products/files/ENERGY_STAR_Lamps_V1_0_Draft%203.pdf).

### DEFINITION OF BASELINE EQUIPMENT

The baseline is a specialty incandescent light bulb including those exempt of the EISA 2007 standard: three-way, plant light, daylight bulb, bug light, post light, globes G40 (<40W), candelabra base (<60W), vibration service bulb, decorative candle with medium or intermediate base (<40W), shatter resistant and reflector bulbs and standard bulbs greater than 2601 lumens, and those non-exempt from EISA 2007: dimmable, globes (less than 5" diameter and >40W), candle (shapes B, BA, CA >40W, candelabra base lamps (>60W) and intermediate base lamps (>40W).

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 6.8 year[1273].

Exterior bulbs: The expected measure life is 3.2 years[1274] for bulbs installed June 2012 – May 2017. For bulbs installed June 2017-May 2018 this would be reduced to 3 years.

### DEEMED MEASURE COST

For the Retail (Time of Sale) measure, the incremental capital cost for this measure is $5[1275].

For the Direct Install measure, the full cost of $8.50 should be used plus $5 labor[1276] for a total of $13.50. However actual program delivery costs should be utilized if available.

For bulbs provided in Efficiency Kits, the actual program delivery costs should be utilized..

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[1272] RES v C&I split is based on a weighted (by sales volume) average of ComEd PY4, PY5 and PY6 and Ameren PY5 and PY6 in store intercept survey results. See 'RESvCI Split_122014.xls'.

[1273] The assumed measure life for the specialty bulb measure characterization was reported in "Residential Lighting Measure Life Study", Nexus Market Research, June 4, 2008 (measure life for markdown bulbs). Measure life estimate does not distinguish between equipment life and measure persistence. Measure life includes products that were installed and operated until failure (i.e., equipment life) as well as those that were retired early and permanently removed from service for any reason, be it early failure, breakage, or the respondent not liking the product (i.e., measure persistence).

[1274] Based on using 8,000 hour rated life assumption since more switching and use ourdoors. 8,000/2475 = 3.2years

[1275] NEEP Residential Lighting Survey, 2011

[1276] Based on 15 minutes at $20 per hour.

### LOADSHAPE

Loadshape R06 - Residential Indoor Lighting

Loadshape R07 - Residential Outdoor Lighting

### COINCIDENCE FACTOR

Unlike standard CFLs that could be installed in any room, certain types of specialty CFLs are more likely to be found in specific rooms, which affects the coincident peak factor. Coincidence factors by bulb types are presented below[1277]

| Bulb Type | Peak CF |
|---|---|
| Three-way | 0.078[1278] |
| Dimmable | 0.078[1279] |
| Interior reflector (incl. dimmable) | 0.091 |
| Exterior reflector | 0.273 |
| Candelabra base and candle medium and intermediate base | 0.121 |
| Bug light | 0.273 |
| Post light (>100W) | 0.273 |
| Daylight | 0.081 |
| Plant light | 0.081 |
| Globe | 0.075 |
| Vibration or shatterproof | 0.081 |
| Standard spirals >= 2601 lumens, Residential, Multi-family in unit | 0.071 |
| Standard spirals >= 2601 lumens, unknown | 0.081 |
| Standard spirals >= 2601 lumens, exterior | 0.273 |
| Specialty - Generic | 0.081 |

[1277] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.
[1278] Based on average of bedroom, dining room, office and living room results from the lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.
[1279] Ibid

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

ΔkWh = ((WattsBase - WattsEE) / 1000) * ISR * (1-Leakage) * Hours * WHFe

Where:

WattsBase = Actual wattage equivalent of incandescent specialty bulb, use the tables below to obtain the incandescent bulb equivalent wattage[1280]; use 60W if unknown[1281]

EISA exempt bulb types:

| Bulb Type | Lower Lumen Range | Upper Lumen Range | WattsBase |
|---|---|---|---|
| **Standard Spirals >=2601** | 2601 | 2999 | 150 |
| | 3000 | 5279 | 200 |
| | 5280 | 6209 | 300 |
| **3-Way** | 250 | 449 | 25 |
| | 450 | 799 | 40 |
| | 800 | 1099 | 60 |
| | 1100 | 1599 | 75 |
| | 1600 | 1999 | 100 |
| | 2000 | 2549 | 125 |
| | 2550 | 2999 | 150 |
| **Globe (medium and intermediate bases less than 750 lumens)** | 90 | 179 | 10 |
| | 180 | 249 | 15 |
| | 250 | 349 | 25 |
| | 350 | 749 | 40 |
| **Decorative** | 70 | 89 | 10 |

[1280] Based upon the draft ENERGY STAR specification for lamps (http://energystar.gov/products/specs/sites/products/files/ENERGY_STAR_Lamps_V1_0_Draft%203.pdf) and the Energy Policy and Conservation Act of 2012.

[1281] A 2006-2008 California Upstream Lighting Evaluation found an average incandescent wattage of 61.7 Watts (KEMA, Inc, The Cadmus Group, Itron, Inc, PA Consulting Group, Jai J. Mitchell Analytics, Draft Evaluation Report: Upstream Lighting Program. Prepared for the California Public Utilities Commission, Energy Division. December 10, 2009)

| Bulb Type | Lower Lumen Range | Upper Lumen Range | WattsBase |
|---|---|---|---|
| **(Shapes B, BA, C, CA, DC, F, G, medium and intermediate bases less than 750 lumens)** | 90 | 149 | 15 |
| | 150 | 299 | 25 |
| | 300 | 749 | 40 |
| **Globe (candelabra bases less than 1050 lumens)** | 90 | 179 | 10 |
| | 180 | 249 | 15 |
| | 250 | 349 | 25 |
| | 350 | 499 | 40 |
| | 500 | 1049 | 60 |
| **Decorative (Shapes B, BA, C, CA, DC, F, G, candelabra bases less than 1050 lumens)** | 70 | 89 | 10 |
| | 90 | 149 | 15 |
| | 150 | 299 | 25 |
| | 300 | 499 | 40 |
| | 500 | 1049 | 60 |

**Directional Lamps -** ENERGY STAR Minimum Luminous Efficacy = 40Lm/W for lamps with rated wattages less than 20Wand 50 Lm/W for lamps with rated wattages >= 20 watts[1282].

For Directional R, BR, and ER lamp types[1283]:

| Bulb Type | Lower Lumen Range | Upper Lumen Range | $Watts_{Base}$ |
|---|---|---|---|
| **R, ER, BR with medium screw bases w/ diameter >2.25" (*see exceptions below)** | 420 | 472 | 40 |
| | 473 | 524 | 45 |
| | 525 | 714 | 50 |
| | 715 | 937 | 65 |
| | 938 | 1259 | 75 |
| | 1260 | 1399 | 90 |
| | 1400 | 1739 | 100 |
| | 1740 | 2174 | 120 |
| | 2175 | 2624 | 150 |

[1282] From pg 10 of the Energy Star Specification for lamps v1.1
[1283] From pg 11 of the Energy Star Specification for lamps v1.1

| Bulb Type | Lower Lumen Range | Upper Lumen Range | $Watts_{Base}$ |
|---|---|---|---|
| | 2625 | 2999 | 175 |
| | 3000 | 4500 | 200 |
| ***R, BR, and ER with medium screw bases w/ diameter <=2.25"** | 400 | 449 | 40 |
| | 450 | 499 | 45 |
| | 500 | 649 | 50 |
| | 650 | 1199 | 65 |
| ***ER30, BR30, BR40, or ER40** | 400 | 449 | 40 |
| | 450 | 499 | 45 |
| | 500 | 649 | 50 |
| ***BR30, BR40, or ER40** | 650 | 1419 | 65 |
| ***R20** | 400 | 449 | 40 |
| | 450 | 719 | 45 |
| ***All reflector lamps below lumen ranges specified above** | 200 | 299 | 20 |
| | 300 | 399 | 30 |

Directional lamps are exempt from EISA regulations.

For PAR, MR, and MRX Lamps Types:

For these highly focused directional lamp types, it is necessary to have Center Beam Candle Power (CBCP) and beam angle measurements to accurately estimate the equivalent baseline wattage. The formula below is based on the Energy Star Center Beam Candle Power tool.[1284] If CBCP and beam angle information are not available, refer to the R, BR, and ER lumen based method above.

$$\text{Wattsbase} =$$

$$375.1 - 4.355(D) - \sqrt{227{,}800 - 937.9(D) - 0.9903(D^2) - 1479(BA) - 12.02(D * BA) + 14.69(BA^2) - 16{,}720 * \ln(CBCP)}$$

Where:

D = Bulb diameter (e.g. for PAR20 D = 20)

BA = Beam angle

CBCP = Center beam candle power

The result of the equation above should be rounded DOWN to the nearest wattage established by Energy Star:

[1284] http://energystar.supportportal.com/link/portal/23002/23018/Article/32655/

| Diameter | Permitted Wattages |
|---|---|
| 16 | 20, 35, 40, 45, 50, 60, 75 |
| 20 | 50 |
| 30S | 40, 45, 50, 60, 75 |
| 30L | 50, 75 |
| 38 | 40, 45, 50, 55, 60, 65, 75, 85, 90, 100, 120, 150, 250 |

EISA non-exempt bulb types:

| Bulb Type | Lower Lumen Range | Upper Lumen Range | Incandescent Equivalent Post-EISA 2007 (WattsBase) |
|---|---|---|---|
| **Dimmable Twist, Globe (less than 5" in diameter and > 749 lumens), candle (shapes B, BA, CA > 749 lumens), Candelabra Base Lamps (>1049 lumens), Intermediate Base Lamps (>749 lumens)** | 310 | 749 | 29 |
| | 750 | 1049 | 43 |
| | 1050 | 1489 | 53 |
| | 1490 | 2600 | 72 |

WattsEE = Actual wattage of energy efficient specialty bulb purchased, use 15W if unknown[1285]

ISR = In Service Rate, the percentage of units rebated that are actually in service.

| Program | Weighted Average 1st year In Service Rate (ISR) | 2nd year Installations | 3rd year Installations | Final Lifetime In Service Rate |
|---|---|---|---|---|
| Retail (Time of Sale) | 88.0%[1286] | 5.4% | 4.6% | 98.0%[1287] |

[1285] An evaluation (Energy Efficiency / Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: Residential Energy Star ® Lighting http://ilsag.org/yahoo_site_admin/assets/docs/ComEd_Res_Lighting_PY2_Evaluation_Report_2010-12-21_Final.12113928.pdf ) reported 13-17W as the most common specialty CFL wattage (69% of program bulbs). 2009 California data also reported an average CFL wattage of 15.5 Watts (KEMA, Inc, The Cadmus Group, Itron, Inc, PA Consulting Group, Jai J. Mitchell Analytics, Draft Evaluation Report: Upstream Lighting Program, Prepared for the California Public Utilities Commission, Energy Division. December 10, 2009).

[1286] 1st year in service rate is based upon review of PY4-6 evaluations from ComEd and PY5-6 from Ameren (see 'IL RES Lighting ISR_122014.xls' for more information. The average first year ISR was calculated weighted by the number of bulbs in the each year's survey.

[1287] The 98% Lifetime ISR assumption is consistent with the assumption for standard CFLs (in the absence of evidence that it should be different for this bulb type) based upon review of two evaluations:
'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This

| Program | | Weighted Average 1st year In Service Rate (ISR) | 2nd year Installations | 3rd year Installations | Final Lifetime In Service Rate |
|---|---|---|---|---|---|
| Direct Install | | 96.9%[1288] | | | |
| Efficiency Kits[1289] | CFL Distribution[1290] | 59% | 13% | 11% | 83% |
| | School Kits[1291] | 61% | 13% | 11% | 86% |
| | Direct Mail Kits[1292] | 66% | 14% | 12% | 93% |

Leakage = Adjustment to account for the percentage of bulbs purchased that move out (and in if deemed appropriate) of the Utility Jurisdiction.

Upstream (TOS) Lighting programs = Determined through evaluation[1293].

All other programs = 0

Hours = Average hours of use per year, varies by bulb type as presented below:[1294]

| Bulb Type | Annual hours of use (HOU) |
|---|---|
| Three-way | 850 |
| Dimmable | 850 |
| Interior reflector (incl. dimmable) | 861 |
| Exterior reflector | 2475 |

implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings.

[1288] Consistent with assumption for standard CFLs (in the absence of evidence that it should be different for this bulb type). Based upon review of the PY2 and PY3 ComEd Direct Install program surveys. This value includes bulb failures in the 1st year to be consistent with the Commission approval of annualization of savings for first year savings claims. ComEd PY2 All Electric Single Family Home Energy Performance Tune-Up Program Evaluation, Navigant Consulting, December 21, 2010. http://www.icc.illinois.gov/downloads/public/edocket/287090.pdf.

[1289] In Service Rates provided are for the bulb within a kit only. Given the significant differences in program design and the level of education provided through Efficiency Kits programs, the evaluators should apply the ISR estimated through evaluations (either past evaluations or the current program year evaluation) of the specific Efficiency Kits program. In cases where program-specific evaluation results for an ISR are unavailable, the default ISR values for Efficiency Kits provide may be used.

[1290] Free bulbs provided without request, with little or no education. Consistent with Standard CFL assumptions.

[1291] Kits provided free to students through school, with education program. Consistent with Standard CFL assumptions.

[1292] Opt-in program to receive kits via mail, with little or no education. Consistent with Standard CFL assumptions.

[1293] Using a leakage estimate from the current program year evaluation, from past evaluation results, or a rolling average of leakage estimates from previous years.

[1294] Hours of use by specialty bulb type calculated using the average hours of use in locations or rooms where each type of specialty bulb is most commonly found. Values for Reflector, Decorative and Globe are taken directly from the lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation. All other hours have been updated based on the room specific hours of use from the PY5/PY6 logger study.

| Bulb Type | Annual hours of use (HOU) |
|---|---|
| Candelabra base and candle medium and intermediate base | 1190 |
| Bug light | 2475 |
| Post light (>100W) | 2475 |
| Daylight | 847 |
| Plant light | 847 |
| Globe | 639 |
| Vibration or shatterproof | 847 |
| Standard Spiral >2601 lumens, Residential, Multi Family in-unit | 759 |
| Standard Spiral >2601 lumens, unknown | 847 |
| Standard Spiral >2601 lumens, Exterior | 2475 |
| Specialty - Generic | 847 |

WHFe = Waste heat factor for energy to account for cooling savings from efficient lighting

| Bulb Location | WHFe |
|---|---|
| Interior single family or unknown location | 1.06 [1295] |
| Multi family in unit | 1.04 [1296] |
| Exterior or uncooled location | 1.0 |

### DEFERRED INSTALLS

As presented above, the characterization assumes that a percentage of bulbs purchased are not installed until Year

[1295] The value is estimated at 1.06 (calculated as 1 + (0.66*(0.27 / 2.8)). Based on cooling loads decreasing by 27% of the lighting savings (average result from REMRate modeling of several different configurations and IL locations of homes), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm (-0.02 * SEER2) + (1.12 * SEER) (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP) and 66% of homes in Illinois having central cooling ("Table HC7.9 Air Conditioning in Homes in Midwest Region, Divisions, and States, 2009 from Energy Information Administration", 2009 Residential Energy Consumption Survey; http://www.eia.gov/consumption/residential/data/2009/xls/HC7.9%20Air%20Conditioning%20in%20Midwest%20Region.xls )

[1296] As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average); http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls

2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

| | |
|---|---|
| Year 1 (Purchase Year) installs: | Characterized using assumptions provided above or evaluated assumptions if available. |
| Year 2 and 3 installs: | Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.<br>The NTG factor for the Purchase Year should be applied. |

For example, for a 13W dimmable CFL impacted by EISA 2007 (60W standard incandescent and 43W EISA qualified incandescent/halogen) purchased in 2013.

$\Delta kWH_{1st\ year\ installs}$ = ((60 - 13) / 1000) * 0.823 * 850 * 1.06

= 34.9 kWh

$\Delta kWH_{2nd\ year\ installs}$ = ((43 - 13) / 1000) * 0.085 * 850 * 1.06

= 2.3 kWh

Note: Here we assume no change in hours assumption. NTG value from Purchase year applied.

$\Delta kWH_{3rd\ year\ installs}$ = ((43 - 13) / 1000) * 0.072 * 850 * 1.06

= 1.9 kWh

Note: delta watts is equivalent to install year. Here we assume no change in hours assumption.

### HEATING PENALTY

If electric heated home (if heating fuel is unknown assume gas, see Natural Gas section):

ΔkWh[1297] = - (((WattsBase - WattsEE) / 1000) * ISR * Hours * HF) / ηHeat

Where:

HF = Heating Factor or percentage of light savings that must be heated

= 49%[1298] for interior or unknown location

= 0% for exterior location

ηHeat = Efficiency in COP of Heating equipment

= actual. If not available use[1299]:

| System Type | Age of Equipment | HSPF Estimate | ηHeat |
|---|---|---|---|

[1297] Negative value because this is an increase in heating consumption due to the efficient lighting.

[1298] This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

[1299] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

| | | | (COP Estimate) |
|---|---|---|---|
| Heat Pump | Before 2006 | 6.8 | 2.00 |
| | 2006 - 2014 | 7.7 | 2.26 |
| | 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1.00 |

For example, a 15W globe CFL replacing a 60W incandescent specialty bulb installed in home with 2.0 COP Heat Pump:

$\Delta kWh_{1st\ year}$ = - (((60 - 15) / 1000) * 0.823 * 639 * 0.49) / 2.0

= - 5.8 kWh

Second and third year savings should be calculated using the appropriate ISR.

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW =((WattsBase - WattsEE) / 1000) * ISR * WHFd * CF

Where:

WHFd = Waste heat factor for demand to account for cooling savings from efficient lighting.

| Bulb Location | WHFd |
|---|---|
| Interior single family or unknown location | 1.11[1300] |
| Multi family in unit | 1.07[1301] |
| Exterior or uncooled location | 1.0 |

CF = Summer Peak Coincidence Factor for measure. Coincidence factors by bulb types are presented below[1302]

| Bulb Type | Peak CF |
|---|---|
| Three-way | 0.078[1303] |

[1300] The value is estimated at 1.11 (calculated as 1 + (0.66 * 0.466 / 2.8)). See footnote relating to WHFe for details. Note the 46.6% factor represents the average Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load.

[1301] As above but using estimate of 45% of multifamily buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average); http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls.

[1302] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

[1303] Based on average of bedroom, dining room, office and living room results from the lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

| Bulb Type | Peak CF |
|---|---|
| Dimmable | 0.078[1304] |
| Interior reflector (incl. dimmable) | 0.091 |
| Exterior reflector | 0.273 |
| Candelabra base and candle medium and intermediate base | 0.121 |
| Bug light | 0.273 |
| Post light (>100W) | 0.273 |
| Daylight | 0.081 |
| Plant light | 0.081 |
| Globe | 0.075 |
| Vibration or shatterproof | 0.081 |
| Standard Spiral >=2601 lumens, Residential, Multi-family in unit | 0.071 |
| Standard spirals >= 2601 lumens, unknown | 0.081 |
| Standard spirals >= 2601 lumens, exterior | 0.273 |
| Specialty - Generic | 0.081 |

Other factors as defined above

For example, a 15W specialty CFL replacing a 60W incandescent specialty bulb:

$\Delta kW_{1st\ year}$ = ((60 - 15) / 1000) * 0.823 * 1.11 * 0.081

= 0.003 kW

Second and third year savings should be calculated using the appropriate ISR.

**NATURAL GAS SAVINGS**

Heating Penalty if Natural Gas heated home (or if heating fuel is unknown):

ΔTherms[1305] = - (((WattsBase - WattsEE) / 1000) * ISR * Hours * HF * 0.03412) / ηHeat

Where:

HF = Heating Factor or percentage of light savings that must be heated

= 49%[1306] for interior or unknown location

= 0% for exterior location

[1304] Ibid

[1305] Negative value because this is an increase in heating consumption due to the efficient lighting.

[1306] This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

| | |
|---|---|
| 0.03412 | =Converts kWh to Therms |
| ηHeat | = Efficiency of heating system |
| | =70%[1307] |

For example, a 15W Globe specialty CFL replacing a 60W incandescent specialty bulb:

ΔTherms = - (((60 - 15) / 1000) * 0. 823 * 639 * 0.49 * 0.03412) / 0.7

= - 0.57 Therms

Second and third year savings should be calculated using the appropriate ISR.

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

For those bulbs types exempt from EISA the following O&M assumptions should be used: Life of the baseline bulb is assumed to be 1.32 year[1308]; baseline replacement cost is assumed to be $3.5[1309].

For non-exempt EISA bulb types defined above, the following O&M assumptions should be used: Life of the baseline bulb is assumed to be 1.32 year[1310]; baseline replacement cost is assumed to be $5[1311].

**MEASURE CODE: RS-LTG-ESCC-V03-150601**

[1307] This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.)
In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:
(0.24*0.92) + (0.76*0.8) * (1-0.15) = 0.70
[1308] Assuming 1000 hour rated life for incandescent bulb: 1000/759 = 1.32
[1309] NEEP Residential Lighting Survey, 2011
[1310] Assuming 1000 hour rated life for halogen bulb: 1000/759 = 1.32
[1311] NEEP Residential Lighting Survey, 2011

## 5.5.3 ENERGY STAR Torchiere

### DESCRIPTION

A high efficiency ENERGY STAR fluorescent torchiere is purchased in place of a baseline mix of halogen and incandescent torchieres and installed in a residential setting.

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the fluorescent torchiere must meet ENERGY STAR efficiency standards.

### DEFINITION OF BASELINE EQUIPMENT

The baseline is based on a mix of halogen and incandescent torchieres.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The lifetime of the measure is assumed to be 8 years[1312].

### DEEMED MEASURE COST

The incremental cost for this measure is assumed to be $5[1313].

### LOADSHAPE

Loadshape R06 - Residential Indoor Lighting

Loadshape R07 - Residential Outdoor Lighting

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is 7.1% for Residential and in-unit Multi Family bulbs and 8.1% for bulbs installed in unknown locations[1314].

[1312] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.
[1313] DEER 2008 Database Technology and Measure Cost Data (www.deeresources.com) and consistent with Efficiency Vermont TRM.
[1314] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

**Algorithm**

## CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

ΔkWh = ((ΔWatts) /1000) * ISR * (1-Leakage) * HOURS * WHFe

Where:

ΔWatts = Average delta watts per purchased ENERGY STAR torchiere

= 115.8 [1315]

ISR = In Service Rate or percentage of units rebated that get installed.

= 0.86 [1316]

Leakage = Adjustment to account for the percentage of bulbs purchased that move out (and in if deemed appropriate) of the Utility Jurisdiction.

Upstream (TOS) Lighting programs = Determined through evaluation[1317].

All other programs = 0

HOURS = Average hours of use per year

| Installation Location | Hours |
|---|---|
| Residential and in-unit Multi Family | 1095 (3.0 hrs per day)[1318] |

WHFe = Waste Heat Factor for Energy to account for cooling savings from efficient lighting

| Bulb Location | WHFe |
|---|---|
| Interior single family or unknown location | 1.06 [1319] |

[1315] Nexus Market Research, "Impact Evaluation of the Massachusetts, Rhode Island and Vermont 2003 Residential Lighting Programs", Final Report, October 1, 2004, p. 43 (Table 4-9)

[1316] Nexus Market Research, RLW Analytics "Impact Evaluation of the Massachusetts, Rhode Island, and Vermont 2003 Residential Lighting Programs" table 6-3 on p63 indicates that 86% torchieres were installed in year one. http://publicservice.vermont.gov/energy/ee_files/efficiency/eval/marivtreportfinal100104.pdf

[1317] Using a leakage estimate from the current program year evaluation, from past evaluation results, or a rolling average of leakage estimates from previous years.

[1318] Nexus Market Research, "Impact Evaluation of the Massachusetts, Rhode Island and Vermont 2003 Residential Lighting Programs", Final Report, October 1, 2004, p. 104 (Table 9-7)

[1319] The value is estimated at 1.06 (calculated as 1 + (0.66*(0.27 / 2.8)). Based on cooling loads decreasing by 27% of the lighting savings (average result from REMRate modeling of several different configurations and IL locations of homes), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm (-0.02 * SEER2) + (1.12 * SEER) (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP) and 66% of homes in Illinois having central cooling ("Table HC7.9 Air Conditioning in Homes in Midwest Region, Divisions, and States, 2009 from Energy Information Administration", 2009 Residential Energy Consumption Survey; http://www.eia.gov/consumption/residential/data/2009/xls/HC7.9%20Air%20Conditioning%20in%20Midwest%20Region.xls )

| Bulb Location | WHFe |
|---|---|
| Multi family in unit | 1.04 [1320] |
| Exterior or uncooled location | 1.0 |

For single family buildings:

ΔkWh = (115.8 /1000) * 0.86 * 1095 * 1.06

= 116 kWh

For multi family in unit:

ΔkWh = (115.8 /1000) * 0.86 * 1095 * 1.04

= 113 kWh

### HEATING PENALTY

If electric heated home (if heating fuel is unknown assume gas, see Natural Gas section):

ΔkWh[1321] = - ((ΔWatts) /1000) * ISR * HOURS * HF) / ηHeat

Where:

HF = Heating Factor or percentage of light savings that must be heated

= 49%[1322] for interior or unknown location

ηHeat = Efficiency in COP of Heating equipment

= Actual. If not available use defaults provided below[1323]:

| System Type | Age of Equipment | HSPF Estimate | ηHeat (COP Estimate) |
|---|---|---|---|
| Heat Pump | Before 2006 | 6.8 | 2.00 |
| | 2006 - 2014 | 7.7 | 2.26 |
| | 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1.00 |

[1320] As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average); http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls

[1321] Negative value because this is an increase in heating consumption due to the efficient lighting.

[1322] This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

[1323] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

For example, an ES torchiere installed in a house with a newer heat pump:

ΔkWh = - ((115.8) / 1000) * 0.86 * 1095 * 0.49) / 2.26

= - 23.6 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ((ΔWatts) /1000) * ISR * WHFd * CF

Where:

WHFd = Waste Heat Factor for Demand to account for cooling savings from efficient lighting

| Bulb Location | WHFd |
|---|---|
| Interior single family or unknown location | 1.11[1324] |
| Multi family in unit | 1.07[1325] |
| Exterior or uncooled location | 1.0 |

CF = Summer Peak Coincidence Factor for measure

| Bulb Location | CF[1326] |
|---|---|
| Interior single family or Multi family in unit | 7.1% |
| Unknown location | 8.1% |

For single family and multi-family in unit buildings:

ΔkW = (115.8 / 1000) * 0.86 * 1.11 * 0.071

= 0.008kW

For unknown location:

ΔkW = (115.8 / 1000) * 0.86 * 1.07 * 0.081

= 0.009 kW

[1324] The value is estimated at 1.11 (calculated as 1 + (0.66 * 0.466 / 2.8)). See footnote relating to WHFe for details. Note the 46.6% factor represents the average Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load.

[1325] As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average); http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls.

[1326] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

### NATURAL GAS SAVINGS

Heating penalty if Natural Gas heated home, or if heating fuel is unknown.

$\Delta Therms_{WH}$ = - (((ΔWatts) /1000) * ISR * HOURS * 0.03412 * HF) / ηHeat

Where:

$\Delta Therms_{WH}$ = gross customer annual heating fuel increased usage for the measure from the reduction in lighting heat in therms.

0.03412 = conversion from kWh to therms

HF = Heating Factor or percentage of light savings that must be heated

= 49% [1327]

ηHeat = average heating system efficiency

= 70% [1328]

$\Delta Therms_{WH}$ = - ((115.8 / 1000) * 0.86 * 1095 * 0.03412 * 0.49) / 0.70

= - 2.60 therms

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

Life of the baseline bulb is assumed to be 1.83 years[1329] for residential and multifamily in unit. Baseline bulb cost replacement is assumed to be $6.[1330]

### MEASURE CODE: RS-LTG-ESTO-V02-150601

[1327] This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

[1328] This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:
(0.24*0.92) + (0.76*0.8) * (1-0.15) = 0.70

[1329] Based on VEIC assumption of baseline bulb (mix of incandescent and halogen) average rated life of 2000 hours, 2000/1095 = 1.83 years.

[1330] Derived from Efficiency Vermont TRM.

## 5.5.4 Exterior Hardwired Compact Fluorescent Lamp (CFL) Fixture

### DESCRIPTION

An ENERGY STAR lighting fixture wired for exclusive use with pin-based compact fluorescent lamps is installed in an exterior residential setting. This measure could relate to either a fixture replacement or new installation (i.e. time of sale).

Federal legislation stemming from the Energy Independence and Security Act of 2007 required all general-purpose light bulbs between 40 and 100W to be approximately 30% more energy efficient than current incandescent bulbs. Production of 100W, standard efficacy incandescent lamps ends in 2012, followed by restrictions on 75W in 2013 and 60W and 40W in 2014. The baseline for this measure has therefore become bulbs (improved incandescent or halogen) that meet the new standard.

A provision in the EISA regulations requires that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in essence making the baseline equivalent to a current day CFL. Therefore the measure life (number of years that savings should be claimed) should be reduced once the assumed lifetime of the bulb exceeds 2020. Due to expected delay in clearing retail inventory and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until mid-2020.

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is an ENERGY STAR lighting exterior fixture for pin-based compact fluorescent lamps.

### DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a standard EISA qualified incandescent or halogen exterior fixture as provided in the table provided in the Electric Energy Savings section.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected life of an exterior fixture is 20 years[1331]. However due to the backstop provision in the Energy Independence and Security Act of 2007 that requires by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, the baseline replacement would become a CFL in that year. The expected measure life for CFL fixtures installed June 2012 – May 2013 is therefore assumed to be 8 years. For bulbs installed June 2013 – May 2014, this would be reduced to 7 years and should be reduced each year[1332].

### DEEMED MEASURE COST

The incremental cost for an exterior fixture is assumed to be $32[1333].

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[1331] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007 (http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf) gives 20 years for an interior fluorescent fixture.

[1332] Due to expected delay in clearing stock from retail outlets and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until mid-2020.

1333 ENERGY STAR Qualified Lighting Savings Calculator default incremental cost input for exterior fixture (http://www.energystar.gov/buildings/sites/default/uploads/files/light_fixture_ceiling_fan_calculator.xlsx?4349-303e=&b6b3-3efd&b6b3-3efd)

### LOADSHAPE

Loadshape R07 - Residential Outdoor Lighting

### COINCIDENCE FACTOR

The summer peak coincidence factor is assumed to be 27.3%[1334].

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

ΔkWh =((WattsBase - WattsEE) / 1000) * ISR * (1-Leakage) * Hours

Where:

WattsBase = Based on lumens of CFL bulb and program year purchased:

| Minimum Lumens | Maximum Lumens | Incandescent Equivalent Post-EISA 2007 (WattsBase) |
|---|---|---|
| 5280 | 6209 | 300 |
| 3000 | 5279 | 200 |
| 2601 | 2999 | 150 |
| 1490 | 2600 | 72 |
| 1050 | 1489 | 53 |
| 750 | 1049 | 43 |
| 310 | 749 | 29 |
| 250 | 309 | 25 |

WattsEE = Actual wattage of CFL purchased

ISR = In Service Rate or the percentage of units rebated that get installed.

[1334] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

| Program | Weighted Average 1st year In Service Rate (ISR) | 2nd year Installations | 3rd year Installations | Final Lifetime In Service Rate |
|---|---|---|---|---|
| Retail (Time of Sale) | 87.5%[1335] | 5.7% | 4.8% | 98.0%[1336] |
| Direct Install | 96.9[1337] | | | |

Leakage = Adjustment to account for the percentage of bulbs purchased that move out (and in if deemed appropriate) of the Utility Jurisdiction.

Upstream (TOS) Lighting programs = Determined through evaluation[1338].

All other programs = 0

Hours = Average hours of use per year

=2475 (6.78 hrs per day)[1339]

### DEFERRED INSTALLS

As presented above, the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

Year 1 (Purchase Year) installs: Characterized using assumptions provided above or evaluated assumptions if available.

Year 2 and 3 installs: Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.

[1335] 1st year in service rate is based upon review of PY2-3 evaluations from ComEd (see 'IL RES Lighting ISR.xls' for more information. The average first year ISR was calculated weighted by the number of bulbs in the each year's survey.

[1336] The 98% Lifetime ISR assumption is consistent with the assumption for standard CFLs (in the absence of evidence that it should be different for this bulb type) based upon review of two evaluations:
'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings.

[1337] In the absence of evaluation results for Direct Install Fixtures specifically, this is made consistent with the Direct Install CFL measure which is based upon review of the PY2 and PY3 ComEd Direct Install program surveys.

[1338] Using a leakage estimate from the current program year evaluation, from past evaluation results, or a rolling average of leakage estimates from previous years.

[1339] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

The NTG factor for the Purchase Year should be applied.

For example, for a 2 x 14W pin based CFL fixture (43W EISA qualified incandescent/halogen) purchased in 2014.

$\Delta kWH_{1st\ year\ installs}$ = ((86 - 28) / 1000) * 0.875 * 2475

= 125.6 kWh

$\Delta kWH_{2nd\ year\ installs}$ = ((86 - 28) / 1000) * 0.057 * 2475

= 8.2 kWh

Note: Here we assume no change in hours assumption. NTG value from Purchase year applied.

$\Delta kWH_{3rd\ year\ installs}$ = ((86 - 28) / 1000) * 0.048 * 2475

= 6.9 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ((WattsBase - WattsEE) / 1 000) * ISR * CF

Where:

CF = Summer Peak Coincidence Factor for measure.

= 27.3%[1340]

Other factors as defined above

For example, a 2 x 14W pin-based CFL fixture is purchased in 2013:

$\Delta kW_{1st\ year}$ = ((86 - 28) / 1000) * 0.875 * 0.273

= 0.0142 kW

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year.

**NATURAL GAS SAVINGS**

N/A

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

[1340] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

### DEEMED O&M COST ADJUSTMENT CALCULATION

Bulb replacement costs assumed in the O&M calculations are provided below[1341].

| | Std Inc. | EISA Compliant Halogen |
|---|---|---|
| 2014 | $0.34 | $1.25 |
| 2015 | $0.34 | $0.90 |
| 2016 | $0.34 | $0.80 |
| 2017 | $0.34 | $0.70 |
| 2018 | $0.34 | $0.60 |
| 2019 | $0.34 | $0.60 |
| 2020 & after | $0.34 | N/A |

In order to account for the falling EISA Qualified bulb replacement cost provided above, an equivalent annual levelized baseline replacement cost over the lifetime of the CFL bulb is calculated. Note that the measure life for these measures is capped to the number of years remaining until 2020 and that the efficient case also assumes replacement cost only if the first replacement occurs before the end of the measure life. The delta O&M cost should be used in cost effectiveness screening

The NPV for replacement lamps and annual levelized replacement costs using the statewide real discount rate of 5.23% are presented below:

| Lumen Level | NPV of replacement costs for period | | | Levelized annual replacement cost savings | | |
|---|---|---|---|---|---|---|
| | June 2015 - May 2016 | June 2016 - May 2017 | June 2017 - May 2018 | June 2015 - May 2016 | June 2016 - May 2017 | June 2017 - May 2018 |
| Lumens <310 or >2600 (non-EISA compliant) | $3.30 | $2.64 | $1.96 | $0.65 | $0.61 | $0.56 |
| Lumens ≥ 310 and ≤ 2600 (EISA compliant) | $6.90 | $5.17 | $3.60 | $1.37 | $1.20 | $1.02 |
| Efficient bulb CFL | $0.06 | $0 - No replacement bulb within measure life | | $0.01 | $0 - No replacement bulb within measure life | |

For halogen bulbs, we assume the same replacement cycle as incandescent bulbs.[1342] The replacement cycle is based on the location of the lamp and varies based on the hours of use for that location. Both incandescent and halogen lamps are assumed to last for 1,000 hours before needing replacement, CFLs in Residential and in-unit multifamily assume 8000 hours and multifamily common areas assume 10,000 (longer run hours and less switching

[1341] Based upon pricing forecast developed by Applied Proactive Technologies Inc (APT) based on industry input and provided to Ameren.

[1342] The manufacturers of the new minimally compliant EISA Halogens are using regular incandescent lamps with halogen fill gas rather than halogen infrared to meet the standard and so the component rated life is equal to the standard incandescent.

leads to longer lamp life).

**MEASURE CODE: RS-LTG-EFIX-V04-150601**

## 5.5.5 Interior Hardwired Compact Fluorescent Lamp (CFL) Fixture

### DESCRIPTION

An ENERGY STAR lighting fixture wired for exclusive use with pin-based compact fluorescent lamps is installed in an interior residential setting. This measure could relate to either a fixture replacement or new installation (i.e. time of sale).

Federal legislation stemming from the Energy Independence and Security Act of 2007 required all general-purpose light bulbs between 40 and 100W to be approximately 30% more energy efficient than current incandescent bulbs. Production of 100W, standard efficacy incandescent lamps ends in 2012, followed by restrictions on 75W in 2013 and 60W and 40W in 2014. The baseline for this measure has therefore become bulbs (improved incandescent or halogen) that meet the new standard.

A provision in the EISA regulations requires that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in essence making the baseline equivalent to a current day CFL. Therefore the measure life (number of years that savings should be claimed) should be reduced once the assumed lifetime of the bulb exceeds 2020. Due to expected delay in clearing retail inventory and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until mid-2020.

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is an ENERGY STAR lighting interior fixture for pin-based compact fluorescent lamps.

### DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a standard EISA qualified incandescent or halogen interior fixture as provided in the table provided in the Electric Energy Savings section.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected life of an interior fixture is 20 years[1343]. However due to the backstop provision in the Energy Independence and Security Act of 2007 that requires by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, the baseline replacement would become equivalent to a CFL in that year. The expected measure life for CFL fixtures installed June 2012 – May 2013 is therefore assumed to be 8 years. For bulbs installed June 2013 – May 2014, this would be reduced to 7 years and should be reduced each year[1344].

### DEEMED MEASURE COST

The incremental cost for an interior fixture is assumed to be $32[1345].

---

[1343] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007 (http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf ) gives 20 years for an interior fluorescent fixture.

1344 Due to expected delay in clearing stock from retail outlets and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until mid-2020.

1345 ENERGY STAR Qualified Lighting Savings Calculator default incremental cost input for interior fixture (http://www.energystar.gov/buildings/sites/default/uploads/files/light_fixture_ceiling_fan_calculator.xlsx?4349-303e=&b6b3-3efd&b6b3-3efd)

### LOADSHAPE

Loadshape R06 - Residential Indoor Lighting

### COINCIDENCE FACTOR

The summer peak coincidence factor is assumed to be 7.1%[1346] for Residential and in-unit Multi Family bulbs.

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

ΔkWh = ((WattsBase - WattsEE) / 1000) * ISR * (1-Leakage) * Hours * WHFe

Where:

WattsBase = Based on lumens of CFL bulb and program year purchased:

| Minimum Lumens | Maximum Lumens | Incandescent Equivalent Post-EISA 2007 ($Watts_{Base}$) |
|---|---|---|
| 5280 | 6209 | 300 |
| 3000 | 5279 | 200 |
| 2601 | 2999 | 150 |
| 1490 | 2600 | 72 |
| 1050 | 1489 | 53 |
| 750 | 1049 | 43 |
| 310 | 749 | 29 |
| 250 | 309 | 25 |

WattsEE = Actual wattage of CFL purchased

ISR = In Service Rate or the percentage of units rebated that get installed.

[1346] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

| Program | Weighted Average 1st year In Service Rate (ISR) | 2nd year Installations | 3rd year Installations | Final Lifetime In Service Rate |
|---|---|---|---|---|
| Retail (Time of Sale) | 87.5%[1347] | 5.7% | 4.8% | 98.0%[1348] |
| Direct Install | 96.9[1349] | | | |

Leakage = Adjustment to account for the percentage of bulbs purchased that move out (and in if deemed appropriate) of the Utility Jurisdiction.

Upstream (TOS) Lighting programs = Determined through evaluation[1350].

All other programs = 0

Hours = Average hours of use per year

| Installation Location | Hours |
|---|---|
| Residential and in-unit Multi Family | 759 [1351] |

WHFe = Waste heat factor for energy to account for cooling energy savings from efficient lighting

| Bulb Location | WHFe |
|---|---|
| Interior single family or unknown location | 1.06 [1352] |

[1347] 1st year in service rate is based upon review of PY2-3 evaluations from ComEd (see 'IL RES Lighting ISR.xls' for more information. The average first year ISR was calculated weighted by the number of bulbs in the each year's survey.

[1348] The 98% Lifetime ISR assumption is consistent with the assumption for standard CFLs (in the absence of evidence that it should be different for this bulb type) based upon review of two evaluations:
'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings.

[1349] In the absence of evaluation results for Direct Install Fixtures specifically, this is made consistent with the Direct Install CFL measure which is based upon review of the PY2 and PY3 ComEd Direct Install program surveys.

[1350] Using a leakage estimate from the current program year evaluation, from past evaluation results, or a rolling average of leakage estimates from previous years.

[1351] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

[1352] The value is estimated at 1.06 (calculated as 1 + (0.66*(0.27 / 2.8)). Based on cooling loads decreasing by 27% of the lighting savings (average result from REMRate modeling of several different configurations and IL locations of homes), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm (-0.02 * SEER2) + (1.12 * SEER) (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP) and 66% of homes in Illinois having central cooling ("Table HC7.9 Air Conditioning in Homes in Midwest Region, Divisions, and States, 2009 from Energy Information Administration", 2009 Residential Energy Consumption Survey;

| Bulb Location | WHFe |
|---|---|
| Multi family in unit | 1.04 [1353] |

### DEFERRED INSTALLS

As presented above, the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

Year 1 (Purchase Year) installs: Characterized using assumptions provided above or evaluated assumptions if available.

Year 2 and 3 installs: Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.

The NTG factor for the Purchase Year should be applied.

For example, for a 2 x 14W pin based CFL fixture (43W EISA qualified incandescent/halogen) purchased in 2013.

$\Delta kWH_{1st\ year\ installs}$ = ((86 - 28) / 1000) * 0.875 * 759 * 1.06

= 40.8 kWh

$\Delta kWH_{2nd\ year\ installs}$ = ((86 - 28) / 1000) * 0.057 * 759 * 1.06

= 2.7 kWh

Note: Here we assume no change in hours assumption. NTG value from Purchase year applied.

$\Delta kWH_{3rd\ year\ installs}$ = ((86 - 28) / 1000) * 0.048 * 759 * 1.06

= 2.2 kWh

### HEATING PENALTY

If electric heated building:

$\Delta kWh$[1354] = - (((WattsBase - WattsEE) / 1000) * ISR * Hours * HF) / ηHeat

Where:

HF = Heating Factor or percentage of light savings that must be heated

= 49%[1355] for interior or unknown location

= 0% for unheated location

http://www.eia.gov/consumption/residential/data/2009/xls/HC7.9%20Air%20Conditioning%20in%20Midwest%20Region.xls)

[1353] As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average); http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls

[1354] Negative value because this is an increase in heating consumption due to the efficient lighting.

[1355] This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

ηHeat = Efficiency in COP of Heating equipment

= actual. If not available use[1356]:

| System Type | Age of Equipment | HSPF Estimate | ηHeat (COP Estimate) |
|---|---|---|---|
| Heat Pump | Before 2006 | 6.8 | 2.00 |
| | 2006 -2014 | 7.7 | 2.26 |
| | 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1.00 |

For example, a 2 x 14W pin-based CFL fixture is purchased in 2013 and installed in home with 2.0 COP Heat Pump:

$\Delta kWh_{1st\ year}$ = - (((86 – 28) / 1000) * 0.875 * 759 * 0.49) / 2.0

= - 9.4 kWh

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year.

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ((WattsBase - WattsEE) / 1 000) * ISR * WHFd * CF

Where:

WHFd = Waste heat factor for demand to account for cooling savings from efficient lighting.

| Bulb Location | WHFd |
|---|---|
| Interior single family or unknown location | 1.11[1357] |
| Multi family in unit | 1.07[1358] |
| Exterior or uncooled location | 1.0 |

[1356] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

[1357] The value is estimated at 1.11 (calculated as 1 + (0.66 * 0.466 / 2.8)). See footnote relating to WHFe for details. Note the 46.6% factor represents the average Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load.

[1358] As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average); http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls .

CF = Summer Peak Coincidence Factor for measure.

| Bulb Location | CF[1359] |
|---|---|
| Interior single family or unknown location | 7.1% |
| Multi family in unit | 7.1% |

Other factors as defined above

For example, a 14W pin-based CFL fixture is purchased in 2013:

$\Delta kW_{1st\ year}$ = ((86- 28) / 1000) * 0.875 * 1.11 * 0.071

= 0.004 kW

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year.

**NATURAL GAS SAVINGS**

ΔTherms[1360] = - (((WattsBase - WattsEE) / 1000) * ISR * Hours * HF * 0.03412) / ηHeat

Where:

HF = Heating Factor or percentage of light savings that must be heated

= 49%[1361] for interior or unknown location

= 0% for unheated location

0.03412 =Converts kWh to Therms

ηHeat = Efficiency of heating system

=70%[1362]

[1359] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

[1360] Negative value because this is an increase in heating consumption due to the efficient lighting.

[1361] This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

[1362] This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls)) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:
(0.24*0.92) + (0.76*0.8) * (1-0.15) = 0.70

For example, a 2 x 14W pin-based CFL fixture is purchased in 2013 and installed in home with gas heat at 70% efficiency:

$$\Delta Therms_{1st\ year} = -((86 - 28) / 1000) * 0.875 * 759 * 0.49 * 0.03412) / 0.7$$

$$= - 0.9\ Therms$$

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year.

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

Bulb replacement costs assumed in the O&M calculations are provided below[1363].

| | Std Inc. | EISA Compliant Halogen |
|---|---|---|
| 2014 | $0.34 | $1.25 |
| 2015 | $0.34 | $0.90 |
| 2016 | $0.34 | $0.80 |
| 2017 | $0.34 | $0.70 |
| 2018 | $0.34 | $0.60 |
| 2019 | $0.34 | $0.60 |
| 2020 & after | $0.34 | N/A |

In order to account for the falling EISA Qualified bulb replacement cost provided above, an equivalent annual levelized baseline replacement cost over the lifetime of the CFL bulb is calculated. Note that the measure life for these measures is capped to the number of years remaining until 2020 and that the efficient case also assumes replacement cost only if the first replacement occurs before the end of the measure life. The delta O&M cost should be used in cost effectiveness screening

The NPV for replacement lamps and annual levelized replacement costs using the statewide real discount rate of 5.23% are presented below:

[1363] Based upon pricing forecast developed by Applied Proactive Technologies Inc (APT) based on industry input and provided to Ameren.

| Location | Lumen Level | NPV of replacement costs for period | | | Levelized annual replacement cost savings | | |
|---|---|---|---|---|---|---|---|
| | | June 2015 - May 2016 | June 2016 - May 2017 | June 2017 - May 2018 | June 2015 - May 2016 | June 2016 - May 2017 | June 2017 - May 2018 |
| Residential and in-unit Multi Family | Lumens <310 or >2600 (non-EISA compliant) | $0.86 | $0.66 | $0.45 | $0.17 | $0.15 | $0.13 |
| | Lumens ≥ 310 and ≤ 2600 (EISA compliant) | $1.73 | $1.24 | $0.80 | $0.34 | $0.29 | $0.23 |
| | Efficient bulb CFL | $0 - No replacement bulb within measure life | | | $0 - No replacement bulb within measure life | | |

For halogen bulbs, we assume the same replacement cycle as incandescent bulbs.[1364] The replacement cycle is based on the location of the lamp and varies based on the hours of use for that location. Both incandescent and halogen lamps are assumed to last for 1,000 hours before needing replacement, CFLs in Residential and in-unit multi family assume 8000 hours

**MEASURE CODE: RS-LTG-IFIX-V04-150601**

[1364] The manufacturers of the new minimally compliant EISA Halogens are using regular incandescent lamps with halogen fill gas rather than halogen infrared to meet the standard and so the component rated life is equal to the standard incandescent.

## 5.5.6 LED Downlights

### DESCRIPTION

This measure describes savings from a variety of LED downlight lamp types. This characterization assumes that the LED lamp or fixture is installed in a residential location. Where the implementation strategy does not allow for the installation location to be known (e.g. an upstream retail program) evaluation data could be used to determine an appropriate residential v commercial split. If this is not available, it is recommended to use this residential characterization for all installs in unknown locations to be appropriately conservative in savings assumptions.

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be an ENERGY STAR LED lamp or fixture.

### DEFINITION OF BASELINE EQUIPMENT

The baseline condition is assumed to be an incandescent/halogen lamp for all lamp types.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is given in the following table.[1365]

| Bulb Type | Measure Life (yr) |
|---|---|
| PAR20, PAR30, PAR38 screw-in lamps | 10 |
| MR16/PAR16 pin-based lamps | 10 |
| Recessed downlight luminaries | 15 |
| Track lights | 15 |

### DEEMED MEASURE COST

The price of LED lamps is falling quickly. Where possible the actual cost should be used and compared to the baseline cost provided below. If the incremental cost is unknown, assume the following[1366]:

| Bulb Type | Baseline Cost | LED Cost | Incremental Cost |
|---|---|---|---|
| PAR20, PAR30, PAR38 screw-in lamps | $4.00 | $44.00 | $40.00 |
| MR16/PAR16 pin-based lamps | $3.00 | $28.00 | $25.00 |

[1365] Limited by persistence. NEEP EMV Emerging Technologies Research Report (December 2011)

[1366] Costs are provided as the best estimate from VEIC and are based on review of available product and of price reports provided to Efficiency Vermont by a number of manufacturers and retailers.

| Bulb Type | Baseline Cost | LED Cost | Incremental Cost |
|---|---|---|---|
| Recessed downlight luminaries | $4.00 | $94.00 | $90.00 |
| Track lights | $4.00 | $60.00 | $56.00 |

### LOADSHAPE

Loadshape R06 - Residential Indoor Lighting

Loadshape R07 - Residential Outdoor Lighting

### COINCIDENCE FACTOR

The summer Peak Coincidence Factor is assumed to be 9.1% for Residential and in-unit Multi Family bulbs, 27.3% for bulbs installed in Exterior locations, and 9.4% for bulbs installed in unknown locations[1367].

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

ΔkWh = ((WattsBase - WattsEE) / 1000) * ISR * (1-Leakage) * Hours * WHFe

Where:

$Watts_{base}$ = Input wattage of the existing or baseline system. Reference the table below for default values.

$Watts_{EE}$ = Actual wattage of LED purchased / installed. If unknown, use default provided below:

- ENERGY STAR Minimum Luminous Efficacy = 40Lm/W for lamps with rated wattages less than 20Wand 50 Lm/W for lamps with rated wattages >= 20 watts[1368].

For Directional R, BR, and ER lamp types[1369]:

| Bulb Type | Lower Lumen Range | Upper Lumen Range | $Watts_{Base}$ | Lumens used to calculate LED Wattage (midpoint) | LED Wattage ($Watts_{EE}$) | Delta Watts |
|---|---|---|---|---|---|---|
| **R, ER, BR with medium screw bases w/** | 420 | 472 | 40 | 446 | 11 | 29 |
| | 473 | 524 | 45 | 499 | 12 | 33 |
| | 525 | 714 | 50 | 620 | 15 | 35 |

[1367] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.
[1368] From pg 10 of the Energy Star Specification for lamps v1.1
[1369] From pg 11 of the Energy Star Specification for lamps v1.1

| Bulb Type | Lower Lumen Range | Upper Lumen Range | $Watts_{Base}$ | Lumens used to calculate LED Wattage (midpoint) | LED Wattage ($Watts_{EE}$) | Delta Watts |
|---|---|---|---|---|---|---|
| **diameter >2.25" (*see exceptions below)** | 715 | 937 | 65 | 826 | 21 | 44 |
| | 938 | 1259 | 75 | 1099 | 22 | 53 |
| | 1260 | 1399 | 90 | 1330 | 27 | 63 |
| | 1400 | 1739 | 100 | 1570 | 31 | 69 |
| | 1740 | 2174 | 120 | 1957 | 39 | 81 |
| | 2175 | 2624 | 150 | 2400 | 48 | 102 |
| | 2625 | 2999 | 175 | 2812 | 56 | 119 |
| | 3000 | 4500 | 200 | 3750 | 75 | 125 |
| ***R, BR, and ER with medium screw bases w/ diameter <=2.25"** | 400 | 449 | 40 | 425 | 11 | 29 |
| | 450 | 499 | 45 | 475 | 12 | 33 |
| | 500 | 649 | 50 | 575 | 14 | 36 |
| | 650 | 1199 | 65 | 925 | 23 | 42 |
| ***ER30, BR30, BR40, or ER40** | 400 | 449 | 40 | 425 | 11 | 29 |
| | 450 | 499 | 45 | 475 | 12 | 33 |
| | 500 | 649 | 50 | 575 | 14 | 36 |
| ***BR30, BR40, or ER40** | 650 | 1419 | 65 | 1035 | 21 | 44 |
| ***R20** | 400 | 449 | 40 | 425 | 11 | 29 |
| | 450 | 719 | 45 | 585 | 15 | 30 |
| ***All reflector lamps below lumen ranges specified above** | 200 | 299 | 20 | 250 | 6 | 14 |
| | 300 | 399 | 30 | 350 | 9 | 21 |

Directional lamps are exempt from EISA regulations.

For PAR, MR, and MRX Lamps Types:

For these highly focused directional lamp types, it is necessary to have Center Beam Candle Power (CBCP) and beam angle measurements to accurately estimate the equivalent baseline wattage. The formula below is based on the Energy Star Center Beam Candle Power tool.[1370] If CBCP and beam angle information are not available, refer to the R, BR, and ER lumen based method above.

$$\text{Wattsbase} =$$

$$375.1 - 4.355(D) - \sqrt{227{,}800 - 937.9(D) - 0.9903(D^2) - 1479(BA) - 12.02(D * BA) + 14.69(BA^2) - 16{,}720 * \ln(CBCP)}$$

Where:

[1370] http://energystar.supportportal.com/link/portal/23002/23018/Article/32655/

D = Bulb diameter (e.g. for PAR20 D = 20)

BA = Beam angle

CBCP = Center beam candle power

The result of the equation above should be rounded DOWN to the nearest wattage established by Energy Star:

| Diameter | Permitted Wattages |
|---|---|
| 16 | 20, 35, 40, 45, 50, 60, 75 |
| 20 | 50 |
| 30S | 40, 45, 50, 60, 75 |
| 30L | 50, 75 |
| 38 | 40, 45, 50, 55, 60, 65, 75, 85, 90, 100, 120, 150, 250 |

ISR = In Service Rate or the percentage of units rebated that get installed[1371]

| Bulb Type | ISR |
|---|---|
| PAR20, PAR30, PAR38 screw-in lamps | 0.95 |
| MR16/PAR16 pin-based lamps | 0.95 |
| Recessed downlight luminaries | 1.0 |
| Track lights | 1.0 |

Leakage = Adjustment to account for the percentage of bulbs purchased that move out (and in if deemed appropriate) of the Utility Jurisdiction.

Upstream (TOS) Lighting programs = Determined through evaluation[1372].

All other programs = 0

Hours = Average hours of use per year

| Installation Location | Hours[1373] |
|---|---|
| Residential and in-unit Multi Family | 861 |

[1371] NEEP EMV Emerging Technologies Research Report (December 2011)

[1372] Using a leakage estimate from the current program year evaluation, from past evaluation results, or a rolling average of leakage estimates from previous years.

[1373] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluations.

| Installation Location | Hours[1373] |
|---|---|
| Unknown location | 891 |
| Exterior | 2475 |

WHFe = Waste heat factor for energy to account for cooling savings from efficient lighting

| Bulb Location | WHFe |
|---|---|
| Interior single family or unknown location | 1.06 [1374] |
| Multi family in unit | 1.04 [1375] |
| Exterior or uncooled location | 1.0 |

For example, a 13W PAR20 LED is installed in place of a 750 lumen PAR20 incandescent screw-in lamp with medium screw base, diameter >2.5", installed in single family interior location:

ΔkWh = ((45 - 13) / 1000) * 0.95 * 861 * 1.06

= 27.7 kWh

**HEATING PENALTY**

If electric heated home (if heating fuel is unknown assume gas, see Natural Gas section):

ΔkWh[1376] = - (((WattsBase - WattsEE) / 1000) * ISR * Hours * HF) / ηHeat

Where:

HF = Heating Factor or percentage of light savings that must be heated

= 49%[1377] for interior or unknown location

[1374] The value is estimated at 1.06 (calculated as 1 + (0.66*(0.27 / 2.8)). Based on cooling loads decreasing by 27% of the lighting savings (average result from REMRate modeling of several different configurations and IL locations of homes), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm (-0.02 * SEER2) + (1.12 * SEER) (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP) and 66% of homes in Illinois having central cooling ("Table HC7.9 Air Conditioning in Homes in Midwest Region, Divisions, and States, 2009 from Energy Information Administration", 2009 Residential Energy Consumption Survey; http://www.eia.gov/consumption/residential/data/2009/xls/HC7.9%20Air%20Conditioning%20in%20Midwest%20Region.xls)

[1375] As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average); http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls

[1376] Negative value because this is an increase in heating consumption due to the efficient lighting.

= 0% for exterior location

ηHeat = Efficiency in COP of Heating equipment

= Actual. If not available use:[1378]

| System Type | Age of Equipment | HSPF Estimate | ηHeat (COP Estimate) |
|---|---|---|---|
| Heat Pump | Before 2006 | 6.8 | 2.00 |
| | After 2006 - 2014 | 7.7 | 2.26 |
| | 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1.00 |

For example, a 13W PAR20 LED is installed in place of a 750 lumen PAR20 incandescent screw-in lamp with medium screw base, diameter >2.5",installed in single family interior location:

ΔkWh = - ((45 - 13) / 1000) * 0.95 * 861 * 0.49) / 2.26

= - 5.67 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = ((WattsBase - WattsEE) / 1000) * ISR * WHFd * CF

Where:

WHFd = Waste heat factor for demand to account for cooling savings from efficient lighting.

| Bulb Location | WHFd |
|---|---|
| Interior single family or unknown location | 1.11[1379] |
| Multi family in unit | 1.07[1380] |

[1377] This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

[1378] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

[1379] The value is estimated at 1.11 (calculated as 1 + (0.66 * 0.466 / 2.8)). See footnote relating to WHFe for details. Note the 46.6% factor represents the average Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load.

[1380] As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average); http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Ty

| Bulb Location | WHFd |
|---|---|
| Exterior or uncooled location | 1.0 |

CF = Summer Peak Coincidence Factor for measure, see above for values.

| Bulb Location | CF[1381] |
|---|---|
| Interior single family or Multi-family in unit | 9.1% |
| Unknown Location | 9.4% |
| Exterior Locations | 27.3% |

Other factors as defined above

For example, a 13W PAR20 LED is installed in place of a 750 lumen PAR20 incandescent screw-in lamp with medium screw base, diameter >2.5", installed in single family interior location:

ΔkW = ((45 - 13) / 1000) * 0.95 * 1.11* 0.091

= 0.0031 kW

**NATURAL GAS SAVINGS**

Heating penalty if Natural Gas heated home, or if heating fuel is unknown.

Δtherms = - (((WattsBase - WattsEE) / 1000) * ISR * Hours * HF * 0.03412) / ηHeat

Where:

HF = Heating factor, or percentage of lighting savings that must be replaced by heating system.

= 49% [1382] for interior or unknown location

= 0% for exterior location

0.03412 = Converts kWh to Therms

ηHeat = Average heating system efficiency.

= 0.70 [1383]

pe.xls.

[1381] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

[1382] Average result from REMRate modeling of several different configurations and IL locations of homes

[1383] This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls)) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State.

Other factors as defined above

For example, a 13W PAR20 LED is installed in place of a 750 lumen PAR20 incandescent screw-in lamp with medium screw base, diameter >2.5", installed in single family interior location with gas heating at 70% total efficiency:

Δtherms = - (((45 - 13) / 1000) * 0.95 * 861 * 0.49* 0.03412) / 0.70

= - 0.63 therms

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

The life of the baseline bulb and the cost of its replacement is presented in the following table:

| Lamp Type | Baseline Lamp Life (hours) | Baseline Life (Single Family and in unit Multifamily - 1010 hours) | Baseline Replacement Cost |
|---|---|---|---|
| PAR20, PAR30, PAR38 screw-in lamps | 2000 | 2.0 | $4.00 |
| MR16/PAR16 pin-based lamps | 2000 | 2.0 | $3.00 |
| Recessed downlight luminaries | 2000 | 2.0 | $4.00 |
| Track lights | 2000 | 2.0 | $4.00 |

### MEASURE CODE: RS-LTG-LEDD-V04-150601

Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:
(0.24*0.92) + (0.76*0.8) * (1-0.15) = 0.70

## 5.5.7 LED Exit Signs

### DESCRIPTION

This measure characterizes the savings associated with installing a Light Emitting Diode (LED) exit sign in place of a fluorescent or incandescent exit sign in a MultiFamily building. Light Emitting Diode exit signs have a string of very small, typically red or green, glowing LEDs arranged in a circle or oval. The LEDs may also be arranged in a line on the side, top or bottom of the exit sign. LED exit signs provide the best balance of safety, low maintenance, and very low energy usage compared to other exit sign technologies.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is assumed to be an exit sign illuminated by LEDs.

### DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be a fluorescent or incandescent model.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 16 years[1384].

### DEEMED MEASURE COST

The incremental cost for this measure is assumed to be $30[1385].

### LOADSHAPE

Loadshape C53 - Flat

### COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 100%[1386].

**Algorithm**

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = ((WattsBase - WattsEE) / 1000) * HOURS * WHF_e$$

Where:

[1384] 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, "Effective/Remaining Useful Life Values", California Public Utilities Commission, December 16, 2008.
[1385] NYSERDA Deemed Savings Database, Labor cost assumes 25 minutes @ $18/hr.
[1386] Assuming continuous operation of an LED exit sign, the Summer Peak Coincidence Factor is assumed to equal 1.0.

WattsBase = Actual wattage if known, if unknown assume the following:

| Baseline Type | WattsBase |
|---|---|
| Incandescent | 35W[1387] |
| Fluorescent | 11W[1388] |
| Unknown (e.g. time of sale) | 11W |

WattsEE = Actual wattage if known, if unknown assume 2W[1389]

HOURS = Annual operating hours

= 8766

$WHF_e$ = Waste heat factor for energy; accounts for cooling savings from efficient lighting.

= 1.04[1390] for multi family buildings

Default if replacing incandescent fixture

ΔkWH = (35 – 2)/1000 * 8766 * 1.04

= 301 kWh

Default if replacing fluorescent fixture

ΔkWH = (11 – 2)/1000 * 8766 * 1.04

= 82 kWh

**HEATING PENALTY**

If electric heated building (if heating fuel is unknown assume gas, see Natural Gas section):

ΔkWh[1391] = - (((WattsBase - WattsEE) / 1000) * Hours * HF) / ηHeat

Where:

[1387] Based on review of available product.

[1388] Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, February, 19, 2010

[1389] Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, February, 19, 2010

[1390] The value is estimated at 1.04 (calculated as 1 + (0.45*(0.27 / 2.8)). Based on cooling loads decreasing by 27% of the lighting savings (average result from REMRate modeling of several different configurations and IL locations of homes), assuming typical cooling system operating efficiency of 3.1 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm (-0.02 * SEER2) + (1.12 * SEER) (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP) and estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average); http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls

[1391] Negative value because this is an increase in heating consumption due to the efficient lighting.

HF = Heating Factor or percentage of light savings that must be heated

= 49%[1392]

ηHeat = Efficiency in COP of Heating equipment

= Actual. If not available use:[1393]

| System Type | Age of Equipment | HSPF Estimate | ηHeat (COP Estimate) |
|---|---|---|---|
| Heat Pump | Before 2006 | 6.8 | 2.00 |
| | After 2006 | 7.7 | 2.26 |
| Resistance | N/A | N/A | 1.00 |

For example, a 2.0COP Heat Pump heated building:

If incandescent fixture: ΔkWH = -((35 – 2)/1000 * 8766 * 0.49) / 2

= -71 kWh

If fluorescent fixture ΔkWH = -((11 – 2)/1000 * 8766 * 0.49) / 2

= -19 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

$$\Delta kW = ((WattsBase - WattsEE) / 1000) * WHF_d * CF$$

Where:

$WHF_d$ = Waste heat factor for demand to account for cooling savings from efficient lighting. The cooling savings are only added to the summer peak savings.

=1.07[1394] for multi family buildings

CF = Summer Peak Coincidence Factor for measure

= 1.0

Default if incandescent fixture

ΔkW = (35 – 2)/1000 * 1.07 * 1.0

= 0.035 kW

[1392] This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

[1393] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

[1394] The value is estimated at 1.11 (calculated as 1 + (0.45 * 0.466 / 2.8)). See footnote relating to WHFe for details. Note the 46.6% factor represents the average Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load.

Default if fluorescent fixture

ΔkW = (11 – 2)/1000 * 1.07 * 1.0

= 0.0096 kW

**NATURAL GAS SAVINGS**

Heating penalty if Natural Gas heated building, or if heating fuel is unknown.

Δtherms = - (((WattsBase - WattsEE) / 1000) * Hours * HF * 0.03412) / ηHeat

Where:

HF = Heating factor, or percentage of lighting savings that must be replaced by heating system.

= 49% [1395]

0.03412 = Converts kWh to Therms

ηHeat = Average heating system efficiency.

= 0.70 [1396]

Other factors as defined above

Default if incandescent fixture

Δtherms = - (((35 - 2) / 1000) * 8766 * 0.49* 0.03412) / 0.70

= -6.9 therms

Default if fluorescent fixture

Δtherms = - (((11 - 2) / 1000) * 8766 * 0.49* 0.03412) / 0.70

= -1.9 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

[1395] Average result from REMRate modeling of several different configurations and IL locations of homes

[1396] This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls)) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:
(0.24*0.92) + (0.76*0.8) * (1-0.15) = 0.70

### DEEMED O&M COST ADJUSTMENT CALCULATION

The annual O&M Cost Adjustment savings should be calculated using the following component costs and lifetimes.

| | Baseline Measures | |
|---|---|---|
| Component | Cost | Life (yrs) |
| Lamp | $7.00[1397] | 1.37 years[1398] |

### MEASURE CODE: RS-LTG-LEDE-V01-120601

[1397] Consistent with assumption for a Standard CFL bulb with an estimated labor cost of $4.50 (assuming $18/hour and a task time of 15 minutes).
[1398] Assumes a lamp life of 12,000 hours and 8766 run hours 12000/8766 = 1.37 years.

## 5.5.8 LED Screw Based Omnidirectional Bulbs

### DESCRIPTION

This characterization provides savings assumptions for LED Screw Based Omnidirectional (e.g. A-Type lamps) lamps within the residential and multifamily sectors. This characterization assumes that the LED lamp or fixture is installed in a residential location. Where the implementation strategy does not allow for the installation location to be known (e.g. an upstream retail program) evaluation data could be used to determine an appropriate residential v commercial split. If this is not available, it is recommended to use this residential characterization for all installs in unknown locations to be appropriately conservative in savings assumptions.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, new lamps must be Energy Star labeled.

### DEFINITION OF BASELINE EQUIPMENT

In 2012, Federal legislation stemming from the Energy Independence and Security Act of 2007 (EIAS) will require all general-purpose light bulbs between 40 watts and 100 watts to have ~30% increased efficiency, essentially phasing out standard incandescent technology. In 2012, the 100 w lamp standards apply; in 2013 the 75 w lamp standards will apply, followed by restrictions on the 60 w and 40 w lamps in 2014. Since measures installed under this TRM all occur after 2014, baseline equipment are the values after EISA. These are shown in the baseline table below.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

13.7 years (exterior) to 26 years (residential home), however all installations are capped at 10 years[1399].

### DEEMED MEASURE COST

Wherever possible, actual incremental costs should be used. Refer to reference table "Residential LED component Cost & Lifetime" for defaults.

### LOADSHAPE

Loadshape R06 – Residential Indoor Lighting

Loadshape R07 – Residential Outdoor Lighting

### COINCIDENCE FACTOR

The summer peak coincidence factor is assumed to be 7.1% for Residential and in-unit Multi Family bulbs, 27.3% for exterior bulbs and 8.1% for unknown[1400].

[1399] Based on recommendation in the Dunsky Energy Consulting, Livingston Energy Innovations and Opinion Dynamics Corporation; NEEP Emerging Technology Research Report: https://www.neep.org/Assets/uploads/files/emv/emv-products/NEEP_EMV_EmergingTechResearch_Report_Final.pdf, p 6-18.

[1400] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

$$\Delta kWh = ((Watts_{base}-Watts_{EE})/1000) * ISR * (1-Leakage) * Hours * WHF_e$$

Where:

$Watts_{base}$ = Input wattage of the existing or baseline system. Reference the "LED New and Baseline Assumptions" table for default values.

$Watts_{EE}$ = Actual wattage of LED purchased / installed. If unknown, use default provided below:

**LED New and Baseline Assumptions Table**

| Minimum Lumens | Maximum Lumens | Lumens used to calculate LED Wattage (midpoint) | LED Wattage [1401] (WattsEE) | Baseline 2014-2019 (WattsBase) | Delta Watts 2014-2019 (WattsEE) | Baseline Post EISA 2020 requirement[1402] (WattsBase) | Delta Watts Post 2020 (WattsEE) |
|---|---|---|---|---|---|---|---|
| 5280 | 6209 | 5745 | 104.4 | 300.0 | 195.6 | 300.0 | 195.6 |
| 3000 | 5279 | 4140 | 75.3 | 200.0 | 124.7 | 200.0 | 124.7 |
| 2601 | 2999 | 2800 | 50.9 | 150.0 | 99.1 | 150.0 | 99.1 |
| 1490 | 2600 | 2045 | 37.2 | 72.0 | 34.8 | 45.4 | 8.3 |
| 1050 | 1489 | 1270 | 23.1 | 53.0 | 29.9 | 28.2 | 5.1 |
| 750 | 1049 | 900 | 16.4 | 43.0 | 26.6 | 20.0 | 3.6 |
| 310 | 749 | 530 | 9.6 | 29.0 | 19.4 | 11.8 | 2.1 |
| 250 | 309 | 280 | 5.6 | 25.0 | 19.4 | 25.0 | 19.4 |

ISR = In Service Rate, the percentage of units rebated that are actually in service.

[1401] Based on ENERGY STAR specs – minimum luminous efficacy for Omnidirectional Lamps. For LED lamp power <10W = 50lm/W and for LED lamp power >=10W = 55lm/W.
[1402] Calculated as 45lm/W for all EISA non-exempt bulbs.

| Program | | Weighted Average 1st year In Service Rate (ISR) | 2nd year Installations | 3rd year Installations | Final Lifetime In Service Rate |
|---|---|---|---|---|---|
| Retail (Time of Sale) | | 95%[1403] | 1.6% | 1.4% | 98.0%[1404] |
| Direct Install | | 96.9%[1405] | | | |
| Efficiency Kits[1406] | CFL Distribution[1407] | 59% | 13% | 11% | 83% |
| | School Kits[1408] | 61% | 13% | 11% | 86% |
| | Direct Mail Kits[1409] | 66% | 14% | 12% | 93% |

Leakage = Adjustment to account for the percentage of bulbs purchased that move out (and in if deemed appropriate) of the Utility Jurisdiction.

Upstream (TOS) Lighting programs = Determined through evaluation[1410].

All other programs = 0

Hours = Average hours of use per year

[1403] 1st year in service rate is based upon analysis of ComEd PY7 intercept data.

[1404] The 98% Lifetime ISR assumption is based upon the standard CFL measure in the absence of any better reference. This value is based upon review of two evaluations:
'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings.

[1405] Based upon Standard CFL assumption in the absence of better data, and is based upon review of the PY2 and PY3 ComEd Direct Install program surveys. This value includes bulb failures in the 1st year to be consistent with the Commission approval of annualization of savings for first year savings claims. ComEd PY2 All Electric Single Family Home Energy Performance Tune-Up Program Evaluation, Navigant Consulting, December 21, 2010.
http://www.icc.illinois.gov/downloads/public/edocket/287090.pdf.

[1406] In Service Rates provided are for the bulb within a kit only. Given the significant differences in program design and the level of education provided through Efficiency Kits programs, the evaluators should apply the ISR estimated through evaluations (either past evaluations or the current program year evaluation) of the specific Efficiency Kits program. In cases where program-specific evaluation results for an ISR are unavailable, the default ISR values for Efficiency Kits provide may be used.

[1407] Free bulbs provided without request, with little or no education. Consistent with Standard CFL assumptions.

[1408] Kits provided free to students through school, with education program. Consistent with Standard CFL assumptions.

[1409] Opt-in program to receive kits via mail, with little or no education. Consistent with Standard CFL assumptions.

[1410] Using a leakage estimate from the current program year evaluation, from past evaluation results, or a rolling average of leakage estimates from previous years.

| Installation Location | Hours[1411] |
|---|---|
| Residential and in-unit Multi Family | 759 |
| Exterior | 2475 |
| Unknown | 847 |

WHFe = Waste heat factor for energy to account for cooling energy savings from efficient lighting

| Bulb Location | WHFe |
|---|---|
| Interior single family or unknown location | 1.06 [1412] |
| Multi family in unit | 1.04 [1413] |
| Exterior or uncooled location | 1.0 |

**Mid Life Baseline Adjustment**

During the lifetime of a standard Omnidirectional LED, the baseline incandescent/halogen bulb would need to be replaced multiple times. Since the baseline bulb changes over time (except for <300 and 2600+ lumen lamps) the annual savings claim must be reduced within the life of the measure to account for this baseline shift.

For example, for 60W equivalent bulbs installed in 2014, the full savings (as calculated above in the Algorithm) should be claimed for the first six years, but a reduced annual savings (calculated energy savings above multiplied by the adjustment factor in the table below) claimed for the remainder of the measure life.

[1411] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

[1412] The value is estimated at 1.06 (calculated as 1 + (0.66*(0.27 / 2.8)). Based on cooling loads decreasing by 27% of the lighting savings (average result from REMRate modeling of several different configurations and IL locations of homes), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm (-0.02 * SEER2) + (1.12 * SEER) (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP) and 66% of homes in Illinois having central cooling ("Table HC7.9 Air Conditioning in Homes in Midwest Region, Divisions, and States, 2009 from Energy Information Administration", 2009 Residential Energy Consumption Survey; http://www.eia.gov/consumption/residential/data/2009/xls/HC7.9%20Air%20Conditioning%20in%20Midwest%20Region.xls )

[1413] As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average); http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls

| Minimum Lumens | Maximum Lumens | LED Wattage (WattsEE) | Delta Watts 2014-2019 (WattsEE) | Delta Watts Post 2020 (WattsEE) | Mid Life adjustment (made from June 2020) to first year savings |
|---|---|---|---|---|---|
| 1490 | 2600 | 37.2 | 34.8 | 8.3 | 23.8% |
| 1050 | 1489 | 23.1 | 29.9 | 5.1 | 17.1% |
| 750 | 1049 | 16.4 | 26.6 | 3.6 | 13.5% |
| 310 | 749 | 9.6 | 19.4 | 2.1 | 10.8% |

For example, an 8W LED lamp, 450 lumens, is installed in the interior of a home in 2014. The customer purchased the lamp through an upstream program:

$$\Delta kWH = ((29-8/1000) * 847 * 1.06 * 0.92$$

$$= 17.3 \text{ kWh}$$

This value should be claimed for six years, i.e. June 2014 – May 2020, but from May 2020 until the end of the measure life for that same bulb, savings should be reduced to (17.3 * 0.108 =) 1.9 kWh for the remainder of the measure life. Note these adjustments should be applied to kW and fuel impacts as well.

### DEFERRED INSTALLS

As presented above, the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

| | |
|---|---|
| Year 1 (Purchase Year) installs: | Characterized using assumptions provided above or evaluated assumptions if available. |
| Year 2 and 3 installs: | Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.<br><br>The NTG factor for the Purchase Year should be applied. |

Using the example from above, for an 8W LED, 450 Lumens purchased for the interior of a residential homes through an upstream program in 2014.

$$\Delta kWH_{1st\ year\ installs} = ((29-8/1000)*847*1.06*0.92$$

$$= 17.3 \text{ kWh}$$

$$\Delta kWH_{2nd\ year\ installs} = ((29-8/1000)*847*1.06*0.032$$

$$= 0.6 \text{ kWh}$$

Note: Here we assume no change in hours assumption. NTG value from Purchase year applied.

### HEATING PENALTY

If electric heated home (if heating fuel is unknown assume gas, see Natural Gas section):

$$\Delta kWh^{1414} = -(((WattsBase - WattsEE) / 1000) * ISR * Hours * HF) / \eta Heat$$

Where:

HF = Heating Factor or percentage of light savings that must be heated

= 49%[1415] for interior or unknown location

= 0% for exterior or unheated location

ηHeat = Efficiency in COP of Heating equipment

= actual. If not available use[1416]:

| System Type | Age of Equipment | HSPF Estimate | ηHeat (COP Estimate) |
|---|---|---|---|
| Heat Pump | Before 2006 | 6.8 | 2.00 |
| | After 2006 - 2014 | 7.7 | 2.26 |
| | 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1.00 |

Using the same 8 W LED that is installed in home with 2.0 COP Heat Pump (i.e., the heat pump was installed prior to 2006):

$\Delta kWh_{1st\ year}$ = - (((29-8) / 1000) * 0.92 * 759 * 0.49) / 2.0

= - 3.6 kWh

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year. The appropriate baseline shift adjustment should then be applied to all installs.

[1414] Negative value because this is an increase in heating consumption due to the efficient lighting.

[1415] This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

[1416] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

### SUMMER COINCIDENT PEAK DEMAND SAVINGS

ΔkW = ((WattsBase - WattsEE) / 1 000) * ISR * WHFd * CF

Where:

WHFd = Waste heat factor for demand to account for cooling savings from efficient lighting.

| Bulb Location | WHFd |
|---|---|
| Interior single family or unknown location | 1.11[1417] |
| Multi family in unit | 1.07[1418] |
| Exterior or uncooled location | 1.0 |

CF = Summer Peak Coincidence Factor for measure.

| Bulb Location | CF[1419] |
|---|---|
| Interior single family or unknown location or Multi family in unit | 7.1% |
| Exterior | 27.3% |
| Unknown | 8.1% |

Other factors as defined above

For the same 8 W LED that is installed in a single family interior location in 2014, the demand savings are:

ΔkW = ((29-8) / 1000) * 0.92* 1.11 * 0.071

= 0.0015 kW

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year. The appropriate baseline shift adjustment should then be applied to all installs.

[1417] The value is estimated at 1.11 (calculated as 1 + (0.66 * 0.466 / 2.8)). See footnote relating to WHFe for details. Note the 46.6% factor represents the average Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load.

[1418] As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average); http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls.

[1419] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluations.

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

Bulb replacement costs assumed in the O&M calculations are provided below[1420].

| | Std Inc. | EISA Compliant Halogen | CFL | LED-A |
|---|---|---|---|---|
| 2014 | $0.34 | $1.25 | $2.50 | $13.81 |
| 2015 | $0.34 | $0.90 | $2.50 | $10.86 |
| 2016 | $0.34 | $0.80 | $2.50 | $8.60 |
| 2017 | $0.34 | $0.70 | $2.50 | $7.74 |
| 2018 | $0.34 | $0.60 | $2.50 | $6.96 |
| 2019 | $0.34 | $0.60 | $2.50 | $6.27 |
| 2020 & after | $0.34 | N/A | $2.50 | $5.64 |

In order to account for the shift in baseline due to the Energy Independence and Security Act of 2007, an equivalent annual levelized baseline replacement cost over the lifetime of the LED bulb is calculated. The key assumptions used in this calculation are documented below:

| Installation Location | Omnidirectional LED Measure Hours | Hours of Use per year [1421] | Measure Life in Years (capped at 10) |
|---|---|---|---|
| Residential and in-unit Multi Family | 25,000 | 759 | 10 |
| Exterior | 25,000 | 2475 | 10 |
| Unknown | 25,000 | 847 | 10 |

The NPV for replacement lamps and annual levelized replacement costs using the statewide real discount rate of 5.23% are presented below:

[1420] Based upon pricing forecast developed by Applied Proactive Technologies Inc (APT) based on industry input and provided to Ameren.
[1421] Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluations.

| Location | Lumen Level | NPV of replacement costs for period | | | Levelized annual replacement cost savings | | |
|---|---|---|---|---|---|---|---|
| | | June 2015 - May 2016 | June 2016 - May 2017 | June 2017 - May 2018 | June 2015 - May 2016 | June 2016 - May 2017 | June 2017 - May 2018 |
| Residential and in-unit Multi Family | Lumens <310 or >2600 (non-EISA compliant) | $1.73 | $1.73 | $1.73 | $0.23 | $0.23 | $0.23 |
| | Lumens ≥ 310 and ≤ 2600 (EISA compliant) | $2.52 | $2.22 | $1.97 | $0.33 | $0.29 | $0.26 |
| Exterior | Lumens <310 or >2600 (non-EISA compliant) | $6.10 | $6.10 | $6.10 | $0.80 | $0.80 | $0.80 |
| | Lumens ≥ 310 and ≤ 2600 (EISA compliant) | $9.48 | $8.35 | $7.55 | $1.24 | $1.09 | $0.99 |
| Unknown | Lumens <310 or >2600 (non-EISA compliant) | $1.93 | $1.93 | $1.93 | $0.25 | $0.25 | $0.25 |
| | Lumens ≥ 310 and ≤ 2600 (EISA compliant) | $2.81 | $2.47 | $2.20 | $0.37 | $0.32 | $0.29 |

Note incandescent lamps in lumen range <310 and >2600 are exempt from EISA. For halogen bulbs, we assume the same replacement cycle as incandescent bulbs.[1422] The replacement cycle is based on the location of the lamp and varies based on the hours of use for that location. Both incandescent and halogen lamps are assumed to last for 1,000 hours before needing replacement.

**MEASURE CODE: RS-LTG-LEDA-V03-150601**

[1422] The manufacturers of the new minimally compliant EISA Halogens are using regular incandescent lamps with halogen fill gas rather than halogen infrared to meet the standard and so the component rated life is equal to the standard incandescent.

# 5.6 Shell End Use

## 5.6.1 Air Sealing

### DESCRIPTION

Thermal shell air leaks are sealed through strategic use and location of air-tight materials. Leaks are detected and leakage rates measured with the assistance of a blower-door. The algorithm for this measure can be used when the program implementation does not allow for more detailed forecasting through the use of residential modeling software.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

Air sealing materials and diagnostic testing should meet all eligibility program qualification criteria. The initial and final tested leakage rates should be performed in such a manner that the identified reductions can be properly discerned, particularly in situations wherein multiple building envelope measures may be implemented simultaneously.

### DEFINITION OF BASELINE EQUIPMENT

The existing air leakage should be determined through approved and appropriate test methods using a blower door. The baseline condition of a building upon first inspection significantly impacts the opportunity for cost-effective energy savings through air-sealing.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years.[1423]

### DEEMED MEASURE COST

The actual capital cost for this measure should be used in screening.

### LOADSHAPE

Loadshape R08 - Residential Cooling

Loadshape R09 - Residential Electric Space Heat

Loadshape R10 - Residential Electric Heating and Cooling

### COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during utility peak hour)

[1423] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

= 68%[1424]

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)

= 72%%[1425]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)

= 46.6%[1426]

**Algorithm**

**CALCULATION OF SAVINGS**

**ELECTRIC ENERGY SAVINGS**

ΔkWh = ΔkWh_cooling + ΔkWh_heating

Where:

ΔkWh_cooling = If central cooling, reduction in annual cooling requirement due to air sealing

= [(((CFM50_existing - CFM50_new)/N_cool) * 60 * 24 * CDD * DUA * 0.018) / (1000 * ηCool)] * LM

CFM50_existing = Infiltration at 50 Pascals as measured by blower door before air sealing.

= Actual

CFM50_new = Infiltration at 50 Pascals as measured by blower door after air sealing.

= Actual

N_cool = Conversion factor from leakage at 50 Pascal to leakage at natural conditions

=Dependent on exposure:[1427]

| Climate Zone | Exposure | N-Factor |
|---|---|---|
| Zone 2 | Well Shielded | 22.2 |
| | Normal | 18.5 |
| | Exposed | 16.7 |
| Zone 3 | Well Shielded | 25.8 |
| | Normal | 21.5 |

[1424] Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.
[1425] Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.
[1426] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.
[1427] N-factor is used to convert 50-pascal blower door air flows to natural air flows and is dependent on geographic location and exposure of the home to wind (impacts of stack effect based on height of building will not be significant because of reduced delta T during the cooling season) , based on methodology developed by Lawrence Berkeley Laboratory (LBL). N-factor values copied from J. Krigger, C. Dorsi; "Residential Energy: Cost Savings and Comfort for Existing Buildings", p284.

| Climate Zone | Exposure | N-Factor |
|---|---|---|
| | Exposed | 19.4 |

60 * 24 = Converts Cubic Feet per Minute to Cubic Feet per Day

CDD = Cooling Degree Days

= Dependent on location[1428]:

| Climate Zone (City based upon) | CDD 65 |
|---|---|
| 1 (Rockford) | 820 |
| 2 (Chicago) | 842 |
| 3 (Springfield) | 1,108 |
| 4 (Belleville) | 1,570 |
| 5 (Marion) | 1,370 |

DUA = Discretionary Use Adjustment (reflects the fact that people do not always operate their AC when conditions may call for it).

= 0.75 [1429]

0.018 = Specific Heat Capacity of Air (Btu/ft$^3$*°F)

1000 = Converts Btu to kBtu

ηCool = Efficiency (SEER) of Air Conditioning equipment (kBtu/kWh)

= Actual (where it is possible to measure or reasonably estimate). If unknown assume the following[1430]:

| Age of Equipment | SEER Estimate |
|---|---|
| Before 2006 | 10 |
| 2006 - 2014 | 13 |
| Central AC After 1/1/2015 | 13 |
| Heat Pump After 1/1/2015 | 14 |

[1428] National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 65°F.

[1429] This factor's source is: Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research", p31.

[1430] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

LM = Latent multiplier to account for latent cooling demand

= dependent on location: [1431]

| Climate Zone (City based upon) | LM |
|---|---|
| 1 (Rockford) | 8.5 |
| 2 (Chicago) | 6.2 |
| 3 (Springfield) | 6.6 |
| 4 (St. Louis, MO) | 5.8 |
| 5 (Evansville, IN) | 6.6 |

ΔkWh_heating = If electric heat (resistance or heat pump), reduction in annual electric heating due to air sealing

= (((CFM50_existing - CFM50_new)/N_heat) * 60 * 24 * HDD * 0.018) / (ηHeat * 3,412)

N_heat = Conversion factor from leakage at 50 Pascal to leakage at natural conditions

= Based on climate zone, building height and exposure level:[1432]

| | # Stories: | 1 | 1.5 | 2 | 3 |
|---|---|---|---|---|---|
| Zone 2 | Well Shielded | 22.2 | 20.0 | 17.8 | 15.5 |
| | Normal | 18.5 | 16.7 | 14.8 | 13.0 |
| | Exposed | 16.7 | 15.0 | 13.3 | 11.7 |
| Zone 3 | Well Shielded | 25.8 | 23.2 | 20.6 | 18.1 |
| | Normal | 21.5 | 19.4 | 17.2 | 15.1 |
| | Exposed | 19.4 | 17.4 | 15.5 | 13.5 |

HDD = Heating Degree Days

= Dependent on location:[1433]

[1431] The Latent Multiplier is used to convert the sensible cooling savings calculated to a value representing sensible and latent cooling loads. The values are derived from Harriman et al "Dehumidification and Cooling Loads From Ventilation Air", ASHRAE Journal, by adding the latent and sensible loads to determine the total, then dividing the total by the sensible load. Where this specialized data was not available, a nearby city was chosen.

[1432] N-factor is used to convert 50-pascal blower door air flows to natural air flows and is dependent on geographic location, height of building (stack effect) and exposure of the home to wind, based on methodology developed by Lawrence Berkeley Laboratory (LBL). N-factor values copied from J. Krigger, C. Dorsi; “Residential Energy: Cost Savings and Comfort for Existing Buildings”, p284.

[1433] National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 65°F. The base temperature was selected to account for the fact that homes receiving airsealing efforts are likely to be more leaky homes where the inside and outside air temperature is more consistent and therefore is more likely to require heating as temperatures drop below 65 degrees. Using this base temperature also reconciles the resulting savings estimates with the results of more sophisticated

| Climate Zone (City based upon) | HDD 65 |
|---|---|
| 1 (Rockford) | 6,569 |
| 2 (Chicago) | 6,339 |
| 3 (Springfield) | 5,497 |
| 4 (Belleville) | 4,379 |
| 5 (Marion) | 4,476 |

ηHeat = Efficiency of heating system

= Actual. If not available refer to default table below[1434]:

| System Type | Age of Equipment | HSPF Estimate | ηHeat (Effective COP Estimate)= (HSPF/3.413)*0.85 |
|---|---|---|---|
| Heat Pump | Before 2006 | 6.8 | 1.7 |
| | 2006 - 2014 | 7.7 | 1.92 |
| | 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1 |

3412 = Converts Btu to kWh

For example, a well shielded, 2 story single family home in Chicago with 10.5 SEER central cooling and a heat pump with COP of 2 (1.92 including distribution losses), has pre and post blower door test results of 3,400 and 2,250:

ΔkWh = ΔkWh_cooling + ΔkWh_heating

= [((((3,400 – 2,250) / 22.2) * 60 * 24 * 842 * 0.75 * 0.018) / (1000 * 10.5)) * 6.2] + [((3,400 – 2,250) / 17.8)) * 60 * 24 * 6339 * 0.018 / (1.92 * 3,412)]

= 501 + 1620

= 2,121 kWh

ΔkWh_heating = If gas *furnace* heat, kWh savings for reduction in fan run time

modeling software.

[1434] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.

= ΔTherms * $F_e$ * 29.3

$F_e$ = Furnace Fan energy consumption as a percentage of annual fuel consumption

= 3.14%[1435]

29.3 = kWh per therm

For example, a well shielded, 2 story single family home in Chicago with a gas furnace with system efficiency of 70%, has pre and post blower door test results of 3,400 and 2,250 (see therm calculation in Natural Gas Savings section:

ΔkWh = 152 * 0.0314 * 29.3

= 140 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = (ΔkWh_cooling / FLH_cooling) * CF

Where:

FLH_cooling = Full load hours of air conditioning

= Dependent on location[1436]:

| Climate Zone (City based upon) | Single Family | Multifamily |
|---|---|---|
| 1 (Rockford) | 512 | 467 |
| 2 (Chicago) | 570 | 506 |
| 3 (Springfield) | 730 | 663 |
| 4 (Belleville) | 1,035 | 940 |
| 5 (Marion) | 903 | 820 |

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)

= 68%[1437]

[1435] $F_e$ is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% $F_e$. See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

[1436] Full load hours for Chicago, Moline and Rockford are provided in "Final Evaluation Report: Central Air Conditioning Efficiency Services (CACES), 2010, Navigant Consulting", http://ilsag.org/yahoo_site_admin/assets/docs/ComEd_PY2_CACES_Evaluation_Report_2010-10-18.299122020.pdf p.33. An average FLH/Cooling Degree Day (from NCDC) ratio was calculated for these locations and applied to the CDD of the other locations in order to estimate FLH.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)

= 72%%[1438]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)

= 46.6%[1439]

Other factors as defined above

For example, a well shielded, 2 story single family home in Chicago with 10.5 SEER central cooling and a heat pump with COP of 2.0, has pre and post blower door test results of 3,400 and 2,250:

$\Delta kW_{SSP}$ = 501 / 570 * 0.68

= 0.60 kW

$\Delta kW_{PJM}$ = 501 / 570 * 0.466

= 0.410 kW

**NATURAL GAS SAVINGS**

If Natural Gas heating:

ΔTherms = (((CFM50_existing - CFM50_new)/N_heat) * 60 * 24 * HDD * 0.018) / (ηHeat * 100,000)

Where:

N_heat = Conversion factor from leakage at 50 Pascal to leakage at natural conditions

= Based on climate zone, building height and exposure level[1440]:

| | # Stories: | 1 | 1.5 | 2 | 3 |
|---|---|---|---|---|---|
| Zone 2 | Well Shielded | 22.2 | 20.0 | 17.8 | 15.5 |
| | Normal | 18.5 | 16.7 | 14.8 | 13.0 |
| | Exposed | 16.7 | 15.0 | 13.3 | 11.7 |
| Zone 3 | Well Shielded | 25.8 | 23.2 | 20.6 | 18.1 |
| | Normal | 21.5 | 19.4 | 17.2 | 15.1 |
| | Exposed | 19.4 | 17.4 | 15.5 | 13.5 |

HDD = Heating Degree Days

[1437] Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.
[1438] Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.
[1439] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.
[1440] N-factor is used to convert 50-pascal blower door air flows to natural air flows and is dependent on geographic location, height of building (stack effect) and exposure of the home to wind, based on methodology developed by Lawrence Berkeley Laboratory (LBL). N-factor values copied from J. Krigger, C. Dorsi; "Residential Energy: Cost Savings and Comfort for Existing Buildings", p284.

= dependent on location[1441]:

| Climate Zone (City based upon) | HDD 65 |
|---|---|
| 1 (Rockford) | 6,569 |
| 2 (Chicago) | 6,339 |
| 3 (Springfield) | 5,497 |
| 4 (Belleville) | 4,379 |
| 5 (Marion) | 4,476 |

ηHeat = Efficiency of heating system

= Equipment efficiency * distribution efficiency

= Actual[1442]. If not available use 70%[1443].

Other factors as defined above

For example, a well shielded, 2 story single family home in Chicago with a gas furnace with system efficiency of 70%, has pre and post blower door test results of 3,400 and 2,250:

ΔTherms = ((3,400 – 2,250)/17.8) * 60 * 24 * 6339 * 0.018) / (0.7 * 100,000)

= 152 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

[1441] National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 60°F, consistent with the findings of Belzer and Cort, Pacific Northwest National Laboratory in "Statistical Analysis of Historical State-Level Residential Energy Consumption Trends," 2004..

[1442] Ideally, the System Efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The Distribution Efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute: (http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf or by performing duct blaster testing.

[1443] This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls ) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:
(0.24*0.92) + (0.76*0.8) * (1-0.15) = 0.70

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-SHL-AIRS-V03-150601**

## 5.6.2 Basement Sidewall Insulation

### DESCRIPTION

Insulation is added to a basement or crawl space. Insulation added above ground in conditioned space is modeled the same as wall insulation. Below ground insulation is adjusted with an approximation of the thermal resistance of the ground. Insulation in unconditioned spaces is modeled by reducing the degree days to reflect the smaller but non-zero contribution to heating and cooling load. Cooling savings only consider above grade insulation, as below grade has little temperature difference during the cooling season.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

This measure requires a member of the implementation staff or a participating contractor to evaluate the pre and post R-values and measure surface areas. The requirements for participation in the program will be defined by the utilities.

### DEFINITION OF BASELINE EQUIPMENT

The existing condition will be evaluated by implementation staff or a participating contractor and is likely to be no basement wall or ceiling insulation.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 25 years.[1444]

### DEEMED MEASURE COST

The actual installed cost for this measure should be used in screening.

### DEEMED O&M COST ADJUSTMENTS

N/A

### LOADSHAPE

Loadshape R08 - Residential Cooling

Loadshape R09 - Residential Electric Space Heat

Loadshape R10 - Residential Electric Heating and Cooling

### COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

[1444] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during utility peak hour)

= 68%[1445]

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)

= 72%%[1446]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)

= 46.6%[1447]

## Algorithm

### CALCULATION OF SAVINGS

### ELECTRIC ENERGY SAVINGS

Where available savings from shell insulation measures should be determined through a custom analysis. When that is not feasible for the program the following engineering algorithms can be used with the inclusion of an adjustment factor to de-rate the heating savings.

ΔkWh = (ΔkWh_cooling + ΔkWh_heating)

Where:

ΔkWh_cooling = If central cooling, reduction in annual cooling requirement due to insulation

= (((1/R_old_AG - 1/(R_added+R_old_AG)) * L_basement_wall_total * H_basement_wall_AG * (1-Framing_factor)) * 24 * CDD * DUA) / (1000 * ηCool))

R_added = R-value of additional spray foam, rigid foam, or cavity insulation.

R_old_AG = R-value value of foundation wall above grade.

= Actual, if unknown assume 1.0[1448]

L_basement_wall_total = Length of basement wall around the entire insulated perimeter (ft)

H_basement_wall_AG = Height of insulated basement wall above grade (ft)

Framing_factor = Adjustment to account for area of framing when cavity insulation is used

[1445] Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.
[1446] Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.
[1447] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.
[1448] ORNL Builders Foundation Handbook, crawl space data from Table 5-5: Initial Effective R-values for Uninsulated Foundation System and Adjacent Soil, 1991, http://www.ornl.gov/sci/roofs+walls/foundation/ORNL_CON-295.pdf

= 0% if Spray Foam or External Rigid Foam

= 25% if studs and cavity insulation[1449]

24 = Converts hours to days

CDD = Cooling Degree Days

= Dependent on location and whether basement is conditioned:[1450]

| Climate Zone (City based upon) | Conditioned CDD 65 | Unconditioned CDD 65[1451] |
|---|---|---|
| 1 (Rockford) | 820 | 263 |
| 2 (Chicago) | 842 | 281 |
| 3 (Springfield) | 1,108 | 436 |
| 4 (Belleville) | 1,570 | 538 |
| 5 (Marion) | 1,370 | 570 |
| Weighted Average[1452] | 947 | 325 |

DUA = Discretionary Use Adjustment (reflects the fact that people do not always operate their AC when conditions may call for it).

= 0.75 [1453]

1000 = Converts Btu to kBtu

ηCool = Seasonal Energy Efficiency Ratio of cooling system (kBtu/kWh)

= Actual (where it is possible to measure or reasonably estimate). If unknown assume the following:[1454]

[1449] ASHRAE, 2001, "Characterization of Framing Factors for New Low-Rise Residential Building Envelopes (904-RP)," Table 7.1
[1450] National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 65°F. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.
[1451] Five year average cooling degree days with 75F base temp from DegreeDays.net were used in this table because the 30 year climate normals from NCDC used elsewhere are not available at base temps above 72F.
[1452] Weighted based on number of occupied residential housing units in each zone.
[1453] This factor's source is: Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research", p31.
[1454] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

| Age of Equipment | ηCool Estimate |
|---|---|
| Before 2006 | 10 |
| 2006 - 2014 | 13 |
| Central AC After 1/1/2015 | 13 |
| Heat Pump After 1/1/2015 | 14 |

ΔkWh_heating = If electric heat (resistance or heat pump), reduction in annual electric heating due to insulation

= ([((1/R_old_AG - 1/(R_added+R_old_AG)) * L_basement_wall_total * H_basement_wall_AG * (1-Framing_factor)) + ((1/(R_old_BG - 1/(R_added+R_old_BG)) * L_basement_wall_total * (H_basement_wall_total - H_basement_wall_AG) * (1-Framing_factor))] * 24 * HDD) / (3,412 * ηHeat)) * $ADJ_{Basement}$

Where

R_old_BG = R-value value of foundation wall below grade (including thermal resistance of the earth) [1455]

= dependent on depth of foundation (H_basement_wall_total – H_basement_wall_AG):

= Actual R-value of wall plus average earth R-value by depth in table below

| Below Grade R-value | | | | | | | | | |
|---|---|---|---|---|---|---|---|---|---|
| Depth below grade (ft) | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 |
| Earth R-value (°F-ft$^2$-h/Btu) | 2.44 | 4.50 | 6.30 | 8.40 | 10.44 | 12.66 | 14.49 | 17.00 | 20.00 |
| Average Earth R-value (°F-ft2-h/Btu) | 2.44 | 3.47 | 4.41 | 5.41 | 6.42 | 7.46 | 8.46 | 9.53 | 10.69 |
| Total BG R-value (earth + R-1.0 foundation) default | 3.44 | 4.47 | 5.41 | 6.41 | 7.42 | 8.46 | 9.46 | 10.53 | 11.69 |

H_basement_wall_total = Total height of basement wall (ft)

HDD = Heating Degree Days

[1455] Adapted from Table 1, page 24.4, of the 1977 ASHRAE Fundamentals Handbook

= dependent on location and whether basement is conditioned:[1456]

| Climate Zone (City based upon) | Conditioned HDD 60 | Unconditioned HDD 50 |
|---|---|---|
| 1 (Rockford) | 5,352 | 3,322 |
| 2 (Chicago) | 5,113 | 3,079 |
| 3 (Springfield) | 4,379 | 2,550 |
| 4 (Belleville) | 3,378 | 1,789 |
| 5 (Marion) | 3,438 | 1,796 |
| Weighted Average[1457] | 4,860 | 2,895 |

ηHeat = Efficiency of heating system

= Actual. If not available refer to default table below:[1458]

| System Type | Age of Equipment | HSPF Estimate | ηHeat (Effective COP Estimate) (HSPF/3.413)*0.85 |
|---|---|---|---|
| Heat Pump | Before 2006 | 6.8 | 1.7 |
| | After 2006 - 2014 | 7.7 | 1.92 |
| | 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1 |

$ADJ_{Basement}$ = Adjustment for basement wall insulation to account for prescriptive engineering algorithms overclaiming savings.

= 88%[1459]

[1456] National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 60°F for a conditioned basement and 50°F for an unconditioned basement), consistent with the findings of Belzer and Cort, Pacific Northwest National Laboratory in "Statistical Analysis of Historical State-Level Residential Energy Consumption Trends," 2004. There is a county mapping table in the front of the TRM providing the appropriate city to use for each county of Illinois.
[1457] Weighted based on number of occupied residential housing units in each zone.
[1458] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.
[1459] Based upon comparing algorithm derived savings estimate and evaluated bill analysis estimate in the following 2012

For example, a single family home in Chicago with a 20 by 25 by 7 foot R-2.25 basement, with 3 feet above grade, insulated with R-13 of interior spray foam, 10.5 SEER Central AC and 2.26 COP Heat Pump:

ΔkWh = (ΔkWh_cooling + ΔkWh_heating)

= [(((1/2.25 - 1/(13 + 2.25))*(20+25+20+25) * 3 * (1 - 0)) * 24 * 281 * 0.75)/(1000 * 10.5)] + [(((((1/2.25 - 1/(13 + 2.25)) * (20+25+20+25) * 3 * (1-0)) + ((1 / (2.25 + 6.42) – 1 / (13 + 2.25 + 6.42)) * (20+25+20+25) * 4 * (1-0))) * 24 * 3079) / (3412 * 1.92)) * 0.88]

= (49.3 + 1263.0)

= 1312.3 kWh

ΔkWh_heating = If gas *furnace* heat, kWh savings for reduction in fan run time

= ΔTherms * $F_e$ * 29.3

$F_e$ = Furnace Fan energy consumption as a percentage of annual fuel consumption

= 3.14%[1460]

29.3 = kWh per therm

For example, a single family home in Chicago with a 20 by 25 by 7 foot unconditioned basement, with 3 feet above grade, insulated with R-13 of interior spray foam, and a 70% efficient furnace (for therm calculation see Natural Gas Savings section :

= 118.1 * 0.0314 * 29.3

= 109 kWh

**SUMMER COINCIDENT PEAK DEMAND**

ΔkW = (ΔkWh_cooling / FLH_cooling) * CF

Where:

FLH_cooling = Full load hours of air conditioning

Massachusetts report: "Home Energy Services Impact Evaluation", August 2012. See "Insulation ADJ calculations.xls" for details or calculation.

[1460] $F_e$ is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% $F_e$. See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

= dependent on location[1461]:

| Climate Zone (City based upon) | Single Family | Multifamily |
|---|---|---|
| 1 (Rockford) | 512 | 467 |
| 2 (Chicago) | 570 | 506 |
| 3 (Springfield) | 730 | 663 |
| 4 (Belleville) | 1,035 | 940 |
| 5 (Marion) | 903 | 820 |
| Weighted Average[1462] | 629 | 564 |

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)

= 68%[1463]

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)

= 72%%[1464]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)

= 46.6%[1465]

For example, a single family home in Chicago with a 20 by 25 by 7 foot unconditioned basement, with 3 feet above grade, insulated with R-13 of interior spray foam, 10.5 SEER Central AC and 2.26 COP Heat Pump:

$\Delta kW_{SSP}$ = 49.3 / 570 * 0.68

= 0.059 kW

$\Delta kW_{PJM}$ = 49.3 / 570 * 0.466

= 0.040 kW

[1461] Full load hours for Chicago, Moline and Rockford are provided in "Final Evaluation Report: Central Air Conditioning Efficiency Services (CACES), 2010, Navigant Consulting", http://ilsagfiles.org/SAG_files/Evaluation_Documents/ComEd/ComEd%20EPY2%20Evaluation%20Reports/ComEd_Central_AC_Efficiency_Services_PY2_Evaluation_Report_Final.pdf, p.33. An average FLH/Cooling Degree Day (from NCDC) ratio was calculated for these locations and applied to the CDD of the other locations in order to estimate FLH. There is a county mapping table in the front of the TRM providing the appropriate city to use for each county of Illinois.

[1462] Weighted based on number of occupied residential housing units in each zone.

[1463] Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

[1464] Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

[1465] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

### NATURAL GAS SAVINGS

If Natural Gas heating:

ΔTherms = [((((1/R_old_AG - 1/(R_added+R_old_AG)) * L_basement_wall_total * H_basement_wall_AG * (1-Framing_factor) + (1/(R_old_BG - 1/(R_added+R_old_BG)) * L_basement_wall_total * (H_basement_wall_total - H_basement_wall_AG) * (1-Framing_factor)] * 24 * HDD) / (ηHeat * 100,067)] * $ADJ_{Basement}$

ηHeat = Efficiency of heating system

= Equipment efficiency * distribution efficiency

= Actual. If unknown assume 70%[1466]

Other factors as defined above

For example, a single family home in Chicago with a 20 by 25 by 7 foot R-2.25 basement, with 3 feet above grade, insulated with R-13 of interior spray foam, and a 70% efficient furnace:

= ((1/2.25 - 1/(13 + 2.25)) * (20+25+20+25) * 3 * (1-0) + (1/8.67 - 1/(13 + 8.67)) * (20+25+20+25) * 4 * (1 - 0)) * 24 * 3079) / (0.7 * 100,067) * 0.88

= 118.1 therms

### WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

### DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

### MEASURE CODE: RS-SHL-BINS-V06-150601

[1466] This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls)) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:
(0.24*0.92) + (0.76*0.8) * (1-0.15) = 0.70

## 5.6.3 Floor Insulation Above Crawlspace

### DESCRIPTION

Insulation is added to the floor above a vented crawl space that does not contain pipes or HVAC equipment. If there are pipes, HVAC, or a basement, it is desirable to keep them within the conditioned space by insulating the crawl space walls and ground. Insulating the floor separates the conditioned space above from the space below the floor, and is only acceptable when there is nothing underneath that could freeze or would operate less efficiently in an environment resembling the outdoors. Even in the case of an empty, unvented crawl space, it is still considered best practice to seal and insulate the crawl space perimeter rather than the floor. Not only is there generally less area to insulate this way, but there are also moisture control benefits. There is a "Basement Insulation" measure for perimeter sealing and insulation. This measure assumes the insulation is installed above an unvented crawl space and should not be used in other situations.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

This measure requires a member of the implementation staff or a participating contractor to evaluate the pre and post R-values and measure surface areas. The requirements for participation in the program will be defined by the utilities.

### DEFINITION OF BASELINE EQUIPMENT

The existing condition will be evaluated by implementation staff or a participating contractor and is likely to be no insulation on any surface surrounding a crawl space.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 25 years.[1467]

### DEEMED MEASURE COST

The actual installed cost for this measure should be used in screening.

### DEEMED O&M COST ADJUSTMENTS

N/A

### LOADSHAPE

Loadshape R08 - Residential Cooling

Loadshape R09 - Residential Electric Space Heat

Loadshape R10 - Residential Electric Heating and Cooling

### COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to

[1467] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during utility peak hour)

= 68%[1468]

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)

= 72%%[1469]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)

= 46.6%[1470]

## Algorithm

### CALCULATION OF SAVINGS

#### ELECTRIC ENERGY SAVINGS

Where available savings from shell insulation measures should be determined through a custom analysis. When that is not feasible for the program the following engineering algorithms can be used with the inclusion of an adjustment factor to de-rate the heating savings.

ΔkWh = (ΔkWh_cooling + ΔkWh_heating)

Where:

ΔkWh_cooling = If central cooling, reduction in annual cooling requirement due to insulation

= (((1/R_old - 1/(R_added+R_old)) * Area * (1-Framing_factor)) * 24 * CDD * DUA) / (1000 * ηCool))

R_old = R-value value of floor before insulation, assuming 3/4" plywood subfloor and carpet with pad

= Actual. If unknown assume 3.96 [1471]

R_added = R-value of additional spray foam, rigid foam, or cavity insulation.

Area = Total floor area to be insulated

Framing_factor = Adjustment to account for area of framing

= 12% [1472]

[1468] Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

[1469] Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

[1470] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

[1471] Based on 2005 ASHRAE Handbook – Fundamentals: assuming 2x8 joists, 16" OC, ¾" subfloor, ½" carpet with rubber pad, and accounting for a still air film above and below: 1/ [(0.85 cavity share of area / (0.68 + 0.94 + 1.23 + 0.68)) + (0.15 framing share / (0.68 + 7.5" * 1.25 R/in + 0.94 + 1.23 + 0.68))] = 3.96

[1472] ASHRAE, 2001, "Characterization of Framing Factors for New Low-Rise Residential Building Envelopes (904-RP)," Table 7.1

24 = Converts hours to days

CDD = Cooling Degree Days

| Climate Zone (City based upon) | Unconditioned CDD[1473] |
|---|---|
| 1 (Rockford) | 263 |
| 2 (Chicago) | 281 |
| 3 (Springfield) | 436 |
| 4 (Belleville) | 538 |
| 5 (Marion) | 570 |
| Weighted Average[1474] | 325 |

DUA = Discretionary Use Adjustment (reflects the fact that people do not always operate their AC when conditions may call for it).

= 0.75 [1475]

1000 = Converts Btu to kBtu

ηCool = Seasonal Energy Efficiency Ratio of cooling system (kBtu/kWh)

= Actual (where it is possible to measure or reasonably estimate). If unknown assume the following:[1476]

| Age of Equipment | ηCool Estimate |
|---|---|
| Before 2006 | 10 |
| 2006 - 2014 | 13 |
| Central AC After 1/1/2015 | 13 |
| Heat Pump After 1/1/2015 | 14 |

ΔkWh_heating = If electric heat (resistance or heat pump), reduction in annual electric heating due to insulation

= ((((1/R_old - 1/(R_added + R_old)) * Area * (1-Framing_factor) * 24 * HDD)/ (3,412 * ηHeat)) * $ADJ_{Floor}$

[1473] Five year average cooling degree days with 75F base temp from DegreeDays.net were used in this table because the 30 year climate normals from NCDC used elsewhere are not available at base temps above 72F.
[1474] Weighted based on number of occupied residential housing units in each zone.
[1475] Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research", p31.
[1476] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

HDD = Heating Degree Days:[1477]

| Climate Zone (City based upon) | Unconditioned HDD |
|---|---|
| 1 (Rockford) | 3,322 |
| 2 (Chicago) | 3,079 |
| 3 (Springfield) | 2,550 |
| 4 (Belleville) | 1,789 |
| 5 (Marion) | 1,796 |
| Weighted Average[1478] | 2,895 |

ηHeat = Efficiency of heating system

= Actual. If not available refer to default table below:[1479]

| System Type | Age of Equipment | HSPF Estimate | ηHeat (Effective COP Estimate) (HSPF/3.413)*0.85 |
|---|---|---|---|
| Heat Pump | Before 2006 | 6.8 | 1.7 |
| | 2006 - 2014 | 7.7 | 1.92 |
| | 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1 |

$ADJ_{Floor}$ = Adjustment for floor insulation to account for prescriptive engineering algorithms overclaiming savings.

= 88%[1480]

Other factors as defined above

[1477] National Climatic Data Center, Heating Degree Days with a base temp of 50°F to account for lower impact of unconditioned space on heating system. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.

[1478] Weighted based on number of occupied residential housing units in each zone.

[1479] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.

[1480] Based upon comparing algorithm derived savings estimate and evaluated bill analysis estimate in the following 2012 Massachusetts report: "Home Energy Services Impact Evaluation", August 2012. See "Insulation ADJ calculations.xls" for details or calculation. Note that basement wall is used as a proxy for crawlspace ceiling.

For example, a single family home in Chicago with a 20 by 25 footprint, insulated with R-30 spray foam above the crawlspace, a 10.5 SEER Central AC and a newer heat pump:

ΔkWh = (ΔkWh_cooling + ΔkWh_heating)

= (((1/3.96 -1/(30+3.96))*(20*25)*(1-0.12)* 24 * 281*0.75)/(1000*10.5) + (((1/3.96 - 1/(30+3.96))*(20*25)*(1-0.15) * 24 * 3079)/(3412*1.92)) * 0.88)

= (47.3 + 941.1)

= 988.4 kWh

ΔkWh_heating = If gas *furnace* heat, kWh savings for reduction in fan run time

= ΔTherms * $F_e$ * 29.3

$F_e$ = Furnace Fan energy consumption as a percentage of annual fuel consumption

= 3.14%[1481]

29.3 = kWh per therm

For example, a single family home in Chicago with a 20 by 25 footprint, insulated with R-30 spray foam above the crawlspace, and a 70% efficient furnace (for therm calculation see Natural Gas Savings section):

ΔkWh = 91.2 * 0.0314 * 29.3

= 83.9 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = (ΔkWh_cooling / FLH_cooling) * CF

Where:

FLH_cooling = Full load hours of air conditioning

= Dependent on location:[1482]

| Climate Zone (City based upon) | Single Family | Multifamily |
|---|---|---|
| 1 (Rockford) | 512 | 467 |
| 2 (Chicago) | 570 | 506 |

[1481] $F_e$ is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% $F_e$. See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

[1482] Full load hours for Chicago, Moline and Rockford are provided in "Final Evaluation Report: Central Air Conditioning Efficiency Services (CACES), 2010, Navigant Consulting", http://ilsag.org/yahoo_site_admin/assets/docs/ComEd_PY2_CACES_Evaluation_Report_2010-10-18.299122020.pdf, p.33. An average FLH/Cooling Degree Day (from NCDC) ratio was calculated for these locations and applied to the CDD of the other locations in order to estimate FLH. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.

| Climate Zone (City based upon) | Single Family | Multifamily |
|---|---|---|
| 3 (Springfield) | 730 | 663 |
| 4 (Belleville) | 1,035 | 940 |
| 5 (Marion) | 903 | 820 |
| Weighted Average[1483] | 629 | 564 |

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)

= 68%[1484]

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)

= 72%%[1485]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)

= 46.6%[1486]

For example, a single family home in Chicago with a 20 by 25 footprint, insulated with R-30 spray foam above the crawlspace, a 10.5 SEER Central AC and a newer heat pump:

$\Delta kW_{SSP}$ = 47.3 / 570 * 0.68

= 0.056 kW

$\Delta kW_{SSP}$ = 47.3 / 570 * 0.466

= 0.039 kW

**NATURAL GAS SAVINGS**

If Natural Gas heating:

ΔTherms = (1/R_old - 1/(R_added+R_old)) * Area * (1-Framing_factor)) * 24 * HDD) / (100,000 * ηHeat) * $ADJ_{FloorGasHeat}$

Where

[1483] Weighted based on number of occupied residential housing units in each zone.
[1484] Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.
[1485] Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.
[1486] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

ηHeat = Efficiency of heating system

= Equipment efficiency * distribution efficiency

= Actual. If unknown assume 70%[1487]

Other factors as defined above

For example, a single family home in Chicago with a 20 by 25 footprint, insulated with R-30 spray foam above the crawlspace, and a 70% efficient furnace:

ΔTherms = (1 / 3.96 – 1 /(30 + 3.96))*(20 * 25) * (1 - 0.12) * 24 * 3079) / (100,000 * 0.70) * 0.88

= 91.2 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

## MEASURE CODE: RS-SHL-FINS-V06-150601

[1487] This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls)) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:
(0.24*0.92) + (0.76*0.8) * (1-0.15) = 0.70

## 5.6.4 Wall and Ceiling/Attic Insulation

### DESCRIPTION

Insulation is added to wall cavities, and/or attic. This measure requires a member of the implementation staff evaluating the pre and post R-values and measure surface areas. The efficiency of the heating and cooling equipment in the home should also be evaluated if possible.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

### DEFINITION OF EFFICIENT EQUIPMENT

This measure requires a member of the implementation staff or a participating contractor to evaluate the pre and post R-values and measure surface areas. The requirements for participation in the program will be defined by the utilities.

### DEFINITION OF BASELINE EQUIPMENT

The existing condition will be evaluated by implementation staff or a participating contractor and is likely to be empty wall cavities and little or no attic insulation.

### DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 25 years.[1488]

### DEEMED MEASURE COST

The actual installed cost for this measure should be used in screening.

### LOADSHAPE

Loadshape R08 - Residential Cooling

Loadshape R09 - Residential Electric Space Heat

Loadshape R10 - Residential Electric Heating and Cooling

### COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during utility peak hour)

= 68%[1489]

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)

[1488] Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

[1489] Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

= 72%%[1490]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)

= 46.6%[1491]

**Algorithm**

**CALCULATION OF SAVINGS**

**ELECTRIC ENERGY SAVINGS**

Where available savings from shell insulation measures should be determined through a custom analysis. When that is not feasible for the program the following engineering algorithms can be used with the inclusion of an adjustment factor to de-rate the heating savings.

ΔkWh = (ΔkWh_cooling + ΔkWh_heating)

Where

ΔkWh_cooling = If central cooling, reduction in annual cooling requirement due to insulation

= [((1/R_old - 1/R_wall) * A_wall * (1-Framing_factor_wall) + (1/R_old - 1/R_attic) * A_attic * (1-Framing_factor_attic)) * 24 * CDD * DUA] / (1000 * ηCool)

R_wall = R-value of new wall assembly (including all layers between inside air and outside air).

R_attic = R-value of new attic assembly (including all layers between inside air and outside air).

R_old = R-value value of existing assemble and any existing insulation.

(Minimum of R-5 for uninsulated assemblies[1492])

A_wall = Net area of insulated wall ($ft^2$)

A_attic = Total area of insulated ceiling/attic ($ft^2$)

Framing_factor_wall = Adjustment to account for area of framing

= 25%[1493]

Framing_factor_attic = Adjustment to account for area of framing

= 7%[1494]

24 = Converts hours to days

CDD = Cooling Degree Days

[1490] Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.
[1491] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.
[1492] An estimate based on review of Madison Gas and Electric, Exterior Wall Insulation, R-value for no insulation in walls, and NREL's Building Energy Simulation Test for Existing Homes (BESTEST-EX).
[1493] ASHRAE, 2001, "Characterization of Framing Factors for New Low-Rise Residential Building Envelopes (904-RP)," Table 7.1
[1494] Ibid.

= dependent on location:[1495]

| Climate Zone (City based upon) | CDD 65 |
|---|---|
| 1 (Rockford) | 820 |
| 2 (Chicago) | 842 |
| 3 (Springfield) | 1,108 |
| 4 (Belleville) | 1,570 |
| 5 (Marion) | 1,370 |
| Weighted Average[1496] | 947 |

DUA = Discretionary Use Adjustment (reflects the fact that people do not always operate their AC when conditions may call for it).

= 0.75 [1497]

1000 = Converts Btu to kBtu

ηCool = Seasonal Energy Efficiency Ratio of cooling system (kBtu/kWh)

= Actual (where it is possible to measure or reasonably estimate). If unknown assume the following:[1498]

| Age of Equipment | ηCool Estimate |
|---|---|
| Before 2006 | 10 |
| 2006 - 2014 | 13 |
| Central AC After 1/1/2015 | 13 |
| Heat Pump After 1/1/2015 | 14 |

kWh_heating = If electric heat (resistance or heat pump), reduction in annual electric heating due to insulation

= (((1/R_old - 1/R_wall) * A_wall * (1-Framing_factor_wall) * $ADJ_{Wall}$ ) + (1/R_old - 1/R_attic) * A_attic * (1-Framing_factor_attic) * $ADJ_{Attic}$) * 24 * HDD] / (ηHeat * 3412)

[1495] National Climatic Data Center, Cooling Degree Days are based on a base temp of 65°F. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.
[1496] Weighted based on number of occupied residential housing units in each zone.
[1497] This factor's source is: Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research", p31.
[1498] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

HDD = Heating Degree Days

= Dependent on location:[1499]

| Climate Zone (City based upon) | HDD 60 |
|---|---|
| 1 (Rockford) | 5,352 |
| 2 (Chicago) | 5,113 |
| 3 (Springfield) | 4,379 |
| 4 (Belleville) | 3,378 |
| 5 (Marion) | 3,438 |
| Weighted Average[1500] | 4,860 |

ηHeat = Efficiency of heating system

= Actual. If not available refer to default table below:[1501]

| System Type | Age of Equipment | HSPF Estimate | ηHeat (Effective COP Estimate) (HSPF/3.413)*0.85 |
|---|---|---|---|
| Heat Pump | Before 2006 | 6.8 | 1.7 |
| | 2006 - 2014 | 7.7 | 1.92 |
| | 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1 |

3412 = Converts Btu to kWh

$ADJ_{Wall}$ = Adjustment for wall insulation to account for prescriptive engineering algorithms overclaiming savings.

= 63%[1502]

[1499] National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 60°F, consistent with the findings of Belzer and Cort, Pacific Northwest National Laboratory in "Statistical Analysis of Historical State-Level Residential Energy Consumption Trends," 2004. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.

[1500] Weighted based on number of occupied residential housing units in each zone.

[1501] These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.

[1502] Based upon comparing algorithm derived savings estimate and evaluated bill analysis estimate in the following 2012 Massachusetts report: "Home Energy Services Impact Evaluation", August 2012. See "Insulation ADJ calculations.xls" for details or calculation.

$ADJ_{Attic}$ = Adjustment for attic insulation to account for prescriptive engineering algorithms overclaiming savings.

= 74%[1503]

For example, a single family home in Chicago with 990 $ft^2$ of R-5 walls insulated to R-11 and 700 $ft^2$ of R-5 attic insulated to R-38, 10.5 SEER Central AC and 2.26 (1.92 including distribution losses) COP Heat Pump:

ΔkWh = (ΔkWh_cooling + ΔkWh_heating)

= ((((1/5 - 1/11) * 990 * (1-0.25)) + ((1/5 - 1/38) * 700 * (1-0.07))) * 842 * 0.75 * 24)/ (1000 * 10.5)) + (((((1/5 - 1/11) * 990 * (1-0.25) * 0.63) + ((1/5 - 1/38) * 700 * (1-0.07) * 0.74)) * 5113 * 24) / (1.92 * 3412))

= 280 + 2523

= 2803 kWh

ΔkWh_heating = If gas *furnace* heat, kWh savings for reduction in fan run time

= ΔTherms * $F_e$ * 29.3

$F_e$ = Furnace Fan energy consumption as a percentage of annual fuel consumption

= 3.14%[1504]

29.3 = kWh per therm

For example, a single family home in Chicago with 990 $ft^2$ of R-5 walls insulated to R-11 and 700 $ft^2$ of R-5 attic insulated to R-38, with a gas furnace with system efficiency of 66% (for therm calculation see Natural Gas Savings section):

ΔkWh = 250.3 * 0.0314 * 29.3

= 230 kWh

**SUMMER COINCIDENT PEAK DEMAND SAVINGS**

ΔkW = (ΔkWh_cooling / FLH_cooling) * CF

Where:

[1503] Based upon comparing algorithm derived savings estimate and evaluated bill analysis estimate in the following 2012 Massachusetts report: "Home Energy Services Impact Evaluation", August 2012. See "Insulation ADJ calculations.xls" for details or calculation. Note that basement walls is used as a proxy for crawlspace ceiling.

[1504] $F_e$ is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% $F_e$. See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

FLH_cooling = Full load hours of air conditioning

= Dependent on location as below:[1505]

| Climate Zone (City based upon) | Single Family | Multifamily |
|---|---|---|
| 1 (Rockford) | 512 | 467 |
| 2 (Chicago) | 570 | 506 |
| 3 (Springfield) | 730 | 663 |
| 4 (Belleville) | 1,035 | 940 |
| 5 (Marion) | 903 | 820 |
| Weighted Average[1506] | 629 | 564 |

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)

= 68%[1507]

$CF_{SSP}$ = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)

72%%[1508]

$CF_{PJM}$ = PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)

= 46.6%[1509]

For example, a single family home in Chicago with 990 $ft^2$ of R-5 walls insulated to R-11 and 700 $ft^2$ of R-5 attic insulated to R-38, 10.5SEER Central AC and 2.26 COP Heat Pump:

$\Delta kW_{SSP}$ = 280 / 570 * 0.68

= 0.33 kW

$\Delta kW_{PJM}$ = 280 / 570 * 0.466

= 0.23 kW

[1505] Based on Full Load Hours from ENERGY Star with adjustments made in a Navigant Evaluation, other cities were scaled using those results and CDD. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.

[1506] Weighted based on number of occupied residential housing units in each zone.

[1507] Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

[1508] Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

[1509] Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

**NATURAL GAS SAVINGS**

If Natural Gas heating:

$$\Delta Therms = ((((1/R_old - 1/R_wall) * A_wall * (1-Framing_factor_wall) * ADJ_{Wall}) + ((1/R_old - 1/R_attic) * A_attic * (1-Framing_factor_attic) * ADJ_{Attic})) * 24 * HDD) / (\eta Heat * 100{,}067\ Btu/therm)$$

Where:

HDD = Heating Degree Days

= Dependent on location:[1510]

| Climate Zone (City based upon) | HDD 60 |
|---|---|
| 1 (Rockford) | 5,352 |
| 2 (Chicago) | 5,113 |
| 3 (Springfield) | 4,379 |
| 4 (Belleville) | 3,378 |
| 5 (Marion) | 3,438 |
| Weighted Average[1511] | 4,860 |

ηHeat = Efficiency of heating system

= Equipment efficiency * distribution efficiency

= Actual.[1512] If unknown assume 70%.[1513]

Other factors as defined above

[1510] National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 60°F, consistent with the findings of Belzer and Cort, Pacific Northwest National Laboratory in "Statistical Analysis of Historical State-Level Residential Energy Consumption Trends," 2004. There is a county mapping table in the Appendix providing the appropriate city to use for each county of Illinois.

[1511] Weighted based on number of occupied residential housing units in each zone.

[1512] Ideally, the System Efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The Distribution Efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute: (http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf) or by performing duct blaster testing.

[1513] This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls)). In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:
(0.24*0.92) + (0.76*0.8) * (1-0.15) = 0.70

For example, a single family home in Chicago with 990 $ft^2$ of R-5 walls insulated to R-11 and 700 $ft^2$ of R-5 attic insulated to R-38, with a gas furnace with system efficiency of 66%:

ΔTherms = ((((1/5 - 1/11) * 990 * (1-0.25) *0.63) + ((1/5 - 1/38) * 700 * (1-0.07)*0.74)) * 24 * 5113) / (0.66 * 100,067)

= 250.3 therms

**WATER IMPACT DESCRIPTIONS AND CALCULATION**

N/A

**DEEMED O&M COST ADJUSTMENT CALCULATION**

N/A

**MEASURE CODE: RS-SHL-AINS-V05-150201**

**Illinois Statewide**

**Technical Reference Manual**

**for Energy Efficiency**

# Attachment A

# Illinois Statewide

# Net-to-Gross

# Methodologies

**February 13, 2015**

**FINAL**

**Effective for Evaluation:**

**June 1st, 2015**

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# I. Illinois Statewide Net-to-Gross Methodologies

## A. Policy Context for this Information

The Illinois Evaluation Teams (Opinion Dynamics, Cadmus Group, Navigant Consulting, Itron, and ADM Associates) are working with the Illinois Stakeholder Advisory Group (SAG) to create an Illinois Statewide Net-to-Gross (NTG) Methodologies document (IL-NTG Methods). The IL-NTG Methods document is included as an attachment to the Illinois Statewide Technical Reference Manual for Energy Efficiency (IL-TRM). Through five different dockets, the Illinois Commerce Commission (ICC) has directed the Evaluation Teams to compile and formalize standard NTG methods for use in Illinois energy-efficiency (EE) evaluation, measurement and verification (EM&V) work. The ICC EE dockets are shown in the following table.

**Table 9. ICC Energy Efficiency Dockets**

| ICC Order Docket No. and Date | Program Administrator | NTG Discussion – Order Pages | ICC Link |
|---|---|---|---|
| **13-0495 (1/28/14)** | Commonwealth Edison Company (ComEd) | 129-130 | ICC Order Docket No. 13-0495 |
| **13-0498 (1/28/14)** | Ameren Illinois Company (Ameren) | 167, 171 | ICC Order Docket No. 13-0498 |
| **13-0499 (1/28/14)** | Illinois Department of Commerce and Economic Opportunity (DCEO) | 20, 23, 49 | ICC Order Docket No. 13-0499 |
| **13-0549 (5/20/14)** | Nicor Gas Company (Nicor) | 41-42, 78 | ICC Order Docket No. 13-0549 |
| **13-0550 (5/20/14)** | North Shore Gas Company (North Shore Gas) and The Peoples Gas Light and Coke Company (Peoples Gas) (collectively, PG&NSG or Integrys) | 54-55, 66 | ICC Order Docket No. 13-0550 |

To provide clarity to the ICC directives, the relevant section on IL-NTG Methods is shown in its entirety from the Nicor Gas Order (Docket No. 13-0549). The Nicor Gas Order provides the most detail on the ICC NTG directive in comparison to the other EE orders. The Nicor language is as follows:

> The Commission believes that Staff's recommendations concerning Commission adoption of consistent statewide net-to-gross methodologies ("IL-NTG Methods") for use by the evaluators are reasonable and will aid in future evaluation of the energy efficiency programs. To help ensure the independence of the evaluators, to improve efficiency in the evaluation process, and to ensure programs across the state as delivered by the various program administrators can be meaningfully and consistently evaluated, the Commission hereby adopts Staff's recommendation that consistent IL-NTG Methods be established for use in the evaluations of comparable energy efficiency programs offered by different Illinois program administrators. The Commission notes that Section 8-104(k) of the Act encourages statewide coordination and consistency between the gas and electric energy efficiency programs and Staff's proposal would help ensure consistency in the evaluation of program performance. The Commission notes that this directive is not to create entirely "new" NTG methodologies for every energy efficiency program, but rather to assess NTG methodologies and survey instruments that have been used to evaluate energy efficiency programs offered in Illinois, and to compile the most

justifiable and well-vetted methodologies (or potentially combine certain components from the existing approaches to better represent the most justifiable and well-vetted method consistent with best practices) in an attachment to the Updated IL-TRM that would get submitted to the Commission for approval. The Commission notes that the IL-NTG Methods will be flexible and adaptable to multiple program designs and budgets and tailored to appropriately assess the specifics of each of the program administrators' energy efficiency programs, consistent with standard NTG methodologies adopted in other states that were filed in this proceeding. The Commission agrees with Staff that in the interest of efficiency, the current program evaluators should take the lead in compiling and formalizing standard methodologies for NTG in Illinois taking into consideration SAG input. Because the existing Plan 1 evaluators are under contract with the Company for the evaluation of the program year three energy efficiency programs, it is appropriate for these existing evaluators to work on and complete the compilation of the IL-NTG Methods over the next year. The Commission recognizes that each year considerable time may be spent vetting NTG methodologies for each program evaluation separately for each utility under the existing evaluation plan review practices; adoption of IL-NTG Methods would save on these limited evaluation resources by having a common reference document for the evaluators to use in estimating net savings for Illinois.

The Commission hereby directs the Company to require its evaluators to collaborate with the other Illinois evaluators and the SAG to use best efforts to reach consensus on the approaches used in assessing NTG in particular markets for both residential and non-residential energy efficiency programs in a manner consistent with the direction described herein. (Pages 41-42)

(16) Northern Illinois Gas Company shall require its evaluators to collaborate with the other Illinois evaluators and the SAG to reach consensus on the most defensible and well-vetted methodologies for assessing net-to-gross ratios in particular markets for both residential and non-residential energy efficiency programs in a manner consistent with the direction provided herein;

(17) ICC Staff shall file the agreed-upon consensus statewide NTG methodologies with the Commission as an attachment to the Updated IL-TRM, and if consensus is not reached on a certain component of the statewide NTG methodologies, that particular non-consensus component should be submitted in a manner consistent with the approach used for non-consensus IL-TRM Updates; (Page 78)

### B. Programs Currently Covered in this Document

This document will be updated over time to cover a range of programs. To facilitate completion of part of the IL-NTG Methods sections prior to March 1, 2015, this document includes methods specific for three program types: 1) Commercial, Industrial, and Public Sector Standard/Prescriptive and Custom programs, 2) Appliance Recycling programs, and 3) Residential Upstream Lighting programs. All NTG data collection and analysis activities occurring after the effective date, June 1, 2015, for the program types covered by this document, shall conform to the NTG methods set forth herein.

### C. Updating the IL-NTG Methods

This attachment is part of the IL-TRM and follows the timeline for updating of the IL-TRM as specified in the IL-TRM Policy Document.[1514] In general, the following will take place:

- Updates will occur annually.
- Any changes to the IL-NTG Methods document will be circulated to the full SAG and SAG participants will have a ten business day review process.
- Updates will be discussed within the SAG and completed by March 1st.
- The ICC Staff will then submit a Staff Report (with the consensus Updated TRM attached) to the Commission with a request for expedited review and approval.

### D. Diverging from the IL-NTG Methods

The NTG methods for the programs outlined in this document are partially binding. The criteria for deviating from the IL-NTG Methods document are set forth below. In all cases, the evaluators (or any interested stakeholder) submits the proposed deviation to the full SAG for a ten business day SAG review and comment period. In the event of an objection by a SAG participant, efforts may be made to see if consensus can be reached on the proposed deviation in a subsequent monthly SAG meeting. In this case, a final opportunity for SAG review and comment to the proposed deviation will be provided following the SAG meeting.

Evaluators may modify the approaches described in this document if the following three conditions have been satisfied:

1. Evaluators must explicate within the annual evaluation research plan (or other document) how specific items in the proposed modified NTG method will diverge from what is written in this document. Evaluators must justify why the divergence is appropriate.
2. Prior to the use of the modified NTG method for a particular program, evaluation teams must be in agreement on the use and execution of the modified NTG method.
3. No objection from SAG participants is received regarding the proposed modified NTG method within a ten business day SAG review and comment period.

Evaluators may test alternative methods of estimating NTG for a particular program (either in lieu of the NTG methods outlined in this document or in addition to the NTG methods outlined in this document), if the following three conditions have been satisfied:

1. Evaluators must explicate within the annual evaluation research plan (or other document) the proposed alternative NTG method. Evaluators must explain why the proposed alternative NTG method might be superior to the NTG methods outlined in this document for the particular program. Evaluators must discuss the foundation for expecting that the proposed alternative NTG method is likely to produce meaningful results.

[1514] Policy Document for the Illinois Statewide Technical Reference Manual for Energy Efficiency. October 25, 2012. http://www.icc.illinois.gov/downloads/public/IL%20TRM%20Policy%20Document.pdf

2. Prior to the use of the alternative NTG method for a particular program, evaluation teams must be in agreement on the key details of the approach for implementing the alternative NTG method.
3. No objection from SAG participants is received regarding the proposed alternative NTG method for the particular program within a ten business day SAG review and comment period.

When performing alternative NTG methods for a particular program, the choice of methods may vary across the state. For example, if ComEd's evaluator chooses to test Methods 1 and 2 for a particular program, Ameren's and DCEO's evaluators do not also have to perform Methods 1 and 2 for a similar program.

### E. Procedure for Non-Consensus Items

Non-consensus items that arise during the development and updating of the IL-NTG Methods document will be handled in substantially the same way as non-consensus IL-TRM Updates are addressed. The approach to be used is as follows.

- Once the Illinois NTG Working Group[1515] has progressed as far as they can on the methodology, and it has been found that there is non-consensus on a specific Net-to-Gross Methods topic or procedure, the Illinois NTG Working Group shall submit to the ICC Staff and the Stakeholder Advisory Group's (SAG) Technical Advisory Committee (TAC) a Comparison Exhibit of Non-Consensus Net-to-Gross Methods topics/procedures *within 1 week* after the Illinois NTG Working Group has failed to reach consensus. The TAC will then deliberate on the issue with a goal of reaching consensus.

- If consensus does not emerge in the TAC regarding a particular Net-to-Gross Methods topic or procedure, the Comparison Exhibit of Non-Consensus NTG Methods topics/procedures is then sent to the full SAG for their deliberations and input. The SAG provides a forum where experts on all sides of the contested issue can present their expert opinions in an effort to inform parties of the contested issue and to also facilitate consensus.

- If the full SAG is unable to reach consensus, the non-consensus item will be referred to the ICC for resolution at the time of the IL-TRM Update proceeding. After receipt of the Comparison Exhibit of Non-Consensus Net-to-Gross Methods topics/procedures, the ICC Staff will submit a Staff Report to the Commission to initiate a proceeding separate from the consensus IL-TRM Update proceeding to resolve the non-consensus Net-to-Gross Methods topics/procedures.

[1515] The Illinois NTG Working Group consists primarily of the subset of Evaluators deliberating on NTG methodologies; however, any interested party may participate in the Illinois NTG Working Group.

## II. Attribution in Energy Efficiency Programs in General

One of the most difficult aspects of evaluation, and not just within evaluation of energy efficiency programs, is attributing results to a program. Attribution provides credible evidence that there is a causal link between the program activities and the outcomes achieved by the program. Attribution research estimates the difference between the outcomes and those that would have occurred absent the program (i.e., the counterfactual). Put in research terms, evaluators must reject the null hypothesis of no causality through probabilistic statements (e.g., "strong evidence", "high probability"). As such, it is important to realize that the concept of the counterfactual cannot be proven with certainty. So even though the NTG ratio is a single value, conceptually it is a probabilistic statement[1516]. One of the main academics within evaluation stated that there is a "…total and inevitable absence of certain knowledge [arising] from the methods social scientists use" when assessing the counterfactual. (Shadish, et al., 2002) This statement is not about poor methods, but about the counterfactual itself. Because programs work with people and are not a laboratory experiment that can be replicated over and over, to find out what actions people would have taken absent an intervention, one would need a time machine to take people back in time and not provide the program. Since time machines do not exist, evaluators have developed methods that approximate the counterfactual to the best of their ability.

For energy efficiency programs, evaluators differentiate between savings at a "gross" and "net" level as described below in the short set of relevant definitions. These definitions are not all encompassing or meant to restrict evaluation in any way, but to provide context before additional detail is provided in later sections. Research to determine attribution occurs to allow for a better understanding of the net level of savings.

**Relevant Definitions:**

| Concept | Term | Definition |
|---|---|---|
| **Consumers** | Nonparticipant | Any consumer who was eligible but did not participate in the subject efficiency program, in a given program year. |
| | Participant | A consumer that received a service offered through the subject efficiency program, in a given program year; also called *program participant*. The term "service" is used in this definition to suggest that the service can be a wide variety of inducements, including financial rebates, technical assistance, product installations, training, energy efficiency information or other services, items, or conditions. Each evaluation plan should define "participant" as it applies to the specific evaluation. |
| **Impacts** | Gross Impacts | The change in energy consumption and/or demand that results directly from program-related actions taken by participants in an energy efficiency program, regardless of why they participated. |

[1516] A probabilistic statement is not the same as the confidence and precision information calculated based on sampling theory.

| Concept | Term | Definition |
|---|---|---|
| **Attribution of Impacts** | Net Impacts | The change in energy consumption and/or demand that is attributable to a particular energy efficiency program. This change in energy use and/or demand may include, implicitly or explicitly, consideration of factors such as free ridership, participant and nonparticipant spillover, and induced market effects. These factors may be considered in how a baseline is defined (e.g., common practice) and/or in adjustments to gross savings values. |
| | Net-to-Gross Ratio | A factor representing net program savings divided by gross program savings that is applied to gross program impacts to convert them into net program impacts. The factor itself may be made up of a variety of factors that create differences between gross and net savings, commonly including free riders and spillover. The factor can be estimated and applied separately to either energy or demand savings. |
| | Free Rider | A program participant who would have implemented the program's measure(s) or practice(s) in the absence of the program. Free riders can be (1) total, in which the participant's activity would have completely replicated the program measure; (2) partial, in which the participant's activity would have partially replicated the program measure; or (3) deferred, in which the participant's activity would have partially or completely replicated the program measure, but at a future time. |
| | Spillover | Reductions in energy consumption and/or demand caused by the presence of an energy efficiency program, beyond the program-claimed gross savings of the participants. There can be participant and/or nonparticipant spillover. *Participant spillover* is the additional energy savings that occur as a result of the program's influence when a program participant independently installs incremental energy efficiency measures or applies energy-saving practices after having participated in the energy efficiency program. *Nonparticipant spillover* refers to energy savings that occur when a program nonparticipant installs energy efficiency measures or applies energy savings practices as a result of a program's influence. |
| **Markets** | Market | The commercial activity (e.g., manufacturing, distributing, buying, and selling) associated with products and services that affect energy use. |
| | Market Effects | A change in the structure of a market or the behavior of participants in a market that is reflective of an increase (or decrease) in the adoption of energy efficient products, services, or practices and is causally related to market interventions (e.g., programs). Examples of market effects include increased levels of awareness of energy efficient technologies among customers and suppliers, increased |

| Concept | Term | Definition |
|---|---|---|
| | | availability of energy efficient technologies through retail channels, reduced prices for energy efficient models, build out of energy efficient model lines, and—the end goal— increased market share for energy efficient goods, services, and design practices. |
| | Market Assessment | An analysis that provides an assessment of how and how well a specific market or market segment is functioning with respect to the definition of well-functioning markets or with respect to other specific policy objectives. A market assessment generally includes a characterization or description of the specific market or market segments, including a description of the types and number of buyers and sellers in the market, the key actors that influence the market, the type and number of transactions that occur on an annual basis, and the extent to which market participants consider energy efficiency an important part of these transactions. This analysis may also include an assessment of whether a market has been sufficiently transformed to justify a reduction or elimination of specific program interventions. Market assessment can be blended with strategic planning analysis to produce recommended program designs or budgets. One particular kind of market assessment effort is a baseline study, or the characterization of a market before the commencement of a specific intervention in the market for the purpose of guiding the intervention and/or assessing its effectiveness later. |

*Source:* Derived from State and Local Energy Efficiency Action Network. 2012. *Energy Efficiency Program Impact Evaluation Guide*. Prepared by Steven R. Schiller, Schiller Consulting, Inc., www.seeaction.energy.gov.

## III. Attribution within the Commercial, Industrial, and Public Sectors

Over thirty programs across a number of types of Commercial, Industrial, or Public Sector programs are expected to be offered in Illinois in electric program year 8 (EPY8) and gas program year 5 (GPY5) (i.e., June 2015 – May 2016). The evaluation team has worked partially through the NTG method for the Commercial & Industrial (C&I) and Public Sector Standard/Prescriptive and Custom programs. Future updates to this document will include a full NTG method for these programs as well as other programs.

### A. Standard/Prescriptive and Custom Programs

All C&I and Public Sector Standard/Prescriptive and Custom programs offered in Illinois in GPY5/EPY8 are similar enough in scope and implementation to fall under the consistent methods outlined in this section. The detail drafted below documents agreements reached by the evaluation teams through approximately 10 hours of discussion spread out over five meetings which began in October 2014 and continued through early January 2015. Additionally, evaluators spent considerable amount of time prior to official meetings delving into NTG details. Consensus reached so far pertains to the self-report approach and is documented below.

#### 1. Free Ridership

There have been several core agreements reached by the evaluation teams. These agreements should reduce potential methodological differences employed by different evaluation contractors. Each is bulleted below.

- **Multiple Questions:** Evaluators will use program participant responses to multiple survey questions as inputs to the free ridership calculation algorithm. Evaluators will not use the response to a single question to establish a survey respondent as either a complete free rider or a complete non-free rider.

- **Program and Non-Program Factors:** Evaluators will administer survey questions to obtain respondent ratings on a numeric scale of the impact, influence, or importance on the decision to implement energy efficiency measures or take energy efficiency actions. A series of questions will focus on factors that the evaluator determines are a function of the program. Such program factors may, for instance, include availability of the program incentive, technical assistance from program staff, program staff recommendations, program-administrator marketing materials, and endorsement or recommendation by utility account manager or program partner staff. Previous experience with the program is not a program factor for purposes of obtaining respondent ratings of program impact, influence, or importance on the decision to implement energy efficiency measures. Evaluators will also administer a series of questions to obtain respondent ratings on a numeric scale of the impact, influence, or importance on the decision to implement energy efficiency measures or factors that the evaluator determines are not a function of the program. Such non-program factors may include, for example, age or condition of existing equipment, previous experience with the measure, standard business or industry practice, and organizational policy or guidelines.

- **Mediation of Numeric Scales:** Evaluators will administer survey questions referencing numeric scoring scales for the purpose of quantifying free ridership. The numeric scales shall be based upon 11 points ranging from 0 to 10. Survey respondent numeric scores obtained from the administration of these questions will serve as inputs to the applied free ridership calculation algorithm. In calculating free ridership, survey respondent numeric scores may be mediated by other algorithmic components.

- **Vendor Recommendations:** Equipment vendor or contractor recommendations may also be a program factor to the extent that such recommendation is a function of the program. The evaluator may administer survey questions to vendors or contractors to verify their involvement with participant projects and to obtain respondent ratings – on a numeric scale – of the impact, influence, or importance of the program on the decision to recommend the energy efficiency measure(s) to the program participant.
- **Counterposing Program and Non-Program Factors:** Evaluators will administer a survey question that asks respondents to quantify the impact, influence, or importance on the decision to implement energy efficiency measures of factors that the evaluator determines are a function of the program relative to factors that the evaluator determines are not a function of the program.
- **Likelihood to Implement:** Evaluators will administer a survey question to obtain respondent ratings on a numeric scale of the likelihood of the respondent, in the absence of the program, to implement specified energy efficiency measures. The evaluator may administer questions to collect respondent self-report data regarding the respondent course of action, in the absence of the program, relating to the likelihood and timing of implementation, project scope, and measure characteristics.
- **Consistency Checks:** Evaluators should administer survey questions as checks on the consistency of responses associated with a core free ridership assessment methodology. Evaluators may also reference available data, including consistency check data, to perform documented modifications to individual free ridership estimates resulting from the application of a core free ridership assessment methodology.

The survey questions referenced above constitute basic guidelines for evaluators to use in the development and application of a core free ridership assessment methodology – these survey questions are not all encompassing and other survey questions may be asked by evaluators.

#### a) Scoring Algorithm

The evaluation teams have not yet reached agreement on the specific algorithm to use. There have been thoughtful discussions around the status quo algorithms, multiplying specific inputs rather than averaging them, and including partial free ridership through a very different approach of time-varying free ridership values[1517].

The evaluation teams will continue discussions in 2015 with the intent of using future evaluations to pilot the algorithms.

### 2. Spillover

Spillover has not yet been discussed by the evaluation teams in terms of reaching consensus on spillover methods. Future methods will be informed by current spillover study results.

[1517] Within time-varying free ridership, free ridership may vary over the course of measure life due to respondents' self-reported timing of implementing actions under the counterfactual scenario (i.e., absence of the program). Free ridership may also vary based on project scope and measure characteristics associated with respondents' self-reported actions under the counterfactual no-program scenario. As stated above, evaluators may, on a pilot basis, separately calculate the free ridership rate applicable to annualized first year gross energy savings and the free ridership rate applicable to gross energy savings occurring over the lifetime of implemented measures.

# IV. Attribution within the Residential and Low Income Sectors

Over 30 programs across a number of types of Residential programs are expected to be offered in Illinois in EPY8/GPY5 (i.e., June 2015 – May 2016). The evaluation team has worked partially through the NTG method for Appliance Recycling programs and Residential Upstream Lighting programs. Future updates to this document will include a full NTG method for these programs as well as other programs.

## A. Appliance Recycling Programs

Appliance recycling programs (ARPs) typically offer some mix of incentives and free pickups for the removal of old-but-operable refrigerators, freezers, or room air conditioners. These programs encourage consumers to undertake the following:

- Discontinue use of secondary or inefficient appliances;
- Relinquish appliances previously used as primary units upon their replacement (rather than keeping the old appliance as a secondary unit); and
- Prevent the continued use of old appliances in other households through direct transfers (i.e., giving it away or selling it) or indirect transfers (resale in the used appliance market).

As the program theory and logic for appliance recycling differ significantly from standard "downstream" incentive programs (which typically offer rebates for purchases of efficient products), the free ridership estimation approach also significantly differs.

There are basic and enhanced methods described next.

### *Basic Method*

#### 1. Free Ridership

Free ridership is based on participants' anticipated plans had the program not been available, thus classifying a free rider as a participant who would have removed the unit from service regardless of the program.

Estimating net savings for ARPs should adopt a multistep process to segment participants into different groups, each with specific attributable savings.

In general, independent of program intervention, participating appliances would have been subject to one of the following options:

1. The appliance would have been kept by the participating household.
2. The appliance would have been discarded in a way that transfers the unit to another customer for continued use.
3. The appliance would have been discarded in a way that would have permanently removed the unit from service.

Only Option 3 constitutes free ridership (the proportion of units that would have been taken off the grid absent the program). Options 1 and 2 both indicate non-free riders. However, these respondents need to be further classified to account for potential induced replacement and secondary market impacts, both described below.

#### a) Data Collection

A participant survey—drawn from a random sample of participants—will serve as the primary source of data collected for estimating NTG for the ARP. To determine the percentage of participants in each of the three options, evaluators will begin by asking surveyed participants about the likely fate of their recycled appliance had it not been decommissioned through the program. Responses provided by participants generally can be categorized as follows:

1. Kept the appliance.
2. Sold the appliance to a private party (either an acquaintance or through a posted advertisement).
3. Sold or gave the appliance to a used-appliance dealer.
4. Gave the appliance to a private party, such as a friend or neighbor.
5. Gave the appliance to a charity organization, such as Goodwill Industries or a church.
6. Had the appliance removed by the dealer from whom the new or replacement appliance was obtained.
7. Hauled the appliance to a landfill or recycling center.
8. Hired someone else to haul the appliance away for junking, dumping, or recycling.

Additional, follow-up questions will be included to validate the viability of all responses.

Next evaluators will assess whether each participant's final response indicates free ridership.

- Some final responses clearly indicate free ridership, such as: "I would have taken it to the landfill or recycling center myself."
- Other responses clearly indicate no free ridership, as when the appliance would have remained active within the participating home ("I would have kept it and continued to use it") or used elsewhere within the utility's service territory ("I would have given it to a family member, neighbor, or friend to use").

If the respondent planned to have the unit picked up by the retailer and the retailer would likely resell the unit in the secondary market, they are not a free rider. Absent retailer survey primary research described in the Enhanced Options below, the evaluators will utilize data from the most recent research conducted of the ComEd program to determine the proportion of free riders unless another metric is mutually agreed upon by the evaluators[1518].

[1518] Note that such retailer interviews are being conducted annually for the ComEd ARP evaluation, and answers are used directly in the calculation of the NTG ratio in cases where: (1) the respondent planned to have the unit picked up by the retailer; and (2) the retailer was interviewed.

## 2. Secondary Market Impacts

In the event that the unit would have been transferred to another household (Option 2 above), the question then becomes what purchasing decisions are made by the would-be acquirers of participating units now that these units are unavailable. These would-be acquirers could:

1. Not purchase/acquire another unit.
2. Purchase/acquire another used unit.

Adjustments to savings based on these factors are referred to as the program's secondary market impacts.

If it is determined that the participant would have directly or indirectly (through a market actor) transferred the unit to another customer on the grid, the next question addresses what that potential acquirer did because that unit was unavailable. There are three possibilities:

**A. None of the would-be acquirers would find another unit.** That is, program participation would result in a one-for-one reduction in the total number of appliances operating on the grid. In this case, the total energy consumption of avoided transfers (participating appliances that otherwise would have been used by another customer) should be credited as savings to the program. This position is consistent with the theory that participating appliances are essentially convenience goods for would-be acquirers. (That is, the potential acquirer would have accepted the appliance had it been readily available, but because the appliance was not a necessity, the potential acquirer would not seek out an alternate unit.)

**B. All of the would-be acquirers would find another unit.** Thus, program participation has no effect on the total number of appliances operating on the grid. This position is consistent with the notion that participating appliances are necessities and that customers will always seek alternative units when participating appliances are unavailable.

**C. Some of the would-be acquirers would find another unit, while others would not.** This possibility reflects the awareness that some acquirers were in the market for an appliance and would acquire another unit, while others were not (and would only have taken the unit opportunistically).

The evaluators will assume Possibility C unless primary research within a utility's service territory to assess the secondary appliance market is undertaken as described in the Enhanced Options below. Specifically, evaluators will assume that half (0.5, the midpoint of Possibilities A and B) of the would-be acquirers of avoided transfers found an alternate unit.

Once the proportion of would-be acquirers who are assumed to find alternate units is determined, the next question is whether the alternate unit was likely to be another used appliance (similar to those recycled through the program) or, with fewer used appliances presumably available in the market due to program activity, would the customer acquire a new standard-efficiency unit instead.

Again, unless primary research is undertaken as described in the Enhanced Options below for an assessment of the appliance market, evaluators will apply a midpoint approach assuming half (0.5) of the would-be acquirers of program units would find a similar, used appliance and half (0.5) would acquire a new, standard-efficiency unit.

### 3. Induced Replacement

If, however, the unit would have been kept by the participating household, the next question is whether the appliance was replaced and, if so, whether the household would have replaced the appliance regardless of the program.

The purchase of a refrigerator in conjunction with program participation does not necessarily indicate induced replacement. (The refrigerator market is continuously replacing older refrigerators with new units, independent of any programmatic effects.) However, if a customer would have not purchased the replacement unit (put another appliance on the grid) in the absence of the program, the net program savings should reflect this fact. This is, in effect, akin to negative spillover and will be used to adjust net program savings downward.

Estimating the proportion of households induced to replace their appliance should be done through participant surveys. As an example, participants could be asked, “Would you have purchased your replacement refrigerator if the recycling program had not been offered?”

Because an incentive ranging from $35 to $50 is unlikely to be sufficient motivation for purchasing an otherwise-unplanned replacement unit (which can cost $500 to $2,000), it is critical that evaluators include a follow-up question. That question should confirm the participants’ assertions that the program alone caused them to replace their refrigerator. For example, participants could be asked, “Let me be sure I understand correctly. Are you saying that you chose to purchase a new appliance because of the appliance recycling program, or are you saying that you would have purchased the new appliance regardless of the program?”

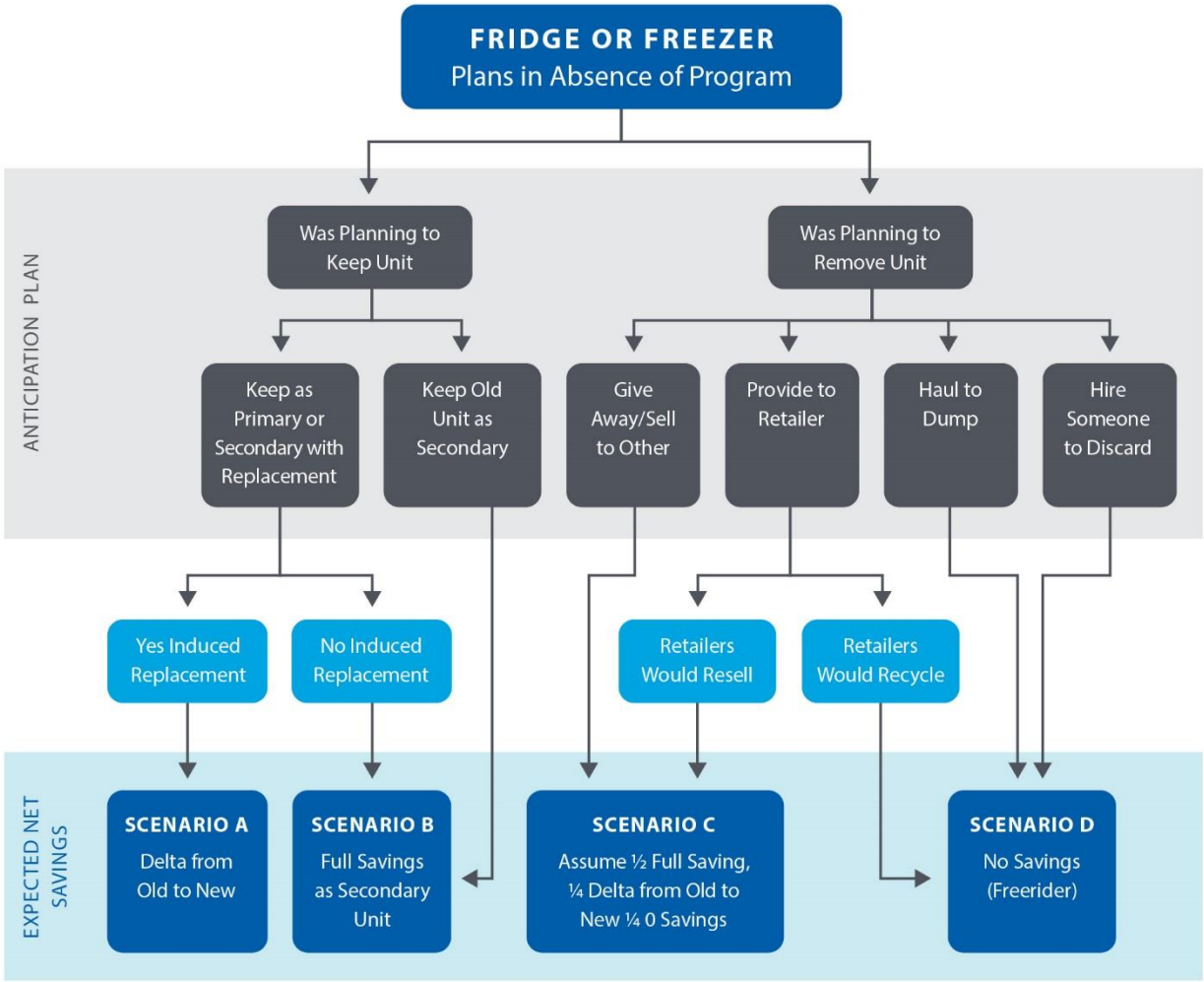
When assessing participant survey responses to calculate induced replacement, evaluators will consider the appliance recycled through the program, as well as the participant’s stated intentions in the absence of the program. For example, if customers indicate they would have discarded their primary refrigerator independent of the program, it is not possible that the replacement was induced (because it is extremely unlikely the participant would live without a primary refrigerator). Induced replacement is a viable response for all other usage types and stated intention combinations.

As one might expect, previous evaluations have shown the number of induced replacements to be considerably smaller than the number of naturally occurring replacements unrelated to the program. Once the number of induced replacements is determined, this information is combined with the energy consumption replacement appliance to determine the total energy consumption induced by the program (on a per-unit basis).

### 4. Integrating Free Ridership, Secondary Market Impacts, and Induced Replacement

The flow chart shown in Figure 3 illustrates how net savings will be derived for an ARP. As shown, below, expected savings fall into four different scenarios.

**Figure 3. Appliance Retirement Scenarios**




*Source:* Adapted from the *Pennsylvania Statewide Evaluator Common Approach for Measuring Net Savings for Appliance Retirement Programs*, Guidance Memo-026, March 14, 2014.

#### b) Scoring Algorithm

Net savings will be assigned individually to each respondent, based on responses provided to the questions discussed above. Net savings will be averaged across all respondents to calculate program-level net savings. The following equation will be used:

$$FR = (free\ ridership\ and\ secondary\ market\ impacts\ \% - induced\ replacement\ \%)$$

Table 10 demonstrates the proportion of a sample population classified into each of the seven potential categories and the resulting weighted net savings.

**Table 10. Net Savings Example for a Sample Population***

| Primary Classification | Secondary Classification | Tertiary Classification | Population (%) | UEC (kWh) w/out Program | UEC (kWh) w/ Program | kWh Savings |
|---|---|---|---|---|---|---|
| **Would have kept unit** | Scenario A: Kept but Induced Replacement | Non-ES unit | 3% | 1,026 | 520 | 506 |
| | | ES unit | 2% | 1,026 | 404 | 622 |
| | Scenario B: Kept but **NO** Induced Replacement | N/A | 25% | 1,026 | 0 | 1,026 |
| **Would have removed unit** | Scenario C: Transferred | Retailer would Recycle | 12.5% | 0 | 0 | 0 |
| | | Retailer would Resell | 12.5% | 1,026 | 520 | 506 |
| | | No Replacement | 25% | 1,026 | 0 | 1,026 |
| | Scenario D: Removed from Service | N/A | 20% | 0 | 0 | 0 |
| **Net Savings (kWh)** | | | | | | 604 |

*The percent values presented in this table serve only as examples; actual research should be conducted to determine the percentage of units falling into each of these categories. Note that Unit Energy Consumption (UEC) values presented in the table represent example values, factoring in part-use.

### *Enhanced Method*

Results can be enhanced by including three additional research efforts. The basic method has defaults where primary research on enhanced approaches cannot be performed:

A retailer survey, to determine the quantity and/or proportion of units returned to a retailer, and that the retailer would deconstruct or recycle. Through this survey, one would determine a retailer's criteria for reselling used units vs. deconstructing them, based on unit age and condition. Results from the survey and analysis would be used to determine the proportion of those who would have returned an old appliance to the retailer that should be included in Scenario D (free riders). This research was conducted for ComEd in EPY6 evaluation and those results were applied to Ameren.

An appliance market assessment study, to determine the size of the secondary appliance market and whether removal of participating units from the market would cause an otherwise would-be receiver to purchase an alternative used or new unit. Savings attributable to these participants are the most difficult to estimate, as the scenario attempts to estimate what the prospective buyer of a used appliance would do in the absence of finding a program-recycled unit in the marketplace (i.e., the program took the unit off the grid, so the prospective purchaser faced, in theory, a

smaller supply of used appliances). It is difficult to answer this question with certainty, absent utility-specific information regarding the change in the total number of appliances (overall and used appliances specifically) that were active before and after program implementation. In some cases outside of Illinois, evaluators have conducted in-depth market research to estimate both the program's impact on the secondary market and the appropriate attribution of savings for this scenario. Although these studies are imperfect, they can provide utility-specific information related to the program's net energy impact. Where feasible, evaluators and utilities should design and implement such an approach. Unfortunately, this type of research tends to be cost-prohibitive, or the necessary data may simply be unavailable.

However, it is possible to estimate through nonparticipant surveys which of the disposal responses given by nonparticipants were most likely to have been to an opportunistic would-be-acquirer. Transfers would most likely have been opportunistic are determined primarily based on the cost to the recipient. If the appliance was sold or transferred to a retailer, there would have been a cost to the recipient of that appliance. If the recipient was willing to pay for the appliance or was willing to exert the effort to visit a retail location, this suggests the recipient was actively seeking an appliance. However, if the unit were given away for free there was little cost to the recipient and is a reasonable proxy for the proportion of opportunistic acquirers. This proportion would replace the 50% default assumption (scenario C in Figure 3) of would-be-acquirers that would or would not find an alternate unit.

A nonparticipant survey can be used to assess how nonparticipants acquire and dispose of used units. As nonparticipants do not have the same perceived response bias as participants, they can help offset some of this potential bias in estimating the true proportion of the population that would have recycled their units in program's absence. The evaluators will average the results of the nonparticipant survey with the participant survey if the nonparticipant survey is of sufficient sample size. Otherwise, results may be used for a qualitative characterization of potential bias. Though recommended, use of a nonparticipant survey need not be required, given budget and time considerations. A nonparticipant survey was completed as part of ComEd's EPY6 evaluation and used qualitatively to validate participant results.

### 5. Participant Spillover

Unlike many programs, recycling programs face reduced opportunities for spillover due to the lack of general energy education and the small likelihood of participants having further units to recycle on their own. This program could directly impact decisions to replace refrigerators or freezers with ENERGY STAR units rather than standard efficiency units, given that the program offers marketing and education related to the operating costs of refrigerators and freezers. Reliable methods of conducting this analysis have yet to be developed. One attempted method compared proportions of ENERGY STAR appliances replaced by program participants to proportions of ENERGY STAR new appliance shipments in a similar area. Due to the difficulty in isolating the shipment area to the program area, this has not yielded noticeable spillover in Illinois.

### 6. Nonparticipant Spillover

The specific approach and method for measuring spillover has not yet been discussed by the evaluation teams to reach a consensus. However, effective program marketing and outreach generates program participation and increases general energy-efficiency awareness among customers. The cumulative effect of sustained utility program marketing (which often occurs concurrently for multiple programs) can affect customers' perceptions of their energy usage and, in some cases, motivates customers to take

efficiency actions outside of the utility's program. This phenomenon—called nonparticipant spillover (NPSO)—results in energy savings. Marketing of the Appliance Recycling program specifically may induce nonparticipants to either reduce the use of the secondary refrigerator or freezer that they keep, or when they are purchasing a new refrigerator or freezer, to buy one that is more energy efficient.

### B. Residential Upstream Lighting Programs

The Illinois Residential Upstream Lighting programs to date have provided discounts on efficient lighting through retailers at the point of purchase. Such programs often remain transparent to customers purchasing incentivized lighting. Program administrators also do not know the identity of most customers purchasing the program-discounted lighting; so these customers cannot easily be contacted once they leave the store for a traditional self-report net-to-gross (NTG) evaluation survey (i.e., an after-the-fact, direct solicitation of customers regarding what they would have done in the program's absence). Similar surveys can be conducted with customers within program retailers after they have made their lighting purchasing decision but before they leave the store. For programs such as this, in store customer surveys are preferable to the traditional self-report telephone surveys that ask customers to recall their past light bulb purchases. Light bulbs are a small and relatively insignificant purchase for most people thus the recall bias could be substantial.

Further, as upstream programs work with multiple market actors and can include wide-reaching marketing campaigns promoting energy efficiency to the general public, they tend to stimulate spillover and "market effects." As a result, estimating NTG for upstream residential lighting programs can be challenging. Multiple methods exist, each with their own strengths and weaknesses.

Ameren and ComEd implement their residential lighting programs comparably, and the evaluation teams have used a consistent primary NTG evaluation method. This section details the consensus NTG methodology, which has been used multiple times for both ComEd and Ameren and is considered the most well-vetted and defensible NTG method that has been successfully used in Illinois.

For EPY5 and EPY6, Ameren and ComEd used a customer self-report methodology to estimate NTG for their upstream residential lighting programs.[1519] Customer self-report data in this method are collected during surveys conducted within program retailers with customers purchasing program bulbs (i.e., in-store intercept surveys). This method separately estimates free ridership, participant spillover, and nonparticipant spillover. Details follow on the primary data collection and scoring algorithms.

#### 1. Free Ridership

Free ridership is the proportion of program bulbs that would have been purchased if the program did not exist. Three alternative scenarios could occur:

1. Full Free Rider: The customer would have purchased the same quantity of efficient bulbs (CFLs or LEDs) in the program's absence.
2. Partial Free Rider: The customer would have purchased fewer efficient bulbs (CFLs or LEDs) in the program's absence.
3. Non-Free Rider: The customer would have not purchased any efficient bulbs (CFLs or LEDs) in the program's absence.

[1519] ComEd has used this method since EPY2. Ameren began using it in EPY5.

Free ridership is calculated as the average of two distinct scores: a program influence score and a non-program score. These scores are defined as follows:

1. The *program influence score* captures the maximum level of program influence, reported by a survey respondent, of the residential lighting program on their decisions to purchase program bulbs on the day of the survey. This program influence can take a number of forms, such as: the monetary incentive provided to decrease the cost of high-efficiency bulbs; program-sponsored educational materials that explain the benefits of efficient lighting; in-store product placement of efficient bulbs; and program bulb recommendations provided by retail store personnel.
2. The *non-program score* is used to estimate how many program bulbs a survey respondent would have purchased in the absence of the residential lighting program.

**a) Data Collection**

To estimate free ridership, the evaluation teams will conduct in-store intercept surveys with customers purchasing program-discounted lighting at participating retailers. Customers are asked questions that are used to estimate a program influence score and a non-program score for each customer and efficient bulb type purchased.

**Primary Program Influence Score Questions**

1. Light bulb purchasing plans for current shopping trip (Yes/No)
2. If planning to purchase bulbs:
   a. Bulb type (CFL, LED, Incandescent, Halogen)
   b. Utility-incentivized bulbs (Yes/No)
3. Influence of various program factors:
   a. Program incentive
   b. In-store information (printed materials or information from utility representatives or retail personnel)
   c. Positioning of discounted bulbs within the store

**Primary Non-Program Score Questions**

1. Stated preference of light bulb purchases had the utility incentive not been available (purchase all, some or none of efficient bulbs)
2. Quantity of light bulbs purchased absent the utility incentive

**b) Scoring Algorithms**

Using the data collected from program participants during the in-store intercept surveys, program influence and non-program scores are calculated for each survey respondent and then combined to estimate a respondent-specific free ridership score.

**Calculation of the Program Influence Score:**

Survey respondents purchasing one or more program-discounted bulbs are assigned a preliminary program influence score based on the maximum program influence level (on a 0 to 10 scale) they assigned to one or more program factors (e.g., monetary incentive/informational materials (printed or from store personnel)/product positioning). The influence level assigned to the monetary incentive

should be increased for survey respondents (using a linear decreasing function[1520]) who indicated that absent the incentive they would not have purchased any of the program bulbs they were purchasing that day.

After the preliminary program influence score is assigned, a secondary algorithm is run that adjusts the preliminary program influence based on survey data regarding the customers purchasing plans when they entered the store. Survey respondents who indicate they planned to purchase high-efficiency bulbs prior to entering the store and had who not come to the store specifically to buy utility-incentivized program bulbs, should have their program influence score cut in half. This adjustment makes the final program influence score reflective of their stated planned intention to purchase efficient bulbs in the program's absence.

**Calculation of the Non-Program Score:**

The non-program score is based on whether a respondent states they would have purchased all, some, or none of the program-discounted bulbs in the absence of utility incentives. Respondents reporting they would have purchased all of the efficient bulbs without the incentive should be considered free riders and receive a non-program score of zero. Those reporting they would have purchased none of the efficient bulbs without the incentives should be classified as non-free riders and receive a non-program score of 10, the maximum. Respondents reporting they would have purchased some of the efficient bulbs without the incentive should be assigned a non-program score between 0 and 10, reflective of the percentage of efficient bulbs they would not have purchased absent the program.

Respondents reporting they would have purchased all of the program-discounted bulbs in the program's absence, but in-store materials provided by the utility had a moderate to high influence on their decision should have their non-program scores adjusted to equal the level of influence they attributed to these program-sponsored informational materials.

**Calculation of Free Ridership:**

$$\text{Free Ridership} = 1 - (\text{Program Influence Score} + \text{Non-Program Score})/20$$

Using the calculated program influence and non-program scores, free ridership is calculated as one minus the sum of the two scores (program influence score plus non-program score), divided by 20. Dividing the sum of scores by 20 results in a ratio (between 0 and 1) that is representative of the average of the two zero to 10 scores. Subtracting this ratio from one reverses the score, thus representing the free ridership level. If either the non-program or program influence scores are missing, free ridership can be calculated using the single available score divided by 10. Evaluators may also reference available data to perform documented modifications to individual free ridership estimates resulting from the application of a core free ridership assessment methodology.

[1520] The function, adjusted monetary score = (monetary score + 10)/2, increases the monetary score using a decreasing linear function. This function results in an increase in the monetary influence score of between 0 and 5 points depending on their original monetary score (i.e., an original score of 0 would become a 5, a 5 would become a 7.5, and a 10 would remain a 10. In past Illinois evaluations, this adjustment has typically changed less than 10% of all monetary scores.

### 2. Participant Spillover

Participant spillover results from purchases of non-discounted efficient bulbs by program bulb purchasers who are influenced by their participation in the residential lighting program to purchase additional non-discounted efficient bulbs.

#### a) Data Collection

Data collected during in-store intercept surveys with customers purchasing program bulbs should be used to estimate participant spillover. During these surveys, customers purchasing program-discounted and non-discounted efficient bulbs should be asked questions to determine whether the residential lighting program influenced their purchases of non-discounted efficient bulbs.

#### b) Scoring Algorithm

To estimate participant spillover, the number of program-influenced, non-discounted efficient bulbs purchased by program participants is divided by the total number of program bulbs purchased by these program participants. This results in the participant spillover rate.

### 3. Nonparticipant Spillover

Nonparticipant spillover results from purchases of non-discounted efficient bulbs by customers who are not purchasing program-discounted bulbs, but report that the residential lighting program influenced their decision to purchase non-discounted efficient bulbs.

#### a) Data Collection

Data collected during in-store intercept surveys with customers purchasing efficient bulbs not discounted by the program should be used to estimate nonparticipant spillover. During these surveys, customers purchasing non-discounted efficient bulbs should be asked questions to determine whether the residential lighting program influenced their purchases of non-discounted efficient bulbs.

#### b) Scoring Algorithm

To estimate nonparticipant spillover, one must first calculate the number of program-influenced, non-discounted efficient bulbs purchased by the population of program nonparticipants surveyed. This yields a survey nonparticipant spillover rate. This rate is then extrapolated to the estimated population of nonparticipating utility customers to determine the estimated total quantity of non-program efficient bulbs being purchased within the utility service territory. Dividing this result by the total number of program bulbs results in the nonparticipant spillover rate.

### 4. Method Advantages and Disadvantages

The in-store intercept method described above has certain advantages and disadvantages.

**Advantages:** This approach catches customers at their point of purchase, before they leave the store and can no longer be contacted directly. Given the interview's timing, customers can more easily recall price factors leading to their purchase choices. Also, as customers are intercepted at the store rather than surveyed by telephone, a higher cooperation rate results.

**Disadvantages:** Customers may not fully connect the impact that in-store education, product placement, and advertising have on their decision making. While many consumers believe they are not influenced by advertising, retailers know advertising and product placement work. Further, store intercepts typically must be coordinated with education events, and many retailers do not allow interviews to take place in their stores. Consequently, results are not based on random samples of

customers purchasing program-discounted lighting throughout the year and across all participating retailers, which could bias the results.

# V. Appendix A: Overview of NTG Methods

The evaluation teams present information in this appendix to provide a relatively quick overview of NTG methods for readers unaccustomed to the possible methods that evaluators may deploy. It is not meant to be a complete or deep discussion about each of the methods presented. However, the evaluators in Illinois considered the inclusion of this appendix to be very important in acknowledging the current suite of methods deployed by evaluators throughout the U.S. and giving a framework for work within Illinois.

Much of the information shown below is taken directly from a single source — the national Uniform Methods Project, Chapter 17: Estimating Net Savings: Common Practices. (Violette and Rathbun, 2014) This document has done a nice job of summarizing the eight most common attribution methods currently in use across the U.S. The evaluation teams recommend that readers go first to this reference for further information. Additionally, while there are slightly over 100 references within the Violette and Rathbun document, other non-duplicative references are included where reasonable as additional resources for those interested in further research into any specific method.

## A. Survey-Based Approaches

Virtually all Illinois based evaluations use a survey-based approach for programs where primary data is used to determine net savings. (The main exception is for Behavioral programs which use statistical analysis based on a randomized control trial program design.) Survey based approaches obtain data from program participants and nonparticipants using a structured data collection instrument implemented via phone, in person or on-line. At times, evaluators create and use an unstructured depth-interview guide to collect information about attribution and this provides both contextual data and quantitative data about a given project.

### 1. Self-Report Approach

The self-report approach relies on the abilities of customers to discuss the program influence as well as the somewhat abstract ideas of the counterfactual (i.e., what would have occurred absent the program) after making a choice to purchase an energy efficient item or take an energy efficient action unrelated to a purchase. For program participants, this could include doing nothing (i.e., leaving the existing equipment as-is), installing the same energy efficient equipment as they did through the program, or an intermediate step of installing equipment that is more efficient than what they had in place previously, but less efficient than what they installed through the program . Evaluators also use this approach when collecting information from trade allies or distributors. This self-report approach is not new, nor is it exclusively used by the energy efficiency industry. An important attribute of this approach is its reliance on well-designed and fielded survey questions, so that the data underlying subsequent analyses are accurate and complete.

The output of this approach is a NTG ratio which can be considered an index of the program's influence on the decision to install energy efficient equipment. The NTG ratio is applied to gross savings in order to obtain an estimate of net savings. The NTG ratio may include free ridership, spillover, or market effects, depending on the survey and analytical design. NTG ratios may be calculated at the measure, suite of measures, or program level and are typically average values weighted by savings. If sufficient information is available, analysis of NTG ratios among certain customer segments may be done to further inform changes to program design.

**References**

- Sudman, 1996
- Stone, et al., 2000
- Bradburn, et al., 2004

### 2. Econometric/Revealed Preference Approach

The econometric/revealed preference approach, while still considered a survey approach due to how data is collected, moves beyond asking people about the counterfactual and instead uses the observations of the evaluator to collect information for analysis of a NTG ratio. Within this approach, evaluators typically deploy similar sampling designs as for the self-report approach to collect data, but actively gather what a person is doing (i.e., what is being purchased in a store) to determine attribution.

## B. Randomized Control Trials (RCT) and Quasi-Experimental Designs

As mentioned earlier, evaluators deploy an RCT for estimating savings from the Behavioral programs within Illinois. Additionally, quasi-experimental designs (QED) have been used in the past in Illinois to estimate net savings from the upstream CFL program, and CFL, insulation and air sealing measures within the Home Performance with Energy Star program.

RCT and QED use statistical analysis to determine regularities within the data that reveal net savings due to a program intervention[1521]. The analytical design attempts to control for factors that can confound net analysis.[1522] When estimating net savings within both an RCT and QED, two groups are included within the analysis: 1) a group that has been exposed to (i.e., treated by) a program and 2) a group that has not been exposed to the program. Evaluators must carefully consider the choice of the non-exposed group (called a control group for RCTs or comparison group for QEDs).

RCT – This design must be integral to a program's implementation. Without the ability to randomly assign customers to one group or another (or at least randomly encourage customers to participate in a program), the ability of the design to yield unambiguous estimates of net impacts is compromised. Evaluators often help design how a program is implemented and, if not involved at the outset, carefully review choices made by the implementation team.

QED – A QED may be designed after a program has been implemented. It relies on determination of an equivalent comparison group, which is often chosen based on energy use. QED is difficult to perform well within the commercial sector due to the heterogeneity of end uses within the sector.

The output of an RCT or QED is the average net savings for the population within the statistical model. Evaluators may also analyze the data to help understand the savings within specific known segments if sufficient information and data points are available.

**References**

- Mohr, 1995
- Shadish, Cook, Campbell, 2002

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[1521] Net savings are calculated when a comparison or control group of non-treated customers are part of the design. Statistical analyses can also obtain gross savings.

[1522] Economists strongly support this approach, but among program evaluators, the idea that an RCT is a "gold standard" for attribution research has been hotly debated for decades.

- Scriven, 2008
- Donaldson, 2009

### C. Deemed or Stipulated NTG Ratios

A deemed (or stipulated) NTG ratio is a value known prior to implementing a program and applied to estimate net savings for that program in a certain year.

Deemed or stipulated NTG ratios may be based on previous primary data collection, review of secondary data, or agreed to among stakeholders. In Illinois, deemed or stipulated NTG ratios should reflect best estimates of likely future actual NTG ratios for the relevant program year, taking into consideration stakeholder input, the evaluator's expertise, and the best and most up-to-date information.

### D. Common Practice Baseline Approaches

For this method, the evaluation team estimates what a typical consumer would have done at the time of the project implementation. Essentially, what is "commonly done" becomes the basis for baseline energy consumption and calculation of net savings. No gross impacts are calculated in this approach. This baseline is defined as the counterfactual "i.e., what would have occurred absent the program" and has been referred to as current practice, common practice or industry standard practice. Evaluators determine these practices through multiple methods, but often can be from self-report or on-site audits. The difference between the energy use of measures installed in the program and the energy use associated with current practice is considered by some to be sufficiently close to the net savings.

This approach is not in use in Illinois, but is used elsewhere in the country such as the Pacific Northwest and Delaware.

### E. Market Analyses

Market analyses can be done in several ways. Market analyses are often used in theory-driven evaluations of market transformation programs.

Other non-sales data market analyses can be postulated on changes specified in program logic such as: 1) changes in the number of energy efficient units manufactured, 2) changes in market actor behavior around promotion or stocking of energy efficient items, or 3) reduction in prices. The analyses involving non-sales data must make a clear link between the program intervention and the changes found in the market. Additionally, outside of Illinois, while evaluators have extrapolated the market changes to specific energy or demand reductions, this activity may be viewed as tenuous due to assumptions that evaluators must make within the analysis.

Illinois is in a position to begin to discuss market analyses and how specific research may be able to interpret changes that have occurred (or may occur in the future) because of the IOU interventions over the past six years. Market analyses can be backward looking through historical tracing, but is best used when the logic of an intervention is described and specific market metrics are tracked over time. This is a switch from the current annual evaluation of programs and has challenges that stakeholders would need to discuss and reach a consensus on an approach that works for Illinois.

### F. Structured Expert Judgment Approaches

Closely tied to market analysis, this approach is a way for evaluators to gather credible evidence of changes that arise due to the intervention of a program. When deployed, it is often used as a cost-effective approach to estimate market effects or reach agreement on a NTG value when several different types of evidence are available. . The key premise of this approach is the use of a select group of known experts that all stakeholders agree can provide unbiased information as well as having sufficient knowledge to judge what may have occurred absent a program intervention.

A Delphi Panel is an example of this approach where data is collected from two or more rounds of data collection (which can occur via email, internet, or in-person). A round is when experts make their thoughts known about a specific subject, the evaluation team synthesizes the data and provides this collated data back to the group to discuss again. Allowing the full experts to see how their peers think about a topic helps to move the group towards consensus.

To date, in Illinois, there has been little need for this approach. However, if more market analyses occur in the future, this is a valuable tool that can be deployed.

**References**

- Mosenthal, et al., 2000
- Powell, 2002

### G. Program Theory-Driven Approach

This approach is not included in the Violette and Rathbun (2014) document as a high level method, but is discussed by the authors under the historical tracing method. The Illinois evaluators believe that it deserves at least a short discussion within this framework.

A program theory is the written narrative about why the activities of a program are expected to bring about change. Typically associated with this approach is the direct graphical explication of the linkages between activities, outputs, and outcomes through an impact logic model.[1523]

A theory-driven evaluation denotes "[A]ny evaluation strategy or approach that explicitly integrates and uses stakeholder, social science, some combination of, or other types of theories in conceptualizing, designing, conducting, interpreting, and applying an evaluation." (Coryn 2011) Within this approach, the ultimate conclusions regarding the efficacy of a program are based on the preponderance of the evidence and not on the results of any single analysis. Coryn and colleagues systematically examined 45 cases of theory-driven evaluations published over a twenty-year period to ascertain how closely theory-driven evaluation practices comport with the key tenants of theory-driven evaluation as described and prescribed by prominent theoretical writers. One output from this analysis was the identification of the core principles and sub-principles of theory-driven evaluation. If interested, please review the reference under Coryn 2011.

As an approach, it is best used for complex programs and/or causal mechanisms that extend far into the future. Evaluators collect evidence that supports or rejects hypotheses that are explicit in the logic model. The case for program attribution is strengthened based on the extent to which an evaluation

[1523] Evaluators may use logic models to show program processes as well, but this is a program flow chart, not an impact model.

shows that the expected changes occur. Additionally, the evaluation team may be able to collect data that will answer questions about the longer term outcomes of a program. This type of data collection may be very similar to market tracking activities described briefly above under Market Analyses.

This approach does not specifically estimate a NTG value, but program administrators can choose to keep, drop or change a program based on intermediary data. Regulators must be convinced that the logic of a program is sound and that the intermediary outcomes are causally linked to expected savings.

**References**

- Weiss, 1997
- Chen, 2000
- Coryn, 2011

### H. Case Studies Design

Case studies are used extensively in social sciences as well as many other disciplines or practice-oriented areas such as political science, economics, education, and public policy. Case studies help to understand the how and why of a situation and typically retain a holistic aspect of real-life events. As such, they may be a useful approach to determine attribution. As with program theory design, though, the data collected and analyzed within a case study approach will not typically yield a specific NTG value, but can provide credible evidence and insight that supports or refutes the changes brought about by program intervention.

To be used to assess attribution, evaluators must carefully design case studies to assure they account for the threats to causality (i.e. internal validity) that arise in any design. While not typically thought of in this manner, case study design can address multiple types of validity such as construct, internal and external validity as well as assuring reliability. When establishing construct validity and reliability, evaluators must use multiple sources of evidence, create and maintain a study database, and maintain a "chain of evidence" within the analysis. Internal validity is shown through analytic tactics such as pattern matching, explanation building, addressing rival explanations, or using logic models. External validity centers on the ability to generalize the analytical findings to other similar situations. External validity may be shown through replication of findings.

**References**

- Yin, 2003
- Stake, 2006

## VI. Appendix B: References

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